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1 Equivalent mechanical load model methodology to simulate lightning strike impact
2 on protected and painted composite structure

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10 **Keywords:** lightning; impact; carbon fibre reinforced plastics; numerical simulations; paint

11

12 **Abstract**

13 Lightning strike damage on aircraft composite structure is very difficult to predict. The main
14 source of this damage is the mechanical constraint generated by the lightning in interaction
15 with the structure made of composite, protected with a metallic mesh and painted. The
16 purpose of this paper is to propose a numerical model able to reproduce the surface
17 overpressure generated by the explosion of the lightning strike protection (LSP) and its
18 confinement with the paint. Comparisons of numerical displacements with lightning
19 experimental ones are presented in order to analyse the predictability of the model. This is a
20 first step in order to assess the damage in the composite structure depending on the design
21 configurations (CFRP, LSP and Paint) when combined with a damage model.

22

23 **1. Introduction and context**

24 In the last 40 years, the use of composites has significantly increased in the aircraft industry,
25 as carbon fibre reinforced composites present a higher strength and stiffness to mass ratio
26 compared to metallic structures. Despite their excellent mechanical properties, composites
27 are poor conductors, making them more susceptible to serious damage due to a lightning
28 strike such as severe delamination, burning and possibly puncture [1]. In order to protect the
29 composite structures, a lightning strike protection layer, usually a metallic mesh, is applied
30 on the external surface.

31 It is extremely difficult to predict the damage incurred by the composite structure since
32 different types of damage occur: fibre tufting and resin decomposition on one hand, and
33 delamination and matrix cracking on the other hand. This is indeed a highly dynamic and
34 multi-physics phenomenon. There is no fully validated theory on all the physics phenomena
35 involved by the arc and its interaction with the structure or the associated chronology. Still, a
36 proposal of the different constraints involved has been developed [2] and illustrated in Figure
37 1. On one side, there are thermal forces with the thermal flux at the arc root, the thermal
38 radiation from the arc channel and the joule heating from the current flow. On the other
39 side, there are mechanical forces with Laplace forces and shock wave from the arc
40 generation.

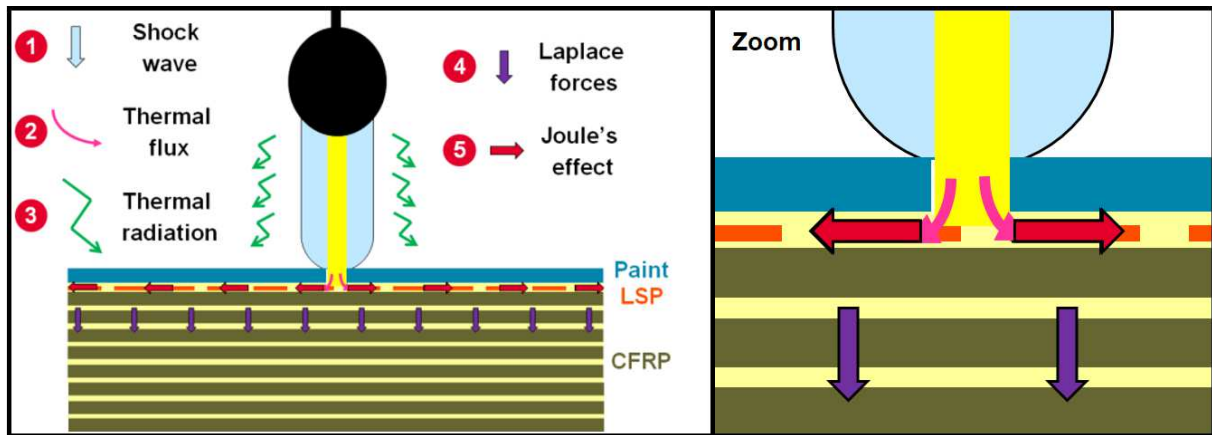
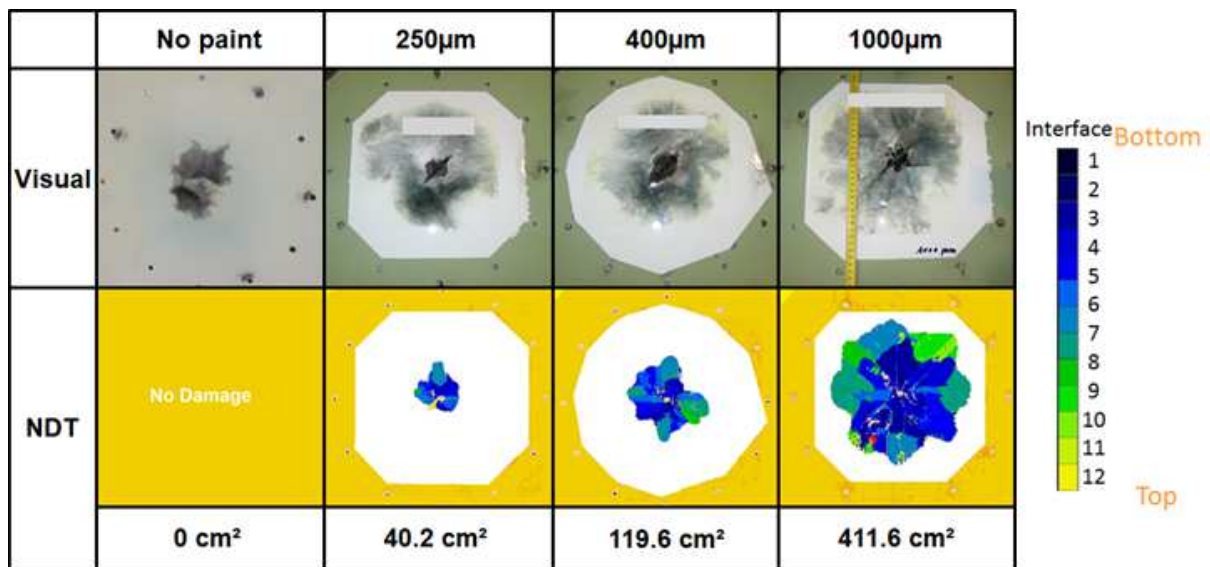


Figure 1 Lightning constraints generation in composite structures

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Work performed in the past to simulate damage induced by a lightning strike, which can be found in the literature, was first based on a simple electro-thermal approach in which the current flow in the composite generates heat that will create damage [3–7]. These models were subsequently refined by considering the degradation of the resin by pyrolysis [3,6–12]. In many cases, the arc was modelled as an equivalent current source with a fixed injection area. It has been demonstrated, however, that the arc root interacts with the structure and that its injection is variable in time and space [13–15]. Finally, this approach showed its limitations since it proved unable to predict the underlying delamination coming from a mechanical constraint. In order to assess delamination, which is the most detrimental effect for the sustainability of the structure, a model of the mechanical sources generated by the arc, i.e. the shock wave and the magnetic pressure, was introduced [16–23]. However, the theory is not fully validated yet and the magnitude and distribution of these forces are highly dependent on the arc root interaction. This sensitivity makes it very difficult to weight these two contributors correctly. Even if a more accurate model could be developed, this would not suffice to predict the damage as the values of these forces that were determined were not high enough to generate significant stress in the composite [21,23].

59 This led some researchers to investigate the explosion effect of the composite but also the
 60 role of the metallic protection on the surface of the structure as the sudden and high
 61 lightning current flowing in the conductor leads to a rapid increase in temperature up to an
 62 explosion [16,24,25]. Finally, it is not enough to consider the different forces that can be
 63 generated by a lightning strike; it is also necessary to take into account the total system that
 64 will interact with these forces. Of course, the composite structure where damage prediction
 65 is essential is part of it. But in addition, the lightning strike protection and the paint must be
 66 introduced in the model [26]. Indeed, lightning tests on protected composite structure with
 67 different paint thickness highlighted its significant influence on the damage generation in
 68 composite structure. In Figure 2 below, the visual damage and the delamination, measured
 69 by Non Destructive Test (NDT), are compared with this increasing parameter:



70
 71 **Figure 2 Lightning damage evolution in CFRP panel (13 plies) protected with ECF195 with different paint thickness**
 72 This influence is mainly due to the confinement effect of the lightning strike protection (LSP)
 73 explosion by the paint that will enhance the overpressure.

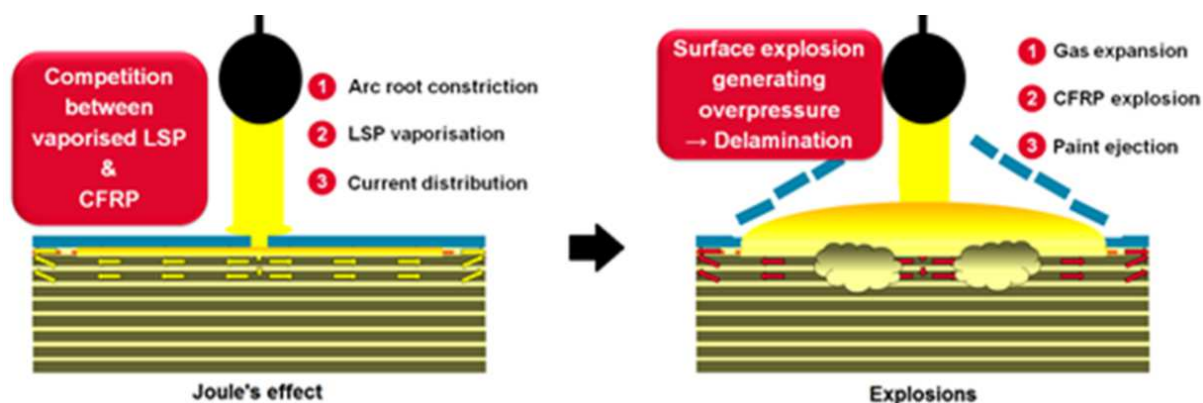


Figure 3 Lightning surface explosion

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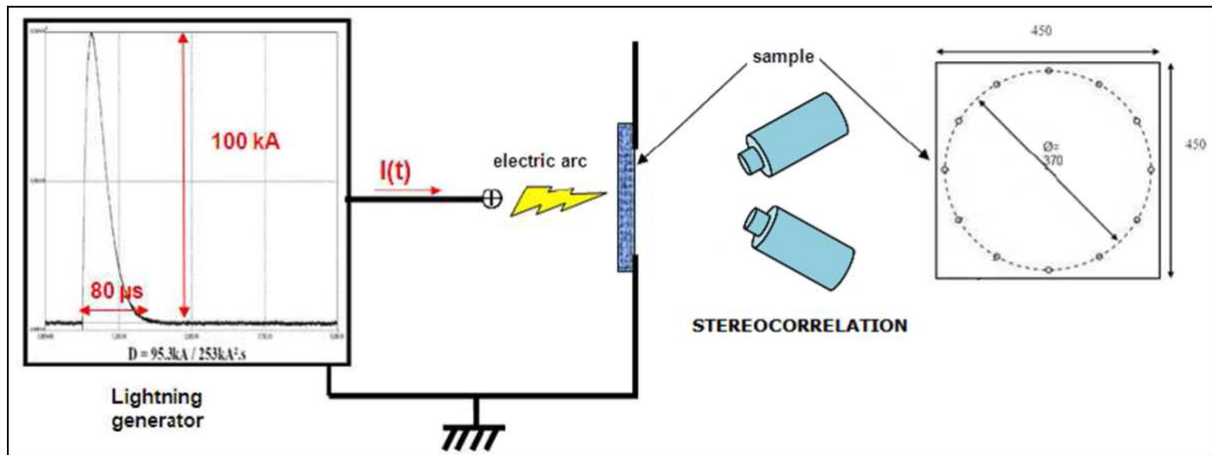
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76 In this paper, we will present a methodology for the construction of the equivalent
 77 mechanical load due to lightning strike on a complete structure, considering the CFRP, the
 78 LSP and the paint.

