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Thermochemical non-equilibrium effects in turbulent hypersonic boundary layers

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A hypersonic, spatially evolving turbulent boundary layer at Mach 12.48 with a cooled wall is analysed by means of direct numerical simulations. At the selected conditions, massive kinetic-to-internal energy conversion triggers thermal and chemical non-equilibrium phenomena. Air is assumed to behave as a five-species reacting mixture, and a two-temperature model is adopted to account for vibrational non-equilibrium. Wall cooling partly counteracts the effects of friction heating, and the temperature rise in the boundary layer excites vibrational energy modes while inducing mild chemical dissociation of oxygen. Vibrational non-equilibrium is mostly driven by molecular nitrogen, characterized by slower relaxation rates than the other molecules in the mixture. The results reveal that thermal non-equilibrium is sustained by turbulent mixing: sweep and ejection events efficiently redistribute the gas, contributing to the generation of a vibrationally under-excited state close to the wall, and an over-excited state in the outer region of the boundary layer. The tight coupling between turbulence and thermal effects is quantified by defining an interaction indicator. A modelling strategy for the vibrational energy turbulent flux is proposed, based on the definition of a vibrational turbulent Prandtl number. The validity of the strong Reynolds analogy under thermal non-equilibrium is also evaluated. Strong compressibility effects promote the translational–vibrational energy exchange, but no preferential correlation was detected between expansions/compressions and vibrational over-/under-excitation, as opposed to what has been observed for unconfined turbulent configurations.

Key words: compressible boundary layers, hypersonic flow, turbulent reacting flows

1. Introduction

High-speed flows represent a challenging topic of interest for manifold configurations, including objects entering a planetary atmosphere or for atmospheric hypersonic flight (Gnoffo *et al.* 1999). In such flows, conversion of massive amounts of kinetic energy into internal energy causes a sudden rise of the flow temperature. The effects triggered by the high temperatures include chemical reactions and vibrational relaxation phenomena on characteristic time scales comparable to the flow time scales. The complex thermochemical state induced by such conditions may affect quantities of interest for the design of high-speed vehicles significantly (Bertin & Cummings 2006; Candler 2019). In the past, thermochemical effects caused by hypersonic conditions have been investigated in more or less simple configurations. Most of the efforts focused on stagnation-point flows, of interest for bluff bodies re-entering the atmosphere (Fay & Riddell 1958; Armenise *et al.* 1996; Bonelli *et al.* 2017; Colonna, Bonelli & Pascazio 2019). In later stages of re-entry trajectories or for suborbital flight conditions characterized by higher free-stream densities, the flow description is further complicated by the transition from a laminar to a turbulent regime. Such a transition may be triggered by free-stream disturbances, erosion and ablation of the thermal protection systems (TPS) or due to surface defects, leading to sharp rise of the skin friction and heat fluxes at the walls. For this reasons, laminar-to-turbulent transition has been identified as a subject of major concern for the accurate prediction of the aerothermodynamic flow fields around objects flying at hypersonic speeds. A significant number of contributions in the literature have addressed the linear and nonlinear stability of flat-plate boundary layers in chemical non-equilibrium, sometimes up to the initial stages of transition. The pioneering studies of Malik & Anderson (1991), Hudson, Chokani & Candler (1997) and Perraud *et al.* (1999) pointed out that chemical reactions tend to promote the so-called second mode instability, similarly to strongly cooled walls. More recently, Marxen *et al.* (2013) and Marxen, Iaccarino & Magin (2014) used direct numerical simulations (DNS) to evaluate the maximum streamwise velocity disturbance caused by large- or small-amplitude waves in presence of chemical reactions. Kline, Chang & Li (2019) and Knisely & Zhong (2019) investigated the effects of thermochemical non-equilibrium (TCNE) on boundary-layer stability; the effect of ablation has also been considered (Mortensen & Zhong 2016; Miró Miró & Pinna 2021). On the other hand, while wall-bounded turbulence under low-enthalpy conditions (i.e. with air behaving as a calorically perfect gas) has been carefully scrutinized over the years, the interplay between TCNE conditions and turbulence has received much less attention. Compressible turbulent boundary-layer (TBL) configurations have been investigated with different turbulence injection techniques and wall treatments; for a thorough overview of DNS of TBLs at relatively low free-stream Mach numbers ($M_\infty \leq 3$), the reader may refer to the work of Wenzel *et al.* (2018) and references therein. Many efforts have been devoted to extending the validity of scaling laws derived for incompressible configurations (Morkovin 1962) to compressible ones. Inspection of the hypersonic regime by means of high-fidelity simulations was initiated by the studies of Duan, Beekman & Martín (2011), who performed DNS of temporally evolving TBLs with nominal free-stream Mach numbers up to 12 to prove the robustness of Morkovin's hypothesis even in high-speed flows. A DNS of a transitional spatially evolving boundary layer at Mach 6 was performed by Franko & Lele (2013), whereas Zhang, Duan & Choudhari (2018) produced a database of cooled boundary layers up to $M_\infty = 14$, with flow conditions representative of those encountered in hypervelocity wind tunnels. Huang *et al.* (2020) performed DNSs of zero-pressure-gradient hypersonic TBLs and conducted *a priori* and *a posteriori* analyses of the performances of different Reynolds-averaged Navier–Stokes (RANS) models. A

handful of works focused on high-enthalpy flows, at free-stream conditions for which dissociation and recombination reactions are triggered (Martin & Candler 2001; Duan & Martín 2009). A comparative study between low- and high-enthalpy, reactive, temporally evolving boundary layers was performed by Duan & Martín (2011b); the authors found that the two closely resemble each other, since the scaling laws validated under low-enthalpy conditions still hold or could be generalized for high-enthalpy applications. The interaction of finite-rate chemistry with transitional and turbulent spatially developing boundary layers was recently investigated by Passiatore *et al.* (2021) and Di Renzo & Urzay (2021), who isolated the effects of chemical non-equilibrium on pseudo-adiabatic and cooled TBLs, respectively. In both studies, the effect of thermal non-equilibrium was neglected since the selected edge conditions were not expected to promote significant vibrational relaxation phenomena. The influence of these processes on turbulent flow behaviour has been investigated only for unconfined flow configurations so far, such as isotropic turbulence decay (Neville *et al.* 2014; Khurshid & Donzis 2019; Zheng *et al.* 2020) and mixing layers (Neville *et al.* 2015; Fiévet *et al.* 2019).

The present work aims at bridging this knowledge gap with the investigation of a TBL exposed to both thermal and chemical non-equilibrium conditions. For that purpose, we perform a DNS of a flat-plate boundary layer, subjected to post-shock conditions of a 6° sharp wedge flying at Mach 20. The edge thermodynamic state is such that vibrational relaxation phenomena are significant within the boundary layer, while a mild chemical activity is developed.

The paper is organized as follows. The governing equations, the numerical method, the outline of the simulation and the main parameters are described in § 2. Numerical results are presented in § 3; focusing mostly on the fully turbulent regime, we illustrate the behaviour of the dynamic field and compare different transformations for the streamwise velocity. Afterwards, thermal non-equilibrium effects are inspected by highlighting the tight coupling with turbulent mixing mechanisms. Correlations deriving from the strong Reynolds analogy are also evaluated, as well as classical closures of the new terms arising in the RANS equations. Lastly, the coupling between compressibility effects and thermal non-equilibrium is investigated. Conclusions are then drawn in § 4.

2. Methodology

2.1. Governing equations and thermodynamic models

The fluid under investigation in the current study is air at high temperature, modelled as a five-species mixture of N₂, O₂, NO, O and N. When considering a gas under thermal non-equilibrium conditions, the vibrational energetic levels of the molecules in the mixture are partially excited and no longer equilibrated with the roto-translational ones. A direct consequence is that, even assuming that particle populations follow a Boltzmann distribution, the utilization of a single static temperature to represent all the energetic modes is no longer valid (Anderson 2006). A classical approach to deal with such conditions, referred to as multi-temperature in the literature, consists of taking into account the vibrational levels separately, by means of additional ‘vibrational’ temperatures for each molecule. In order to keep a reasonable number of equations to be integrated, the two-temperature model of Park (1988) is adopted for the present calculations. Such a model, widely used in previous works (see, e.g. Hudson *et al.* 1997; Franko, MacCormack & Lele 2010; Bitter & Shepherd 2015), assumes that the vibrational energy states of each molecule satisfy a Boltzmann distribution characterized by only one vibrational temperature T_V , common to all diatomic species in the mixture (that is, N₂, O₂ and NO).

A single additional conservation equation is, therefore, needed for the total vibrational energy e_V . Thus, the behaviour of such flows is governed by the compressible Navier–Stokes equations for multicomponent chemically reacting and thermally relaxing gases, which read

$$\frac{\partial \rho}{\partial t} + \frac{\partial \rho u_j}{\partial x_j} = 0 \quad (2.1)$$

$$\frac{\partial \rho u_i}{\partial t} + \frac{\partial (\rho u_i u_j + p \delta_{ij})}{\partial x_j} = \frac{\partial \tau_{ij}}{\partial x_j} \quad (2.2)$$

$$\frac{\partial \rho E}{\partial t} + \frac{\partial [(\rho E + p) u_j]}{\partial x_j} = \frac{\partial (u_i \tau_{ij})}{\partial x_j} - \frac{\partial (q_j^{TR} + q_j^V)}{\partial x_j} - \frac{\partial}{\partial x_j} \left(\sum_{n=1}^{NS} \rho_n u_{nj}^D h_n \right) \quad (2.3)$$

$$\frac{\partial \rho_n}{\partial t} + \frac{\partial (\rho_n u_j)}{\partial x_j} = - \frac{\partial \rho_n u_{nj}^D}{\partial x_j} + \dot{\omega}_n \quad (n = 1, \dots, NS - 1) \quad (2.4)$$

$$\frac{\partial \rho e_V}{\partial t} + \frac{\partial \rho e_V u_j}{\partial x_j} = \frac{\partial}{\partial x_j} \left(-q_j^V - \sum_{m=1}^{NM} \rho_m u_{mj}^D e_{Vm} \right) + \sum_{m=1}^{NM} (Q_{TVm} + \dot{\omega}_m e_{Vm}). \quad (2.5)$$

In the preceding formulation, ρ is the mixture density, t the time coordinate, x_j the space coordinate in the j th direction of a Cartesian coordinate system, with u_j the velocity vector component in the same direction, p is the pressure, δ_{ij} the Kronecker symbol and τ_{ij} the viscous stress tensor, modelled as

$$\tau_{ij} = \mu \left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right) - \frac{2}{3} \mu \frac{\partial u_k}{\partial x_k} \delta_{ij}, \quad (2.6)$$

with μ the mixture dynamic viscosity. In (2.3), $E = e + \frac{1}{2} u_i u_i$ is the specific total energy (with e the mixture internal energy), q_j^{TR} and q_j^V the roto-translational and vibrational contributions to the heat flux, respectively; u_{nj}^D denotes the diffusion velocity and h_n the specific enthalpy for the n th species. In the species conservation equations (2.4), $\rho_n = \rho Y_n$ represents the n th species partial density (Y_n being the mass fraction) and $\dot{\omega}_n$ the rate of production of the n th species. The sum of the partial densities is equal to the mixture density $\rho = \sum_{n=1}^{NS} \rho_n$, NS being the total number of species. To ensure total mass conservation, the mixture density and $NS - 1$ species conservation equations are solved, while the partial density of the NS th species is computed as $\rho_{NS} = \rho - \sum_{n=1}^{NS-1} \rho_n$. In the following, we set such species as molecular nitrogen, since it is the most abundant one throughout the computational domain. In (2.5), e_V represents the mixture vibrational energy, given by

$$e_V = \sum_{m=1}^{NM} Y_m e_{Vm}, \quad (2.7)$$

with e_{Vm} the vibrational energy of the m th molecule and NM their total number. In the same equation, $Q_{TV} = \sum_{m=1}^{NM} Q_{TVm}$ represents the energy exchange between vibrational and translational modes (due to molecular collisions and linked to energy relaxation phenomena) and $\sum_{m=1}^{NM} \dot{\omega}_m e_{Vm}$ the vibrational energy lost or gained due to molecular depletion or production.

Each species is assumed to behave as a thermally perfect gas; Dalton's pressure mixing law leads then to the thermal equation of state

$$p = \rho T \sum_{n=1}^{NS} \frac{\mathcal{R}Y_n}{\mathcal{M}_n} = T \sum_{n=1}^{NS} \rho_n R_n, \quad (2.8)$$

with R_n and \mathcal{M}_n the gas constant and molecular weight of the n th species, respectively, and $\mathcal{R} = 8.314 \text{ J mol}^{-1} \text{ K}^{-1}$ the universal gas constant. The thermodynamic properties of high- T air species are computed considering the contributions of translational, rotational and vibrational modes; specifically, the internal energy reads

$$e = \sum_{n=1}^{NS} Y_n h_n - \frac{p}{\rho}, \quad \text{with} \quad h_n = h_{f,n}^0 + \int_{T_{ref}}^T (c_{p,n}^T + c_{p,n}^R) dT' + e_{v,n}. \quad (2.9)$$

Here, $h_{f,n}^0$ is the n th species enthalpy of formation at the reference temperature ($T_{ref} = 298.15 \text{ K}$), $c_{p,n}^T$ and $c_{p,n}^R$ the translational and rotational contributions to the isobaric heat capacity of the n th species, computed as

$$c_{p,n}^T = \frac{5}{2}R_n \quad \text{and} \quad c_{p,n}^R = \begin{cases} R_n & \text{for diatomic species} \\ 0 & \text{for monoatomic species} \end{cases} \quad (2.10a,b)$$

and $e_{v,n}$ the vibrational energy of species n , given by

$$e_{v,n} = \begin{cases} \frac{\theta_n R_n}{\exp(\theta_n/T_V) - 1} & \text{for diatomic species} \\ 0 & \text{for monoatomic species} \end{cases} \quad (2.11)$$

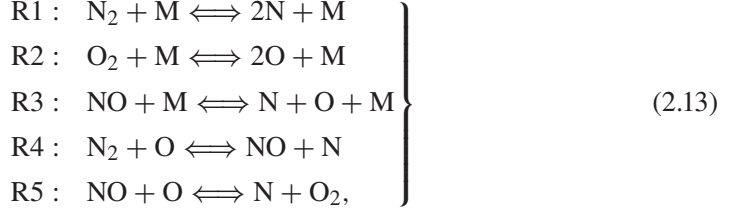
with θ_n the characteristic vibrational temperature of each molecule (3393 K, 2273 K and 2739 K for N_2 , O_2 and NO , respectively). After the numerical integration of the conservation equations, the roto-translational temperature T is computed from the specific internal energy (devoid of the vibrational contribution) directly, whereas an iterative Newton–Raphson method is used to compute T_V from (2.7).

