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Beatrice Battisti, Giovanni Bracco, Michel Bergmann. A multi-fidelity model for wave energy converters. 2024. hal-04496537

HAL Id: hal-04496537 https://hal.science/hal-04496537

Preprint submitted on 8 Mar 2024

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A multi-fidelity model for wave energy converters

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Abstract

The objective of this study is to develop a three-dimensional numerical model for a floating point absorber wave energy converter in the presence of sea waves, considering its interaction with a bi-fluid flow (comprising air and water). The primary aim is to create an efficient computational tool that achieves two key objectives: firstly, reducing the computational time typically associated with high-fidelity Computational Fluid Dynamics (CFD) models, and secondly, curing the lack of accuracy of low-fidelity asymptotic or projection-based reduced-order models in regions subjected to viscous and highly nonlinear effects. To address these objectives, we propose a multi-fidelity model based on domain decomposition. This approach combines a high-fidelity CFD solver, which accurately captures the behavior in viscous and nonlinear regions, with a Reduced Order Model (ROM) based on Proper Orthogonal Decomposition (POD), tailored for weakly nonlinear regions. By integrating these components spatially, we simulate the dynamics of the floating body within a unified framework. This methodology ensures precise predictions of the body's motion for both in-sample (reproduction) and out-of-sample (prediction) configurations.

Keywords: Multi-fidelity model, Proper Orthogonal Decomposition, Reduced Order Model, Coupling methodology, Wave energy.

1 Introduction

Numerical simulations are ubiquitous in industrial sectors such as energy, where technologies undergo a virtual cycle of concept, design, and testing before physical manufacturing. For instance, although wave energy remains an emerging sector compared to established renewable sources, it heavily relies on tractable numerical models for conception and optimization, necessitating a delicate balance between accuracy and computational costs. Wave energy converters, particularly those in the wave-activated body class, require large motion amplitudes for optimal efficiency.

Traditional sea-keeping numerical tools, often based on linear potential theory and commonly used in offshore structure design, are widely utilized in the initial stages of wave energy converter development, providing quick estimates of system parameters [1]. However, discrepancies persist between numerical predictions and wave tank experiments [2]. Introducing some nonlinearities to weakly or fully nonlinear potential flow models partially addresses this issue [3, 4, 5], while maintaining fundamental assumptions such as inviscid flow.

The utilization of high-fidelity solvers such as Computational Fluid Dynamics (CFD) or Smoothed-Particle Hydrodynamics (SPH) allows accurate simulation of all nonlinearities, but remains challenging for wave energy applications due to significant computational time requirements. Indeed, as the number of devices increases, computational costs become prohibitive, particularly in optimization analyses considering large parameter spaces and multiple queries [6, 7]. This complexity necessitates the development of accurate yet computationally efficient models.

To address this need, data-driven model reduction emerges as a strategy to extract essential features and reduce computational complexity. Reduced-Order Models (ROMs) have diverse applications spanning various fields, including thermo-acoustic problems [8], image processing [9], graphics applications [10, 11], and fluid flows [12, 13]. In the context of fluid dynamics, numerous ROMs have been developed for fluid control [14, 15], interaction of flows with varying body shapes [16, 17, 18], and turbulence description [19, 20, 21]. These models find applications in various domains, ranging from fluid-structure interaction in cardiovascular problems [22] to the numerical simulation of the wake behind a wind turbine [23], and the depiction of the dynamics of multi-phase flows [24], as in the case of this study. However, the application of ROMs to highly nonlinear problems like wave energy converters necessitates careful consideration.

Among the model reduction techniques, the Proper Orthogonal Decomposition (POD) method emerges as an effective means for deriving a reduced basis in high-dimensional flow systems. Intrusive ROMs, typically derived through POD and Galerkin projection methods or (least square) Petrov-Galerkin approaches [25, 26], are attractive for linear and weakly nonlinear model reductions and find success in various research domains. However, Galerkin ROMs might encounter challenges related to instability, and the use of the linear affine POD representation can limit efficiency for highly nonlinear problems, even for Petrov-Galerkin projections. Quadratic POD approximations [27] and fully nonlinear approximations [28] have been recently introduced to mitigate the Kolmogorov barrier in nonlinear projection ROMs. However, their intrusive nature often necessitates complex coding modifications [29]. The complexity of intrusive projection-based ROMs is particularly evident when the computational source code is unavailable, as is often the case with some commercial software. To overcome this limitation, we propose a non-intrusive ROM approach where POD ROM is obtained in a Galerkin-free way, coupled with a high-fidelity solver implemented in commercial code.

Similar to other coupling methodologies [30, 31], this approach relies on domain decomposition. In this study, the computational domain is decomposed into low-fidelity (POD) and high-fidelity (Navier-Stokes solver) subdomains. The temporal coefficients of the POD expansion are obtained through a minimization problem defined on both an overlapping domain between the two subdomains and on a domain (discrete points defining sensors) where incoming flow is actually imposed. The high-fidelity domain is usually chosen to be a small surrounding of the floating body (the wave energy converter) where highly nonlinear phenomena usually occur, and the other POD domain is used to both give boundary conditions to the high-fidelity solver and to propagate the information elsewhere using global POD modes. This multi-fidelity numerical model is used to predict the temporal evolutions of velocity field, the water surface, and the body dynamics in wave energy converter simulations.

The paper is organized as follows. The multi-fidelity model, which couples the highfidelity Navier-Stokes model and the low-fidelity POD model, is presented in Section 2. A in-depth sensitivity analysis of several parameters involved in the multi-fidelity model, such as the number of POD modes or the size of the different subdomains, is described for a simple flow configuration in Section 3. A floating point absorber wave energy converter is then studied in Section 4. Finally, some conclusions are drawn in Section 5.

2 Numerical multi-fidelity model

2.1 Flow configuration

The objective is to characterize the performance of a wave energy harvesting device under the influence of sea waves. A three-dimensional domain is considered, and a two-dimensional vertical section is schematically represented in Figure 1. We consider a point absorber-type structure, wherein the sole degree of freedom is the vertical motion. The densities and viscosities are represented by ρ and μ , with subscripts *a* and *w* indicating air and water, respectively. The floater, characterized by a density ρ_s , occupies domain Ω_s , while the fluid (comprising air and water) is flowing within domain Ω_f . The fluid domain Ω_f is further partitioned into two distinct regions, Ω_a for air and Ω_w for water, with an interface denoted by Γ_f .

Given the nature of a floating structure, we have $\rho_a < \rho_s < \rho_w$.



Figure 1: Sketch of the flow configuration.

Before introducing the multi-fidelity model (Sec. 2.4), we give a description of the two models involved in the coupling procedure, the high-fidelity model (Sec. 2.2) and the low-fidelity POD approximation (Sec. 2.3).

2.2 High-fidelity numerical model

The flow described in the previous section is modelled by the two-phase Navier-Stokes equations. Let α represent the local fraction of water, referred to as the Volume Of Fluid (VOF), where $0 \le \alpha \le 1$ [32]. The fluid density and dynamic viscosity can be expressed as follows:

$$\rho(\boldsymbol{x}, t) = \rho_a + (\rho_w - \rho_a) \,\alpha(\boldsymbol{x}, t), \tag{1}$$

$$\mu(\boldsymbol{x}, t) = \mu_a + (\mu_w - \mu_a) \,\alpha(\boldsymbol{x}, t), \qquad (2)$$

where $\alpha = 0$ and $\alpha = 1$ denote air and water, respectively. The bi-fluid interface is approximated by $\alpha = 0.5$.

Let $D(\boldsymbol{u}) = \frac{\boldsymbol{\nabla}\boldsymbol{u} + (\boldsymbol{\nabla}\boldsymbol{u})^T}{2}$ denote the viscous part of the stress tensor, where the velocity field is $\boldsymbol{u} = (u, v, w)$ with u, v, and w representing the velocity components in the x, y, and z directions, respectively. Let also p be the pressure field and \boldsymbol{g} the gravity acceleration along the y axis. The incompressible Navier-Stokes equations for both fluids in the domain $\Omega_f = \Omega_a \cup \Omega_w$ are:

$$\frac{\partial \boldsymbol{u}}{\partial t} + (\boldsymbol{u} \cdot \boldsymbol{\nabla})\boldsymbol{u} = -\frac{1}{\rho}\boldsymbol{\nabla}p + \frac{1}{\rho}\boldsymbol{\nabla} \cdot 2\mu D(\boldsymbol{u}) + \boldsymbol{g} \quad \text{in } \Omega_f,$$
(3)

$$\nabla \cdot \boldsymbol{u} = 0 \quad \text{in } \Omega_f, \tag{4}$$

where the temporal evolution of the bi-fluid interface, and thus μ and ρ , is governed by the transport equation:

$$\frac{\partial \alpha}{\partial t} + \boldsymbol{u} \cdot \boldsymbol{\nabla} \alpha = 0 \quad \text{in } \Omega_f.$$
⁽⁵⁾

This system is complemented with initial conditions $\boldsymbol{u}(\boldsymbol{x}, t = 0) = \boldsymbol{u}_0, \ p(\boldsymbol{x}, t = 0) = p_0$, and $\alpha(\boldsymbol{x}, t = 0) = \alpha_0$, boundary conditions on the external boundary Γ_e for velocity $\boldsymbol{u}(\boldsymbol{x}, t)$ and VOF $\alpha(\boldsymbol{x}, t)$, and boundary conditions on the structure boundary Γ_s , as well as conditions through the bi-fluid interface Γ_f [33].

