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### <sup>1</sup> Controlling chaotic oscillations in a symmetric two-mass model of the vocal <sup>2</sup> folds

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#### 8 Abstract

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Human phonation is a highly non-linear process in which subglottal flow emanating from the lungs induces self-oscillations of the vocal folds. In normal conditions, this results in the generation of a regularly pulsating 10 volume velocity that becomes the source of acoustic waves, which once modulated by the vocal tract, get 11 emitted outwards as voice. However, vocal folds oscillations can become chaotic under many circumstances. 12 For instance, even in the case of healthy symmetric vocal folds, an excess value of the subglottal pressure can 13 trigger chaotic motion. In this paper, we derive a chaos control strategy for a two-mass model of the vocal 14 cords to revert this situation and render the motion regular again. The approach relies on slightly altering 15 the system energy to move it to a stable state. Given that no external control forces can be applied to the 16 vocal cords, it is proposed to add a third mass to the original two-mass model, which is assumed to be made 17 of an ideal smart material. The mass of the smart material is presumed negligible in comparison to the 18 two masses of the vocal folds model, but its damping and stiffness can be tuned to evolve with time. For a 19 fixed subglottal pressure for which the motion is chaotic, it is shown how periodicity can be recovered using 20 adequate damping laws, by either attaching the smart material onto the larger vocal fold mass or onto the 21 smaller one. For the latter, control turns to be more difficult and the damping of the smart material has to 22 quickly vary with time. On the other hand, given that the subglottal pressure would rarely be constant in a 23 real situation, we also introduce a damping law to avoid chaotic motion as the subglottal pressure augments 24 or diminishes. Finally, it is shown that control can not only be achieved by acting on the damping of the 25 smart material but also on its stiffness. A stiffness law to prevent chaotic oscillations and get a healthy 26 pulsating volume velocity is therefore implemented. A brief discussion on the mid-long term potential of 27 the presented solution for practical cases is included. 28

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Keywords: Vocal fold pacemaker, Vocal fold dynamics, Chaotic self-oscillations, Chaos control, Smart
 materials, Two-mass models

#### 32 1. Introduction

The essentials of the myoelastic aerodynamic theory of human phonation are nowadays well understood [1, 2]. In a nutshell, the air emanating from the lungs reaches the larynx and exerts a pressure on the vocal folds which open once surpassed a certain threshold value. This opening causes the pressure to suddenly drop and as a consequence the vocal folds close again until the subglottal pressure becomes strong enough to force a new aperture. In this way, self-oscillations of the vocal folds are established. The regular opening/closing of the vocal cords results in a glottal volume velocity that acts as a source term for the

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waves. The latter propagate and become modulated inside the vocal tract and get finally emitted outside the mouth, as voice.

The process of phonation is highly non-linear due to the complex combination of non-linear biomechanics 41 involving impacts and aerodynamics, and can often lead to chaotic oscillations of the vocal folds, which may 42 induce abnormal and unpleasant voice. This is often the case in voice pathologies like unilateral paralyses 43 or in lesions involving nodules, polyps or cysts. To better understand the physics behind those phenomena, 44 several low-dimensional models of the vocal folds were developed during the past half century, consisting 45 of mass-spring-damper systems driven by glottal flow excitation. The investigation on the so-called mass 46 models began in the earlier seventies, being probably the two-mass model of Ishizaka and Flanagan the 47 most celebrated one [3]. However, it was not until the nineties that non-linear dynamics methods were 48 used to understand the bifurcation processes that led from regular vocal fold motion to chaotic one (see 49 e.g., [4–6]). Investigations on lumped mass models of increasing complexity [7–10] continued through the 50 next decades and low dimensional models for the vocal fold self-oscillations [11-13] and glottal flows [14, 15]51 still constitute a very active area of experimental and theoretical research. In fact, it would be interesting 52 to check whether advances and new proposals in the classification of attractors and bifurcation theory [16], 53 like the identification of hidden attractors [17-19] and/or hyperchaos [20-22] could be of relevance for some 54 of these lumped models of the vocal folds and if they could have physical implications in terms of voice 55 production. 56

Despite all efforts made to understand the physical mechanisms of voice disorders, the possibility of 57 establishing control strategies that could revert chaotic vocal fold vibrations to regular ones have not been 58 explored yet. The main novelty of this paper is to examine if that could be feasible or not. To the best of our 59 knowledge this has not been attempted before. Using low dimensional mass models, we aim at a theoretical 60 design of a vocal fold pacemaker that could improve voice quality. It is to be noted that laryngeal pacemakers 61 have been amongst us for decades [23–25] and recent progress have been achieved on them [26–28]. However, 62 such pacemakers are not intended at all to deal with voice generation problems but are designed to solve 63 critical medical problems related to abduction and inspiration. 64

In this paper we will make use of the two-mass model in [5] and propose a control approach for chaotic 65 vocal fold oscillations. To facilitate the analysis we will deal with the easiest case in which the two-mass 66 model is totally symmetric, i.e. there are no geometrical or physical differences between the left and right 67 vocal cords. It was shown in [29] that even in this simple situation chaotic motion could arise for high 68 enough subglottal pressure values. This results in irregular glottal volume velocity and, therefore, in poor 69 voice quality. There exist several feedback chaos control schemes that are useful for non-linear mechanical 70 systems involving impacts, like the ones we encounter in the modelling of the vocal folds. The popular 71 Ott-Grebogi-Yorke (OGY) strategy [30] essentially locates an unstable periodic orbit embedded in a chaotic 72 attractor and then stabilizes the orbit by inducing small perturbations on a control parameter. The OGY 73 method has been successfully applied to a large variety of problems and it performs well in those with impacts, 74 see e.g. [31]. On the other hand, since the proposal of delayed feedback control strategies in [32, 33] a vast 75 amount of feedback control algorithms have been successfully developed (see e.g., the reviews in [34-36]), 76 which have also turned to be very efficient in stabilizing a large variety of mechanical problems that include 77 contact [37–40]. More recently, another feedback strategy was suggested which does not require knowing 78 the system equations and basically relies on its output time series [41, 42]. The basic idea in [41, 42] is 79 to slightly alter the time averaged oscillation energy (kinetic plus potential) of the system to move it to a 80 value where it becomes stable. Such approach has proved valid for classical non-linear oscillators (e.g., Van 81 der Pol and Duffing oscillators), for networks of oscillators, and for colliding elements as well [43]. Given 82 that our two-mass model driven by the glottal flow involves more than one mass and also deals with contact 83 between vocal folds, we have relied on the strategy in [41, 42] to help transitioning the chaotic motion of 84 the vocal folds to periodic one, and get an appropriate glottal volume velocity. It is to be remarked that 85 the volume velocity will be our variable of primary interest throughout the paper. 86

The main novel contributions of this work are as follows. First, let us note that most chaos controlling approaches, like that in [41, 42], depend on applying an external control force to stabilize the system. Given that this would not be possible in the case of the vocal folds, our proposal is to substitute that force by the inclusion of a third mass, made of a smart material, in the original two-mass model of the vocal folds. The



Figure 1: Schematic representation of the two-mass model of the vocal folds described by Eq. (4). As we are considering the symmetric case, only the left vocal cords and its associated variables and parameters are represented.

smart material is assumed to be ideal in the sense of its mass being negligible in comparison to the lower 91 large mass and upper small mass of the vocal folds model, and also in the sense of having the property 92 that its damping and stiffness can be tuned to evolve with time without restrictions. In some way, the 93 suggested strategy corresponds to swapping the external control force from the right hand side of the system 94 of equations to the left hand side, and integrate it as an internal force with variable parameters. After 95 establishing the new model, we propose different laws for reverting the vocal cords motion from chaotic 96 to periodic. For a fixed subglottal pressure value, if one attaches the smart material to the large mass of 97 the vocal folds it suffices to increase the overall damping value to transition to a stable state. However, 98 achieving control by gluing the smart material to the small mass is by far more subtle and the time evolving 99 law in [41, 42] becomes necessary to get a robust solution, in which the vocal folds still collide closing the 100 glottis. On the other hand, given that in practice the subglottal pressure that drives the vocal cord self-101 102 oscillations will not be constant, we introduce a new feedback mechanism to guarantee a periodic volume velocity in such situation. Finally, we show that chaos control can not only be achieved by modifying the 103 damping of the smart material but also by changing its stiffness. An appropriate law is introduced to do so. 104 The paper is organized as follows. In section 2, we present the ODE system corresponding to the 105 symmetric two-mass model in [5, 29]. Section 3 contains a brief review of the performance of that model 106 when varying the subglottal pressure and the contact stiffness between masses. Special emphasis is placed 107 on the consequences for the glottal volume velocity when transitioning to chaotic motion. In section 4, we 108 start reviewing the control strategy in [41, 42] and then propose the idea of applying it to our problem by 109 including a third smart material mass into the system. The section continues by exploring several damping 110 laws to either control the large mass or the small mass of the system and analyses the impact that this 111 has on the volume velocity. Then, the case of non-constant subglottal pressure is addressed and a chaos 112 control approach is derived for it. Finally, it is shown that stabilization can also be achieved by modifying 113 the stiffness of the smart material. A short discussion follows considering the mid-long term potential of 114 the proposed theoretical solution based on currently existing smart materials and their properties [44]. The 115 conclusions close the paper in section 5. 116

