

## Added viscous damping in a finite-size microperforated plate within a nonlinear framework

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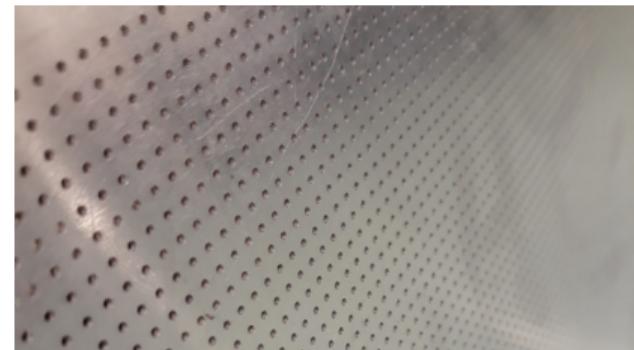
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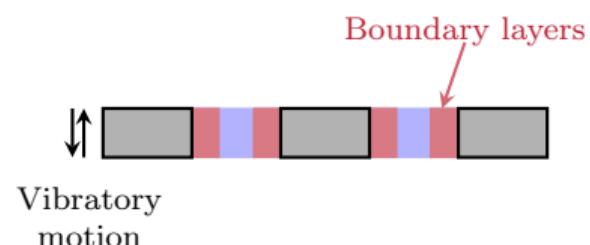
## Microperforated plate (MPP)

- ▶ Simple and lightweight structures
- ▶ Sound and vibration absorbers on its own
- ▶ Added viscous damping in low frequency range<sup>1</sup>



## Dissipation mechanisms

- ▶ Visco-thermic effects in boundary layers
- ▶ Fluid-structure interaction
- ▶ Sensitive to excitation level

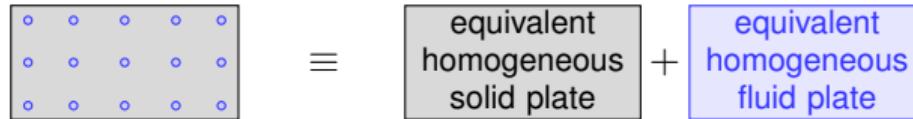
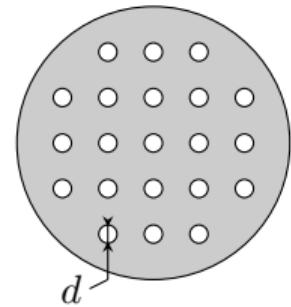


**Objective** Explore the impact of excitation level on MPP added damping

<sup>1</sup> Gallerand et al. "Vibration and damping analysis of a thin finite-size microperforated plate". (2022)

# Linear regime – Modeling steps

1. MPP  $\equiv$  Porous medium<sup>2</sup>  $\Rightarrow \phi(d), \sigma_0(d)$  and  $\alpha_\infty(\phi, d)$
2. Equivalent mechanical properties<sup>3</sup>  $\Rightarrow \rho_{\text{eq}}(\phi)$  and  $E_{\text{eq}}(\phi)$
3. Dynamics of MPPs  $\equiv$  Dynamics of porous plates<sup>4</sup>



<sup>2</sup>Atalla and Sgard. "Modeling of perforated plates and screens using rigid frame porous models". (2007)

<sup>3</sup>Boccaccini and Fan. "A new approach for the Young's modulus-porosity correlation of ceramic materials". (1997)

<sup>4</sup>Leclaire, Horoshenkov, and Cummings. "Transverse vibrations of a thin rectangular porous plate saturated by a fluid". (2001)

# Linear regime – Dynamics of finite-size MPPs

## Governing equations

$$\left(D_{\text{eq}}^*(\phi) + \frac{\alpha^2 M h^3}{12}\right) \nabla^4 w_s + h(\rho_{\text{eq}}(\phi) \ddot{w}_s + \rho_f \ddot{w}) = 0 \quad (1a)$$

$$\alpha M \nabla^2 w_s - \left(\rho_f \ddot{w}_s + \frac{\rho_f \alpha_\infty}{\phi} \ddot{w}\right) - \sigma_0(d) \dot{w} = 0 \quad (1b)$$

