

# Temperature wall function adaptation for mixed convection airflows: LBM-LES simulations of a hot wall jet developing in a room

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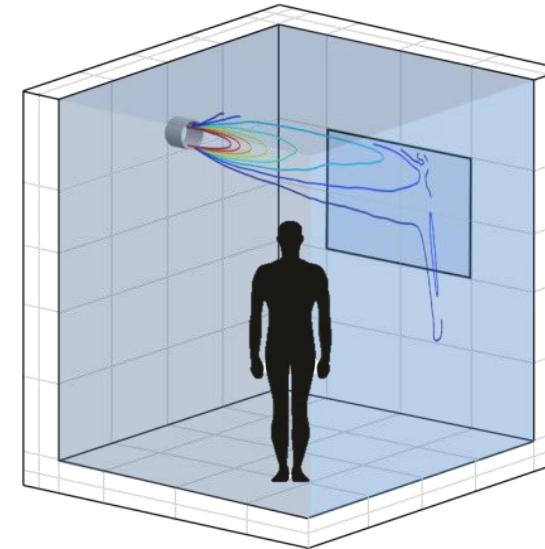
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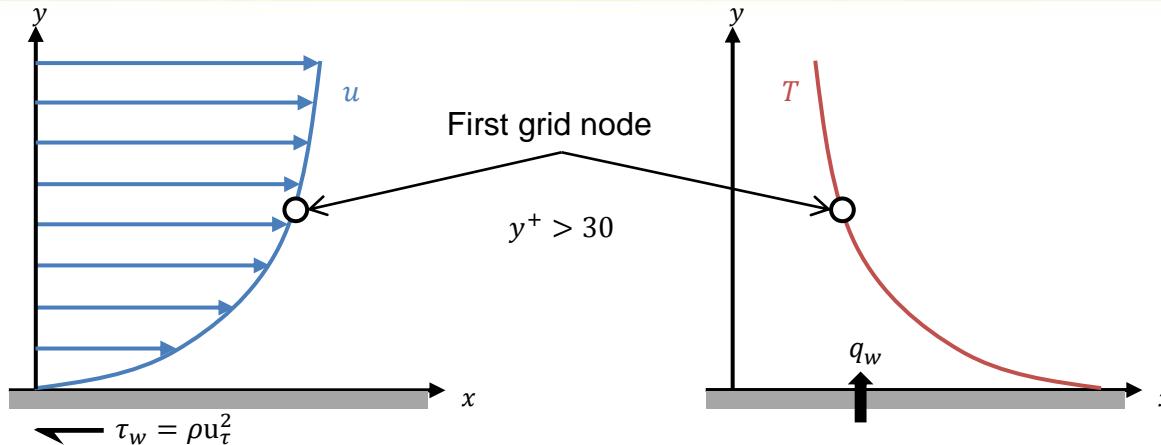
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# Introduction

- Indoor airflows characteristics
    - Wall bounded interactions, vortices, buoyancy
    - Turbulent, transient and laminar regions
    - Low velocities, low  $Re$
  - Challenges for CFD
    - Turbulence modeling
    - Buoyancy modeling
    - Near-wall modeling
  - Limits of standard wall functions
    - Developed for high  $Re$  forced convection flows
    - Underestimate the convective heat transfer for mixed convection indoor airflows ([X. Yuan and al. 1999](#))
- How can the wall functions be adapted ?



# Standard wall functions



- One-dimensional boundary layer (BL) equations

Momentum:  $v \frac{\partial u}{\partial y} = \nu \frac{\partial^2 u}{\partial y^2}$

Energy:  $v \frac{\partial T}{\partial y} = \alpha \frac{\partial^2 T}{\partial y^2}$



$$\begin{cases} u^+ = \frac{1}{\kappa} \ln(y^+) + B \\ T^+ = Pr_{t,w}(u^+ + P) \end{cases}$$

$$\begin{aligned} y^+ &= \frac{yu_\tau}{v} \\ u^+ &= \frac{u}{u_\tau} \\ T^+ &= \frac{(T_w - T)\rho c_p u_\tau}{q_w} \end{aligned}$$

