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Optimized and Robust Orbit Jump for Nonlinear Vibration Energy Harvesting

Camille Saint-Martin · Adrien Morel ·
Ludovic Charleux · Emile Roux · David
Gibus · Aya Benhemou · Adrien Badel

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Abstract This paper introduces an optimized orbit jump strategy for nonlinear Vibration Energy Harvesters (VEHs). Nonlinear VEHs are a promising alternative to linear VEHs due to their broadband characteristics. However, they exhibit complex dynamical behaviors, including not only high-power inter-well orbits but also low-power intra-well orbits and chaos. The existence of low-power orbits in their dynamics can restrict their energy harvesting performance. In order to overcome this issue, this study investigates an *orbit jump strategy* that allows the VEH to transition from low-power intra-well orbits to high-power inter-well orbits. This strategy, based on varying the buckling level of a bistable VEH, has been previously studied but not yet optimized. In this study, we define an optimization criterion that maximizes both the effectiveness and robustness of the orbit jump strategy. We developed a Python CUDA code using GPU parallel computing to handle the large number of numerical resolutions of the nonlinear VEH model. Experimental tests were performed on a bistable VEH over a frequency range of 30 Hz, validating the numerical results obtained with the optimized orbit jump strategy. The results indicate that the energy consumption required for a successful orbit jump ranges between 0.2 mJ and 1 mJ, and can be restored within 0.2 s in the worst case. Experimental results show an average success rate of 48%, despite a variation of $\pm 15\%$ in the starting and ending times of the jump, leading to a robust and optimized orbit jump strategy. The proposed optimization procedure can be applied to other orbit jump strategies, and other types of nonlinear VEHs.

Keywords Orbit jump, Optimization, GPU parallel computing, Buckling adjustments, Bistability, Nonlinear dynamics, Energy harvesting.

Camille Saint-Martin
Savoie Mont Blanc University, SYMME, F-74000, Annecy, France
E-mail: camille.saint-martin@univ-smb.fr

Adrien Morel
Savoie Mont Blanc University, SYMME, F-74000, Annecy, France
E-mail: adrien.morel@univ-smb.fr

1 Introduction

Energy harvesting is seen as a viable alternative to the use of batteries for supplying low-power electronic systems. The sources of energy that can be harvested are diverse and numerous, including solar radiations, fluid flows, electromagnetic waves, and mechanical vibrations. In particular, vibration energy is naturally ubiquitous even in confined environments with little solar and thermal energies available. This study focuses on energy harvesters that convert vibrational energy from ambient sources into electricity [1].

Vibration Energy Harvesters (VEHs) can be divided into two categories: linear VEHs, which rely on linear oscillators, and nonlinear VEHs that exploit nonlinear oscillators. Historically, linear VEHs have been studied because their behavior is easier to predict and because they can be more easily manufactured. However, linear VEHs have a narrow frequency bandwidth, and as a result, their energy harvesting performance drastically decreases when there is a mismatch between the driving frequency and their natural frequency [2, 3]. This makes linear VEHs unsuitable for applications with a time-varying spectrum, limiting their use in most environments. This has led to an increased interest in the development of nonlinear VEHs, especially bistable VEHs.

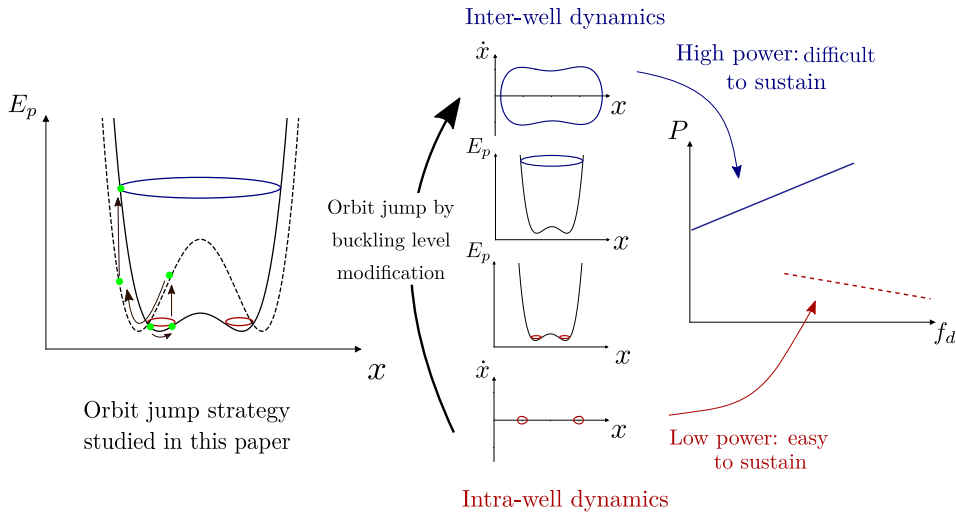


Fig. 1: Orbit jump strategy using buckling level adjustments of bistable VEH to switch from intra-well to inter-well orbits. Illustration inspired from [4].

The study of bistable VEHs started in 2008–2009 with the works of Shahruz *et al.* [5] and Cottone *et al.* [6]. Nonlinear VEHs have the advantage of exhibiting broadband behavior [7, 8], but their complex dynamics with multiple orbits can result in a drastic difference in power for a given driving frequency, as shown in Fig.1. Many studies have aimed to better understand the underlying dynamics of multi-stable energy harvesters [9–11] (for reviews, see [12–14]). In particular,

low-power intra-well orbits can lead to poor energy harvesting performance, which hinders the advantages of nonlinear VEHs, and is a major limitation of this type of energy harvester.

To enhance the performance of nonlinear VEHs, researchers have developed methods called *orbit jump strategies*. Orbit jump strategies enable nonlinear VEHs to transition from low-power orbits to higher power orbits, maximizing energy harvesting performance and exploiting their full potential (for review in multi-stable VEHs control, see [13] and for broader review of nonlinear dynamical system control, see [15]). The concept of orbit jump strategies in energy harvesting was first introduced a decade ago, with early studies conducted by Erturk *et al.* [16], Sebald *et al.* [7, 17] and Masuda *et al.* [18]. Erturk *et al.* [16] applied a “hand impulse” to impart enough velocity to a piezomagnetoelastic energy harvester, causing the nonlinear VEH to transition to the high-power orbit. To the best of our knowledge, this is the first experimentally and numerically demonstrated orbit jump strategy in the literature, using an additional velocity input to enhance the performance of nonlinear VEH. Sebald *et al.* [7] proposed a method, *Fast Burst Perturbation* (FBP), which consists in adding an external sinusoidal excitation during a few cycles. This perturbation is added to either the ambient excitation or the voltage of the electromechanical transducer in order to use the latter as an actuator (which is limited by the maximum amplitude that can be injected into the electromechanical transducer before it undergoes a dielectric breakdown). The authors validated the FBP method through numerical simulations [7] and experimental measurements [17]. Thereafter, Masuda *et al.* [18] investigated an orbit jump strategy by theoretically and numerically analyzing the variations in the load resistance value as a function of the displacement amplitude. They implemented negative resistance, acting as a negative damping, to destabilize low-power orbits during periods of low displacement amplitude. Once the nonlinear VEH stabilizes on a high-power orbit, with a large amplitude of displacement, the load resistance returns to its initial positive value. However, the study is limited to numerical simulations and requires further experimental validation. Moreover, this orbit jump strategy is only valid for a specific range of accelerations and frequencies.

Subsequently, as illustrated in Table 1, the manner in which the nonlinear VEH is perturbed permits the classification of orbit jump strategies into two distinct categories:

- (i) orbit jump strategies that add a temporary *external force* to the nonlinear VEH (e.g., a pulse on the voltage across the electromechanical transducer) [7, 16, 17];
- (ii) orbit jump strategies which involve temporarily *modifying the dynamic characteristics* of the nonlinear VEH (e.g., its damping or stiffness) [18].

Furthermore, subsequent studies have placed increased emphasis on analyzing the energy expenditure associated with orbit jump strategies, which is a critical factor to consider. Indeed, if the energy required to realize the orbit jump is not quickly recovered, the effectiveness of this approach is questionable. With regard to orbit jump strategies which introduce an external signal to disturb the system, Mallick *et al.* [19] used the FBP technique by superimposing a sinusoidal signal on the voltage across the electromechanical transducer over 15 cycles to use the transducer as an actuator.

