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Transverse Particle Trapping Using Finite Bessel Beams based on Acoustic Metamaterials

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11	(Dated: November $6, 2023$)
12	We investigate the transverse trapping force acting on a small particle on/off the central axis
13	of zero-order finite Bessel beams. Combining the simulated fields with the Gorkov force potential,
L4	the transverse trapping behaviors on small objects are analyzed, and the reversal of the trapping
15	behaviors is recovered when varying the paraxiality parameter of the Bessel beam. The results prove
L6	the possibility of using axisymmetric Bessel beams to trap both dense and stiff particles as well as
L7	light and soft ones. The particles can be trapped at the maximum central pressure in the main lobe
18	and also the maximum pressure of the other transverse locations. A lossy acoustic metastructure
19	with the ability to decouple the phase and amplitude modulations is used to generate the desired
20	sound field, which is used to trap a foam ball as an example. Our research opens a promising avenue
21	to design and develop simplified acoustic tweezers.

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I. INTRODUCTION

Acoustic waves, as one of the most important forms in 23 the classical wave system, play a significant role in the 24 25 development of science and technology. When an object is placed in an acoustic field, there exists an acoustic 26 radiation force acting on it. The force originates from the 27 momentum transfer from the field to the object due to 28 the scattering, reflection, and absorption. The acoustic 29 radiation force acts throughout the object, but owing to 30 the overall momentum conservation, the radiation force 31 can be reduced to an integration of the time-averaged 32 momentum-flux tensor over an arbitrary closed surface 33 enclosing the whole object. [1] Acoustic radiation forces 34 ³⁵ have been studied for a long time in regards to the sound waves of different forms. [2–17] The trapping 36 behaviors generated by standing waves on small objects 37 (object sizes are much smaller than the wavelength) 38 have been well understood, [2] where relatively light 39 40 and soft particles (like droplets) are trapped to the ⁴¹ pressure maximum (anti-node), yet relatively dense and 42 stiff particles (like elastic objects) would be trapped 43 to pressure minimum (node). On the other hand, 44 the recent studies on acoustic radiation forces based $_{45}$ on non-diffracting beams [4–6, 14, 15, 18–20] have ⁴⁶ predicted the possibility for particle manipulations with 47 more flexibility. However, in practical situations and 48 applications, non-diffracting beams, such as acoustic

⁴⁹ Bessel beams, would be difficult to generate by a finite ⁵⁰ source due to relatively complex transverse structures. ⁵¹ Hence, it is significantly important to precisely generate 52 and control the desired sound field. The conventional ⁵³ method would use a speaker/transducer array connected 54 with a multi-signal generator, and then the desired sound 55 field could be generated by adjusting the signal for ⁵⁶ each speaker/transducer individually. [10] Sound field 57 manipulation based on such an active method has the 58 advantage of flexibility, yet as the complexity of the ⁵⁹ array system increases, the cost, and the complexity 60 of the signal processing will also increase. Hence, it 61 is desired to realize precise field manipulations with 62 simplicity and low cost. Recently, the emergence 63 and the rapid development of acoustic metamaterials 64 [21] open a field in acoustics, and also provide an 65 approach for field manipulations. [22–39] Acoustic ⁶⁶ metamaterials can achieve extraordinary properties and 67 phenomena, which natural materials would be difficult ⁶⁸ to achieve, such as negative refraction/reflection, [22, 23] ⁶⁹ cloaking, [40–44] hologram, [11, 36, 45–47] and specific 70 field generations. [22, 24, 32]

In this work, we investigate the transverse acoustic 71 72 radiation force generated by finite Bessel beams based 73 on acoustic metamaterials. The transverse trapping behaviors of a small particle on/off the central axis of the 74 75 Bessel beams are studied numerically and experimentally. ⁷⁶ A lossy acoustic metastructure, capable of decoupling 77 the phase and amplitude modulations, is designed to 78 generate the desired Bessel field, which is then used to ⁷⁹ trap a small foam ball to the beam center as an example. ⁸⁰ Our work gains insight into some unusual and unique ⁸¹ trapping behaviors, which will help the further study 82 of particle manipulations using acoustic radiation forces 83 and acoustic Bessel fields.

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FIG. 1. Illustration of transverse particle trapping by acoustic Bessel beams.