Note: no adverse pressure gradient and friction

# Adaptation for mixed convection flows

## 1. Adjustment of the turbulent Prandtl number $Pr_{t,w}$

Outdoor ([T. Defraeye and al 2012](#))  
 Indoor ([T. Zhang and al. 2013](#))

$$q_w = \frac{(T_w - T) \rho c_p u_\tau}{Pr_{t,w} \left[ \frac{1}{\kappa} \ln(y^+) + B + P \right]}$$

→ Increasing  $q_w$  by decreasing  $Pr_{t,w}$

Limits: no physical basis, lack universality

## 2. Integration of the buoyancy effect in the BL equations ([B. Chen and al. 2021](#))

Momentum:  $v \frac{\partial u}{\partial y} = v \frac{\partial^2 u}{\partial y^2} + g\beta(T_w - T)$



Energy:  $v \frac{\partial T}{\partial y} = \alpha \frac{\partial^2 T}{\partial y^2}$

$$\begin{cases} u^+ = \frac{1}{\kappa} \left( 2\sqrt{1 - y^+ T^+ S^+} - \ln \frac{1 + \sqrt{1 - y^+ T^+ S^+}}{1 - \sqrt{1 - y^+ T^+ S^+}} \right) + C_u \\ T^+ = \frac{1 - \sqrt{1 - 2y^+ S^+ Pr_t(u^+ + C_t)}}{y^+ S^+} \end{cases}$$

$$S^+ = \frac{g\beta v q_w}{\rho c_p u_\tau^4}$$

$$y^+ T^+ S^+ = \frac{y g \beta (T_w - T)}{u_\tau^2}$$

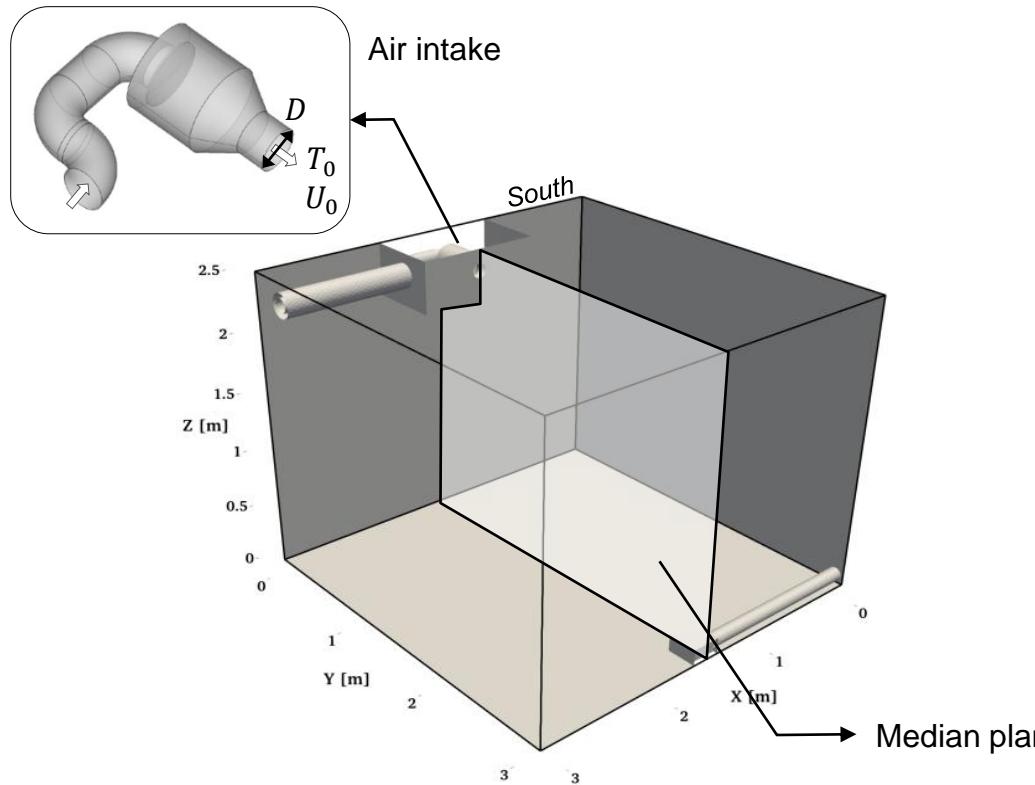
$$y^+ S^+ = \frac{y g \beta q_w}{\rho c_p u_\tau^3}$$

