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# Direct absorption nanofluid-based solar collectors for low and medium temperatures. A review

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## **ABSTRACT**

Solar energy is expected to play an important role in the decarbonization of the energy and industrial sectors. Low and medium temperature (< 400 °C) solar thermal collectors have proved to be a reliable solution to supply heat and decarbonize the industrial sector, with over 800 Solar Heat for Industrial Processes (SHIP) plants put in operation in the last decade. Governmental support policy is a key factor for solar thermal energy to play a major role in CO<sub>2</sub> emission reduction, which require improving the efficiency of solar collectors and reducing costs. Recent studies have demonstrated the potential of nanoparticles to enhance the optical properties of heat transfer fluids for direct absorption solar collectors (DASC). In a DASC the transfer fluid absorbs volumetrically the incident radiation, resulting in a more homogeneous temperature distribution and less heat losses than in conventional surface collectors. In this paper, the current state-of-the-art of SHIP installations and conventional surface collectors is presented, and a critical literature review dedicated to nanofluid-based DASC for both concentrating and non-concentrating collectors is provided. The key findings and the challenges to be overcome toward promoting the development of nanofluid-based DASC for SHIP applications are discussed.

## **KEYWORDS**

Solar Energy; Direct Absorption Solar Collector (DASC); Nanofluid; Solar heat for industrial processes (SHIP); Concentrating and non-concentrating technologies

## **HIGHLIGHTS**

- Parabolic trough collector is the leading technology for SHIP installations.
  - Nanofluid-based DASC improve the efficiency of conventional surface collector.
  - Carbon nanoparticles show promising properties for direct absorption collectors.
  - Low nanoparticle concentrations result in strong changes in absorption properties.
  - The nanoparticle stability is highly affected by the fluid temperature.
-

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## 1. Introduction

The International Energy Agency (IEA) reported in 2009 that 47% of the final energy used worldwide was related to heat demand [2]. The industry sector accounts for 32% of the global energy demand, and is currently the most energy consuming sector in the world [3]. 74% of the total energy demand in the industrial sector corresponds to heat (82 EJ), a three-fold larger value than the electricity demand. High temperature ( $>400\text{ }^{\circ}\text{C}$ ) heat represents 48% of the heat demand, while low and medium temperature ( $<400\text{ }^{\circ}\text{C}$ ) heat cover the remaining 52%. Modular size solar thermal collectors can supply low and medium temperature heat in industry, which would reduce the greenhouse gas emissions of the factory. Moreover, solar thermal energy systems can be hybridized with fossil or renewable fuels to supply heat on demand. The number of Solar Heat for Industrial Processes (SHIP) projects has increased considerably in the last 15 years, with at least 891 installations (791 MW<sub>th</sub>) worldwide in 2020 [4]. Solar thermal collectors are still not affordable enough to represent a competitive alternative to fossil fuel-based heat, and intense research efforts are still required toward increasing the SHIP share in the industrial sector [5].

Nowadays, either concentrating or non-concentrating solar thermal collectors are used to convert the incident solar radiation into process heat. The incoming sunlight is absorbed at the receiver/absorber surface and transferred to a heat transfer fluid (HTF), usually water or thermal oil. Significant research efforts have been dedicated to the efficiency enhancement of surface collectors, either by increasing the heat transfer or by reducing heat losses with glass covers or vacuum chambers (see section III). The integration of micro or nano-size particles to the HTF in order to increase their thermal conductivity and improve the collector efficiency has also led to many recent publications [6–10].

Direct (or volumetric) absorption solar collectors (DASC) have recently been suggested as a way to enhance heat exchanges within conventional solar collectors, using a semi-transparent fluid acting both as an absorber and as a HTF [11,12]. Conventional HTF exhibit weak radiative properties, making them unsuitable for sunlight absorption. However, the addition of small-size particles offers a path for better tuning the absorption capability of the fluid. For example, carbonaceous micro-sized particles can improve the radiative properties of the fluid but suffer from settling or blocking of pumps due to their inappropriate size [13,14]. The dispersion of nanoparticles in the so-called nanofluids allows mitigating the settling and blocking issues commonly encountered with larger size particles, while enhancing the absorbing properties of the fluid. Nanofluid-based direct absorption solar collectors (NDASC) have three advantages over traditional surface collectors, 1) heat is absorbed volumetrically, which enhances internal heat distribution; 2) peak temperatures occur far from the walls, where most heat losses occur; and 3) NDASC offer versatility for hybrid photovoltaic-thermal (PVT) collectors, photo-chemistry, and other applications.

Most of the previously published reviews in the field of nanofluids for solar energy systems discuss their enhanced thermal conductivity as well as their suitability for hybrid concepts (PVT) with little focus on NDASC [14–25]. Trong Tam *et al.* [26] and Kumar *et al.* [27] reviewed NDASC collectors using carbon-based nanofluids and plasmonic nanofluids respectively. Rasih *et al.* published in 2019 two reviews centered in concentrating NDASC [28] and in numerical investigation advances of NDASC studies [29]. The review of Fu *et al.* [30] studies NDASC systems at medium-to-high temperatures, while that of Karami *et al.* [31] focus on low-temperature NDASC. The state-of-the-art of Gorji *et al.* [32] and Chamsa-Ard *et al.* [33] give a complete overview of advances in nanofluid properties and synthesis, and their use in NDASC collectors up to 2017. In this literature review, recent advances

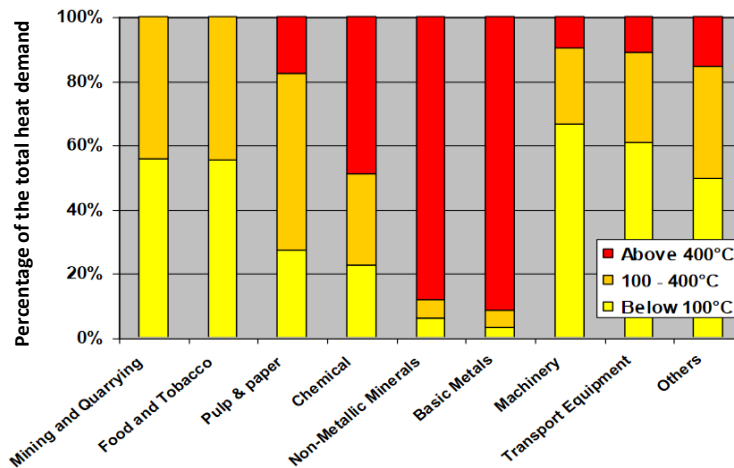
in nanofluid-based DASC are presented together with a detailed analysis of SHIP applications and commercial surface collectors for future NDASC commercialization.

In this paper, we aim at gathering, classifying and discussing the most promising applications for low and medium temperature collectors (SHIP installations). Current commercial collectors used for SHIP installations are described and analyzed to better grasp the challenges likely to be met by NDASC toward commercialization, and a literature review dedicated to NDASC for both concentrating and non-concentrating collectors is provided. In the final section, the challenges to overcome toward promoting the development and the practical implementation of NDASC are discussed, and the key findings of this review are summarized.

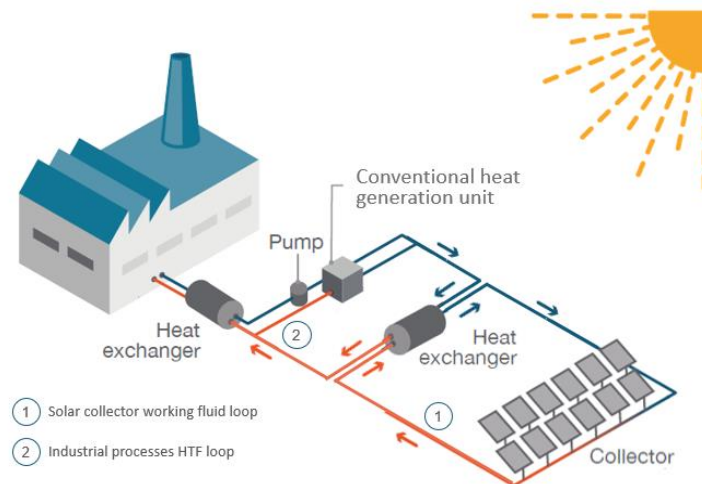
<b>Nomenclature</b>			
$A$	area ( $m^2$ )	<i>opt</i>	optical
$C$	concentration ratio	<i>out</i>	outlet
$D$	tube diameter ( $m$ )	<i>rec</i>	receiver
$I_0$	incident radiation ( $W/m^2$ )	<i>sca</i>	scattering
$I_{b,\lambda}$	blackbody spectral irradiance ( $W/m^2.sr.nm$ )	<i>th</i>	thermal
$I_\lambda$	spectral irradiance ( $W/m^2.sr$ )	<b>Acronyms</b>	
$k$	absorption, scattering or extinction coefficient ( $1/m$ )	CPC	Compound Parabolic Concentrator
$L$	Length ( $m$ )	DAPTC	Direct Absorption Parabolic Trough Collector
$\dot{m}$	mass flow rate ( $kg/s$ )	DASC	Direct Absorption Solar Collector
$s$	radiation beam direction vector	DNI	Direct Normal Irradiance
$T$	Temperature ( $^\circ C$ )	ETC	Evacuated Tube Collector
$\Delta T$	temperature difference ( $^\circ C$ )	FPC	Flat Plate Collector
$W$	width ( $m$ )	HTF	Heat Transfer Fluid
<b>Greek symbols</b>		LFC	Linear Fresnel Collector
$\eta$	thermal or optical efficiency (%)	MWCNT	Multi-Wall Carbon Nano-Tube
$\lambda$	wavelength ( $nm$ )	NDASC	Nanofluid-based Direct Absorption Solar Collector
$\phi$	scattering phase function	PDC	Parabolic Dish Collector
$\omega$	solid angle ( $sr$ )	PTC	Parabolic Trough Collector
<b>Subscripts</b>		PV	Photovoltaic cell
<i>abs</i>	absorption	PVT	Hybrid Photovoltaic-Thermal collector
<i>aper</i>	aperture	RTE	Radiative Transfer Equation
<i>ext</i>	external	SHIP	Solar Heat for Industrial Processes
<i>in</i>	inlet; inner (diameter)	SWCNH	Single-Wall Carbon Nano-Horns
		vol%	volume fraction (%)
		wt%	weight fraction (%)

## 2. Solar Heat for Industrial Processes

**Figure 1** shows the heat demand distribution by temperature range, for several key industrial sectors worldwide [34]. In most sectors, more than 80% of the total heat demand is associated with low and medium temperature heat, a temperature range particularly suited for SHIP plants (the mining and quarrying, as well as the food and tobacco industries showing the highest potential). In the case of the chemical sector, 50% of the heat demand could be supplied with a SHIP plant, but a hybrid installation is necessary to provide the high temperature heat demand. For the non-metallic and basic metals sectors, classical SHIP plants are not suitable due to the discrepancy in the typical operating temperatures of the SHIP plants and the metallurgical processes involved in these industries. This latter is an open field for research and innovation.



**Figure 1:** Temperature level of the industrial heat demand by industry sector. Reprinted from Ref. [34] with permission from the author.



**Figure 2:** Scheme of the general structure of a hybrid SHIP installation. The scheme is based on the figure from the Solar Payback report [35].

Conventional industrial heating systems use a HTF closed circuit that supplies heat to the different processes [36]. The most common standard HTF used for low ( $\leq 100$  °C) and medium temperature (100-400 °C) heat demand are liquid water, steam, and oil. Solar thermal collectors also use these fluids as HTF, which is a major advantage for the integration of solar energy in industrial plants. There are currently two possible integration solutions: 1) direct coupling of the solar field with the general HTF loop of the factory via a heat exchanger (**Figure 2**); and 2) coupling of the solar field with a particular single process of the factory. In most SHIP plants, solar energy is coupled with another thermal energy source (forming a *hybrid* installation) because of the intermittent nature of solar

energy (**Figure 2**) [37]. Solar energy can supply most of the heat demand while the conventional heat generation unit is run to provide the unmet heat demand (either because the solar power contribution is not sufficient, or because the heat temperature is too low).

In a SHIP plant, the land footprint of the solar field may constitute a serious hurdle. Rooftop solar collectors offer a relevant solution to the space availability problem, but may suffer from excessive wind loads, impeding their practical implementation. The rooftop area may not be sufficient to supply all the heat demand and additional ground areas can be required. The solar resource is largely determined by the location of the factory: potential market areas for SHIP include North African countries, the Middle East, the Mediterranean countries, Australia, USA, India, China, and South America [37]. Nevertheless, SHIP installations may also be worth implementing in countries with lower solar irradiation, and could contribute reducing the environmental impact and the energy consumption in these locations as well (as illustrated by Germany, which currently hosts the largest number of SHIP plants in Europe (33 projects), despite its limited solar resource [38]).