#### <sup>117</sup> 2. Non-linear ODE system for the symmetric vocal fold two-mass model

Through years, several models have been proposed to simulate the underlying physics of phonation and 118 in particular the onset of self-oscillations of the vocal folds. The latter are located in the larynx, which 119 connects the trachea and the vocal tract, and can be mechanically represented as a distributed system of 120 masses, springs and dampers. The glottis is the opening between vocal folds. The easiest models that are 121 able to reproduce the essential physiological and mechanical properties of phonation, namely the interaction 122 between the vocal fold geometry and the glottal airflow obeying Bernouilli's equation are the two-mass 123 models [3]. In such models, a vocal cord is separated in depth (thickness) into an upper and lower part, 124 each one consisting of an oscillator characterized by mass, stiffness and damping values (see Fig. 1). If one 125 considers the case of symmetric vocal folds, it suffices to study the dynamics of either the right or left ones. 126 The latter have been chosen in this work. The phase lag between the lower and upper masses  $m_{1l}$  and  $m_{2l}$ 127 in Fig. 1 is essential for the generation of self-sustained oscillations, which can not be reproduced by single 128 mass models. 129

Assuming that the cubic nonlinearity of the oscillators in the Ishizaka and Flanagan two-mass model is small, that the interaction of the vocal folds and glottal flow dynamics are independent of subglottal and supraglottal resonances in the trachea and vocal tract, and that the approximation of Bernouilli's flow applies, the equations of motion for  $m_{1l}$  and  $m_{2l}$  can be derived from Newton's second law (see e.g., [5, 29] for more details),

$$m_{1l}\ddot{x}_{1l} + r_{1l}\dot{x}_{1l} + k_{1l}x_{1l} + k_{cl}(x_{1l} - x_{2l}) + \Theta(-a_1)c_{1l}\frac{a_1}{2L} = P_1Ld_{1l},$$
(1a)

$$m_{2l}\ddot{x}_{2l} + r_{2l}\dot{x}_{2l} + k_{2l}x_{2l} + k_{cl}(x_{2l} - x_{1l}) + \Theta(-a_2)c_{2l}\frac{a_2}{2L} = 0,$$
(1b)

where,  $r_{il}$  (i = 1, 2) stand for the damping constants,  $k_{il}$  (i = 1, 2) for the stiffness ones and  $k_{cl}$  is the coupling stiffness between the upper and lower masses (see Fig. 1). The dampers  $r_{il}$  account for the mass stickiness when the moist surfaces of the left and right vocal folds do have contact [3]. The springs with stiffness  $k_{il}$  are a representation of the tension of the vocal folds, which is controlled by the contraction of the anterior cricothyroid muscle, among others [2, 3]. In what concerns the coupling stiffness,  $k_{cl}$ , it characterizes the flexural stiffness in the lateral direction of the vocal cords. Likewise, the fifth term on the left hand side (l.h.s) of Eqs. (1a) and (1b) is the non-linear collision force between the right and left vocal folds.  $a_i = a_{0i} + 2Lx_{il}$  (i=1,2) are the lower and upper glottal areas with L being the length of the glottis and  $a_{0i} = 2Lx_{0il}$  (i=1,2) the lower and upper glottal rest areas ( $x_{0il}$  stands for the distance of  $m_{il}$  to the midline in the rest position). The collision function  $\Theta$  is approximated by  $\Theta(z) = \tanh(50z/z_0)$  for z > 0 and becomes zero if  $z \leq 0$ .  $z_0$  is taken as  $a_{01}$  in the computations.  $c_{il}$  (i = 1, 2) are additional spring constants during collision. As regards the force term in the right hand side (r.h.s) of Eq. (1a),  $d_{1l}$  is the thickness of  $m_{1l}$  and  $P_1$  is the glottal pressure acting on it. Bernoulli's equation allows one to derive the following expression for  $P_1$ ,

$$P_1 = P_s \left[ 1 - \Theta(a_{\min}) \left( \frac{a_{\min}}{a_1} \right)^2 \right] \Theta(a_1), \tag{2}$$

with  $P_s$  being the subglottal pressure and  $a_{\min} = a_1$  if  $x_{1l} < x_{2l}$  while  $a_{\min} = a_2$  if  $x_{2l} \le x_{1l}$ .

An important parameter for voice generation is the volume flow velocity, U. This acts as a source term which excites acoustic waves in the vocal tract that become radiated as voice outside the mouth. U is typically prescribed as a boundary condition at the glottis in numerical voice production studies (see e.g., [45–48]). The volume flow velocity can be computed as,

$$U = \left(\frac{2P_s}{\rho}\right)^{1/2} a_{\min}\Theta(a_{\min}).$$
(3)

Eqs. (1a) and (1b) can be converted into a first-order non-linear ODE system of the type  $\dot{\boldsymbol{y}} = \mathbf{A}(\boldsymbol{y})\boldsymbol{y} + \boldsymbol{f}(\boldsymbol{y})$  by introducing the mass velocities  $v_{1l} = \dot{x}_{1l}$  and  $v_{2l} = \dot{x}_{2l}$  as independent variables. This yields,

$$\dot{\boldsymbol{y}} \equiv \begin{pmatrix} \dot{\boldsymbol{v}} \\ \dot{\boldsymbol{x}} \end{pmatrix} = \begin{pmatrix} -\mathbf{M}^{-1}\mathbf{C} & -\mathbf{M}^{-1}(\mathbf{K} + \boldsymbol{\Theta}(\boldsymbol{x})) \\ \mathbf{I} & \mathbf{0} \end{pmatrix} \begin{pmatrix} \boldsymbol{v} \\ \boldsymbol{x} \end{pmatrix} + \begin{pmatrix} \mathbf{M}^{-1}\boldsymbol{f}_{\boldsymbol{v}}(\boldsymbol{x}) \\ \mathbf{0} \end{pmatrix}$$
$$\equiv \mathbf{A}(\boldsymbol{y})\boldsymbol{y} + \boldsymbol{f}(\boldsymbol{y}) \equiv \boldsymbol{g}(\boldsymbol{y}), \tag{4}$$

where  $\boldsymbol{v} = (v_{1l}, v_{2l})^{\top}$  is the mass velocity vector and  $\boldsymbol{x} = (x_{1l}, x_{2l})^{\top}$  the displacement one, so that  $\boldsymbol{y} = (\boldsymbol{v}, \boldsymbol{x})^{\top}$ , as identified in Eq. (4). The force vector component is given by  $\boldsymbol{f}_{\boldsymbol{v}} = (P_1 L d_{1l}, 0)^{\top}$ . It is also common to express Eq. (4) as  $\dot{\boldsymbol{y}} = \boldsymbol{g}(\boldsymbol{y})$ , with  $\boldsymbol{g}(\boldsymbol{y}) = \mathbf{A}(\boldsymbol{y})\boldsymbol{y} + \boldsymbol{f}(\boldsymbol{y})$ , as shown in the last equality of Eq. (4). The block matrices in  $\mathbf{A}$  correspond to the identity  $\mathbf{I}$ , and to the mass,  $\mathbf{M}$ , damping,  $\mathbf{C}$ , stiffness,  $\mathbf{K}$  and collision,  $\boldsymbol{\Theta}$ , matrices given by,

$$\mathbf{M} = \operatorname{diag}(m_{1l}, m_{2l}), \qquad \mathbf{C} = \operatorname{diag}(r_{1l}, r_{2l}), \mathbf{K} = \begin{pmatrix} k_{1l} + k_{cl} & -k_{cl} \\ -k_{cl} & k_{2l} + k_{cl} \end{pmatrix} \qquad \mathbf{\Theta} = \operatorname{diag}\left(\Theta(-a_1)c_{1l}\left[\frac{a_{01}}{2Lx_{1l}} + 1\right], \Theta(-a_2)c_{2l}\left[\frac{a_{02}}{2Lx_{2l}} + 1\right]\right).$$
(5)

<sup>131</sup> Note that the non-linearity enters the ODE system through  $\Theta(x)$  and  $f_v(x)$ . The dynamics of Eq. (4) <sup>132</sup> has been analysed at extent in [5, 29], showing that for slightly higher values of the coupling constant  $k_{cl}$ <sup>133</sup> than usual, and for large subglottal pressure values,  $P_s$ , the motion of the two vocal fold masses can become <sup>134</sup> aperiodic, or even chaotic. We will briefly review some of these aspects in the following section. It is to be <sup>135</sup> noted that such abnormal behaviour of the vocal folds can have a profound impact on the glottal volume <sup>136</sup> flow velocity U, and therefore, in the quality of the generated voice.