External Forcing	Authors	Parameter or variable modified	Perturbation waveform	Validity range	Energy cost	Recovery time	Optimality
Hand impulse	Erturk <i>et al.</i> [16]	Velocity	 0.25 s*	Multiple freq. 6 – 8 Hz	N/A	N/A	✗
Fast Burst Perturbation	Sebald <i>et al.</i> [17]	Voltage	 0.7 s*	Multiple freq. 27.3 – 29.8 Hz	N/A	1.5 s*	✗
Impact-induced	Zhou <i>et al.</i> [20]	Velocity	 0.1 s*	Multiple freq. 4 – 23 Hz	N/A	N/A	✗
Electrical switching	Mallick <i>et al.</i> [19]	Voltage	 0.2 s	Single freq. 70 Hz	0.563 mJ*	1 s*	✗
Attractor selection	Udani <i>et al.</i> [4]	Voltage	 2 s	Single freq. 19.8 Hz	1.21 mJ	5.66 s	✓

Characteristic Modulation	Authors	Parameter or variable modified	Perturbation waveform	Validity range	Energy cost	Recovery time	Optimality
Negative resistance	Lan <i>et al.</i> [21]	Resistance	 0.1 s	Multiple freq. 9 – 11 Hz	0.2 mJ*	0.535 s	✗
Load perturbation	Wang <i>et al.</i> [22]	Stiffness and damping	 4 s	Single freq. 5.2 Hz	N/A	N/A	✗
Negative resistance	Ushiki <i>et al.</i> [23]	Resistance	 0.9 s*	Single freq. 70 Hz	35 mJ*	20 s	✗
Bidirectional Energy Conversion Circuit	Wang <i>et al.</i> [24]	Stiffness and damping	 10.9 s	Single freq. 7.6 Hz	22 mJ	120 s	✗
Buckling modification	Huguet <i>et al.</i> [25]	Buckling level	 20 ms*	Multiple freq. 30 – 70 Hz	1 mJ*	1 s	~
Voltage Inversion Excitation	Yan <i>et al.</i> [26]	Stiffness	 1.5 s.	Multiple freq. 48.6 – 49.5 Hz	1.43 mJ	23 s	✗
Adjustment strategy	Huang <i>et al.</i> [27]	Buckling level and voltage	 90 s*	Multiple freq. 35 – 40 Hz	2.8 mJ	120 s	✗

Table 1: Main properties associated to the both groups of orbit jump strategies defined in the current state of literature. * indicates that the values have been estimated based on the given papers. N/A denotes the absence of data.

They pointed out the effect of the phase shift between the ambient excitation and the resulting excitation¹ on the success of the orbit jump. That means, the success of the orbit jump strategy depends both on the nature of the perturbation and on the control of its timing. Their work is also among the first to consider the energy cost of the orbit jump strategy and gives the time needed to recover the energy consumed during the orbit jump (2 s), as shown in Table 1.

Udani *et al.* [4] added an artificial excitation to the ambient excitation creating a new excitation phase-shifted from the original ambient excitation. They demonstrated that the resulting modification of the dynamics and the basins of attraction of the orbits could facilitate escaping from the potential well. However, modifying the ambient excitation is not easy to implement in practice, limiting the applicability of their study. In a previous study [28], they developed a search algorithm in order to design an efficient *attractor selection strategy*. Notably, their approach was the first to search for the parameters of the perturbing signal that make their

¹ which is the effective excitation during the orbit jump, i.e., the FBP application on the ambient excitation (which is harmonic in this study).

strategy efficient.

On the other hand, a number of orbit jump strategies that involve temporarily modifying the nonlinear VEH's dynamic characteristics have been developed. Lan *et al.* [21] employed a method that emulates negative resistance using a negative impedance converter, similar to the approach taken by Masuda *et al.* [18]. They highlighted that the primary factor that disrupts the system is the increase in piezoelectric voltage resulting from negative resistance emulation, rather than damping modification, since the duration of orbit jump is brief.

Similarly, Ushiki *et al.* [23] defined a *self-powered stabilization* method using a negative impedance converter. Although they successfully destabilized low-power orbits across a frequency band of 23 Hz, the process of achieving positive energetic balance takes between 10 and 100 s, indicating that there is room for improvement in this aspect of the study.

Wang *et al.* [22] defined a *load perturbation* method based on the electrical load effects on the dynamics to attain high-power orbits. They disconnected the electrical load by opening a switch that was in series connection with the load and driven by an integrated circuit chip. This resulted in a reduction of the total damping rate of the VEH. However, this orbit jump strategy is only applicable for a specific combination of driving frequencies and amplitudes, making it non-robust and non-reproducible. Later, in order to decrease the energy injection of the orbit jump, Wang *et al.* [24] used a Bidirectional Energy Conversion Circuit (BECC) that includes the energy extraction circuit. The experimental test shows a jump duration of 10.9 s, which requires an energy of 22 mJ. The corresponding recovery time of 2 min suggests that the strategy could benefit from optimization.

Several recent studies have investigated the modification of the buckling level in nonlinear VEHs as illustrated in Fig.1. Huguet *et al.* [25] introduced the *buckling level modification* technique, using an additional electromechanical transducer² to alter the buckling level of the VEH. Their study demonstrated the effectiveness and reproducibility of the method through numerous experimental tests, computing jumping probabilities across six tested driving frequencies. Notably, this study also reported the first experimental jumps on sub-harmonic orbits in the literature. Furthermore, the study demonstrated that the energy consumed by the orbit jump strategy was quickly restored (in approximately 1 s), as shown in Table 1. Although this strategy has been partially empirically optimized, a complete optimization has not yet been conducted, which could further enhance its robustness and effectiveness.

Huang *et al.* [27] introduced a new *Voltage Inversion Excitation* (VIE) method, which reverses the voltage of the piezoelectric actuator at specific times to provide additional excitation to nonlinear VEH. However, this method consumes a significant amount of energy. To address this issue, they developed a more complex combination of two orbit jump strategies, which generally involve longer jump durations, as depicted in Table 1.

Yan *et al.* [26] used a stiffness modulation circuit to temporarily adjust the stiffness of a monostable softening VEH and experimentally demonstrated the VIE technique at 3 frequencies, which can be expanded to more frequencies.

Although there is a large pool of research on designing orbit jump strategies, in

² There are two electromechanical transducers, one for energy harvesting and the other for buckling level tuning.

general, very few strategies are optimized (i.e., a comprehensive optimization of the orbit jump parameters) in the literature as can be seen from Table 1. In most articles, the orbit jump parameters are determined through a preliminary numerical study or intuitive reasoning, rather than effective optimization. Furthermore, among these strategies, many have been experimentally validated over a limited frequency range (see Table 1), while they deserve to be tested over a wider frequency range in order to prove their reproducibility and robustness.

In this paper, we focus our analysis on the orbit jump strategy based on buckling level modification, as introduced in [25]. This orbit jump strategy stands out as one of the most promising strategies due to its short duration to achieve positive energy balance after the orbit jump (as shown in Table 1) and its ease of implementation, even though it has only been partially optimized experimentally. This paper presents the associated optimized orbit jump strategy and experimentally verifies its performance on a nonlinear VEH. To evaluate the performance of the orbit jump strategy, we performed experimental tests at starting and ending times of the jump with a variation of $\pm 15\%$ from 30 to 60 Hz. Specifically, we evaluate the strategy's robustness in terms of its average success rate, as well as its energy consumption during the jump. The proposed optimization method can be applied to any bistable VEH regardless of its energy conversion mechanism (e.g., with electromagnetic or electrostatic conversions). Figure 1 provides a summary of the orbit jump strategy process of this article and the motivations for applying an orbit jump strategy to nonlinear VEHs. This article is organized as follows: section 2 gives the electromechanical model of the bistable VEH and an overview of its dynamics. Then, section 3 presents the orbit jump strategy and its optimization based on a criterion which takes into account both the effectiveness and the robustness of the orbit jump strategy. Finally, section 4 presents experimental validation of the optimized strategy.

2 Electromechanical dynamics of bistable VEH

This section introduces the electromechanical model of a bistable VEH, along with a summary of the underlying dynamics with multiple behaviors, highlighting the interest of introducing orbit jump strategies.

2.1 Bistable VEH model

This paper studies a Duffing-type bistable VEH shown in Fig.2. This VEH (for more details on its design, see [29]) consists of buckled steel beams of length L to which a proof mass M is attached that can oscillate between two stable equilibrium positions, $-x_w$ and x_w . The VEH is driven by a sinusoidal excitation with a driving frequency $f_d = \omega_d/2\pi$ and a constant acceleration amplitude A . Two Amplified Piezoelectric Actuators (APA) are employed, with the smaller – *Energy harvesting APA* – having a force factor α , a clamped capacitance C_p , and the capacity to extract energy from the mechanical oscillator.

The electrodes of the energy harvesting APA are connected to a resistance R . The second and stiffer APA – *Tuning APA* – acts as an actuator to implement the orbit jump strategy by temporarily modifying the buckling level of the nonlinear VEH.

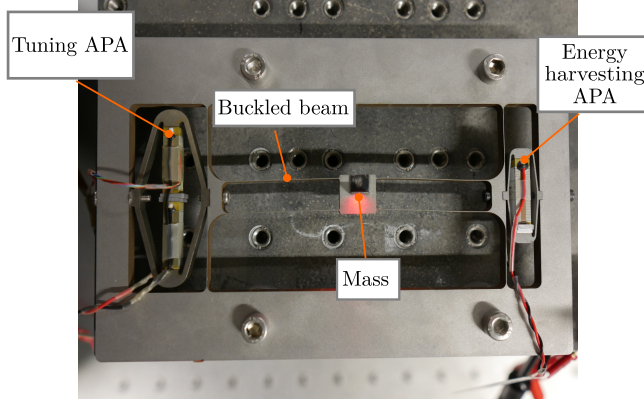
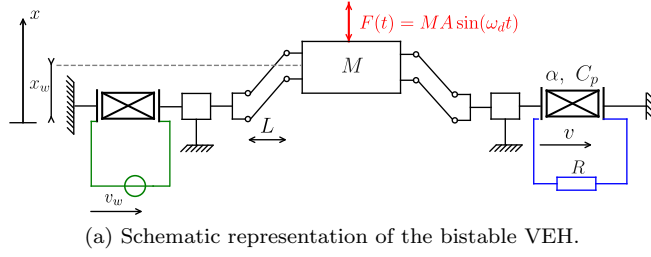


Fig. 2: (a) Schematic structure of the bistable VEH. (b) Experimental bistable VEH studied in this article from [29].

