



Transition to stress focusing for locally curved sheets

Thomas Barois, Ilyes Jalis, Loïc Tadriss, Emmanuel Viro

► To cite this version:

Thomas Barois, Ilyes Jalis, Loïc Tadriss, Emmanuel Viro. Transition to stress focusing for locally curved sheets. Physical Review E , 2021, 104 (1), 10.1103/PhysRevE.104.014801 . hal-03285055

HAL Id: hal-03285055

<https://hal.science/hal-03285055>

Submitted on 13 Jul 2021

HAL is a multi-disciplinary open access archive for the deposit and dissemination of scientific research documents, whether they are published or not. The documents may come from teaching and research institutions in France or abroad, or from public or private research centers.

L'archive ouverte pluridisciplinaire **HAL**, est destinée au dépôt et à la diffusion de documents scientifiques de niveau recherche, publiés ou non, émanant des établissements d'enseignement et de recherche français ou étrangers, des laboratoires publics ou privés.

Transition to stress focusing for locally curved sheets

Thomas Barois¹, Ilyes Jalisso¹, Loïc Tadrist² and Emmanuel Viot³

¹*Univ. Bordeaux, CNRS, LOMA, UMR 5798, F-33400 Talence, France*

²*Aix-Marseille Univ., CNRS, ISM, Marseille, France*

³*hap2U, 75 Avenue Gabriel Péri, 38400 Saint Martin d'Hères, France*



(Received 8 January 2021; revised 16 March 2021; accepted 4 June 2021; published 9 July 2021)

A rectangular thin elastic sheet is deformed by forcing a contact between two points at the middle of its length. A transition to buckling with stress focusing is reported for the sheets sufficiently narrow with a critical width proportional to the sheet length with an exponent $2/3$ in the small thickness limit. Additionally, a spring network model is solved to explore the thick sheet limit and to validate the scaling behavior of the transition in the thin sheet limit. The numerical results reveal that buckling does not exist for the thickest sheets, and a stability criterion is established for the buckling of a curved sheet.

DOI: [10.1103/PhysRevE.104.014801](https://doi.org/10.1103/PhysRevE.104.014801)

I. INTRODUCTION

The crumpling of a paper sheet is an usual outcome for someone trying to write a deep and meaningful piece of text, such as the introduction of a scientific paper. After being crumpled, the paper is irreversibly damaged, and its topography may be viewed as a disordered network of ridges. The persistence of such ridges is a structural memory of stresses focused during the forced manipulation of the sheet.

Deformations without stretching should prevail for thin sheets due to the large mismatch between thickness and typical size [1]. However, stretching deformations appear during crumpling because of stress focusing [2]. Stress focusing is observed with the indentation of a thin plate against a circular contour [3–5]. The resulting structure is called a developable cone (or d cone [6]), and it conveys the idea that the plate adopts a conical shape to satisfy developability. Experimental d cones do not rigorously match with conical surfaces. A U-shaped scar is found at the tip of a d cone and its formation is the consequence of the sheet avoiding diverging curvatures at the apex. This picture of stress focusing can be generalized to crumpling [7–12]: A crumpled configuration is a state occupying a small volume that combines regions approaching developability and localized ridges focusing the stretching deformations.

In the previous situations reported, a confinement is imposed to a thin sheet and stress focusing is compulsory because there are no developable deformations that can be contained within the imposed volume. In this paper, we identify the existence of a controlled transition to stress focusing for a thin sheet submitted to a bending deformation and without volume confinement. The main outreach of this paper is about the fundamental understanding of crumpling dynamics with the appearance of stress focusing whereas smooth solutions of pure bending satisfy the imposed condition. A second outcome is the characterization of the buckling threshold. This has a practical interest for the safe handling of a piece of thin

material without inducing stress focusing that would result in structural damages [13,14]. The characterization of this transition is also of interest in a context of exploiting buckling and crumpling for the realization of functional materials [15–21].

The point of this paper is to evidence a route to crumpling via the driving of low-energy bending deformations, which can be viewed as a one direction confinement. Stress focusing was previously observed for one direction confinement but with peculiar boundary constraints, such as an imposed three-buckle shape [22] or a pulling force [23]. In other situations with frames, the transition to stress focusing can be reversed, and defects can appear at small confinement before developable [24] or smooth [25] solutions establish for larger external loads. For draping [26], a transition to stress focusing exists for a sheet bent under its own weight, but the supporting tip adds a localized stress.

II. SELF-CONTACT EXPERIMENT

The present paper is based on a simple self-contact procedure applied to a thin sheet in order to characterize the transition to stress focusing. Figure 1(a) represents an elastic sheet of length L and width W . The thickness t is small compared to W and L . The material coordinates of the sheet u, v verify $u = x$ and $v = y$ for the flat sheet. The origin for xyz is the center of the sheet. Figure 1 shows two elongated sheets for (b) a small ($L/W \approx 2$) and (c) a large aspect ratio ($L/W \approx 6$).