# <sup>137</sup> 3. Chaotic motion of the vocal folds at high subglottal pressure and effects on the glottal <sup>138</sup> volume flow velocity

For subsequent simulations, the physical and geometrical parameters in Table 1 have been employed. 139 Throughout this text, the dimensions of the various variables will not be written (though they will be explic-140 itly indicated in figures). They correspond to appropriate combinations of taking the following dimensions 141 for length [L] = cm, time [T] = ms and mass [M] = g. It is to be noted that the standard value for the 142 coupling stiffness of  $k_{cl} = 0.025$  has been increased to  $k_{cl} = 0.09$  in the simulations following [29]. As said, 143 the system in Eq. (4) was studied in detail in the past. Its dynamic behaviour dependence on the values 144 of various model parameters was investigated and standard methods of non-linear analysis were applied 145 to it. In [5], the existence and stability of fixed points was addressed through linearisation, while in [29] 146 a combination of techniques were used to discriminate under which conditions the motion of the masses 147 becomes chaotic. The authors found that the system exhibits one positive Lyapunov exponent for large 148 enough subglottal pressure values, which proves the existence of chaos (a Kaplan-Yorke dimension of 2.4 149 was reported indicating low-dimensional chaos). Despite being four dimensional, hyperchaos is therefore not 150 possible in this particular system because at least two positive Lyapunov exponents are necessary for that 151 to occur (see e.g. [20-22]). Strange attractors at the phase spaces of the two masses were reported in [29], as 152 well as broadband noisy spectra and complex Poincaré sections. Bifurcation plots for the mass displacement 153 amplitudes in terms of  $k_{cl}$ ,  $P_s$  and further parameters were shown pointing out to a period doubling route 154 to chaos. 155

In this section, we will only outline the essentials of the behaviour of the system in Eq. (4) in a nutshell. The interested reader is referred to the original works [5, 29] for details. In contrast to the general approach in [5, 29], however, special emphasis will be placed herein on the effects that the vocal fold dynamics have on the volume velocity U. All simulations in this work have been carried out implementing a standard fourth-order Runge-Kutta numerical method.

One can basically distinguish three different regimes for the vocal folds motion when we fix all system parameters but progressively increase the subglottal pressure  $P_s$ . These are captured in Fig. 2, where we plot different type of results for  $P_s = \{0.0035, 0.021, 0.036\}$ . Each row in the figure corresponds to a value <sup>164</sup> of  $P_s$  that is representative of such regimes. The first column contains trajectories in the phase space for  $m_{1l}$ , <sup>165</sup> the second column shows the time evolution of  $x_{1l}$  and the third one plots the time evolution of U. The <sup>166</sup> results for  $m_{2l}$  are very similar to those of  $m_{1l}$  and add no physical insight into the problem. Consequently <sup>167</sup> they have not been included for the sake of brevity. The blue lines in the figure have been obtained from <sup>168</sup> the solution of Eq. (4) using the set of initial conditions  $y_1(0) = (0, 0, 0.1, 0.1)^{\top}$ , while the red dashed ones <sup>169</sup> correspond to the neighbouring initial condition  $y_2(0) = (0, 0, 0.11, 0.11)^{\top}$ .

If we focus on the first row of the figure we observe that the pressure  $P_s = 0.0035$  is unable to generate 170 self-oscillations of the vocal folds. The trajectories in phase space are spirals that decay from the initial 171 condition to the fixed point (0,0), i.e. the mass  $m_{1l}$  reaches the zero rest position where it stands still. 172 Observe that the two spirals in the figure corresponding to the initial conditions  $y_1(0)$  and  $y_2(0)$  remain 173 close to each other until they reach the origin. As regards the time evolution of  $x_{1l}$ , it shows a few oscillations 174 until the mass stops. Again, the two lines in the figure remain very close one to each other but do not match. 175 The glottal volume flow velocity U in the third subfigure of the first row becomes zero in a few occasions 176 because of the impact between vocal folds, and then acquires a small residual constant value because the 177 vocal folds do not contact in the rest position. 178

The situation in the second row would be the desirable one for voice generation (taking into account 179 the very severe simplifications assumed in the model of Eq. (4)). When the subglottal pressure increases, 180 at some point the solution exhibits a Hopf bifurcation and a limit cycle in phase space is formed. It can be 181 observed in the first figure of the second row, that the two trajectories arising from  $y_1(0)$  and  $y_2(0)$  merge 182 at the limit cycle and then oscillate with exactly the same time period (see the plot of  $x_{1l}$  versus time). 183 The volume flow velocity is also periodic, as observed in the third figure of the second row. It remains zero 184 during the vocal fold impacts, then increases to a maximum and then decreases again until the next impact 185 occurs. 186

The conditions in the third row are the undesirable ones. As long as  $P_s$  grows, the system bifurcates 187 again and the periodicity of the vocal fold self-oscillations is lost. For  $P_s = 0.041$  the motion has already 188 become chaotic, exhibiting a strong sensitivity to the initial conditions. This is apparent in the phase space 189 plot, where strange attractors are observed and the trajectories starting from  $y_1(0)$  and  $y_2(0)$  quickly depart 190 from each other. If we look at the plot of  $x_{1l}$  versus time, we also perceive that, after a few oscillations, the 191 evolution of  $x_{1l}$  when starting at  $y_1(0)$  clearly differs from that starting at  $y_2(0)$ . The motion is no longer 192 periodic at all and exhibits sudden amplitude transitions. The same occurs for the volume flux velocity: 193 neither the impacts nor the amplitudes of U are regular. Such circumstance could result in a severe loss of 194 quality of the generated voice. 195

To better recognize the effects of the chaotic mass motion on the glottal volume flow velocity, U, in the first row of Fig. 3 we have taken advantage of Takens's theorem to recover the attractor in the phase space  $(\dot{U}, U)$  by plotting  $(U(t+\tau), U(t))$ . Here,  $\tau$  is a time lag. There are different options for choosing  $\tau$  (e.g., one could take the first local minimum of the mutual information usually applied to determine the correlation dimension in time series analysis [49] c.f., [50]). For our purposes, a simpler option suffices. We have taken

Physical parameters	Geometry parameters
$m_{1l} = 2.5 \text{ g}$	$d_{1l} = 0.25 \text{ cm}$
$m_{2l} = 0.125 \text{ g}$	$d_{2l} = 0.05 \text{ cm}$
$r_{1l} = 0.02 \text{ g ms}^{-1}$	L = 1.4  cm
$r_{2l} = 0.01 \text{ g ms}^{-1}$	$a_{01} = 0.05 \text{ cm}^2$
$k_{1l} = 0.08 \text{ g}\mathrm{ms}^{-2}$	$a_{02} = 0.05 \ \mathrm{cm}^2$
$k_{2l} = 0.008 \text{ g ms}^{-2}$	
$k_{cl} = 0.09 \text{ g ms}^{-2}$	
$c_{1l} = 3k_{1l} \text{ g ms}^{-2}$	
$c_{2l} = 3k_{2l} \text{ g ms}^{-2}$	
$ ho = 0.00113 \ { m g}  { m cm}^{-3}$	

Table 1: Physical and geometrical parameters of the vocal folds (see [5, 29]).



Figure 2: Vocal fold dynamics for different values of the subglottal pressure  $P_s$ . Blue line: set of initial conditions  $y_1(0)$ . Red dashed line: set of initial conditions  $y_2(0)$ . Left column: phase space plot of  $v_{1l}$  versus  $x_{1l}$  for mass  $m_{1l}$ . Mid column: time evolution of the position  $x_{1l}$  of  $m_{1l}$ . Right column: time evolution of the glottal volume flow velocity.