**Table 1:** SHIP projects by industry sector [36,38–41]. Technology abbreviations refer to: parabolic trough collector (PTC), flat plate collector (FPC), linear Fresnel collector (LFC), evacuated tube collector (ETC), and parabolic dish collector (PDC).

Industry sector	Country	Year	Company	Process	Technology	T (°C)
<b>Food</b>	Mexico	2013	Durango dairy	Boiler preheating	PTC	20-95
	Mexico	2014	Nestle Dairy Plant	Pasteurization	PTC	80-95
	India	2018	Hatsun Dairy	Drying	PTC	-
	Czech Republic	2003	PETA Bohemia Pekarna Sobeslav	Bakery	FPC	10 - 90
	Switzerland	2012	Emmi Dairy Saignelsgier	Drying processes	PTC	140-180
	Italy	2015	Nuova Sarda Industria Caaseria	Process heating	LFC	200
	Spain	2013	Papes Safor	Boiler preheating	PTC	200-250
	Greece	1999	Alpino SA	Boiler preheating	FPC	20 - 70
	Portugal	1987	Knorr Best Foods	Tools washing	FPC	40 - 45
	Spain	2015	Grasas del Guadalquivir	Process heating	LFC	130
	USA	2008	Frito Lay	Steam for heating	PTC	243
<b>Breweries</b>	Mexico	2014	Quesos La Doñita	Pasteurization	PTC	60-95
	Cyprus	2021	KEAN soft drinks	Steam for cleaning, sterilization and pasteurization	PTC	188
	Germany	2009	Hofmuhl Brewery	Bottle washing	ETC	20-110
<b>Textile</b>	China	2007	Daly Textile	Dyeing process	FPC	55
	China	2011	Jiangsu Printing & Dyeing	Preheating: printing & dyeing	ETC	50
	Germany	-	Meiser Textile	Process heating	PTC	140
<b>Fabricated Metal</b>	Germany	2010	Alanod	Aluminium process	PTC	143
	France	-	Viessmann Faulquemont	Cleaning bath	ETC	60
	Portugal	2014	Silamos S.A.	Drying products	PTC	50-160
<b>Medical</b>	India	2014	PSG Hospitals	Laundry and sterilization	PTC	150
<b>Paper</b>	Canada	2014	Parc Solaire Alain Lemaire	Boiler pre-heating	PTC	120-140
	India	2011	B.S. Paper Mill	Process heating	PDC	90-100
<b>Automobile</b>	India	2015	Harite Seatings Systems Limited	Cleaning automobile parts	ETC	55-60
	India	2010	Mahindra Vehicle Manufacturers	Washing engine components	PDC	120
<b>Pharmaceutical</b>	Egypt	-	El Nasr Pharmaceutical	Process steam	PTC	173

	India	2014	Abbott Healthcare	Boiler heating	PDC	-
	Spain	-	Covex	Process heat	PDC	50-90
<b>Chemical</b>	China	2016	Procter & Gamble	Boiler heating	PTC	130
<b>Mining</b>	South Africa	2011	Xstrata Elands Mine	Cleaning	ETC	60

SHIP are particularly developed among the food sector, with India and Mexico hosting the largest number of SHIP installations [38,42] (**Table 1**). In 2013, 61% of the Indian solar thermal capacity was used for industrial processes and community cooking (208 installed projects in 2017) [37,42]. In Mexico, 83 SHIP projects were installed in 2020, 48 of which provide heat to the food sector [38]. A direct coupling integration method is preferred among SHIP plants with most plants working at temperatures between 60-150 °C.

International organizations and governments are pushing forward the development of SHIP projects at a national and international level. The International Energy Agency (IEA) and SolarPACES founded the Solar Heating and Cooling (SHC) program to promote solar energy for low and medium heat demand [43]. Among their projects, “Task 64/IV – Solar Process Heat” is centered in SHIP projects and their development, as well as in creating a SHIP market guideline [44]. In Europe, the Solar Heat Europe association strives for the growth of SHIP installations by analyzing market statistics and by working with EU policymakers to increase the share of solar heating technologies in the European economy [45].



### 3. Commercial solar thermal collectors

Solar thermal collectors can be classified into two groups: non-concentrating (or stationary) and concentrating collectors. Non-concentrating collectors involve the absorption of the incident radiation by the absorbing surface (or volume) directly exposed to the solar flux. Concentrating collectors use reflecting mirrors to intercept and focus the incident solar radiation onto a smaller area, and require sun tracking to efficiently collect and concentrate solar radiation (unlike non-concentrating collectors, concentrators only exploit Direct Normal Irradiation (DNI)).

#### 3.A.- Non-concentrating collectors

The two main non-concentrating technologies are Flat Plate Collectors (FPC) and Evacuated Tube Collectors (ETC) [37], both technologies using water as HTF due to their low working temperature [46]. **Table 2** presents the main characteristics of the two types of collectors.

##### 3.A.1.- Flat plate collectors (FPC)

FPC convert the incident solar radiation into heat via an absorbing black plate coated with a selective absorbing material. The absorbed heat is then conducted through copper pipes, and collected by the working fluid flowing inside [36]. Conventional FPC operating temperatures range from 20 °C to 80 °C [5,46] with a 60% average collector efficiency. Integration of a double-glass cover or a vacuum chamber can level the maximum operating temperature up to 150 °C [5] and the collector efficiency up to 70% in the 20-80 °C temperature range [2,5]. FPC are used in the food and textile sectors (see **Table 1**) [5,36,38,47].

##### 3.A.2.- Evacuated tube collectors (ETC)

An ETC consists of a row of parallel evacuated tubes absorbing independently the incident solar irradiation. The tubes are composed of one or two glass layers, a vacuum chamber, a metallic absorbing layer with a selective coating, and fluid tubes containing the working fluid [46,49] (see [5,46,48] for more information about these designs). FPC performances are examined in [37,46,50]. Standard ETC work at temperatures up to 120 °C with a collector efficiency of ~60%, however new generations allow temperatures of 170 °C or 180 °C to be reached [2,5,37,46]. For further information, the Solar Rating & Certification Corporation provides a detailed comparison of most commercialized tubular solar collectors [51].

*Table 2: Summary of non-concentrating solar thermal collectors.*

Collector [Ref.]	Absorber	Dimension	Temperature	Advantages (Adv.) and Disadvantages (Disadv.)
FPC [1,4,27–29,36,41]	Planar	-	50-120 °C	<u>Adv.:</u> <ul style="list-style-type: none"> <li>◦ Light and affordable.</li> <li>◦ Main technology for low-temperature SHIP (40-90 °C).</li> </ul> <u>Disadv.:</u> <ul style="list-style-type: none"> <li>◦ Low efficiencies (&lt;60 %) beyond ~80 °C.</li> </ul>
ETC [2,5,37,46,50,51]	Tubular	D <sub>tube</sub> = 50-100 mm	60-180 °C	<u>Adv.:</u> <ul style="list-style-type: none"> <li>◦ Large temperature range (up to 120 °C) with nearly constant efficiency (60 %).</li> <li>◦ Popular for residential heating and cooling.</li> </ul> <u>Disadv.:</u> <ul style="list-style-type: none"> <li>◦ More expensive than FPC.</li> <li>◦ Less popular than FPC for low-temperature SHIP and than PTC for medium-temperature SHIP.</li> </ul>

### 3.B.- Concentrating collectors

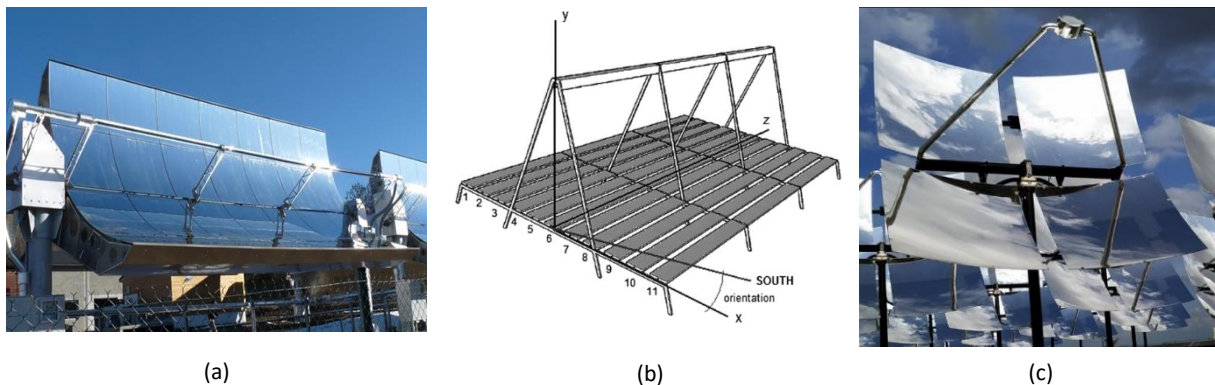
Two distinct families of optical concentrators can be distinguished based on the tracking strategy followed. One-axis tracking systems involve tracking the course of the sun along one single axis (east-west or, less commonly, north-south [35,52,53]), the most popular axis tracking systems to date being parabolic trough collectors (PTC) and linear Fresnel collectors (LFC). Two axes sun-tracking offer an extra degree of freedom in the quest of high concentration ratios. Two main technologies dominate this family of systems: parabolic dish collectors (PDC) and solar towers (ST). Solar towers use a field of heliostats (mirrors) to concentrate the incident radiation on a fixed-focus central receiver located atop a tower. This technology being solely used for electricity generation at high temperatures (up to 560 °C for molten salt solar power towers), it will not be addressed in this review. PTC, LFC, and PDC generally use either water, steam or thermal oil as HTF. **Table 3** summarizes the main characteristics of the three concentrating collectors.

#### 3.B.1.- Parabolic trough collectors (PTC)

PTC use cylindrical shape reflectors to focus direct solar radiation onto the moving linear receiver situated at the collector focus (focal line) (**Figure 3.a**). The receiver consists of an evacuated glass tube encompassing an inner metal tube coated with a selective material [52,54]. Small and medium PTC, whose typical dimensions are summarized in **Table 3**, are able to work at temperatures comprised between 100 °C and 300 °C [37] and are widely used in SHIP plants for medium temperature heat applications [4,38], residential heating and heat-driven refrigeration and cooling [54].

#### 3.B.2.- Linear Fresnel collectors (LFC)

LFC use horizontally aligned flat or quasi-flat mirrors to track the sun and reflect solar energy towards a fixed focus where the receiver is located (**Figure 3.b**). Unlike PTC, standard LFC are not able to concentrate sunlight on a small-size focal line due to the planarity of the reflectors and, therefore, use cavity receivers with secondary reflectors [53,55]. As PTC, large LFC reach around 400 °C and are used for electricity generation [56,57], while medium size LFC (see **Table 3**) have an operating temperature range of 80-300 °C and supply heat for SHIP plants, residential heating and heat-driven refrigeration [5,37–39].



**Figure 3:** Medium-size concentrating solar thermal collectors: (a) a PTC (PROMES laboratory); (b) scheme of a LFC; and (c) the TCT-RED PDC proposed by Thermal Cooling Technology®. Figures (a), (b) and (c) reprinted from Ref. [58] with permission from PROMES, Ref. [59] with permission from Elsevier and Ref. [40] with permission from Thermal Cooling Technology®, respectively.

#### 3.B.3.- Parabolic dish collectors (PDC)

PDC consist of a parabolic reflector dish focusing sunlight onto the receiver, consisting in a cylindrical cavity with an absorbing plate. Several PDC designs can be found in the market, such as the ARUN dish [60], the Scheffler dish [61,62] or the TCT-RED proposed by Thermal Cooling Technology®

(Figure 3.c) [40]. In India, PDC is the leading solar technology supplying heat for community cooking and industrial process heat, with 88 projects installed during 2017 [64]. Outside India, PDC for heat generation are not very developed, with only a few SHIP projects using PDC in Spain or Argentina [40,65,66].

Table 3: Summary of concentrating solar thermal collectors.