The heat fluxes are modelled by means of Fourier's law, $q_j^{TR} = -\lambda_{TR}(\partial T/\partial x_j)$ and $q_j^V = -\lambda_V(\partial T_V/\partial x_j)$, λ_{TR} and λ_V being the roto-translational and vibrational thermal conductivities, respectively. Pure species' viscosity and thermal conductivities are computed using curve fits by Blottner, Johnson & Ellis (1971) and Eucken's relations, respectively (Hirschfelder & Curtiss 1969). The corresponding mixture properties are evaluated by means of Wilke's mixing rules (Wilke 1950). Mass diffusion is modelled by means of Fick's law

$$\rho_n u_{nj}^D = -\rho D_n \frac{\partial Y_n}{\partial x_j} + \rho_n \sum_{n=1}^{NS} D_n \frac{\partial Y_n}{\partial x_j}, \quad (2.12)$$

where the first term on the right-hand side represents the effective diffusion velocity and the second one is a mass corrector term that should be taken into account in order to satisfy the continuity equation when dealing with non-constant species diffusion coefficients (Poinsot & Veynante 2005). Specifically, D_n is an equivalent diffusion coefficient of species n into the mixture, computed following Hirschfelder's approximation (Hirschfelder

& Curtiss 1969), starting from the binary diffusion coefficients which are curve fitted in Gupta *et al.* (1990). Note that the molecular weight gradient contribution is neglected in (2.12), which therefore represents a rather simple model (albeit allowing for variable mass diffusion coefficients and non-constant Lewis numbers). The five species interact with each other through a reaction mechanism consisting of five reversible chemical steps (Park 1990)



with M being the third body (any of the five species considered). Dissociation and recombination processes are described by reactions R1, R2 and R3, whereas the shuffle reactions R4 and R5 represent rearrangement processes. The mass rate of production of the n th species is governed by the law of mass action

$$\dot{\omega}_n = \mathcal{M}_n \sum_{r=1}^{NR} (v''_{nr} - v'_{nr}) \times \left[k_{f,r} \prod_{n=1}^{NS} \left(\frac{\rho Y_n}{\mathcal{M}_n} \right)^{v'_{nr}} - k_{b,r} \prod_{n=1}^{NS} \left(\frac{\rho Y_n}{\mathcal{M}_n} \right)^{v''_{nr}} \right], \quad (2.14)$$

where v'_{nr} and v''_{nr} are the stoichiometric coefficients for reactants and products in the r th reaction for the n th species, respectively, and NR is the total number of reactions. Furthermore, $k_{f,r}$ and $k_{b,r}$ denote the forward and backward reaction rates of reaction r , modelled by means of Arrhenius' law. The tight coupling between chemical and thermal non-equilibrium, due to their concurrent presence in such flows, is taken into account by means of a suitable modification of the temperature values used for computing the reaction rates. A geometric-averaged temperature is considered for the dissociation reactions R1, R2 and R3 in (2.13), computed as $T_{avg} = T^q T_V^{1-q}$ with $q = 0.7$ (Park 1988).

Lastly, the vibrational–translational energy exchange is computed as

$$Q_{TV} = \sum_{m=1}^{NM} Q_{TV,m} = \sum_{m=1}^{NM} \rho_m \frac{e_{Vm}(T) - e_{Vm}(T_V)}{t_m}, \quad (2.15)$$

where t_m is the m th molecular relaxation time evaluated by means of the expression of Millikan & White (1963). Specifically, the relaxation time of the m th molecule with respect to the n th species writes

$$t_{mn}^{MW} = \frac{p}{p_{atm}} \exp \left[a_{mn} (T^{-(1/3)} - b_{mn}) - 18.42 \right], \quad (2.16)$$

where p is the pressure, $p_{atm} = 101\,325$ Pa and a_{mn} and b_{mn} are coefficients reported in Park (1993). Since this expression tends to underestimate the experimental data at temperatures above 5000 K, a high-temperature correction was proposed by Park (1989)

$$t_{mn} = t_{mn}^{MW} + t_{mn}^c \quad \text{with } t_{mn}^c = \sqrt{\frac{\phi_{mn}}{\mathcal{M}_m \sigma}}. \quad (2.17)$$

Here, $\phi_{mn} = \mathcal{M}_m \mathcal{M}_n / (\mathcal{M}_m + \mathcal{M}_n)$ and $\sigma = \sqrt{8\mathcal{R}/T\pi} (7.5 \times 10^{-12} NA/T)$, NA being Avogadro's number. The mean value is then evaluated with a weighted harmonic average

$$t_m = \sum_{n=1}^{NS} \frac{\rho_n}{\mathcal{M}_n} \sum_{n=1}^{NS} \frac{t_{mn}}{\rho_n/\mathcal{M}_n}. \quad (2.18)$$

The complete formulation of transport coefficients laws and thermochemical models is provided in [Appendix A](#).

2.2. Numerical method

The Navier–Stokes equations are integrated numerically by using a high-order centred finite-difference scheme (Sciacovelli *et al.* 2021). The convective fluxes are discretized by means of central tenth-order differences, supplemented with a higher-order adaptive artificial dissipation. This consists in a blend of a ninth-order accurate dissipation term based on tenth-order derivatives of the conservative variables (used to damp grid-to-grid oscillations) along with a low-order shock-capturing term. A highly selective sensor, based on Ducros' extension of Jameson's pressure-based sensor (Ducros *et al.* 1999) is used to turn on shock capturing in the immediate vicinity of flow discontinuities for all equations except the vibrational energy equation. For the latter, a shock sensor based on second-order derivatives of the vibrational temperature was preferred to ensure appropriate damping of spurious oscillations. Standard fourth-order differences are used for the viscous fluxes. Time integration is carried out by means of a third-order total variation diminishing (TVD) Runge–Kutta scheme (Gottlieb & Shu 1998). More details about the present numerical technique, as well as a complete assessment for a variety of highly compressible flow problems including chemically reacting hypersonic boundary layers can be found in Sciacovelli *et al.* (2021).

2.3. Computational set-up

The configuration under investigation is a spatially evolving, zero-pressure-gradient flat-plate boundary layer, sketched in [figure 1](#). The prescribed edge conditions of $M_e = 12.48$, $T_e = 594.3$ K and $p_e = 4656$ Pa are representative of those downstream of a shock wave generated by a 6° sharp wedge flying at $M = 20$ at an altitude of approximately 36 km. The stagnation enthalpy at the edge of the boundary layer is $H_e = h_e + u_e^2/2 = 18.66$ MJ kg $^{-1}$, a value comparable to those of the high-enthalpy cases considered by Duan & Martín (2011b) and Di Renzo & Urzay (2021). Air at such free-stream conditions is supposed to be in thermochemical equilibrium ($X_{N_2} = 0.79$, $X_{O_2} = 0.21$, X_n being the n th species molar fraction). Of note, a similar scenario has been considered by Kline *et al.* (2019) for stability studies. The computational domain is a rectangular box enclosed within the shock layer, highlighted in red in [figure 1](#). The extent of the domain is $L_x \times L_y \times L_z = 3000\delta_{in}^* \times 120\delta_{in}^* \times 30\pi\delta_{in}^*$, with x , y and z denoting the streamwise, wall-normal and spanwise directions, respectively, and δ_{in}^* the displacement thickness of the boundary layer at the inlet section, computed as $\delta^* = \int_0^\delta (1 - \rho u/\rho_e u_e) dy$ (δ being the boundary-layer thickness at 99% of the edge velocity). The computational grid is $N_x \times N_y \times N_z = 9660 \times 480 \times 512$, for a total of approximately 2.4 billions grid points. The grid spacing is uniform in the streamwise and spanwise directions, whereas a constant grid stretching of 0.7% is applied in the wall-normal direction. The calculation is initiated in the laminar region at a distance x_0 from the plate leading edge. The profiles of

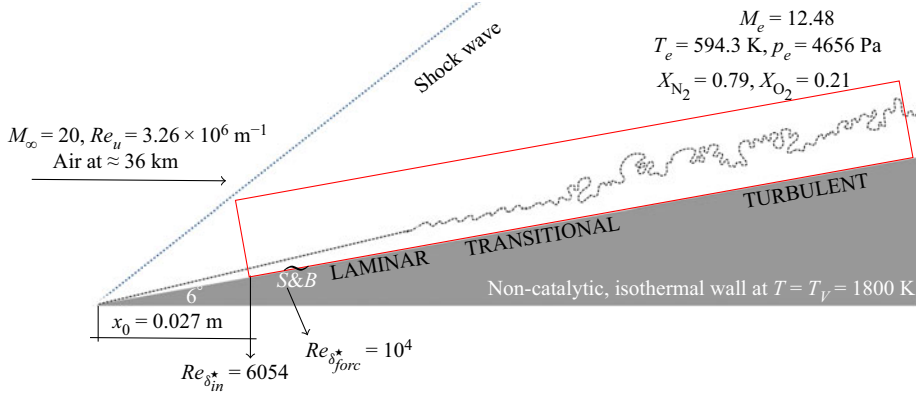


Figure 1. Sketch of the configuration under investigation.

the conservative variables prescribed at the inflow are generated by solving the locally self-similar theory for a two-dimensional chemically out-of-equilibrium and vibrationally equilibrated boundary layer (Sciacovelli *et al.* 2021); the inflow Reynolds number based on the inlet displacement thickness is $Re_{\delta_{in}^*} = 6054$. Using a thermal equilibrium simplifies the numerical setting and is not expected to alter the qualitative behaviour of the turbulent zone, of interest in the following analyses. A sponge layer is applied downstream of the inlet boundary, up to $(x - x_0)/\delta_{in}^* = 20$, to prevent abrupt distortions of the boundary layer. Characteristic-based boundary conditions are imposed at the top and right boundaries, and periodic conditions are set in the spanwise direction. The wall is assumed non-catalytic (i.e. $\partial Y_n/\partial y = 0$) and isothermal with $T = T_V = 1800 \text{ K}$. The first condition implies that the surface does not participate to chemical processes. This is an idealization of what actually happens in practical flight conditions; the material of TPS of flight vehicles may be catalytic, thus promoting recombination of the atoms in the mixture in the near-wall region. The investigation of finite-rate catalysis is beyond the scope of the present discussion, but it could be of interest to understand the contribution of wall catalysis to thermal stresses (Bonelli, Pascasio & Colonna 2021) in future works. On the other hand, the hypothesis of wall thermal equilibrium is mainly dictated by a lack of knowledge about the most appropriate conditions to be used for vibrational temperature. In previous studies of laminar flows around hollow cylinder flares (Kianvashrad & Knight 2017, 2019), both adiabatic and isothermal boundary conditions were utilized for the vibrational energy, leading to similar results in terms of heat transfer. Furthermore, past boundary-layer stability studies accounting for thermal non-equilibrium assumed thermal equilibrium at the wall (Bitter & Shepherd 2015; Kline *et al.* 2019; Knisely & Zhong 2019). Unfortunately, no information is available for turbulent flows. Nevertheless, the contribution of the vibrational heat transfer to the total wall heat flux will be shown to be small with respect to the roto-translational counterpart (as later detailed in § 3), justifying *a posteriori* the choice of a Dirichlet boundary condition as a first approximation. The selected wall temperature value, on the other hand, is representative of realistic flight data (Park 2004).

Transition to turbulence is induced by means of a suction and blowing strategy, similar to the one adopted in Passiatore *et al.* (2021). Specifically, the following wall-normal velocity

is imposed along a wall strip close to the inflow:

$$\frac{v_{wall}}{u_{\infty}} = e^{-0.4g(x)^2} A \left[\sin(2\pi g(x) - \omega t) + \cos(\beta z) + 0.05 \sin\left(2\pi g(x) - \omega t + \frac{\pi}{4}\right) \cos(\beta z) \right], \quad (2.19)$$

where $g(x) = (x - x_{forc})/(\delta_{in}^* \sqrt{2}\sigma)$, with $\sqrt{2}\sigma = 0.85(2\pi/\omega)$ and x_{forc} is the centre of the Gaussian-like distribution which modulates the forcing function. The Reynolds number at the forcing location is $Re_{\delta_{forc}^*} = 10^4$, corresponding to $x_{forc} = 7.4 \times 10^{-2}$ m. In (2.19), the non-dimensional pulsation $\omega = \tilde{\omega}\delta_{in}^*/c_{\infty}$ (c_{∞} being the free-stream speed of sound) corresponds to a dimensional frequency of $\tilde{f} = \tilde{\omega}/2\pi = 75$ kHz and $\beta = 0.4/\delta_{in}^*$ is the spanwise wavenumber. Lastly, A is the forcing amplitude, which has been set equal to 5%. The presence of stationary modes at a strong amplitude in the suction-and-blowing injection strip mimics the effects of roughness spots and is used to speed up breakdown to turbulence within the prescribed computational domain.

2.4. Data collection and analysis

In the results presented below, flow statistics are computed by averaging in time and in the spanwise homogeneous direction, after that the initial transient is evacuated. First- and second-order moments of various flow quantities will be presented and discussed. For a given variable f , we denote with $\bar{f} = f - f'$ the standard time and spanwise average, with f' the corresponding fluctuation, whereas $\tilde{f} = f - f''$ denotes the density-weighted Favre averaging, with f'' the Favre fluctuation and $\tilde{f} = \overline{\rho f}/\bar{\rho}$. The sampling time interval is constant and corresponds exactly to 300 samples per each period of the forcing harmonic; specifically, $\Delta t_{stats}^+ = \Delta t_{stats} (u_{\tau}^2 \bar{\rho}_w / \bar{\mu}_w) = 0.16$, with $u_{\tau} = \sqrt{\bar{\tau}_w / \bar{\rho}_w}$ the friction velocity based on the averaged wall shear stress $\bar{\tau}_w$ at the end of the computational domain. Data are collected for more than three turnover times, corresponding to $T_{stats} (u_{\tau}^2 / \bar{\mu}_w / \bar{\rho}_w) \approx 6700$ for a total of $\approx 40\,000$ temporal snapshots. In addition to flow statistics, instantaneous planes and specific meshlines are extracted with a frequency twenty and forty times higher with respect to the fundamental mode in the forcing function, for a total of 3200 and 6400 samples, respectively. The analysis will mainly focus on five selected streamwise stations; one located in the laminar region (for purpose of comparison), one in the transitional region and the last three in the turbulent portion of the domain. [Table 1](#) reports the values of the Reynolds number based on the distance from the leading edge $Re_x = \rho_e u_e x / \mu_e$ at the selected stations and some boundary-layer properties. The Reynolds number based on the local momentum thickness $Re_{\theta} = \rho_e u_e \theta / \mu_e$ (with $\theta = \int_0^{\delta} \rho u / \rho_e u_e (1 - \rho u / \rho_e u_e) dy$) reaches values close to ≈ 6000 in the fully turbulent region, corresponding to $Re_{\theta}^{inc} = Re_{\theta} \mu_e / \bar{\mu}_w$ of approximately 3000. The displacement-thickness-based Reynolds number $Re_{\delta^*} = \rho_e u_e \delta^* / \mu_e$ and the friction Reynolds number $Re_{\tau} = \bar{\rho}_w u_{\tau} \delta / \bar{\mu}_w$ reach values up to $\approx 16 \times 10^4$ and ≈ 1100 , respectively. Of note, the grid spacings ensure a DNS-like spatial resolution everywhere. Here, the notation ‘ \bullet^+ ’ denotes normalization with respect to the viscous length scale $l_v = \bar{\mu}_w / (\bar{\rho}_w u_{\tau})$. Unless otherwise specified, the wall-normal evolution of statistics is displayed in inner semi-local units $y^* = \bar{\rho} u_{\tau}^* y / \bar{\mu}$, with $u_{\tau}^* = \sqrt{\bar{\tau}_w / \bar{\rho}}$.