79 2. Basis of the work

80 2.1. Lightning experimental results

81 In order to study the different parameters that play a role in the generation of lightning
 82 damage, specific lightning tests were performed during the EDIFISS project. All the samples
 83 were flat square panels of 450mm lateral dimension, mechanically and electrically perfectly
 84 bonded to a circular metallic frame through 12 fasteners distributed along its 370mm
 85 diameter. A lightning strike of 100kA following waveform D [27] is performed in the centre
 86 of the panel on the front face (side where the LSP is applied). The rear face displacement is
 87 measured thanks to the image correlation technique used by DGA-TA [28]. Two high speed
 88 cameras Photron SA5 are installed at specific angles behind the panel which has been
 89 painted with a black and white pattern calibrated through a focus frame. Pictures are taken
 90 at a rate of 262,500 frames per second (fps) with a resolution of 128x128 pixels, covering a
 91 central area of the sample of 80x80 mm². Figure 4 illustrates the test setup:



92

93 **Figure 4 Lightning test setup**

94 In a first step, the surface explosion due to Joule heating in LSP layer was studied

95 independently from the CFRP internal damage thanks to specific tests based on the GFRP

96 substrate. Indeed, due to its high resistivity, the lightning current will not be diverted into

97 the glass fibre composite panel and will only flow in the metallic LSP. Therefore, the

98 displacement of the panel is due only to the explosion occurring at the surface. Another

99 interest of this configuration is that no damage is created in the laminate, therefore an

100 elastic model of the panel without a damage law is sufficient to simulate its mechanical

101 behaviour.

102 In a second step, CFRP substrates were tested. At the surface, the confinement effect was

103 studied through the influence of paint thickness on the damage severity. The paint

104 thicknesses considered were 50 μm , 250 μm , 400 μm , and 1000 μm). In the core, the influence

105 of the global panel stiffness on the final extent of damage was studied thanks to a thicker

106 panel. The LSP was in all cases ECF195 from Dexmet[®].

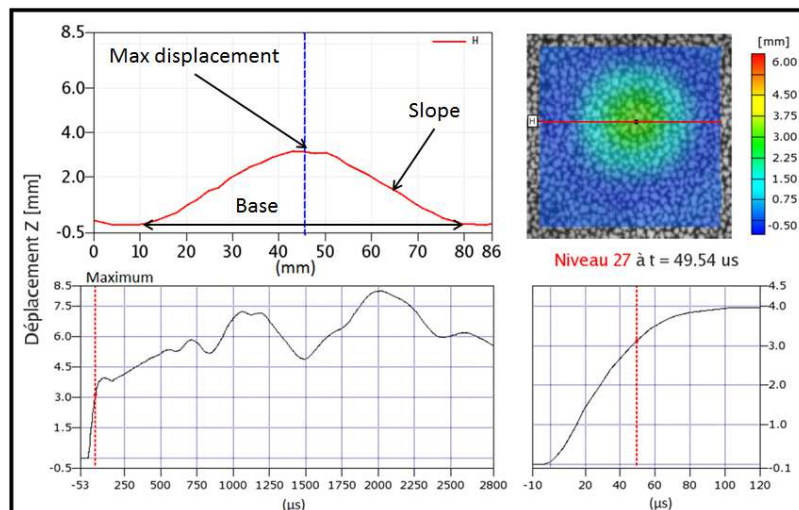
107 A summary of the user cases is listed in Table 1 below:

| Rationale | Substrate | Paint thickness (μm) |
|-------------------------------------|--------------------------------|-----------------------------------|
| Confined surface explosion baseline | GFRP 2,75mm / 11 plies | 400 |
| Free surface explosion baseline | GFRP 2,75mm / 11 plies | 0 |
| Baseline CFRP structure | CFRP 1,651mm / 13 plies | 400 |
| Extreme surface confinement | CFRP 1,651mm / 13 plies | 1000 |
| Light surface confinement | CFRP 1,651mm / 13 plies | 250 |

| | | |
|-----------------------------------|--------------------------------|-----|
| Stiffness influence | CFRP 3,302mm / 26 plies | 400 |
| Stacking and CFRP grade influence | CFRP 1,651mm / 9 plies | 400 |

108 **Table 1 Summary of user cases**

109 The profile of the out-of-plane displacement along an 80 mm long line along the ($x=0^\circ$)
110 direction centred on the rear face central point is reported as a function of time. From the
111 curves, several types of information can be extracted such as the maximum displacement
112 and its rate at the centre versus time, or the slope and base of displacement along ($x=0^\circ$)
113 versus time. The maximum of displacement is mainly reached before $100\mu s$. The comparison
114 between the model and the test will be performed up to this moment. The principle is
115 illustrated in Figure 5 :



116
117 **Figure 5 Rear face out of plane displacement measurement from image correlation**

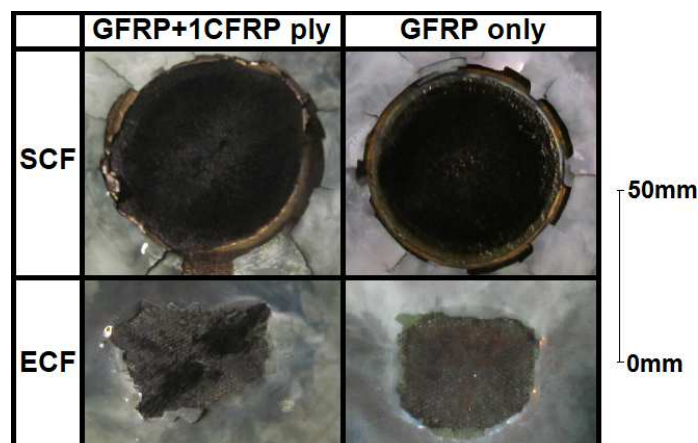
118
119 **2.2. The surface mechanical load hypothesis**

120 The model proposed here considers only the contributions on the surface of the composite.
121 These contributors are the explosion of the lightning strike protection, the confinement from
122 the paint and possibly the lightning arc shock wave. The main hypothesis taken here is that
123 the thermal damage in the composite structure due to the lightning current flow in the
124 fibres is negligible. This thermal source could have two different influences: the contribution

125 of the explosion of the composite ply in the global overpressure [29] and the modification of
 126 the composite panel mechanical properties.

127 2.2.1. The composite explosion

128 Since we cannot measure the pressure generated by a lightning strike, we measured the
 129 displacement of the back of 11 GFRP ply (2.75mm) panels protected with LSP, with or
 130 without a CFRP ply (127 μ m) in between, thanks to the DIC method, and painted with 400 μ m
 131 of typical aeronautic white paint. The added contribution of the explosion of the underlying
 132 composite ply to the total overpressure generated by the explosion of the metallic
 133 protection was limited: Rear face deflections measured without and with the underlying
 134 CFRP ply on top of the GFRP substrate were very similar. First, for the SCF configuration, the
 135 amplitudes and shapes of the measured displacement profile around the rear face centre at
 136 different instants after the test strike were quasi-identical with or without the CFRP ply. In
 137 this configuration, there was no current flowing into the LSP in opposition to ECF, therefore
 138 the overpressure was due only to explosion of the LSP. The presence of dry fibre is visible
 139 with ECF but not with SCF with 400 μ m of paint as illustrated in Figure 6.



140
 141 **Figure 6 Visual damage observation between ECF and SCF with or without underlying CFRP ply**

142 The deflection is not changed by the presence of the CFRP ply. Indeed, the equivalent
 143 bending modulus from the laminate theory [30] of the GFRP panel with an additional CFRP

144 ply at 45° is very similar to the reference GFRP panel. If we consider, in a simple approach,
 145 an equivalent shell, we can define an equivalent elastic bending modulus E_{fx} in x direction for
 146 the composite laminate:

$$E_{fx} = \frac{12}{h^3} \times \bar{E}I_{11} \quad 1$$

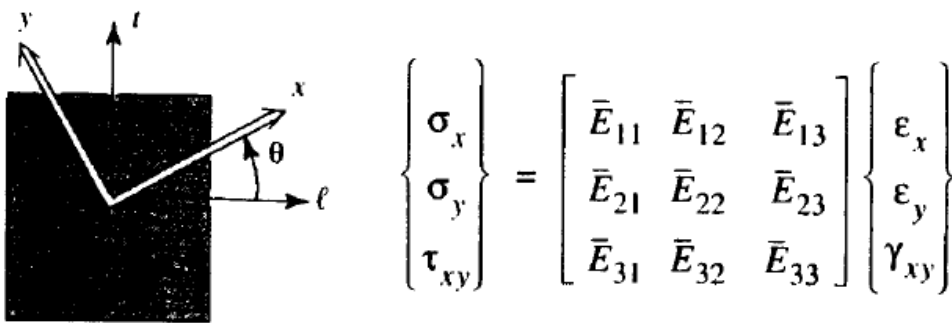
147

148 With h the laminate thickness and $\bar{E}I_{11}$ the laminate equivalent elastic stiffness. From the
 149 laminate theory explicated by Gay [30], this elastic stiffness of a balanced laminate can be
 150 calculated from the bending stiffness factors C_{ij} :

$$\left\{ \begin{array}{l} \bar{E}I_{11} = C_{11} - \frac{C_{12}^2}{C_{22}} \\ C_{ij} = \sum_{k=1^{st}ply}^{n^{th}ply} \bar{E}_{ij}^k \frac{(z_k^3 - z_{k-1}^3)}{3} \end{array} \right. \quad 2$$

151

152 The factor C_{ij} is dependent on the laminate distribution, ply orientation and property \bar{E}_{ij} of
 153 each ply. The location of the ply k in the laminate depth is identified by z_k and z_{k-1} ,
 154 respectively the top and bottom height positions of the ply in the laminate with the origin
 155 located in the medium plane. This stiffness coefficient \bar{E}_{ij} ($i,j=1..3$) is a projected value of the
 156 ply coordinate system (l,t) in the global coordinate system (x,y) of the ply elastic modulus as
 157 presented in Figure 7 below:



158

159 Figure 7 Elastic relationship projection in the global coordinate system [30]

160 For the assessment of the equivalent elastic bending modulus E_{fx} , we need to compute the
 161 factors $(\bar{E}_{11}, \bar{E}_{22}, \bar{E}_{12})$ for each ply orientation θ :

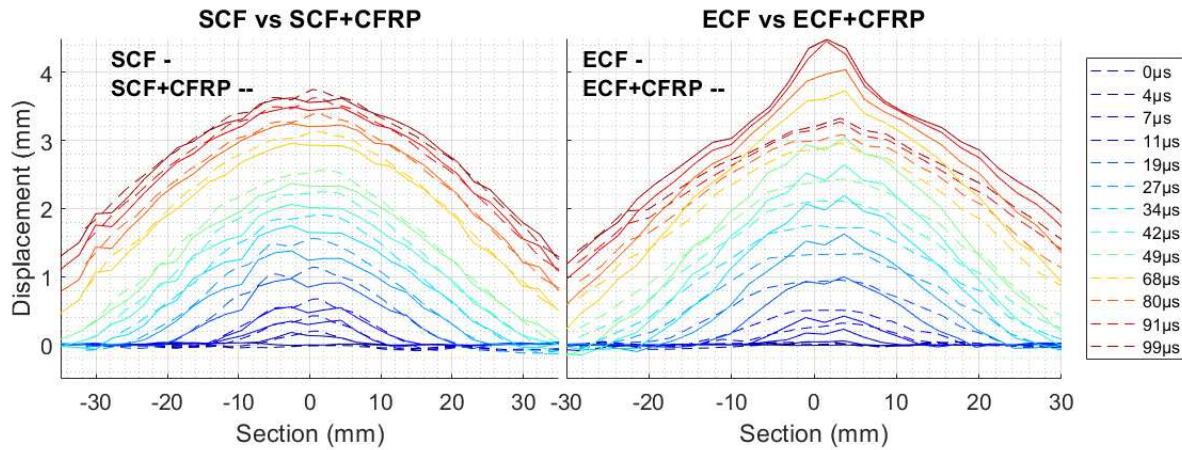
$$\begin{cases} \bar{E}_{11} = c^4 \bar{E}_l + s^4 \bar{E}_t + 2c^2 s^2 (v_{tl} \bar{E}_l + 2G_{lt}) \\ \bar{E}_{22} = s^4 \bar{E}_l + c^4 \bar{E}_t + 2c^2 s^2 (v_{tl} \bar{E}_l + 2G_{lt}) \\ \bar{E}_{12} = c^2 s^2 (\bar{E}_l + \bar{E}_t - 4G_{lt}) + (c^4 + s^4) v_{tl} \bar{E}_l \end{cases} \quad 3$$

162

163 with $\bar{E}_l = E_l / (1 - v_{tl} v_{lt})$, $\bar{E}_t = E_t / (1 - v_{tl} v_{lt})$ and $c = \cos \theta$, $s = \sin \theta$.