Collector [Ref.]	Concentrator <sup>(1)</sup>		Receiver <sup>(2)</sup>	Temperature	Advantages (Adv.) and Disadvantages (Disadv.)
<i><sup>(1)</sup> Length (L), width (W), aperture area (<math>A_{aper}</math>), and concentrating ratio (C); <sup>(2)</sup> Inner diameter (<math>D_{in}</math>) and outer diameter (<math>D_{out}</math>)</i>					
<b>PTC</b> [4,38,67–74]	Linear	L = 2 - 5 m W = 1 - 3 m C = 15 - 40	Tubular $D_{in}$ = 20 - 80 mm $D_{ext}$ = 80 - 140 mm	100-300 °C	<u>Adv.:</u> <ul style="list-style-type: none"> <li>High optical and thermal efficiencies.</li> <li>Most popular technology for medium temperature SHIP (100-250 °C).</li> </ul> <u>Disadv.:</u> <ul style="list-style-type: none"> <li>Receiver, which entails important constraints for junctions.</li> </ul>
<b>LFC</b> [5,37–39,75–81]	Linear	L = 3 - 20 m W = 1 - 7 m C = 10 - 25	Cavity with a secondary reflector	80-300 °C	<u>Adv.:</u> <ul style="list-style-type: none"> <li>Higher land-use efficiencies and lower wind loads than PTC, important constraints in rooftop installations.</li> <li>Flat reflectors are more affordable than parabolic reflectors.</li> <li>Fixed receiver.</li> </ul> <u>Disadv.:</u> <ul style="list-style-type: none"> <li>Lower optical efficiency than PTC.</li> <li>Technology less developed than PTC.</li> </ul>
<b>PDC</b> [40,60–66]	Point focus	$A_{aper}$ = 16-100 m <sup>2</sup> C = 100-250	Cavity	100-300 °C	<u>Adv.:</u> <ul style="list-style-type: none"> <li>Better optical efficiencies than PTC and LFC.</li> <li>Well known technology with a large number of SHIP installations in India.</li> </ul> <u>Disadv.:</u> <ul style="list-style-type: none"> <li>Highly affected by wind loads.</li> </ul>

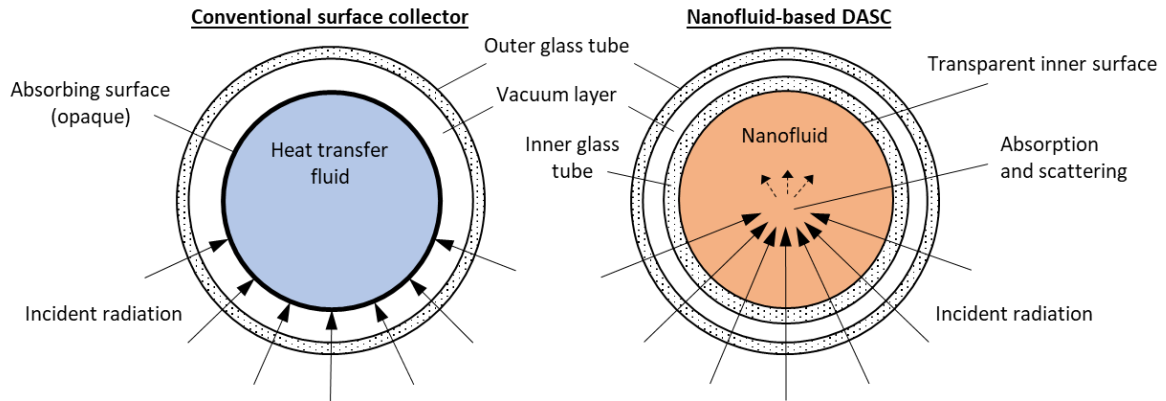
### 3.C.- Conclusion

Despite their affordability, non-concentrating collectors suffer from low efficiencies above 150 °C, a temperature level beyond which concentrating collectors demonstrate higher performances. PDC suffer from high wind loads, an important parameter when SHIP rooftop installations are intended. LFC and PTC have the advantage of a higher working temperature range than ETC but strongly depend on the typical DNI of the location. In conclusion, ETC are better suited for SHIP installation in locations characterized by a low annual average DNI, while LFC and PTC are more favorable for locations with high DNI and offer a higher heat temperature range. Considering the use of nanofluids for DASC, linear tubular collectors are more favorable than point focus or planar collectors owing to the progressive linear absorption and the fluid flow direction. Planar collectors, like nanofluid-based FPC, may be favorable for hybrid PVT systems with the nanofluid acting as an optical filter. Further detail regarding non-concentrating and medium-size collectors for SHIP applications can be found in the following articles and reports [5,46,48,54,57,82].

## 4. Nanofluid-based direct absorption solar collectors (NDASC)

In this section, the underlying physical principles of the NDASC are discussed, and the main nanoparticles properties as well as their effect on NDASC operation are described. A wide range of studies have already discussed how the integration of nanofluids may improve the absorption properties of these solar collectors, focusing solely on the optical properties of the nanofluids. Here, we aim at extending our analysis beyond these aspects and discuss literature outcomes related to the volumetric absorber itself, both on numerical and experimental grounds.

The absorber container (plate or tube) as well as the absorbing medium, constitute two key elements differentiating surface collectors from DASC. In a DASC, the radiation is absorbed in the volume of the HTF rather than at the surface of the collector (as illustrated in **Figure 4**). Consequently, the absorber container must be transparent to solar radiation.



**Figure 4:** Diagram of the optical principle of a conventional tubular surface collector and a nanofluid-based tubular DASC.

### 4.A.- Physical principles and properties of nanofluids

Since solar radiation is absorbed in the volume of the HTF, the extinction of sunlight through the fluid must be understood. The Radiative Transfer Equation (RTE) defines the variation of the radiation intensity due to absorption, scattering, and emission of the medium (Eq. 2):

$$\frac{d}{ds} I_{\lambda}(s) = -k_{abs,\lambda} I_{\lambda}(s) - k_{sca,\lambda} I_{\lambda}(s) + k_{abs,\lambda} I_{b,\lambda}(s) + \frac{1}{4\pi} \int_{4\pi} k_{sca,\lambda} I_{\lambda}(s) \phi_{\lambda}(\omega' \rightarrow \omega) d\omega' \quad (2)$$

where  $I_{\lambda}$  is the spectral radiative intensity,  $\lambda$  the wavelength,  $k_{abs}$  the absorption coefficient,  $k_{sca}$  the scattering coefficient,  $I_{b,\lambda}$  the blackbody spectral radiative intensity,  $\phi_{\lambda}$  the scattering phase function, and  $\omega$  the solid angle. The parameter “ $s$ ” represents the distance travelled by the radiation beam through the medium. The most common assumption to solve the RTE considers the scattering coefficient of the nanofluid negligible compared to the absorption coefficient. In addition, as the working temperatures of DASC are relatively low, the emission term of the RTE can generally be neglected too. These assumptions lead to a well-known form of the RTE, the Beer-Lambert equation (eq. 3), which points out the exponential decrease of the solar irradiation intensity along its propagation path in the medium.

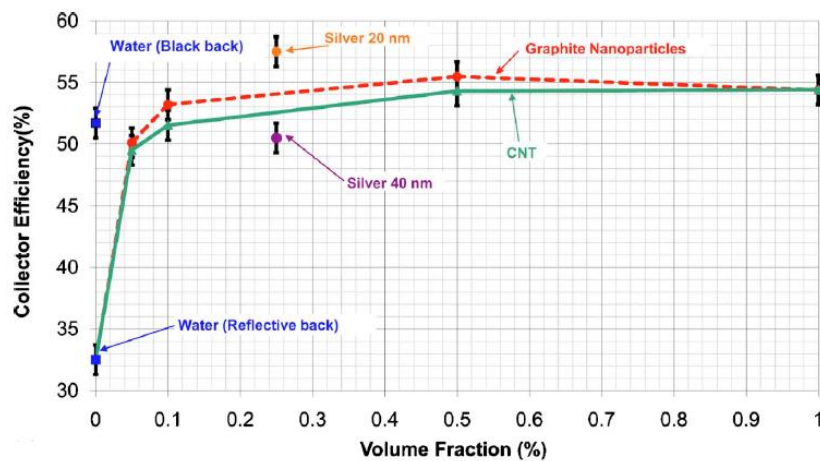
$$I_{\lambda}(s) = I_{\lambda}(0) \exp(-k_{abs,\lambda} \cdot s) \quad (3)$$

Several methods are commonly used to characterize the absorption properties of the nanofluid depending on the particle type, shape and mean size. The Rayleigh scattering approximation is the most accurate and popular solution method [1,12,26,31,32,83–94], other methods such as the Mie scattering theory [92,95–97] or the Maxwell-Garnett theory being less used [91,98]. Among metal nanoparticles, gold (Au), silver (Ag), copper (Cu), and aluminum (Al) show the best optical properties

for NDASC. Metal oxides showing the best potential for direct absorption applications are silica ( $\text{SiO}_2$ ), iron oxide ( $\text{Fe}_3\text{O}_2$ ), copper oxide ( $\text{CuO}$ ), and alumina ( $\text{Al}_2\text{O}_3$ ). Carbon-based nanoparticles most commonly used for NDASC are carbon nanotubes (CNT) (single-walled or multi-walled), single-walled carbon nanohorns (SWCNH), and graphene nanoplatelets. The performance of graphene oxide and reduced-graphene oxide nanoparticles has also been investigated [99,100]. The nanoparticles' shape and size are evaluated microscopically and spectroscopically with well-known methods such as scanning electron microscopy (SEM), transmission electron microscopy (TEM), or atomic force microscopy (AFM) [101–108]. Three base fluids are essentially used: water, ethylene glycol and thermal oils. Further information about the thermophysical and optical properties of nanofluids for DASC can be found in references [109–115].

#### 4.B.- Non-concentrating NDASC

The main characteristics and performances of non-concentrating NDASC studies are summarized in **Table 4** chronologically. The following section will discuss these findings into more details.

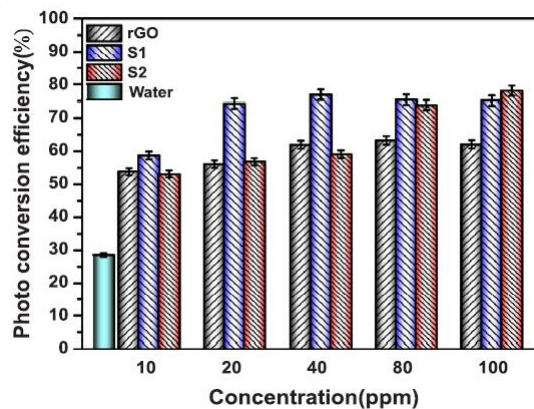


**Figure 5:** Experimental collector efficiencies as a function of the vol% for NDASC with different nanoparticles dispersed in water. The surface collector is plotted under the name “Water (black back)”. Reprinted from Ref. [13] with the permission of AIP Publishing.

In 1975, Minardi and Chuang [11] studied experimentally for the first time the performance of a volumetric planar collector using a black Indian ink and found collectors' efficiencies up to 80%. Their work underlined the expected advantages of fluid-based DASC over conventional planar collectors: temperature distribution, lower heat losses, versatility and lower cost. With the rise of nanomaterials science, a broad range of studies has been dedicated to the optical characterization of nanofluids as well as their possible integration in solar thermal collectors. Tyagi *et al.* [12] were the first to numerically study the efficiency of a 2D planar NDASC incorporating Al nanoparticles dispersed in water. Their model predicts efficiencies up to 80% with a 3.0% nanoparticle volume fraction (vol%) and a 1.2 mm collector thickness, representing a 10% efficiency increase relative to conventional FPC, and laid the groundwork for future studies on numerical NDASCs.

Lee *et al.* [95] and Cregan and Myers [84] studied numerically Al nanoparticles dispersed in water with lower concentrations, between 0.0005 and 0.02 vol%, and demonstrated relatively high efficiencies despite the low concentration levels and collector heights (see **Table 4**). Among metals, Ag nanoparticles have been widely studied for NDASC as a result of their high absorption properties at low volume fractions [13,85,86,116–118]. Otanicar *et al.* [13,116] demonstrated higher efficiencies with Ag particles rather than carbon nanotubes or graphite particles, even at lower Ag particle

concentrations (0.25 vol%) (**Figure 5**). However, later numerical and experimental studies [85,86,117] aiming at comparing Ag, graphite and magnetite nanoparticles at low concentrations (less than 0.004 vol%), shown higher thermal efficiencies with magnetite particles, followed by graphite. This discrepancy outlines the high impact of the particle size, the particle volume fraction and the collector height on efficiency and outlines the compromise that should be made between these parameters. Other metal particles, such as Cu, Au, Fe or Ni, have also been studied numerically and spectrophotometrically, and demonstrated interesting characteristics for direct absorption applications [119,120,110,121–123], despite the limited range of experimental studies available. Metal-oxides such as CuO, SiO<sub>2</sub>, Al<sub>2</sub>O<sub>3</sub> or TiO<sub>2</sub> demonstrated promising absorbing properties but also lower efficiencies than their metal or carbon based counterparts [96,124–127]. Nevertheless, metal-oxides can be relatively easily coupled with metal or carbon particles to create hybrid nanofluids.



