

# An ill-posed mechanical problem with friction

Michelle Schatzman, Claude-Henri Lamarque, Jérôme Bastien

# ▶ To cite this version:

Michelle Schatzman, Claude-Henri Lamarque, Jérôme Bastien. An ill-posed mechanical problem with friction. European Journal of Mechanics - A/Solids, 1999, 18 (3), pp.415-420. 10.1016/S0997-7538(99)00134-5. hal-04772343

# HAL Id: hal-04772343 https://hal.science/hal-04772343v1

Submitted on 7 Nov 2024

**HAL** is a multi-disciplinary open access archive for the deposit and dissemination of scientific research documents, whether they are published or not. The documents may come from teaching and research institutions in France or abroad, or from public or private research centers. L'archive ouverte pluridisciplinaire **HAL**, est destinée au dépôt et à la diffusion de documents scientifiques de niveau recherche, publiés ou non, émanant des établissements d'enseignement et de recherche français ou étrangers, des laboratoires publics ou privés.



Distributed under a Creative Commons Attribution - NonCommercial 4.0 International License

# An ill-posed mechanical problem with friction

Michelle Schatzman<sup>a</sup>, Claude-Henri Lamarque<sup>b,\*</sup>, Jérôme Bastien<sup>b</sup>

<sup>a</sup> UMR 5585 CNRS, Analyse Numérique, Université Claude Bernard Lyon I, 69622 Villeurbanne Cedex, France

<sup>b</sup> URA 1652 CNRS Département Génie Civil et Bâtiment, Laboratoire Géomatériaux, École Nationale des Travaux Publics de l'Etat, rue Maurice Audin, 69518 Vaulx-en-Velin Cedex, France

Abstract – Many models involve the Coulomb's law in order to describe dynamical properties of friction phenomena. In order to generalize this Coulomb's law and to deal with its correct mathematical expression, we study a nonlinear equation where we take into account a maximal monotone graph. In the particular case of Coulomb's law, existence and uniqueness are proved. But in the general case, only existence persists. A counter-example to uniqueness is given.

friction / Coulomb's law / maximal monotone graph

### 1. Introduction

For twenty years, many studies have been devoted to nonlinear oscillations. Either they can deal with smooth nonlinearities due to the geometry (oscillations of a pendulum or large flexural displacements of a beam), or to constitutive laws (nonlinear elasticity, etc.), or to active forces (aeroelastic dry and lift forces depending on velocity and displacement of a structure). They can also investigate non-smooth non-linearities that modelize for example impacts (Whiston, 1987) or constitutive laws (Capecchi and Vestroni, 1995), or friction phenomena (Jean and Pratt, 1985).

In this study, we only deal with mechanical models including dry friction. We consider two distinct classes of models. A first class involves all the mechanical models with Coulomb's friction, and a second one involves all the other models: this last class contains all the models with a friction force which is not a constant during the dynamical phase.

We consider the one-dimensional motion of a material point on a plane support moving at velocity v(t); this point is submitted to a normal force directed downwards,  $F_n$ , which creates a longitudinal friction force. Moreover, the material point is submitted to given dynamics. The mathematical formulation of this problem is

$$\ddot{x}(t) = F(t, x(t), \dot{x}(t)) - \mu_D F_n \operatorname{sgn}(\dot{x}(t) - v(t)), \quad \text{if } \dot{x}(t) - v(t) \neq 0,$$
(1)

$$\ddot{x}(t) = F(t, x(t), \dot{x}(t)) + F_l$$
, with  $F_l \in [-\mu_S F_n, \mu_S F_n]$ , if  $\dot{x}(t) - v(t) = 0$ , (2)

where  $\mu_D$  and  $\mu_S$  are the dynamical and static friction coefficients;  $\dot{x}(t)$  denotes velocity relative to the support, and *F* describe the longitudinal dynamics. We assume that *F* is a Lipschitz continuous function with respect to its last arguments  $(x, \dot{x})$ .

<sup>\*</sup> Correspondence and reprints. E-mail: claude.lamarque@entpe.fr

This model had been studied for a long time because it is very simple, especially in the case of a spring-mass system with a passive external solicitation. In such a case, F is written as

$$F(t,\alpha,\beta) = -\alpha\beta - \omega^2\alpha \tag{3}$$

with specific (per unit mass) viscous damping a and eigenfrequency of the system  $\omega$ .

Den Hartog (1956) studied the Coulomb's model with a linear spring and viscous damping. This article focused on the instability induced by friction. Other authors deal with friction coefficients  $\mu_D$  and  $\mu_S$  which are constants (Awrejcewiczs and Delfs, 1990), which are increasing functions of time (Caudet, 1964; Baumberger, et al. 1990) or which are functions of the relative velocity (Dowell and Schwarz, 1983; Popp, 1989; Rabinowicz, 1996). If  $\mu_D$  is a constant equal to  $\mu_D$ , a proof of existence and uniqueness of a solution has been given in (Monteiro Marques, 1994). This paper by Monteiro Marques provides two sufficient conditions for uniqueness.