The motion of the floating rigid body is governed by Newton's laws:

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$$m\frac{\mathrm{d}\boldsymbol{\overline{u}}}{\mathrm{d}t} = \boldsymbol{F}_{ext},\tag{6}$$

$$\frac{\mathrm{d}J\boldsymbol{\omega}}{\mathrm{d}t} = \boldsymbol{M}_{ext},\tag{7}$$

where m and J are the mass and inertia matrix of the body, and \overline{u} and ω denote the linear and angular velocities.

The forces and torques are computed by:

$$\boldsymbol{F}_{ext} = -\int_{\Gamma_s} \mathbb{T}(\boldsymbol{u}, p) \, \boldsymbol{n} \, \mathrm{d}\boldsymbol{x} + \boldsymbol{g}, \tag{8}$$

$$\boldsymbol{M}_{ext} = -\int_{\Gamma_s} \boldsymbol{r} \wedge \mathbb{T}(\boldsymbol{u}, p) \, \boldsymbol{n} \, \mathrm{d}\boldsymbol{x}, \tag{9}$$

where $\mathbb{T}(\boldsymbol{u}, p) = -p\boldsymbol{I} + \mu(\nabla \boldsymbol{u} + \nabla \boldsymbol{u}^T)$ is the stress tensor, \boldsymbol{n} is the unit outward vector to Γ_s , and $\boldsymbol{r} = \boldsymbol{x} - \boldsymbol{x}_G$ with \boldsymbol{x}_G representing the body center of mass.

In this study, the numerical solution of the Navier-Stokes equations is computed using the commercial software StarCCM+ [34]. The governing equations are discretized on an overset Chimera mesh, where a foreground body-fitted grid refined near the body is superimposed on an octree backward grid refined around the air-water interface. A typical computational mesh used in this study is shown in Figure 2.



Figure 2: Sketch of the computational mesh.

For open flows, artificial boundary conditions are typically applied at external boundaries. These may include Dirichlet or Neumann conditions, or more sophisticated absorbing or non-reflecting conditions [35, 36], depending on the complexity of the problem under consideration. Imposing appropriate boundary conditions remains one of the main challenges in Computational Fluid Dynamics (CFD). In open flows, such as the one described in Figure 1, boundary conditions are usually applied far from the body, resulting in a large computational domain. The computational costs required to solve the Navier-Stokes equations in such a computational domain can become prohibitively large. To decrease these costs, reduced-order models can be employed, either as a substitution model or to provide more suitable boundary conditions that allow for a reduction in the size of the computational domain [37, 38].

Instead of computing the solution of the Navier-Stokes equations on a fine mesh with possibly millions of cells, we can approximate the solution using a few global basis functions (typically on the order of $\mathcal{O}(10)$). In this study, the global basis functions are computed using Proper Orthogonal Decomposition (POD).

2.3 Low-fidelity subspace: Proper Orthogonal Decomposition

The goal is to approximate the numerical high-fidelity solution U_h of the governing equations (3)-(5) by a POD ansatz \tilde{U} with few $N_r = \mathcal{O}(10)$ global basis functions $\{\phi_i\}_{i=1}^{N_r}$, *i.e.*

$$\boldsymbol{U}_{h}(\boldsymbol{x}, t) \approx \widetilde{\boldsymbol{U}}(\boldsymbol{x}, t) = \overline{\boldsymbol{U}}(\boldsymbol{x}) + \sum_{n=1}^{N_{r}} a_{n}(t)\boldsymbol{\phi}_{n}(\boldsymbol{x}).$$
(10)

The field U can be the velocity field u, the pressure field p, the density field ρ , the VOF field α , or even a mixed field (u, p, ρ, α) (or other alternatives), and \overline{U} is a reference field. The latter is usually the average in time, a shift or non equilibrium mode [39, 40], or zero.

The Proper Orthogonal Decomposition (POD) was first introduced in turbulence by Lumley [41] in 1967 to identify the coherent structures in a turbulent flow. For a review of the POD, readers are referred to [42, 43]. The goal is to find the deterministic functions $\phi(\mathbf{x})$ that are most similar in an average sense to data $\mathbf{U}(\mathbf{x},t) = \mathbf{U}_h(\mathbf{x},t) - \overline{\mathbf{U}}(\mathbf{x})$. In what follows, the snapshot method introduced by Sirovich [44] is used.

The construction of the low-fidelity subspace, the POD basis functions, is called the *offline* stage. It is based on the following steps.

First, one has to collect a set of N_s snapshots $\{U_h(\boldsymbol{x}, t_i)\}_{i=1}^{N_s}$. Usually, the snapshots are extracted at discrete times $\{t_i\}_{i=1}^{N_s}$ where $t_i = T_0 + (i-1)\Delta T$, for given values of T_0 and ΔT . Note that snapshots can also be sampled on different input parameters η_i (as it will be the case hereafter for different wave parameters). Without loss of generality, the POD description only consider time, and we then build $U(\boldsymbol{x}, t_i) = U_h(\boldsymbol{x}, t_i) - \overline{U}(\boldsymbol{x})$, where $\overline{U}(\boldsymbol{x})$ is the temporal average of the N_s snapshot $U_h(\boldsymbol{x}, t_i)$.

In what follows, the snapshots are extracted (and possibly interpolated) on a uniform Cartesian grid composed by n_c cells. In this study, we consider $\boldsymbol{U} = (u, v, w, \alpha)^T$. One possible drawback of computing POD basis functions from velocity and VOF fields together is that the same number of modes is considered for both velocity and VOF, leading to a sub-optimal data compression. However, the main advantage is that only one POD is necessary, and all the variables are closely linked. If needed, one could also add the (scaled) pressure field to \boldsymbol{U} . The snapshots matrix is $\mathbf{S} = [\boldsymbol{U}(\boldsymbol{x}, t_1)^T \quad \boldsymbol{U}(\boldsymbol{x}, t_2)^T \quad \cdots \quad \boldsymbol{U}(\boldsymbol{x}, t_{N_s})^T] \in \mathbb{R}^{4n_c \times N_s}$. Numerically, we have:

$$\mathbf{S} = \begin{pmatrix} u(x_{1},t_{1}) & u(x_{1},t_{2}) & \cdots & u(x_{1},t_{N_{s}}) \\ \vdots & \vdots & & \vdots \\ u(x_{n_{c}},t_{1}) & u(x_{n_{c}},t_{2}) & \cdots & u(x_{n_{c}},t_{N_{s}}) \\ v(x_{1},t_{1}) & v(x_{1},t_{2}) & \cdots & v(x_{1},t_{N_{s}}) \\ \vdots & \vdots & & \vdots \\ v(x_{n_{c}},t_{1}) & v(x_{n_{c}},t_{2}) & \cdots & v(x_{n_{c}},t_{N_{s}}) \\ \vdots & \vdots & & \vdots \\ w(x_{1},t_{1}) & w(x_{1},t_{2}) & \cdots & w(x_{1},t_{N_{s}}) \\ \vdots & \vdots & & \vdots \\ w(x_{n_{c}},t_{1}) & w(x_{n_{c}},t_{2}) & \cdots & w(x_{n_{c}},t_{N_{s}}) \\ \vdots & \vdots & & \vdots \\ \alpha(x_{n_{c}},t_{1}) & \alpha(x_{1},t_{2}) & \cdots & \alpha(x_{n_{c}},t_{N_{s}}) \\ \vdots & \vdots & & \vdots \\ \alpha(x_{n_{c}},t_{1}) & \alpha(x_{n_{c}},t_{2}) & \cdots & \alpha(x_{n_{c}},t_{N_{s}}) \end{pmatrix}$$

$$(11)$$

The temporal correlation matrix is

$$\mathbf{R} = \mathbf{S}^T \, \mathbf{S} \in \mathbb{R}^{N_s \times N_s}. \tag{12}$$

If one wants to consider another parameter η , the snapshots matrix become $\mathbf{S} = (\mathbf{S}_{\eta_1}, \mathbf{S}_{\eta_2}, \dots, \mathbf{S}_{\eta_{N_m}})$, where N_m is the number of sampling points for parameters η , and \mathbf{S}_{η_i} is the snapshot matrix (11) built with $\eta = \eta_i$, and we finally have $N_s := N_s \times N_m$ snapshots.

We then solve the following eigenvalues problem

$$\mathbf{R}\psi_n = \lambda_n \psi_n, \quad n = 1, \cdots, N_s \tag{13}$$

where the $\lambda_1 \geq \lambda_2 \geq \cdots \geq \lambda_{N_s} \geq 0$.

The POD basis functions, also called POD modes, are:

$$\phi_n = \frac{1}{\sqrt{\lambda_n}} \mathbf{S} \psi_n, \quad n = 1, \cdots, N_s, \tag{14}$$

and the associated POD temporal coefficients are $a_n = \frac{\psi_n}{\sqrt{\lambda_n}}$.