**Figure 6:** Photo-thermal conversion efficiencies as a function of the nanoparticle concentration for the different samples: water, reduced-graphene oxide (rGO), plasmonic fluid with 0.15g of Ag (S1), and plasmonic fluid with 0.30g of Ag (S2). Reprinted from Ref. [99], an open access article not-requiring permission.

As outlined by several authors, controlling the particle size and configuration of metal nanoparticles, e.g. core-shell particles, could lead to more efficient absorbing nanofluids thanks to the plasmon resonance phenomenon [128,129]. Lee *et al.* [95,130] studied analytically the efficiency of a planar NDASC incorporating core-shell nanoparticles, namely SiO<sub>2</sub> core and Au shell, dispersed in water. The authors outlined that scattering phenomena needs to be considered when plasmonic particles are used, as the surface plasmon enhances both absorption and scattering. The optical analysis emphasizes the absorption wavelengths variation with the nanoparticle configuration (core and shell dimensions). By selectively choosing the core-shell configuration of the particles and their relative fraction in the solution, the authors obtained a nanofluid demonstrating higher performances in comparison with Al-based nanofluid. Mehrali *et al.* [99] demonstrated higher thermal efficiencies with graphene-silver hybrid plasmonic nanoparticles than with reduced-graphene oxide particles, using water as based-fluid (**Figure 6**). The experiment analyzed the photo-thermal conversion with a static (no-flow) sample of the nanofluids, and thus results should be verified with under flow conditions. Further information about plasmonic particles used in DASC can be found in Mallah *et al.* [112,131] and Kumar *et al.* [27] reviews. The book recently published by Zafar Said, *Hybrid Nanofluids* [115], gives a complete overview of the history of hybrid nanofluids, their preparation and their potential applications. Plasmonic and hybrid nanofluids constitute promising options toward enhancing the absorption properties of common nanofluids for NDASC at low volume fraction. However the complexity of their synthesis and their high production cost could impede their future commercialization [27].

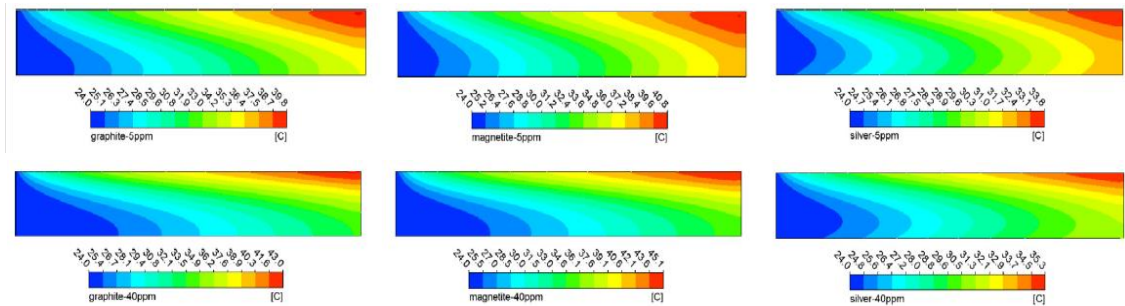
Carbon-based nanofluids have been considered as the most promising working fluid for NDASC owing to their outstanding thermal and optical properties [26,111]. Otanicar *et al.* [13,116] characterized both graphite and carbon nanotubes, and showed slightly higher performances for the former (**Figure 5**). Both multiwalled (MWCNT) or single-walled (SWCNT) carbon nanotubes are the most commonly used carbon-based nanoparticles for NDASC, efficiencies up to 89% were obtained with relatively low concentrations (less than 100 ppm) [13,87,132,133]. Luo *et al.* [96] demonstrated higher efficiencies with graphite, relative to Al<sub>2</sub>O<sub>3</sub> nanoparticles, using a 50 times lower volume fraction, both on theoretical and experimental grounds. Similarly, Gorji and Ranjbar [85,86,117] revealed a 30% improvement in the thermal efficiency of graphite-based nanoparticles, in comparison with Ag nanoparticles, that are currently considered as one of the most promising metal nanoparticles for NDASC. The new promising 2D carbon-based nanoparticle, graphene, has shown enormous potential for NDASC with over 90% efficiencies at low concentrations (0.02-0.005 weight fraction (wt%)) [100,134,135]. Graphene oxide and reduced graphene oxide show promising absorption properties as well as versatility for hybridization with metal nanoparticles such as silver [99,100]. Further experimental studies are still needed to determine the optimal graphene volume fractions as well as the optimal working conditions (flow rate, fluid temperature, etc...). The analysis provided by Khullar *et al.* [111] offers a full overview of the thermal and optical properties of the principal nanoparticles used in NDASC, and concluded that carbon-based nanofluids are the most suitable candidates. Further comparisons between carbon nanofluids, plasmonic, and hybrid nanofluids including all aspects of the technologies (see **section 5**) are still required to determine the best volumetric absorbers today.

As revealed in **Table 4**, the most widely used base fluid for non-concentrating NDASC is water. Owing to the moderate temperature increase, water is the most affordable and practical base fluid. The influence of the base fluid on the performance of the collectors is scarcely addressed in the literature. The absorption properties of water-based nanofluids incorporating carbon nanoballs were shown to be slightly higher than ethylene glycol-based samples [136], which was confirmed numerically by Moradi *et al.* [137] considering a water and an ethylene glycol based NDASC using SWCNH. Thermal oils and glycols may offer an advantage over water when temperatures beyond 100 °C are attained due to their higher boiling temperature, issue addressed for concentrating NDASC in the next subsection.

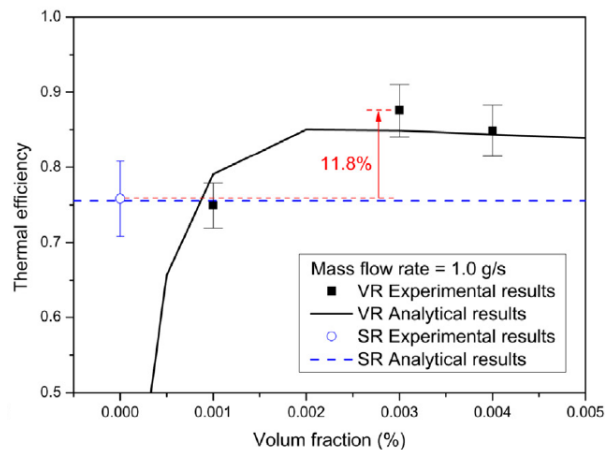
The role of the particle size was investigated by several authors: Tyagi *et al.* [12] suggested that the collector efficiency is weakly correlated with particle size, a conclusion supported by other researchers in later studies [31,83,84,90,110]. However, Otanicar *et al.* [116] and Chen *et al.* [119] noticed an efficiency drop inversely correlated to particle size, for both Ag and Au nanoparticles. He *et al.* [138] outlined that the heat transfer properties of nanofluids are improved for decreasing particle size, while the optical properties are slightly worsened: such trends could explain the discrepancies in the conclusions reached by these authors. It can be concluded that the particle size has a limited impact on the optical properties of DASC nanofluids for particle size below 100 nm. However, since the heat transfer properties are enhanced for decreasing particle size [139], smaller particles might be preferred for systems involving large temperature gradients.

Most metal-based DASC studies involve spherical-shaped nanoparticles, since metal particles are mostly spherical and modeling theories are based on the spherical assumption (Rayleigh scattering approximation, Mie scattering theory, etc...). Carbon-based nanofluids incorporate tubular or flat shaped nanoparticles (CNT or graphene for example), which makes them preferable over metals or oxides as their larger surface area increase the heat transfer rate between the particles and the base

fluid [19,26,140]. Morphology studies aiming at understanding how the shape of the particles affect the system performance were published for DASC system involving plasmonic or hybrid nanofluids [27,95,130,131,141–143]. As outlined by Mallah *et al.* [131], the nanoparticle shape influences the spectral absorption of the nanofluid, offering the possibility of creating blended nanofluids with increased absorption properties, and thus higher NDASC performances, by carefully choosing the particle shape and volume fraction.



**Figure 7:** Temperature contours for three nanofluids (graphene (50 nm), magnetite (15 nm), and silver (20 nm)) with two different nanoparticle's concentration, 5 ppm and 40 ppm. The simulated collector is 12.1 cm long, 5 cm wide, and 2 cm high. Reprinted from Ref. [86] with permission from Elsevier.



**Figure 8:** Thermal efficiency (analytical and experimental results) for the surface receiver (SR) and the volumetric receivers (VR) at different MWCNT vol%. Reprinted from Ref. [133] with permission from Elsevier.

It is widely accepted that the two crucial parameters affecting the collector's performance are 1) the particle volume fraction and 2) the fluid depth (distance travelled by the incident light throughout the nanofluid). The absorption coefficient of a nanofluid increases with the particle volume fraction. However, as can be deduced from the Beer-Lambert's law (eq. 3), the absorbed radiation in a fluid varies exponentially with the fluid depth, resulting in a higher fraction of the incident radiation absorbed in the first fluid layers. Therefore, increasing the particle concentration over a specific threshold value can lead to the complete absorption of the incident radiation on the first nanofluid layers, and result in temperature contours similar to those of conventional surface collectors. **Figure 7** from Gorji and Ranjbar [86] reflects the increase of the top layer temperatures proportionally with the particle volume fraction. Higher temperatures in the top fluid layers would induce higher heat losses and negatively impact the collector efficiency. A compromise between the volume fraction and the collector's depth needs to be found to determine the optimal nanofluid, which differs for each particle-type and working conditions, as outlined in **Figure 5**, **Figure 6** and **Figure 8** [13,99,133,144]. The collector and thermal efficiencies of various NDASC are given in **Table 4**. Three important limitations currently preventing NDASC commercialization are nanofluid stability, production cost and environmental impact (see **section 5**): these parameters appear to be strongly penalized by the



increase in the nanoparticle volume fraction, and future research work aiming at optimizing these parameters for lower volume fraction nanofluids should thus be pursued.

Similarly to conventional surface collectors, the efficiency of a NDASC is affected by the increase of the fluid temperature due to higher heat losses [86,87,100,126,134]. The temperature increase may also affect the nanofluid stability due to the temperature-dependence of the surfactant degradation, organic compounds used to improve chemically the stability of nanofluids, which constitutes a major factor of nanofluid instability and reduction of absorption properties (refer to **section 5** for further information about the stability challenge). By increasing the nanofluid flowrate, the time during which the fluid is exposed to radiation is reduced, which lowers the fluid temperature and improves the thermal efficiency of the collector [86,87,135]. Thanks to the progressive absorption of radiation along the fluid volume, NDASC are not constrained by the flow regime unlike surface collectors, which need high Reynolds number regimes for increasing the convective part of the heat transfer fluid. Yet, Struchalin *et al.* [145] demonstrated that the deposition efficiency (capability of nanoparticles to deposit in the collector's inner surface) decreases from 49% in a laminar flow to 1-2% in a turbulent flow, making high Reynolds number regimes preferable. The sensitivity of nanofluids to the flow regime is strongly linked to the elaboration method (base fluid, particle properties, additives). The investigation carried by Bhalla *et al.* [146] provides a comprehensive understanding of the influence of different working parameters (mass flow rate, incident radiation, collector depth, etc...) on the collector efficiency using amorphous carbon-based nanofluids. Studies summarized in **Table 4** demonstrate the capability of NDASC to outperform surface planar collectors, with efficiencies enhanced by 10-15 % using low particle volume fraction. Novel NDASC using porous materials [147] or in between absorbing plates [148] can also be found in **Table 4**.

Table 4: Summary of non-concentrating NDASC reviewed.