Therefore, this orbit jump strategy is classified as a type of nonlinear VEH characteristic modulation strategy. The energy harvesting APA is the APA120S, and the tuning APA is the APA100M manufactured by Cedrat Technologies (France). The model of bistable VEH [25] is given in equation (1),

$$\begin{cases} \ddot{x} + \frac{\omega_0^2}{2} \left(\frac{x^2}{x_w^2} - 1 \right) x + \frac{\omega_0}{Q} \dot{x} + 2 \frac{\alpha}{ML} xv = A \sin(\omega_d t) & (1a) \\ \frac{2\alpha}{L} x \dot{x} = C_p \dot{v} + \frac{1}{R} v & (1b) \end{cases}$$

Where x denotes the mass displacement, $\dot{x}$ its velocity and $\ddot{x}$ its acceleration. The voltage in the the energy harvesting APA is noted v . Note that the equations of the model (1) do not contain any term related to the tuning APA due to its higher stiffness compared to the harvesting APA, and thus does not have any significant influence on the dynamics of the VEH. The natural angular frequency ω_0 and the quality factor Q of the considered symmetrical bistable VEH are determined by the underlying equivalent linear model [30], which is obtained by considering small oscillations of the mass around one of its two stable equilibrium positions. The tuning APA voltage, denoted v_w , is used to modulate the buckling level of

the bistable VEH and facilitate transitions from low-power to high-power orbits. Table 2 shows the parameter values of the bistable VEH studied in this paper, which were determined experimentally through low-power orbit characterizations using weak sinusoidal vibrations. Note that for simplicity, we assumed that the force factor of the bistable VEH was the same as that of the energy harvesting APA.

Parameters	Values	Units
x_w	0.71	mm
M	6	g
ω_0	295	rad/s
Q	160	
α	0.139	N/V
C_p	1	μF

Table 2: Parameter values for the buckled-beam nonlinear VEH [29].

2.2 Bistable VEH behaviors

The dynamics of a bistable VEH may exhibit multiple behaviors for a given driving frequency, including low-power *intra-well* orbits, high-power *inter-well* orbits, and *chaotic* orbits. In this study, we define an orbit as *robust* if it is less sensitive to perturbations and easily attainable. In order to detect all possible behaviors in the frequency range of [20 Hz, 100 Hz] and $A = 4 \text{ m/s}^2$, the nonlinear Ordinary Differential Equations (ODEs) system (1) was solved for a large number of initial conditions using the Dormand-Prince method [31].

Since the nonlinear ODEs (1) can be solved independently across multiple resolutions, this problem is well-suited to parallel computing, that can greatly enhance computational performance. For this task, a custom Python CUDA code was executed on an NVIDIA RTX A5000 GPU featuring 8 192 CUDA cores, enabling the resolution of (1) with 80 000 distinct initial conditions for each driving frequency. In symmetric bistable VEH, the elastic potential energy is a quartic function of (t, x) whose expression is given in (2). Note that the natural angular frequency ω_0 depends on the value of x_w and will therefore be influenced during the orbit jump. The mean harvested power (for a given orbit) of the bistable VEH is the mean power dissipated in R and is expressed by (3),

$$E_p(t) = \frac{M\omega_0^2}{8x_w^2} (x + x_w)^2 (x - x_w)^2 \quad (2)$$

$$P_h = \frac{1}{T} \int_0^T \frac{v^2}{R} dt \quad (3)$$

where T is the period of the displacement x . Figure 3(a) shows the mean harvested power associated with existing orbits as a function of driving frequency f_d in [20 Hz, 100 Hz] when $R = 1/2C_p\omega_d$ for each driving frequency (which corresponds to the resistance value maximizing electrically induced damping [32] whose formula is valid for a harmonic excitation). Note that “Other” gathers sub-harmonic orbits and chaos [8,33]. Producing Fig.3(a) requires $80 \times 80\,000$ numerical computations, which can be completed in just a few minutes using parallel computing instead of the several hours required for sequential computing on CPU. It is worth mentioning that both power and existence of orbits vary with the driving frequency.

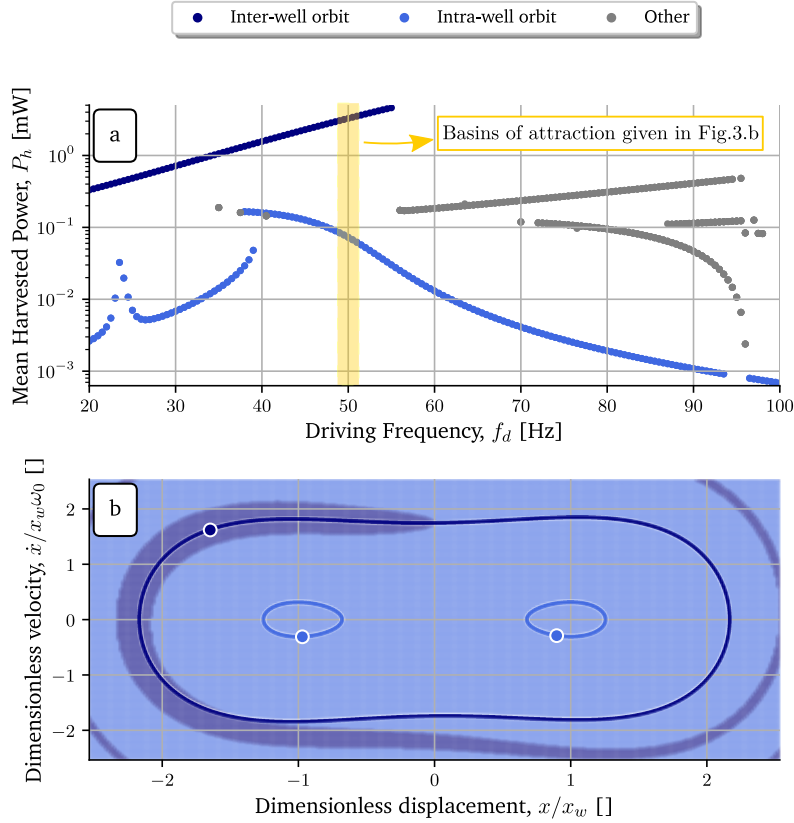


Fig. 3: (a) Mean harvested power P_h as a function of the driving frequency f_d , (b) inter-well (in dark blue) and intra-well (in light blue) orbits trajectories, their associated basins of attraction and attractor in the dimensionless phase plane $(x/x_w, \dot{x}/x_w\omega_0)$ for $f_d = 50$ Hz. The denomination “Other” (in gray) regroups all the orbits not indicated in the legend (i.e., sub-harmonic orbits and chaos). The basins of attraction in (b) were obtained after 80 000 resolutions of the ODE system (1).

As seen in Fig. 3(a), there exist multiple orbits for a given driving frequency with various power in the bistable VEH dynamics. As a matter of example, the high-power inter-well orbit allows to harvest 102 times more power than low-power

intra-well orbits for $f_d = 30$ Hz.

The high-power inter-well orbits see their power increases with the driving frequency while they stop existing beyond a particular frequency (the cutoff frequency). As given in Fig.3(a), the cutoff frequency of high-power inter-well orbits occurs at 55 Hz. The yellowish window in Fig.3(a) highlights the driving frequency $f_d = 50$ Hz whose *basins of attraction*³, orbits and *attractors*⁴ are plotted in the dimensionless phase plane $(x/x_w, \dot{x}/x_w \omega_0)$ in Fig.3(b). The surfaces of the basins vary with the driving frequency [33]: the high-power inter-well orbits see their basins surface decrease with the driving frequency. Figure 3(b) shows respectively basins of attraction of the intra-well (in light blue) and inter-well (in dark blue) orbits at $f_d = 50$ Hz. It is worth noting that the basin of attraction of the inter-well orbits becomes thinner as they approach their cutoff frequency. On the other hand, the power gap between intra-well and inter-well orbits becomes larger for frequencies near the cutoff-frequency of the inter-well orbit, as seen in Fig.3(a). Hence, as the driving frequency approaches the cutoff frequency of the inter-well orbits, it becomes increasingly difficult to attain inter-well orbits.

Thus, the primary challenges stem from:

1. there are multiple orbits with different harvested power for a given driving frequency;
2. the harvesting power gap between intra-well and inter-well orbits tends to increase with the driving frequency (particularly when $\omega_d > \omega_0$);
3. the inter-well orbits are less robust with larger driving frequencies (resulting in a narrowing of their basin of attraction).

The larger the power gap between intra-well and inter-well orbits, the greater the benefit in defining an orbit jump strategy. However, as the inter-well orbits become less robust with frequency, this task becomes increasingly difficult. All of these aforementioned difficulties are challenging to overcome and motivate the design of a robust orbit jump strategy in order to facilitate nonlinear VEHs to operate on high-power inter-well orbit as often as possible.

3 Orbit jump strategy: numerical modeling and optimization

This section introduces the orbit jump strategy [25] studied in this paper and its optimization using an evolutionary strategy algorithm.

3.1 Strategy description

The considered orbit jump strategy is based on the modification of the buckling level of bistable VEH. This strategy has already been studied and experimentally validated in multiple studies [25, 34], with promising results. In most of these studies, the ending time of the jump has been fixed to the instant when the mass reaches its maximum displacement which may not be the optimal time to minimize

³ which correspond to the initial conditions converging toward the considered orbit.