NUMERICAL SIMULATION II. 84

We consider here a particle within a finite 85 axisymmetric Bessel beam [Fig. 1], which can be 86 approximately considered as a non-diffracting beam 87 within a certain transverse radius and a certain 88 propagating distance. The pressure field within such 89 regions $\operatorname{Re}[p(\mathbf{r},t)]$ (Re denotes the real part) of the finite 90 zero-order Bessel beams propagating along a z axis is 91 92 given by,

$$p(\mathbf{r},t) = p_0 J_0(\mu\rho) \exp(i\kappa z - i\omega t), \tag{1}$$

⁹⁴ where p_0 is a real-valued amplitude, J_0 is zero-order Bessel function, $\mathbf{r}(\rho, \phi, z)$ is the field point in cylindrical 95 coordinates, and the transverse wavenumber μ and axial 96 wavenumber κ are related to the total wavenumber k =97 98 $\sqrt{\mu^2 + \kappa^2} = \omega/c_0$ (c_0 is sound speed in the surrounding ⁹⁹ media) through a paraxial parameter β with $\mu = k \sin \beta$, 100 and $\kappa = k \cos \beta$.

The force potential U is then obtained based on the 101 ¹⁰² simulated pressure p and velocity fields \mathbf{v} , [2] i.e.,

¹⁰³
$$U = (\pi a^3/3)[f_1|p|^2/(\rho_0 c_0^2) - (3/2)f_2\rho_0|\mathbf{v}|^2], \quad (2)$$

where a is the radius of the object, ρ_0 and c_0 are the mass 105 density and the sound speed of the background medium, 106 and f_1 and f_2 are the monopole and dipole factors 107 depending on the mass density ratio and bulk modulus 114 ¹⁰⁸ ratio between the object and the surrounding medium. ¹¹⁵ Bessel beam with $\beta = 40^{\circ}$ [see Fig. 2(a)]. 109 ¹¹⁰ the calculation since for most solid objects, the density ¹¹⁷ U [Eq. (2)] along the transverse location r/λ with λ ¹¹¹ and bulk modulus are much larger than the background ¹¹⁸ being the wavelength, and the black solid line shows the ¹¹² medium, air, so the results here are applicable to most ¹¹⁹ corresponding sound pressure. Particles will be trapped ¹¹³ elastic objects.



FIG. 2. Reversal of transverse particle trapping behaviors when reducing the angle from (a) $\beta = 40^{\circ}$ to (b) $\beta = 15^{\circ}$. Black solid lines: normalized squared pressure, $|p|^2$, at $z = 3\lambda$; red dots: normalized force potential U computed from Eq. (2) based on the simulated pressure and velocity fields. Insets: two-dimensional normalized squared pressure (color plots) and the corresponding radiation forces (white arrows). (c) Variation of the first trapping location r_0 near the beam axis. Transition angle happens at $\beta \sim 29^{\circ}$. (d) Spatial range R for approximated Bessel beams from a finite sound source. Red dots: simulated results obtained based on the -3dB location in the z direction; black line: curve fitting, $R \approx 0.9 H / \tan \beta$ with H being the half length of the finite source.

We first simulate the sound field of the finite zero-order The [15] Here, rigid particles with $f_1 = 1 = f_2$ were used in ¹¹⁶ red dots show the variation of the force potential[2] 120 to the local potential minimum, hence, for this case,

¹²¹ the rigid particle cannot be trapped to the pressure 122 maximum at the beam center (r = 0), which agrees with the previous conclusion from the standing fields. [2] In 123 addition, we find that for the off-axis transverse locations 124 (r > 0), the force potential minimum coincides with the 125 ¹²⁶ corresponding pressure minimum (or at most with a little ¹²⁷ shift), where the particles can be trapped. In particular, the nearest trapping location to the beam center for this 128 129 case is at $r_0 \sim 0.5\lambda$, which is useful for trapping particles [see Inset for the two-dimensional pressure profile (color 130 plot) and the radiation forces computed based on the 131 force potential, i.e., $\mathbf{F} = -\nabla U$ (white arrows)]. However, 132 when the angle β is reduced to be, for example, 15° 133 [see Fig. 2(b)], the trapping behavior becomes different. 134 Rigid particles will be instead trapped to the central 135 136 pressure maximum [see white arrows in Inset for the ¹³⁷ radiation force], and in addition, almost all the potential ¹³⁸ minimum along the transverse location corresponds to ¹³⁹ the pressure maximum, where the rigid particle can be ¹⁴⁰ trapped. Comparing Fig. 2(a) and (b), one can find ¹⁴¹ that for a large β [see (a)], there exist more potential ¹⁴² minima at the same transverse distance, so the spatial ¹⁴³ resolution for the trapping will be improved. In addition, ¹⁴⁴ the transverse radiation force for a large β would be larger due to the large gradient of the force potential. 145