This article is organised as follows. In Section 2, we present a simple model where  $\mu_D$  is a constant and  $\mu_S$  is an increasing function of time (this case includes Coulomb's friction). In Section 3, we give an example of non-uniqueness, and in Section 4 we give a numerical scheme which has the same non-uniqueness properties as the continuous problem. Finally we draw conclusions.

#### 2. Introduction of the model

Let us consider again Eqs (1) and (2). Let us choose  $F_n = 1$ . We intend to study a solid with abscissa x(t), moving on a plane rigid support, with velocity v(t), submitted to an external smooth force  $F(t, x(t), \dot{x}(t))$  and to friction  $\phi(t)$ ;  $\dot{x}(t)$  denotes the velocity relative to the referential. Thus this model can be expressed as

$$\ddot{x}(t) = F(t, x(t), \dot{x}(t)) + \phi(t), \tag{4}$$

$$x(0) = x_0, \tag{5}$$

$$\dot{x}(0) = \dot{x}_0. \tag{6}$$

Now let us describe the friction term: our model of friction takes into account the points of view of Caudet (1964) and Baumberger et al. (1990). This model is based upon the motion of last blocking time. Let w be a continuous function on [0, T] and let

$$b(t, w) = \begin{cases} t, & \text{if } w(t) \neq 0, \\ \sup\{s \in [0, t[ \text{ such that } w(s) \neq 0\}, & \text{if } w(t) = 0. \end{cases}$$
(7)

This definition assumes that if w(0) = 0, a non positive b(0, w) is given. We consider the different phases: static ones  $(\dot{x}(t) - v(t) = 0)$  and dynamic ones  $(\dot{x}(t) - v(t) \neq 0)$ .

We introduce two coefficients: a dynamic friction coefficient  $\mu_D$  and a static friction coefficient  $\mu_S(t)$ . The coefficient  $\mu_D$  is a strictly positive constant and  $\mu_S(t)$  is an increasing positive function, bounded on every static phase. So we have

$$\begin{cases} \phi(t) = -\mu_D \operatorname{sgn}(\dot{x}(t) - v(t)), & \text{if } \dot{x}(t) - v(t) \neq 0, \\ \phi(t) \in [-\mu_S(t), \mu_S(t)], & \text{if } \dot{x}(t) - v(t) = 0, \end{cases}$$
(8)

where

$$\mu_{S}(t) = h(t - b(t, \dot{x} - v)), \tag{9}$$

where *h* is an increasing bounded positive function from  $]0, +\infty[$  to  $]0, +\infty[$ . We can simplify (8) and (9) by setting

$$h(0) = \mu_D \tag{10}$$

with  $\mu_D \leq \mu_S$ , and we consider the maximal monotone graph  $\sigma$  defined by

$$\sigma(x) = \begin{cases} -1, & \text{if } x < 0, \\ 1, & \text{if } x > 0, \\ [-1, 1], & \text{if } x = 0. \end{cases}$$
(11)

So (8) and (9) are equivalent to

$$\phi(t) \in -\sigma\left(\dot{x}(t) - v(t)\right)h\left(t - b(t, \dot{x} - v)\right). \tag{12}$$

Thus, for given F, v, b,  $\sigma$ , h,  $x_0$ , and  $\dot{x}_0$ , we seak functions x and  $\phi$  from [0, T] to  $\mathbb{R}$ , such that Eqs (4) and (12) are verified with initial conditions (5) and (6). Let us observe that  $\phi$  can be eliminated from (4) and (12)

$$\ddot{x}(t) + \sigma (\dot{x}(t) - u(t)) h (t - b(t, \dot{x} - v)) \ni F (t, x(t), \dot{x}(t)).$$
(13)

If we assume that h is a positive constant  $\mu = \mu_D$ , (12) is equivalent to

$$\begin{cases} \phi(t) = -\mu \operatorname{sgn}(\dot{x}(t) - v(t)), & \text{if } \dot{x}(t) - v(t) \neq 0, \\ \phi(t) \in [-\mu, \mu], & \text{if } \dot{x}(t) - v(t) = 0 \end{cases}$$
(14)

and we find again the particular case of Coulomb's model. If velocity  $\dot{x}$  is denoted by y, Eqs (5), (6) and (13) are equivalent to

$$\dot{y}(t) + \sigma \left( y(t) - v(t) \right) h \left( t - b(t, y - v) \right) \ni F \left( t, x_0 + \int_0^t y(s) \, \mathrm{d}s, \, y(t) \right), \tag{15}$$

where *y* verifies the initial condition

$$y(0) = \dot{x}_0.$$
 (16)