$(x - x_0)/\delta_{in}^*$	744	1396	1757	2329	2900
$Re_x \times 10^{-6}$	4.51	8.45	10.6	14.1	17.6
$Re_{\delta^*} \times 10^{-4}$	6.80	9.91	13.25	13.27	16.16
Re_{θ}	1600	2960	3560	5030	6200
Re_{θ}^{inc}	774	1426	1716	2423	2994
Re_{τ}	134	670	845	983	1128
Ma_{τ}	0.14	0.27	0.23	0.22	0.21
B_q^{TR}	0.14	0.38	0.30	0.26	0.24
B_q^V	0.016	0.01	0.01	0.016	0.017
Δx^+	4.38	10.54	8.80	7.77	7.26
Δz^+	2.60	6.25	5.21	4.60	4.30
$\Delta y_w^+ - \Delta y_{\delta}^+$	0.39–1.36	0.93–5.80	0.78–6.90	0.70–7.82	0.64–8.83

Table 1. Boundary-layer properties at five selected downstream stations. In the table, $Ma_{\tau} = u_{\tau}/\bar{c}_w$ is the friction Mach number, $B_q^{TR} = \bar{q}_w^{TR}/(\bar{\rho}_w u_{\tau} \bar{h}_w)$ and $B_q^V = \bar{q}_w^V/(\bar{\rho}_w u_{\tau} \bar{h}_w)$ are the roto-translational and vibrational dimensionless heat fluxes. Lastly, Δx^+ , Δy_w^+ , Δy_{δ}^+ and Δz^+ denote the grid sizes in inner variables in the x -direction, y -direction at the wall and at the boundary-layer edge and in the z -direction, respectively.

3. Results

3.1. Global flow properties

The streamwise evolution of selected quantities at the wall is first discussed. [Figures 2\(a\)](#) and [2\(b\)](#) report the distributions of the skin friction coefficient C_f and the heat flux coefficients C_q and C_q^V , defined as

$$C_f = \frac{2\bar{\tau}_w}{\rho_e u_e^2}, \quad C_q^{TR} = \frac{\bar{q}_w^{TR}}{\rho_e u_e^3}, \quad C_q^V = \frac{\bar{q}_w^V}{\rho_e u_e^3}, \quad (3.1a-c)$$

the net total heat flux being given by the sum of the two contributions. A ramp up starting at $(x - x_0)/\delta_{in}^* \approx 1100$ leads to a sharp overshoot in the C_f profile, with a peak of the wall shear stress almost five times larger than its corresponding laminar value. The flow achieves a turbulent regime starting from $(x - x_0)/\delta_{in}^* \approx 1800$, with a smoothly decreasing C_f up to the end of the computational domain. The roto-translational contribution of the wall heat flux C_q^{TR} is shown to be largely predominant with respect to the vibrational counterpart C_q^V (which is multiplied by ten to match the range of [figure 2b](#)), mainly because of the vibrational thermal conductivity $\bar{\lambda}_V$, whose values are approximately one order of magnitude smaller than $\bar{\lambda}_{TR}$. While C_q^{TR} closely follows the skin friction coefficient profile, C_q^V exhibits a different evolution. It rapidly decreases after the forcing strip and stays approximately constant up to the breakdown to turbulence; afterwards, the value in the turbulent region almost doubles the one in the laminar regime and keeps a constant value up to the end of the domain. The different trend of the heat fluxes is related to the evolution of vibrational temperature gradients across the boundary layer, as discussed later in [§ 3.3](#). In order to isolate the contributions of the mean and fluctuating field to the skin friction, the decomposition of Renard & Deck ([2016](#)) (later extended to compressible flows by Li *et al.* [2019](#)) has been computed; of note, the general hypotheses under which it is derived allows a straightforward application even in presence of thermal and chemical non-equilibrium effects. The skin friction coefficient can then be rewritten under the

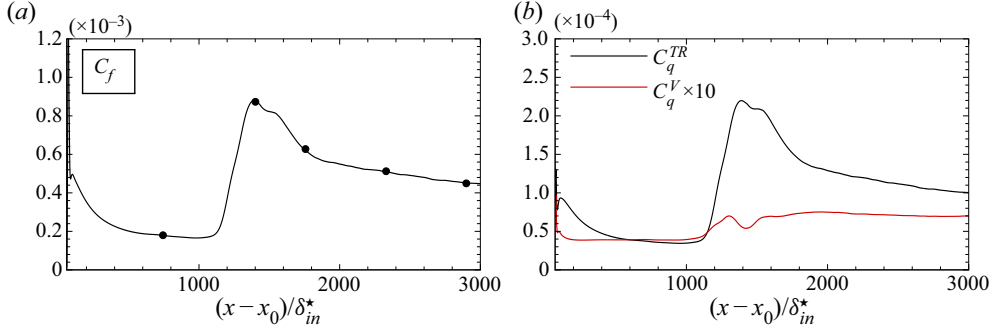


Figure 2. Evolution of (a) the skin friction coefficient C_f (black line) and (b) the heat flux coefficients C_q^{TR} and C_q^V (black and red lines, respectively). The filled symbols in (a) denote the streamwise location of the five stations selected in [table 1](#).

following form:

$$\begin{aligned}
 C_f = & \underbrace{\frac{2}{\rho_\infty u_\infty^3} \int_0^\delta \bar{\tau}_{xy} \frac{\partial \tilde{u}}{\partial y} dy}_{C_{f,1}} + \underbrace{\frac{2}{\rho_\infty u_\infty^3} \int_0^\delta -\overline{\rho u'' v''} \frac{\partial \tilde{u}}{\partial y} dy}_{C_{f,2}} \\
 & + \underbrace{\frac{2}{\rho_\infty u_\infty^3} \int_0^\delta (\tilde{u} - u_\infty) \left[\bar{\rho} \left(\tilde{u} \frac{\partial \tilde{u}}{\partial x} + \tilde{v} \frac{\partial \tilde{u}}{\partial y} \right) - \frac{\partial}{\partial x} (\bar{\tau}_{xx} - \bar{\rho} \tilde{u}'' - \bar{p}) \right] dy}_{C_{f,3}}, \quad (3.2)
 \end{aligned}$$

where $C_{f,1}$, $C_{f,2}$ and $C_{f,3}$ represent the mean-field molecular dissipation, the turbulent dissipation and the effects related to boundary-layer spatial growth, respectively. The sum of the three terms and their separate contributions are displayed in [figure 3](#). An excellent agreement with respect to the direct C_f computation is observed in the pseudo-laminar and fully turbulent regions, with only minor deviations in the transition region. The $C_{f,3}$ contribution is negligible everywhere but at the breakdown-to-turbulence location, where it takes negative values due to the abrupt thickening of the boundary layer. The decrease of $C_{f,3}$ is counterbalanced by a large increase of the Reynolds stress-related term, whereas the growth of the mean-field contribution, $C_{f,1}$, is slightly delayed with respect to the other two. In the turbulent region, $C_{f,1}$ and $C_{f,2}$ are almost superposed, differently from what observed by [Passiatore *et al.* \(2021\)](#) where the former was shown to be predominant. Such a discrepancy can be ascribed to the much larger friction Reynolds numbers reached in the current configuration, leading to an increased turbulent contribution ([Fan, Li & Pirozzoli 2019](#)).

3.2. Mean flow analysis

Different scalings for the profiles of the averaged streamwise velocity have been tested. First, the transformations of [Van Driest \(1956\)](#) and [Trettel & Larsson \(2016\)](#)

$$u_{VD} = \frac{1}{u_\tau} \int_0^{\tilde{u}} \sqrt{\frac{\bar{\rho}}{\bar{\rho}_w}} du, \quad u_{TL} = \int_0^{\tilde{u}} \left(\frac{\bar{\rho}}{\bar{\rho}_w} \right) \left[1 + \frac{1}{2} \frac{1}{\bar{\rho}} \frac{d\bar{\rho}}{dy} y - \frac{1}{\bar{\mu}} \frac{d\bar{\mu}}{dy} y \right] du \quad (3.3a,b)$$

are shown for the three turbulent stations in [figures 4\(a\)](#) and [4\(b\)](#), respectively. The former scaling has been shown to work reasonably well for adiabatic boundary layers, even at

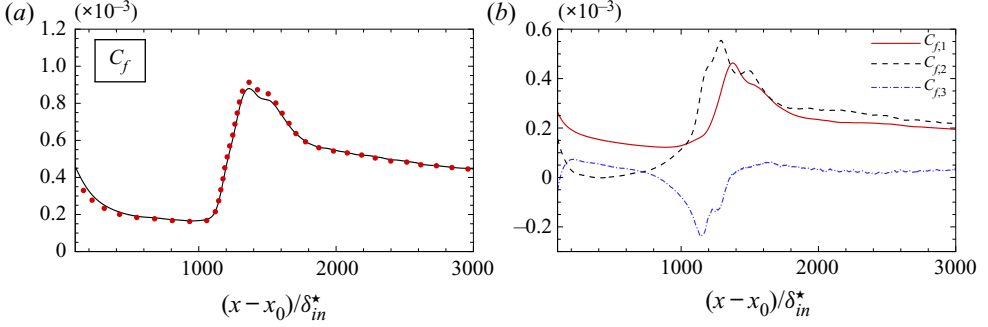


Figure 3. Evolution of (a) the skin friction coefficient C_f (black line) and Renard–Deck decomposition (red symbols) and (b) separate contribution of each term of (3.2).

high speeds (Passiatore *et al.* 2021). On the other hand, it becomes inaccurate when highly cooled configurations are considered, whether they be boundary layers (Zhang *et al.* 2018; Huang *et al.* 2020), pipe flows (Ghosh, Foysi & Friedrich 2010) or channel flows (Modesti & Pirozzoli 2016; Sciacovelli, Cinnella & Gloerfelt 2017). Figure 4(a) confirms such a trend, both the linear and logarithmic regions being offset with respect to the analytical laws. Of note, in figure 4 the logarithmic region is described by $(1/\kappa) \log y^+ + C$ and $(1/\kappa) \log y^* + C$, with $\kappa = 0.41$ and $C = 5.2$. The semi-local scaling of Trettel & Larsson, as expected, improves the near-wall prediction since it explicitly accounts for the stress-balance condition within the entire inner layer. A large scatter is, however, observed in the logarithmic region which shows a Re -dependence similar to the van Driest scaling. Although such a scaling works reasonably well for internal flows, it is not as good for external configurations, most likely because of an interaction with the wake region which is shown to be over-stretched in figure 4(b). Recently, Griffin, Fu & Moin (2021) proposed a new total-stress-based transformation (called hereafter Griffin–Fu–Moin scaling, u_{GFM}) with a constant-stress-layer assumption, which reads

$$u_{GFM} = \int_0^\delta \frac{\frac{1}{\mu^+} \frac{\partial u^+}{\partial y^*}}{1 + \frac{1}{\mu^+} \frac{\partial u^+}{\partial y^*} - \mu^+ \frac{\partial u^+}{\partial y^+}} dy^* \quad (3.4)$$

where μ^+ denotes normalization of the mean viscosity with respect to its wall value. Such a scaling has been shown to successfully collapse channels, pipe flows and boundary-layer configurations, even at large Mach numbers. A comparison between u_{TL} and u_{GFM} as a function of y^* is shown in figure 4(c) for a velocity profile extracted at $Re_\theta = 6200$. In the inner layer, viscous stresses are predominant and roughly correspond to the total shear stress resulting in a very good collapse for both scalings. On the contrary, turbulent shear stresses become dominant in the logarithmic region, which effect is most correctly taken into account by the total-stress-based scaling. This results in a better collapse of u_{GFM} onto the universal logarithmic profile with respect to u_{TL} , for which the slope of the logarithmic region is largely overestimated. Yet, the u_{GFM} transformation still predicts a higher slope and intercept in the logarithmic region compared with the classical incompressible values, in accordance with the recent study of Lee, Martin & Williams (2021). Finally, we verified that the transformation based on the constant-stress-layer assumption (i.e. $\tau/\tau_w \approx 1$ across the boundary layer, leading to (3.4)) and the one based on the actual total shear stress (extracted from DNS data) do not exhibit any discernible differences, confirming the

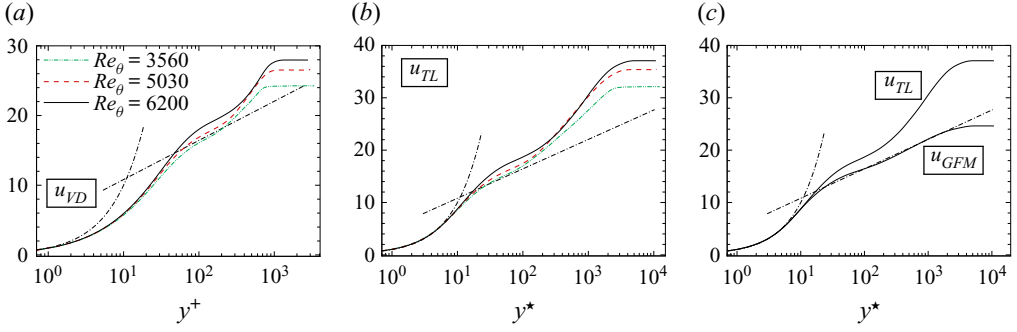


Figure 4. Wall-normal profiles of (a) the van Driest-transformed streamwise velocity, (b) Trettel & Larsson's transformation and (c) comparison between Trettel & Larsson transformation and total-stress-based scaling of Griffin *et al.* (2021) at $Re_\theta = 6200$.

validity of the constant-stress-layer hypothesis. Neither chemical activity nor thermal relaxation process should substantially alter the validity of the transformation since a certain degree of decoupling is observed between the thermochemical and turbulent activities, as previously noticed by Passiatore *et al.* (2021) and Di Renzo & Urzay (2021).