Limits: - only for horizontal walls

- strong validity conditions  $0 < y^+ T^+ S^+ < 1$

$$2y^+ S^+ Pr_t(u^+ + C_t) < 1$$

# MINIBAT test-cell (Kuznik 2005)



**Experimental conditions: hot jet**

- Inlet

| $T_0$ [ $^{\circ}$ C] | $Re_D$ | $U_0$ [m/s] |
|-----------------------|--------|-------------|
| 30.9                  | 21600  | 2.70        |

- Wall temperatures [ $^{\circ}$ C]

| South | North | East | West | Floor | Ceiling |
|-------|-------|------|------|-------|---------|
| 24.4  | 25.0  | 24.6 | 24.7 | 24.5  | 25.5    |

# CFD modeling



## Thermal LBM

- LBM-BGK equation (velocity and density)

$$f_i(x + c_i \delta t, t + \delta t) = f_i^{eq}(x, t) + \left(1 - \frac{1}{\tau}\right) f_i^{neq}(x, t)$$

With  $\tau = \frac{\nu}{c_s^2} + \frac{\Delta t}{2}$  the relaxation time

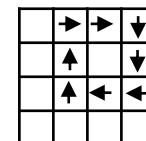
$f_i^{eq}$ : 3rd order equilibrium distribution function

$f_i^{neq}$ : Hybrid Recursive Reconstruction procedure ([Jacob and al. 2018](#))

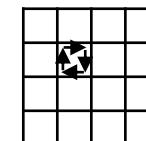
- Energy equation (temperature)  
→ FV method
- Boussinesq source term

## LES

- Principle:



Resolved



Modeled

- Eddy viscosity paradigm:

$$R_{ij} = \overline{u'_i u'_j} \cong -2\nu_t S_{ij} + R_{kk} \delta_{ij}$$

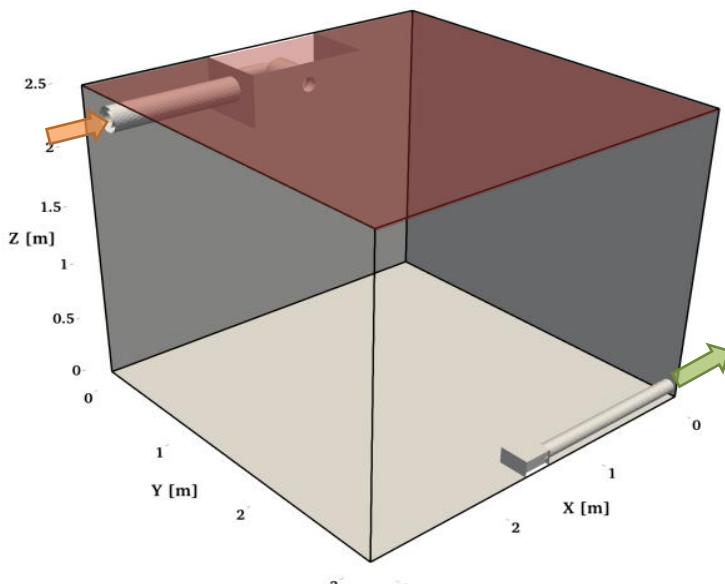
- $\nu = \nu_{mol} + \nu_t$  in the relaxation time

## Boundary conditions

- Immersed boundary approach
- Macroscopic quantities at boundary nodes:
  - Dirichlet: interpolation
  - Neumann: extrapolation
  - Wall functions
- Computation of the velocity gradients on boundary nodes
- Reconstruction of the distribution functions