#### 3. Existence and uniqueness results: ill-posed model

*F* is continuous with respect to all its arguments and Lipschitz continuous with respect to its last two arguments *x* and  $\dot{x}$ ; then there exists a solution *x* to (5), (6) and (13). In the particular case of Coulomb's friction, *x* is unique. In the general case, uniqueness is not true. Moreover, one can exhibit an example with non-uniqueness for the problem (15), (16). We assume now that

1. 
$$y(0) = 0, \quad v = 0,$$
 (17)

2. *h* is increasing, continuous and strictly positive on  $\mathbb{R}_+$ , (18)

3. Let also *F* be defined by F(t, a, b) = h(t). (19)

A family of distinct functions  $\{y_u\}_{u \in [0,T]}$  is defined as follows

$$\forall t \in [0, u], \quad y_u(t) = 0, \text{ and } \forall t \in [u, T], \qquad y_u(t) = \int_u^t h(s) \, \mathrm{d}s - h(0)(t - u).$$
 (20)

We verify readily that  $y_u$  solves (15), (16). We also notice that  $y_u \neq y_v$  when  $u \neq v$ . Thus, the solution to Eqs (15) and (16) is not unique.

By using the same idea, can show the sensitivity of the solutions to data. Let us set

$$\forall t \in [0, T], \quad y_0(t) = \int_0^t h(s) \, \mathrm{d}s - h(0)t, \quad \text{and} \quad y_T \equiv 0$$
 (21)

and for every  $\varepsilon \in \mathbb{R}$ 

$$f_{\varepsilon} = h + \varepsilon. \tag{22}$$

Consider functions  $\{Y_{\varepsilon}\}_{\varepsilon \in \mathbb{R}}$  such that for every  $\varepsilon \in \mathbb{R}$  and for every  $t \in [0, T]$ 

$$Y_{\varepsilon}(t) = \begin{cases} y_0(t) + \varepsilon t, & \text{if } \varepsilon > 0\\ 0, & \text{if } \varepsilon \leqslant 0. \end{cases}$$
(23)

For every  $\varepsilon \in [-2h(0), +\infty[$ , we have

$$\dot{Y}_{\varepsilon} + \sigma (Y_{\varepsilon}(t))h(t - b(t, Y_{\varepsilon})) \ni f_{\varepsilon}, \text{ and } Y_{\varepsilon}(0) = 0.$$
 (24)

But  $y_0$  and  $y_T$  are two distinct solutions to (15), (16) and

$$\lim_{\varepsilon \to 0^+} Y_{\varepsilon} = y_0, \qquad \lim_{\varepsilon \to 0^-} Y_{\varepsilon} = y_T, \qquad \lim_{\varepsilon \to 0} f_{\varepsilon} = h, \quad \text{in } C^0([0, T]).$$
(25)

Thus the solutions are not continuous with respect to the data.

## 4. Numerical scheme

Let us consider the counter-example to uniqueness of Section 3 defined by Eqs (15) and (20) and assumptions (17), (18) and (19). Its solutions can be approximated by a numerical scheme similar to the implicit Euler scheme for first-order differential equations. Let  $\delta \in [0, T]$ , let *N* be the integer part of  $T/\delta$  and  $t_n = n\delta$ , for every  $n \in \{0, ..., N\}$ . Define continuous and piecewise linear functions  $z_{\delta}$  on [0, T] by their values at the nodes  $t_n$ 

$$\forall n \in \{0, \dots, N\}, \quad Z_n = z_\delta(t_n). \tag{26}$$

Let  $Z_0 = y_0$  and assume that the  $Z_n$ 's satisfy the following relation

$$\forall n \in \{0, \dots, N-1\}, \quad \frac{Z_{n+1} - Z_n}{\delta} + \sigma(Z_{n+1})h(t_{n+1} - b(t_{n+1}, z_{\delta})) \ni h(t_{n+1}).$$
(27)

According to Brezis (1973), if A is a maximal monotone graph, for every  $\lambda > 0$ ,  $I + \lambda A$  is invertible; i.e., for every  $y \in \mathbb{R}$ , the equation

$$x + \lambda A(x) \ni y \tag{28}$$

admits a unique solution, denoted

$$x = (I + \lambda A)^{-1}(y).$$
 (29)

In the case of the Coulomb's friction, we have  $h = \mu > 0$  and according to (27), (28) and (29)  $Z_{n+1}$  is defined by

$$\forall n \in \{0, \dots, N-1\}, \quad Z_{n+1} = (I + \mu \delta \sigma)^{-1} \big(\delta h(t_{n+1}) + Z_n\big).$$
(30)

Let us show that with assumptions (17), (18), (19), scheme (27) has several solutions.