⁴ which correspond to the states (when $t = kT_d$, for $k \in \mathbb{Z}$) into which the bistable VEH converges when stabilized.

the energy cost of the orbit jump and maximize its robustness. Therefore, in this study, we chose to optimize this ending time. The orbit jump strategy adjusts the buckling level of the bistable VEH from x_w to $k_w x_w$ at a starting time t_0 for a (relatively short) duration Δt .

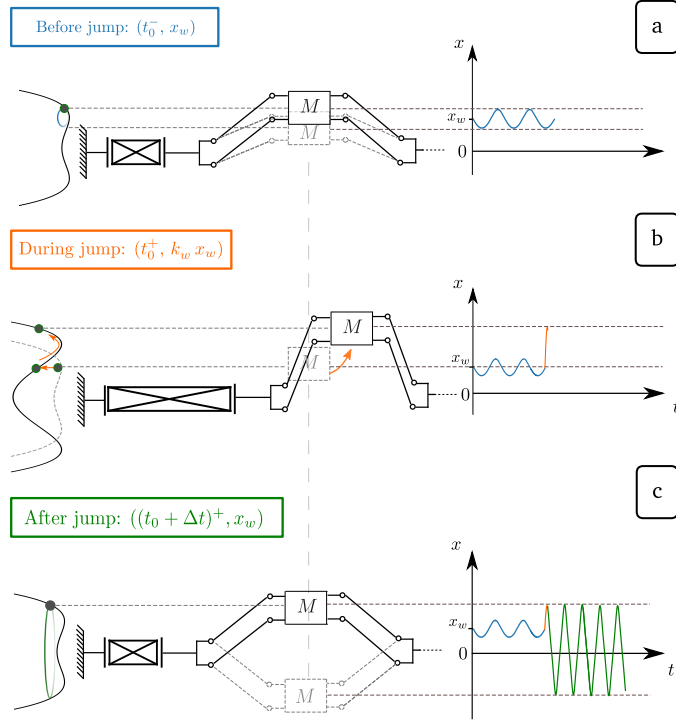


Fig. 4: Different steps of the aforementioned orbit jump strategy using buckling level modifications. From the left to the right: potential wells, the APA-mass system, and the displacement of the mechanical oscillator. Colored frames give corresponding equilibrium position and instant in the orbit jump strategy. It is worth noting that the motions in the central diagram are deliberately large in order to highlight the consequences of the variation of the buckling level on the mass.

Figure 4 illustrates the important steps of the aforementioned orbit jump strategy. For each step (before, during and after the orbit jump), the potential wells, the evolution of the tuning APA-mass system, and the displacement waveform of the mass are shown. As seen in Fig.4(a), at the beginning of the orbit jump process (when $t < t_0$, denoted t_0^- in Fig.4(a)) the mass oscillates around one of the two stable equilibrium positions at $x = x_w$ (low-power intra-well orbit oscillations). The gray point in Fig.4 illustrates the mass position in the potential well curve. Thereafter, between t_0 and $t_0 + \Delta t$ (when $t_0 \leq t \leq t_0 + \Delta t$, denoted t_0^+ in Fig.4(b)), the voltage of the tuning APA v_w changes and the buckling level

increases⁵ to $k_w x_w$ (with $k_w > 1$). It is worth noting that, while the buckling level theoretically increases instantaneously, a certain amount of time is required in practice. As seen in Fig.4(b), the potential well changes: equilibrium positions are greater ($x = \pm k_w x_w$) and the potential energy barrier is also larger. Thus, the gray point which was in the previous potential well (in gray dashed line) is now in a higher position, meaning that the inertial mass received potential energy during the buckling level modification. Finally, at $(t_0 + \Delta t)^+$ (that is, when $t > t_0 + \Delta t$), the initial buckling level is restored, reintroducing potential energy to the mass and setting both equilibrium positions back to $x = \pm x_w$. As illustrated in Fig.4, if the values of the orbit jump parameters $(t_0, \Delta t, k_w)$ are properly set, the bistable VEH should operate in its high-power inter-well orbit. For the sake of generality, we considered t_0 and Δt multiples of the driving period T_d and used the both following dimensionless times:

- $\tau_0 = t_0/T_d$, the dimensionless starting time;
- $\Delta\tau = \Delta t/T_d$, the dimensionless orbit jump duration.

Figure 5 shows an example of application of this orbit jump strategy for $f_d = 50$ Hz (we intentionally chose orbit jump parameters that make possible to jump on high-power inter-well orbit in order to illustrate the approach). Figure 5(c,d) shows the impact of orbit jump parameters $(\tau_0, \Delta\tau, k_w)$ on stable equilibrium positions and potential wells during the orbit jump strategy. Blue points in Fig.5(a,b,d) represent the instant when starting the orbit jump process, denoted by t_{ref} . Triangle up (resp. down) markers represent the instants when the buckling level of the bistable VEH increases (resp. decreases) at $t - t_{\text{ref}} = t_0$ (resp. $t - t_{\text{ref}} = t_0 + \Delta t$). As illustrated in Fig.5(d), when the buckling level is increased, then decreased, the inertial mass acquires potential energy that comes from the APA actuating system. This energy, called *invested energy* E_{inv} , consists in the potential energy (2) difference between t_0 and $t_0 + \Delta t$. As shown by (4), E_{inv} can be computed from the potential energy expression given by (2). The *total harvested energy* E_{tot} (5) is the invested energy subtracted from the harvested energy over a duration of $100 T_d$ from the instant t_{ref} . Note that we arbitrary take a duration of $100 T_d$ for the evaluation of the orbit jump strategy in the rest of the paper, as it is long enough to yield significant total energy if we successfully jump to a high orbit, while also being short enough to account for the invested energy during the orbit jump.

$$E_{\text{inv}}(t_0, \Delta t, k_w) = E_p[t_0^+] - E_p[t_0^-] + E_p[(t_0 + \Delta t)^+] - E_p[(t_0 + \Delta t)^-] \quad (4)$$

$$= \Delta E_0 + \Delta E_1$$

$$E_{\text{tot}}(t_0, \Delta t, k_w) = \int_{t_{\text{ref}}}^{t_{\text{ref}} + 100 T_d} \frac{v^2}{R} dt - E_{\text{inv}}(t_0, \Delta t, k_w) \quad (5)$$

As a matter of example, (4) and (5) allow to estimate the invested energy ($E_{\text{inv}} \simeq 1.27$ mJ) and the total harvested energy over a duration of $100 T_d$ ($E_{\text{tot}} \simeq 5.54$ mJ) in the orbit jump shown in Fig.5. The harvested power in high orbit (after the jump) is about 45 times larger than the power in low orbit (before the jump), for this driving frequency.

⁵ The elongation of the tuning APA leads to a higher level of buckling, resulting in the two equilibrium positions moving further apart from each other.

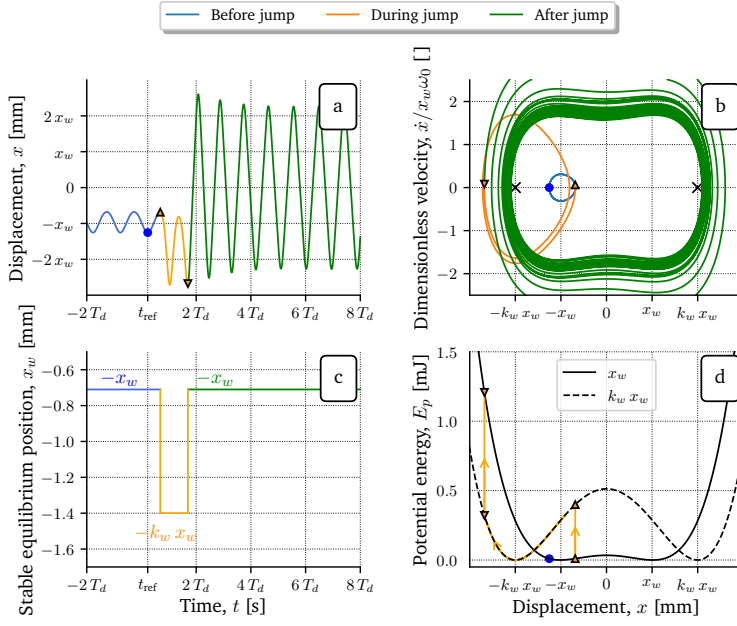


Fig. 5: Example of a successful orbit jump strategy for $f_d = 50$ Hz with $(\tau_0, \Delta\tau, k_w) = (0.46, 1.01, 2.00)$: (a) time displacement signal, (b) trajectories in the phase plane $(x, \dot{x}/x_w\omega_0)$, (c) evolution of the left stable equilibrium position before (in blue), during (in orange) and after (in green) the application of the orbit jump strategy and (d) the both elastic potential energy curves associated to the both buckling levels.

On the other hand, inherent experimental imprecision exists due to the non-ideal experimental setup (such as delays and parasitics effects) and imperfect experimental identification (with uncertainties regarding the values of ω_0 or x_w). All of these possible variations in parameters must be considered, which emphasizes the need to optimize the orbit jump strategy to reduce its sensitivity to parameter variations (i.e., its robustness) and enhance its performance. In the rest of the paper, we will investigate the optimization of the orbit jump strategy (for several driving frequencies) that can enhance its effectiveness based on its energy cost and its robustness against variations.