- ▶  $w_s$ : solid motion
- ▶  $w$ : relative fluid-solid motion
- ▶ Biot elastic coefficient  $\alpha$  and  $M$
- ▶ Isotropic structural damping  $\Rightarrow$  complex bending coefficient

$$D_{\text{eq}}^*(\phi) = \frac{E_{\text{eq}}(\phi) h^3}{12(1 - \nu^2)} (1 + j\eta_s) \quad \text{with} \quad E_{\text{eq}}(\phi) = \frac{E(1 - \phi^2)}{1 + (2 - 3\nu^2)} \quad (2)$$

# Linear regime – Dynamics of finite-size MPPs

## Boundary conditions

- ▶ Free
- ▶ Simply-supported
- ▶ Clamped

## Eigenvalue problem

- ▶ Displacement approximation via modal superposition

$$w_s(x, y, t) = \sum_{m=1}^{\infty} \sum_{n=1}^{\infty} w_{mn}^s e^{\lambda_{mn} t} \Psi_{mn}(x, y) \quad (3)$$

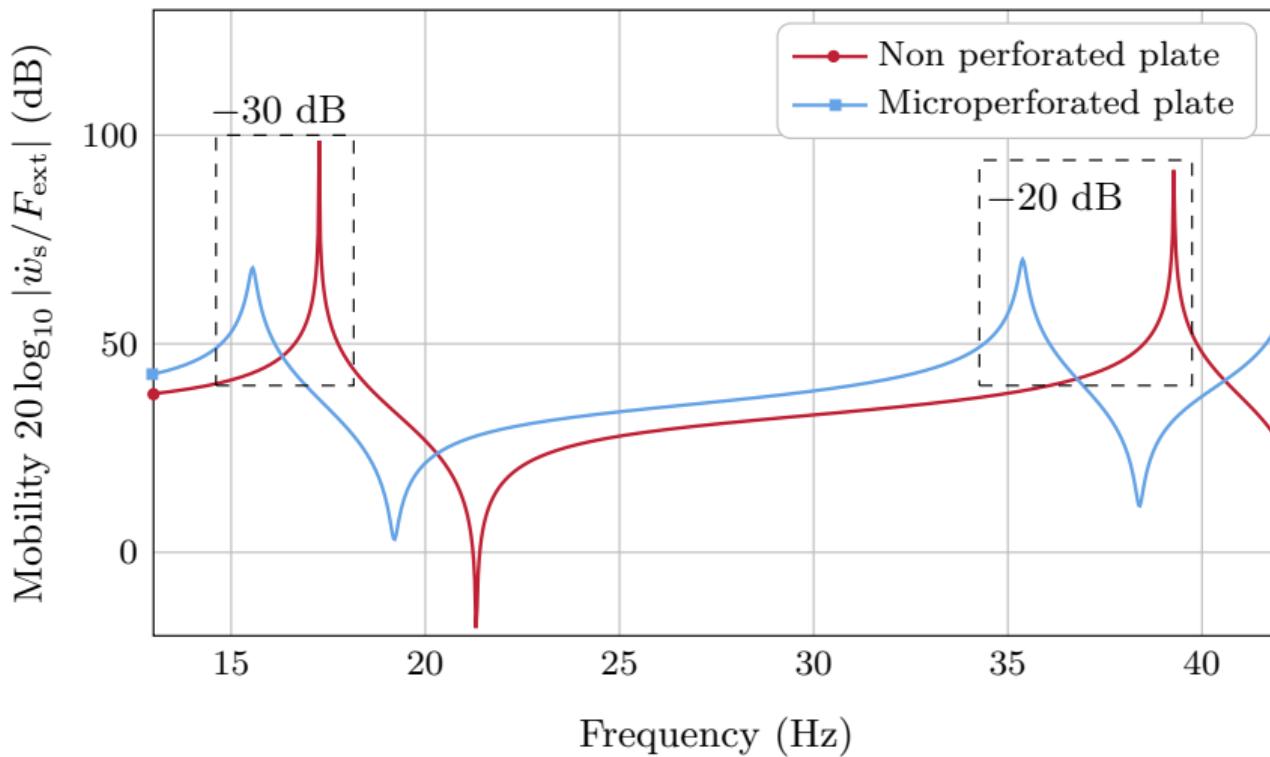
- ▶ Similar formulation for the relative fluid-solid motion  $w(x, y, t)$

## Resolution

Eigenvalues:  $\lambda_{mn} = -\zeta_{mn}\omega_{mn} \pm j\omega_{mn}\sqrt{1 - \zeta_{mn}^2}$