164 If we neglect that the laminate is not fully balanced with the addition of the CFRP ply on the
 165 GFRP substrate, we can calculate the bending modulus with the same approach. The GFRP
 166 plate presented a bending modulus of 24GPa and the addition of the CFRP ply change the
 167 modulus to 27GPa which is only 12% higher. The assessment of the laminate behaviour
 168 based on the flexion is limited but it provides a first framework of comparison to study the
 169 influence of such a configuration. A similar approach was used by Soulas [18] considering an
 170 equivalent bending stiffness factor, as the mean value of C_{11} and C_{22} , in order to compute an
 171 equivalent impulse.

172 For the ECF configuration, the influence of the presence of a CFRP layer changes the
 173 maximum amplitude of deflection slightly but also the span. The maximum deflection is
 174 lower, possibly because part of the current has been diverted to the CFRP which decreases
 175 the explosion pressure of the LSP in this area. This is illustrated in Figure 8 below on the right
 176 hand side.



177

178 **Figure 8 CFRP explosion influence on total deflection (Bump due to digital image correlation defect during test)**

179 With these test results, we can generally, in a first stage, neglect the contribution of the
 180 CFRP explosion in the overpressure generation that will contribute to the mechanical
 181 damage. In addition, we neglect the internal damage of the CFRP due to Joule heating.

182 **2.2.2. The thermal damage**

183 The current flow in the composite generates Joule heating which can break the composite
 184 fibres and deteriorate the resin. This damage occurs at the same time as the mechanical load
 185 and could impinge the mechanical resistance of the panel to the impact. A thick paint
 186 applied on a composite structure constrains the arc expansion, forcing the lightning current
 187 to flow deeper in the composite than with a thinner coating. In Figure 9 below, the plies
 188 damaged by the current flow are visible thanks to a microcut done in the centre. The first ply
 189 in this section is destroyed and the second ply is highly damaged. Even with this very severe
 190 configuration where the arc root is highly constrained, the deepest ply burned is the fourth
 191 one among the thirteen plies of the panel with a total thickness of 1.65 mm.



192

193 **Figure 9 Internal damage study for severe configuration: CFRP (13plies) + ECF195 + 1000 μ m of paint**

194 This configuration is considered severe since 1000 μ m of paint is not expected on an aircraft,
 195 where the usual thickness is between 250 μ m and 400 μ m. For this nominal configuration,
 196 only the first ply is burned. Due to the high electrical resistance of the composite, the
 197 current penetrates locally in the depth of the laminate but soon after flows back to the
 198 surface metallic protection. The thermal damage is therefore highly localised even for a
 199 severe configuration. Then, the thermal expansion is also supposed to be limited to these
 200 affected plies which are already damaged by Joule heating and its extension to the
 201 surrounding plies will be limited due to the slow process of heat transfer. It is therefore not
 202 expected to significantly change the mechanical behaviour of the plate.

203

204 **2.3. The mechanical model**

205 Due to the circular clamping mechanical boundary condition, only the disk in this area will be
 206 modelled and its boundary will be “encastre” in the Abaqus® input file
 207 ($U_1=U_2=U_3=UR_1=UR_2=UR_3=0$). In Abaqus/Explicit®, a single layer of shell finite elements (S3
 208 and S4R) with elastic material properties, as defined in Table 6 5, and a user defined load
 209 (VDLOAD) on the top surface were implemented. The model proposed here considers only
 210 the contributions on the surface of the composite. These contributors are the explosion of

211 the lightning strike protection, the confinement from the paint and possibly the lightning arc
 212 shock wave.

| | Density | E ₁₁ | E ₂₂ | G ₁₂ | G ₁₃ | G ₂₃ | V ₁₂ | Ply thickness | Ply number | Stacking |
|-------------|-------------------|-----------------|-----------------|-----------------|-----------------|-----------------|-----------------|---------------|------------|--|
| GFRP | 1.88 | 24 | 24 | 4.8 | 4.8 | 4.5 | 0.28 | 0.25 | 11 | 0° for all plies |
| CFRP | 1.58 | 165 | 8.5 | 4.2 | 4.2 | 3.36 | 0.35 | 0.127 | 13 | 45/-45/90/45/-45/0/90/0/-45/45/90/-45/45 |
| Unit | g/cm ³ | GPa | GPa | GPa | GPa | GPa | N/A | mm | N/A | N/A |

213 **Table 2 GFRP material properties**

214 A composite layup is defined in the shell property to simulate all the plies in the unique shell
 215 layer. Due to the quick spatio-temporal expansion of the pressure applied with the VDLOAD,
 216 the mesh is refined to 0.5mm and ordered in a square of 150mm on the centre of the disk.

217 The mesh details are given below:

| No. of elements | Element type | No. of nodes | DOF |
|-----------------|--------------|--------------|---------|
| 64,090 | S4R | 155,026 | 930,156 |
| 181,468 | S3 | | |

218 **Table 3 GFRP shell mesh details**

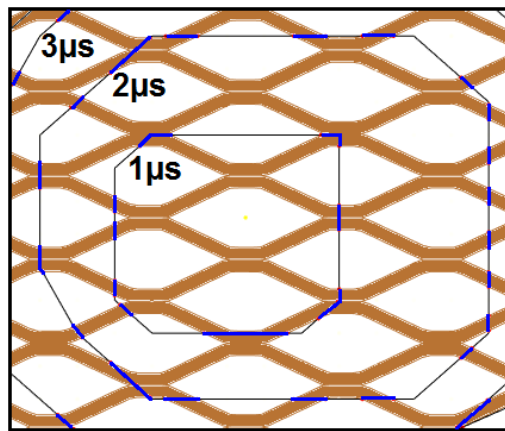
219 **3. Lightning equivalent load methodology**

220 Lightning strike on a composite structure generates high overpressure due to the explosion
 221 of the lightning strike protection on its surface, enhanced by the confinement due to the
 222 presence of paint. The profile of this overpressure is not constant in time and space but
 223 variable and dependant on the lightning protection and the paint. This paragraph presents
 224 the methodology to build the surface overpressure profile equivalent to a lightning strike.
 225 Because the pressure load on the top face of the composite surface is not constant in time
 226 and space, we need to use a user subroutine that will allow us to define the magnitude of
 227 the load in Abaqus/Explicit® with a space-time function. As explained earlier, one of the
 228 main contributors is the overpressure generated by the rapid vaporization of the metallic
 229 lightning strike protection that covers the composite aircraft surface and aims at diverting
 230 lightning current. Our focus is on ECF195 which is mainly used for lightning protection. Based

231 on the development detailed in [13], the lightning strike protection is approximated in this
 232 part by a web of wires of $\varnothing 125\mu\text{m}$ for this configuration. Each wire is considered as a source
 233 of overpressure dependent on current density, which will generate Joule heating, which is
 234 assessed as follows:

$$J_n = I_n / S \quad 4$$

235
 236 with I_n , the total injected current (100kA for waveform D) divided by the number of wires in
 237 intersection with the vaporization profile and S, the section of the wire. The vaporisation
 238 profile was obtained thanks to high speed camera measurements through transparent GFRP
 239 panels [13]. The boundary of this profile is recorded every micro second and is
 240 superimposed on the ECF pattern in order to count the number of intersections as shown in
 241 Figure 10 below:



242
 243 **Figure 10 Vaporisation profile intersection with ECF**

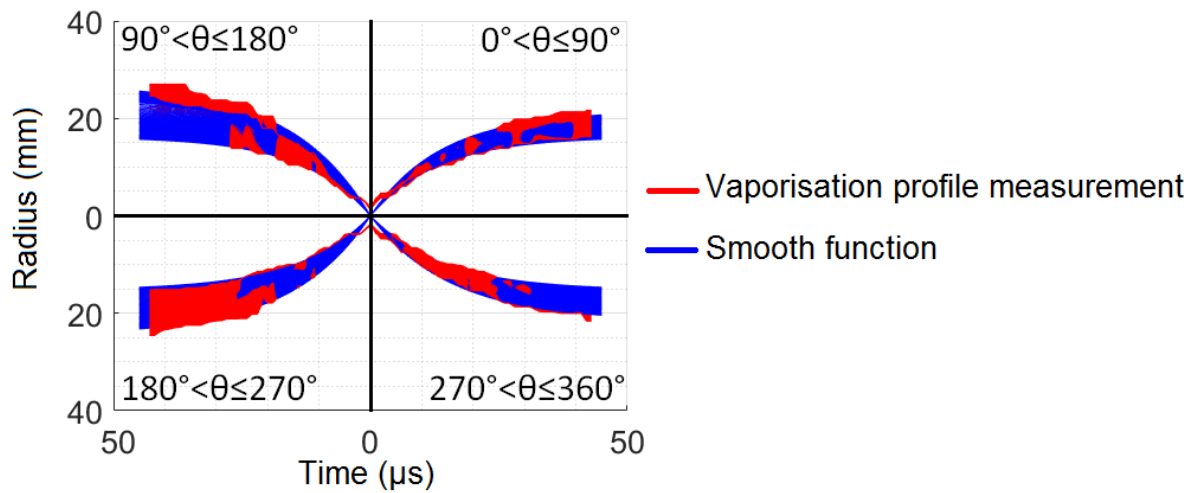
244 Since the measurement of the vaporisation profile is noisy due to the filter dependency on
 245 light intensity which varies during the test, a smooth profile was developed thanks to Matlab
 246 fit function in order to determine the parameter α of equation 5. For this purpose, the
 247 vaporisation profile was converted into the polar coordinate system and a radius function was
 248 defined per angle degree as follows:

$$R(\theta, t) = R_{max}(\theta)(1 - e^{-\alpha(\theta) \times t}) \quad 5$$

249

250 with θ the angle, from 1 to 360°, t the time of the vaporisation profile expansion, R_{\max} the
251 maximum radius reached at θ angle direction and α a fitting parameter dependent on the
252 angle. The function, illustrated in Figure 11, is therefore continuous with time and starts at

253 zero:

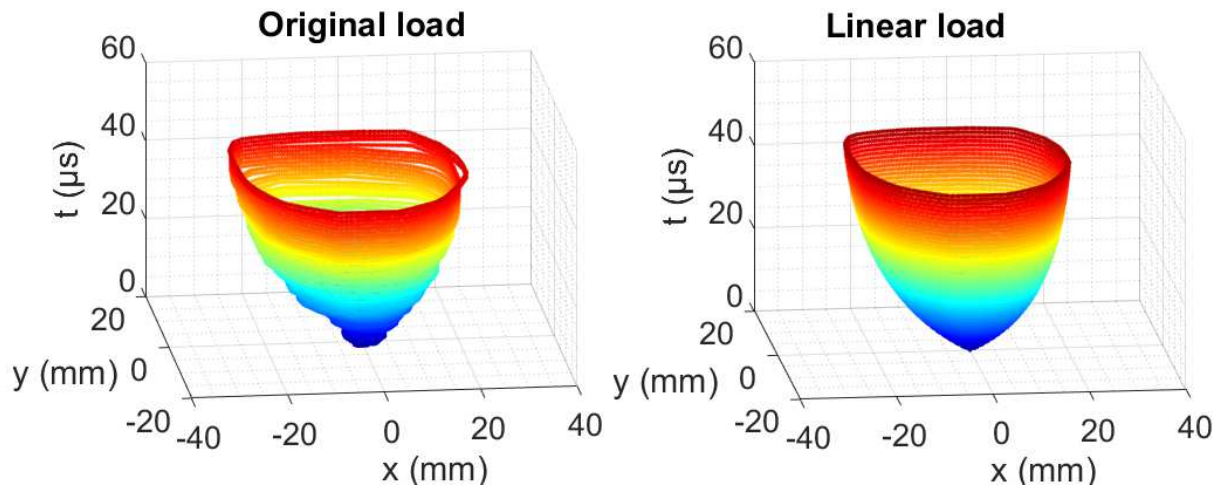


254

255 **Figure 11 Radius fitting function - Angle view**

256 The main advantage of such a definition is that it is independent from the picture sampling
257 per μs and that it removes the variability due to measurement error, giving an increasing
258 profile with time. This linearization of the spatial distribution of the vaporisation profile is
259 illustrated in Figure 12 where the profile is projected on the time axis:

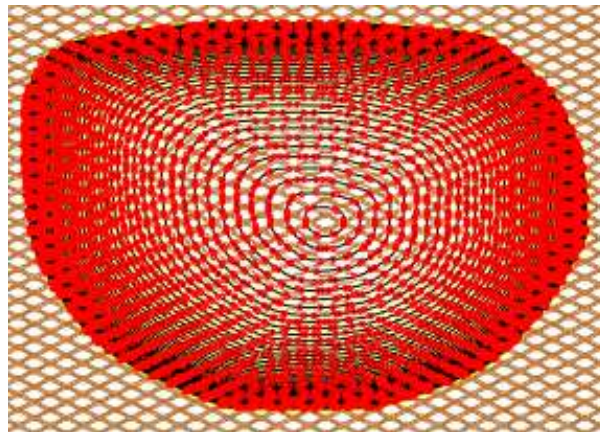
260



261

262 **Figure 12 Vaporisation profile linearization**

263 Based on this linearization of the spatial vaporisation, we can now assess the number of
 264 intersections per wire of ECF at each time step, as illustrated in **Figure 13** With this number, a
 265 current amplitude per wire can be defined by considering a homogenous distribution of the
 266 current along the vaporisation boundary in each wire in intersection at a given time.



267

268 **Figure 13 Intersection detection with linear vaporisation profile (ECF195+400µm of paint)**

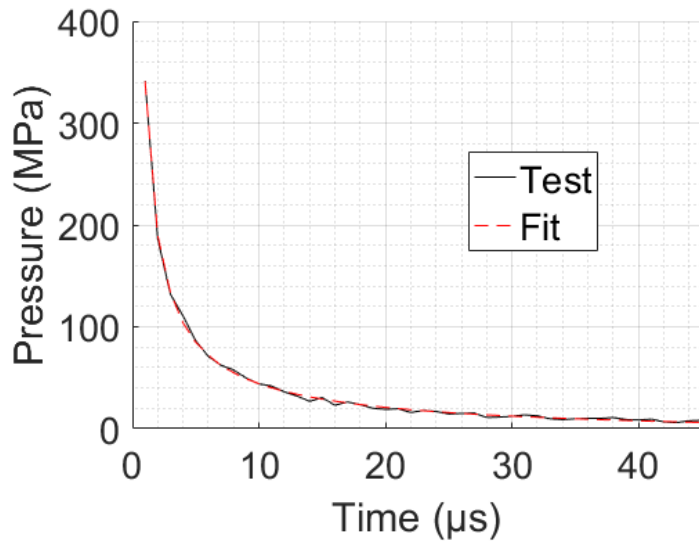
269 We can relate the pressure peak amplitude ΔP^+ of the wire explosion to the current amplitude
 270 I_{max} flowing in it based on experimental tests **[13]**:

$$P_{exp}(I_{max}) = a_1 I_{max}^{a_2} + a_3 \tag{6}$$

271

272 In this configuration, $a_1=80.3$ (MPa.A^{-b}), $a_2=0.64$ and $a_3=-62.88$ MPa. Those parameters were
 273 determined with a function fit. Then, based on the vaporisation profile defined in the section,

274 we have a relationship between the explosion time and the current amplitude in the wire I_n .
 275 Therefore, we can relate the pressure peak directly to the explosion time along the
 276 vaporisation profile in order to simplify the VDLOAD as presented in **Figure 14**.



277
 278 **Figure 14 Shock wave peak pressure dependency on the explosion time**

279 Knowing the vaporisation profile spatial distribution at each time step, we can associate the
 280 peak pressure amplitude with a polynomial function dependent on the explosion time thanks
 281 to Matlab fit function:

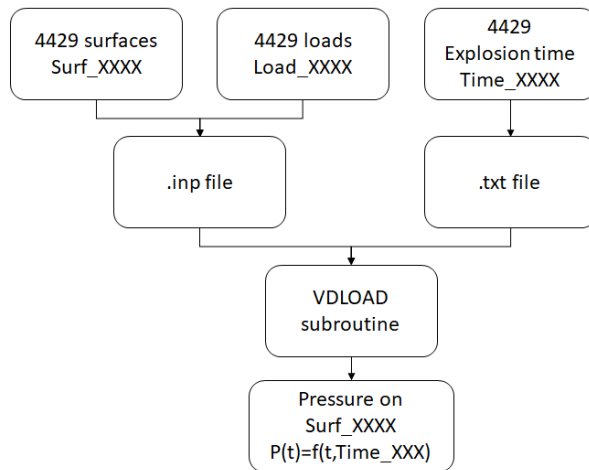
$$P_{exp}(t_{exp}) = p_1 t_{exp}^{p_2} + p_3 \quad 7$$

282
 283 with t_{exp} the explosion time of the wire along the vaporisation profile in μs , P_{exp} the associated
 284 peak pressure in MPa. $p_1=350.64812$ ($MPa \cdot \mu s^{-1}$), $p_2=-0.81381945$ and $p_3=-9.6754547$ (MPa)
 285 for this configuration of the ECF195 vaporisation profile confined by $400\mu m$ of paint. This
 286 process will be performed for each configuration of LSP and paint thickness since it will change
 287 the vaporisation profile, the explosion time and the peak pressure amplitude with the current
 288 density.

289 This function 7 will be useful for the VDLOAD, to prescribe the pressure load in each finite
 290 element at each instant. Note that, nevertheless, it uses the initial position of the grid to

291 determine the maximum pressure. Consequently, to apply the corresponding pressure, it is
292 necessary to locate the initial position of the application point.

293 Indeed, in reality, during the load application, a deflection of the sample will occur, thus in the
294 numerical model the finite element mesh cells are expected to move as well from their original
295 position. Therefore, the comparison of the element location with the vaporisation profile
296 distribution will lead to “jumps” in the load condition and an unexpected variation of the
297 pressure with time. Unfortunately, in the source file of an Abaqus® VDLOAD subroutine, it is
298 not possible to identify each mesh element number individually which could have been an
299 easy way to locate its initial position and assign a pressure history. In order to solve the issue,
300 the VDLOAD was not applied on a single top surface but on several surface elements. To
301 achieve this, the mesh coordinates of the composite front surface defined in the Abaqus®
302 ‘.inp’ file were reviewed and superimposed with the space-time vaporisation profile. Each
303 square mesh face of 0.5mmx0.5mm, that is included in the total load profile, was clearly
304 identified and an explosion time was associated. In this case, 4429 elementary surfaces were
305 defined. The ‘.inp’ file was modified to include the elementary surfaces, called “Surf_XXXX”
306 with XXXX the element number defined, and its associated load called “Load_ XXXX”. Apart
307 from the ‘.inp’ modification, a text file was created in which an explosion time is associated to
308 the load n°XXXX. During the computation, the VDLOAD applies the load XXX on the associated
309 surface XXX with an amplitude dependent on its associated explosion time as defined in **Figure**
310 **15:**



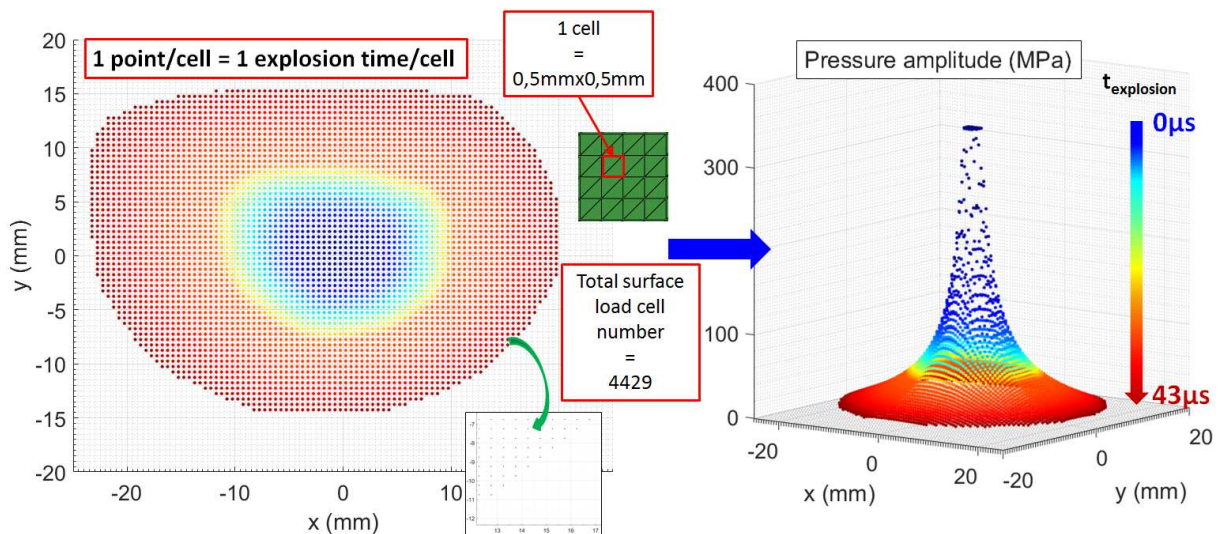
311

312

Figure 15 VDLOAD inputs

313

314 The VDLOAD subroutine calls the text file and identifies the explosion time. Therefore, each
 315 load cell is identified and its maximum pressure can be defined thanks to the equation 69
 316 established above based on its explosion time.



317

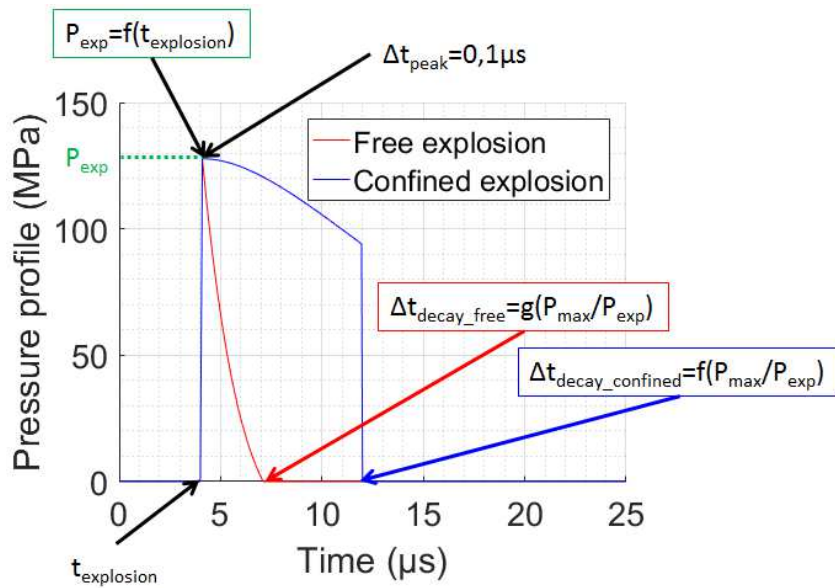
318 Figure 16 Vaporisation profile projection on mesh grid

319 As shown in Figure 16, the vaporisation profile defined as a function of time t and space (x,y)
 320 was projected and the explosion time is identified at the centre of each cell. One cell is one
 321 square of the mesh on the top face of the composite panel modelled. In our case, the first
 322 composite ply orientation is oriented at 45° , therefore one cell is made of 2 prismatic finite
 323 elements' top faces.