→ Need validation for application to indoor airflows with thermal effects

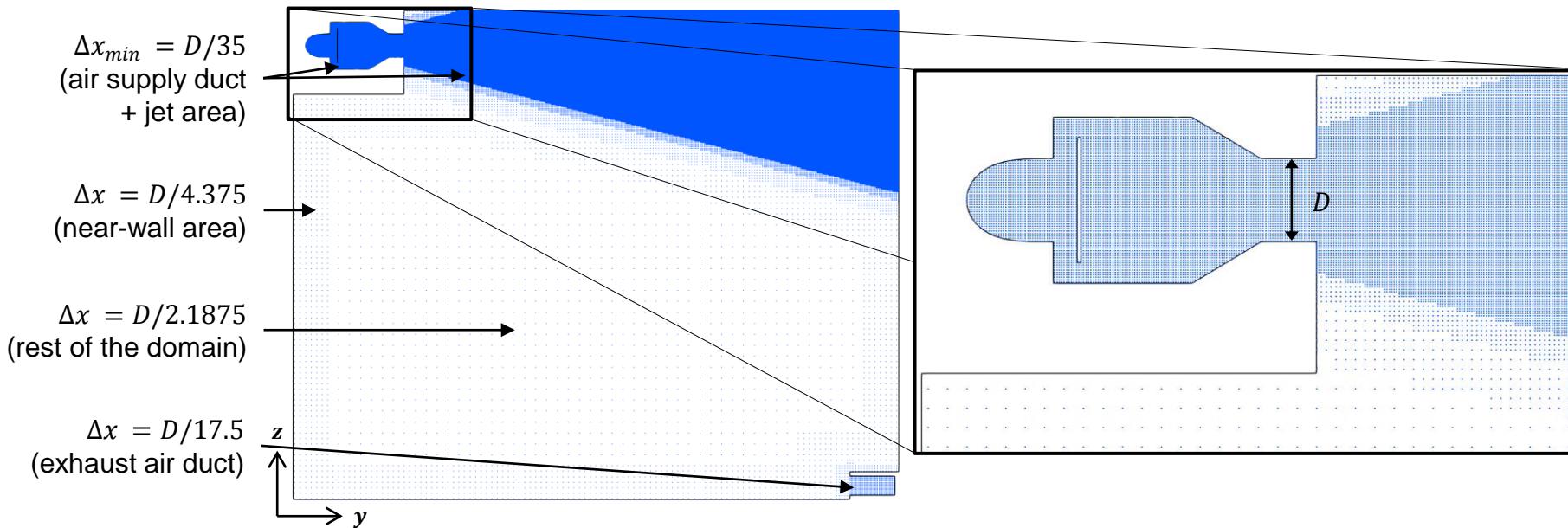
# Boundary conditions



|                                  | <b>Inlet</b><br>Dirichlet velocity<br>Neumann density | <b>Outlet</b><br>Dirichlet density<br>Neumann velocity |
|----------------------------------|---|--|
| <b>Walls</b>                     | <p><b>Ceiling</b></p>                                 | <p><b>Other walls</b></p>                              |
| <i>Eddy viscosity modelling</i>  |   |  |
| <i>Velocity wall function</i>    | Explicit log law<br>(Cai and Sagaut 2021)             | Implicit log law<br>(Afzal 1996)                       |
| <i>Temperature wall function</i> | Log law (Kader 1981)<br>with $Pr_{t,w} = 0.2$         | Log law (Kader 1981)<br>with $Pr_{t,w} = 0.025$        |

$v_t^RANS$ : mixing length model /  $v_t^{LES}$ : Vreman's SGS model ( $C_{smag} = 0.1$ )