Finally, from the relative information content $RIC(n) = \sum_{i=1}^{n} \lambda_i / \sum_{i=1}^{N_S} \lambda_i$, the POD basis is truncated to keep only the $N_r \ll N_s$ most dominant modes $\{\phi_i\}_{i=1}^{N_r}$, such that $RIC(N_r) > \delta$, with a user defined threshold δ usually taken to be larger than 99%. Let $\mathbf{\Phi} \in \mathbb{R}^{4 n_c \times N_r}$ be the matrix built with the N_r first POD modes, *i.e.*

$$\boldsymbol{\Phi} = \begin{pmatrix} \phi^{u}(x_{1},t_{1}) & \phi^{u}(x_{1},t_{2}) & \cdots & \phi^{u}(x_{1},t_{N_{r}}) \\ \vdots & \vdots & & \vdots \\ \phi^{u}(x_{n_{c}},t_{1}) & \phi^{u}(x_{n_{c}},t_{2}) & \cdots & \phi^{u}(x_{n_{c}},t_{N_{r}}) \\ \phi^{v}(x_{1},t_{1}) & \phi^{v}(x_{1},t_{2}) & \cdots & \phi^{v}(x_{1},t_{N_{r}}) \\ \vdots & \vdots & & \vdots \\ \phi^{v}(x_{n_{c}},t_{1}) & \phi^{v}(x_{n_{c}},t_{2}) & \cdots & \phi^{v}(x_{n_{c}},t_{N_{r}}) \\ \phi^{w}(x_{1},t_{1}) & \phi^{w}(x_{1},t_{2}) & \cdots & \phi^{w}(x_{1},t_{N_{r}}) \\ \vdots & \vdots & & \vdots \\ \phi^{w}(x_{n_{c}},t_{1}) & \phi^{w}(x_{n_{c}},t_{2}) & \cdots & \phi^{w}(x_{n_{c}},t_{N_{r}}) \\ \phi^{\alpha}(x_{1},t_{1}) & \phi^{\alpha}(x_{1},t_{2}) & \cdots & \phi^{\alpha}(x_{1},t_{N_{r}}) \\ \vdots & \vdots & & \vdots \\ \phi^{\alpha}(x_{n_{c}},t_{1}) & \phi^{\alpha}(x_{n_{c}},t_{2}) & \cdots & \phi^{\alpha}(x_{n_{c}},t_{N_{r}}) \end{pmatrix}$$

$$(15)$$

The equation (10) writes:

$$U \simeq \widetilde{U} = \overline{U} + \Phi a, \tag{16}$$

with $\boldsymbol{a} = (a_1, a_2, \dots, a_{N_r})^T$.

Defining the sub-matrices

$$\Phi^{u} = \begin{pmatrix} \phi^{u}(x_{1},t_{1}) & \phi^{u}(x_{1},t_{2}) & \cdots & \phi^{u}(x_{1},t_{N_{r}}) \\ \vdots & \vdots & & \vdots \\ \phi^{u}(x_{n_{c}},t_{1}) & \phi^{u}(x_{n_{c}},t_{2}) & \cdots & \phi^{u}(x_{n_{c}},t_{N_{r}}) \end{pmatrix} \in \mathbb{R}^{n_{c} \times N_{r}} \\
\Phi^{v} = \begin{pmatrix} \phi^{v}(x_{1},t_{1}) & \phi^{v}(x_{1},t_{1}) & \cdots & \phi^{v}(x_{1},t_{N_{r}}) \\ \vdots & \vdots & & \vdots \\ \phi^{v}(x_{n_{c}},t_{1}) & \phi^{v}(x_{n_{c}},t_{2}) & \cdots & \phi^{v}(x_{n_{c}},t_{N_{r}}) \end{pmatrix} \in \mathbb{R}^{n_{c} \times N_{r}} \\
\Phi^{w} = \begin{pmatrix} \phi^{w}(x_{1},t_{1}) & \phi^{w}(x_{1},t_{2}) & \cdots & \phi^{w}(x_{1},t_{N_{r}}) \\ \vdots & \vdots & & \vdots \\ \phi^{w}(x_{n_{c}},t_{1}) & \phi^{w}(x_{n_{c}},t_{2}) & \cdots & \phi^{w}(x_{n_{c}},t_{N_{r}}) \end{pmatrix} \in \mathbb{R}^{n_{c} \times N_{r}} \\
\Phi^{\alpha} = \begin{pmatrix} \phi^{\alpha}(x_{1},t_{1}) & \phi^{\alpha}(x_{1},t_{2}) & \cdots & \phi^{\alpha}(x_{1},t_{N_{r}}) \\ \vdots & \vdots & & \vdots \\ \phi^{\alpha}(x_{n_{c}},t_{1}) & \phi^{\alpha}(x_{n_{c}},t_{2}) & \cdots & \phi^{\alpha}(x_{n_{c}},t_{N_{r}}) \end{pmatrix} \in \mathbb{R}^{n_{c} \times N_{r}},$$
(17)

each physical field can be approximated onto these basis:

$$u \simeq \widetilde{u} = \overline{u} + \Phi^{u} \boldsymbol{a},$$

$$v \simeq \widetilde{v} = \overline{v} + \Phi^{v} \boldsymbol{a},$$

$$w \simeq \widetilde{w} = \overline{w} + \Phi^{w} \boldsymbol{a},$$

$$\alpha \simeq \widetilde{\alpha} = \overline{\alpha} + \Phi^{\alpha} \boldsymbol{a}.$$

One advantage of computing a global POD basis with $\boldsymbol{U} = (u, v, w, \alpha)^T$ is that the same temporal coefficients \boldsymbol{a} are used for each component.

Note that for out-of-sample computation, *i.e.* for $t \neq t_i$ or $\eta \neq \eta_i$ (different operating conditions), the temporal coefficients \boldsymbol{a} have to be computed using an external model. Usually, a projection-based reduced order model is used, like a Galerkin or Petrov-Galerkin model. However, for the bi-fluid case studied here, the last term in equation (3) would lead to a fourth-order tensor that is not compatible with model order reduction. Hyper-reduction techniques [45, 46] can be in principle used to mitigate that problem. However these intrusive approaches are impossible in the context of a commercial software where the sources, and thus the numerical residuals, are not available. A modified and extended version of the Galerkin-free approach introduced in [37] is used and presented hereafter.

In what follows, the POD basis functions are numerically computed on a Cartesian mesh that corresponds to the coarser cell size present in the high-fidelity mesh (see Figure 2). If a body is present in the computational domain, mesh points lying in all possible body locations (during motion) are removed. Usually, a Cartesian box is removed. One solution is computed the POD basis functions only on union of the two gray domains in Figure 3. Indeed, the body is subjected to move only in the white domain.

2.4 Multi-fidelity hybrid model

2.4.1 General process

The multi-fidelity model proposed here couples a high-fidelity solver with a low-fidelity solver, where the Proper Orthogonal Decomposition (POD) temporal coefficients $\{a_i\}_{i=1}^{N_r}$ are obtained using localized information. This information is provided by high-fidelity solutions on subdomains, and possibly by imposed values at external local sensors. Let Ω_{lf} denote the subdomain where the low-fidelity model is used, Ω_{hf} denote the subdomain where the highfidelity model is used, and $\Omega_o = \Omega_{lf} \cup \Omega_{hf}$ represent the overlapping domain where localized information is exchanged between high-fidelity and low-fidelity models. Let $\{\boldsymbol{x}_i^S\}_{i=1}^M$ be the locations of M sensors, where some information $\hat{\boldsymbol{U}}_i = (\hat{u}_i, \hat{v}_i, \hat{w}_i, \hat{\alpha}_i)^T$ is given (at least partially). For some applications, we may have M = 0. A sketch of the domain decomposition is provided in Figure 3.



Figure 3: Domain decomposition for the coupling methodology. The overlapping domain Ω_o is dark-gray colored. The high-fidelity domain Ω_{hf} is the union of the white and dark gray domains, and the low-fidelity domain Ω_{lf} is the union of the light and dark gray domains.

The main requirements for the high-fidelity domain are as follows:

- It should be as small as possible to reduce computational costs.
- It should be located in regions where the POD representation is not accurate enough, such as highly nonlinear regions, especially in the vicinity of structures like floating bodies.
- It should surround all possible unsteady locations of the floater (if any).
- It should have at least $N_r M$ discrete points to obtain a well-posed minimization problem for $\{a_i\}_{i=1}^{N_r}$ described hereafter.

The objective is to find the temporal coefficients $\boldsymbol{a}^* = (a_1^*, a_2^*, \cdots, a_{N_r}^*)^T$ such that the reduced solution $\widetilde{\boldsymbol{U}} = \overline{\boldsymbol{U}} + \boldsymbol{\Phi} \boldsymbol{a}$ is as close as possible to the high-fidelity solution \boldsymbol{U}_{hf} on the overlapping domain Ω_o , while also being close to the observed (and possibly imposed) values $\{\widehat{\boldsymbol{U}}_i\}_{i=1}^M$ at the M sensor points. Similarly to data $\widehat{\boldsymbol{U}}$, the field $\widetilde{\boldsymbol{U}}$ evaluated at a point-wise location \boldsymbol{x}_i^S is denoted $\widetilde{\boldsymbol{U}}_i$. Depending on the sensor locations, interpolations may be required to compute $\widetilde{\boldsymbol{U}}_i$.

Mathematically, at a given time t, the temporal coefficients a_i^* satisfy:

$$\boldsymbol{a}^{*} = \operatorname*{arg\,min}_{\boldsymbol{a}} \left(\frac{1}{|\Omega_{o}|} \int_{\Omega_{o}} \left(\boldsymbol{U}_{hf} - \widetilde{\boldsymbol{U}} \right)^{2} \mathrm{d}\boldsymbol{x} + \frac{\beta}{M} \sum_{i=1}^{M} \left(\widehat{\boldsymbol{U}}_{i} - \widetilde{\boldsymbol{U}}_{i} \right)^{2} \right), \quad \text{with} \quad \widetilde{\boldsymbol{U}} = \overline{\boldsymbol{U}} + \boldsymbol{\Phi} \boldsymbol{a},$$
(18)

where the parameter β is used to weight the sensor contributions. For low values of β , the Reduced Order Model (ROM) approximation, $\tilde{U} = \overline{U} + \Phi a^*$, is mostly influenced by the Full Order Model (FOM) solution U_{hf} . For large values of β , the ROM is mostly influenced by the sensor data \hat{U} , thus providing new information to the FOM. Values in between, $\beta \approx 1$, offer a compromise between FOM and sensor data contributions.