324 The pressure profile per cell will be defined as shown in Figure 17. Before the explosion time
 325 associated to the cell, no pressure is applied. Then, the pressure linearly increases up to its
 326 maximum as defined by equation 8 in $\Delta t_{\text{peak}}=0.1\mu\text{s}$. In theory, the time to peak for a shock
 327 wave is considered null but a small Δt is introduced to limit numerical discontinuities. After
 328 reaching its maximum, the pressure decreases. The decrease shape and duration (Δt_{decay})
 329 should depend on the confinement. For a free explosion of the LSP, i.e. an unpainted panel,
 330 the pressure shape is considered similar to the shock wave law based on the Friedlander
 331 equation:

$$P(t) = P_{\text{exp}} e^{-\frac{(t-t_{\text{explosion}})}{\Delta t_{\text{decay_free}}}} \times \left(1 - \frac{(t-t_{\text{explosion}})}{\Delta t_{\text{decay_free}}} \right) \quad 8$$

332
 333 The decay time $\Delta t_{\text{decay_free}}$ is considered dependent on the amplitude of the explosion where
 334 the decay is longer with the pressure amplitude decrease. For a confined configuration where
 335 a paint layer is present above the LSP and confines the gas of the explosion, this quick decay
 336 of the pressure cannot be considered anymore. Several shapes of pressure decay dependent
 337 on decay duration were tested and will be presented in the next section. The decay duration
 338 is defined dependent on the pressure ratio between the maximum pressure P_{max} that will be
 339 applied on the panel, corresponding to the most central area where the current density is the
 340 highest, and the cell pressure of explosion P_{exp} . Indeed, our hypothesis is that a high pressure
 341 amplitude will quickly eject the paint above and release the pressure. In the meantime, the
 342 pressure is almost fully maintained. When the pressure produced by the explosion decreases
 343 as the Joule heating in the LSP is lowered, it will delay the ejection of the paint and the
 344 pressure will be applied during a longer period.



345

346 **Figure 17 Pressure profile for VDLOAD**

347

348 **4. Results and analysis**

349 **4.1. Surface explosion validation**

350 In order to validate the contribution from the surface explosion only, a simulation for GFRP
 351 panels was performed. As already mentioned, specific lightning tests have been performed
 352 in laboratories where different LSP have been applied on a GFRP substrate which has the
 353 main advantage of being highly resistive. Therefore, all the lightning current will flow in the
 354 metallic protection, leading to its explosion. With this configuration, no delamination or
 355 Joule heating occur in the laminate, thus the substrate can be simulated with a simple elastic
 356 behaviour. The VDLOAD subroutine is used to apply a pressure profile dependent on time
 357 and space as described earlier. The only unknown is the decrease in the pressure when the
 358 metallic explosion is confined by paint or free. We have thus simulated several load
 359 configurations in order to assess the influence of the decay on the panel deflection.

360 **4.1.1. Free surface explosion**

361 In a first step, we considered the free explosion of the LSP with different decay times Δt_{decay} .

362 It was not possible to use the method defined in [13] to derive a space-time vaporisation

363 profile, since the light coming from the arc column was not filtered in the absence of paint

364 and disturbed the light capture. Therefore, we used the vaporisation profile simulated with

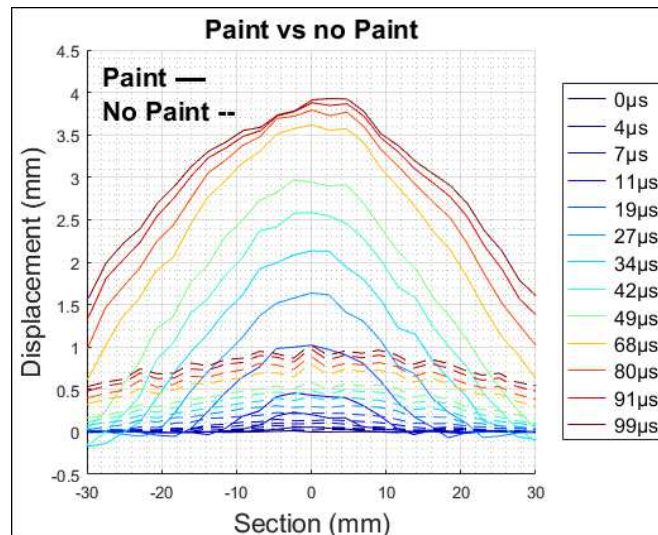
365 COMSOL® [31] since it presented good correlation with the experimental melting profile.

366 The pressure decay follows this Friedlander law defined above.

367 The displacement due to a free explosion of ECF195 is very limited. For the GFRP

368 configuration, the maximum displacement after 100 μs is only around 1mm compared to

369 almost 4mm for a confined explosion with 400 μm of paint as shown in Figure 18 below:



370

371 **Figure 18 Displacement profile comparison: Paint vs no Paint**

372 Considering a uniform decay time over the loaded faces no matter what the explosion

373 pressure amplitude, or considering a decay time inversely proportional to the peak pressure

374 resulted in displacement profiles that were never really comparable to the test result: the

375 displacement profile was too sharp at the start of the deflection, then too slow and the

376 displacement away from the vaporisation profile area was not negligible compared to the

377 central area. This led us to the conclusion that a contributor was missing. Since no current

378 can flow in GFRP panel protected with ECF195, there is no thermal stress due to Joule
 379 heating in the panel and any thermal deformation due to heat transfer will occur on a later
 380 stage [32] as it is a slow process compared to surface explosion. This is not supposed to be
 381 the missing contributor and we surmised that it could be the acoustic shock wave from the
 382 arc column. We therefore added this pressure to the LSP explosion pressure taking into
 383 account the definition proposed by Karch [33]. This is an easy-to-use and very pragmatic
 384 approach which does not consider any Magneto-Hydro-Dynamic (MHD) effect but provides
 385 and gives the correct order of magnitude and can be implemented easily in a VDLOAD. The
 386 spatial distribution is a growing disk of a radius R depending on time:

$$R(t) = 1.004 \times \left(\frac{E_0}{\rho_0}\right)^{0.25} \sqrt{t} \quad 9$$

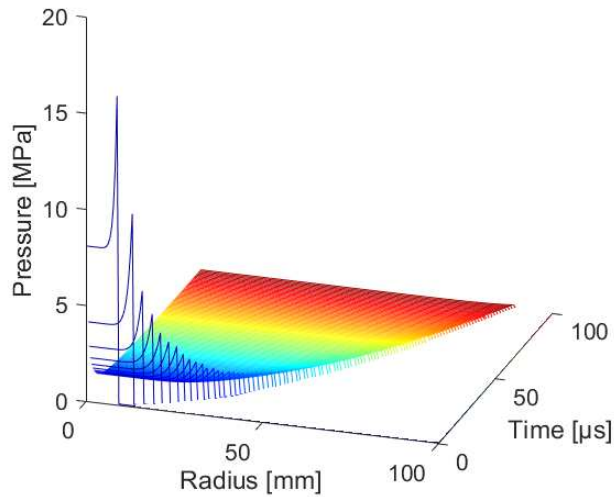
387
 388 with ρ_0 the initial air density (kg/m^3) and E_0 , the initial arc energy density (J/m). The latter
 389 parameter is a function of the maximum amplitude of the lightning current I_{\max} :

$$E_0 = 0.45 \times 10^{-2} (I_{\max})^{1.25} \quad 10$$

390
 391 Then, the amplitude of the shock wave can be defined as follows:

$$P(\eta, t) = 0.18 \times (\rho_0 E_0)^{0.25} \times \frac{f(\eta)}{t} \quad 11$$

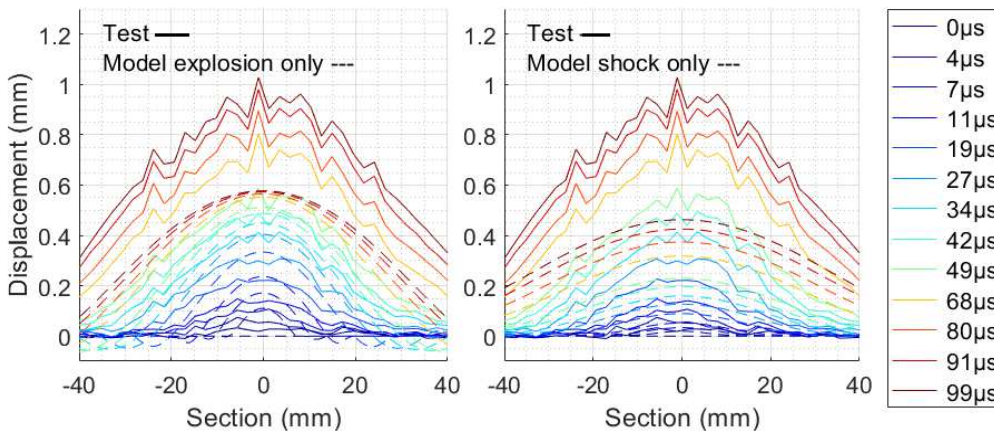
392
 393 with $f(\eta)$, defined in [33], a dimensionless shape function dependent on the ratio of the
 394 radius within the pressure disk and the shock front radius: $\eta=r/R(t)$. The shockwave pressure
 395 is shown in Figure 19, from $1\mu\text{s}$ to $100\mu\text{s}$, illustrating that a pressure is maintained after the
 396 shock front passage and slowly decreases:



397

398 **Figure 19 Shock wave pressure distribution**

399 Looking at the contribution of the shock wave only on the panel deflection, we can conclude
 400 that its contribution is about half the maximum displacement, and so is not negligible
 401 compared to a free surface explosion, as illustrated on Figure 20:

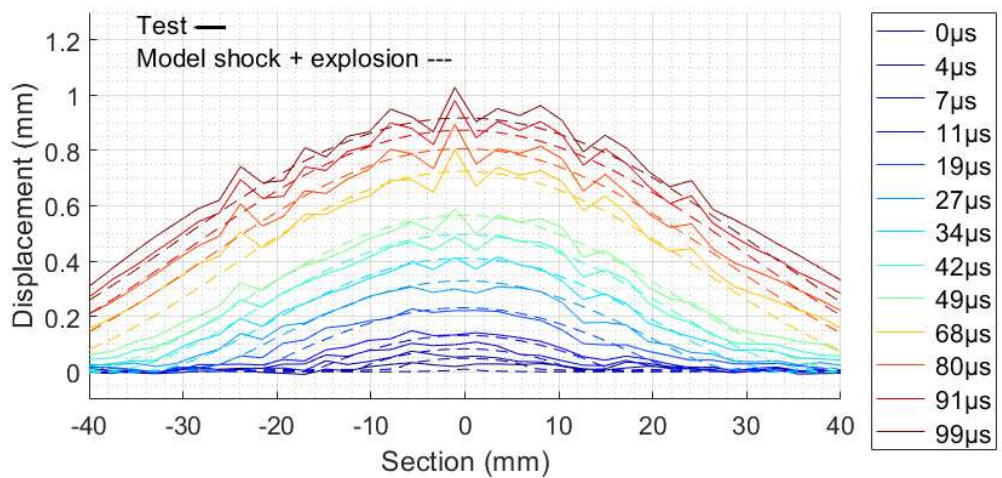


402

403 **Figure 20 GFRP displacement comparison between free explosion lightning test and simulated contribution of explosion**
 404 **only (left) or shock only (right)**

405 Finally, summing both contributions of the arc shock pressure coming from outside the LSP
 406 and the explosion of the LSP, the displacement of the GFRP panel, protected with ECF195,
 407 unpainted and subjected to lightning waveform D is well simulated (see Figure 21). The decay
 408 time for the explosion is inversely proportional to the peak pressure and the minimum decay
 409 time in the centre is equal to $0.85\mu\text{s}$. A comparison of rear face deflections with the LSP
 410 surface explosion alone and with the addition of the shock wave is presented in Figure 21.