Figure 5 shows the Reynolds stress profiles at the turbulent station $Re_\theta = 6200$ normalized with respect to the semi-local friction velocity u_τ^* ; results are compared with low-enthalpy configurations extracted from Zhang *et al.* (2018) (cases M14Tw018 and M8Tw048) and Xu *et al.* (2021) (case M8T1). Despite the important differences in the values of the free-stream Mach numbers, friction Reynolds numbers, absolute wall temperatures and wall-cooling rates, the turbulent intensity profiles are comparable and do not exhibit any marked influence attributable to TCNE effects. Consistently with previous observations (Duan *et al.* 2011; Lagha *et al.* 2011; Zhang *et al.* 2018), the larger streamwise component and the smaller cross-flow one may indicate the presence of strong compressibility effects, as discussed later in § 3.6. Another scaling often investigated in wall-bounded compressible turbulence is the one that relates the mean velocity to the mean temperature profile for zero-pressure-gradient boundary layers. The modified Crocco relation derived by Walz (1969) writes

$$\frac{\tilde{T}}{T_e} = \frac{T_w}{T_e} + \frac{T_{aw} - T_w}{T_e} \left(\frac{\tilde{u}}{u_e} \right) + \frac{T_e - T_{aw}}{T_e} \left(\frac{\tilde{u}}{u_e} \right)^2, \quad (3.5)$$

with $T_{aw}/T_e = 1 + r((\gamma - 1)/2)M_e^2$ and r the recovery factor set equal to 0.9; the relation is shown in figure 6(a). In previous studies with high Mach numbers, wall-cooled, high-enthalpy boundary layers (Duan & Martín 2011b), it was found that such a relation deviates from the exact \tilde{T}/T_e profile extracted from DNS data; a significant discrepancy is indeed shown in the range $0.2 < \tilde{u}/u_e < 0.7$. This should not be surprising since relation (3.5) was derived under calorically perfect gas hypotheses. In order to remove the explicit dependence on thermal and chemical models, Duan & Martín (2011b) proposed an analogous enthalpy-based equation, which reads

$$\frac{\tilde{h}}{h_e} = \frac{h_w}{h_e} + \frac{h_{aw} - h_w}{h_e} f \left(\frac{\tilde{u}}{u_e} \right) - r \frac{u_e^2}{2h_e} \left(\frac{\tilde{u}}{u_e} \right)^2, \quad (3.6)$$

where $h_{aw} = h_e + \frac{1}{2}ru_e^2$. The function $f(\tilde{u}/u_e)$ has to be independent of free-stream conditions, wall temperature and surface catalysis (if any). For calorically perfect gases,

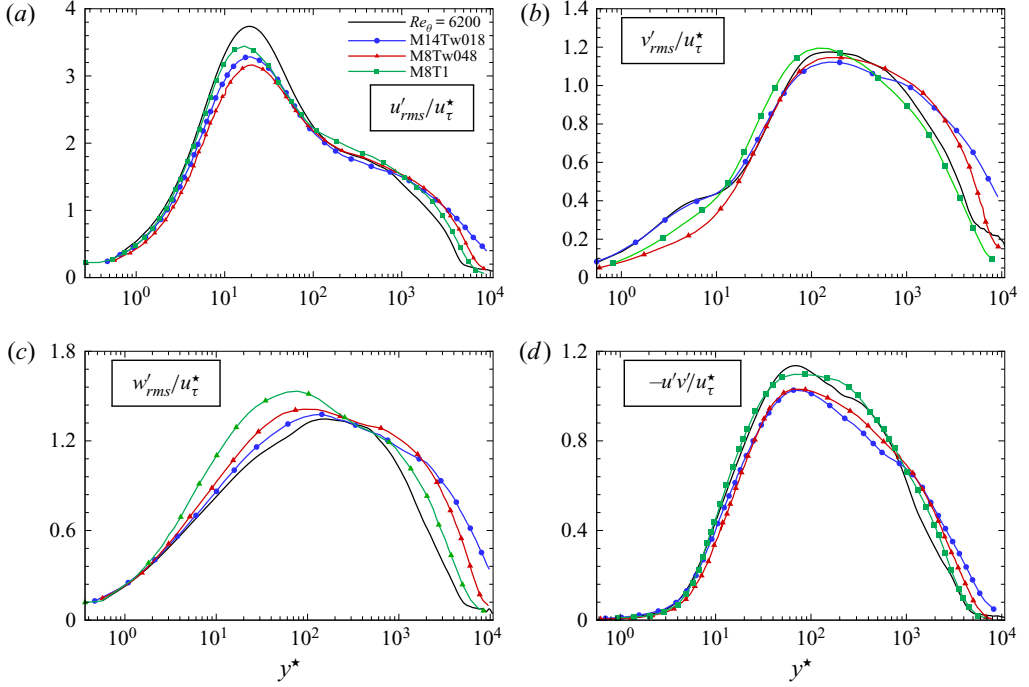


Figure 5. Reynolds stresses in the (a) streamwise, (b) wall-normal and (c) spanwise directions, and (d) Reynolds shear stress. The M14Tw018 and M8Tw048 profiles are extracted from Zhang *et al.* (2018), whereas M8T1 is from Xu *et al.* (2021).

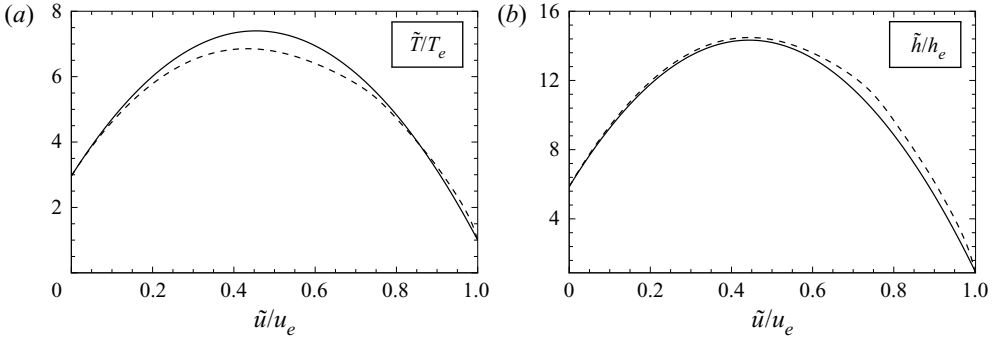


Figure 6. Walz's equation (a) and modified Walz's equation (b) as shown in (6) and (3.5), respectively, at $Re_\theta = 6200$. Dashed lines represent the exact ratio extracted from the DNS data.

(3.6) reduces to (3.5), apart from the term $f(\tilde{u}/u_e)$ which is equal to \tilde{u}/u_e in the classical formulation. By curve fitting the DNS data, Duan & Martín (2011b) obtained the following relation for $f(\tilde{u}/u_e)$:

$$f\left(\frac{\tilde{u}}{u_e}\right) = 0.1741 \left(\frac{\tilde{u}}{u_e}\right)^2 + 0.8259 \left(\frac{\tilde{u}}{u_e}\right). \quad (3.7)$$

The results of this transformation are displayed in figure 6(b) and show an improved collapse with respect to the classical temperature-based formulation.

3.3. Thermal non-equilibrium effects

The presence of thermal non-equilibrium is analysed by comparing the vibrational relaxation time with characteristic time scales of the dynamic field. For that purpose, the following vibrational Damköhler numbers are considered:

$$Da_v^{VS} = \frac{\bar{\mu}_w/\bar{\tau}_w}{t_m}, \quad Da_v^{LE} = \frac{\delta^*/u_\tau}{t_m}, \quad Da_v^{RT} = \frac{x/u_e}{t_m}, \quad (3.8a-c)$$

with $Da_v^{(\bullet)} = \mathcal{O}(1)$ denoting the regime in which non-equilibrium effects are relevant. The numerator of each Damköhler number is fixed for a given streamwise station and equal for each of the three molecules; the different trends of these parameters, for each of the three diatomic species, are therefore entirely dictated by the species molecular relaxation times t_m , which depend upon the local state of the gas. The evolution of these dimensionless numbers is displayed in [figure 7](#). The first ratio in (3.8a–c) relates the characteristic time of fluid motion in the viscous sublayer (VS) to the vibrational relaxation time of molecule m . [Figure 7\(a,d,g\)](#) shows the evolution of this quantity for the three molecules present in the mixture. It is shown that $Da_v^{VS} \ll 1$ everywhere for N_2 and O_2 whereas it is of the order of unity only for NO , whose concentration is, however, almost negligible (see § 3.5). The decoupling of the two time scales indicates that there exists a layer of thickness l_v in the inner (colder) region of the boundary layer in which the molecules are vibrationally frozen. The higher values of $Da_v^{VS} \ll 1$ observed at the laminar position are due to the predominance of viscous stresses. Of major interest is Da_v^{LE} , representing the ratio between the large-eddy (LE) turnover time and the relaxation time, constructed by means of the displacement thickness δ^* and u_τ . The orders of magnitude of the two time scales are comparable for molecular nitrogen ([figure 7b](#)), for which the peak of Da_v^{LE} ranges from 0.4 to 0.7 at $y^* \approx 10$ as the flow evolves in the turbulent region. On the other hand, [figures 7\(e\)](#) and [7\(h\)](#) show that $Da_v^{LE} \gg 1$ for O_2 and NO , implying that the two molecules reach a vibrational equilibrium with respect to the characteristic time of the large-scale turbulent motions much faster. Similar considerations hold for Da_v^{RT} , which compares the vibrational relaxation time with the local flow residence time (RT). As the flow evolves along the plate, there is enough time for O_2 and NO to achieve thermal equilibrium, whereas N_2 remains significantly out of equilibrium at all stations. This result is a consequence of the high vibrational temperature (and slower relaxation rate) characteristic of molecular nitrogen. Of note, both Da_v^{LE} and Da_v^{RT} exhibit much higher values at the three turbulent stations due to the contribution of turbulent mixing. Wall-normal profiles of the normalized mean translational and vibrational temperatures are reported in [figures 8\(a\)](#) and [8\(b\)](#), respectively. The translational temperature reaches a peak value of approximately $[2.3 - 2.5]T_w$, at a wall-normal location $y^* \approx 10$, corresponding to the peak of turbulent intensity. The vibrational temperature \tilde{T}_V is 20 % to 30 % smaller than \tilde{T} across the boundary layer, proving the existence of a vibrational–relaxation layer for the thermodynamic conditions under investigation, and also exhibits a peak located between the buffer layer and the lower logarithmic region. The major gap is visible at $Re_\theta = 2960$, which corresponds to the C_f peak. Additionally, the wall gradient $\partial\tilde{T}_V/\partial y^*$ is little sensitive to the peak’s magnitude and location, explaining the nearly constant trend of C_q^V previously observed in [figure 2\(b\)](#).

The normalized temperature difference $(\tilde{T} - \tilde{T}_V)/T_w$ is reported in [figure 9\(a,b\)](#) as a function of y^* and y/δ , respectively. Such a quantity increases in the transitional region, reaches a maximum of the order of ≈ 2000 K where C_f peaks, and then tends to decrease

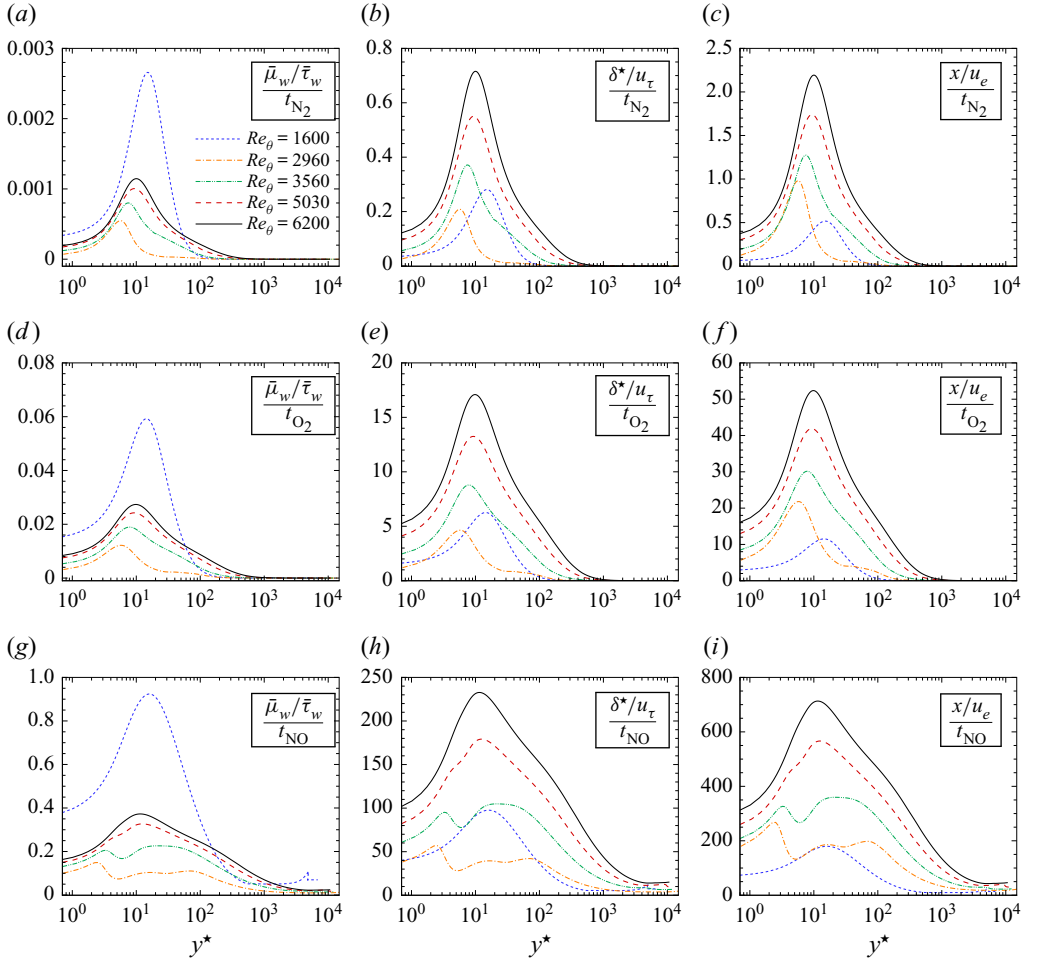


Figure 7. Vibrational Damköhler numbers for N_2 (a–c), O_2 (d–f) and NO (g–i). Damköhler number based on inner units (a)–(d)–(g), large-eddy turnover times (b)–(e)–(h) and flow residence time (c)–(f)–(i).

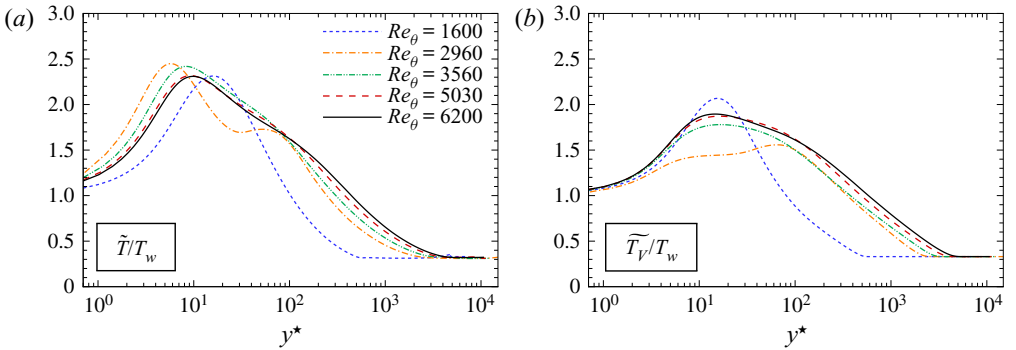


Figure 8. Normalized mean translational temperature (a) and vibrational temperature (b) at the selected streamwise stations.