In what follows, we consider only the Volume of Fluid (VOF), $\hat{\alpha}_i$, as measurable by sensors. These data represent water elevation, which is usually easy to measure. The minimization problem (18) is thus:

$$\boldsymbol{a}^{*} = \operatorname*{arg\,min}_{\boldsymbol{a}} \left(\frac{1}{|\Omega_{o}|} \int_{\Omega_{o}} \left(\boldsymbol{U}_{hf} - \widetilde{\boldsymbol{U}} \right)^{2} \mathrm{d}\boldsymbol{x} + \frac{\beta}{M} \sum_{i=1}^{M} \left(\widehat{\alpha}_{i} - \widetilde{\alpha}_{i} \right)^{2} \right). \tag{19}$$

However, the following minimization problem:

$$\boldsymbol{a}^{*} = \operatorname*{arg\,min}_{\boldsymbol{a}} \left(\frac{1}{|\Omega_{o}|} \int_{\Omega_{o}} \left(\alpha_{hf} - \widetilde{\alpha} \right)^{2} \mathrm{d}\boldsymbol{x} + \frac{\beta}{M} \sum_{i=1}^{M} \left(\widehat{\alpha}_{i} - \widetilde{\alpha}_{i} \right)^{2} \right), \tag{20}$$

with $\tilde{\alpha} = \overline{\alpha} + \Phi^{\alpha} a$, provides better results and is used in the remaining part of this study. Each physical field used in the POD snapshots can finally be approximated as:

$$\widetilde{u} = \overline{u} + \Phi^u \boldsymbol{a}^*, \tag{21}$$

$$\widetilde{v} = \overline{v} + \Phi^v \boldsymbol{a}^*, \tag{22}$$

$$\widetilde{w} = \overline{w} + \Phi^w \boldsymbol{a}^*, \tag{23}$$

$$\widetilde{\alpha} = \overline{\alpha} + \Phi^{\alpha} \boldsymbol{a}^*. \tag{24}$$

2.4.2 Continuous multi-fidelity model

The multi-fidelity model finally writes:

$$\frac{\partial \boldsymbol{u}}{\partial t} + (\boldsymbol{u} \cdot \boldsymbol{\nabla})\boldsymbol{u} = -\frac{1}{\rho}\boldsymbol{\nabla}p + \frac{1}{\rho}\boldsymbol{\nabla} \cdot 2\mu D(\boldsymbol{u}) + \boldsymbol{g} \quad \text{in } \Omega_{hf},$$
(25)

$$\nabla \cdot \boldsymbol{u} = 0 \quad \text{in } \Omega_{hf}, \tag{26}$$

$$\frac{\partial \alpha}{\partial t} + \boldsymbol{\nabla} \cdot (\alpha \boldsymbol{u}) = 0 \quad \text{in } \Omega_{hf}, \tag{27}$$

$$u = \widetilde{u} = \overline{u} + \mathbf{\Phi}^u \mathbf{a}^* \quad \text{on } \Gamma_{hf}, \tag{28}$$

$$v = \widetilde{v} = \overline{v} + \mathbf{\Phi}^v \mathbf{a}^* \quad \text{on } \Gamma_{hf}, \tag{29}$$

$$w = \widetilde{w} = \overline{w} + \Phi^w a^* \quad \text{on } \Gamma_{hf}, \tag{30}$$

$$\alpha = \widetilde{\alpha} = \overline{\alpha} + \mathbf{\Phi}^{\alpha} \mathbf{a}^* \quad \text{on } \Gamma_{hf}, \tag{31}$$

$$\boldsymbol{a}^{*} = \arg\min_{\boldsymbol{a}} \left(\frac{1}{|\Omega_{o}|} \int_{\Omega_{o}} \left(\alpha_{hf} - \widetilde{\alpha} \right)^{2} \mathrm{d}\boldsymbol{x} + \frac{\beta}{M} \sum_{i=1}^{M} \left(\widehat{\alpha}_{i} - \widetilde{\alpha}_{i} \right)^{2} \right) \quad \text{with } \widetilde{\alpha} = \overline{\alpha} + \boldsymbol{\Phi}^{\alpha} \boldsymbol{a}.$$
(32)

Here, Dirichlet boundary conditions are used for both velocity and the VOF fields.

Note that the field reconstructions (28)-(31) actually extend over the entire domain Ω_{lf} (including the boundaries Γ_{hf}), thus providing a propagation model.

2.4.3 Discrete approach

The high-fidelity and low-fidelity solutions can be computed on different meshes. In this study, the high-fidelity solution U is computed on an octree grid refined near the interface, possibly coupled with an overset grid (when floating structures are considered). The low-fidelity solution \tilde{U} is computed on a uniform Cartesian grid, corresponding to the coarser level of refinement in the octree grid. Interpolations are thus required for equations (28)-(32), where information between subdomains has to be exchanged. Different interpolation functions are introduced:

- \mathcal{I}_{\downarrow} is an interpolator from the high-fidelity mesh (octree) to the low-fidelity mesh (Cartesian uniform). It is used, for instance, to numerically compute the integral over Ω_o (the first term in (32)).
- \mathcal{I}_{\uparrow} is an interpolator from the low-fidelity mesh to the high-fidelity mesh. It is used, for instance, to evaluate the Dirichlet boundary conditions (28)-(31) on Γ_{hf} , $u = \mathcal{I}_{\uparrow}(\widetilde{u})$ (similarly for v, w, and α). The property $\mathcal{I}_{\uparrow} = \mathcal{I}_{\downarrow}^{-1}$ is expected, however it might not be satisfied numerically.
- \mathcal{I}_s is an interpolator from the low-fidelity Cartesian mesh to sensor locations $\boldsymbol{x}_i^S \in \Omega_{lf}$, i.e., $\alpha_i = \mathcal{I}_s(\alpha)$. It is used to evaluate the second term in (32).

Equations (28)-(31) become:

$$u = \mathcal{I}_{\uparrow}(\widetilde{u}) \quad v = \mathcal{I}_{\uparrow}(\widetilde{v}) \quad w = \mathcal{I}_{\uparrow}(\widetilde{w}) \quad \alpha = \mathcal{I}_{\uparrow}(\widetilde{\alpha}) \quad \text{on} \quad \Gamma_{hf},$$
(33)

and equation (32) becomes:

$$\boldsymbol{a}^{*} = \operatorname*{arg\,min}_{\boldsymbol{a}} \left(\frac{1}{|\Omega_{o}|} \int_{\Omega_{o}} \left(\mathcal{I}_{\downarrow}(\alpha) - \widetilde{\alpha} \right)^{2} \mathrm{d}\boldsymbol{x} + \frac{\beta}{M} \sum_{i=1}^{M} \left(\widehat{\alpha} - \mathcal{I}_{s}(\widetilde{\alpha}) \right)^{2} \right).$$
(34)

The integral over the overlapping domain Ω_o is numerically computed on the low-fidelity Cartesian mesh for simplicity reasons. One can prefer to use the high-fidelity octree mesh, and the first term in equation (34) would be replaced by $\frac{1}{|\Omega_o|} \int_{\Omega_o} (\alpha - \mathcal{I}_{\uparrow}(\tilde{\alpha}))^2 d\boldsymbol{x}$.

2.4.4 Errors computations

In what follows, ξ can be a velocity component u, v or w, or the VOF α . Let ξ_{HF} be the reference high-fidelity solution computed on the large computational domain Ω_{HF} . The two following fields are defined from the coupling algorithm: The field ξ_{hf} is the high-fidelity solution computed on the small domain Ω_{hf} , and the field $\tilde{\xi}$ is the low-fidelity solution computed in domain Ω_{lf} . The relative errors are:

$$\varepsilon_{hf}^{\xi}(t) = \frac{\sqrt{\int_{\Omega_{hf}} (\xi_{HF}(\boldsymbol{x}, t) - \xi_{hf}(\boldsymbol{x}, t))^2 \mathrm{d}\boldsymbol{x}}}{\sqrt{\int_{\Omega_{hf}} \xi_{HF}(\boldsymbol{x}, t)^2 \mathrm{d}\boldsymbol{x}}},$$
(35)

$$\varepsilon_{lf}^{\xi}(t) = \frac{\sqrt{\int_{\Omega_{lf} \setminus \Omega_o} (\xi_{HF}(\boldsymbol{x}, t) - \tilde{\xi}(\boldsymbol{x}, t))^2 \mathrm{d}\boldsymbol{x}}}{\sqrt{\int_{\Omega_{lf} \setminus \Omega_o} \xi_{HF}(\boldsymbol{x}, t)^2 \mathrm{d}\boldsymbol{x}}}.$$
(36)

Solutions ξ_{HF} , ξ_{hf} and $\overline{\xi}$ are computed on different grids. It is, thus, more convenient to compute the error on the same grid, and we chose the uniform Cartesian grid used to computed the low-fidelity solution. Numerically, errors (35) and (36) are approximated by:

$$\widehat{\varepsilon}_{hf}^{\xi}(t) = \frac{\|\mathcal{I}_{\downarrow}(\xi_{HF}(t)) - \mathcal{I}_{\downarrow}(\xi_{hf}(t))\|_{\Omega_{hf}}}{\|\mathcal{I}_{\downarrow}(\xi_{HF}(t))\|_{\Omega_{hf}}},\tag{37}$$

$$\widetilde{\varepsilon}_{lf}^{\xi}(t) = \frac{\|\mathcal{I}_{\downarrow}(\xi_{HF}(t)) - \widetilde{\xi}(t)\|_{\Omega_{lf} \setminus \Omega_o}}{\|\mathcal{I}_{\downarrow}(\xi_{HF}(t))\|_{\Omega_{lf} \setminus \Omega_o}},\tag{38}$$

where $\|\cdot\|_{\Omega_i}$ stands for the discrete L_2 -norm on a domain Ω_i .