3.2 Optimization of the orbit jump strategy

As noticed in the previous subsection, the success of an orbit jump strategy depends drastically on the values of its parameters $(\tau_0, \Delta t, k_w)$ (which then depend on the driving frequency or the starting intra-well orbit for example). Properly defining both time parameters $(\tau_0, \Delta\tau)$ is crucial to the success of the orbit jump, regardless of the buckling factor (k_w). For example, setting the starting time of the orbit jump τ_0 to 0.9 renders the orbit jump strategy in Fig.5 ineffective, meaning that the VEh remains in low-power intra-well orbit even after the orbit jump. To

ensure an effective orbit jump strategy, we conduct a numerical investigation of the optimal values of the orbit jump parameters which:

- ($\mathcal{C}_1$) maximize the total harvested energy over 100 cycles, E_{tot} ;
- ($\mathcal{C}_2$) maximize the success rate of the orbit jump within a neighborhood of the orbit jump parameter values, with a variation of $\pm 15\%$.

The criterion ($\mathcal{C}_1$) allows to select orbit jump parameters that maximize the harvested energy while minimizing the invested energy. This criterion allows to evaluate effectiveness of the orbit jump strategy. While the criterion ($\mathcal{C}_2$) makes it possible to anticipate potential experimental deviations in the characteristics of the VEH or in the parameters of the orbit jump strategy. This criterion allows to evaluate robustness of the orbit jump strategy. Then, the optimization of the orbit jump parameters according both ($\mathcal{C}_1$) and ($\mathcal{C}_2$) criteria is performed by means of an evolutionary strategy algorithm [35] implemented in our in-house Python CUDA code. Evolutionary strategy algorithm has been selected due to its robustness in handling multi-extremal and discontinuous fitness functions, as well as its ability to benefit from GPU parallel computing. For that we define the *average total harvested energy* in (6) which is the fitness function⁶ to maximize,

$$\overline{E_{\text{tot}}}(\tau_0, \Delta\tau, k_w) = \sum_{i=0}^{N-1} E_{\text{tot}}(\tau_0^i, \Delta\tau^i, k_w^i) / N, \quad \forall N > 1, N \in \mathbb{N} \quad (6)$$

where $N > 1$ is the number of parameter combinations tested (e.g., $N = 8000$) and for all $i \in \llbracket 0, N-1 \rrbracket$, $(\tau_0^i, \Delta\tau^i, k_w^i) \in \mathcal{V}(\tau_0, \Delta\tau, k_w)$, the neighborhood of a given parameters combination $(\tau_0, \Delta\tau, k_w)$ with a variation of $\pm 15\%$. Therefore, the optimization problem to solve is formulated as (7).

$$\mathcal{S} : \max \left\{ \overline{E_{\text{tot}}}(\tau_0, \Delta\tau, k_w) \mid (\tau_0, \Delta\tau, k_w) \in \mathcal{D} \right\} \quad (7)$$

$$\text{where } \mathcal{D} = [0.2, 1.2] \times [0.2, 1.5] \times [1, 2]$$

Note that we only consider τ_0 and $\Delta\tau$ larger than 0.2 for the ease of experimental implementations. Moreover, the maximum mechanical constraints that can be supported by the considered prototype of bistable VEH have been taken into account by limiting the k_w to 2. $k_w > 1$ was chosen due to the prototype's reference buckling level x_w being close to the estimated minimum value, and optimization results showed no interesting solutions for $k_w < 1$.

The detailed optimization procedure is described in Appendix A. Figure 6 presents a comparison between the optimized and suboptimized 50 Hz orbit jump strategy (as shown previously in Fig.5). The optimized orbit jump strategy requires an invested energy of 0.49 mJ and yields a total harvested energy of 6.06 mJ over 100 oscillation cycles. It is worth noting that the end of the optimized orbit jump is defined slightly after the maximum displacement of the mass, in contrast to previous studies where the end time was generally defined at the instant of maximum displacement. The next section will investigate optimal orbit jump parameters combination $(\tau_0, \Delta\tau, k_w)$ satisfying (7) for [30 Hz, 60 Hz], and the optimization results will be presented jointly with the experimental results.

⁶ which is used for evaluating how close a given solution is to the optimum solution.

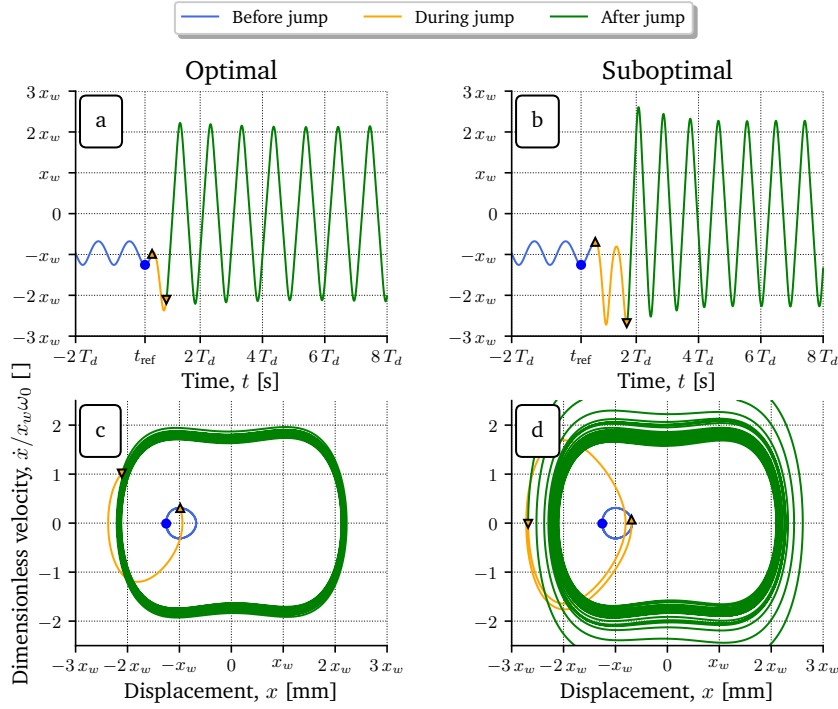


Fig. 6: Comparison between optimal (a,c) and suboptimal (b,d) of time displacement signals and trajectories in the phase plane $(x, \dot{x}/x_w \omega_0)$ for $f_d = 50$ Hz before (in blue), during (in orange) and after (in green) the application of the orbit jump strategy. Blue points correspond to the beginning of the orbit jump process, triangle up (resp. down) markers refer to the instant when the buckling level increased (resp. returned to its initial value). Optimal orbit jump parameter values $(\tau_0^{\text{opt}}, \Delta\tau^{\text{opt}}, k_w^{\text{opt}}) = (0.23, 0.46, 1.81)$. Suboptimal orbit jump parameter values $(\tau_0^{\text{sub}}, \Delta\tau^{\text{sub}}, k_w^{\text{sub}}) = (0.46, 1.01, 2.00)$.

4 Optimized orbit jump strategy: experimental validation and energy analysis

This section compares experimental and numerical results of the optimized orbit jump strategy.

4.1 Experimental validation

In order to experimentally validate the aforementioned optimized orbit jump strategy, experimental tests have been made around each optimal orbit jump parameters combination for driving frequency in $[30 \text{ Hz}, 60 \text{ Hz}]$. Figure 7 shows the experimental setup. The bistable VEH prototype shown in Fig.2(b) is fixed on an electromagnetic shaker driven by a power amplifier. The acceleration amplitude A of the shaker is measured by an accelerometer and sent to the control board. As

illustrated in Fig.7(b), the amplitude of the signal driving the power amplifier (v_A) is regulated in order to maintain a constant acceleration amplitude $A = 4 \text{ m/s}^2$ by means of an internal Proportional Integral (PI) controller. The piezoelectric electrodes of the energy harvesting APA (in blue) are connected to:

- a voltage follower in order to prevent the control board's impedance impacting the piezoelectric element and to avoid the control board to be exposed to a voltage strictly higher than 10 V which could damage it;
- a resistive decade box whose resistive value can be adjusted with a signal sent from the control board.

Displacement and velocity ($x, \dot{x}$) of the inertial mass are sensed with a laser differential vibrometer. At given times, when modifying the buckling level of the bistable VEH, the control board sends a signal to the high speed bipolar amplifier which controls the voltage across the tuning APA, v_w .

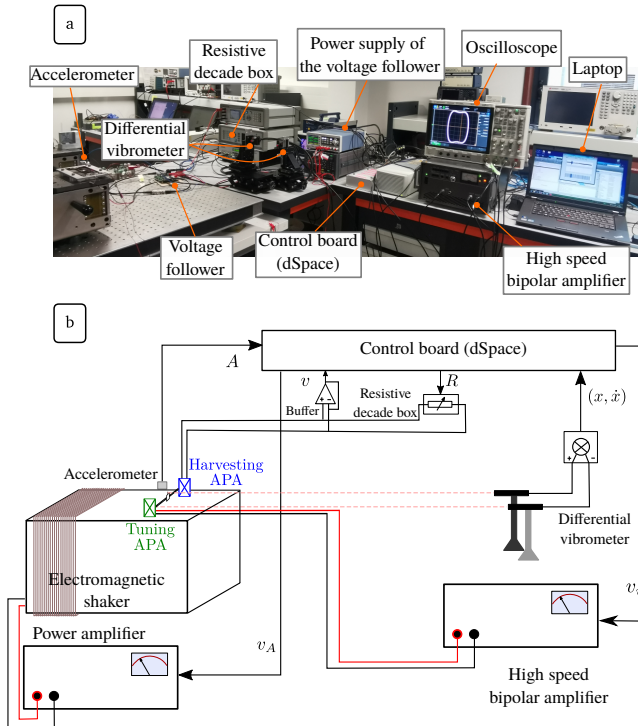


Fig. 7: (a) Experimental setup used to test the optimized orbit jump strategy and (b) its schematic representation.