411 Since the explosion is quickly released, the overpressure generated on the panel is limited,
412 hence the need to take into account the shock wave overpressure which lasts longer.



413

414 **Figure 21 GFRP displacement comparison between free explosion lightning test and simulated contribution of explosion**
415 **and shock wave**

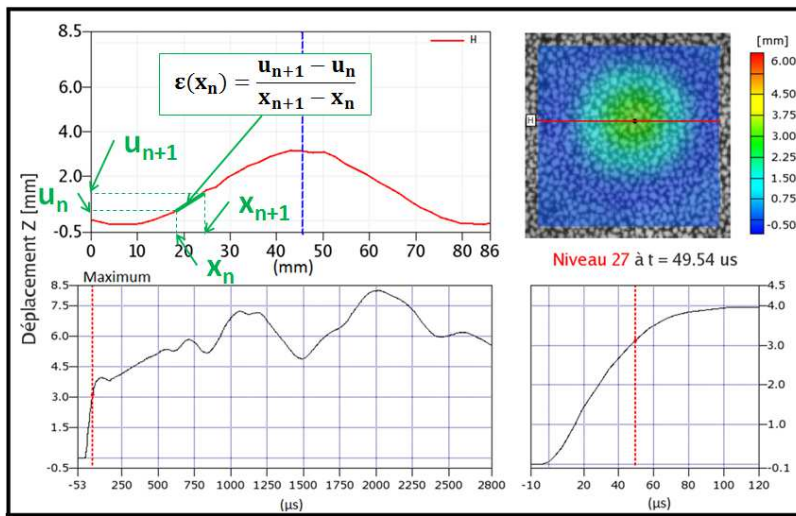
416 This first step without any confinement due to the paint highlights that the approach of the
417 overpressure due to the explosion as a free shock wave is a fairly good hypothesis but the
418 shock wave due to the lightning arc generation cannot be neglected. In order to support the
419 validation of this approach, other configuration of lightning strike protection with a different
420 profile of explosion is necessary.

421

422 **4.1.2. Confined surface explosion**

423 In a second step, we will now consider a more complex configuration: the displacement of a
424 GFRP panel, protected with ECF195 and painted. For this case, the deflection of the panel
425 will be enhanced since the surface explosion is confined by the presence of the paint. Adding
426 only the same pressure as for an unpainted plate subjected to the shock is no longer valid
427 because the paint confines the explosion pressure which is also different due to the arc
428 constriction. The space-time distribution comes from the vaporisation profile measurement

429 [13]. It is therefore necessary to propose another method to estimate the pressure decay
 430 after the maximum value given by the explosion.
 431 In order to compare the influence of different explosion profiles with the lightning test
 432 result, we chose to consider two criteria of the panel displacement. First, the maximum rear
 433 face displacement which occurs at the centre and second, the maximum of the deflection
 434 slope ϵ as defined in Figure 22, will be recorded. The maximum slope is computed on the
 435 horizontal and central cross section of the centre point displacement profile.



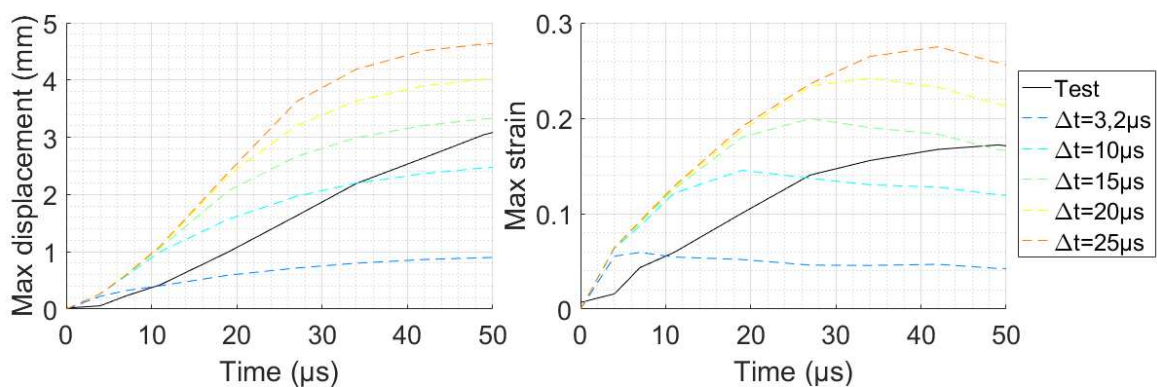
436
 437 **Figure 22 Maximum strain definition**

438 Since the effect of the confinement on the explosion is not fully known, different pressure
 439 decay waveforms are considered and implemented as a surface load through a VDLOAD. The
 440 first configuration sets a constant decay time Δt_{decay} for all elementary loads, whatever the
 441 pressure amplitude. The different decay times considered were 25, 20, 15, 10 and $3.2\mu s$. The
 442 pressure decay followed the defined law, illustrated in Figure 17:

$$P(t) = P_{exp} e^{-\frac{(t-t_{explosion})}{\Delta t_{decay}}} \times \left(1 + \frac{(t-t_{explosion})}{\Delta t_{decay}} \right) \quad 12$$

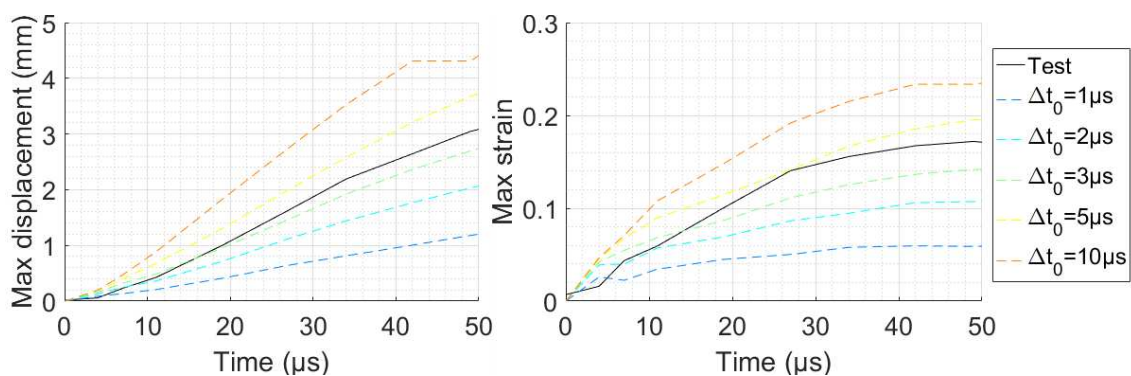
443
 444 On Figure 23, it is clearly visible that a uniform decay time cannot generate the equivalent
 445 overpressure. Compared to the test, the max slope is too steep in the early stage as for the

446 maximum displacement. The configuration that is closest to the maximum of displacement
 447 and slope in the latest stage is the configuration with 15 μ s of decay. This approach cannot
 448 represent the overpressure generated on the whole surface. The paint will be ejected
 449 depending on the stress generated by the overpressure on the paint. As this overpressure
 450 decreases with the current density decrease in the ECF, the paint will take longer and longer
 451 to be ejected. The decay time will thus increase with the increase of distance from the
 452 centre [13].



453
 454 **Figure 23 Overpressure sensitivity analysis with constant decay time**

455 The second configuration considered has a decay time Δt_{decay} which increases with the
 456 inverse of the pressure decrease. Several values chosen a priori were considered: 1, 2, 3, 5
 457 and 10 μ s. As shown in Figure 24, the law which provides the closest deflection behaviour is
 458 the one with the minimum decay of 3 μ s in the centre but again, the max slope and
 459 displacement are higher in the early stage.

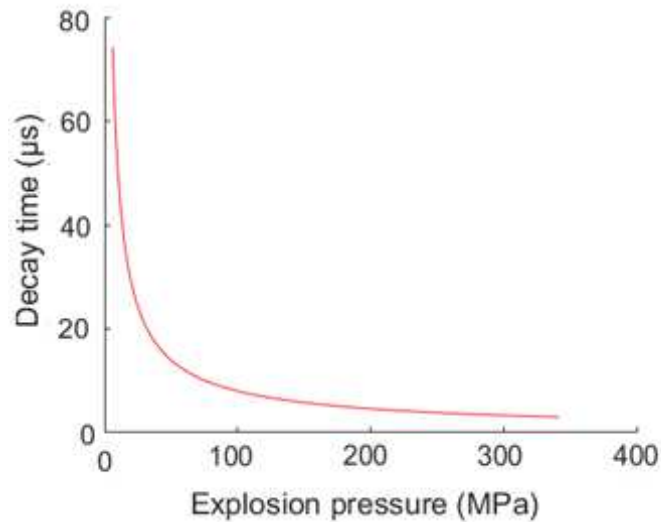


460
 461 **Figure 24 Sensitivity analysis of overpressure with inverse decay law – max strain**

462 For this configuration, no shock wave is applied since its contribution cannot be applied
 463 directly on the panel, no matter what the configuration of the explosion since the ejection of
 464 the paint will disturb the overpressure applied after the shock front. It is too complex to
 465 simply add it as another contributor of the total overpressure to the LSP explosion. In our
 466 approach, we simulate a confinement of the paint with a chosen decay time and introducing
 467 the shock wave would not make a lot of sense. This approximation should not increase the
 468 error significantly since the deflection due to the lightning arc shock wave on the GFRP panel
 469 is less than a maximum of 0.5mm after 100µs compared to a total deflection of almost 4mm
 470 at the same time when there is a confinement with 400µm of paint. The final decay law
 471 chosen for this specification configuration is:

$$\begin{cases} \Delta t_{decay} = \Delta t_0 \left(\frac{P_{max}}{P_{exp}} \right)^\alpha \\ \Delta t_0 = 3 \mu s \text{ and } \alpha = 0.8 \end{cases} \quad 13$$

472
 473 $P_{max} = \max(P_{exp})$ is the maximum pressure explosion reached in the centre where the current
 474 density in the copper wire is the highest. Δt_0 is the decay time in the centre and the shortest
 475 one to represent the fastest paint release due the highest explosion pressure.
 476 The parameter α drives the decay time increase with the explosion pressure decrease as
 477 shown in Figure 25:



478

479

Figure 25 Decay time as a function of the explosion pressure

480 In addition, the pressure profile was linearized as shown previously in Figure 12. This pressure

481 profile is the highest in the centre in one point. However, this is not related to any physical

482 phenomenon. Therefore, the pressure load applied in the centre will be on a disk

483 corresponding to the first measured disk in the back light measurement of the vaporisation

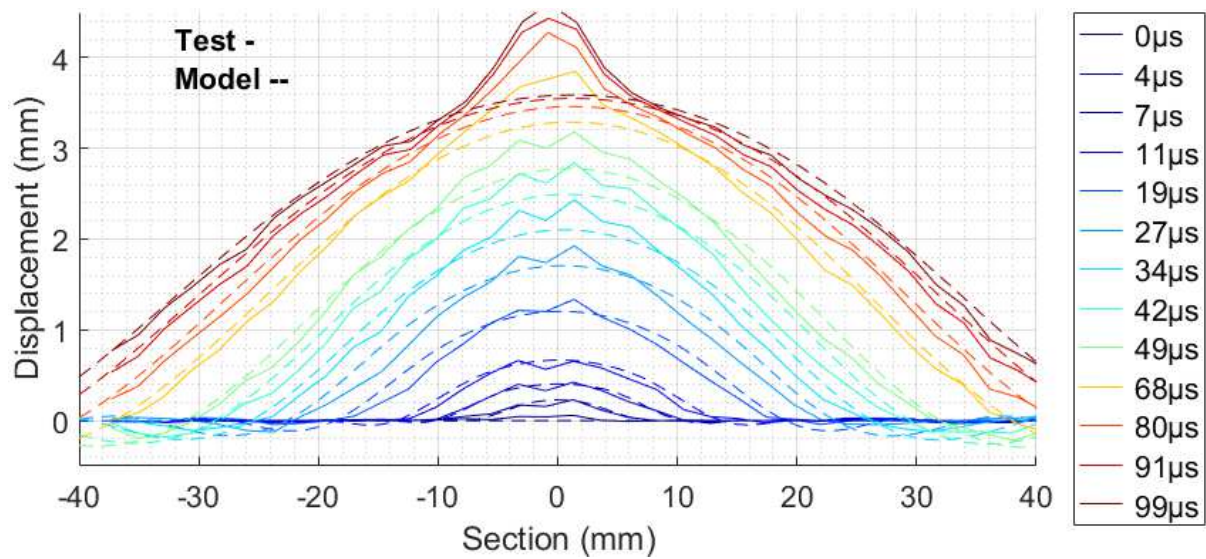
484 profile. The diameter of this disk is smaller than 4mm.