The total numerical error on the whole domain Ω_f can be approximated by $\tilde{\varepsilon}^{\xi}(t) = \tilde{\varepsilon}_{hf}^{\xi}(t) + \tilde{\varepsilon}_{lh}^{\xi}(t)$. However, note that $\tilde{\varepsilon}^{\xi}(t)$ is not equivalent to the relative L_2 error on Ω_{HF} . In what follows we will only consider temporal average error over a given horizon T, and we define:

$$\overline{\varepsilon}_{hf}^{\xi}(t) = \int_{T} \widetilde{\varepsilon}_{hf}^{\xi}(t) \,\mathrm{d}t, \qquad (39)$$

$$\overline{\varepsilon}_{lf}^{\xi}(t) = \int_{T} \widetilde{\varepsilon}_{lf}^{\xi}(t) \,\mathrm{d}t.$$
(40)

3 Parameter sensitivity analysis

As shown in the previous sections, various parameters must be selected, such as the number of Proper Orthogonal Decomposition (POD) modes or the sizes of the computational domains, Ω_{hf} and Ω_o .

In this analysis, we consider a simple dam break-like configuration without a body nor wave forcing (M = 0). This test case is computationally affordable and allows for indepth parameter sensitivity analysis. While other flow configurations may yield different optimal parameter values, this study serves to identify the parameters crucial to the coupling strategy.

A three-dimensional domain Ω_{HF} with dimensions $[L_X \times L_Y \times L_Z] = [10 \times 6 \times 0.4]$ m is considered. The small size of L_Z ensures reasonable computational costs.

The initial position of the water-air interface Γ_f is analytically defined by:

$$\Gamma_f^0 = H_w \, e^{-\frac{1}{2} \left(x + \frac{3}{2}y\right)^2} + H_s,\tag{41}$$

where parameter H_w determines the initial wave height relative to the sea level H_s at $x \to \pm \infty$. The height H_s is adjusted to achieve a desired water volume in the computational domain. Here, we consider $0 \text{ m} \leq H_w \leq 1 \text{ m}$ to obtain smooth waves commonly observed in the wave energy sector, where extreme waves are rare.

The overall configuration is depicted in Figure 4.



Figure 4: Sketch of the flow configuration with different computational domains. The initial condition (41) for $H_w = 0.6$ m and $H_s = 0$ m is shown by the blue line.

No-slip boundary conditions, u = 0 m/s, are enforced on the boundaries orthogonal to the x and y directions, while symmetry boundary conditions are applied to the boundary orthogonal to the z direction.

The objective of this section is to conduct a sensitivity analysis of various physical and numerical parameters. Numerical sensitivities pertain to mesh discretization for the Full Order Model (FOM) (Sec. 3.1), temporal coupling scheme (Sec. 3.2), number of snapshots N_s , and number of POD modes N_r (Sec. 3.3). Geometrical sensitivities involve the size of the small high-fidelity domain Ω_{hf} (Sec. 3.4) and the size of the overlapping domain Ω_o (Sec. 3.5).

Sensitivity analyses are conducted independently on each parameter while keeping all others fixed, potentially introducing biases in the analysis. To mitigate such biases, ideal scenarios will be considered for fixing the remaining parameters.

3.1 Mesh sensitivity analysis

The primary objective is to compute an accurate solution U_h of the Navier-Stokes equations (3)-(5) over the entire domain Ω_{HF} , necessary for the offline stage of Proper Orthogonal Decomposition (POD) computed in $\Omega_{lf} \subset \Omega_{HF}$.

Four different meshes $\{M_i\}_{i=1}^4$ are employed to discretize the domain Ω_{HF} spatially. In all cases, a hierarchical Cartesian (octree) mesh is utilized. Far from the air-water interface, the mesh is uniform in all three directions, i.e., $\Delta x = \Delta y = \Delta z = h$. Each mesh includes two levels of refinement near the air-water interface, with the initial position defined by $H_w = 0.6 \text{ m}$ and $H_s = 0 \text{ m}$ in equation (41). The simulation time-step is set to $\Delta t = 0.002 \text{ s}$. The characteristics of the different meshes are summarized in Table 1, where n_c^i , with $n_c^1 > n_c^2 > n_c^3 > n_c^4$, represents the initial number of cells for mesh M_i . Since Adaptive Mesh Refinement (AMR) is applied, the number of mesh cells may slightly evolve over time as the air-water interface moves. The average relative errors

$$\overline{\varepsilon}_{mesh}^{\xi} = \int_{T} \frac{\sqrt{\int_{\Omega_{HF}} \|\xi_{M_i} - \xi_{M_1}\|_2^2 \mathrm{d}\boldsymbol{x}}}{\sqrt{\int_{\Omega_{HF}} \|\xi_{M_1}\|_2^2 \mathrm{d}\boldsymbol{x}}} \,\mathrm{d}t,$$

for $i = \{2, 3, 4\}$ and $\xi = u, v, w, p$, and α , where integrals over Ω_{HF} are discretized on a uniform Cartesian grid that fits the coarser level of the coarser grid (M_4) , are also reported in this table.

Mesh	h(m)	n_c	$\overline{\varepsilon}^u_{mesh}$	$\overline{\varepsilon}_{mesh}^{v}$	$\overline{\varepsilon}^w_{mesh}$	$\overline{\varepsilon}^{\alpha}_{mesh}$	$\overline{\varepsilon}^p_{mesh}$
M_4	0.2	1.8×10^4	1.7e-3	6.6e-4	3.9e-2	1.8e-4	2.1e-5
M_3	0.1	$8.5 imes 10^4$	1.0e-3	3.3e-4	1.5e-2	1.1e-4	1.0e-5
M_2	0.05	$4.4 imes 10^5$	5.2e-4	1.7e-4	8.6e-3	5.0e-5	3.5e-6
M_1	0.025	$2.5 imes 10^6$	-	-	-	-	-

Table 1: Mesh characteristics and relative errors with respect to the finest mesh M_1 .

For each quantity $(u, v, w, \alpha, \text{ and } p)$, a first-order behavior is observed.

In the subsequent analysis, we will use the mesh M_2 with h = 0.05 m that presents a good trade-off between computational cost and accuracy.

3.2 Sensitivity of the multi-fidelity model resolution

Once the solution U_h is obtained on the M_2 mesh, the POD basis functions $\{\Phi_i\}_{i=1}^{N_r}$ can be computed in an offline process, and the coupling model (25)-(32) can then be integrated in time. The final goal is to compute, at each time step, the POD coefficients $\{a_i^*\}_{i=1}^{N_r}$.

The Navier-Stokes equations (25)-(27) are discretized in time using an implicit scheme. It is thus natural to impose the boundary conditions (28)-(31) at time $t^{(n+1)} = t^{(n)} + \Delta t$, and the minimization problem becomes a fully coupled non-linear problem, where all equations (25)-(32) have to be solved at the same time, *i.e.* the solution $U = (u, v, w, \alpha)$ and POD coefficients a^* must be solved simultaneously. This non-linear minimization can be expensive in the numerical viewpoint, and the goal is, thus, to compare the solution of the implicit coupling with a more simple explicit one, where boundary conditions (28)-(31) are provided in an explicit way, at time $t^{(n)}$. The minimization problem (32) (or (34)) for an explicit coupling reduces to a least square problem, where the N_r POD coefficients $\{a^*\}_{i=1}^{N_r}$ are computed from the known solution U obtained at the previous time step. As already mentioned, the number of discrete points N_p over the overlapping domain Ω_o should satisfy $N_p > N_r - M$. Then, the least square problem requires to compute N_r unknowns using N_p equations. In the algebraic point of view, the over-determined minimization problem (34) is approximated by $a^* = \arg \min(Aa - b)^2$, where $A \in \mathbb{R}^{N_p \times N_r}$ and $b \in \mathbb{R}^{N_p}$.

The first $N_p - M$ lines of $\stackrel{a}{A}$ and \boldsymbol{b} are built from the term

$$\frac{1}{|\Omega_o|} \int_{\Omega_o} \left(\alpha_{hf} - \widetilde{\alpha} \right)^2 \mathrm{d}\boldsymbol{x},$$

while the last M lines are built from the term

$$\frac{\beta}{M} \sum_{i=1}^{M} \left(\widehat{\alpha}_i - \widetilde{\alpha}_i\right)^2.$$

The POD coefficients are obtained solving the associated normal equation $A^T A a^* = A^T b$.

In order to avoid truncation errors in the POD expansion $(N_r \leq N_s)$, boundary conditions (28)-(31) are provided by the reference high-fidelity solution $\tilde{U} = U_{HF}$, pre-computed on the large domain for the same physical configuration (41). Indeed, this is equivalent to consider all the POD modes built from snapshots computed at all times $t^{(k)}$ without any truncation.