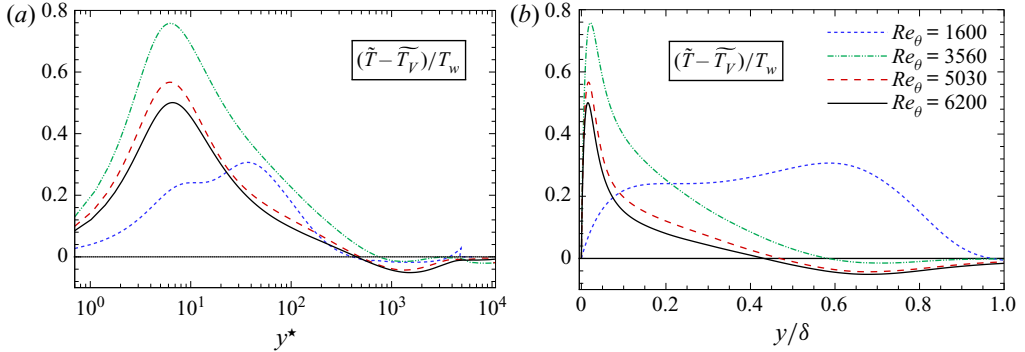


Figure 9. Normalized difference $(\tilde{T} - \tilde{T}_V)/T_w$ at the selected streamwise stations; (a) semi-local units, (b) outer scale. The dotted line corresponds to the zero value.

moving towards the turbulent region. The wall-normal maximum value moves from the outer part of the boundary layer ($y/\delta \approx 0.6$) to the buffer layer, where it stabilizes at $y^* \approx 8$. Thermodynamic non-equilibrium conditions are kept until the end of the computational domain, as already confirmed by inspection of the vibrational Damköhler number values. Positive $\tilde{T} - \tilde{T}_V$ values are observed from the VS (we recall that $T = T_V = 1800$ K at the wall) up to the first portion of the logarithmic region. However, as the flow evolves along the flat plate, a region characterized by $\tilde{T} - \tilde{T}_V < 0$ appears, meaning that the flow becomes vibrationally over-excited. The height at which the sign changes smoothly moves from the edge of the boundary layer up to $y/\delta \approx 0.45$, due to the boundary-layer thickening. This is clearly visible in figure 10, where we report a snapshot of the temperature difference along a longitudinal cut plane (top) and the isocontours of the second invariant of the velocity gradient tensor coloured by $(T - T_V)$. Fiévet *et al.* (2019), who conducted an analysis on the coupling between vibrational non-equilibrium and turbulent mixing, observed that the majority of non-equilibrium states occur in under-excited conditions. The rare regions in which there exists an over-excited non-equilibrium state are dictated by the link between the roto-translational and mechanical energies. The static temperature is coupled with the other variables through the equation of state; hence, a decrease of turbulent intensity (moving towards the edge of the boundary layer) produces a decrease of the translational temperature. At the same time, the vibrational energy remains nearly constant because it relaxes slowly, leading to an over-excitation in the outer layer, until the free-stream equilibrium conditions are reached by both temperatures.

Normalized root mean square (r.m.s.) of the Favre fluctuations of translational and vibrational temperatures across the boundary layer are displayed in figure 11. In figure 11(a), a peak emerges on each side of the \tilde{T}_{max} location ($y^* \approx 10$), as classically observed for wall-cooled TBLs (Duan, Beekman & Martín 2010; Zhang *et al.* 2018; Di Renzo & Urzay 2021). The predominance of the outer r.m.s. peak in the turbulent region is mainly caused by the decrease of \tilde{T} toward the outer boundary, the absolute temperature fluctuations being comparable in the inner and outer regions. In contrast, the vibrational temperature fluctuations exhibit no local minimum (figure 11b), and increase monotonically up to the boundary-layer edge, where they drop abruptly. This is in contrast to profiles typically observed in the laminar region (see the profile at $Re_\theta = 1600$ reported in the figure for comparison), and can be ascribed to thermal non-equilibrium: the vibrational relaxation times being slower than the translational ones and comparable

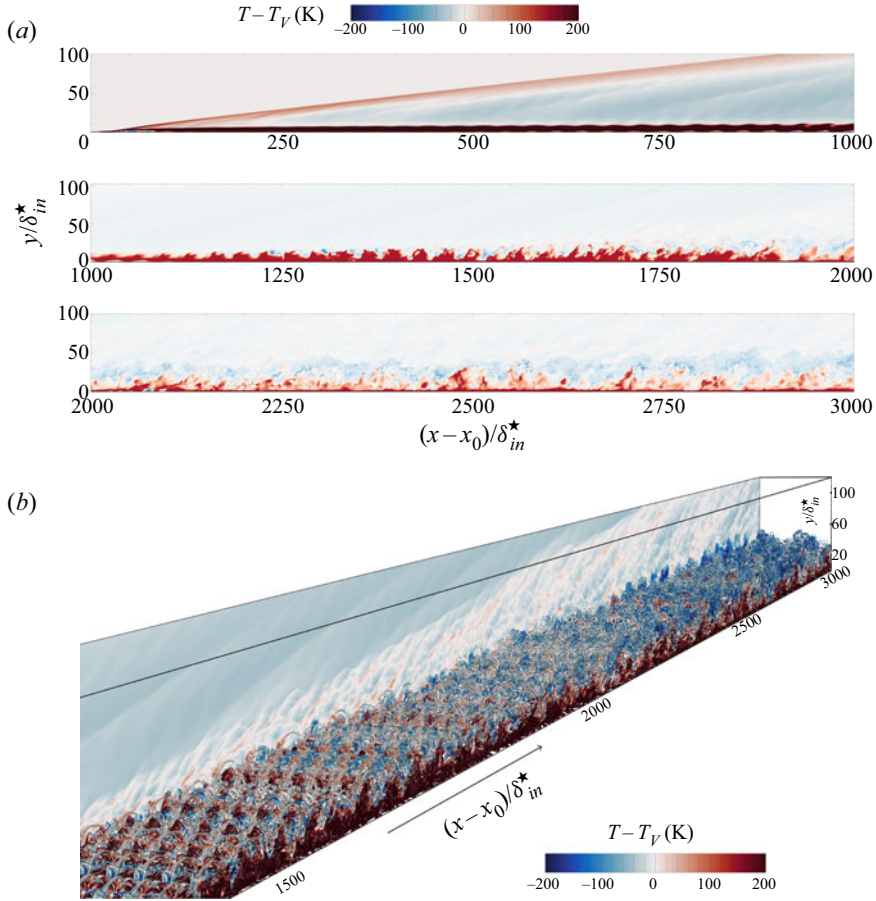


Figure 10. Instantaneous isocontours of $T - T_V$ in a xy -plane and second invariant of the velocity gradient tensor, Q , coloured by $T - T_V$.

to the turbulence time scales, T_V variations due to vibrational–translational energy exchanges occur at similar rates than variations caused by wall-normal turbulent motions. Therefore, the latter play a major role in redistributing vibrational temperature across the boundary layer, smoothing out the local minimum. The premultiplied energy spectra of the fluctuating temperatures, reported in [figure 12](#) as a function of the normalized spanwise wavenumber λ^* , provide a further confirmation of this trend. For both temperatures, a large-scale peak is visible at $\lambda_z^* \approx 1500$ and $y^* \approx 2500$, whereas an inner (slightly weaker) peak is observed only for the roto-translational temperature spectrum, coherently with the r.m.s. temperature fluctuations. Additionally, the well-developed spectra indicate that a fully turbulent state has been reached and the footprint of the forcing strategy has been completely lost. The persistence of thermal non-equilibrium conditions across the boundary layer is tightly related to the transition to a turbulent regime. Preliminary numerical experiments on the same configuration without boundary-layer tripping have shown that, for a laminar flow regime, the difference between the roto-translational and vibrational temperatures is almost negligible (of the order of 50 K) at the end of the plate. The thermal non-equilibrium observed in the turbulent case is strongly related to sweep and ejection turbulent motions, resulting in enhanced mixing as anticipated by [Urzay & Di Renzo \(2021\)](#) with *a priori* considerations. In [figure 13](#) we report the mean values of the

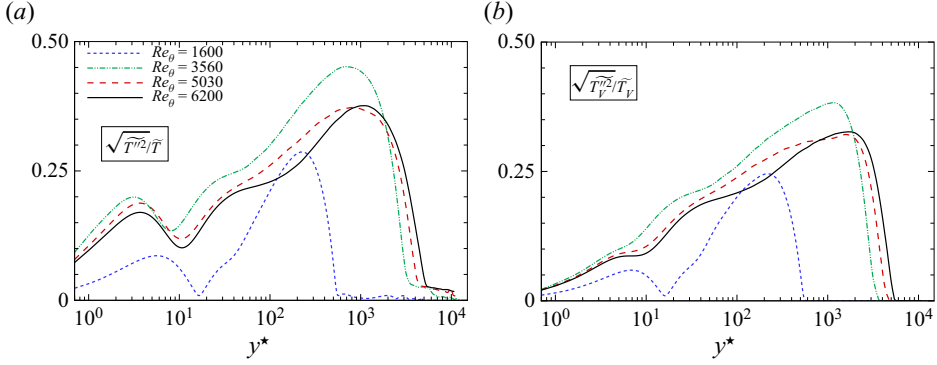


Figure 11. Root-mean-square temperature Favre fluctuations normalized by the local Favre average for translational temperature (a) and vibrational temperature (b).

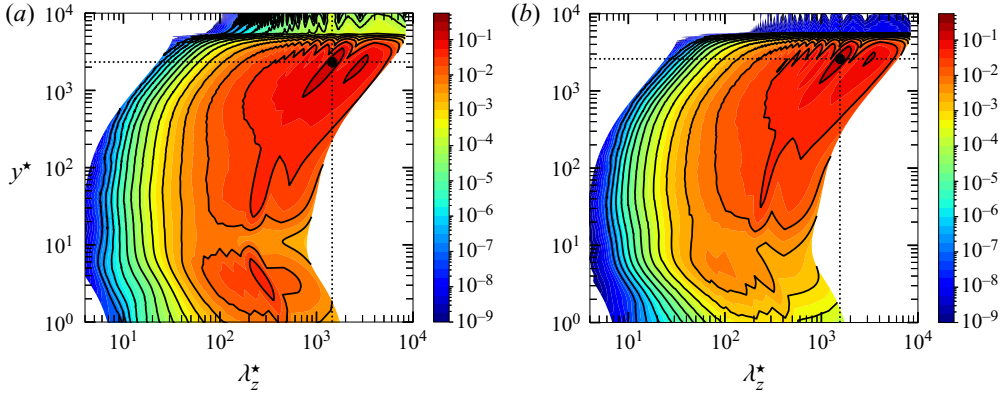


Figure 12. Premultiplied energy spectra $k_z E_{T''T''}/\tilde{T}$ (a) and $k_z E_{T_V''T_V''}/\tilde{T_V}$ (b), at $Re_\theta = 6200$.

two temperatures conditioned on sweep events (characterized by $u'' > 0$ and $v'' < 0$) and ejection events ($u'' < 0$ and $v'' > 0$) at $Re_\theta = 6200$ and for several wall-normal locations. Within the VS, sweeps entrain hot fluid from the overlying regions. Consequently, they are characterized by higher mean temperatures with respect to the total average in the inner region. The opposite is true for ejection events. The largest differences between sweeps and ejections conditionally averaged temperatures are registered for the roto-translational modes due to the sharper peak observed for \tilde{T} . An opposite trend is observed at normal locations beyond the temperature peak ($y^* \approx 10$). In this case, sweeps bring colder air from the outer layers towards the hotter, inner region; here, the translational modes rapidly reach equilibrium, whereas a longer relaxation time is needed for the vibrational ones. As a result, \tilde{T}_V is always smaller than \tilde{T} . On the other hand, ejection motions bring hotter, thermally out-of-equilibrium mixture towards the outer region of the boundary layer, where vibrational relaxation times rapidly increase because of the colder environment; therefore, in this region $\tilde{T}_V > \tilde{T}$. The largest temperature difference between sweeps and ejections events occurs in the first portion of the logarithmic region (up to $y^* \approx 100$), with values as large as 800 K for vibrational modes and 1000 K for the roto-translational ones. Further evidence of this phenomenon is provided by the conditional probability density functions (p.d.f.s) of the two temperatures, shown in [figure 14](#). Three wall-normal stations

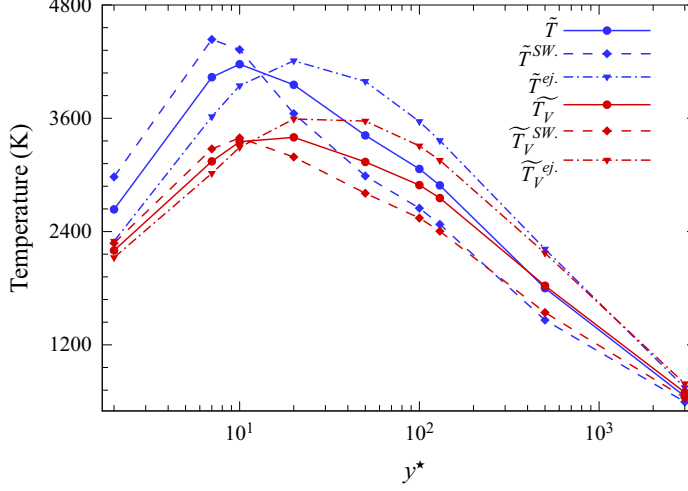


Figure 13. Mean values of \tilde{T} and \tilde{T}_V conditioned on sweep and ejection events, at $Re_\theta = 6200$ at various wall-normal heights. Blue lines: roto-translational modes; red lines: vibrational modes; dashed lines, diamonds: average over sweep events; dashed-dotted lines, triangles: average over ejection events; solid lines, circles: global average.

are considered: the location where the maximum temperature difference is registered (panels a and b, $y^* \approx 10$), the logarithmic region (panels c and d, $y^* \approx 100$) and the boundary layer edge (panels e and f, $y^* \approx 3000$). Close to the boundary layer edge, a heavy right tail is observed for the p.d.f.s of both events, with T_V values slightly larger than T ones. Moving inside the boundary layer, at $y^* \approx 100$, both p.d.f.s tend to become symmetric. The most remarkable differences are observed in the buffer layer with left-skewed distributions for T and symmetric ones for T_V , their shape being related to the presence of a local minimum for $\widetilde{T''^2}$ and a small plateau for $\widetilde{T_V''^2}$ as shown in figure 11.