In order to smooth the variation of the buckling level and avoid to damage the VEH prototype, we implemented in the control board a second-order filter that reduces the sharpness of v_w variations. The rise time of the buckling level is approximately one twentieth of a cycle, which is acceptable. Before any runs, the acceleration amplitude is gradually increased to $A = 4 \text{ m/s}^2$ and the buckling

level is decreased to obtain $x_w = 0.71$ mm. It is worth noting that the parameters of the VEH prototype have been identified in low-power orbit characterization and given in Table 2.

In order to experimentally validate the model described in equation (1) and the numerical orbit jump modeling, 2000 experimental results are launched with several values⁷ of τ_0 and $\Delta\tau$ ($k_w = 1.5, \tau_0 \in [0.2, 1.2], \Delta\tau \in [0.2, 1.5]$) for $f_d = 40$ Hz and arbitrary resistor $R = 20$ k Ω . Identical simulations are performed with 8000 parameters combinations.

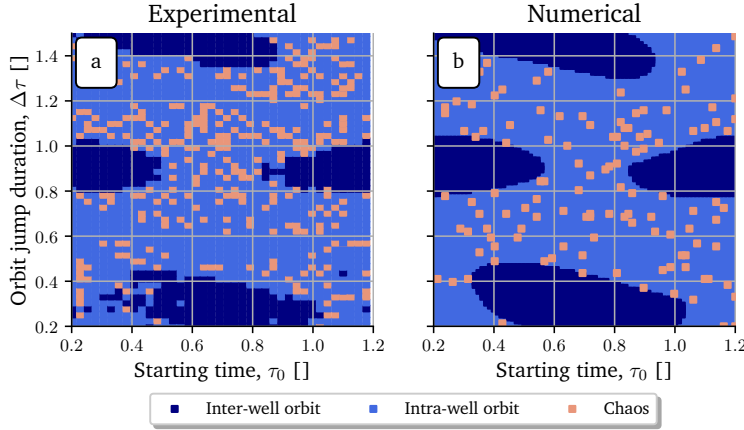


Fig. 8: Experimentally (a) and numerically (b) maps $(\tau_0, \Delta\tau)$ with $k_w = 1.5$, $f_d = 40$ Hz and $R = 20$ k Ω .

Figure 8 shows corresponding experimental and numerical scatter plots $(\tau_0, \Delta\tau)$ with each point associated to its final orbit after the jump and gives comparison between experimental and numerical structures of the basins for $f_d = 40$ Hz and $R = 20$ k Ω . As shown in Fig.8, there are three possible behaviors: low-power intra-well orbit (in light blue), high-power inter-well orbit (in dark blue) and chaos (in dark salmon). As an example, for experimental data in Fig.8(a), increasing the buckling level from x_w to $1.5x_w$ over a duration of $0.3T_d$ starting at $t - t_{\text{ref}} = 0.7T_d$ will result in the bistable VEH operating on the high-power inter-well orbit. The ranges of parameter values where the VEH jumps are approximately the same, although more chaos is observed experimentally. This may be attributed to an insufficient waiting time for the nonlinear VEH to reach steady-state conditions in the experimental setup. However, the experimental and numerical basins' structures given in Fig.8 are almost identical which validates the numerical model of the bistable VEH and numerical model of the orbit jump application. Additionally, Fig.8 shows the pseudo-periodicity of the inter-well orbit's basin in τ_0 (described with the two basins in the middle of Fig.8(a,b)). Therefore, since the starting orbit is T_d -periodic, the values of τ_0 can be restricted to a semi-open interval of length 1 without loss of information and justifies the values of τ_0 .

⁷ Note that we opted to fix k_w and to vary τ_0 and $\Delta\tau$ because the times are more susceptible to experimental variations due to the time delay of the board and the amplifier.

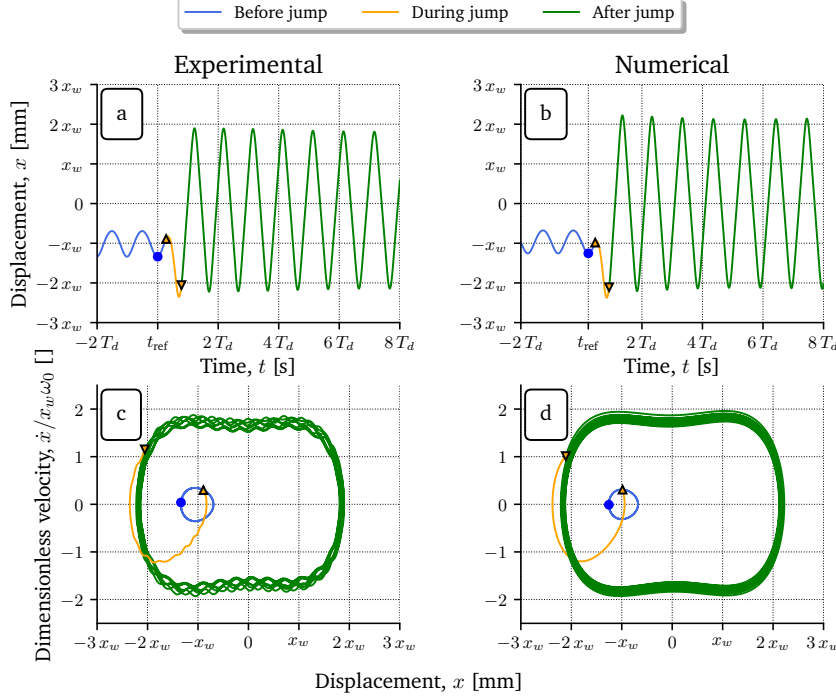


Fig. 9: Comparison between experimental (a,c) and numerical (b,d) of time displacement signals and trajectories in the phase plane $(x, \dot{x}/x_w \omega_0)$ for $f_d = 50$ Hz before (in blue), during (in orange) and after (in green) the application of the orbit jump strategy. Blue points correspond to the beginning of the orbit jump process, triangle up (resp. down) markers refer to the instant when the buckling level increased (resp. returned to its initial value). Experimental orbit jump parameter values $(\tau_0^{\text{exp}}, \Delta\tau^{\text{exp}}, k_w^{\text{exp}}) = (0.26, 0.44, 1.81)$. Numerical orbit jump parameter values $(\tau_0^{\text{num}}, \Delta\tau^{\text{num}}, k_w^{\text{num}}) = (0.23, 0.46, 1.81)$.

In order to validate the model of the bistable VEH and the orbit jump strategy effect, we perform experimental tests around the optimized orbit jump parameters (obtained with the evolutionary strategy algorithm introduced in section 3.2) for $f_d = 50$ Hz. Figure 9 compares experimental (Fig.9(a,c)) and numerical (Fig.9(b,d)) time displacement signals and trajectories in the phase plane respectively before, during, and after the application of the orbit jump strategy. As seen in Fig.9, experimental results are consistent with the numerical results. Experimental orbits are asymmetric, as shown in Fig.9(a,c), which can be attributed to mechanical irregularities resulting from the manufacturing process of the bistable VEH. Moreover, the experimental transient just after the jump in Fig.9(a,c) (in green) shows excitation from higher modes of the VEH prototype due to the quick buckling level variation. The corresponding experimental trajectory in the 3D plane $(t, x, \dot{x}/x_w \omega_0)$ is presented in Appendix B.

In quantitative terms, the mean harvested power is 26.5 times higher after the orbit jump in Fig.9, while the experimental invested energy required is equal to 0.65 mJ and can be recovered in 0.21 s. Then, we optimize the orbit jump strategy in the frequency range [30 Hz, 60 Hz] and obtain an optimal triplet $(\tau_0^{\text{opt}}, \Delta\tau^{\text{opt}}, k_w^{\text{opt}})$ satisfying the criterion $\mathcal{S}$ (7) for each driving frequency. Subsequently, we launch experimental maps around each optimal triplet and driving frequency in order to evaluate the robustness of the approach. It is worth mentioning that experimental maps are defined with 49 parameters values and a variation rate of $\pm 15\%$. Specifically, we take 7 values in $[0.85 \times \tau_0^{\text{opt}}, 1.15 \times \tau_0^{\text{opt}}]$ for τ_0 , 7 values in $[0.85 \times \Delta\tau^{\text{opt}}, 1.15 \times \Delta\tau^{\text{opt}}]$ for $\Delta\tau$ and $k_w = k_w^{\text{opt}}$.