In the implicit version, conditions (28)-(31) are imposed using $\tilde{U}^{(n+1)} = U_{HF}^{(n+1)}$, while in the explicit version, the conditions (28)-(31) are imposed from $\tilde{U}^{(n)} = U_{HF}^{(n)}$. In both versions, due to possible different meshes used in large computational domain Ω_{HF} and in the small computational domain Ω_{hf} , interpolations are performed, and we have $\tilde{U}^{(k)} = \mathcal{I}_{\downarrow}(U_{HF}^{(k)})$, where k = n for the explicit version, and k = n + 1 for the implicit version. The boundary conditions imposed for the small computation domain (28)-(31) are thus $U^{(k)} = \mathcal{I}_{\uparrow} \circ \mathcal{I}_{\downarrow}(U_{HF}^{(k)})$. Unfortunately, on the numerical viewpoint, we do not have $\mathcal{I}_{\downarrow} = \mathcal{I}_{\uparrow}^{-1}$, and numerical errors are, thus, necessarily introduced, leading to a non-zero error even if appropriate implicit boundary conditions are applied. Moreover, due to interpolations on the velocity fields, the mass conservation may be locally violated, possibly leading to some pressure oscillations.

For the following numerical example, a high-fidelity solution U_{HF} (velocity, VOF and pressure) is first computed on the entire large domain Ω_{HF} presented in Figure 4 up to T = 3 s, and snapshots are extracted at each computational time step Δt , chosen to be fixed in this study. A high-fidelity solution U_{hf} is then computed in the small high-fidelity domain Ω_{hf} , where boundary conditions $U = \mathcal{I}_{\uparrow} \circ \mathcal{I}_{\downarrow}(U_{HF})$ are applied on Γ_{hf} .

From now, and until the sensitivity analysis of the size of the domain Ω_{hf} , we consider the case R6 that will be presented in Section 3.4. No overlapping region is need here.

Domain	Case		$ar{arepsilon}^u_{hf}$	$\bar{\varepsilon}^v_{hf}$	$\bar{\varepsilon}^w_{hf}$	$\bar{\varepsilon}^{lpha}_{hf}$	$ar{\epsilon}^p_{hf}$
	Implicit	$dt = 0.002 {\rm s}$	2.60e-5	1.69e-5	1.89e-3	5.78e-6	2.06e-6
Ω_{hf} –		$dt = 0.004 \mathrm{s}$	7.15e-5	5.13e-5	3.19e-3	1.49e-5	4.89e-6
	Explicit	$dt = 0.002 \mathrm{s}$	2.98e-5	1.91e-5	1.94e-3	7.58e-6	2.37e-6
		$dt = 0.004 {\rm s}$	8.67 e-5	5.93e-5	3.28e-3	2.23e-5	6.00e-6

Table 2: Relative errors of the multi-fidelity approach with two time steps, for explicit and implicit coupling.

Two cases are considered for this multi-fidelity coupling, a time step $\Delta t = 0.002$ s and a time step $\Delta t = 0.004$ s. For both time steps, the errors (39) (computed for the velocity, the volume fraction, and for the pressure) are comparable in the implicit and explicit cases, see Table 2. Due to the interpolations, the errors using implicit boundary conditions are not zero. Since implicit and explicit coupling give comparable errors, for sake of simplicity and efficiency, the explicit version with $\Delta t = 0.004$ s will be used in what follows.

For in-sample test, *i.e.* when same parameters in (41) are used in both Ω_{HF} and Ω_{lf} , the error (37) computed with exact explicit boundary condition gives a lower bound for the coupling methodology. The exact boundary conditions can be equivalently computed by POD using $N_r = N_s = N_i$, with N_r the number of POD modes, N_s the number of snapshots, and N_i the number of temporal iterations required by the high-fidelity solution. Extra errors will, thus, inevitably be added when considering reduced basis with $N_r \ll N_s \leq N_i$.

3.3 POD Sensitivity Analysis

The goal now is to perform sensitivity analysis of the number of retained POD modes N_r , and of the number of snapshots N_s . A numerical simulation is run in domain Ω_{HF} with $H_w = 0.60$ m and $H_s = 0$ m up to T = 3 s. For the sensitivity of the number of POD modes N_r , snapshots of $U_{HF} = (u, v, w, \alpha)^T$ are collected at each time-step $t_i = i \Delta t$ with $\Delta t = 0.004$ s, and we have $N_s = N_i = 750$. The POD is computed from modified snapshots $U = U_{HF} - \overline{U}$, where \overline{U} denotes the temporal average over the N_s snapshots.

 $U = U_{HF} - \overline{U}$, where \overline{U} denotes the temporal average over the N_s snapshots. The POD eigenvalue spectrum $\{\lambda_i/\lambda_1\}_{i=1}^{N_s}$ and relative information content $RIC(N_r) = \sum_{i=1}^{N_r} \lambda_i / \sum_{i=1}^{N_s} \lambda_i$ are plotted in Figure 5.



Figure 5: Eigenvalue spectrum (left) and RIC (right) with respect to the number of POD modes N_r .

The relative projection errors for $\xi = u, v, w$ and α over the domain Ω_{HF}

$$\overline{\varepsilon}_{proj}^{\xi}(N_r) = \int_T \frac{\sum_{i=1}^{N_i} \|\mathcal{I}_{\downarrow}(\xi_{HF}(t_i)) - (\overline{\xi} + \sum_{k=1}^{N_r} a_k^{proj}(t_i) \mathbf{\Phi}_k^{\xi}))\|_{\Omega_{HF}}}{\sum_{i=1}^{N_i} \|\mathcal{I}_{\downarrow}(\xi_{HF}(t_i))\|_{\Omega_{HF}}} \,\mathrm{d}t, \tag{42}$$

are plotted in Figure 6, where the temporal coefficients $a_k^{proj}(t_i)$ correspond to the POD eigenvectors. These errors traduce the filtering effect of the POD truncation, and quantify, thus, the error due to the truncated modes.



Figure 6: Evolution of the projection error ε_{proj} with respect to the number of POD modes N_r .

The trend of the errors plotted in Figure 6 follows the trend of the POD eigenvalues in Figure 5: the errors are decreasing with the number of POD modes.

In what follows, a sensitivity analysis of the number of retained POD modes, N_r , and the number of snapshots, N_s , is performed on the hybrid multi-fidelity model (25)-(27). The errors (37) and (38) are computed for velocity and VOF.

3.3.1 Sensitivity of the number of modes N_r

The sensitivity of the number of modes N_r is studied keeping constant the number of snapshots $N_s = 750$ over the entire domain Ω_{HF} . We recall that the goal of the hybrid fidelity coupling is twofold. The POD reduced order solution $\widetilde{U} = \overline{U} + \Phi a^*$ with the optimized coefficients a^* is used to give appropriate boundary conditions for the small high-fidelity domain Ω_{hf} , and also to propagate information through the large domain, $\Omega_{lf} \setminus \Omega_o = \Omega_{HF} \setminus \Omega_{hf}$.

From now, and until the sensitivity analysis of the size of the overlapping domain Ω_o that will be presented in Section 3.5, we consider $\Omega_o = \Omega_{hf}$.

Domain	Case	$\bar{\varepsilon}_u$	$\bar{\varepsilon}_v$	$\bar{\varepsilon}_w$	$\bar{\varepsilon}_{lpha}$
	$N_r = 5$	7.28e-4	6.53e-4	1.25e-3	1.10e-4
	$N_r = 10$	2.57e-4	1.88e-4	9.98e-4	4.75e-5
	$N_r = 20$	1.17e-4	8.09e-5	7.98e-4	2.60e-5
$\Delta \mathcal{L}_{lf} \setminus \Delta \mathcal{L}_{O}$	$N_r = 30$	7.63e-5	5.35e-5	7.09e-4	1.86e-5
	$N_r = 50$	6.01e-5	4.61e-5	6.33e-4	1.28e-5
	$N_r = 100$	5.71e-5	4.68e-5	5.95e-4	1.11e-5
	$N_r = 5$	1.36e-3	1.10e-3	1.06e-1	2.61e-4
	$N_r = 10$	4.42e-4	3.21e-4	2.11e-2	6.55e-5
Ω_{hf}	$N_r = 20$	3.34e-4	2.27e-4	2.91e-2	2.58e-5
	$N_r = 30$	1.91e-4	1.32e-4	1.08e-2	2.63e-5
	$N_r = 50$	1.49e-4	1.08e-4	7.81e-3	2.65e-5
	$N_r = 100$	1.16e-4	8.40e-5	4.86e-3	2.73e-5

Both errors (39) and (40) are reported in Table 3. One of the most important variable for

Table 3: Relative errors over the high-fidelity domain Ω_{hf} and over the remaining low-fidelity domain $\Omega_{lf} \setminus \Omega_o$.





the future wave energy converter application is the VOF α , and Table 3 shows that starting from $N_r = 20$, the VOF error is almost constant in the domain Ω_{hf} . The higher VOF POD modes are probably being filtered by the interpolator \mathcal{I}_{\uparrow} used to impose boundary conditions. Since all the other errors are still decreasing with N_r , we arbitrarily chose a slightly larger value, $N_r = 30$.

A comparison of the temporal evolutions of the first five POD coefficients $a_k^{proj}(t)$ obtained by projection (eigenvectors of the POD) and by the coupling strategy $a^*(t)$, is given in Figure 7. Good agreements are observed, validating, thus, the reduced order model obtained with the Galerkin-free coupling strategy.