the simulations Here, were conducted in a 146 two-dimensional axisymmetric model using the 147 148 commercial software, 149 50λ in the z direction. Radiation boundaries were used 181 the angle β for a finite source is roughly 150 to eliminate the reflection. Maximum mesh size was set 151 to be $\lambda/10$ to obtain precise and accurate sound fields. 152 The source is located at z = 0 and the background 153 medium is air with the density of 1.21 kg/m^3 and the 154 sound speed of 343 m/s. 155

We have numerically observed the reversal of the 156 157 158 which recovers the previous theoretical prediction for the 188 distance from the source. 159 ideal Bessel beams. [15] In order to examine the exact 160 transition angle β for finite Bessel beams, a series of 161 simulations were conducted for β from 10° to 70°. The 162 nearest trapping location to the beam center is examined 163 as a function of β [see Fig. 2(c)]. From the results, the 164 trapping location first increases and then decreases as the 190 165 angle β increases. The transition angle happens around ¹⁹¹ using acoustic metamaterials. 166 167 ¹⁶⁸ Fan and Zhang [15] within tolerance. The explanation of ¹⁹³ and the phase of the reproduced sound field by the 169 170 axial velocity to the force potential and by momentum 195 phase-only modulated metamaterials generally ignore the projection therein. 171

172 173 an infinite amount of energy. Hence, the approximated 199 preferred here. 174 Bessel beams only exist within a certain spatial range. 200 In this work, a lossy acoustic metastructure is used 175 176 177 spatial range R with the angle β [see red dots in 202 Fig. 3(a) for the cross-section of one element in the



(a) Illustration of one element of the artificial FIG. 3. structure. Parameters are chosen as $h = \lambda$, $D = 0.25\lambda$, $d = 0.8D, h_2 = 0.5\lambda$, and $h = h_1 + h_2 + h_3$. λ is the wavelength. By adjusting the geometry parameters w and h_1 , (b) the amplitude and (c) the phase of the reflected field could be modulated separately [Eq. (4)]. (d) The artificial structure used in the simulation for $\beta = 15^{\circ}$, and the corresponding geometry parameters w (black circles) and h_1 (red dots) for the 40 elements.

COMSOL MULTIPHYSICS. 179 β takes the location of -3dB along the propagating z The size of the computation domain is 10λ in radius and 180 direction. The relation between the spatial range R and

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$$R \approx 0.9 H / \tan \beta \tag{3}$$

¹⁸³ with H being the half length of the source. Based on 184 Eq. (3), the field can be approximately considered as 185 a Bessel field within the allowable spatial range R for transverse trapping behaviors for rigid particles on the $_{186}$ a given β and a source size H, and the Bessel beam central beam axis (r = 0) as the angle β is reduced, 187 with a small β could trap particles from a relatively long

EXPERIMENTAL REALIZATION III.

The desired acoustic Bessel beams could be generated There usually exist $\beta \sim 29^{\circ}$, agreeing with the predicted value $\beta \sim 28^{\circ}$ in 192 relatively complex couplings between the amplitude the reversal has been interpreted by the contribution of 194 sophisticated artificial structure, and the conventional ¹⁹⁶ error caused by the unavoidable amplitude variation. Like plane waves, an ideal Bessel beam cannot be 197 Hence, acoustic metamaterials with the ability of created since it requires an infinitely large source and also 198 decoupling the phase and amplitude modulations are

We examine the relationship between the approximated $_{201}$ to generate an acoustic Bessel beam with $\beta = 15^{\circ}$ [see ¹⁷⁸ Fig. 2(d)], where the spatial range R for each angle $_{203}$ structure]. [36] This structure can achieve the decoupled



FIG. 4. Particle manipulations using finite Bessel beams. (a) Measured and simulated fields generated by the metastructure with the results compared with an ideal continuous distributed source. Comparison of the field profile $|p|^2$ at $z = 12\lambda$ (white dashed lines). J_0 is the zero-order Bessel function. (b) Experimental photo of the transverse particle trapping using acoustic metamaterials, where a foam ball is trapped at the center of the Bessel beam. The size of the sample is $20 \times 20 \times 2$ cm³, and the diameter of the foam ball is ~ 2 mm. Inset: zoom-in photo of the sample. (c) Illustration of the experimental setup.