 $\tau = 4T_{P_s=0.021} = 27.56$  with  $T_{P_s=0.021}$  being the period of U when the pressure is  $P_s = 0.021$  (see Fig. 2, second row, third column), which results in the blue line of the left-top subfigure in Fig. 3. Note that if we chose a different time delay, like  $\tau = 20$ , we get an homeomorphic result to the previous one (green line in the figure). However, when we increase the subglottal pressure to  $P_s = 0.041$ , a strange attractor develops in the phase space of U, as illustrated in the right-top plot of Fig. 3. While in the results of Fig. 2 the initial transients were included to show the dependence on initial conditions, Fig. 3 does not contemplate them. For completeness, in the bottom row of Fig. 3 we have plotted the value of U depending on the positions  $x_{1l}$ 



Figure 3: First row: phase space plot for the glottal volume velocity  $U(t + \tau)$  versus U(t) for  $P_s = 0.021$  and  $P_s = 0.041$ . Second row: volume velocity depending on the position of the two masses  $x_{1l}$  and  $x_2l$ , i.e.,  $U(x_{1l}, x_{2l})$ . The red colour indicates zero values for U which occurs when the left and right masses do contact. Different scales have been used to better appreciate the difference between plots.

and  $x_{2l}$  of the two masses, namely  $U(x_{1l}, x_{2l})$ . The red color indicates the values of  $x_{1l}$  and  $x_{2l}$  for which 208 U equals zero, i.e. either the left first or second masses contact the right ones and no volume flow can pass 209 through the glottis. The blue colour corresponds to positive values of U. The comparison between the two 210 211 figures in the bottom row supports the information found from the attractors in the top row. Setting the subglottal pressure to  $P_s = 0.041$  entails an abnormal, chaotic, behaviour of the motion of the masses and 212 hence of U. Please note that different scales have been used for the left and right columns of Fig. 3 for 213 better inspection, as increasing  $P_s$  from 0.021 to 0.041 implies a significant growth of the amplitude of the 214 volume velocity. 215

Finally, and to complete this brief overview on the behaviour of the symmetric two-mass model, in Fig. 4 216 we have plotted bifurcation plots of the maximum amplitude displacements of  $m_{1l}$ , namely max  $|x_{1l}|$ , for two 217 different situations. The left subfigure corresponds to fixing the subglottal pressure at  $P_s = 0.041$  and then 218 increasing the coupling stiffness constant from  $k_{cl} = 0.025$  to  $k_{cl} = 0.09$ . As illustrated in the figure, for low 219 values of  $k_{cl}$  the motion remains periodic but for  $k_{cl}$  close to 0.057 it becomes chaotic. Then, a window of 220 aperiodic motion can be observed until the motion becomes chaotic again. In contrast, in the right subfigure 221 we have kept  $k_{cl} = 0.09$  fixed and changed the subglottal pressure from  $P_s = 0.0001$  to  $P_s = 0.08$ . For 222 small values of  $P_s$ , self-oscillations cannot be triggered and, as mentioned before, the masses stop after some 223 initial transients. However, when the pressure overcomes a threshold value close to  $P_s \approx 0.0056$ , a Hopf 224 bifurcation occurs and a limit cycle is established. The oscillations increase their amplitude as  $P_s$  augments, 225



Figure 4: Left subfigure: bifurcation plot for fixed  $P_s = 0.041$  and varying stiffness coupling between masses from  $k_{cl} = 0.025$  to  $k_{cl} = 0.09$ . Right subfigure: bifurcation plot for fixed  $k_{cl} = 0.09$  and varying subglottal pressure from  $P_s = 0.015$  to  $P_s = 0.08$ .

<sup>226</sup> but once reached another limit value of  $P_s \approx 0.0377$  they become chaotic. The right subfigure of Fig. 4 <sup>227</sup> clearly exposes the process by which the three regimes previously shown in Fig. 2 develop.

Our goal in this paper is to determine whether it would be possible to establish some chaos controlling mechanism so that the results in the third row of Fig. 2 could look like those in the second row, and the phase space top-right subfigure in Fig. 3 for the volume flow velocity could be like the top-left one, even when the subglottal pressure is high. As said in the introduction of the paper, rather than trying to control the system by means of some external force, we aim to do so by attaching some kind of smart material to the vocal folds and modify their effective mechanical properties.

#### 234 4. Controlling the chaotic motion of the vocal folds

#### <sup>235</sup> 4.1. Chaos control by altering the system energy through external forces

Let us next summarize the basics of the chaos control approach proposed in [41–43] and adapt it to the problem at hand. The key idea in those works is to either increase or lower the overall energy of the system to a value in which the system gets stabilized. For instance, if a change in a system parameter leads to a period-doubling bifurcation, its energy would typically grow as sub-harmonics would contribute to the fundamental frequency [41]. The sum of kinetic and potential energy of the vocal folds is given by

$$E = \frac{1}{2} \boldsymbol{v}^{\top} \mathbf{M} \boldsymbol{v} + \frac{1}{2} \boldsymbol{x}^{\top} \mathbf{K} \boldsymbol{x}, \tag{6}$$

which once averaged over time T becomes

$$\langle E \rangle = \frac{1}{T} \int_0^T \left( \frac{1}{2} \boldsymbol{v}^\top \mathbf{M} \boldsymbol{v} + \frac{1}{2} \boldsymbol{x}^\top \mathbf{K} \boldsymbol{x} \right) dt.$$
(7)

If the system dynamics is periodic, T would correspond to its period, while if it is chaotic  $T \to \infty$ . The rate of energy change of the system is obtained from the time derivative of Eq. (6), taking into account that all involved block matrices in the first row of Eq. (4) are symmetric,

$$\dot{E} = \boldsymbol{v}^{\top} \left( \mathbf{M} \dot{\boldsymbol{v}} + \mathbf{K} \boldsymbol{x} \right) = \boldsymbol{v}^{\top} \left[ -\mathbf{C} \boldsymbol{v} - \boldsymbol{\Theta}(\boldsymbol{x}) \boldsymbol{x} + \boldsymbol{f}_{\boldsymbol{v}}(\boldsymbol{x}) \right]$$
(8)



Figure 5: Attaching a smart material to mass  $m_{1l}$  (left column) or to mass  $m_{2l}$  (right column). The damping  $r_{ilcon}(\mathbf{p}, t)$  and stiffness  $k_{ilcon}(\mathbf{p}, t)$  can depend on variables such as  $\mathbf{p} = \{\mathbf{x}, \mathbf{v}, U\}$  and time, and be tuned so as to suppress the chaotic motion of the vocal folds.

The control strategy in [41] essentially consists in adding a control force to Eq. (8) typically depending on the velocity and in some cases on position, namely  $f_c(v, x)$ , so that this equation becomes

$$\dot{E} = \boldsymbol{v}^{\top} \left( \mathbf{M} \dot{\boldsymbol{v}} + \mathbf{K} \boldsymbol{x} \right) = \boldsymbol{v}^{\top} \left[ -\mathbf{C} \boldsymbol{v} - \boldsymbol{\Theta}(\boldsymbol{x}) \boldsymbol{x} + \boldsymbol{f}_{\boldsymbol{v}}(\boldsymbol{x}) + \boldsymbol{f}_{c}(\boldsymbol{v}, \boldsymbol{x}) \right],$$
(9)

and the control force is such that the power  $v^{\top} f_c(v, x)$  is either strictly positive or negative. This means 236 that the control force will help increasing or decreasing the average system energy in Eq. (7), depending 237 on the remaining non-linear components response to that change. The underlying idea is that the energy 238 shift helps transitioning the system chaotic dynamics to periodic or quasiperiodic ones. Taking  $f_c(v)$ , i.e., 239 solely depending on the velocity v (typically a linear dependence suffices) has proved useful in stabilizing one 240 dimensional non-linear systems like the Van der Pol and Duffing oscillators, as well as controlling systems 241 with impact forces [41-43]. As said in the introduction of this manuscript, the strategy has also revealed 242 efficient for the case of networks of oscillators with some non-linear terms [41, 42]. Hence, one would expect 243 the method to be effective for our vocal fold dynamics that involves two masses driven by flow pressure, 244 which impact during their motion, closing the glottis. The option of taking  $f_c(x)$ , i.e., a control force solely 245 depending on the displacement x will be also considered in forthcoming sections, in addition to  $f_c(v)$ . But, 246 as explained before, instead of thinking about applying external control forces we will rather integrate them 247 in the system and think in terms of attaching a smart material to the vocal folds whose intrinsic parameters 248 can be modified. 249