3.3.2 Sensitivity of the number of snapshots N_s

The number of POD basis functions is $N_r = 30$. The projection errors with respect to N_s are shown in Table 4. For each variable a first order is observed: the error is divided by n, using n times more snapshots. We arbitrarily choose $N_s = 250$ in what follows.

N_s	Multiple of Δt	RIC (%)	$\bar{\varepsilon}^{u}_{proj}$	$\bar{\varepsilon}_{proj}^{v}$	$\bar{\varepsilon}^w_{proj}$	$\bar{\varepsilon}^{\alpha}_{proj}$
50	15	99.80	6.4e-4	2.6e-4	7.3e-3	2.7e-4
75	10	99.78	4.6e-4	2.0e-4	5.6e-3	1.9e-4
150	5	99.77	2.3e-4	1.1e-4	3.1e-3	9.7e-5
250	3	99.77	1.4e-4	6.4e-5	1.9e-3	5.8e-5
750	1	99.77	4.7e-5	2.1e-5	6.4e-4	1.9e-5

Table 4: Sensitivity analysis on the number of snapshots N_s .

3.4 Sensitivity of the size of high-fidelity domain Ω_{hf}

An important choice concerns the size of the small high-fidelity domain Ω_{hf} . We still consider here $\Omega_o = \Omega_{hf}$. Each configuration under consideration is noted R_D , and is parameterized with the ratio $D = \frac{|\Omega_{HF}|}{|\Omega_{hf}|}$. All the simulations previously shown used R_6 . We consider here two other sizes, namely a larger one with R_3 , and a smaller one with R_8 . The three cases are outlined in Figure 8. Table 5 shows the ratio of the number of mesh cells at initialization, $R_{n_{cells}} = \frac{n_{cells}^{\Omega_{HF}}}{n_{cells}}$. The relation is not linear, but depends on the form and position of Ω_{hf} , because the majority of the cells are concentrated around the free surface, where there is a mesh refinement.



Figure 8: Test cases for varying size of Ω_{hf} .

Case	$R_{n_{cells}}$
R_3	1.69
R_6	3.38
R_8	4.50

Table 5: Number of mesh cells ratio $R_{n_{cells}}$ for the three test case R_3 , R_6 and R_8 .

The errors (39) and (40) are displayed in Table 6. We can observe that the errors obtained for the R_3 and R_6 cases are almost the same, while the error for the R_8 case is larger. A good trade-off between computational time and accuracy seems to be the R_6 case. This case has been chosen in the previous sections and will continue to be used in what follows.

Domain	Case	$\bar{\varepsilon}^u$	$\bar{\varepsilon}^v$	$\bar{\varepsilon}^w$	$\bar{\varepsilon}^{lpha}$
	R_3	6.77e-5	3.92e-5	7.38e-4	1.42e-5
$\Omega_{lf} \setminus \Omega_o$	R_6	7.63e-5	5.35e-5	7.09e-4	1.86e-5
	R_8	1.34e-4	1.20e-4	7.86e-4	2.52e-5
	R_3	1.44e-4	1.41e-4	9.69e-3	2.08e-5
Ω_{hf}	R_6	1.91e-4	1.32e-4	1.08e-2	2.63e-5
	R_8	6.31e-4	3.09e-4	2.27e-2	4.88e-5

Table 6: Errors (39) and (40) for test cases R_3 , R_6 and R_8 .

3.5 Sensitivity of the size of the overlapping domain Ω_o

In the previous analysis, the overlapping region was chosen to be $\Omega_o = \Omega_{hf}$, which corresponds hereafter to the reference case 0. Three other frame-shaped domains Ω_o are also considered and are presented in Figure 9.



Figure 9: Overlapping domains Ω_o tested.

If no sensors are considered (M = 0), the minimization problem for a^* is well-posed if the number of mesh cells N_p included in Ω_o is $N_p \ge N_r$. Each configuration considered here satisfies this requirement.

The average relative errors computed in the small high-fidelity domain Ω_{hf} and in the low-fidelity domain $\Omega_{lf} \setminus \Omega_o$ are reported in Table 7. The errors obtained in case (c) are quite close to those obtained in case 0 (used until now). The errors obtained in cases (a) and (b) are one order of magnitude higher. Indeed, the most sensitive boundary conditions for Ω_{hf} are the lateral ones (left and right) where appropriate conditions have to be imposed, especially for the water level, numerically imposed by Volume of Fluid (VOF) α . A large overlapping zone is thus required in this region, and case (c) satisfies this requirement, and will be used in what follows.

Domain	Case	$\bar{\varepsilon}^u$	$\bar{\varepsilon}^v$	$\bar{\varepsilon}^w$	$\bar{\varepsilon}^{lpha}$
	0	7.61e-5	5.33e-5	7.22e-4	1.86e-5
$0 \downarrow 0$	a	2.23e-3	1.98e-3	4.45e-3	2.92e-4
$\Omega_{lf} \setminus \Omega_{o}$	b	1.81e-3	1.80e-3	2.47e-3	2.59e-4
	с	2.30e-4	2.06e-4	8.05e-4	2.83e-5
	0	1.85e-4	1.29e-4	1.14e-2	2.61e-5
Ω_{hf}	a	3.50e-3	3.63e-3	4.28e-1	3.67e-4
	b	2.48e-3	2.32e-3	1.39e-1	5.12e-4
	c	3.09e-4	2.49e-4	1.17e-2	4.73e-5

Table 7: Relative errors with respect to the overlapping domain.

3.6 Sensitivity of the sampling for out-of-sample prediction

The final configurations resulting from the previous sensitivity analysis are as follows:

- The M_2 mesh is used to discretize the large high-fidelity computational domain Ω_{HF} .
- The time step is $\Delta t = 0.004 \,\mathrm{s}$.
- An explicit coupling is used to exchange information between high and low-fidelity models.
- The number of snapshots is $N_s = 250$ (extracted uniformly with $\Delta t_s = 0.012$ s over T = 3 s).
- The number of retained POD modes is $N_r = 30$.
- The small high-fidelity domain Ω_{hf} corresponds to case R_6 .
- The overlapping domain Ω_o corresponds to case (c).

In the previous sections, the data collected from simulations with initial conditions using $H_w = 0.60$ m are used to build the POD basis functions for the coupled model that reproduces the simulation for the same initial conditions. In other words, we are performing an in-sample test, also called a reproduction problem. In this section, data from simulations based on different initial conditions H_w are collected and used in the training phase, and the coupling algorithm is evaluated for the simulation initialized with $H_w = 0.60$ m (case 0), which is not included in the database. Several training sets are considered (see Figure 10),



Figure 10: Quantitative (left) and graphical (right) definition of the variable parameter, H_w , for the out-of-sample tests. Case 0 is the in-sample test, taken as reference. Case E groups the snapshots from all the other cases, except Case 0.

combining different H_w . The chosen H_w are progressively farther from the target initial conditions (cases A, B, C, and D), or are all collected in a large dataset (case E). Note that we use $N_s = 250$ snapshots for each simulation, and, for instance, the database for case E

Domain	Case	$\bar{\varepsilon}_u$	$\bar{\varepsilon}_v$	$\bar{\varepsilon}_w$	$\bar{\varepsilon}_{lpha}$
	0	6.29e-5	4.23e-5	7.09e-4	1.74e-5
	A	1.09e-4	8.57e-5	1.27e-3	2.41e-5
	В	2.68e-4	2.32e-4	1.43e-3	4.84e-5
$\Delta \mathcal{L}lf \setminus \Delta \mathcal{L}_O$	C	3.54e-4	2.82e-4	1.47e-3	6.75e-5
	D	6.76e-4	5.56e-4	1.57e-3	1.18e-4
	E	3.21e-4	2.68e-4	1.30e-3	5.83e-5
	0	1.72e-4	1.19e-4	1.26e-2	1.37e-5
Ω_{hf}	A	3.43e-4	2.14e-4	2.29e-2	1.95e-5
	В	3.64e-4	2.48e-4	2.11e-2	5.14e-5
	C	4.50e-4	2.98e-4	1.66e-2	5.97e-5
	D	6.18e-4	4.94e-4	2.26e-2	2.02e-4
	E	4.58e-4	2.96e-4	2.04e-2	7.12e-5

is thus composed by 8×250 snapshots. However, for each case (0, A, B, C, D, and E), $N_r = 30$ POD modes are considered.

Table 8: Relative errors on the high-fidelity domain Ω_{hf} and on the remaining low-fidelity domain $\Omega_{lf} \setminus \Omega_o$ for the different sampling test cases.

Table 8 shows that the solution for the baseline configuration (case 0) can be well approximated using training sets that do not contain the in-sample snapshots. In particular, datasets from initial conditions closer to the target (case A) give good results. The relative errors increase when the training points are farther from the baseline configuration. The large dataset (case E) provides good results with an expected better robustness than the other cases A-D. Better sampling strategies can be considered, but it is out of the scope of this paper.

4 Wave energy converter

The goal is to numerically model a class of wave energy converters, where the energy harvesting devices are floating bodies with rigid motions driven by sea waves. Floating wave energy converters are typically deployed in deep waters, ideally located far from the shoreline. These devices are equipped with a mooring system and harness the oscillatory motion generated by the waves to produce energy [47]. They can exhibit varying degrees of freedom, with notable prototypes including point absorbers, overtopping devices, surface attenuators, and rotating mass devices.