3.4. Thermal modelling and Reynolds analogies

For thermally out-of-equilibrium flows, a new unclosed term appears by Favre averaging the vibrational energy equation (2.5), namely, the turbulent transport of vibrational energy $\overline{\rho u_j'' e_V''}$. Such a term is the analogue of the turbulent transport term appearing in the total energy equation. The latter is usually modelled by introducing a turbulent Prandtl number Pr_t , defined as

$$Pr_t = \frac{\overline{\rho u'' v''} \partial \tilde{T} / \partial y}{\overline{\rho v'' T''} \partial \tilde{u} / \partial y}. \quad (3.9)$$

According to the classical strong Reynolds analogy (SRA), Pr_t is expected to take values close to one throughout the flow (Morkovin 1962). Figure 15(a) (displayed in external units for better clarity) shows that the estimation fails at the peak of \tilde{T} , located at $y/\delta \approx 0.03$, since the coexistence of null temperature gradient and turbulent heat flux causes an overshoot. Such a behaviour was previously observed by Duan & Martín (2011b) and Di Renzo & Urzay (2021) (for a discussion about the validity of this assumption for adiabatic hypersonic boundary layers, see Passiatore *et al.* 2021). By analogy, the turbulent flux of

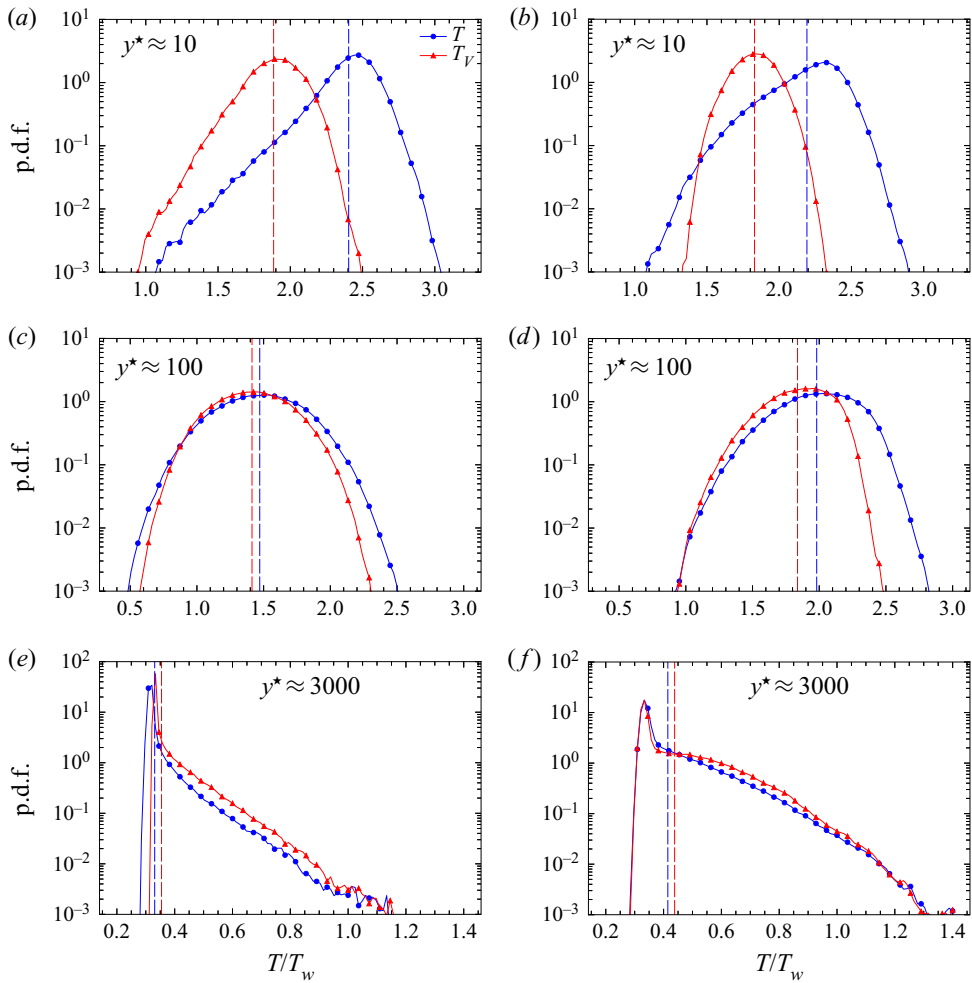


Figure 14. Conditional probability density function (p.d.f.) of translational and vibrational temperatures at $Re_\theta = 6200$ for y^* approximately equal to 10, 100 and 3000, respectively; sweeps events (*a,c,e*) and ejections events (*b,d,f*).

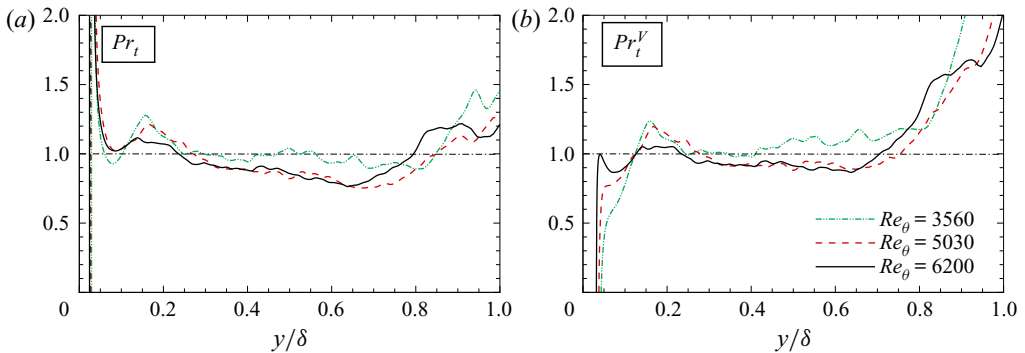


Figure 15. Turbulent Prandtl number (*a*) and turbulent vibrational Prandtl number (*b*).

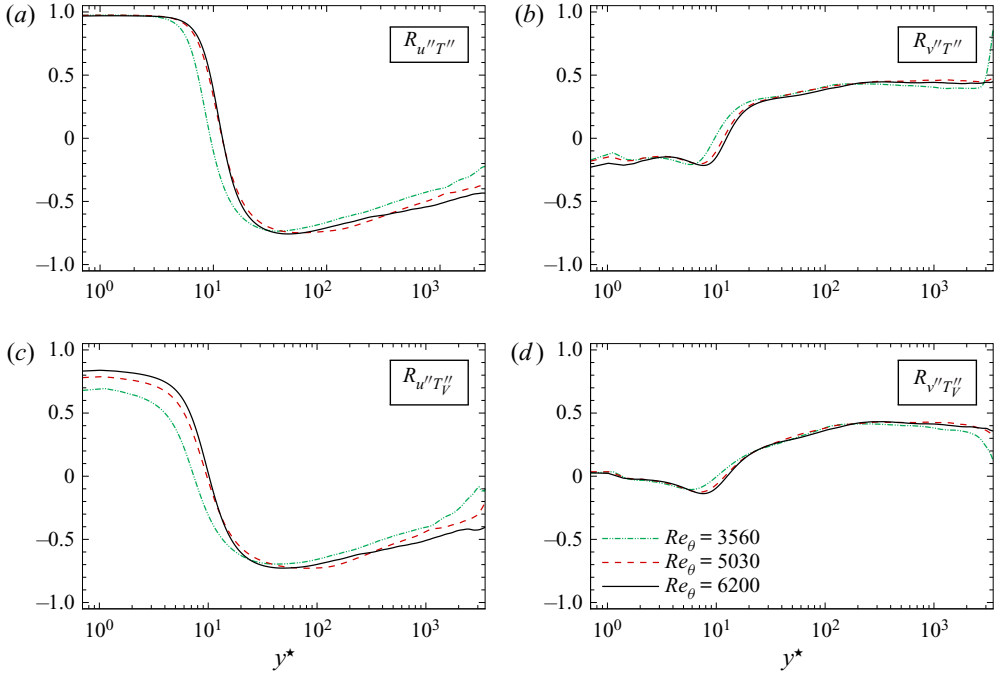


Figure 16. Correlation coefficients between Favre fluctuations of vibrational temperature and streamwise velocity (a) and wall-normal velocity (b) and of translational temperature and streamwise velocity (c) and wall-normal velocity (d).

vibrational energy can be modelled by introducing a vibrational turbulent Prandtl number

$$Pr_t^V = \frac{\overline{\rho u'' v''} \partial \tilde{T}_V / \partial y}{\overline{\rho v'' T_V''} \partial \tilde{u} / \partial y}. \quad (3.10)$$

Here, Pr_t^V (shown in [figure 15b](#)) takes values close to unity in a relatively large region corresponding to $0.2 \lesssim y/\delta \lesssim 0.7$, where turbulent transport dominates, and it diverges at the boundary-layer edge (where $\partial \tilde{u} / \partial y$ tends to vanish) and close to the wall, similarly to Pr_t . We can get further insights by reporting, in [figure 16](#), correlations of u'' and v'' fluctuations with T'' and T_V'' , respectively. As observed in both low- and high-enthalpy boundary layers ([Zhang *et al.* 2018](#); [Di Renzo & Urzay 2021](#)), the logarithmic and outer regions are characterized by a negative correlation between u'' and T'' (a) and positive correlation for v'' and T'' (b). This comes from (i) sweeps ($u'' > 0$, $v'' < 0$) entraining colder ($T'' < 0$) fluid towards the buffer layer, and (ii) ejections ($u'' < 0$, $v'' > 0$) entraining hotter ($T'' > 0$) fluid towards the edge, resulting in $R_{u''T''} < 0$ and $R_{v''T''} > 0$. Below the buffer layer, both correlations change their sign for the same reason, consistently with the trend previously discussed in [figure 13](#). Similar considerations hold for $R_{u''T_V''}$ and $R_{v''T_V''}$. However, while streamwise velocity fluctuations close to the wall are perfectly correlated with the translational temperature ones ($R_{u''T''} \approx 1$), the correlation coefficient with the vibrational temperature (c) is weaker ($R_{u''T_V''}$ is between 0.7 and 0.85). This effect is attributed once again to the larger relaxation time of vibrational modes, T_V needing more time than T for adapting to the surrounding temperature. Together with the evolution of Pr_t and the temperature–velocity fluctuation correlations, the SRA hypothesis

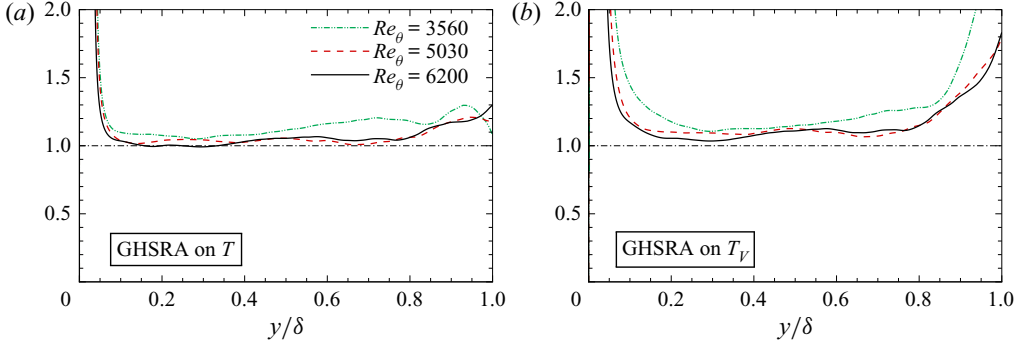


Figure 17. Classical GHSRA expressed in (3.13) (a) and GHSRA based on T_V , reported in (3.14) (b); $-\cdots-$ SRA estimation.

also predicts that

$$\frac{T''/\tilde{T}_{rms}}{(\gamma - 1)M^2(u''/\tilde{u}_{rms})} \approx 1. \quad (3.11)$$

As already observed in the literature, the relation in (3.11) exhibits large deviations from the SRA estimation (not shown) since it fails to take into account the diabatic behaviour of the wall. Several modified forms have been proposed, one of the most successful being that of Huang, Coleman & Bradshaw (1995)

$$\frac{T''/\tilde{T}_{rms}}{(\gamma - 1)M^2(u''/\tilde{u}_{rms})} \approx \frac{1}{Pr_t(1 - (\partial\tilde{T}_t/\partial\tilde{T}))}, \quad (3.12)$$

usually referred to as HSRA, with T_t the stagnation temperature. By removing the assumption of calorically perfect gas (Duan & Martín 2011b), one can write a generalized HSRA (GHSRA)

$$T''_{rms} \approx -\frac{1}{Pr_t} \frac{\partial\tilde{T}}{\partial\tilde{u}} u''_{rms}. \quad (3.13)$$

The wall-normal evolution of the ratio between the right- and the left-hand sides of (3.13) is plotted in figure 17(a), where a satisfactory agreement is observed for most of the boundary-layer thickness (at least in regions where both velocity and temperature fluctuations are not nullified). By using the same procedure followed for Pr_t , the GHSRA can also be extended to the vibrational temperature such that

$$T''_{V,rms} \approx -\frac{1}{Pr_t^V} \frac{\partial\tilde{T}_V}{\partial\tilde{u}} u''_{rms}. \quad (3.14)$$

The results shown in figure 17(b) prove that the GHSRA maintains its validity under thermal non-equilibrium conditions and holds also for vibrational energy.

In addition to turbulent transport, further attention should be paid to the modelling of the nonlinear turbulence–thermal non-equilibrium interactions induced by the source terms of the vibrational energy equation, which are associated with translational–vibrational energy exchanges (Q_{TV}) and chemical dissociation/recombination processes ($\sum_m \dot{\omega}_m e_{vm}$). While thermal non-equilibrium can be reasonably quantified by the magnitude of $\tilde{T} - \tilde{T}_V$, the analysis of the source terms allows one to isolate the physical processes originating

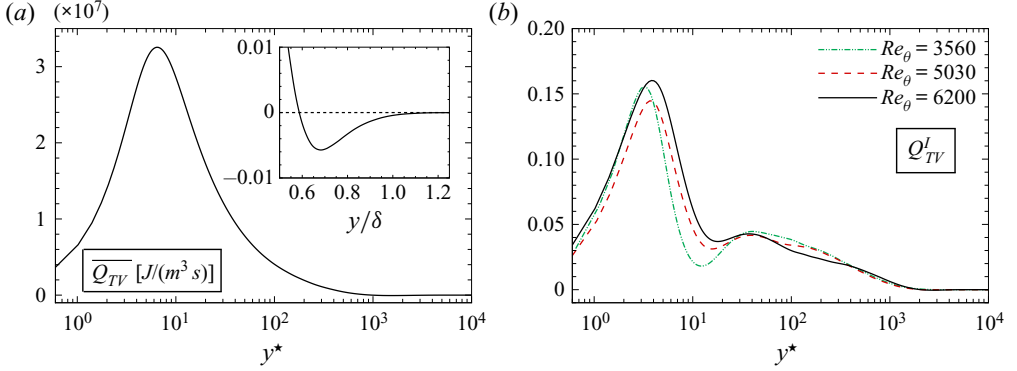


Figure 18. Wall-normal profiles of (a) $\overline{Q_{TV}}$ [$J/(m^3 s)$] and (b) Q_{TV}^I , defined in (3.15). In the inset of (a), a zoom of the profile is shown in outer scale; the dashed line denotes the null value.

such non-equilibrium conditions in the first place. In this specific configuration, chemical activity is extremely weak and therefore $\sum_m \dot{\omega}_m e v_m$ is almost negligible; as a consequence, changes in $\tilde{T} - \tilde{T}_V$ are tightly related to $\overline{Q_{TV}}$ values. Positive values indicate vibrational under-excitation since $T > T_V$ (and the opposite for negative values). The wall-normal profiles of $\overline{Q_{TV}}$, shown in figure 18(a) as a function of y^* and y/δ , closely follow those shown for $(\tilde{T} - \tilde{T}_V)$ in figures 9(a) and 9(b), albeit that the negative values visible in the zoomed view of the outer region are much less pronounced (two orders of magnitude lower) than those registered in the inner zone. Such a large difference is due to the concurrent effects of lower absolute temperature values and (to a lesser extent) smaller $(\tilde{T} - \tilde{T}_V)$ differences, causing the denominator to increase and the numerator to decrease in (2.15), respectively. Although the physical significance of $(\tilde{T} - \tilde{T}_V)$ and $\overline{Q_{TV}}$ is similar in the present case, the interest in analysing the latter quantity stems from a modelling standpoint; that is, only an appropriate modelling of the source term can ensure a correct description of the non-equilibrium state. In this respect, following the technique used by Duan & Martín (2011a) for assessing turbulence/chemistry interactions and with the aim of quantifying the coupling between turbulence and thermal relaxation, we built an indicator defined as

$$Q_{TV}^I = \frac{\overline{Q_{TV}(T, T_V, \rho, p, Y_n)} - Q_{TV}(\tilde{T}, \tilde{T}_V, \tilde{\rho}, \tilde{p}, \tilde{Y}_n)}{\overline{Q_{TV,max}}}, \quad (3.15)$$

where $\overline{Q_{TV,max}}$ denotes the maximum wall-normal value of $\overline{Q_{TV}}$ at the selected station. Deviations from null values represent a measure of vibrational energy variations associated with turbulent fluctuations. Figure 18(b) shows that computing Q_{TV} from averaged quantities would underestimate its value across the whole boundary layer, with discrepancies larger than 15 % of $\overline{Q_{TV,max}}$ in the inner layer. These data further confirm the tight coupling between thermal non-equilibrium and turbulence discussed in the current section and underline that attention must be paid to the development of suitable turbulence closures, especially in the context of RANS modelling.