# Computational grid




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Number of elements

$31.2 \times 10^6$  nodes

Note: Steady state after  $75T^*$

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timestep

$3.37 \times 10^{-5}$  s

Statistical averaging over  $15T^*$

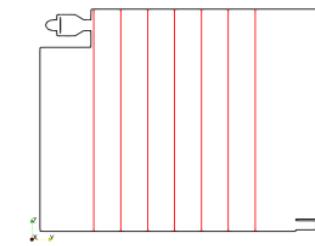
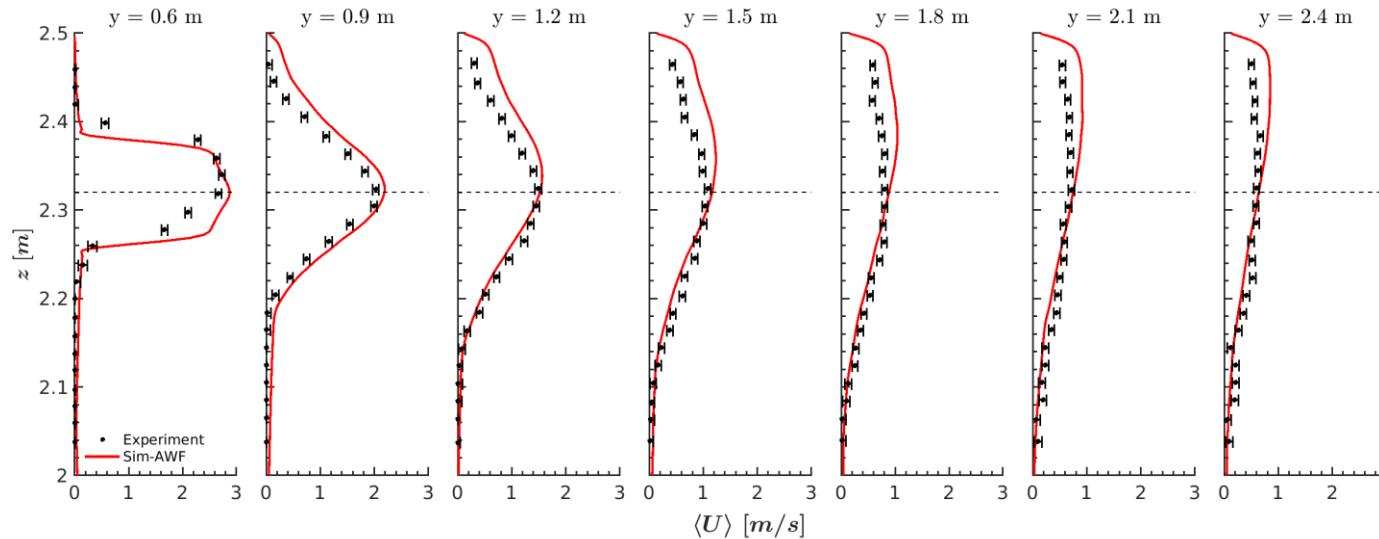
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Calculation cost

$1.7 \times 10^5$  h<sub>CPU</sub> over 816 processors

$T^* = L_c/U_c = 4.65/2$  s

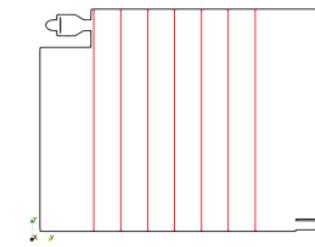
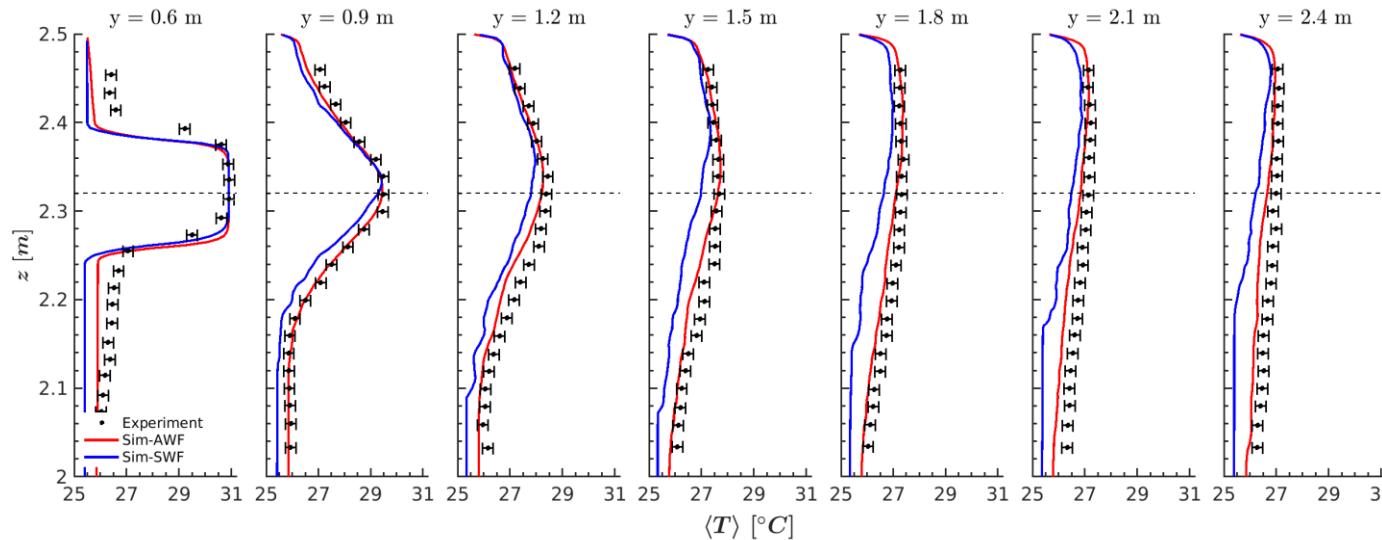
# Mean velocity profiles on median plan