#### <sup>250</sup> 4.2. Controlling the vocal folds with smart materials

In the case of the human vocal folds, adding an external control force  $f_c(v, x)$  to the force vector  $f_v$  is not 251 feasible. At most one could think of varying the vocal folds tension by stimulating the nerves driving them. 252 As exposed in the introduction of the paper, this is at the basis of the few existent vocal fold pacemakers 253 oriented at abduction and respiratory problems, but this approach would not work if the involved nerves 254 were seriously damaged. The alternative we start exploring in this paper is whether it would be possible 255 to control the chaotic motion of the vocal folds by somehow attaching a smart material either to  $m_{1l}$  or to 256  $m_{2l}$ , such that its stiffness and/or damping could be modified at convenience. The situation is schematically 257 depicted in Fig. 5, where in the left column the smart material is added to  $m_{1l}$ , while in the right column 258 is glued to  $m_{2l}$ . The material has mass  $m_{\rm sm}$  and will slightly elongate the glottis in a small quantity  $\Delta L$ . 259 Although these aspects could be easily incorporated in the model of Eq. (4), we will assume them negligible 260



Figure 6: Control by modifying the damping  $r_{1l}$  of mass  $m_{1l}$ . First column: system with no control. Second column: system with control. Damping activated at t = 300. First row: phase space attractor  $(v_{1l}, x_{1l})$ . Second row: time evolution of the displacement  $x_{1l}$ . Third column: time evolution of the glottal volume flow velocity. Results for a subglottal pressure of  $P_s = 0.041$ .

for simplicity in forthcoming simulations (i.e. we consider  $m_{\rm sm} \ll m_{2l}$ ,  $m_{1l}$  and  $\Delta L \ll L$ ). The focus will be placed on finding appropriate laws for the varying smart material damping,  $c_{ilcon}(\mathbf{p}, t)$  and stiffness  $k_{ilcon}(\mathbf{p}, t)$ , i = 1, 2, which may depend on variables  $\mathbf{p} = \{\mathbf{x}, \mathbf{v}, U\}$  and time. As will be seen, most of the forthcoming work will concentrate on tuning the smart material damping to get an overall effective damping that suppresses the vocal folds chaotic motion. A final section on stiffness control will also be provided, though, as well as a brief discussion on the potential of current smart materials to attain our goal. Our exploration will be limited to the simple two-mass model in Eq. (4).



Figure 7: First row: attractors and period-1 orbit for the system with (red) and without (blue) damping control  $r_{1lcon}$ . Left: phase space  $(v_{1l}, x_{1l})$ , right: phase space  $(v_{2l}, x_{2l})$ . Second row: phase space plot for the glottal volume velocity  $U(t+\tau)$  versus U(t) without control (left) and with control (right). Results for a subglottal pressure of  $P_s = 0.041$ .

Instead of adding  $f_c(v, x)$  to the force vector in Eq. (4), our smart material attempt is in fact equivalent to moving this term to the l.h.s of the equation so that the practical consequence is that of modifying the stiffness and/or damping matrices in  $\mathbf{A}(y)$ , by adding some control to them. In other words, we look for the effects of changing

$$\mathbf{C} \longrightarrow \mathbf{C} + \mathbf{C}_c(\boldsymbol{v}, \boldsymbol{x}, \boldsymbol{U}, t), \tag{10}$$

and/or

$$\mathbf{K} \longrightarrow \mathbf{K} + \mathbf{K}_c(\boldsymbol{v}, \boldsymbol{x}, \boldsymbol{U}, t), \tag{11}$$

where  $\mathbf{C}_c$  and  $\mathbf{K}_c$  refer to the damping and stiffness control matrices provided by the smart material. It 268 is important to note here, that one does not need to alter all damping or stiffness entries in  $\mathbf{C}$  and  $\mathbf{K}$ . 269 The good news are that usually  $C_c$  and  $K_c$  will only have a single element different from zero. That is, 270 modifying a single parameter suffices to stabilize the whole system. Before proceeding any further, it should 271 be noticed that there is a trivial modification that could render the system stable. We know from the analysis 272 in section 3 that setting the coupling stiffness to  $k_{cl} = 0.09$  instead of the standard value of  $k_{cl} = 0.025$ 273 (see [5, 29]) triggers chaotic motion beyond a certain threshold subglottal pressure value. Recovering the 274 value of  $k_{cl} = 0.025$  would result in periodic motion of the vocal folds. Nevertheless, the challenge here is 275 to see if that motion can be restored through other parameter modifications rather than  $k_{cl}$ . 276

#### 277 4.2.1. Damping control

We start with one of the simplest possible options, which consists in altering the damping of the largest mass in the system, namely  $m_{1l}$ . This corresponds to taking a force  $f_c(v) = C_c v$  that is integrated in the modified damping matrix of the system as follows,

$$\mathbf{C} = \operatorname{diag}(r_{1l}, r_{2l}) \longrightarrow \mathbf{C} + \mathbf{C}_c = \operatorname{diag}(r_{1l} + r_{1l\operatorname{con}}, r_{2l}), \tag{12}$$



Figure 8: First row: attractors and period-1 orbits for the system with (red) and without (blue) damping control  $r_{2lcon}$ . Left: phase space  $(v_{1l}, x_{1l})$ , right: phase space  $(v_{2l}, x_{2l})$ . Second row: phase space plot for the glottal volume velocity  $U(t+\tau)$  versus U(t) without control (left) and with control (right). Results for a subglottal pressure of  $P_s = 0.041$ .

with  $|r_{1lcon}| < |r_{1l}|$ . Let us consider the case of a subglottal pressure of  $P_s = 0.041$  like in the last row of 278 Fig. 2 and take a small control value  $r_{1lcon} = -0.1r_{1l}$ . Initially, we let the system evolve without control and 279 when t = 300 we activate the new smaller damping. This results in a slight increase of the system average 280 energy in Eq. (7), from  $\langle E \rangle = 6.12 \times 10^{-4}$  to  $\langle E_{\rm con} \rangle = 7.68 \times 10^{-4}$ . The effects of diminishing the damping 281 are plotted in Fig. 6. The first column in the figure corresponds to the results of the simulations without 282 control, while the second column contains those with control. In the first row we observe how the strange 283 attractor transforms to a cycle for the new damping value. The data in these figures have been plotted 28 from t = 500 to t = 1000. In the second row it is seen how the displacement  $x_{1l}$  becomes periodic beyond 285 t= 300, once the control is triggered. Also, the volume flow velocity in the third row becomes regular 286 (compare it with the evolution of U in the last row of the first column), growing from zero, when the glottis 287 is closed, to its maximum value and then decreasing again until the next contact between masses takes 288 place. The time duration of the glottis closure remains constant when the control is activated. Very similar 289 results are obtained for the mass  $m_{2l}$  that will not be reproduced herein for brevity. To better appreciate 290 the stabilization effects of decrementing the damping of  $m_{1l}$ , in the first row of Fig. 7 we have plotted the 291 attractors in phase spaces  $(v_{1l}, x_{1l})$  and  $(v_{2l}, x_{2l})$  when no damping modification is applied (blue lines) and 292 the closed orbits (red lines) achieved when  $r_{1lcon}$  is triggered. In the second row of the figure we make use 293 again of Takens's theorem and plot  $U(t+\tau)$  against U(t) to show how the strange attractor in the left figure 294 gets stabilized through damping in the right figure. The glottal flow volume velocity, which is our final most 295 important variable for voice generation has therefore acquired the right shape. 296

Next, we may wonder what happens if instead of acting on the damping,  $r_{1l}$ , of the first mass, we act on  $r_{2l}$  of the second mass,  $m_{2l}$ . That is, we now make the modification,

$$\mathbf{C} = \operatorname{diag}(r_{1l}, r_{2l}), \longrightarrow \mathbf{C} + \mathbf{C}_c = \operatorname{diag}(r_{1l}, r_{2l} + r_{2l\operatorname{con}}), \tag{13}$$

with  $|r_{2lcon}| < |r_{2l}|$ . The situation is now by far more tricky than when changing  $r_{1l}$ . The mass  $m_{2l}$  is



Figure 9: Dynamic damping modification  $r_{2lcon} \operatorname{sgn}(v_{2l})$  (blue line) depending on the velocity  $v_{2l}$  (black line) of the small mass  $m_{2l}$ .

twenty times smaller than  $m_{1l}$  and acting on it to control the whole dynamics of the system through  $r_{2l}$ is problematic. To begin with, testing  $r_{2lcon} = -0.1r_{2l}$  is not enough to stabilize the system and a higher percentage of variation of  $r_{2l}$  is required. Periodicity is attained when  $r_{2lcon} \sim -0.25r_{2l}$ , but there is a problem associated to it. The time duration of the vocal folds contact is negligible and for slightly higher





Figure 10: First row: attractors and period-1 orbits for the system with (red) and without (blue) damping control  $r_{2l} + r_{2lcon} \operatorname{sgn}(v_{2l})$ . Left: phase space  $(v_{1l}, x_{1l})$ , right: phase space  $(v_{2l}, x_{2l})$ . Second row: phase space plot for the glottal volume velocity  $U(t + \tau)$  versus U(t) without control (left) and with control (right). Results for a subglottal pressure of  $P_s = 0.041$ .