NON-CONCENTRATING NDASC									
Authors (year) [ref.]	Approach			Nanofluid				Results	
	Num./Exp.	Geometry	Fluid depth	Nanoparticles	Base Fluid	Size	Concentration	Temperature	Efficiencies
Otanicar <i>et al.</i> (2009) [116]	Numerical	Planar	0.3 mm	Graphite	Water	D = 30 nm	0.4-1.0 vol%	-	$\eta_{col} = 71\% - 69\%$
				Ag		D = 10 - 30 nm	0.1 vol%	-	$\eta_{col} = 39.8\% - 41.5\%$
Tyagi <i>et al.</i> (2009) [12]	Numerical	Planar	1.2 mm	Al	Water	D = 5 nm	3 vol%	-	$\eta_{col} = 80\%$
Otanicar <i>et al.</i> (2010) [13]	Num./Exp.	Planar	0.15 mm	Graphite	Water	D = 30 nm	0.5 vol%	-	$\eta_{col} = 55.5\%$
				CNT		D = 6 - 20 nm	0.5 vol%	-	$\eta_{col} = 54.0\%$
				Ag		D = 20 - 40 nm	0.25 vol%	-	$\eta_{col} = 50.5\% - 57.5\%$
Lee <i>et al.</i> (2012) [95]	Numerical	Planar	1.5 mm	Au-SiO <sub>2</sub> (Shell-Core)	Water	SiO <sub>2</sub> : D = 20 - 55 nm Au: D = 3 - 10 nm	0.02-0.1 vol%	$\Delta T = 45\text{ }^\circ\text{C}$	$\eta_{col} = 70.0\%$
				Al		D = 5 nm	0.02-0.1 vol%	$\Delta T = 25\text{ }^\circ\text{C}$	$\eta_{col} = 42.0\%$
Kundan and Sharma (2013) [124]	Experimental	Planar	-	CuO	Water	D = 80 nm	0.005 vol%	-	$\eta_{col} = + 6\%$ vs a FPC
							0.05 vol%	-	$\eta_{col} = + 4\%$ vs a FPC
Ladjevardi <i>et al.</i> (2013) [83]	Numerical	Planar	100 mm	Graphite	Water	D = 30 nm	0.0025-0.000025 vol%	-	-
Verma et Kundan (2013) [125]	Experimental	Planar	-	Al <sub>2</sub> O <sub>3</sub>	Water	D = 40 nm	0.05 vol%	-	$\eta_{col} = +3\% - +5\%$ vs a FPC
Karami <i>et al.</i> (2014) [150]	Numerical	Planar	3 - 5 mm	SWCNH	Water	-	0.001-0.05 g/l	$\Delta T = 0.1\text{ }^\circ\text{C}$	$\eta_{col} = 18.0\% - 87.0\%$
Lee <i>et al.</i> (2014) [132]	Num./Exp.	Planar	7.9 mm	Conventional FPC		-	-	-	$\eta_{col} = 75.4\%$
				MWCNT	Water	D = 20 - 30 nm L = 10 $\mu\text{m}$	0.005-0.02 vol%	-	$\eta_{col} = 41.0\% - 90.0\%$
Luo <i>et al.</i> (2014) [96]	Num./Exp.	Planar	25 mm	TiO <sub>2</sub> , Ag, Cu, SiO <sub>2</sub> , CNTs	Texatherm oil	D = 10 - 50 nm	0.01-0.1 vol%	-	-
				Al <sub>2</sub> O <sub>3</sub>		D = 20 nm	0.5 vol%	$T_{out} = 52\text{ }^\circ\text{C}$	$\eta_{col} = 117.5\%$ vs a FPC
				Graphite		D = 35 nm	0.01 vol%	$T_{out} = 50\text{ }^\circ\text{C}$	$\eta_{col} = 122.7\%$ vs a FPC
Cregan and Myers (2015) [84]	Numerical	Planar	1 - 4 mm	Water		-	-	-	$\eta_{col} = 3\% - 5\%$
				Al	Water	D = 20 nm	0.0005 vol%	-	$\eta_{col} = 25\% - 50\%$
				-		0.006 vol%	-	$\eta_{col} = 49\% - 58\%$	
Gorji and Ranjbar (2015) [85]	Numerical	Planar	10 - 20 mm	Graphite	Water	D = 30 nm	0.05; 0.5; 1.0 vol%	-	$\eta_{th} = 49.0\% - 58.0\%$
							0.1 vol%	$T_{out} = 33 - 37\text{ }^\circ\text{C}$	$\eta_{th} = 67.05\% - 78.06\%$
Gupta <i>et al.</i> (2015) [126]	Experimental	Planar	6 mm	Al <sub>2</sub> O <sub>3</sub>	Water	D = 20 - 30 nm	0.001-0.05 vol%	-	$\eta_{col} = + 20 - 40\%$ of water $\eta$
Liu <i>et al.</i> (2015) [134]	Num./Exp.	Tubular	50 mm	Graphene	[HMIM]BF <sub>4</sub>	-	0.0005 wt%	$T_{out} = 56\text{ }^\circ\text{C}$	$\eta_{col} = 70\% - 86\%$
							0.001 wt%	$T_{out} = 77\text{ }^\circ\text{C}$	$\eta_{col} = 55\% - 83\%$

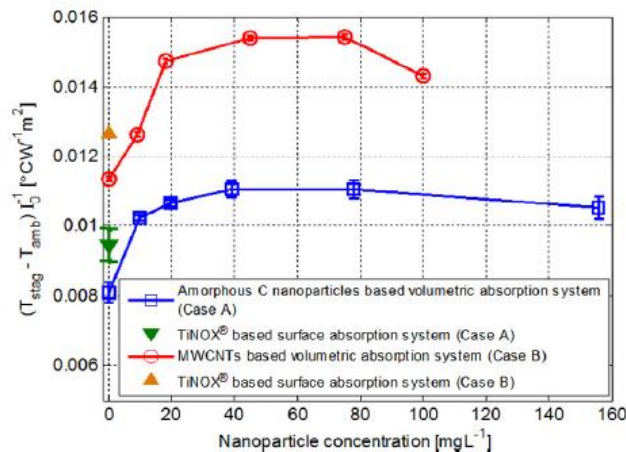
							0.002 wt%	-	$\eta_{col} = 40\% - 81\%$
							0.01 wt%	-	$\eta_{col} = 24\% - 75\%$
<a href="#">Moradi et al. (2015) [137]</a>	<b>Numerical</b>	Tubular	44 mm	SWCNH	Water	D = 2 - 5 nm	0.006 g/l ; 0.02 g/l	$T_{out} = 23 - 25\text{ }^{\circ}\text{C}$	$\eta_{col} = 62.5\% ; 61.0\%$
					Ethylene Glycol	L = 30 - 50 $\mu\text{m}$	0.005 g/l ; 0.02 g/l	$T_{out} = 23 - 28\text{ }^{\circ}\text{C}$	$\eta_{col} = 51.5\% ; 52.5\%$
<a href="#">Delfani et al. (2016) [87]</a>	<b>Num./Exp.</b>	Planar	10 mm	Water				$T_{out} = 36\text{ }^{\circ}\text{C}$	$\eta_{col} = 60.3\%$
				Standard surface collector				$T_{out} = 38\text{ }^{\circ}\text{C}$	$\eta_{col} = 71.7\%$
				MWCNT	Water	D = 10 - 20 nm L = 10 - 30 $\mu\text{m}$	25-100 ppm	$T_{out} = 37 - 40\text{ }^{\circ}\text{C}$	$\eta_{col} = 77.6\% - 89.3\%$
<a href="#">Gorji and Ranjbar (2016) [86]</a>	<b>Num./Exp.</b>	Planar	20 mm	Water				$T_{out} = 31.8\text{ }^{\circ}\text{C}$	$\eta_{th} = 27.5\%$
				Graphite	Water	D = 50 nm	5-40 ppm	$T_{out} = 40 - 43\text{ }^{\circ}\text{C}$	$\eta_{th} = 59.0\% - 72.0\%$
				Magnetite		D = 15 nm	5-40 ppm	$T_{out} = 40 - 45\text{ }^{\circ}\text{C}$	$\eta_{th} = 64.0\% - 70.0\%$
				Ag		D = 20 nm	5-40 ppm	$T_{out} = 34 - 35\text{ }^{\circ}\text{C}$	$\eta_{th} = 35.0\% - 38.0\%$
<a href="#">Lee et al. (2016) [133]</a>	<b>Num./Exp.</b>	Planar	10 mm	Surface receiver				$T_{out} = 120\text{ }^{\circ}\text{C}$	$\eta_{th} = 75.4\%$
				MWCNT	Water	D = 20 nm	0.001-0.004 vol%	$T_{out} = 60 - 100\text{ }^{\circ}\text{C}$	$\eta_{th} = 75.0\% - 87.2\%$
<a href="#">Vakili et al. (2016) [135]</a>	<b>Experimental</b>	Planar	10 mm	Water (deionized)				-	$\eta_{col} = 48.0\% - 69.0\%$
				Graphene	Water	L = 2 $\mu\text{m}$	0.0005 wt%	-	$\eta_{col} = 60.0\% - 83.0\%$
							0.001 wt%	-	$\eta_{col} = 64.0\% - 89.0\%$
							0.005 wt%	-	$\eta_{col} = 78.0\% - 93.0\%$
<a href="#">Chen et al. (2017) [100]</a>	<b>Experimental</b>	Tubular	40 mm	Water				$T_{out} = 71\text{ }^{\circ}\text{C}$	$\eta_{col} = 40\% - 70\%$
				Graphene (G)	Water	-	0.02 wt%	$T_{out} = 75.5\text{ }^{\circ}\text{C}$	$\eta_{col} = 48\% - 94\%$
				G-Oxide (GO)		L = 0.5 - 3 $\mu\text{m}$	0.02 wt%	$T_{out} = 77\text{ }^{\circ}\text{C}$	$\eta_{col} = 57\% - 95\%$
				Reduced-GO		L = 0.5 - 3 $\mu\text{m}$	0.02 wt%	$T_{out} = 82\text{ }^{\circ}\text{C}$	$\eta_{col} = 65\% - 96,93\%$
<a href="#">Gorji and Ranjbar (2017) [117]</a>	<b>Numerical</b>	Planar	20 mm	Water				-	$\eta_{th} = 27.5\%$
				Graphite	Water	D = 50 nm	39.8 ppm	-	$\eta_{th} = 84.6\%$
				Magnetite		D = 15 nm	40 ppm	-	$\eta_{th} = 94.3\%$
				Ag		D = 20 nm	38.6 ppm	-	$\eta_{th} = 51.4\%$
<a href="#">Mehrali et al. (2018) [99]</a>	<b>Experimental</b>	Planar	20 mm	Water (deionized)				-	$\eta_{th} = 28\%$
				Reduced-Graphene Oxide	Water	-	10-100 ppm	$\Delta T = 18\text{ }^{\circ}\text{C}$	$\eta_{th} = 54\% - 62\%$
				Ag-rGO (0.15 g of Ag)		-	10-100 ppm	$\Delta T = 18 - 24\text{ }^{\circ}\text{C}$	$\eta_{th} = 58\% - 77\%$
				Ag-rGO (0.30 g of Ag)		-	10-100 ppm	$\Delta T = 18 - 24\text{ }^{\circ}\text{C}$	$\eta_{th} = 53\% - 78\%$
<a href="#">Siavashi et al. (2018) [148]</a>	<b>Numerical</b>	Planar	5 - 20 mm	Water (deionized)				$T_{out} = 21 - 28\text{ }^{\circ}\text{C}$	$\eta_{col} = 15\% - 30\%$
				Water (deionized) with absorbing plate at the bottom				$T_{out} = 25 - 34\text{ }^{\circ}\text{C}$	$\eta_{col} = 37\% - 60\%$

				SWCNH	Water	D = 67 nm	5 - 40 ppm	$T_{out} = 22 - 36 \text{ }^\circ\text{C}$	$\eta_{col} = 25\% - 55\%$
				SWCNH with absorbing plate at the bottom				$T_{out} = 23 - 35 \text{ }^\circ\text{C}$	$\eta_{col} = 35\% - 59\%$
<b>Bhalla et al.</b> (2019) [146]	<b>Numerical</b>	Planar	6 - 10 mm	Amorphous carbon	Water	D < 50 nm	0.02 g/l	$\Delta T = 10 \text{ }^\circ\text{C}$	$\eta_{col} = 40\%$
							0.04 g/l	$\Delta T = 17 \text{ }^\circ\text{C}$	$\eta_{col} = 70\%$
							0.06 g/l	$\Delta T = 18.61 \text{ }^\circ\text{C}$	$\eta_{col} = 77\%$
<b>Hooshmand et al.</b> (2021) [147]	<b>Experimental</b>	Planar	20 mm	Water (deionized)				-	$\eta_{col} = 35\% - 65\%$
				Porous foam (SiC)	Water	-	-	-	$\eta_{col} = 40\% - 72\%$
				SiC	Water	D = 65 nm	1000 ppm	-	$\eta_{col} = 48\% - 78\%$
				SiC with porous foam (SiC)				-	$\eta_{col} = 57\% - 87\%$
<b>Struchalin et al.</b> (2021) [145]	<b>Num./Exp.</b>	Tubular	20 mm	Commercial FPC					$\eta_{th} = 65\% - 75\%$
				Commercial ETC					$\eta_{th} = 74\% - 77\%$
				MWCNT	Water	D = 49 - 72 nm L = 5 $\mu\text{m}$	0.0015 wt%	$\eta_{th} = 45\% - 70\%$	
							0.0040 wt%	$\eta_{th} = 60\% - 87\%$	
							0.0080 wt%	$\eta_{th} = 75\% - 94\%$	
							0.01 wt%	$\eta_{th} = 70\% - 97\%$	
0.02 wt%	$\eta_{th} = 58\% - 81\%$								

#### 4.C.- Concentrating NDASC

The majority of studies on concentrating NDASC involve small or medium size PTC with tubular or planar linear receivers. As fluid temperatures are higher compared to non-concentrating NDASC, thermal oils are usually preferred over water. The main characteristics and performances of concentrating NDASC are summarized chronologically in **Table 5** and reviewed in the following paragraphs.