*AWF: Adjusted wall function*

- Good agreement with experimental profiles
- Deviation close to the ceiling < 0,4 m/s  
→ Underestimation of the shear stress, buoyancy

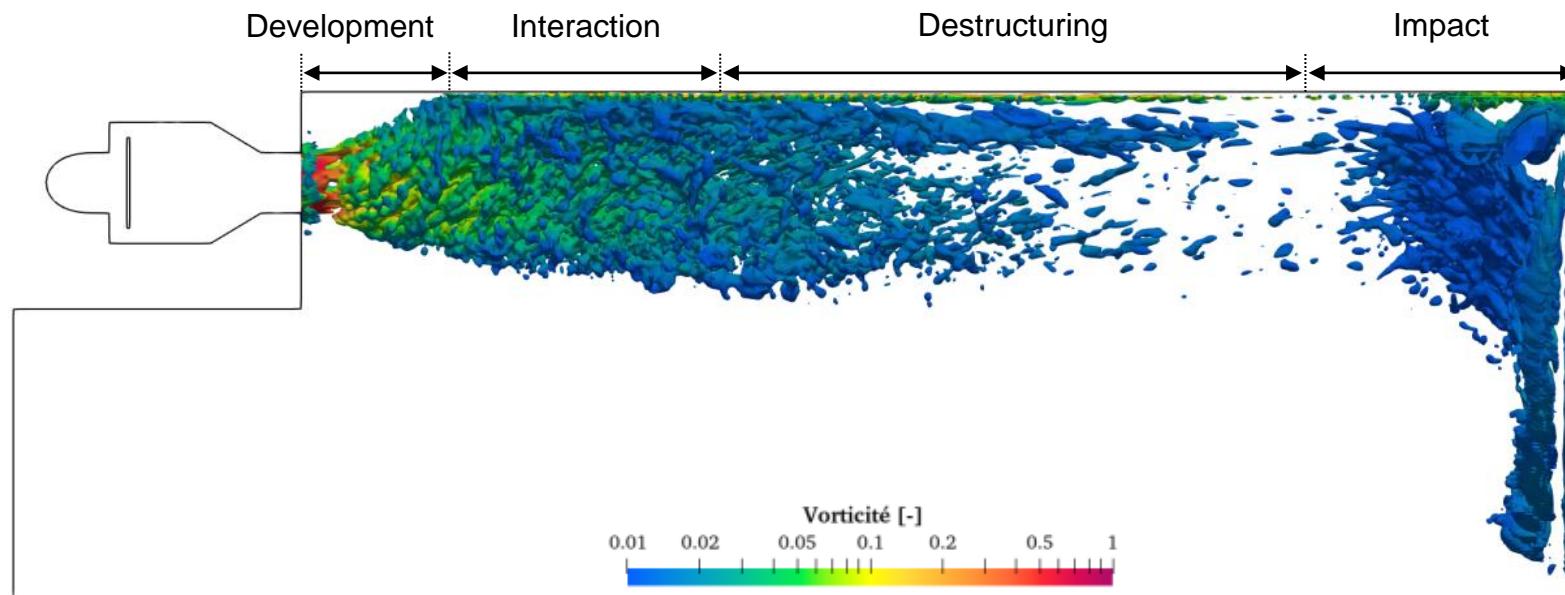
# Mean temperature profiles on median plan