U will be pulsating, it would never be zero, giving place to a breathy voice which is undesirable. The 303 problem is clearly illustrated in Fig. 8 that shows equivalent subplots to the previous ones in Fig. 7. As 304 observed in the first row, the motion of the two masses gets stabilized and two period-1 cycles clearly form. 305 However, when we look at the volume velocity in the second row (bottom-right figure), it becomes apparent 306 that its corresponding closed orbit does not cross the axes so the breathy effect will clearly manifest. In 307 this case, the system average energy has decreased from  $\langle E \rangle = 6.12 \times 10^{-4}$  to  $\langle E_{\rm con} \rangle = 5.9 \times 10^{-4}$ . One 308 may conclude that acting on the first vocal fold mass is clearly an easier way to get an acceptable glottal 309 flow if damping changes are to be implemented (one could also consider changing both damping values 310 simultaneously, though it would clearly be better to find a strategy involving as less control parameters as 311 possible). 312

A more robust option than simply shifting  $r_{1l}$  or  $r_{2l}$  to new values is that of proposing a damping variation depending on the sign of the mass velocities. Following [41–43], we take

$$\mathbf{C} = \operatorname{diag}(r_{1l}, r_{2l}) \longrightarrow \mathbf{C} + \mathbf{C}_c = \operatorname{diag}(r_{1l} + r_{1l\operatorname{con}}\operatorname{sgn}(v_{1l}), r_{2l}), \tag{14}$$

or, alternatively,

$$\mathbf{C} = \operatorname{diag}(r_{1l}, r_{2l}) \longrightarrow \mathbf{C} + \mathbf{C}_c = \operatorname{diag}(r_{1l}, r_{2l} + r_{2l\operatorname{con}}\operatorname{sgn}(v_{2l})), \tag{15}$$

with  $|r_{ilcon} \operatorname{sgn}(v_{il})| < |r_{il}|, i = 1, 2$ . This obviously implies that one should be able to change the damping 313 of the smart material attached to the vocal folds very rapidly, at least twice per oscillation. This is a real 314 challenge for current smart materials (see the discussion in section 4.2.4). Here, the main difference with the 315 previous proposals is that the amount of increased damping depends on the sign of the mass velocity. Let us 316 focus on the second mass, for which severe problems were found when just changing the value of  $r_{2l}$  to a new 317 one. We implement the time varying damping depicted in Fig. 9, according to the general strategy in [42]. 318 As observed in the figure, when the velocity  $v_{2l}$  augments  $r_{2lcon}$  helps increasing the original damping  $r_{2l}$ , 319 while it diminishes it when the opposite occurs. The effects of such variations are remarkable. Analogous 320 results to those of Fig. 8 are plotted in Fig. 10, using the new dynamic damping law of Eq. (15). The 321 improvement is striking. The contact between vocal folds is similar to that obtained when we acted on 322 the first mass and U becomes totally regular again. With this approach, the average energy of the system 323 slightly increases from  $\langle E \rangle = 6.12 \times 10^{-4}$  to  $\langle E_{\rm con} \rangle = 6.79 \times 10^{-4}$ . 324

#### 325 4.2.2. Damping control with volume velocity feedback for varying subglottal pressure

<sup>326</sup> Up to now, the suggested damping strategies for controlling the chaotic motion of the vocal folds have <sup>327</sup> been presented for a fixed value of the subglottal pressure, namely  $P_s = 0.041$ . However, in practice  $P_s$  will <sup>328</sup> rarely be constant and can exhibit significant differences during phonation. Therefore, it would be highly <sup>329</sup> recommendable to have some type of feedback mechanism that could adapt the damping of the system to <sup>330</sup> subglottal pressure variations. That is the topic of this subsection.

To address the problem, we consider the subglottal pressure evolution plotted in the first row of Fig. 11. As seen,  $P_s$  is taken constant and equal to 0.041 for the first 300 ms of the simulation, and then starts growing following a parabolic profile up to a maximum of  $P_{s,max} = 0.08$ . Once passed the maximum, the subglottal pressure diminishes and recovers its original value of  $P_s = 0.041$  at t = 1300. In the second row of Fig. 11 we add 10% of normally distributed noise to  $P_s$  to check the robustness of the feedback strategy to be presented below.

The basic idea for controlling the vocal fold oscillations for varying  $P_s$  consists in introducing a feedback mechanism that increases or decreases the system damping depending on the effects that the evolution of  $P_s$  has on the glottal volume velocity U. We consider the following strategy for the damping of the second mass,

$$\mathbf{C} = \operatorname{diag}(r_{1l}, r_{2l}) \longrightarrow \mathbf{C} + \mathbf{C}_c = \operatorname{diag}(r_{1l}, r_{2l} + \left[r_{2l\operatorname{con}} + \Delta r_{2l\operatorname{con}}(U(t', t'' \dots t^N))\right] \operatorname{sgn}(v_{2l})).$$
(16)

As compared to Eq. (15), at time t of the system evolution the damping value is not only modified according to the sign of the velocity  $v_{2l}$  of the second mass  $m_{2l}$ , but also by a new term,  $\Delta r_{2lcon}(U(t', t'' \dots t^N))$ ,



Figure 11: Subglottal pressure evolution. Top row:  $P_s$  remains constant for the first 300 ms, then increases following a parabolic profile up to a maximum of  $P_{s,max} = 0.08$  and recovers its original value for t = 1300 ms. Bottom row:  $P_s$  exhibits the same time evolution of the top row but with a 10% of normally distributed noise.



Figure 12: Comparison between the dynamic damping control without U feedback,  $r_{2lcon} \operatorname{sgn}(v_{2l})$  (blue line), and with U feedback  $[r_{2lcon} + \Delta r_{2lcon}(U(t', t'' \dots t^N))] \operatorname{sgn}(v_{2l})$  (red line), when the subglottal pressure  $P_s$  starts growing according to the parabolic profile in Fig. 11 (top row), and when it gets back to its original value (bottom row).



Figure 13: Comparison of damping control strategies when  $P_s$  evolves according to the top row of Fig. 11. First column: no control. Second column: damping control in Eq. (15). Third column: damping control with U feedback in Eq. (16). The NF control is activated for  $t \ge 100$  ms and the FB one for  $t \ge 300$  ms. First row: phase space trajectories corresponding to the coloured time interval stripes in the second row. Second row: evolution of the first mass displacement  $x_{1l}$ . Third row: evolution of the glottal volume velocity U.

which depends on previous time values  $t', t'' \dots t^N$  of the glottal volume velocity U.  $\Delta r_{2lcon}(U(t', t'' \dots t^N))$ is computed as follows. It is first assigned a zero value. As the system evolves, every 20 ms we find the peaks of U in the preceding 30 ms and compute a linear regression for them. Usually four to five peaks are found,  $U_{pk,i}$ ,  $i = 1 \dots 4, 5$ . If the resulting slope is positive and the absolute value of the difference between the maximum and minimum peaks of U exceeds a threshold value  $U_{tol} = 0.1$ , i.e.  $|U_{pk,max} - U_{pk,min}| > U_{tol}$ we take  $\Delta r_{2lcon}(U(t', t'' \dots t^N)) = 3 \times 10^{-4}$  and the damping control increases. If  $|U_{pk,max} - U_{pk,min}| > U_{tol}$ 



Figure 14: First row: phase space trajectories for the glottal volume velocity  $U(t+\tau)$  versus U(t) in the time interval [700, 800] ms (red stripe in Fig. 13) for the cases of no control, damping control for constant  $P_s$  in Eq. (15) and damping control with U feedback for varying  $P_s$  in Eq. (16). Second row: plots of  $U(x_{1l}, x_{2l})$  for the same time interval. The red colour indicates zero values for U which occurs whenever the left and right masses contact.

but the slope is negative then  $\Delta r_{2lcon}(U(t', t'' \dots t^N)) = -3 \times 10^{-4}$  and the control damping diminishes. If 345  $|U_{pk,\max} - U_{pk,\min}| < U_{tol}$  no modification is made. The difference between the control damping computed 346 according to Eq. (15) and to Eq. (16) is shown in Fig. 12. The first row in the figure presents the evolution 347 of  $r_{2lcon} \operatorname{sgn}(v_{2l})$  (blue line) and of  $[r_{2lcon} + \Delta r_{2lcon}(U(t', t'' \dots t^N))] \operatorname{sgn}(v_{2l})$  (red line) between t = 250 and 348 t = 500, when the subglottal pressure starts augmenting (see Fig. 11). As observed, the damping with U 349 feedback control increases with  $P_s$ , while that without control remains unaltered. In the second row of the 350 figure, we depict the control damping evolution between t = 1050 and t = 1300, when  $P_s$  diminishes to its 351 original value  $P_s = 0.041$ . As seen in the figure, the damping with U feedback control (red line) progressively 352 diminishes until it recovers that computed without U feedback (blue line). In the case of a more abrupt 353 and complex time evolution of  $P_s$  than that in Fig. 11, one could think of designing alternative feedback 354 strategies if the proposed one was not sufficient. 355