- ▶  $\zeta_{mn}$  : modal damping factor of mode  $mn$
- ▶  $\omega_{mn}$  : undamped natural frequency

# Linear regime – Analytical result



Aluminum simply-supported MPP of size  $490 \text{ mm} \times 570 \text{ mm} \times 1 \text{ mm}$  with  $d = 1.4 \text{ mm}$  and  $\phi = 10\%$

# Linear regime – Maximum damping

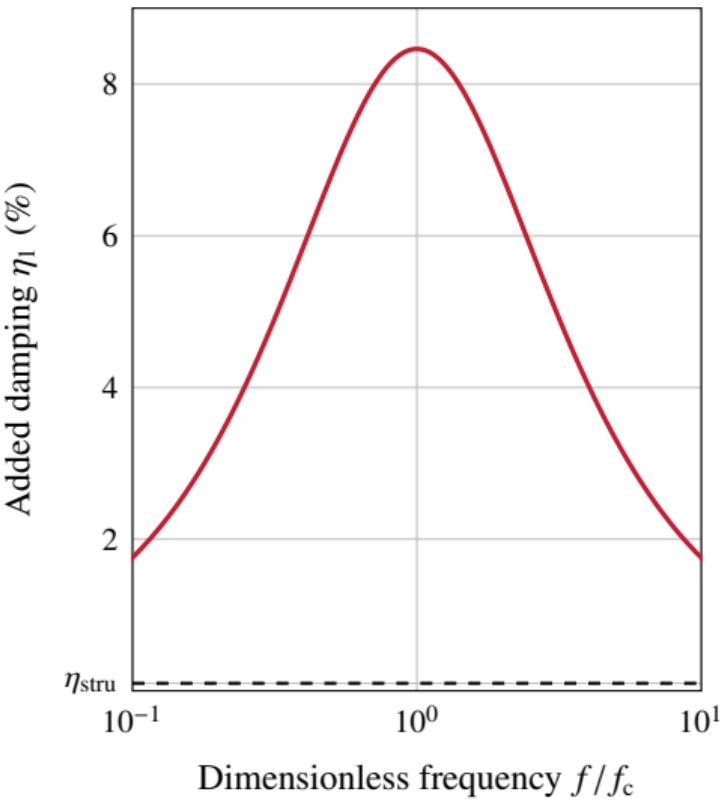
## Characteristic frequency

$$f_c(d) = \frac{32\mu_f}{2\pi\rho_f\alpha_\infty d^2} \quad (4)$$

- ▶  $f_c$  independent of the system
- ▶  $f_c \approx f_1 \Rightarrow 85$  times added damping

## Low frequency effect

- ▶ Greater the lower the frequencies
- ▶ Maximum effect on the first MPP mode



Aluminum simply-supported MPP of size 490 mm × 570 mm × 1 mm with  $d = 1.4$  mm and  $\phi = 10\%$

## Acoustic non-linear regime

- ▶ High airflow velocity in perforations  $\Rightarrow$  inertial effect
- ▶ Captured by a Forchheimer correction<sup>5</sup>

$$\sigma(d, \dot{w}(x, y, t)) = \sigma_0(1 + \varepsilon |\dot{w}(x, y, t)|) \quad (5)$$

## Acoustic and geometric non-linear regimes

- ▶ High airflow velocity in perforations  $10^{-2} \text{ mm s}^{-1} < V_{\text{fluid}} < 10^3 \text{ mm s}^{-1}$
- ▶ Significant structural displacement  $\frac{V_{\text{solid}}}{h} > 0.2 \text{ s}^{-1}$

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<sup>5</sup>Forchheimer. "Wasserbewegung durch Boden". (1901)

# Non-Linear regime – Governing equations

$$\left(D_{\text{eq}}^* + \frac{\alpha^2 M h^3}{12}\right) \nabla^4 w_s + h(\rho \ddot{w}_s + \rho_f \ddot{w}) = 0 \quad (6a)$$

$$\alpha M \nabla^2 w_s - \left(\rho_f \ddot{w}_s + \frac{\rho_f \alpha_\infty}{\phi} \ddot{w}\right) - \sigma_0(d) \dot{w} + \sigma_0(d) \varepsilon \dot{w} |\dot{w}| = 0 \quad (6b)$$