AWF: Adjusted wall function  
SWF: Standard wall function

- AWF in better agreement with experimental profiles
- Deviation on the center of the room :  $0.6^{\circ}\text{C} \rightarrow 0.1^{\circ}\text{C}$   
→ Improvement of the convective heat fluxes prediction

# Turbulent structures visualization with Q-criterion



# Conclusions

- ✓ Capacity of the LBM-LES approach to simulate thermal airflows within buildings
- ✓ Adjustment of the  $Pr_{t,w}$  allows to recover correct temperature profiles

## Work in progress

- Derivation of an adaptive formula based on the Richardson number to calculate the  $Pr_{t,w}$  (*J. Allegrini and al. 2013*)
- Implementation of the new wall functions integrating the buoyancy effect

## Perspectives

- Extension to natural ventilation
- Coupling with a dynamic and three-dimensional building model (*Gresse and al. 2021*)
- Thermal comfort dynamics study with thermo-physiological human model

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### Questions & Comments

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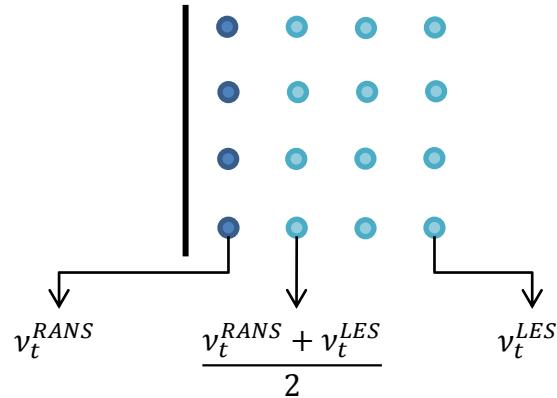
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# Appendix

- Subgrid viscosity

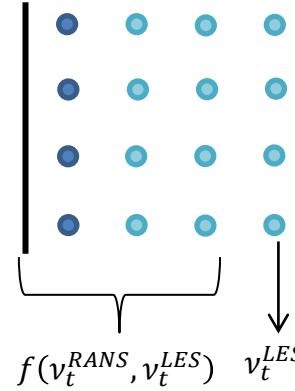
*Viscosity « simple »*



$$\nu_t^{RANS} = (\kappa y^+ A(y^+))^2 \left| \frac{\partial u}{\partial y} \right|$$

$\nu_t^{LES}$  from subgrid model

*Viscosity « blending »  
(Maeyama et al. 2021)*



$$f = K^\gamma \nu_t^{LES} + (1 - K^\gamma) \nu_t^{RANS}$$

$$K = \max \left[ \min \left( \frac{d_{IP} - y}{d_{IP}(1-\alpha')}, 1 \right), 0 \right]$$

- **Velocity wall functions**

- Implicit (*Afzal 1996*):

$$u^+ = \left(1 - e^{\frac{-y^+}{C_{11}}}\right) \left[ \frac{1}{\kappa} \ln(y^+) + B + f_{\text{pres}} + f_{\text{courb}} + f_{\text{rough}} \right]$$

- Explicit (*Cai et al. 2021*):

$$u^+ = Re_y \left[ \left(1 - \tanh \frac{Re_y}{s}\right)^p \sqrt{Re_y} + \left(\tanh \frac{Re_y}{s}\right)^p \frac{1}{E} e^{W(\kappa E Re_y)} \right]^{-1}$$

- **Temperature wall function**

- (*Kader 1981*):

$$T^+ = e^\Gamma T_{lam}^+ + e^\Gamma T_{turb}^+$$

$$\Gamma = \frac{0.01(Pr y^+)^4}{1+5Pr^3 y^+}, \quad T_{lam}^+ = Pr y^+, \quad T_{turb}^+ = Pr_{t,w} \left( \frac{1}{\kappa} \ln(y^+) + B + P_J \right)$$