The effectiveness of the U feedback damping control strategy on the vocal folds behaviour is shown in 356 Fig. 13. The first column in the figure corresponds to the case in which no control is applied to the system. 357 The second column contains the results for the constant damping control strategy without U feedback in 358 Eq. (15), and the third column shows the results when the U feedback control of Eq. (16) is considered. The 359 constant control strategy is activated for  $t \ge 100$  ms and the U feedback one for  $t \ge 300$  ms. As regards 360 the row by row results in the figure, let us focus on the second row contents before commenting on the first 361 one. The second row of the figure presents the evolution of  $x_{1l}$  with time for a total of 1300 ms, during 362 which  $P_s$  varies according to the top row of Fig. 11. As we have decided to plot the entire time interval, 363 individual oscillations are hardly visible. However, one can perceive from the first subfigure in the second 364 row that the lack of control leads to strong chaotic motion, while this is slightly smoothed when applying 365 the damping control without U feedback, and definitely turned into periodic motion when implementing 366 the U feedback control. To better distinguish such behaviour, we have depicted three coloured stripes in 367 the figure, green, blue and red, which respectively cover the time intervals [200, 300] ms, [500, 600] ms and 368 [700, 800] ms. The phase space plots  $(v_{1l}, x_{1l})$  corresponding to these time intervals are presented in the first 369 row of Fig. 13, using the same colour code. In the case of no control, the trajectories evolve into strange 370 attractors of increasing amplitude as  $P_s$  augments, as one could have expected. When applying control 371 without U feedback, it is seen that the motion becomes periodic for [200, 300] ms because for this interval 372



Figure 15: Phase space trajectories for the glottal volume velocity  $U(t+\tau)$  versus U(t) in the time interval [700, 800] ms (red stripe in Fig. 13) for the cases of no control, damping control in Eq. (15) and damping control with U feedback in Eq. (16) for varying  $P_s$  in the presence of noise (bottom row of Fig. 11).

the pressure still has constant value  $P_s = 0.041$ . Therefore a green cycle appears in the second subfigure 373 of the first row in Fig. 13. However, as  $P_s$  gets higher, the control is insufficient and the trajectories in 374 phase space (blue and red) soon become complex, departing from the desired periodic motion. As opposed, 375 when the U feedback is applied, all trajectories in phase space remain periodic. The same green cycle of 376 the previous subfigure is recovered and when the subglottal pressure is increased the motion does not loose 377 periodicity but simply increases its amplitude (see the blue and red lines). Finally, in the third row of Fig. 13 378 we have plotted the time evolution of the glottal volume velocity U. The same tendencies found for  $x_{1l}$  are 379 encountered for U, which seems to exhibit a totally regular behaviour as  $P_s$  grows if the U feedback control 380 is activated. Given that U is the variable of primary importance in this paper, we analyse it in some more 381 detail in Fig. 14. 382

The first row of Fig. 14, presents the trajectories in the phase space (U, U) for the time interval 383 [700, 800] ms (red stripe in Fig. 13) making use of Takens's theorem once more. As observed, and as 384 already shown in previous figures, the trajectories in the case of no control are totally chaotic, which results 385 in a very irregular pattern of U that would lead to critical problems for voice generation. When the control 386 of Eq. (15) is applied some improvement is observed. However, the closure of the vocal folds still has very 387 different durations (look at the zero values of the trajectories in the vertical and horizontal axes of the 388 second subfigure in the first row). This would result again in poor voice quality. We can see, though, that 389 the U feedback control is able to deal with the situation producing a very regular pattern for U that only 390 presents very slight variations of the contact duration and amplitude as  $P_s$  increases. Such behaviour can 391 also be appreciated plotting U against the position of the masses (second row of Fig. 14),  $x_{1l}$  and  $x_{2l}$ , as 392 we did in the second row of Fig. 3. The problems detected in the phase space trajectories clearly reflect in 393 the plots of  $U(x_{1l}, x_{2l})$ . It is remarkable how the U feedback control is able to stabilize the system so that 394  $U(x_{1l}, x_{2l})$  for  $P_s$  close to 0.08 essentially exhibits the same behaviour than for  $P_s = 0.021$  in Fig. 3, but 395 with a larger amplitude. 396

Finally, in Fig. 15, we check whether the presented controlling mechanisms are robust with respect to 397 noise perturbations. To that end we have applied the damping control strategy in Eq. (15) and that with U 398 feedback in Eq. (16) to the subglottal pressure evolution in the bottom row of Fig. 11. As said, that figure 399 corresponds to  $P_s$  in the top row but with an addition of 10% of normally distributed noise. Therefore, we 400 want to check now if the proposed control strategies would be efficient in the demanding situation where 401 noise is added to a varying subglottal pressure. As observed in Fig. 15, where we have plotted the same 402 phase space plots (U, U) than in the first row of Fig. 14, both control options turn to be quite robust to the 403 presence of noise and do not substantially alter their behaviour. Note that in the case of the U feedback 404 control (third column in Fig. 15) the results are quite neat and topologically resemble those in the first 405 row, third column of Fig. 14. That could lead to admissible self-sustained oscillations of the vocal folds and 406 proper glottal volume velocity generation as compared to control without feedback or no control at all. The 407 robustness under noise perturbations of damping control strategies based on altering the system energy was 408 409

#### 410 4.2.3. Stiffness control

Up to now, the chaos control strategies presented in this work have relied on modifying either the damping  $r_{1l}$  of the first mass  $m_{1l}$  or  $r_{2l}$  of  $m_{2l}$ . In this section, it will be shown that it is also possible to switch from chaotic oscillations to periodic ones by modifying the stiffness matrix of the system Eq. (4), instead of the damping one. We may consider the options of taking,

$$\mathbf{K} = \begin{pmatrix} k_{1l} + k_{cl} & -k_{cl} \\ -k_{cl} & k_{2l} + k_{cl} \end{pmatrix} \longrightarrow \mathbf{K} + \mathbf{K}_c = \begin{pmatrix} k_{1l} + k_{1lcon} + k_{cl} & -k_{cl} \\ -k_{cl} & k_{2l} + k_{cl} \end{pmatrix}$$
(17)

or alternatively,

$$\mathbf{K} = \begin{pmatrix} k_{1l} + k_{cl} & -k_{cl} \\ -k_{cl} & k_{2l} + k_{cl} \end{pmatrix} \longrightarrow \mathbf{K} + \mathbf{K}_c = \begin{pmatrix} k_{1l} + k_{cl} & -k_{cl} \\ -k_{cl} & k_{2l} + k_{2l\mathrm{con}} + k_{cl} \end{pmatrix}.$$
 (18)

For the sake of brevity, and not to repeat the many tests carried out in the previous sections on damping control, we will restrain to the first option in Eq. (17) and deal with the case of constant subglottal pressure  $P_s = 0.041$ . In particular, we will modify the stiffness associated to the first mass in Eq. (17) according to the following law,

$$k_{1l} \longrightarrow k_{1l} + \Delta k_{1lcon}(U(t)).$$
 (19)

Here, the dependence of  $\Delta k_{1lcon}(U(t))$  on the volume velocity U is such that if  $0 < U(t) \leq U_{tol}$  we set 411  $\Delta k_{1lcon}(U(t)) = 0.75k_{1l}$ , otherwise we set  $\Delta k_{1lcon}(U(t))$  to zero. A few numerical experiments have revealed 412 that taking  $U_{\rm tol} = 1$  provides stabilization. The functioning of the stiffness control strategy is shown in 413 Fig. 16, where the evolution of U and  $\Delta k_{1lcon}(U(t))$  are plotted for the interval [250, 400] ms. Note that the 414 control is activated at t = 300 and a few milliseconds after that, namely beyond t = 350, the volume velocity 415 has already become totally regular. The efficiency of the stiffness control is more clearly shown in Fig. 17, 416 where, as in previous figures, we compare the results for the system with (second column) and without 417 control (first column). As clearly seen from the time evolution of  $x_{1l}$  and U (second and third rows) and the 418 trajectories in the phase space  $(v_{1l}, x_{1l})$  (first row), the stiffness control is totally capable to stabilize the 419 chaotic motion. Moreover, and as in previous tests with control damping laws, in Fig. 18 we have plotted 420 the phase space trajectories in  $(v_{1l}, x_{1l})$  and  $(v_{2l}, x_{2l})$  when there is no control (blue lines) and when the 421 control is switched on (red lines). Both masses oscillate periodically. The results in the phase space  $(\dot{U}, U)$ 422 also reveal the regularity of the volume velocity which is necessary to produce a normal, healthy voice. 423

Finally, it is to be noted that the proposed stiffness control requires rather high values of  $\Delta k_{1lcon}$  (up to 0.75 $k_{1l}$ ), in comparison to the relative increase of the damping value in previous sections. As regards the system average energy, in this case it has slightly increased from  $\langle E \rangle = 6.12 \times 10^{-4}$  to  $\langle E_{con} \rangle = 6.44 \times 10^{-4}$ .