We consider here a sphere floating on sea waves, the latter being imposed by M > 0sensors in the coupling strategy. The floating body is defined as a point absorber, i.e., the only degree of freedom is in the y direction (aligned with gravity). The objective is to predict the wave behavior and the motion of the floating body subjected to an imposed wave. The wave is defined with some characteristics W0 (period, height, and wavelength) defined in Table 9. The sketch of the domain decomposition is depicted in Figure 11. The boundary Γ_{hf} now has an extra part, orthogonal to the z direction (see top figure in Figure 11), and the green points represent the M sensors where information on the incoming wave is measured.

The initial computational domain, noted Ω_{sim} , expands to $85 \text{ m} \times 45 \text{ m} \times 10 \text{ m}$. The large size of this domain is necessary to take into account the water depth, the multiple wavelengths, and artificial forcing zones that avoid spurious reflections. A symmetry condition is used to reduce the computational burden of the FOM simulation, and is also kept in the coupled model, see Figure 11.

However, without loss of generality, we consider a smaller domain $\Omega_{HF} \subset \Omega_{sim}$ to reduce the memory footprint and CPU costs for POD basis functions. Even though it is necessary to compute the actual solution on Ω_{sim} , we only consider its trace on Ω_{HF} . Similar to the previous test case, the domain Ω_{HF} is $10 \text{ m} \times 6 \text{ m} \times 0.8 \text{ m}$. This allows us to consider more than one wavelength in the domain. However, the size of Ω_{HF} in the z direction is twice the



Figure 11: Domain decomposition for the sphere simulation.

one used in the sensitivity analysis section. This is necessary to numerically model threedimensional effects near the floating body. Moreover, the domain has only one symmetry plane, passing through the middle of the sphere. The dimension of Ω_{hf} also slightly changes. Even though the x and y dimensions still correspond to the R6 case, there is a modification in the z direction, with the creation of the new boundary (see top of Figure 11). Likewise, the overlapping region conserves the same size in the x and y directions, corresponding to test case (c) presented in the dedicated section, but a new area is added in the z direction. With the new dimensions, the volume ratio between Ω_{HF} and Ω_{hf} becomes 8, yielding a bigger reduction with respect to the test case used in the sensitivity analysis. The domain Ω_{hf} is $4 \text{ m} \times 2.5 \text{ m} \times 0.6 \text{ m}$, which represents 0.015% of the volume of the real simulation domain $(\frac{|\Omega_{sin}|}{|\Omega_{hf}|} = 6375)$.

Numerically, Ω_{HF} is composed of 413457 cells, Ω_{hf} is composed of 53703 cells, and Ω_o is composed of 38493 cells. For simplicity reasons, we use M = 3482 sensors covering a domain of dimensions $0.2 \text{ m} \times 2 \text{ m} \times 0.8 \text{ m}$, see figure 13.

An overset mesh is used for the sphere in the whole domain Ω_{sim} , and in the small domain Ω_{hf} . The POD basis functions are only computed on $\Omega_{lf} = \Omega_o \cup (\Omega_{HF} \setminus \Omega_{hf})$.

We denote by IS the In-Sample case where the POD basis is computed with database from the baseline condition W0, and by OOS the Out-Of-Sample case where the POD basis is computed with database from conditions W1 and W2. Waves W0, W1 and W2 are defined in Table 9.

Wave	Wave period (s)	Wave height (m)	Wavelength (m)
W0	2	0.2	6.25
W1	1.95	0.18	5.94
W2	2.05	0.22	6.56

Table 9: Wave characteristics.

The number of POD modes is always $N_r = 30$. Each simulation lasts T = 6 s, allowing an incoming wave to impact the floater. The snapshots are collected at $\Delta t = 0.012$ s, for a total of $N_S = 500$ for each simulation.

The computed errors are reported in Table 10. The errors obtained in the Out-Of-Sample (OOS) case are slightly larger than in the In-Sample (IS) case, but they are comparable. The evolution of the vertical translation of the body and the vertical force acting on it are plotted in Figure 12. For the IS case, the temporal evolution of the force and the position of the body is very close to the reference one (HF). For the OOS case, the difference between the predicted solution and the reference one is slightly larger, but the overall behavior is

Domain	Case	$\bar{\varepsilon}_u$	$\bar{\varepsilon}_v$	$\bar{\varepsilon}_w$	$\bar{\varepsilon}_{lpha}$
$\Omega_{lf}\setminus\Omega_o$	IS	2.40e-4	1.67e-4	4.35e-4	3.02e-5
	OOS	4.74e-4	3.39e-4	1.09e-3	5.23e-5
Ω_{hf}	IS	4.35e-4	3.08e-4	1.59e-3	4.38e-5
	OOS	4.80e-4	3.87e-4	1.62e-3	7.77e-5

good. It is thus possible to obtain good predictions for Out-Of-Sample cases with large CPU gains. Indeed, we recall that $\frac{|\Omega_{hf}|}{|\Omega_{sim}|} = 0.015\%$, allowing significant CPU savings.

Table 10: Errors for In-Sample (IS) and Out-Of-Sample (OOS) tests.



Figure 12: Evolution of the force acting on the body and the heave motion of the body.

Finally, a snapshot of the air-water interface (from the VOF variable α) with the spherical point absorber obtained with the multi-fidelity model is given in figure 13. The air-water interface is continuous across the whole domain, no spurious effects being observed near the overlapping zone.

5 Conclusion

The primary objective of this study was to efficiently model the behavior of Wave Energy Converters (WECs) when exposed to sea waves. This problem is intricate, as it involves the interactions between a floating structure and the fluid corresponding to actual sea waves. In this study, we considered a bi-fluid model for air and water, wherein the interface is embedded and modelled using the Volume Of Fluid (VOF) method. The Full Order Model (FOM), based one the Navier-Stokes equations with VOF and fluid-structure interactions, was solved using the commercial software STARCCM+. While the solution of the Full Order Model is accurate, it becomes prohibitively costly when applied to large-scale WEC problems. Indeed, modelling actual sea waves necessitates a significantly large computational domain to accurately generate and potentially dampen waves, particularly to prevent reflections.

In light of these challenges, Reduced Order Models (ROMs) are essential to save computational resources. In this study, we employed Proper Orthogonal Decomposition (POD)



Figure 13: Snapshot of the air-water interface (from the VOF variable α) with the spherical point absorber obtained with the multi-fidelity model. The green domain represents Ω_{HF} , the violet domains represent the total support of the minimization problem, given by both the overlapping domain (around the sphere) and the sensor locations, here taken over a region close to the inlet boundary (on the left). Only one half of the domain in the z direction is actually computed, the visualization in the whole domain is obtained by symmetry.

as the data compression reduction technique. However, classical POD projection-based (Petrov-)Galerkin ROM for bi-fluid flows poses compatibility issues with model order reduction. Its complexity, manifested in the form of a fourth-order tensor in the leading dynamical system, presents significant hurdles. To overcome this difficulty, hyper-reduction techniques can be used but are challenging or impossible to apply with commercial codes where the source code is not directly available.

To address these challenges, we have proposed a non-intrusive approach that does not require access to numerical residuals or mathematical models. We have developed a domain decomposition Galerkin-free approach coupled with sensor information to impose incoming waves. Our multi-fidelity model integrates POD ROM and FOM spatially, with FOM utilized near obstacles where viscous effects and non-linearities prevail, while a POD ansatz is employed for smoother flow regions.

Our coupling methodology simplifies to a least square minimization problem, making it easily applicable to simulations of moving bodies, including wave energy conversion.

We have began by considering a simple prototype, a point absorber sphere, but our methodology can easily be extended to more complex and full-scale WECs with multiple degrees of freedom. Furthermore, without altering the coupling strategy, additional details such as mooring or control systems can be incorporated for a more realistic simulation.

Parameter tuning, such as adjusting domain sizes or the number of POD modes, is problem-specific and can be tailored to strike a balance between accuracy and speed-ups. The accuracy is improved considering a large number of POD modes with a large highfidelity domain. Conversely, the speed-up can be increased considering a small high-fidelity domain coupled with few POD modes.

The approach has been validated for in-sample cases, where the POD subspace was trained using appropriate wave parameters. This represents a reproduction scenario, as we previously simulated this problem to obtain the POD modes. More notably, our method is also able to predict solutions for unseen parameters in out-of-sample simulations. The results maintain accuracy while significantly reducing CPU costs. This is achieved by leveraging a small high-fidelity domain, which represents only a fraction of the original domain used in training the POD modes. The efficiency of our hybrid fidelity model enables intensive simulations, making it suitable for optimization tasks and facilitating multiple-query simulations.

Our ongoing work includes addressing the optimal sensor location problem to further improve results. Additionally, we are exploring clustering techniques on snapshots to enhance efficiency. This involves reducing the POD offline stage while simultaneously improving the online stage by decreasing the number of POD modes for each cluster. This strategy can be viewed as a method for approximating a non-linear problem with piece-wise linear approaches.

Our ultimate goal is to model and optimize farms comprising multiple WECs. We envision achieving this through an algorithm that integrates multiple POD/FOM patches, augmenting the size of the minimization problem while retaining simplicity as a least-square problem.

Acknowledgments

The authors acknowledge on-premise HPC cluster resources made available by Optimad Srl for conducting the research reported in this paper. The authors acknowledge the support by European Unions Horizon 2020 research and innovation programme under the Marie Skłodowska-Curie Actions, grant agreement 872442 (ARIA).

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