²⁰⁴ modulation of the amplitude A and the phase Φ , [36] i.e., ²⁴¹ 2 cm. The experimental diagram is shown in Fig. 4(c).

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$$A = \frac{d^4 - w^4}{d^4 + w^4}, \Phi = -\frac{4\pi h_1}{\lambda}.$$
 (4)

206 207 208 209 Bessel beams in our case. The viscosity effect of the ²⁵⁰ move freely in the transverse direction. 210 metastructure here can be ignored since the boundary 211 layer thickness is much smaller than the width of the 212 ²¹³ elements. [48, 49] The cross-section of the structure and ²⁵¹ the corresponding geometry parameters w and h_1 for the 214 40 elements are shown in Fig. 3(d), which would be used $_{252}$ 215 216

217 218 219 220 221 222 223 224 metastructure (green diamonds and blue circles) agree 261 generation of zero-order Bessel beams, which are then 225 pretty well with the ideal source of the same size (red 262 used to trap a small foam ball. Our work is beneficial to 226 dots), and both of them follow the squared zero-order 263 the further development of simplified acoustic tweezers 227 this structure, the zero-order Bessel beam is effectively 265 applications involving particle manipulations. 228 generated and can be utilized for particle manipulations. 229 As an example, the transverse particle trapping behavior 230 on a small foam ball (character size of ~ 2 mm) has ²⁶⁶ been examined and experimentally observed, where the 232 particle is transversely trapped to the center of the field, ²⁶⁷ 233 see Fig. 4(b). Here, the sample of the metastructure 268 Science Foundation of China (Grant No. 234 235 is fabricated using a 3D printer (Wiiboox Four) with 269 and No. 11974186), the Natural Science Foundation 236 the material PLA. The printing resolution is 0.1mm. A 270 of Jiangsu Province (Grant No. BK20220924 and No. ²³⁷ signal generator (RIGOL DG1032) and a power amplifier ²⁷¹ BK20200070), and the open research foundation of Key 238 (CROWN XLi2500) are connected to a loudspeaker with 272 Laboratory of Modern Acoustics, Ministry of Education, ²³⁹ a rated power of 100 W (Lutiane DL-630). The working ²⁷³ Nanjing University. $_{240}$ frequency is 17 kHz, corresponding to the wavelength $\lambda \sim _{274}$ X. F. and Y. Z. contributed equally to this work.

²⁴² The loudspeaker is fixed for a 45-degree incidence to the ²⁴³ sample. An absorption panel is set at a specific position 244 to separate incident and reflected acoustic fields. The ²⁴⁵ acoustic fields are scanned in the target region with a The decoupled modulation of the amplitude and the 246 step of 0.5 cm. Then interpolation process is made on phase allows for all the combinations in their full 247 the measured data. To better investigate the transverse parameter ranges [that is, $A \in [0, 1]$ and $\Phi \in [0, 2\pi]$; see ²⁴⁸ particle trapping, the foam is hanging by a light string Fig. 3(b) and (c)], which is necessary to generate acoustic 249 with a diameter of about 0.07 mm, and the particle can

CONCLUSION IV.

In summary, we have analyzed both on-axis and to generated an zero-order Bessel beam with $\beta = 15^{\circ}$. 253 off-axis particle trapping behaviors generated by finite The measured and simulated fields $|p|^2$ generated via 254 acoustic axisymmetric Bessel beams. The reversal of the acoustic metastructure are shown in Fig. 4(a). The 255 the trapping behaviors on the beam axis has been results are further compared with the field generated by $_{256}$ recovered when varying the paraxiality parameter β of an ideal continuously distributed Bessel source of the 257 the Bessel beam. In addition, the applicable spatial same size. We have examined the transverse profile of 256 range for the approximated Bessel beam from a sound the field at $z = 12\lambda$ (white dashed lines). From the ²⁵⁹ source of finite size has been quantitatively investigated. results, one can find the fields generated via the acoustic 260 A lossy acoustic metastructure has been used for the Bessel function (black solid line). Hence, with the aid of 264 using ordinary Bessel beams, which are desired for many

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