For the aspect ratio $L/W \approx 2$, the sheet is transversely curved without stress focusing. The sheet is, however, not free of stretching deformation, and the presence of a small longitudinal stretching is responsible for the symmetric opening of the sheet. This coupling between transverse bending and longitudinal stretching related to a persistence length for the curved region [27–33]. The limit for a sheet free of stretching would be obtained for a small length $L \ll W$. In this case, the sheet profile would be invariant along x , and it would match

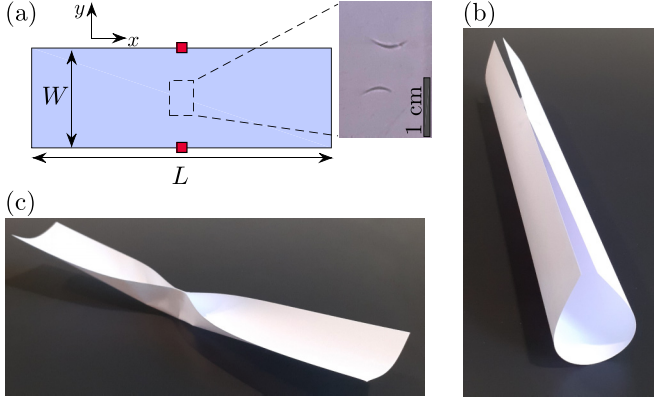


FIG. 1. (a) Schematics of a flat sheet of width W and length L . The square markers indicate the midlength points. A picture shows the scars formed after a self-contact procedure with an acetate sheet ($L = 29.7$ cm, $W = 8$ cm, $t = 0.2$ mm) (b) Rectangular paper strip ($L = 29.7$ cm, $W = 15$ cm, $t = 0.1$ mm) with midlength points at contact and maintained by an adhesive strip. (c) Strip with a smaller width ($L = 29.7$ cm, W cm, t mm).

with Euler's elastica [34] model on the projection plane yz . This limit identifies a smooth solution with self-contact and free of stretching: For any L , a developable surface can be obtained from an elastica profile on the plane yz invariant in the direction x .

For the large aspect ratio L/W , the sheet buckles and scars appear close to the midline $u = 0$. The picture in Fig. 1(a) shows the two scars at the apex of the d cones formed after self-contact ($L = 29.7$ cm, $W = 8$ cm, and $t = 0.2$ mm). For the paper sheet in Fig. 1(c) four d cones are formed. In this case, L/W is larger than for the acetate sheet. This points towards a route to crumpling with an increasing disorder for the sheets of large ratio L/W . Here, the experiments are performed close to the transition and the postbuckling morphology does not show the usual ridges network [35] observed at high compaction.

The appearance of stress focusing for a sheet simply curved, even locally, is counterintuitive because the bending modes have an elastic energy orders of magnitude below the stretching modes. For a typical scale $W = 5$ cm for the sheet in Fig. 1(c), the ratio bending-to-stretching energy with linear coupling is on the order of $(t/W)^2 \sim 4 \times 10^{-6}$. Stress focusing is even more surprising here because bending is a well-known strategy to prevent collapse under external loads, according to the process of curvature-induced rigidity [31,33].

III. SCALING LAW FOR THE BUCKLING TRANSITION

The self-contact procedure could be sensitive to unintentional poking or unwanted gravity load. It is, therefore, crucial to ensure that these contributions can be ruled out. This is performed by comparing experimental data to a numerical model of the self-contact procedure. A set of experiments is performed with acetate sheets ($t = 0.2$ mm) to identify the buckling transition. Figure 2 presents the state of a sheet at self-contact as a function of L/t and W/t . The numerical results are obtained with a spring-network model that will be discussed further. A postcard and a metro ticket are shown to

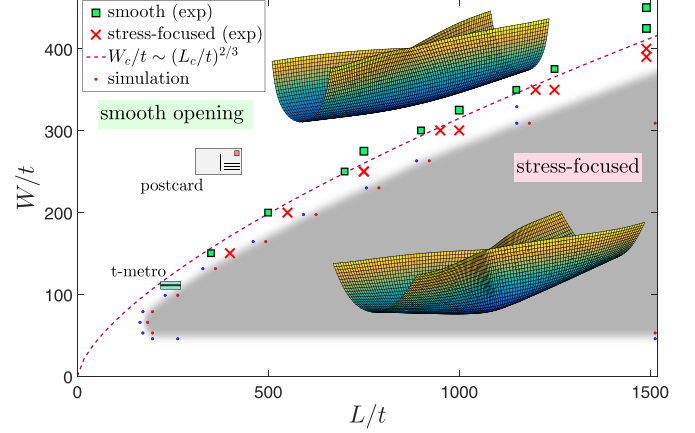


FIG. 2. State of a thin sheet in self-contact (L : length and W : width). The experiments are with an acetate sheet (thickness $t = 0.2$ mm) with squares for a smooth opening and crosses for sheets with stress focusing. The dashed line is a power law with an exponent $2/3$ and a numeric prefactor 3.15. The gray area indicates a stress focusing region obtained from the simulation (blue dots without stress focusing and red dots with stress focusing). A representation of two simulated sheets ($W = 200$) is presented with $L = 500$ for the smooth opening and $L = 550$ for the opening with focused stresses. A paper sheet (A4 format) at self-contact remains smooth ($L/t \sim 3000$ and $W/t \sim 2000$, outside the plot range).

get a sense of the orders of magnitude without the corresponding experiments actually performed.