3.5. Chemical activity

In this section we investigate chemical activity triggered in the high-enthalpy boundary layer by friction heating. The profiles of Favre-averaged mass fractions of species present

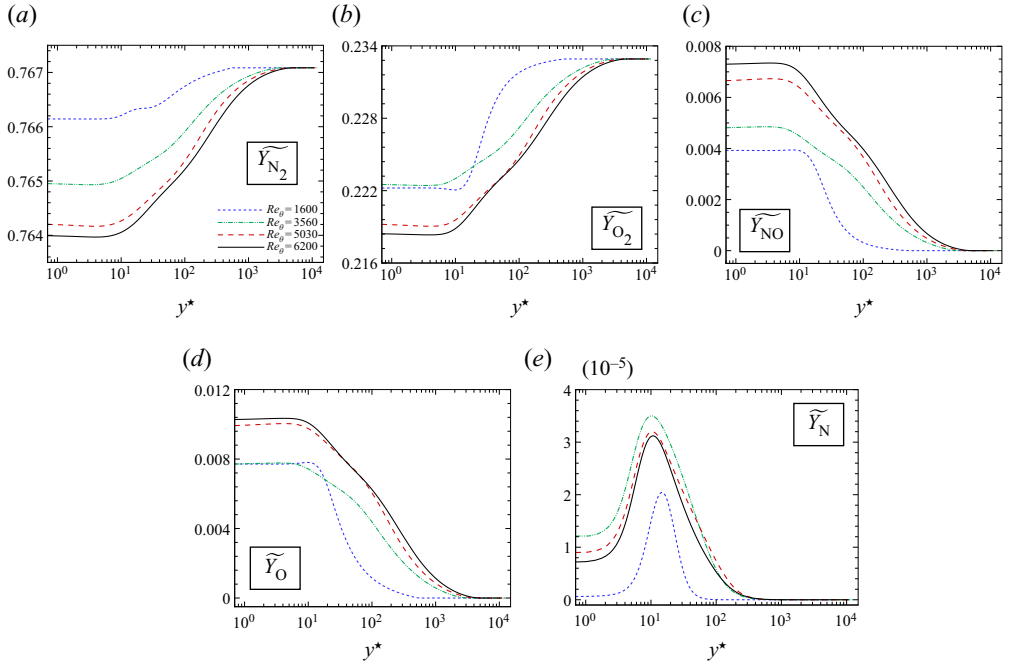


Figure 19. Mean mass fraction of molecular nitrogen (a), molecular oxygen (b), nitric oxide (c), atomic oxygen (d) and atomic nitrogen (e).

in the mixture are reported in [figure 19](#). At the flow conditions under investigation, molecular oxygen is the species which dissociates most, leading to the formation of atomic oxygen. The maximum mass fraction of atomic oxygen in the mixture is approximately 1%. The maximum temperatures in the boundary layer are too low to allow nitrogen dissociation in any significant amount, even at the temperature peak, and consequently the quantity of atomic nitrogen is negligible compared with the other species. Nevertheless, the present conditions are such that a small amount of nitric oxide (approximately 0.7%) is produced by the forward shuffle reaction R4 and by the backward shuffle reaction R5. The rather low chemical activity is partly due to the strong wall cooling, limiting the maximum temperature to values just above the threshold for oxygen dissociation ($T > 2000$ K). Thermal non-equilibrium further limits chemical reactions. Highly vibrationally excited molecules are meant to dissociate more rapidly, since they require less energy exchange during the collisions. On the contrary, in under-excitation conditions, dissociation is less likely to occur. This mechanism is numerically taken into account in the computation of the reaction rate coefficients (for the calculation of which a geometric average of the two temperatures is considered, $T^{0.7}T_V^{0.3}$); they turn out to be lower when strong non-equilibrium effects are present. On the other hand, the endothermic chemical reactions tend to drain roto-translational energy, although this effect is partly compensated by the relaxation of vibrational modes, which supply energy and partly counteract temperature reduction. The chemical Damköhler number based on the LE turnover time $(\delta^*/u_\tau)/(\rho/\dot{\omega}_n)$ is at most of the order of 10^{-2} for all species, denoting little interaction between chemical reactions and turbulent motions and an almost frozen-chemistry behaviour. A detailed discussion can be found in [Passiatore et al. \(2021\)](#) for an adiabatic turbulent chemically out-of-equilibrium boundary

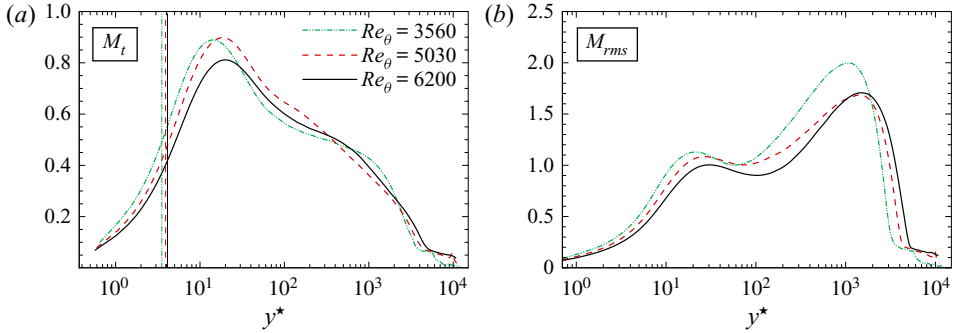


Figure 20. Profiles of turbulent Mach number (a) and r.m.s. Mach number (b).

layer; despite the different conditions, similar qualitative observations can be made in the present work.

3.6. Compressibility effects

The present analysis is completed by discussing the role of compressibility effects and their interaction with thermal non-equilibrium conditions. In figure 20 we report the evolutions of the turbulent and fluctuating Mach numbers, defined as $M_t = u_{rms}/\bar{c}$ and $M_{rms} = (u/c)_{rms}$, respectively. Despite the high external Mach number, M_t is shown to be smaller than one everywhere, with maximum values of approximately 0.8–0.9 registered at the location where the turbulent kinetic energy production peaks. Such values are, however, large enough to enable the formation of eddy shocklets, as also proven by the location of the sonic line (identified by the three vertical lines in figure 20a) within the VS. Accordingly, the fluctuating Mach number takes values larger than one starting from $y^* \gtrsim 10$. The outer M_{rms} peaks at $y^* \in [1000, 2000]$ (whose values are compatible with those obtained by Zhang *et al.* (2018) for high-speed boundary layers of calorically perfect gases) are linked to the large fluctuations of thermodynamic properties resulting from the presence of turbulent/non-turbulent interfaces along the boundary-layer edge. These structures are clearly visible in figure 21, displaying the normalized density gradient magnitude on a streamwise slice of the computational domain. Such large fluctuations of the thermodynamic variables are not accompanied by an intense activity of the dynamic field, as testified by the values of the semi-locally scaled fluctuating velocity divergence $\vartheta'' \bar{\mu}/\bar{\tau}_w$ (with $\vartheta = \partial u_k/\partial x_k$) shown in figure 22 in the last portion of the domain. The stronger dilatation events are registered close to the wall and their magnitude monotonically decreases with the wall distance, as shown in figure 23(a), where results from Xu *et al.* (2021) are reported for comparison. Figure 23(b) displays two p.d.f.s of $\vartheta'' \bar{\mu}/\bar{\tau}_w$ computed at wall-normal positions corresponding to $y^* \approx 10$ and 1000. Both the profiles are slightly skewed toward negative values, highlighting a mild predominance of the occurrence of strong compressions over strong expansions, although such events are much stronger in the inner region. However, joint p.d.f.s of $(\vartheta''(\bar{\mu}/\bar{\tau}_w), |\nabla \rho|/(\rho_e/\delta_{99,end}))$ (not shown) did not exhibit any preferential correlations; this amounts to saying that, even if shocklets might exist inside the boundary layer, their number and/or intensity is not large enough to engender any remarkable effect on thermodynamic and dynamic quantities.

The interplay between compressibility and thermal non-equilibrium effects has previously been investigated in mixing layers (Fiévet *et al.* 2019) and homogeneous isotropic turbulence (Zheng *et al.* 2020) configurations, in which it was observed that

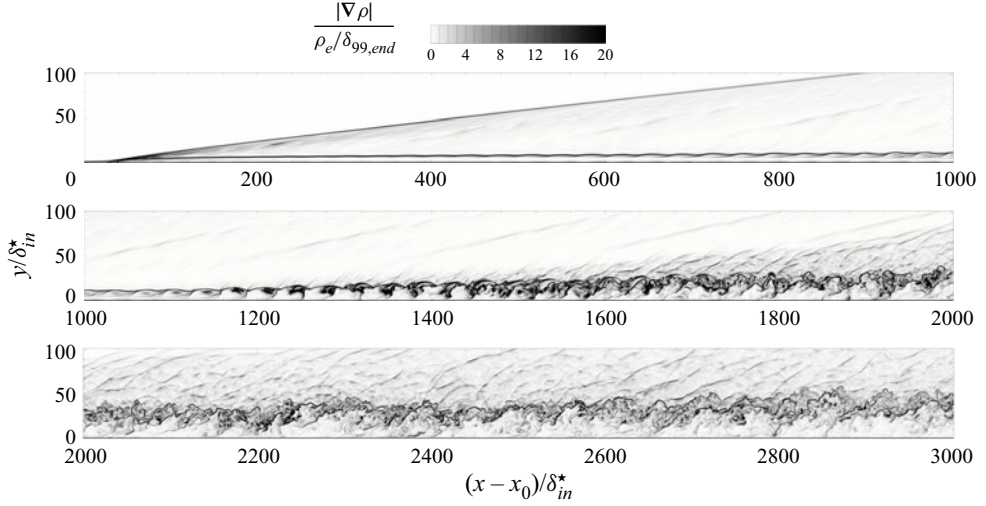


Figure 21. Instantaneous snapshot of the normalized density gradient magnitude along a slice of the computational domain.

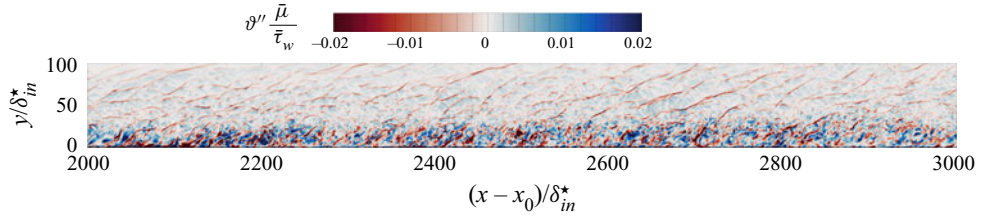


Figure 22. Instantaneous snapshot of the normalized divergence along the last portion of the computational domain.

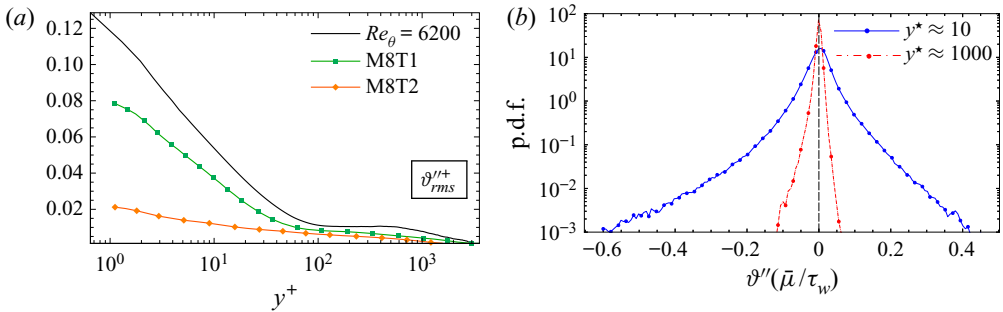


Figure 23. Fluctuating values of velocity divergence normalized in inner-scale units (a) and p.d.f. of normalized divergence at $Re_\theta = 6200$ for $y^* \approx 10$ and $y^* \approx 1000$. M8T1 and M8T2 from Xu *et al.* (2021).

vibrational non-equilibrium is somehow enhanced by strong dilatation motions (whether they be expansions or compressions). Impulsive changes of thermodynamic quantities, as in the case of strong density gradients, can increase the difference between the two temperatures because of their different relaxation times. Here, high- $|\nabla\rho|$ regions are mainly found near the boundary-layer edge where the average temperature values are

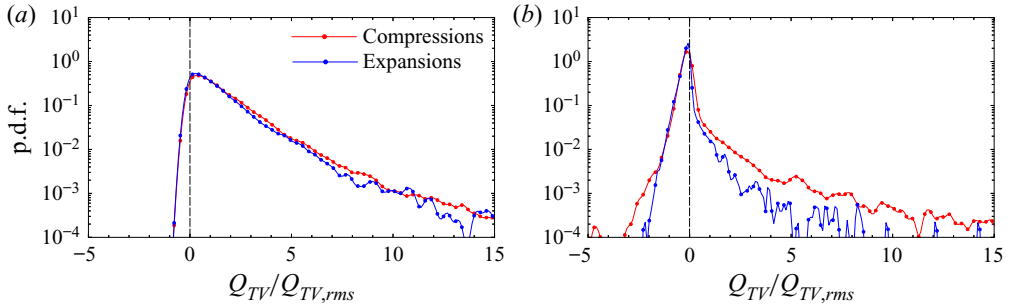


Figure 24. Conditional p.d.f. of normalized translational–vibrational energy exchange at $Re_\theta = 6200$ for $y^* \approx 10$ (a) and $y^* \approx 1000$ (b). The p.d.f.s are conditioned on strong events, defined by $|\vartheta/\vartheta_{rms}| \geq 2$.

relatively low; as a consequence, compressibility/thermal non-equilibrium interactions are expected to be less marked for boundary-layer configurations. For this purpose, we show in [figure 24](#) the p.d.f.s of $Q_{TV}/Q_{TV,rms}$ at the last turbulent station for the same wall-normal positions shown in [figure 23\(b\)](#). The p.d.f.s are conditioned on strong dilatation motions, defined as $|\vartheta/\vartheta_{rms}| \geq 2$. The p.d.f.s in [figures 24\(a\)](#) and [24\(b\)](#) peak at positive (negative) values of $Q_{TV}/Q_{TV,rms}$, coherently with the local mean under-excitation (over-excitation) state depicted in [figure 18](#). All the profiles are strongly right skewed because of the predominance (in magnitude and number) of under-excited states, as explained in [§ 3.3](#). At both locations, strong compressions are associated with slightly higher $Q_{TV}/Q_{TV,rms}$ values than strong expansions; the wider differences between the distributions in panel (b) are caused by the much smaller source term fluctuations observed in the outer region. Globally, the differences are shown to be marginal, validating our previous statement.