Statements made in the previous subsection regarding the influence of the type of nanoparticle, particle size and shape, particle volume fraction and fluid depths can be applied to concentrating NDASC. Carbon-based particles have shown better performances than metal and metal-oxide particles in concentrating collectors [88,151], as concluded in previous section by other authors [26,86,96,111,116]. To the authors knowledge, the scientific literature dedicated to hybrid [152,153] and plasmonic [154] nanofluids is very scarce, preventing any conclusion concerning their potential in concentrating NDASC to be established. The influence of the volume fraction and fluid depth has been evaluated and similar conclusions are achieved: there is an optimum volume fraction and fluid depth for which the collector efficiency reaches a maximum value (**Figure 9**) [89,155–158]. To our knowledge, the impact of the particle size and shape on the efficiency of concentrating NDASC has not been studied. Other important parameters, such as the nature of the base fluid or the working temperature, will be discussed in detail in the following paragraphs.



**Figure 9:** Normalized average stagnation temperature as a function of the nanoparticle concentration for two-carbon based NDASC (amorphous carbon and MWCNT) and two surface collectors. Reprinted from Ref. [144] with permission from Elsevier.

Direct absorption parabolic trough collector (DAPTC) is by far the most commonly used concentrating technology in NDASC (see **Table 5**), thanks to its maturity (compared to LFC). Concentrating ratios (C) of DAPTC vary between 10 and 40 with relatively small receivers, around 20 mm in diameter in most cases, which can be classified as medium-size PTC (see **section 3**). O’Keeffe *et al.* [156] analyzed numerically the impact of C on the DAPTC efficiency and showed an increase in the collector efficiency until a threshold concentration value beyond which the efficiency reaches an asymptotic value due to increasing heat losses. Khullar *et al.* [159] concluded that the efficiency of the DAPTC decreases with the decrease of the incident radiation. From these works, it can be concluded that an optimum concentration ratio can be found for each system considering the fluid depths, which is related to the receiver size, and the size characteristics of the concentrator. Two studies used Fresnel lens as concentrating system, Singh *et al.* [158] and Li *et al.* [160,161], but no LFC or PDC have been used for concentrating NDASC.

Despite the domination of tubular receivers in the field of NDASC, planar receivers has also instigated some interest for NDASC [158,162,163]. Planar collectors offer the possibility of adding inner reflecting surfaces recovering the not-absorbed radiation by the fluid, which may lead to higher collector efficiencies as suggested by Singh *et al.* [158]. Otherwise, no-advantage of planar receivers over tubular receivers has been found. Khullar *et al.* [159] showed that the presence of vacuum layer surrounding the receiver led to reduced convective heat losses, translating into a 5 fold decrease in the heat losses and a 35% increase in the efficiency, relative to non-vacuum receivers. As outlined by several authors, vacuumed receivers only perform better than non-vacuumed receivers beyond a crucial working temperature [155,161]. When operating temperatures are below this crucial temperature, the lower convective heat losses do not compensate the lower optical efficiency of vacuumed receivers. Anti-emissivity coated receivers performance is also determined by a crucial operating temperature below which non-coated receivers are more efficient [156,157]. Therefore, vacuum receivers and anti-emissive coatings should be prioritized for concentrating NDASC operated at temperatures typically exceeding 150 °C.

The boiling temperature of water-based nanofluids limits their working range, the nanofluids being necessarily in liquid state to prevent particle deposition. Dugaria *et al.* [163] numerically showed outlet temperatures up to 143 °C using SWCNH at low concentrations (0.05 g/l) with a PTC and a planar receiver under pressure. Previous experimental study in similar conditions by the same authors demonstrated a significant particle deposition in the collector, lowering the collector efficiency by 18%. The stability issue was ascribed to an important degradation of the surfactant with increasing temperature [162]. Oil-based NDASC, however, can work at temperatures up to 250 °C (see **Table 5**) with Therminol VP-1 and WD synthetic oils being the most commonly used. Kasaeian *et al.* [89,155,164,165] studied ethylene glycol based nanofluids in a DAPTC; and proved the feasibility of such collector at temperatures up to 70 °C. Menbari *et al.* [166] compared a CuO/Al<sub>2</sub>O<sub>3</sub> based nanofluid using water and a water-ethylene glycol mixture (50% each), and found slightly better efficiencies for the water based nanofluids. In general terms, water-based NDASC seem to perform better than oil and ethylene glycol based-nanofluids, but the temperature restriction may be an important constraint for practical applications like SHIP.

Compared to conventional concentrating surface collectors, concentrating NDASC have demonstrated higher efficiencies at temperatures up to ~150 °C (see **Table 5**) [89,144,152,159–163,167–169]. Xu *et al.* [168] and Chen *et al.* [169] outlined that there is a threshold temperature beyond which surface collectors outperform NDASC due to higher radiative losses compared to that of the selective absorbing surface. Anti-emissivity coating should be used in these cases as recommended by Li *et al.* [161] and O’Keeffe *et al.* [156]. Temperature should be carefully controlled as it is a critical parameter affecting the collector efficiency as well as the stability of nanoparticles (see section 5 for more information about the influence of temperature in the nanofluid stability) [162]. An innovative solution has been proposed by Heyhat *et al.* [170] adding porous metal foams inside the nanofluid-based volumetric receiver. They found higher thermal efficiencies (19% higher) than for a CuO-based concentrating NDASC. Singh *et al.* [158] studied experimentally the efficiency of a DAPTC using an ‘used engine oil’ containing carbon nanoparticles with a rectangular receiver with reflective lateral surfaces and obtained up to 60% thermal efficiencies. Novel designs may help improving the performance of current concentrating NDASC but their commercial viability should be further analyzed.

Table 5: Summary of concentrating NDASC studies in the literature

CONCENTRATING NDASC											
Authors (year) [ref]	Approach				Nanofluid				Results		
	Exp./Num.	Geometry	$I_0$ ( $W/m^2$ )	Fluid depth	Nanoparticles	Base Fluid	Size	Concentration	Temperature	Efficiency	
Khullar <i>et al.</i> (2012) [159]	Numerical	PTC - Tube receiver	~ 60000	70 mm	SEGS LS-2 plant PTC collector (Syltherm 800)				$\Delta T = 16-240$ °C	$\eta_{th} = 70\% - 73\%$	
					Volumetric receiver (Therminol VP-1)				$\Delta T = 16-240$ °C	$\eta_{th} = 20\% - 25\%$	
					Al	Therminol VP-1	D = 5 nm	0.05 vol%	$\Delta T = 16-240$ °C	$\eta_{th} = 75\% - 81\%$	
									$\Delta T = 50-200$ °C	$\eta_{col} = 78\% - 81\%$	
Khullar <i>et al.</i> (2014) [144]	Experimental	Planar receiver	3000-5000	8 - 10 mm	Amorphous Carbon	Ethylene Glycol	D = 5 nm	10-150 mg/L	$\Delta T = \sim 50$ °C	-	
					MWCNT	Water	D = 31.5 nm L = 1.25 $\mu$ m	10-150 mg/L	$\Delta T = \sim 47$ °C	-	
Li <i>et al.</i> (2015) [160]	Experimental	Fresnel lens - Tube receivers	4800	10 mm	Surface tube receiver (coated copper tube)				$\Delta T = 10-60$ °C	$\eta_{col} = 74\% - 85\%$	
					MWCNT	Water	D = 6-13 nm L = 2.5-20 $\mu$ m	50 mg/L	$\Delta T = 10-60$ °C	$\eta_{col} = 54\% - 73\%$	
Xu <i>et al.</i> (2015) [168]	Numerical	PTC - Tube receiver	5152	45 mm	Surface receiver (WD synthetic oil)				$\Delta T = 125$ °C	$\eta_{col} = 42\% - 52.5\%$	
					CuO	WD synthetic oil	D = 200 nm	0.055 wt%	$\Delta T = 105$ °C	$\eta_{col} = 40\% - 62\%$	
Chen <i>et al.</i> (2016) [169]	Numerical	PTC - Tube receiver	4560	45 mm	Surface receiver (WD-350 synthetic oil)				-	$\eta_{col} = 64.0\%$	
					CuO	WD350 synth. oil	D = 200 nm	0.05 wt%	$T_{in} = 40-110$ °C	$\eta_{col} = 63\% - 70\%$	
									0.06 wt%	$T_{in} = 40-110$ °C	$\eta_{col} = 66\% - 72\%$
									0.075 wt%	$T_{in} = 40-110$ °C	$\eta_{col} = 65\% - 72\%$
									0.1 wt%	$T_{in} = 40-110$ °C	$\eta_{col} = 64\% - 71\%$
Li <i>et al.</i> (2016) [161]	Num./Exp.	Fresnel lens - Tube pipes	3840	10 mm	Surface tube receiver - Therminol 55				$T_{out} = 106-205$ °C	$\eta_{col} = 47\% - 68\%$	
					MWCNT	Therminol 55	-	50 mg/ml	$T_{out} = 105-204$ °C	$\eta_{col} = 26\% - 54\%$	
Membari <i>et al.</i> (2016) [171]	Num./Exp.	PTC - Tube receiver	~ 44600	20 mm	CuO	Water	D = 100 nm	0.002 vol%	$T_{out} = 28-36$ °C	$\eta_{th} = 28\% - 33\%$	
								0.004 vol%	$T_{out} = 30-39$ °C	$\eta_{th} = 36\% - 40\%$	
								0.008 vol%	$T_{out} = 31-42$ °C	$\eta_{th} = 41\% - 49\%$	
Toppin-Hector and Singh (2016) [88]	Numerical	PTC - Tube receiver	26000	5 - 25 mm	Graphene	Therminol VP-1	D = 5 nm	0.02 vol%	$T_{out} = 127-267$ °C	-	
					Al		D = 5 nm	0.09 vol%	$T_{out} = 127-267$ °C	-	
Bortolato <i>et al.</i> (2017) [162]	Experimental	PTC - Planar receiver	~ 37000	18 mm	Surface receiver (water)				-	$\eta_{col} = \sim 80\%$	
					SWCNT	Water	D = 2-5 nm L = 30-50 nm	0.02 g/L	-	$\eta_{col} = 80\%$ (3h exposure)	
									-	$\eta_{col} = 65\%$ (8h exposure)	
	Numerical		6531	26 mm	Volumetric receiver (Ethylene Glycol)				$T_{out} = 44-48$ °C	$\eta_{th} = 52.2\% - 59.5\%$	