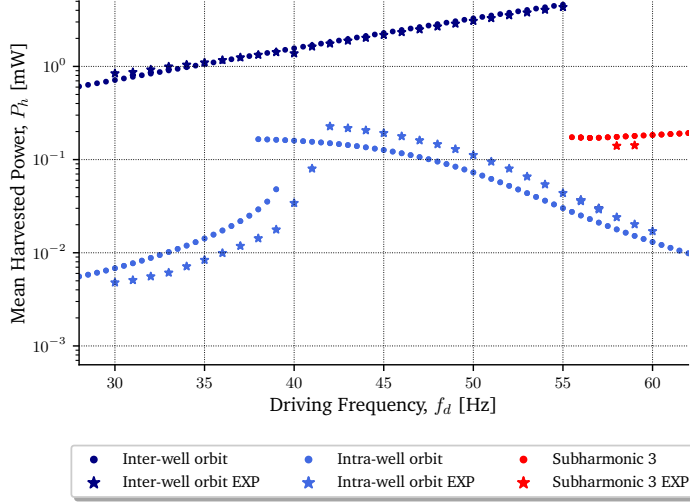


Fig. 10: Orbital mean harvested power P_h obtained numerically (points) and experimentally (stars) as a function of the driving frequency f_d for a sinusoidal excitation of amplitude $A = 4 \text{ m/s}^2$ with optimal resistor $R = 1/2C_p\omega_d$. The experimental data (stars) were obtained through the implementation of the optimized orbit jump strategy.

Figure 10 compares numerically (points) and experimentally (stars) mean harvested power (3) as a function of the driving frequency. Note that experimental power (stars) plotted in Fig.10 comes from experimental results of the orbit jump. The VEH starts in an intra-well orbit at each driving frequency. Then, the optimal jump is applied, and the power is measured in order to evaluate the inter-well orbit power. Through optimization of the orbit jump strategy, the highest orbit was achieved at each driving frequency both experimentally and numerically, as illustrated in Fig. 10. Differences between experimental and numerical data may result from the mismatch between the numerical model and our vibration harvester prototype. It is worth mentioning that applying an orbit jump strategy always yields a significant increase in power in this frequency range. As a matter of example, for $f_d = 55 \text{ Hz}$ the experimental mean harvested power of the intra-well orbit is 0.044 mW despite of 4.44 mW for the inter-well orbit leading to a power gain of

100 after a successful orbit jump. Then, as shown in Fig.(8, 9, 10), experimental results are consistent with numerical results and allow to validate the model and the proposed orbit jump strategy.

4.2 Energy harvesting performance analysis

Figure 11 shows the optimal orbit jump parameter combinations (blue points) and successful experimental parameters closest to the optimal (red stars) for each driving frequency. Note that the automated pre-characterization of the relationship between x_w and v_w allows the experimental determination of the modified buckling level $k_w x_w$ with good accuracy. As shown in Fig.11, experimental data with a variation of $\pm 15\%$ around the optimal times parameters and fixed $k_w = k_w^{\text{opt}}$ are in good agreement with the optimized data except for $f_d = 40$ Hz. The discrepancy between experimental and numerical models observed at $f_d = 40$ Hz can be attributed to the sudden change in behavior of the intra-well orbit due to softening nonlinearity of the potential wells for this particular acceleration amplitude (which equals 4 m/s^2), as seen in Fig.10. The experimental success rate associated with tested $(\tau_0, \Delta\tau)$ pairs, distributed with a variation of $\pm 15\%$, is shown in Fig.12. It is worth noting that despite the relatively large variation around the optimal times parameters, the average experimental success rate is about 48 %, which demonstrates the robustness of the optimized orbit jump strategy. However, the highest inter-well orbit ceases to exist beyond 55 Hz (both numerically and experimentally), resulting in the subharmonic 3 [8] becoming the highest inter-well orbit. Nonetheless, this orbit is challenging to reach and highly unlikely, leading to a decline in success rate between 55 Hz and 60 Hz.

Figure 13 shows invested energy (4) and total harvested energy (5) over 100 cycles as a function of the driving frequency for both successful experimental data and optimal numerical data. The dotted orange curve corresponds to the minimum difference in mechanical energy between the high-power inter-well orbit and the low-power intra-well orbit, $\Delta E_{\min}$, whose expression is given by (8).

$$\Delta E_{\min} = \min_{\forall t \in [0, T_d[} \left(\left[E_p(t) + \frac{1}{2} m \dot{x}(t)^2 \right]_{\text{inter-well}} - \left[E_p(t) + \frac{1}{2} m \dot{x}(t)^2 \right]_{\text{intra-well}} \right) \quad (8)$$

As shown in Fig.13, the invested energy required for the orbit jump does not exceed 1 mJ, even experimentally. Moreover, the invested energy associated with the optimal parameters is close to this minimum energy limit, validating the optimization strategy. Note that a portion of the electrical energy injected into the system is currently lost as electrostatic energy in the tuning APA. For example, at $f_d = 50$ Hz, the mechanical energy injected into the system is equal to 0.6 mJ, while the electrostatic energy lost in the tuning APA is 4.15 mJ. As a result, the total invested energy is 4.75 mJ, and the recovery time should not exceed 2 s. To address this issue, a power electronic converter could be used to store the lost electrostatic energy in the tuning APA and reintroduce it into the system at the appropriate time, although this approach was not implemented in this study.

As illustrated in Fig.13, the driving frequency increases, achieving a high-power orbit becomes more challenging. This leads to an increase in the amplification factor, the invested energy and the total harvested energy over 100 cycles.

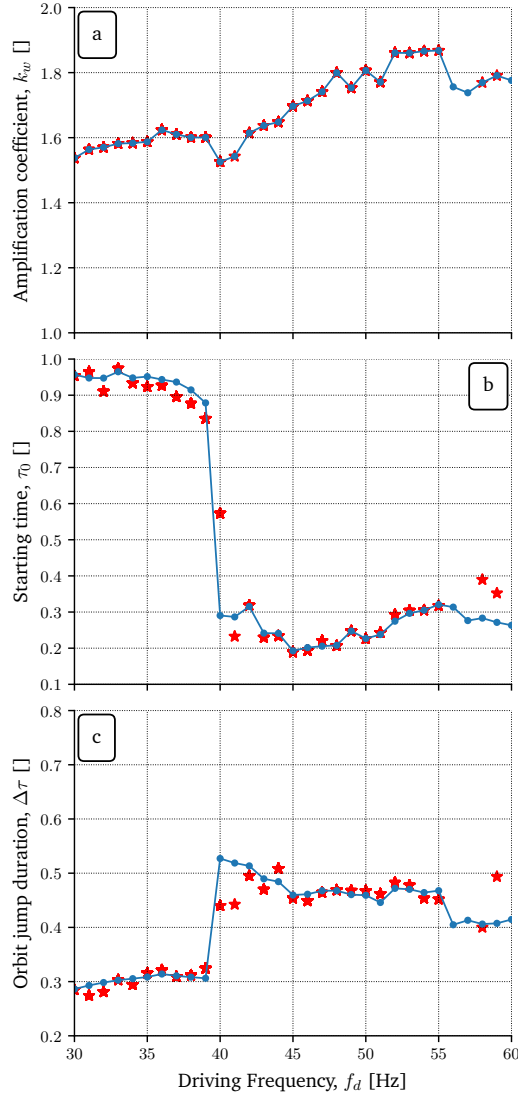


Fig. 11: The optimal numerical values (blue points) and the optimal experimental values (red stars) of (a) the amplification coefficient k_w , (b) the starting time τ_0 and (c) the orbit jump duration $\Delta\tau$ for successful orbit jumps as a function of the driving frequency.

The evaluation of the invested energy for orbit jumping is a major parameter for analyzing the quality of an orbit jump strategy. Additionally, the recovery time to achieve a positive energy balance allows the evaluation of the interest of jumping and the assessment of the cost-effectiveness ratio. Table 3 compares orbit jump strategies from the literature with results presented in this paper based on these aforementioned parameters (the jump duration, the invested energy and the recov-

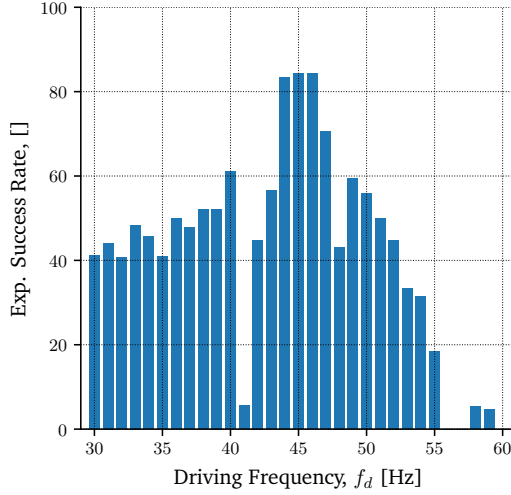


Fig. 12: The experimental success rate provided by the maps of $\pm 15\%$ around each optimal times parameter combination as a function of the driving frequency.