485 The deflection measurement of the GFRP panel protected with ECF195 and confined by

486 400µm of paint during the lightning test performed with WFD is compared with our

487 simulation with the VDLOAD following the decay law defined in Equation 6 36 and is

488 presented in Figure 26:



490

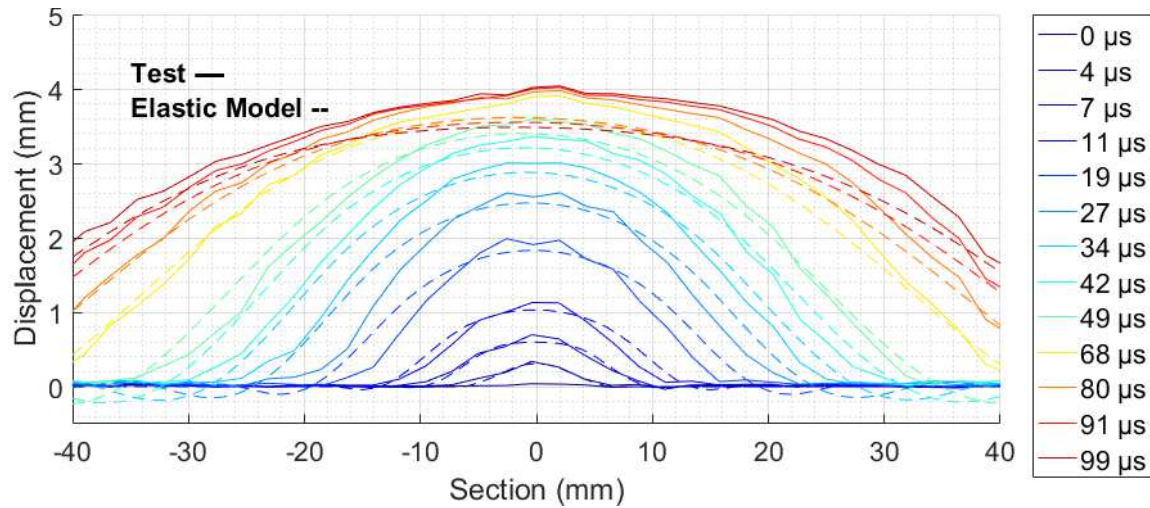
491 **Figure 26 GFRP displacement due to ECF195 explosion confined by 400µm of paint with WFD: Test vs shell**

492 The small bump than can be observed in the central area of the test panel deflection is likely
 493 related to the ejection of the paint used for the DIC pattern. This disturbs the profile
 494 reconstruction but should not be interpreted as a real local displacement. The deflection
 495 profiles between the test and the model are fairly similar. The maximum and the width of
 496 displacement are very close. The only main differences concern the maximum strain, as
 497 defined in Figure 22, and the waves on each side of the main deflection which are more
 498 pronounced in the model than in the test. But this is likely to have second order effect on
 499 the damage generation.

500 The maximum amplitudes of the explosion generated by each ECF wire are of key
 501 importance in determining the overpressure profile but this is also the case for the decay
 502 profile. The sensitivity analyses presented above are extracted from a modelling considering
 503 several decay laws. In order to build a more predictive approach without a supposed decay
 504 law, it will be necessary to simulate the confining effect of the paint and the mechanical
 505 constraints necessary to eject the paint. For this purpose, mechanical characterization tests
 506 of the paint and of the paint adhesive resistance are necessary.

507 **4.2. Effect on a CFRP laminate**

508 The interest of the work done previously was to study the deflection of a panel that has no
509 interaction with the lightning arc. A GFRP panel is highly resistive, therefore all the current
510 from the lightning arc will flow in the LSP, leading to its explosion. The comparison between
511 the simulated deflection from our VDLOAD model and the test results presented fairly good
512 agreement. The next step is now to analyse the effect of this overpressure profile on a CFRP
513 structure for which the interaction with the arc is more complex, especially with the
514 presence of paint that will constrain the arc and lead to current injection into the first plies
515 of the CFRP. The current flow in the CFRP which can generate internal damage due to
516 electro-thermal effects was not considered. As explained in 1.2, the contribution of these
517 effects are supposed negligible in the overall overpressure that will lead to mechanical
518 damage in the composite. Considering now the painted configuration, we can observe on
519 Figure 27 that the deflection profiles are fairly similar between the test of a painted laminate
520 and the simulation of an elastic laminate in the early stages, but not after $60\mu\text{s}$. With this
521 simulation with a simple elastic law, since no damage is allowed, the stiffness of the panel is
522 kept constant. But we know that damage and delamination occur early during the lightning
523 test. This delamination could explain the higher deflection profile for the test that is related
524 to the internal damage that reduces the global flexural rigidity of the panel which would
525 increase the deflection of the panel. This is a hypothesis but other contributors could
526 possibly modify the surface load as the arc root constriction which could locally increase the
527 constraint. This would require further investigations.



528

529 **Figure 27 CFRP displacement due to ECF195 explosion confined by 400μm of paint with WFD: Test vs shell**

530 The hypothesis considering that the internal damage of CFRP due to the current flow into
 531 the structure can be neglected in the assessment of the total mechanical constraints due to
 532 lightning strike tends to be validated by this work. Still, the deflection produced by the load
 533 profile validated previously presents some differences with the test which will required
 534 additional investigation in the current distribution between the CFRP and the LSP.

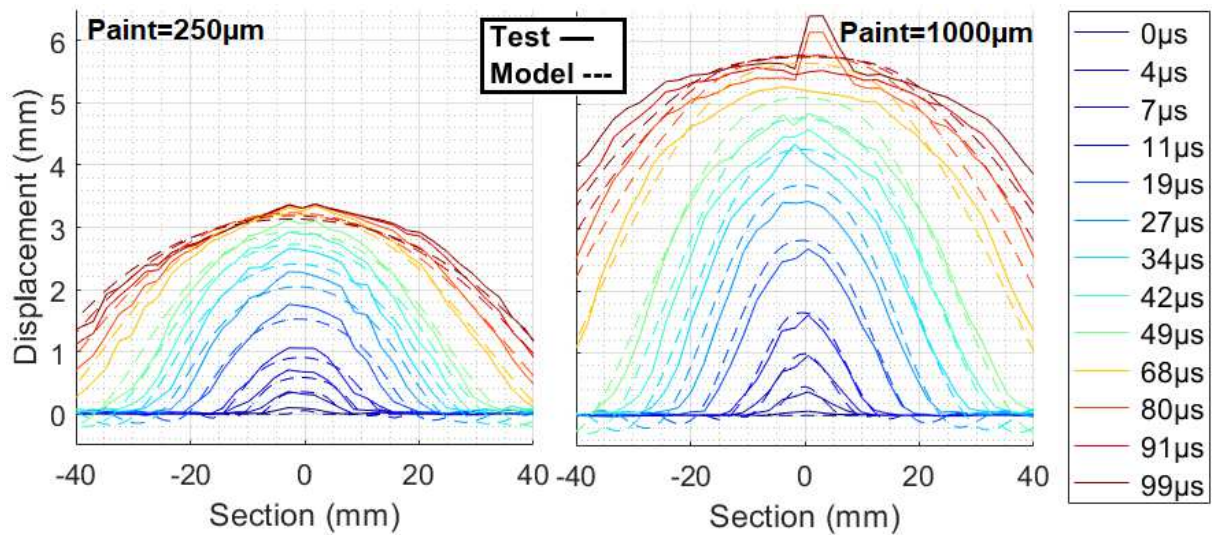
535 **4.3. Effect of explosion confinement**

536 As explained previously, the paint thickness has an influence on the confinement of the LSP
 537 explosion. In the previous work, we defined a confinement law for the pressure considering
 538 400μm of paint defined in Equation 12 and 13. In order to define the parameters of this law,
 539 a sensitivity analysis was necessary, requiring the performance of many model
 540 computations. However, this confinement law will be different with another paint thickness.
 541 In our lightning test campaign, we assessed the damage and deflection of a CFRP panel
 542 protected by ECF195 with different paint thicknesses: 250, 400 and 1000μm. In a first simple
 543 approach, we consider the effect of the paint on the confinement based on a linear
 544 relationship between the initial decay time and the paint thickness:

$$\begin{cases} \Delta t_{0\delta\mu m} \\ \Delta t_{0\beta\mu m} \end{cases} = \frac{\delta\mu m}{\beta\mu m} \quad 14$$

545

546 The power decay law α was kept identical at 0.8 for all the paint configurations. In addition
547 to adapting the pressure decay law for the pressure amplitude, we used the vaporisation
548 profile measured for the spatio-temporal distribution [13]. The distribution obtained with
549 200 μm of paint, which is close to a diamond shape, was used for the CFRP configuration with
550 250 μm of paint. The distribution with a paint thickness of 550 μm is close to an axisymmetric
551 evolution which means that a higher paint thickness will not change the vaporisation profile
552 significantly. Therefore, this distribution was used for the CFRP configuration with 1000 μm
553 of paint. Again, the model used here does not consider damage as it leads to disturbance in
554 the displacement. As with our reference configuration with 400 μm of paint, the
555 displacement profile is well predicted by this confinement law dependent on paint thickness,
556 as shown in Figure 28:

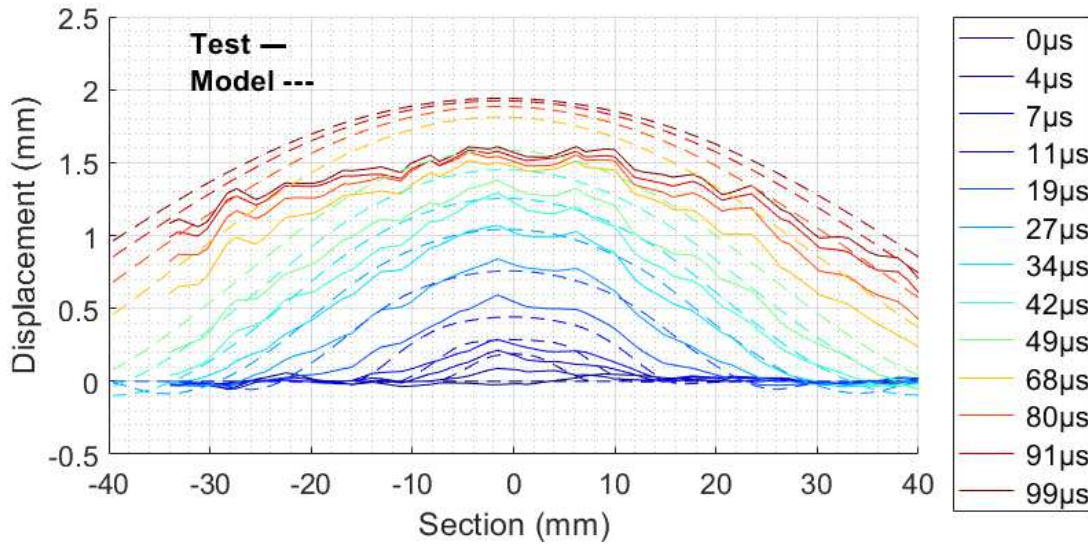


557

558 **Figure 28 Deflection profile comparison between test and model for different paint thicknesses**