<a href="#">Kasaeian et al. (2017.a)</a> [155]		PTC - Tube receiver			Silica (SiO <sub>2</sub> )	Ethylene Glycol	D = 6-15 nm	0.2-0.4 vol%	T <sub>out</sub> = 44-55 °C	η <sub>th</sub> = 55.9% - 70.9%
					MWCNT	Ethylene Glycol	D = 4-10 nm	0.2-0.5 vol%	T <sub>out</sub> = 50-61 °C	η <sub>th</sub> = 61% - 80.7%
<a href="#">Kasaeian et al. (2017.b)</a> [164]	Experimental	PTC - Tube receiver	7464-9330	24 mm	Volumetric receiver (Ethylene Glycol)				T <sub>out</sub> = 48.8 °C	η <sub>th</sub> = 55.8%
					Silica (SiO <sub>2</sub> )	Ethylene Glycol	D = 15 nm L = 10 μm	0.1-0.2- 0.3 vol%	T <sub>out</sub> = 51-57 °C	η <sub>th</sub> = 58% - 60% - 64%
					MWCNT		D = 10 nm L = 100 μm	0.1-0.2-0.3 vol%	T <sub>out</sub> = 55-65 °C	η <sub>th</sub> = 61% - 65% - 73%
<a href="#">Membari et al. (2017)</a> [166]	Experimental	PTC - Tube receiver	~ 42500	20 mm	Binary nanofluid: CuO + Al <sub>2</sub> O <sub>3</sub>	Water or a water/ethylene glycol mixture (50% each)	CuO: D < 100 nm	0.008 vol% CuO 0.2 vol% Al <sub>2</sub> O <sub>3</sub>	T <sub>out</sub> = 38-59 °C	η <sub>th</sub> = 36% - 48% (water)
							Al <sub>2</sub> O <sub>3</sub> : D < 40 nm	0.004 vol% CuO 0.1 vol% Al <sub>2</sub> O <sub>3</sub>	T <sub>out</sub> = 24-45 °C	η <sub>th</sub> = 35% - 45% (mix)
								0.002 vol% CuO 0.05 vol% Al <sub>2</sub> O <sub>3</sub>	T <sub>out</sub> = 30-46 °C	η <sub>th</sub> = 31% - 39% (water)
									T <sub>out</sub> = 31-50 °C	η <sub>th</sub> = 29% - 37% (mix)
									T <sub>out</sub> = 29-49 °C	η <sub>th</sub> = 23% - 31% (water)
		T <sub>out</sub> = 28-42 °C	η <sub>th</sub> = 22% - 30% (mix)							
<a href="#">Wang et al. (2017)</a> [151]	Experimental	CPC - Tube receiver	-	-	Graphene	WD thermal oil	L = 0.5-2 μm	0.06 mg/ml	-	η <sub>col</sub> = 80.0%
					Graphite		-	0.06 mg/ml	-	η <sub>col</sub> = 72.0%
					CuO		-	0.06 mg/ml	-	η <sub>col</sub> = 68.0%
<a href="#">Dugaria et al. (2018)</a> [163]	Numerical	PTC - Planar receiver	38700	12 - 18 mm	Surface receiver (water)				-	η <sub>th</sub> = 77% - 85%
					SWCNH	Water	-	0.006 g/l	T <sub>out</sub> = 130 °C	η <sub>th</sub> = 76% - 80.7%
								0.01 g/l	T <sub>out</sub> = 132 °C	η <sub>th</sub> = 81.8% - 84.2%
								0.02 - 0.05 g/l	T <sub>out</sub> = 134-143 °C	η <sub>th</sub> = 84.8% - 84.9%
<a href="#">O'Keeffe et al. (2018.a)</a> [156]	Numerical	PTC - Tube receiver	1000-25000	16 mm	Volumetric receiver (Therminol VP-1)				-	η <sub>col</sub> = 25.0%
					Al	Therminol VP-1	-	0.001 - 0.006 vol%	-	η <sub>col</sub> = 71% - 80%
<a href="#">O'Keeffe et al. (2018.b)</a> [157]	Numerical	PTC - Tube receiver	5000-20000	5 mm	Al	Therminol VP-1	-	0.001 vol%	-	η <sub>col</sub> = ~25.0%
								0.006 vol%	-	η <sub>col</sub> = ~73% - 76%
								0.01 vol%	-	η <sub>col</sub> = ~75% - 78%
<a href="#">Kasaeian et al. (2019)</a> [89]	Numerical	PTC - Tube receiver	7464-9330	26 mm	MWCNT	Ethylene Glycol	D = 4-10 nm	0.4 vol%	T <sub>out</sub> = 56-65 °C	η <sub>th</sub> = 68% - 79%
								0.5 vol%	T <sub>out</sub> = 59-69 °C	η <sub>th</sub> = 62.5% - 82.3%
								0.6 vol%	T <sub>out</sub> = 62-72 °C	η <sub>th</sub> = 76% - 82.3%
<a href="#">Heyhat et al. (2020)</a> [170]	Experimental	PTC - Tube receiver	25500-28500	22 mm	Porous foam (CuO)	Water	-	-	-	η <sub>th</sub> = 49.4%
					CuO	Water	D = 30 nm	0.01 - 0.1 vol%	-	η <sub>th</sub> = 53.5% - 64.6%
					CuO with porous foam			0.01 - 0.1 vol%	-	η <sub>th</sub> = 70% - 79.3%
	Experimental		~63000	15 mm	<u>Water (deionized)</u>			ΔT = 2.1 - 2.7 °C	η <sub>th</sub> = 13.8% - 15.9%	



<a href="#">Joseph et al. (2020) [154]</a>		PTC - Tube receiver			SiO <sub>2</sub> /Ag-CuO	Water	SiO <sub>2</sub> /Ag: D = 300 nm CuO: D < 50 nm	SiO <sub>2</sub> /Ag: 0.21 g/l CuO: 0.86 g/l	ΔT = 10.8 - 8.4 °C	η <sub>th</sub> = 55% - 64.1%
<a href="#">Khalil et al. (2020) [152]</a>	<b>Experimental</b>	PTC - Tube receiver	15000-19000	15 mm	Conventional PTC with Al <sub>2</sub> O <sub>3</sub> -CuO	Water	Al <sub>2</sub> O <sub>3</sub> : D < 100 nm	0.11 - 0.55 vol%	ΔT = 4.5 - 6 °C	η <sub>th</sub> = 26.6% - 45.4%
					Al <sub>2</sub> O <sub>3</sub> -CuO		CuO: D < 100 nm			
<a href="#">Ham et al. (2021) [167]</a>	<b>Experimental</b>	CPC - Tube receivers	1015	18 mm	Surface receiver (water)				-	η <sub>th</sub> = 42%
					Surface receiver (Fe <sub>3</sub> O <sub>4</sub> nanofluid with 0.001-0.1 wt%)				-	η <sub>th</sub> = 36% - 38%
					Fe <sub>3</sub> O <sub>4</sub>	Water	D = 6-15 nm	0.001 - 0.1 wt%	-	η <sub>th</sub> = 42% - 50 %
<a href="#">Saray and Heyhat (2022) [153]</a>	<b>Numerical</b>	PTC - Tube receiver	~28700	20 - 50 mm	Water (deionized)				-	η <sub>th</sub> = 30% - 34%
					CuO	Water	-	0.1 - 0.05 wt%	-	η <sub>th</sub> = 39% - 48%
					MWCNT		-	0.0015 wt%	-	η <sub>th</sub> = 51% - 55%
					CuO-MWCNT		-	CuO: 0.1 - 0.05 wt% MWCNT: 0.0015 wt%	-	η <sub>th</sub> = 60% - 66%

#### 4.D.- Conclusion

Non-concentrating NDASC are able to enhance the performance of conventional planar collectors by 10-15%, while concentrating NDASC exhibit only minor improvements in comparison with conventional concentrating surface collectors. Carbon nanoparticles typically show slightly better performances than metal or metal-oxide nanoparticles in both non-concentrating and concentrating collectors. The choice of the base fluid does not significantly affect the thermal performance of the collector but is an important factor limiting the working temperature range in concentrating NDASC. The thermal conductivity is enhanced by the addition of nanoparticles but represents less than 4% of the absolute thermal efficiency increase [86,87], and therefore should not be considered as a crucial parameter for the optimization of DASC efficiency. Operating temperatures exceeding 150 °C have been proved to affect the nanofluid stability considerably due to an important surfactant degradation, and thus prevent NDASC from outperforming conventional surface collectors. In the light of these observations, NDASC applications should mainly be focused in low or medium temperature heat supply. As outlined in section 2, SHIP plants offer a promising application sector for medium-size solar collectors with numerous case studies and well documented commercialization strategies. The typical dimensions of the different families of medium-size solar collectors being consistent with the characteristic size of NDASC, the combination of these two technologies could thus provide a significant added value for SHIP applications over other competing technologies. To our knowledge, no studies addressed the integration of NDASC in SHIP applications, only few studies being dedicated to residential heating and cooling [135,172] or water desalination [173]. Future NDASC studies considering the working conditions of SHIP applications should be carried out to quantify the potential of concentrating NDASC in the industrial sector.

### 5. Challenges

Alagumalai *et al.* [174] identified, analyzed and quantified ten main barriers preventing nanofluid commercialization, and suggested a roadmap to overcome them. In the following, five main challenges are addressed.

#### 5.A.- Stability of nanofluids

The long-term stability of nanofluids is a fundamental requirement for most of the practical applications and an important hurdle hindering nanofluid commercialization. Due to various forces (Van der Waals forces, electrostatic force, Brownian motion, etc...), nanoparticles are likely to agglomerate, causing sedimentation which alter the thermo-physical and optical properties of nanofluids, and may cause clogging of the hydraulic components. Some factors highly influencing the stability of nanofluids are: the dielectric constant of the base fluid, the zeta potential and the pH value of the nanofluid, the particle size and shape, and the particle volume fraction. The dielectric constant is directly proportional to the potential difference between two charged nanoparticles, which works as a barrier preventing particle agglomeration caused by Van der Waals forces [175]. Thus, base fluids with higher dielectric constant would offer more stable nanoparticle dispersions. The dielectric constant of water (78.5 at 20 °C) is higher than that of ethylene glycol or ethanol (24.6 at 20 °C), which makes water a better base fluid candidate [175,176]. Hordy *et al.* [177] analyzed the stability of plasma-functionalized MWCNT dispersed in various base fluids as a function of temperature and showed a high instability of the Therminol VP-1 nanofluid, while water and ethylene glycol based nanofluids showed a satisfying stability for 8 months.

The zeta potential indicates the degree of electrostatic repulsion between nanoparticles, with higher absolute values of the zeta potential indicating higher electrostatic repulsive forces. As an order

of magnitude, a nanoparticle dispersion is considered stable when the zeta potential is higher than 30 mV or lower than -30 mV [114,175,176,178,179] (**Figure 10**). The zeta potential, dependent of the particle material, decreases with the pH of the nanofluids. Consequently, low pH solutions (less than 6 [176,178]) exhibit a better stability than medium pH dispersions (**Figure 10**). If high pH values are reached, the zeta potential may drop below -30 mV, promoting the nanofluid stability [180,181,166]. However, very high (pH above 9) or very low (pH below 4) values may trigger an accelerated deterioration of the hydraulic equipment due to corrosion and restrict the possible nanofluid applications [114,179].

The stability of the nanofluid decreases with increasing particle size [114,176,182]. On the other hand, increasing the particle volume fraction induces higher Van der Waals attractive forces and therefore higher particle agglomeration [182]. This conclusion was achieved by several studies and suggests that low volume fraction nanofluids should be further investigated. Last, the working conditions of the collector have a critical influence on the stability of the dispersion. High temperatures induce high diffusion coefficients, likely to enhance the particle aggregation due to the high number of collisions [176]. Also, high temperatures have been proven to accelerate the surfactant degradation in nanofluids and thus affect the dispersion stability [114,162]. Deposition of nanoparticles is shown to be reduced in turbulent flow regime [145].

The method followed for synthesizing the nanofluid (which is either a one-step or a two-step method), is a key factor in the stability of nanofluids [114]. The one-step method allows a fine tuning of the particle size and shape, minimized particle agglomeration and, in turn, highly stable nanofluids. Unfortunately, the one-step method is only suitable for low quantities of nanofluid, and induce high production costs. The two-steps method is a cost-effective option adapted to large-scale production, but suffers from higher particle agglomeration which necessarily requires stabilization methods to be applied. Further information regarding preparation methods and supplementary stabilization techniques can be found in Mukherjee *et al.* [175] and Chakraborty and Panigrahi [176] reviews. New emerging synthesis techniques, such as the innovative method of Carbon Waters® for aqueous dispersions of exfoliated nanocarbons [183], are expected to improve the stability and to reduce the current production cost of nanofluids.

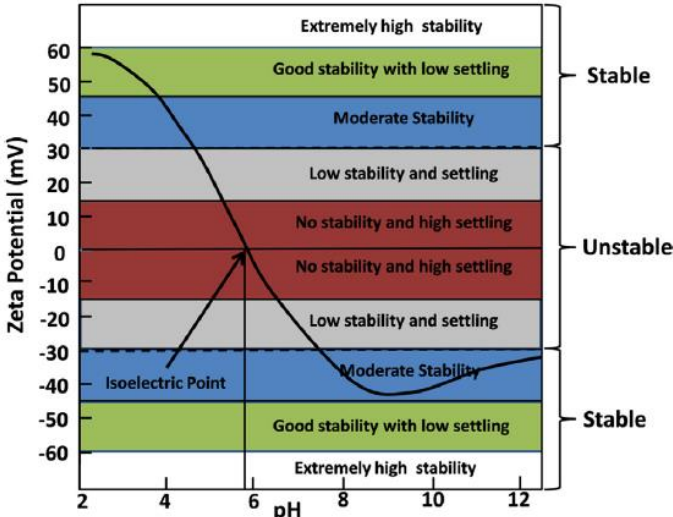


Figure 10: Scheme of the stability zones as a function of the zeta potential (mV) and the pH of the fluid. Reprinted from Ref. [176] with permission from Elsevier.