4. Conclusions

The present work illustrates the results of the numerical simulation of a hypersonic, high-enthalpy TBL. The selected edge conditions are such that the flow undergoes TCNE conditions. The interaction between high-temperature effects and turbulent quantities is then investigated, with specific insights on the vibrational–relaxation dynamics.

The forcing strip located right after the inlet similarity solution induces an abrupt transition to turbulence, characterized by a steep increment of the wall shear stress in the transitional region and an asymptotic decay in the fully turbulent portion. The skin friction decomposition of Renard & Deck (2016) shows that the mean field and turbulent terms contribute equally to the C_f value, whereas the term related to the boundary-layer growth is non-negligible only in the region of breakdown to turbulence. With regard to the wall heat flux, it is observed that its roto-translational contribution follows the same trend of C_f , whereas the vibrational counterpart is nearly constant across the turbulent region, with values being one order of magnitude smaller. The van Driest and Trettel & Larsson transformations perform poorly in collapsing the streamwise velocity profiles onto the universal logarithmic law. On the contrary, the Trettel & Larsson transformation and the new total-stress-based transformation of Griffin *et al.* (2021) provide a perfect match with the linear law, the latter improving the collapse in the logarithmic region as well. Furthermore, the relation between the mean velocity and mean temperature profiles derived by Walz (1969) resulted to be inappropriate for calorically imperfect gases; an improvement was found with the analogous enthalpy-based equation proposed by Duan & Martín (2011b), despite the presence of thermochemical out-of-equilibrium effects.

With the aim of quantifying the thermal non-equilibrium, three vibrational Damköhler numbers are defined based on different characteristic flow time scales. The results illustrate that, aside from the near-wall region where the flow is vibrationally frozen, strong thermal non-equilibrium conditions persist up to the edge of the boundary layer. Molecular nitrogen is responsible for thermal non-equilibrium, since its characteristic temperature is high enough to cause slow relaxation rates. The vibrational excitation is largely sustained by turbulent mixing, as observed by the tight coupling with sweep and ejection events inside the boundary layer. Sweeps convey colder air from the outer regions towards the flat-plate wall, whereas ejections bring hotter air far from the inner layer. The translational modes rapidly adapt to such large temperature fluctuations, contrary to the vibrational modes which re-adjust slowly. At the macroscopic level, the direct consequence is that vibrational temperature lags behind the translational one up to the first half of the logarithmic layer; from there onwards, the flow transitions towards a vibrationally over-excited state which is kept until the edge of the boundary layer. Turbulence/thermal relaxation interactions, quantified by means of a parameter measuring the vibrational energy variations due to turbulent fluctuations, are shown to be substantial across the buffer layer, confirming that turbulent transport plays a major role. Following usual RANS modelling strategies, a vibrational turbulent Prandtl number Pr_t^V can be defined to close the turbulent vibrational energy flux and it assumes values close to unity for most of the boundary layer. Similarly to the classical Pr_t , Pr_t^V deviates from unity in the inner region and at the boundary-layer edge, where $\partial\tilde{u}/\partial y$ vanishes. The remaining SRA relations exhibit an acceptable agreement with the estimation and the GHSRA was found to work reasonably well even when extended to vibrational temperature.

Second-order statistics show classical self-similar trends for the Reynolds stresses, whose values are similar to those observed at low-enthalpy conditions for comparable values of the free-stream Mach numbers. The isothermal wall condition leads to a double-peak-shaped profile for the temperature fluctuations, the outer maximum being predominant. On the contrary, vibrational temperature fluctuations exhibit monotonously increasing profiles due to the marked non-equilibrium conditions existing in the buffer layer.

Gas dissociation at the wall is almost negligible since the imposed wall temperatures ($T = T_V = 1800$ K) are not high enough to trigger significant chemical activity. Little interaction is observed between chemical reactions and turbulent motions, the Damköhler numbers being at most of the order of 10^{-2} for all species. Statistical analyses on compressibility effects, measured by means of the semi-locally scaled velocity divergence fluctuations, have shown that strong dilatation motions are mainly found close to the wall, with strong compressions slightly more probable than strong expansions. With the aim of relating compressibility effects and thermal non-equilibrium, p.d.f.s of the translational–vibrational energy exchange source term Q_{TV} conditioned on the local divergence values are computed. It is observed that strong compressions are associated with slightly higher values of Q_{TV} , as observed in previous studies about thermally out-of-equilibrium mixing layers and isotropic turbulence configurations. However, this correlation is shown to be much weaker for boundary-layer configurations, since the strongest density and temperature gradients are registered in the outer portion of the boundary layer where the flow exhibits a vibrationally frozen behaviour due to the cold temperature.

The present DNS represents a first contribution towards the knowledge of wall-bounded turbulence in presence of thermochemical effects. Many aspects require future research efforts. For instance, the behaviour of the wall in terms of participation to the chemical

	A_n	B_n	C_n
N ₂	0.0268142	0.3177838	-11.3155513
O ₂	0.0449290	-0.0826158	-9.2019475
NO	0.0436378	-0.0335511	-9.5767430
O	0.0203144	0.4294404	-11.6021403
N	0.0115572	0.6031679	-12.4327495

Table 2. Coefficients to compute pure species viscosity from Blottner *et al.* (1971).

	A_2	A_3	A_4
N ₂ – N ₂	0.0112	1.6182	-11.3091
N ₂ – O ₂	0.0465	0.9271	-8.1137
N ₂ –NO	0.0291	1.2676	-9.6878
N ₂ –O	0.0140	1.5824	-10.8891
N ₂ – N	0.0195	1.4880	-10.3654
O ₂ – O ₂	0.0410	1.0023	-8.3597
O ₂ –NO	0.0438	0.9647	-8.2380
O ₂ – O	0.0226	1.3700	-9.6631
O ₂ – N	0.0169	1.4848	-10.2810
NO – NO	0.0364	1.1176	-8.9695
NO – O	0.0179	1.4848	-10.3155
NO – N	0.0185	1.4882	-10.3301
N – N	0.0033	1.5572	-11.1616
N – O	-0.0048	1.9195	-11.9261
O – O	0.0034	1.5572	-11.1729

Table 3. Constants for diffusion coefficient curve fits from Gupta *et al.* (1990). Of note, coefficient A_1 is identically zero.

processes is of the utmost importance: the hypothesis of non-catalytic surface should not be taken for granted, since the materials adopted for the thermal protection systems become catalytic above certain degrees of dissociation and temperature values. At the same time, ablation causes defects on the surface that could induce different transition-to-turbulence scenarios, to be considered in addition to the effect of wall roughness. Moreover, the different combination of operating conditions, such as Mach number, stagnation enthalpy and altitude, may trigger one phenomenon more intensely than another. Turbulent regimes are more likely to occur at lower altitudes, whereas thermal-relaxation phenomena would be predominant higher in the atmosphere. At the same time, higher densities and kinetic energy contents promote molecular collisions, which may result in more intense chemical activity. Spanning a range of control parameters as wide as possible would enable further insights on high-enthalpy hypersonic flows.

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Declaration of interests. The authors report no conflict of interest.

Reaction		C	a	T_{act}	A_1	A_2	A_3	A_4	A_5
$N_2 + N_2 \rightleftharpoons 2N + N_2$	\rightleftharpoons	7.0×10^{21}	-1.6	113 200	1.4766	1.6291	1.2153	-11.4570	-0.009444
$N_2 + O_2 \rightleftharpoons 2N + O_2$	\rightleftharpoons	7.0×10^{21}	-1.6	113 200	1.4766	1.6291	1.2153	-11.4570	-0.009444
$N_2 + NO \rightleftharpoons 2N + NO$	\rightleftharpoons	7.0×10^{21}	-1.6	113 200	1.4766	1.6291	1.2153	-11.4570	-0.009444
$N_2 + O \rightleftharpoons 2N + O$	\rightleftharpoons	3.0×10^{22}	-1.6	113 200	1.4766	1.6291	1.2153	-11.4570	-0.009444
$N_2 + N \rightleftharpoons 2N + N$	\rightleftharpoons	3.0×10^{22}	-1.6	113 200	1.4766	1.6291	1.2153	-11.4570	-0.009444
$O_2 + N_2 \rightleftharpoons 2O + N_2$	\rightleftharpoons	2.0×10^{21}	-1.5	59 500	0.50989	2.4773	1.7132	-6.5441	0.029591
$O_2 + O_2 \rightleftharpoons 2O + O_2$	\rightleftharpoons	2.0×10^{21}	-1.5	59 500	0.50989	2.4773	1.7132	-6.5441	0.029591
$O_2 + NO \rightleftharpoons 2O + NO$	\rightleftharpoons	2.0×10^{21}	-1.5	59 500	0.50989	2.4773	1.7132	-6.5441	0.029591
$O_2 + N \rightleftharpoons 2O + N$	\rightleftharpoons	1.0×10^{22}	-1.5	59 500	0.50989	2.4773	1.7132	-6.5441	0.029591
$O_2 + N_2 \rightleftharpoons N + O + N_2$	\rightleftharpoons	5.0×10^{15}	0	75 500	0.50765	0.73575	0.48042	-7.4979	-0.016247
$NO + O_2 \rightleftharpoons N + O + O_2$	\rightleftharpoons	5.0×10^{15}	0	75 500	0.50765	0.73575	0.48042	-7.4979	-0.016247
$NO + NO \rightleftharpoons N + O + NO$	\rightleftharpoons	5.0×10^{15}	0	75 500	0.50765	0.73575	0.48042	-7.4979	-0.016247
$NO + O \rightleftharpoons N + O + O$	\rightleftharpoons	1.0×10^{17}	0	75 500	0.50765	0.73575	0.48042	-7.4979	-0.016247
$NO + N \rightleftharpoons N + O + N$	\rightleftharpoons	1.0×10^{17}	0	75 500	0.50765	0.73575	0.48042	-7.4979	-0.016247
$N_2 + O \rightleftharpoons NO + N$	\rightleftharpoons	6.4×10^{17}	-1.0	38 400	0.96921	0.89329	0.73531	-3.9596	0.006818
$NO + O \rightleftharpoons N + O_2$	\rightleftharpoons	8.4×10^{12}	0	19 400	-0.002428	-1.7415	-1.2331	-0.95365	-0.04585

Table 4. Parameters needed for the five-species chemical model of Park (1990). Pre-exponential C, exponent a, activation temperature T_{act} and coefficients A_i for computing the equilibrium constant K_{eq} .

	M	<i>a</i>	<i>b</i>
N ₂ – M	N ₂	221	0.0290
	O ₂	229	0.0295
	NO	225	0.0293
	O	72.4	0.0150
	N	180	0.0262
O ₂ – M	N ₂	134	0.0295
	O ₂	138	0.0300
	NO	136	0.0298
	O	47.7	0.0590
	N	72.4	0.0150
NO-M	N ₂	49.5	0.0420
	O ₂	49.5	0.0420
	NO	49.5	0.0420
	O	49.5	0.0420
	N	49.5	0.0420

Table 5. Coefficients for computing the relaxation times t_m from Park (1993).

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Appendix A. Thermochemical model and transport properties

Pure species' viscosity and thermal conductivities are computed using curve fits by Blottner *et al.* (1971) and Eucken's relations (Hirschfelder & Curtiss 1969)

$$\mu_n = 0.1 \exp[(A_n \ln T + B_n) \ln T + C_n] \quad (\text{A1})$$

$$\lambda_{TR,n} = \mu_n \left(\frac{5}{2} c_{v,n}^T + c_{v,n}^R \right), \quad \lambda_{V,n} = \mu_n c_{v,n}^V, \quad (\text{A2a,b})$$

where A_n , B_n and C_n are fitted parameters reported in table 2. The corresponding mixture properties are evaluated by means of the mixing rules of Wilke (1950)

$$f = \frac{\sum_{n=1}^{NS} X_n f_n}{\sum_{m=1}^{NS} X_m \phi_{nm}}, \quad (\text{A3})$$

with f being either μ , λ_{TR} or λ_V , and

$$\phi_{nm} = \frac{1}{\sqrt{8}} \left(1 + \frac{\mathcal{M}_n}{\mathcal{M}_m} \right)^{-(1/2)} \left[1 + \left(\frac{\mu_n}{\mu_m} \right)^{-(1/2)} \left(\frac{\mathcal{M}_m}{\mathcal{M}_n} \right)^{1/4} \right]^2. \quad (\text{A4})$$

As for mass diffusion processes, the equivalent coefficient D_n of (2.12) is computed as in Hirschfelder & Curtiss (1969)

$$D_n = \frac{1 - Y_n}{\sum_{\substack{m=1 \\ m \neq n}}^{NS} \frac{X_m}{D_{mn}}}, \quad \text{with } D_{mn} = \frac{1}{p} \exp(A_{4,mn}) T^{[A_{1,mn}(\ln T)^2 + A_{2,mn} \ln T + A_{3,mn}]}, \quad (\text{A5})$$

where D_{mn} is the binary diffusion coefficient of species m into species n , and $A_{1,mn}, \dots, A_{4,mn}$ are curve-fitted coefficients computed as in Gupta *et al.* (1990); the values are reported in table 3.

Concerning the chemical model of Park (1990), the forward and backward reaction rates $k_{f,r}$ and $k_{b,r}$ shown in (2.14) are modelled by means of Arrhenius' law

$$k_{f,r} = C_r T^{a_r} \exp\left(-\frac{T_{act,r}}{T_{avg}}\right) \quad \text{and} \quad k_{b,r} = \frac{k_{f,r}}{K_{eq,r}}. \quad (\text{A6a,b})$$

Here, $T_{act,r}$ is the activation temperature of reaction r , whose values are reported in table 4, together with the values of the pre-exponential parameter C_r and of the exponent a_r . In (A6a,b), $K_{eq,r}$ is the equilibrium constant of the r th reaction, for which Park employed a curve fit of the form

$$K_{eq,r} = \exp\left(A_{1r} + A_{2r}Z + A_{3r}Z^2 + A_{4r}Z^3 + A_{5r}Z^4\right), \quad (\text{A7})$$

with $Z = 10\,000/T_{avg}$, and the constants $A_{\bullet,r}$ given in table 4. Lastly, the coefficients a_{mn} and b_{mn} , used to compute the relaxation time in (2.16), are reported in table 5.

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