- ▶ Space semi-discretization  $\Rightarrow$  Matrix system of PDE

$$\mathbf{K}_1 \mathbf{w}_s(t) + \mathbf{M}_1 \ddot{\mathbf{w}}_s(t) + \mathbf{M} \ddot{\mathbf{w}}(t) = \mathbf{F}(t), \quad (7a)$$

$$\mathbf{K}_2 \mathbf{w}_s(t) + \mathbf{M} \ddot{\mathbf{w}}_s(t) + \mathbf{M}_2 \ddot{\mathbf{w}}(t) + \mathbf{C}_2 \dot{\mathbf{w}}(t) + \mathbf{F}_{\text{NL}}(\dot{\mathbf{w}}(t)) = \mathbf{0}, \quad (7b)$$

- ▶  $\mathbf{F}_{\text{NL}}(\dot{\mathbf{w}}(t))$  stores

$$f_r^{\text{NL}}(\dot{\mathbf{w}}(t)) = \sigma_0(d) \varepsilon \int_S \sum_{m=1}^M \sum_{n=1}^N \dot{w}_{mn}(t) \Psi_{mn}(x, y) \left| \sum_{p=1}^M \sum_{q=1}^N \dot{w}_{pq}(t) \Psi_{pq}(x, y) \right| \Psi_r(x, y) dS \quad (8)$$

## Non-Linear regime – Harmonic balance method resolution

- ▶ Fourier series truncated at the harmonic  $N_h$

$$\mathbf{w}_s(t) = \sum_{k=1}^{N_h} \mathbf{A}_k \cos k\omega t + \mathbf{B}_k \sin(k\omega t) \quad \text{and} \quad \dot{\mathbf{w}}(t) = \sum_{k=1}^{N_h} \mathbf{C}_k \cos k\omega t + \mathbf{D}_k \sin k\omega t \quad (9)$$

- ▶ Fourier's expansion of Equation (8)

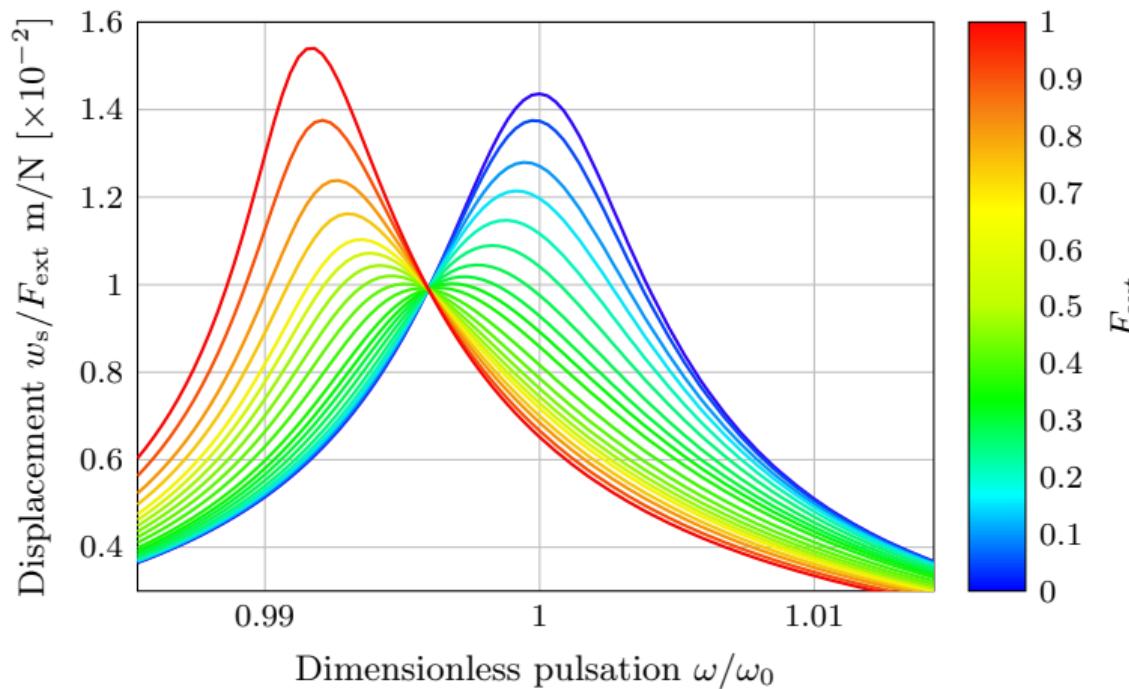
$$f_r^{\text{NL}}(\dot{\mathbf{w}}(t)) = \sigma_0(d)\varepsilon \sum_{m=1}^M \sum_{n=1}^N \sum_{k=1}^{N_h} (\Gamma_{mnkr} \cos k\omega t + \Pi_{mnkr} \sin k\omega t) \quad (10)$$