### 5.B.- Toxicity and environmental impact

Due to their very small size, nanoparticles are able to enter living organisms, facilitating bacterial mutation, cell and protein degeneration and other harmful changes affecting living beings. Toxicological studies have demonstrated that the toxicity level increases for decreasing nanoparticle size [139,184]. The geometry and the chemical properties also affect the toxicity level of a given family of nanoparticles [185,186]. SWCNTs, tubular particles, were reported to induce higher pulmonary toxicity than spherical carbon black particles [184]. These particles show higher toxicity than quartz dust, that are known for causing serious lung diseases [187,188]. For metal nanoparticles, Schrand *et al.* [189] compared and ranked commonly used nanoparticles based on *in vitro testing*: Zn-based > Cu > Ag > Fe<sub>3</sub>O<sub>4</sub> > Al<sub>2</sub>O<sub>3</sub> > TiO<sub>2</sub>. The toxicity level of nanofluids used in NDASC is highly difficult to evaluate, since it is affected by many different factors (base fluid, particle, surfactants, stability, volume fraction, composition, uniformity, etc...). Further studies and experimental data on the toxicity of nanofluids are needed to determine the toxicity limit, the security measures, and the nano-wastes management in the future [27,109,174,190].

As outlined above, NDASC can achieve better efficiencies than conventional collectors, which may generate important energy savings and CO<sub>2</sub> emissions drop. Otanicar *et al.* [191] evaluated the CO<sub>2</sub> emissions saving associated with a planar NDASC over a 15 year lifetime, and estimated that respectively 740 kg and 23 000 kg CO<sub>2</sub> emissions could be saved in comparison with a conventional FPC and a traditional water heater, respectively. The use of a CuO<sub>2</sub> water-based nanofluid as HTF on a conventional FPC was shown to offset CO<sub>2</sub> emissions by 175 kg in comparison with water. Gupta *et al.* [192] evaluated the energy savings associated with the substitution of 50% of the conventional collectors currently installed by NDASC, and concluded that the corresponding energy reduction would represent 857.5 millions of MJ/year. However, at the end of their life cycle, NDASC generate nano-wastes, whose management still needs to be improved and standardized (see Part *et al.* [193] for further information about nano-waste characterization and separation techniques). In the Younis *et al.* [194] book section, the nano-waste risk profiles of different nanomaterials are presented and classified, allowing the reader to identify and estimate the precautions for managing nano-waste.

### 5.C.- Pumping power

The addition of nanoparticles induces an increase in the viscosity of the base fluid, which entails higher pumping power and, thus, higher operational costs [114]. The viscosity of the base fluid was shown to be only slightly affected by the incorporation of graphene nanoplatelets [108] and nitrogen-doped graphene [107] for low quantities of nanoparticles (less than 0.1 wt%). The influence of the MWCNT shape on the nanofluid viscosity was scrutinized by Omrani *et al.* [195], who demonstrated that high aspect ratio particles induce less viscosity growth. Chiam *et al.* [196] quantified dependence of viscosity on the volume fraction of nanoparticles and underlined the significant viscosity growth expected for large volume fractions. However, the range of volume fraction investigated by these authors exceeds the typical particle concentration commonly associated with NDASC (which is typically comprised between 0.001 and 0.1 vol% (see **Table 4** and **5**)) and thus expected viscosity enhancements are low. Nevertheless, due to the positive correlation between viscosity and nanoparticle concentration, nanofluids characterized by lower particle volume fraction should offer promising properties and should thus be privileged.

### 5.D.- Erosion and corrosion of components

Erosion and corrosion associated with the incorporation of nanoparticles constitute a serious hurdle which must necessarily be tackled. Celata *et al.* [197] and Fotowat *et al.* [198] studied the corrosion of both metal and metal-oxides nanoparticles and observed that stainless steel pipes were

not affected, unlike aluminum and copper pipes, which both show significant degradations. Stainless steel components are therefore recommended for the hydraulic circuit of the NDASC installation. The influence of the type of nanoparticles was also investigated by several authors, who demonstrated lower corrosion for carbon-based nanofluids in comparison with metal or oxide based nanofluids [113,199,200].

### 5.E.- Production costs

The cost of selective nanoparticles together with the synthesis costs leads to nanofluids costs that are much higher than those of conventional HTF in surface collectors (water or thermal oils). The economic analysis of Wciślik [201] and Alirezaie *et al.* [202] underlines the low cost of metal-oxides nanoparticles ( $\text{TiO}_2$ ,  $\text{Al}_2\text{O}_3$ ,  $\text{CuO}$  or  $\text{SiO}_2$ ) in comparison with metal particles (Ag, Al, Cu or Au), which are typically 10 times more expensive. Carbon-based nanotubes (single or double walled) are typically 10 times more expensive than metal nanoparticles, and 100 times more expensive than metal-oxides [202]. The prices of a given nanofluid is more subtle to quantify, because of the influence of the synthesis method as well as the cost calculation scenario followed [83,191,201,202]. As an order of magnitude, nanofluid cost with volume fractions of 0.04-0.1 are between 2-10 US\$/L for metal-oxides, 100-500 US\$/L for metals, and 300-1000 US\$/L for carbon nanotubes [201,202]. Despite their lower efficiency in comparison with carbon-based NDASC, metal-oxide base NDASC are more cost-effective and thus show higher potential for rapid commercialization. Nanofluid prices are expected to decrease in the following years thanks to new synthesis techniques, however the increase of nanomaterial demand for numerous novel nanotechnology could offset these trends and prevent the rapid commercialization of NDASC.

## 6. Discussion and perspectives

The present work provides an overview of the application of solar energy for industrial processes involving low and medium temperature operation; the current solar thermal technologies used are exposed, and the latest progress on nanofluid-based direct absorption solar collectors (NDASC) are discussed. Important conclusions obtained from the literature review are summarized hereafter.

- Solar thermal collectors have huge potential in the industrial sector, particularly in the food sector, in countries with good solar resource ( $>1700 \text{ kWh/m}^2$  yearly total global horizontal irradiation). For less irradiated countries, governmental and international incentives for SHIP plants are required to replace progressively conventional carbon-based energy sources during a transition period, as previously done for photovoltaic (PV).
- The majority of SHIP installations require heat temperatures between  $60 \text{ }^\circ\text{C}$  and  $150 \text{ }^\circ\text{C}$ . Concentrating collectors, mainly PTC, show improved performance over planar collectors in this temperature range. Since, PTC valorize the direct solar irradiation only, yearly energy production must be simulated for choosing the best technology between concentrating and non-concentrating collectors for each application, in particular above  $100^\circ\text{C}$ .
- Among the different technologies discussed in section 3, PTC and LFC are the most promising options for SHIP applications, allowing the integration of a nanofluid-based volumetric receiver.
- The performance of NDASC is highly affected by the fluid temperature, which influences the nanofluid stability and increases heat losses with the environment. The nanofluid stability in medium temperature NDASC (i.e.  $100\text{-}400 \text{ }^\circ\text{C}$ ) needs to be further studied to better understand the temperature limitations. Heat losses may be reduced with vacuumed receivers and low-emissivity coated glass envelopes [156].

- NDASC optimal working conditions and heat demand of SHIP installations are close, which presents the industrial sector as a potential application sector for NDASC. Nevertheless, performance and viability need to be thoughtfully studied.

Further research involving outdoor experimental collectors is needed to carefully evaluate the performance and feasibility of NDASC in real working conditions and analyze the effect of nanoparticle size and shape. Carbon nanoparticles are particularly promising for solar applications, their performance being even higher than those of metal or oxides nanofluids [89,96,116,151,155,164,165]. Further research is needed on the technical-economic optimization of volumetric receivers depending on the working temperature range, focusing on low emissivity coatings for inner glass envelope with solar transmittance greater than 90%. Finally, performance and economic studies should be initiated to quantify the potential of NDASC for SHIP applications, as well as residential heating and cooling. The steps to follow towards the development of a NDASC from the laboratory scale to a commercial collector for process heat applications are resumed in **Figure 11**. Another application of interest which should be investigated is hybrid PVT systems that use nanofluids as optical filters to produce both electricity and process heat [203–226]. In this latter domain the variation of optical properties with wavelength is a key parameter that governs the overall efficiency of the hybrid converter. Photochemistry is a promising application sector for nanofluids as well.

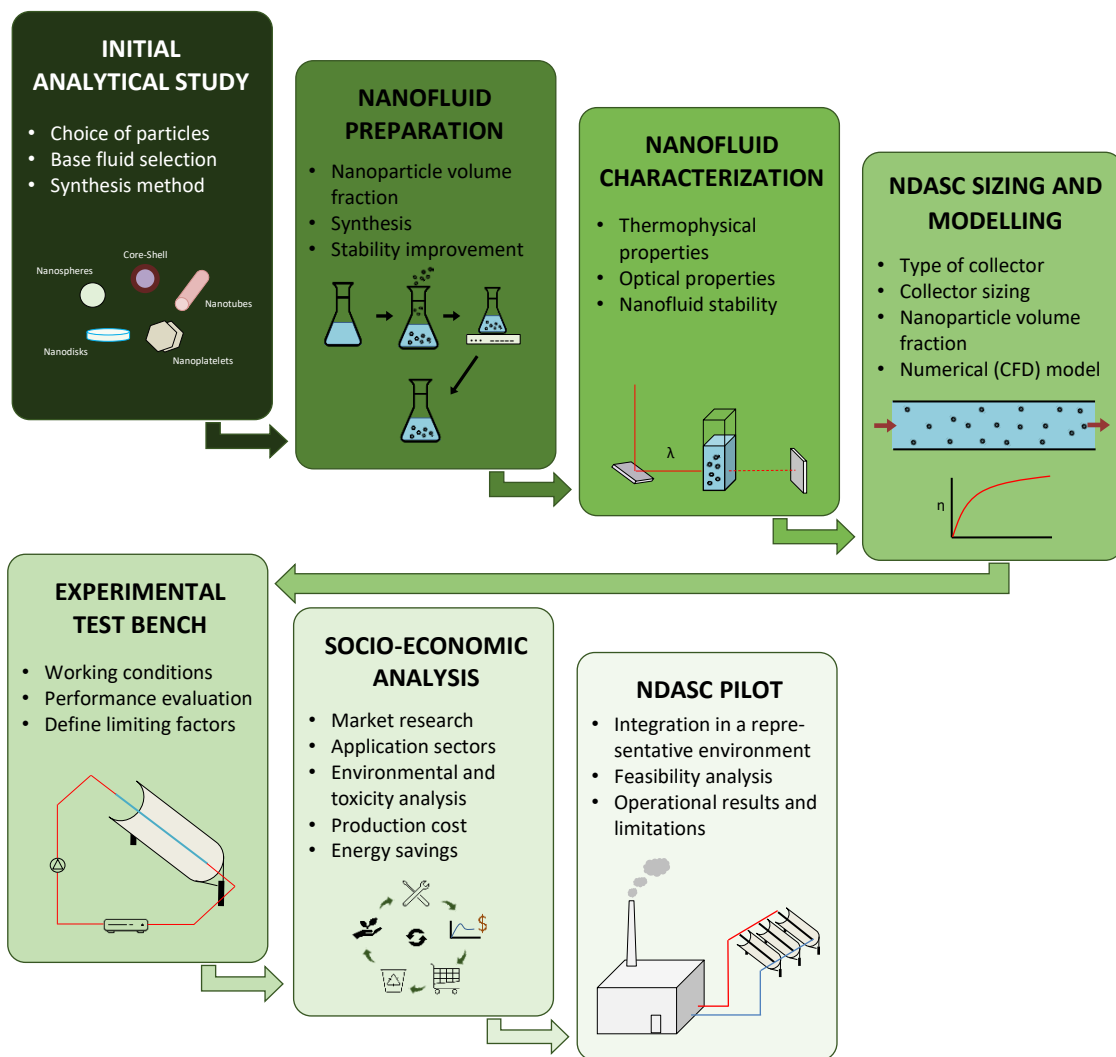


Figure 11: Steps for future NDASC commercialization towards SHIP applications.

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