References	Frequency range	Jump duration	Energy cost	Recovery time	Robustness to parameters shifts	Optimized
Udani <i>et al.</i> [4] (2017)	Single freq. 19.8 Hz	2 s	1.21 mJ	5.66 s	No	✓
Wang <i>et al.</i> [24] (2019)	Single freq. 7 Hz	10.9 s	22 mJ	120 s	No	✗
Huang <i>et al.</i> [27] (2022)	Multiple freq. 35 – 40 Hz	90 s*	4.67 mJ	120 s	No	✗
Huguet <i>et al.</i> [25] (2019)	Multiple freq. 30 – 70 Hz	20 ms*	1 mJ*	1 s	Partial (Exp.)	~
This paper (2023)	Multiple freq. 30 – 60 Hz	8.3 ms	0.6 mJ	0.1 s	Yes (48 % robustness with a $\pm 15\%$ variation)	✓

Table 3: Comparison between the optimized strategy developed in this paper and other previous strategies in literature. * indicates that the values have been estimated based on the given papers.

ery time), but also whether they are optimized or robust to parameter shifts and if they were experimentally tested over a wide frequency range. It can be noted that very few strategies are optimized in the literature and that the only reference where a complete optimization of an orbit jump strategy has been considered (Udani *et al.* [4]) has only been tested for a single driving frequency, which does not validate its robustness, nor the generality of the optimization method. Wang’s *et al.* [24] strategy requires a high amount of energy which can be optimized. On the other hand, Huang *et al.* [27] have defined an innovative strategy that combines two other strategies (buckling level modification and VIE) and is therefore more complex. However, the jump duration is high (90 s), increasing the difficulties of implementation and decreasing both the robustness and efficiency of the strategy with a long recovery time equals to 120 s. Huguet *et al.* [25] introduced

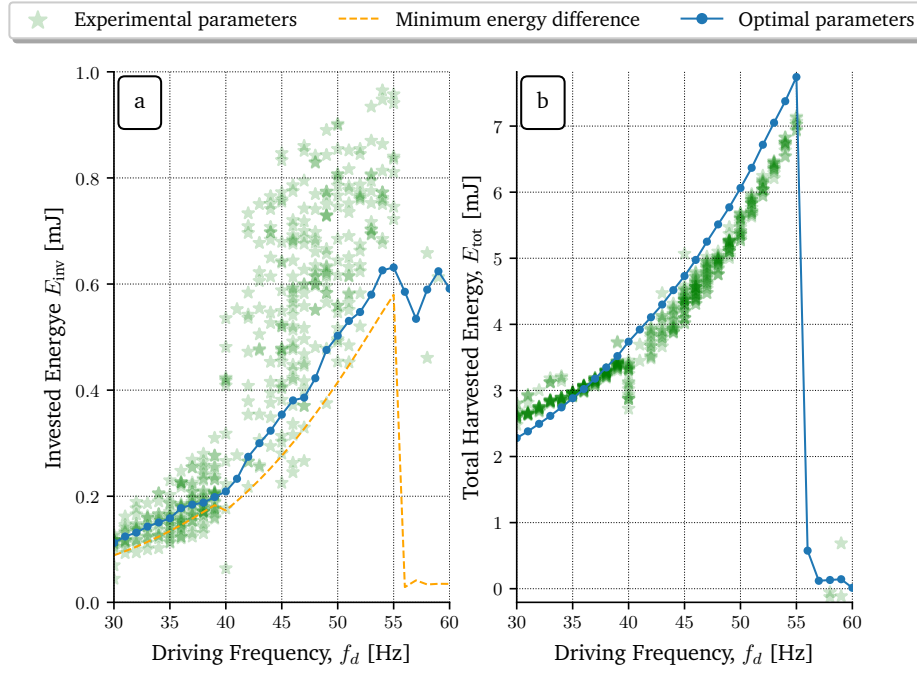


Fig. 13: (a) Invested energy and (b) Total harvested energy over 100 cycles for the numerical optimal orbit jump parameters (in blue dotted curve) and the experimental orbit jump parameters that allowed to jump (green star-shaped markers) as a function of the driving frequency with optimal load resistor. The dotted orange curve represents the minimum mechanical energy difference (8) between the highest inter-well orbit and the lowest intra-well orbit.

the strategy considered in this paper and examined two orbit jump parameters: the starting time of the jump (tested in 4 different values) and the amplification factor of the buckling level (tested in 6 different values). They fixed the ending time as the instant when the mass reaches its maximum displacement. However, using the optimization criterion defined in our study, results show that the optimal ending time for the jump occurs slightly after the maximum displacement. Nonetheless, their study has the merit of presenting numerous experimental trials, which allowed them to partially evaluate the robustness of the approach through statistical analysis of the jumps (with a 0% variation around each combination). The evolutionary strategy algorithm as well as the new optimization criterion proposed in this paper enable the achievement of performant orbit jumps, combining the shortest time duration (8.3 ms), lowest energy cost (0.6 mJ), shortest recovery time (0.1 s) while being robust to large parameters shifts ($\pm 15\%$ variation).

5 Conclusion

Due to the existence of low-power orbits in nonlinear VEHs dynamics, robust and effective orbit jump strategies are essential to ensure good energy harvesting per-

formance by enabling transition from low-power to high-power orbits. To achieve this, orbit jump parameters can be optimized. This paper presents the optimization of an existing orbit jump strategy using an evolutionary strategy algorithm. As a result of the development of an in-house Python CUDA code compatible with GPU computations, the optimization results are accelerated. The experimental results consistently demonstrate that the optimized orbit jump parameters generated high-power inter-well orbits, while maintaining their performance even under potential fluctuations in the VEH bistable environment. By considering the experimental amplification factor at its optimal value and times within a $\pm 15\%$ variation of the optimal times, the optimized strategy's robustness was demonstrated with an average success rate of 48% for the orbit jump. Finally, the energy required for the orbit jump does not exceed 1 mJ, even in experimental conditions. The proposed optimization strategy allows to enhance the confidence of an orbit jump despite fluctuations in the environment of the VEH. This generic approach can be applied to other types of multi-stable VEHs to design optimized orbit jump strategies. In future works, optimization of the final buckling level, which would add a fourth orbit jump parameter associated with the strategy, will be investigated.

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Conflict of interest

The authors declare that they have no conflict of interest.

Data Availability Statement

The datasets generated during and/or analysed during the current study are available from the corresponding author on reasonable request.

A Algorithm

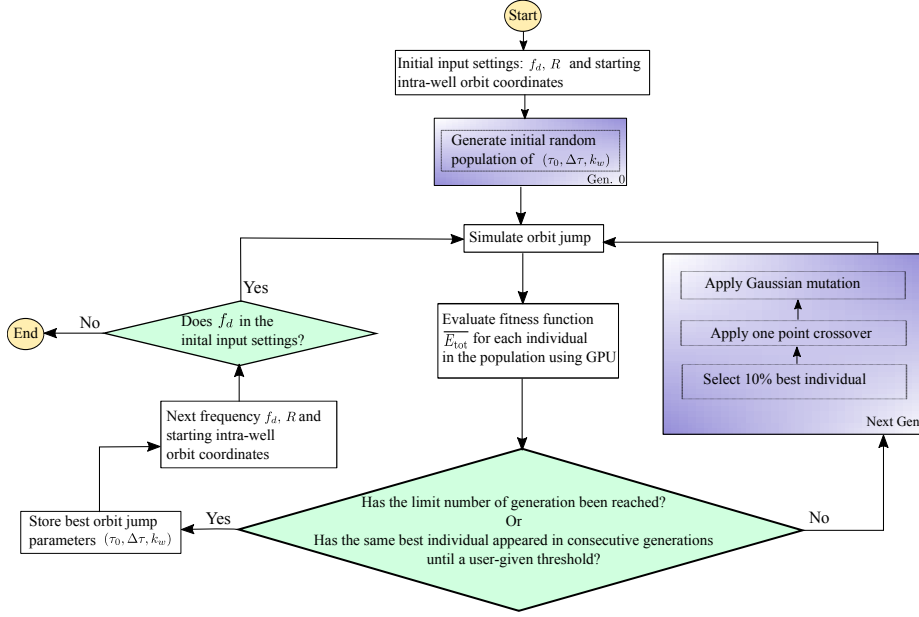


Fig. 14: Evolutionary strategy algorithm for optimizing orbit jump strategy. This flowchart illustrates different steps that allow to determine optimal orbit jump parameters for each driving frequency. Here $(\tau_0, \Delta\tau, k_w) \in \mathcal{D}$ denotes the individuals sequence for a given population in $\mathcal{D}$.

Figure 14 illustrates the various steps involved in the evolutionary strategy algorithm. The procedure starts with a first random generation of 8 000 orbit jump parameters combinations (i.e., individuals). Then, the simulation of the orbit jump associated with each set of parameters is performed, and each individual is evaluated based on its fitness function value (6). This involves considering 7 elements in the neighborhood of each direction, comprising τ_0 , $\Delta\tau$ and k_w . That is, we take $|I| = 7$ then for each individual we launch 7^3 new orbit jump simulations to compute its fitness function value (6). That means that for each generation, we simulate 2 744 000 orbit jumps using parallel GPU computations. Each individual is then evaluated and selected based on its average total harvested energy (6) over 100 cycles. To complete the population of the next generation, classical operations of crossing and mutations are applied to the 10% best individuals present in the next generation.

B Experimental trajectory

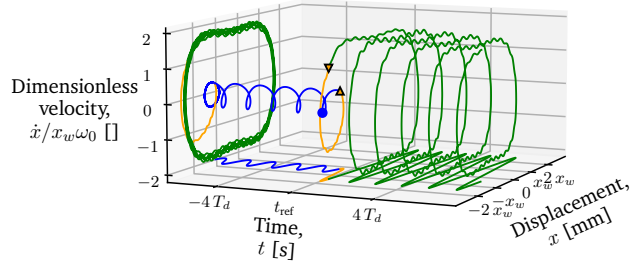


Fig. 15: Experimental trajectory in $(t, x, \dot{x}/x_w \omega_0)$ 3D plane for $f_d = 50$ Hz before (in blue), during (in orange) and after (in green) the application of the orbit jump.