- ▶  $\Gamma_{mnkr}$  and  $\Pi_{mnkr}$ : non-linear functions of  $C_{mnk}$  and  $D_{mnk}$  computed numerically

# Non-Linear regime – Steady state response

►  $F_{\text{ext}}$  softens the system

► Maximum added damping for  $F_{\text{ext}}$  and relative velocity

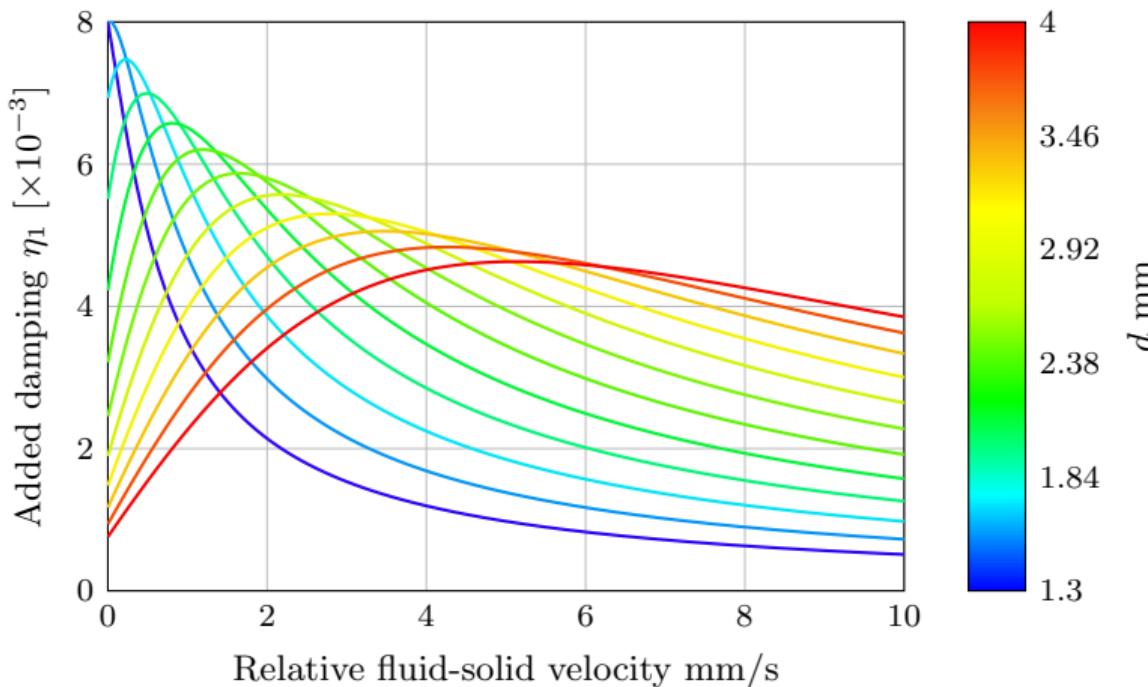


Aluminum simply-supported MPP of size 490 mm × 570 mm × 1 mm with  $d = 2.2$  mm and  $\phi = 10\%$

# Non-Linear regime – Added damping

► Fixed  $\varepsilon$  and MPP dimensions

► Variable  $d$  and amplitude of external excitation



Aluminum simply-supported MPP of size 490 mm × 570 mm × 1 mm with  $\phi = 10\%$

## Linear regime

- ▶ Added damping  $\Rightarrow$  low frequency range
- ▶ Maximum added damping  $\Rightarrow$  characteristic frequency  $f_c(d)$

## Non-linear acoustic regime

- ▶ Added damping  $\Rightarrow$  function of  $F_{\text{ext}}$  and relative fluid-solid velocity
- ▶ Added damping  $\Rightarrow$  maximum for a critical relative fluid-solid velocity
- ▶ Forchheimer  $\Rightarrow$  softening the system