Figure 16: Dynamic stiffness modification  $\Delta k_{1lcon}(U(t))$  (blue line) depending on the volume velocity U(t) (black line).



Figure 17: Control by modifying the stiffness  $k_{1l}$  of mass  $m_{1l}$ . First column: system without control. Second column: system with stiffness control, which is activated at t = 300 ms. First row: phase space attractor  $(v_{1l}, x_{1l})$ . Second row: time evolution of the displacement  $x_{1l}$ . Third column: time evolution of the glottal volume flow velocity. Results for a subglottal pressure of  $P_s = 0.041$ .

#### 427 4.2.4. Brief discussion

In the preceding sections, we have proposed several options to control the chaotic oscillations of a twomass model of the vocal folds and recover a proper glottal pulse for voice production. It is to be mentioned that the control of non-linear oscillators has been studied in detail in the past decades (see e.g. [51, 52]). The first attempts were devoted to single non-linear oscillators like the Duffing and the Van der Pol oscillators (among others), which respectively account for a cubic non-linear stiffness restoring force and a quadratic non-linear damping one. A wide range of strategies have proved useful for that purpose like the OGY



Figure 18: First row: attractors and period-1 orbit for the system with (red) and without (blue) stiffness control  $\Delta k_{1lcon}(U(t))$ . Left: phase space  $(v_{1l}, x_{1l})$ , right: phase space  $(v_{2l}, x_{2l})$ . Second row: phase space plot for the glottal volume velocity  $U(t + \tau)$  versus U(t) without control (left) and with stiffness control (right). Results for a subglottal pressure of  $P_s = 0.041$ .

method [53], feedback strategies [54], non-linear adaptative approaches [55], modified linearization [56], 434 synchronization [57], control without feedback [58], or the altering energy method [41, 42] adopted in this 435 work. As mentioned in the introduction, several of these approaches are also effective when non-linearity 436 manifests because of impacts [37–40, 43, 59]. However, less articles exist on the control of systems of 437 oscillators describing lumped physical models [40, 60, 61], like the one in the current work. In [41, 42], it was 438 checked that the altering energy approach is also very effective for networks of oscillators. This has been 439 confirmed for the two-mass model in the previous sections, when applying the strategy (with variations) to 440 distinct situations. It is expected that similar results would follow for higher dimension models of the vocal 441 folds (e.g., that in [7]). 442

With the exception of the control without feedback in [58], which relies on the classical option of linking 443 a tuned absorption damper to control the oscillator (the lack of feedback may lead to robustness problems), 444 all other strategies in the above paragraph involve an external control force acting on the mechanical system. 445 As this would not be possible for the vocal cords, in the previous sections we have suggested and analyzed the 446 possibility of using ideal smart materials to control the oscillations of the vocal folds, and prevent abnormal 447 voice production. Obviously, there is a long pathway from the theoretical analysis with ideal simple models 448 carried out insofar to any potential real application. In addition to practical issues such as how to obtain 449 the time series where to apply the control strategy (e.g., by means of a thin film accelerometer placed at the 450 vocal cords), a critical aspect concerns whether smart materials exist with the capability of rapidly changing 451 their damping and/or stiffness values at the frequencies needed for chaotic vocal folds control. This is a 452 strong challenge for current smart materials. Besides, while a damping variation in a viscoelastic material 453 can be frequency-dependent as either the storage modulus and the loss modulus change with the frequency 454 of the applied load, a stiffness variation for an elastic material will mostly depend on geometry changes 455 rather than on variations in the elastic modulus, which is an intrinsic property of the material. 456

<sup>457</sup> From the wide range of smart materials described in the literature [44], some look promising and will

be worth exploring in the future. For instance, the actuators or artificial muscles defined in [62] are a class of materials and devices that can reversibly contract, expand, or rotate as a response to an external stimulus. A recent review [63] discusses the mechanisms, limitations and challenges of several actuators including shape memory alloys, ionic-polymer/metal composites and dielectric elastomer actuators, whereas in [64] electroactivated polymers are described as actuators suitable for biomedical applications. A dynamic stimulus of the actuator, might produce a rapid damping or a stiffness variation and recent approaches to understand the mechanisms behind such behaviours have been reported in [65, 66].

#### 465 5. Conclusions

In this paper we have suggested the idea of a voice pacemaker that could control chaotic oscillations of the vocal folds and revert them to periodic ones. That pacemaker could be built by attaching a smart material to the vocal cords, with the capability of having tunable time varying damping and stiffness values. For simplicity, the mass of the smart material has been assumed negligible as compared to those of the vocal folds. Simulations have been performed for a symmetric two-mass model in which oscillations become chaotic for excessive subglottal pressure.

In the two-mass model, each vocal cord is characterized by two masses, the lower one being twenty 472 times bigger than the upper one. To begin with, it has been shown that, for a fixed value of the subglottal 473 pressure, an effective control strategy based on altering the system energy consists in slightly diminish the 474 damping of the biggest mass through the action of the smart material. This stabilizes the motion of the 475 vocal folds, which becomes periodic and closes the glottis at regular time intervals. That can be clearly seen 476 in phase space plots of the motion of the masses, as well as when applying Taken's theorem to visualize the 477 glottal volume velocity in phase space. Remember that a regular glottal volume velocity pulse is the key for 478 proper voice generation. The situation is more intricate if one attempts to control the upper mass of the 479 vocal cords because reducing its damping does not suffice. The vocal fold oscillations can become periodic 480 but they do not collide, i.e., the glottis does not close. This would lead to an undesirable breathy glottal 481 volume velocity and abnormal voice production. A more complex damping law is therefore required to get 482 an admissible glottal pulse if one wants to control the system through the upper mass instead of the lower 483 one. The trick here is to supply additional damping depending on the sign of the velocity of the upper mass. 484 In this way, one can recover a similar control to that achieved with the first mass. 485

On the other hand, in a realistic situation the subglottal pressure will be rarely constant but vary with 486 time. It would be thus interesting to establish a feedback control strategy to deal with that circumstance. 487 This has been done for the very demanding case of controlling the upper mass. For varying subglottal 488 pressure, we do not only change the sign of the damping depending on the upper mass velocity but we also 489 modify its amplitude depending on previous values of the glottal volume velocity. Again, the results have 490 clearly shown the validity of this strategy and how periodicity is recovered yielding a regular glottal pulse 491 for correct voice generation. Another interesting aspect is that in practice, the subglottal pressure will not 492 only evolve with time but present noise perturbations. The feedback control has been also tested in such 493 condition by adding a 10% of normally distributed noise to the subglottal pressure signal. It has been shown 494 that the feedback control is noticeably robust to the presence of noise and that the resulting glottal pulses 495 present admissible variations in amplitude and in the glottis closure time. 496

To finish the paper, we have considered the option of controlling the chaotic motion of the vocal folds 497 by acting on the stiffness of the smart material instead of acting on its damping. For constant subglottal 498 pressure, it has been proposed to modify the stiffness of the lower mass of the vocal folds depending on the 499 value of the glottal volume velocity. If that exceeds a given tolerance value, the stiffness augments by a 500 constant factor, while no change is made for smaller values than the threshold. Once more, time evolution 501 and phase space plots for the mass motion and volume velocity have revealed the efficiency of the method. 502 Therefore, stiffness control could be also an option and in future works it will be worth exploring whether 503 504 it may be also reliable for varying and/or noisy subglottal pressure.

The work presented in this paper intends to open the door to new ways of dealing with pathological voice disorders. It is to be mentioned though, that our analysis has been carried out for a very simple twomass model of the vocal folds. In forthcoming works, we will analyse the smart material control approach for models involving lateral paralyses and polyps which break the symmetry of the current model and also introduce new types of non-linearities. It is expected that the type of chaos control strategy presented herein

<sup>510</sup> could well adapt to those more complex models and situations.

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