Let us assume for  $n \in \{0, ..., N - 1\}, Z_0 = \cdots = Z_n = 0.$ 

(1) If  $Z_{n+1} = 0$ , we have according to Eq. (7)

$$b(t_{n+1}, z_{\delta}) = 0$$
 and  $h(t_{n+1} - b(t_{n+1}, z_{\delta})) = h(t_{n+1})$  (31)

and Eq. (27) is verified.

(2) If  $Z_{n+1} > 0$ , we have according to Eq. (7)

$$b(t_{n+1}, z_{\delta}) = t_{n+1}$$
 and  $h(t_{n+1} - b(t_{n+1}, z_{\delta})) = h(0)$  (32)

and Eqs (27), (28), (29) imply

$$Z_{n+1} = (I + h(0)\delta\sigma)^{-1} (\delta h(t_{n+1})).$$
(33)

Because  $\sigma$  is strictly increasing, one can verify that the value of  $Z_{n+1}$  given by (33) is strictly positive. Then in both cases,  $Z_{n+1}$  is a solution of (27). This numerical scheme has therefore N solutions.

## 5. Conclusion

We have generalized Coulomb's law into a law which involves the last blocking time. We have proved existence for this model, and given a counter-example to uniqueness. Therefore the model under study is defective, and other models which possess better properties should be considered. It would be an interesting perspective to mathematically analyze models or which  $\mu_s$  could depend on the relative abscissa, or the velocity of the support.

Otherwise, in order to justify non-uniqueness of solutions to problem (15) in the general case, we can note that this problem can be written in the form

$$\dot{y}(t) + A_{t,y}(y(t)) \ni F\left(t, x_0 + \int_0^t y(s) \,\mathrm{d}s, y(t)\right),$$
(34)

where at each given time t, the graph  $u \to A_{t,y}(u)$  is maximal monotone one. But, the friction is no longer maximal monotone when t is varying in [0, T]. This remark can explain a posteriori the lack of uniqueness for the studied problem.

#### References

Awrejcewicz J., Delfs J., 1990. Dynamics of a self-excited stick–slip oscillator with two degrees of freedom. Part I: investigation of equilibria. Eur. J. Mech. A/Solids 9 (4), 269–282.

Baumberger T., Caroli C., Perrin B., Ronsin O., 1990. Dynamics of a self-excited stick–slip oscillator with two degrees of freedom. Part II: Slip–stick, slip–slip, stick–slip transitions, periodic and chaotic orbits. Eur. J. Mech. A/Solids 9 (5), 397–418.

Brezis H., 1973. Opérateurs maximaux monotones et semi-groupes de contraction dans les espaces de Hilbert. Mathematics Studies 5, North-Holland, Amsterdam.

Brogliato B., 1996. Nonsmooth Impact Dynamics. Lectures Notes in Control and Information Sciences 220, Springer, NY, Berlin.

Capecchi D., Vestroni F., 1995. Asymptotic response of a two DOF elastoplastic system under harmonic excitation. Internal resonance case. Nonlinear Dyn. 7, 317–333.

Caubet J.J., 1964. Théorie et pratique du frottement. Dunod Technip, Paris.

Dowell E.H., Schwartz H.B., 1983. Forced response of a cantilever beam with a dry friction damper attached, Part I: Theory. J. Sound Vib. 91 (2), 255–267.

Foale S., Bishop S.R., 1994. Bifurcations in impact oscillations. Nonlinear Dyn. 6, 285–299.

Den Hartog, 1956. Mechanical Vibrations. McGraw-Hill, New York.

Kunert A., Pfeiffer F., 1989. Stochastic model for tatling in gear-boxes. In: IUTAM Symp. Nonlinear Dynamics in Engineering Systems, Univ. Stuttgart, Germany, Aug. 21–25.

Jean M., Pratt E., 1985. A system of rigid bodies with dry friction. Int. J. Ing. Sci. 23, 497-513.

Monteiro Marques M.D.P., 1994. An existence, uniqueness and regularity study of the dynamics of systems with one-dimensional friction. Eur. J. Mech. A/Solids 13 (2), 277–306.

Popp K., Stelter P., 1989. Non-linear oscillations of structures induced by dry friction. In: IUTAM Symp. Nonlinear Dynamics in Engineering Systems, Univ. Stuttgart, Germany, Aug. 21–25.

Rabinowicz E., 1996. Friction and Wear of Materials. Wiley and Sons, New York.

Whiston G.S., 1987. Global dynamics of a vibro-impacting linear oscillator. J. Sound Vib. 118 (3), 395-429.