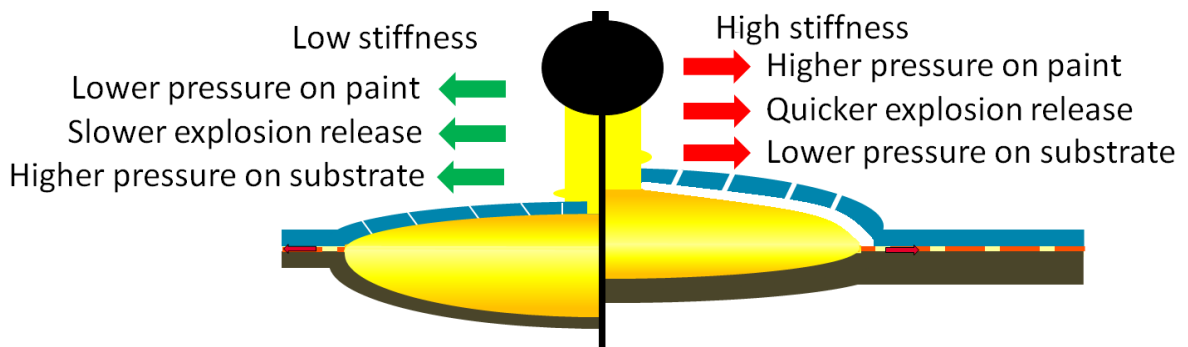
559 The pressure decay due to confinement can be predicted with different paint thicknesses.
560 However, the confinement of the explosion is due, on the one hand, to the paint layer as we
561 demonstrated, but also to the CFRP substrate on the other hand. Considering now a CFRP
562 substrate twice as thick as our baseline, we can assess the influence of the substrate

563 stiffness in the explosion confinement. If we apply the same mechanical load as our
 564 reference configuration (CFRP 13 plies + ECF195 + 400 μ m of paint) on a configuration that is
 565 twice as thick (CFRP 26 plies + ECF195 + 400 μ m of paint), the deflection profile is higher than
 566 expected, as illustrated in Figure 29:



567
 568 **Figure 29 CFRP thick configuration (EDIFISS 100) comparison between test and shell model with reference VDLOAD**

569 A substrate with a higher stiffness sustains the pressure from the explosion more,
 570 transferring a higher stress to the paint layer on the top. The paint is ejected more quickly,
 571 releasing the pressure of the explosion. This theory is illustrated in Figure 30 below:

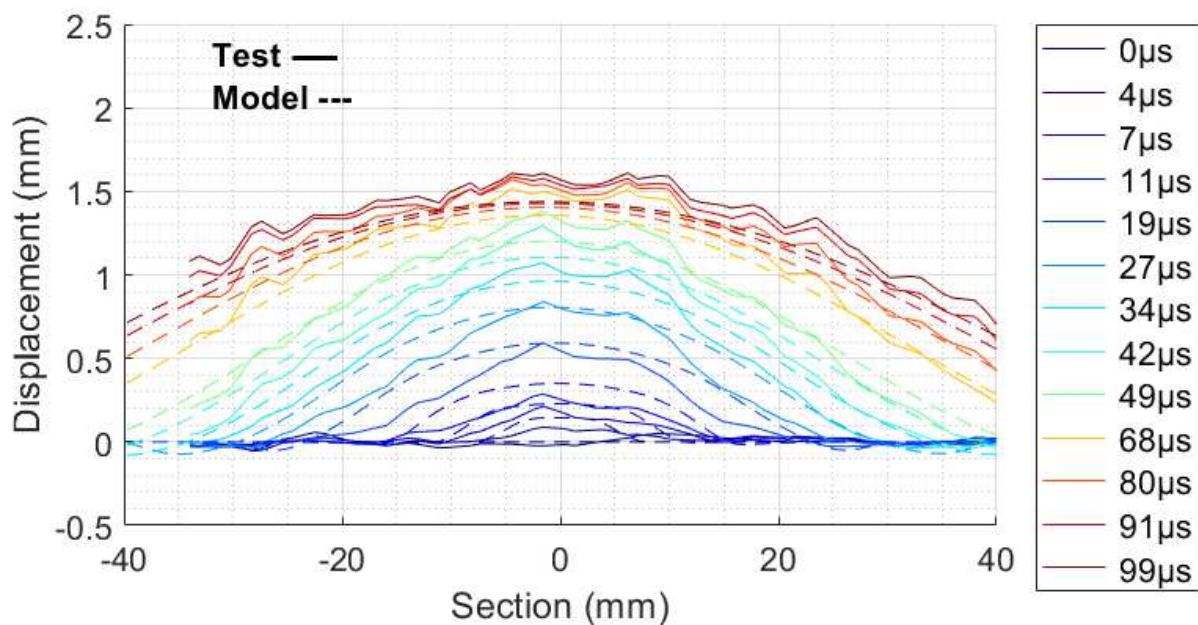


573
 574 **Figure 30 Substrate stiffness influence on explosion release**

575 We can use the laminate theory [30], as explained in 1.2.1, to compare the bending modulus
 576 of our reference CFRP panel made of 13 plies and its thicker configuration made of 26 plies.

577 In this case, the modulus E_f increases from 27GPa to 43GPa. As for the paint thickness
 578 impact, we propose to decrease the decay time Δt_0 with the ratio of E_f for the thick
 579 configuration. With the introduction of the influence of the panel bending stiffness in the
 580 confinement effect, the mechanical load tends to be more representative. As illustrated in
 581 Figure 31, the deflection of the panel due to this adapted mechanical load shows better
 582 agreement with the test measurement.

583

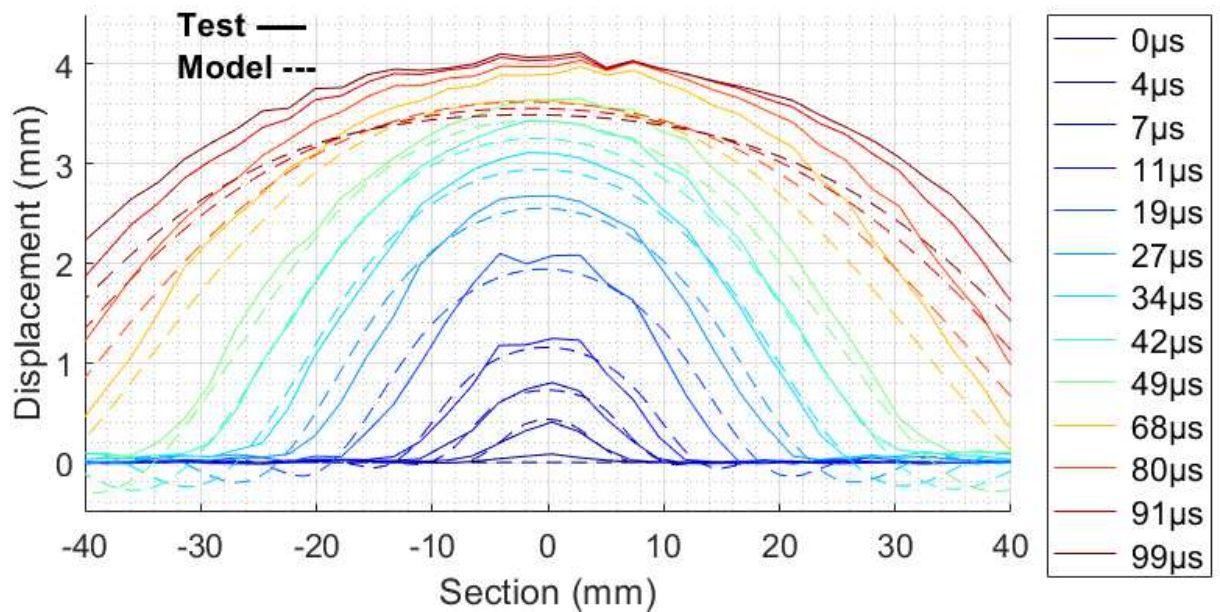


584

585 **Figure 31 CFRP thick configuration (EDIFISS 100) comparison between test and shell model with adapted VDLOAD**

586 On the same principle, we considered a CFRP structure with a different ply grade i.e., the
 587 thickness of the ply is 184 μ m instead of 127 μ m for the reference. But the construction
 588 comprises 9 plies (90/45/-45/0/90/0/-45/45/90) in order to obtain similar mechanical
 589 properties. The equivalent bending modulus is only slightly increased from 27GPa to 28GPa,
 590 therefore the same mechanical load as for the reference was applied. As for the reference,
 591 the model presents the same deflection in the early stage but the deflection then slows
 592 down earlier in comparison to the test results (see Figure 32). The construction of the load
 593 dependent on the bending modulus of the panel predicts fairly correctly the displacement of

594 the panel due to a lightning strike, validating the strong influence of the surface explosion on
595 the mechanical constraints applied on the structure.



596

597 **Figure 32 CFRP medium grade comparison between test and shell model with reference VDLOAD**

598 This work on the explosion confinement highlights the importance of considering the
599 lightning load as dependant on the substrate and the paint that will confine the surface
600 explosion and change the amplitude of the overpressure. This is an important conclusion of
601 this work. In order to complete this activity, it will be also necessary to assess its influence
602 on the arc root confinement and therefore on the current distribution and associated
603 explosion profile. As mentioned before, a model simulating the paint layer and its
604 mechanical properties and damage is a necessary step in order to predict the overpressure
605 and avoid the using of a supposed law of pressure distribution and release.

606

607 **5. Conclusion**

608 The main objective of this paper was to reproduce the mechanical constraints due to a
609 lightning strike. We proposed a methodology to simplify this multiphysics phenomenon into

610 a purely mechanical problem. The ability to reproduce the load produced by the lightning
611 strike is key in order to predict the damage generated by such a phenomenon.

612 From the characterisation of the LSP explosion developed in the previous work which
613 provided the spatio-temporal distribution of the explosion pressure and its amplitude, we
614 could build a complete equivalent mechanical load. In a first step, the explosion pressure
615 was modelled as a pressure applied on the surface of the composite. We built a
616 methodology to convert this profile into a VDLOAD contained in every face of the substrate
617 that will be stressed by the surface explosion pressure. The source of the model input was
618 modified and associated to the user subroutine we developed.

619 In order to validate this model, we chose a set of lightning experimental results which
620 provide essential data. The panel deflection due to the lightning strike was measured thanks
621 to a digital image correlation method.

622 The surface explosion and its confinement by paint were studied independently. Thanks to
623 the use of a GFRP structure which does not divert the lightning current, we were able to
624 study and calibrate the pressure generated by a confined explosion. Since the relaxation
625 time couldn't be measured independently, a simple law for the pressure decay time was
626 therefore developed, dependent on the maximum explosion pressure. This law simulated
627 the confinement due to the paint which maintains the pressure before being ejected by this
628 underlying explosion pressure. Our study of different paint thicknesses and panel stiffnesses
629 demonstrated the dependency of the pressure profile on the confinement. The LSP
630 explosion is sandwiched by the paint and the composite structure. Therefore, a change in
631 their mechanical properties will influence its confinement and will maintain, or not, a high
632 pressure on the structure. We were able to linearly relate the pressure decay law to the

633 variation in stiffness of these parts and predict the panel deflection with this simple
634 approach.

635 The final objective and the next step are to enable the damage prediction with a mechanical
636 model considering the surface load defined in this work.

637 6. Acknowledgements

638 This research was supported by EDIFISS project and funded by Airbus. DGA-TA is gratefully
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640 gratefully thanked for their help and collaboration. This work was performed using HPC
641 resources from CALMIP (Grant 2019-P19051).

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