For the experiments, buckling occurs in a region with a sufficiently large value of L/t for a fixed W/t . The smallest width explored is $W = 3$ cm which corresponds to $W/t = 150$. For this value, the bending deformation for self-contact is such that the sheet reaches a plastic regime, which is out of the scope of this paper. The largest width explored is $W = 9$ cm, slightly above the width for the transition for a sheet of length 29.7 cm ($L/t = 1485$, A4 format). The dashed line is $W_c/t = a(L_c/t)^{2/3}$ with the adjustment $a = 3.15$. The experimental transition is reproducible, and the existence of a scaling regime validates the idea that such crumpling events relate to a controlled mechanical instability. The value of the exponent $2/3$ will be discussed below.

Two strategies were tested to impose the self-contact at the midpoints ($u = 0$, $v \pm W/2$): (i) using two adhesive strips (width 5 mm) and (ii) tightening of a knotted wire (diameter 0.3 mm) between two holes close to the midpoints. Both strategies resulted in the same transition which suggests that the detail of the self-contact does not affect the transition. One could indeed wonder if the pointlike forces at the self-contact might not play the role of an indenting tip [2–5,24]. This is not the case here because the singularities are always formed away from the self-contacted points.

IV. SPRING NETWORK MODEL

For the experimental points below $L/t = 500$, the transition deviates from the scaling regime. To explore the thick sheet regime and the validity of the scaling exponent, a spring-network model is computed. Although crystalline spring networks do not rigorously model the isotropic

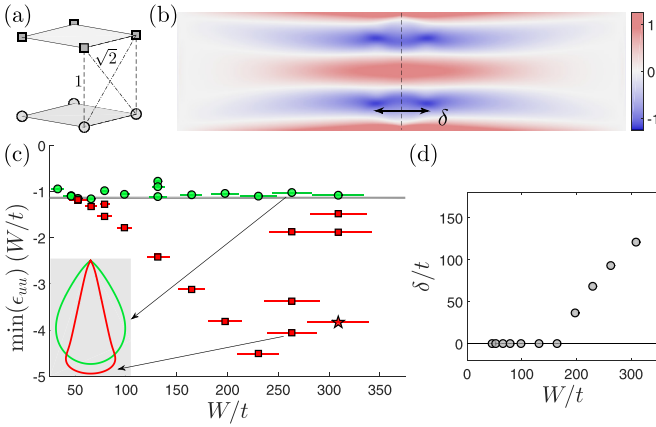


FIG. 3. (a) Unit cell of the spring-network model. Circles and squares represent the lower and the upper surfaces. The linear springs of length 1 and $\sqrt{2}$ are for the first neighbors and the second neighbors, respectively. (b) Normalized longitudinal strain $\epsilon_{uu} \times (W/t)$ of a simulated sheet ($L = 1800$, $W = 470$) just after buckling. The dashed line is the midline $u = 0$. Positive values correspond to a stretching, and negative values are for a compression. Four points of stress focusing are present. (c) Normalized minimal strain $\min(\epsilon_{uu})(W/t)$ (compression) as a function of the dimensionless width W/t . The circles are for simulated sheets with smooth opening and square symbols are for sheets with stress focusing. The horizontal line is at -1.16 . The profile at midlength $u = 0$ illustrates the curvature sign change with buckling. The star symbol indicates the sheet ($L = 1800$, $W = 470$) selected for the strain profile represented in (b). $t = 1.52$. (d) Horizontal spacing of the local minima for the longitudinal strain. $\delta = 0$ means one pair of d cones at $u = 0$ and $\delta > 0$ two pairs at $u \pm \delta/2$ as in (b).

elasticity of continuum media [35–39], such models are well adapted to explore the scaling properties of thin sheet configurations [40]. Figure 3(a) presents the structure of the unit cell used in the simulation. A thin sheet is obtained by two dimensional replication of the unit cell to form a bilayer. First, a flat sheet is generated, and then additional springs are added and tuned to reach self-contact. The equilibrium positions of the vertices are computed by iteration of the spring forces until a static equilibrium is found (see Appendix C 4).

The spring-network model is computed for different sheet dimensions to identify the transition between a smooth opening and a buckled sheet with focused stresses. The criterion to identify a buckled sheet is the sign of the transverse curvature. With a smooth opening, the transverse curvature is always positive. For a buckled sheet, this curvature is partly negative near $u = 0$. Figure 2 represents by dots the simulations performed and a filled area to identify the buckling region.

Stress focusing is not observed in simulations below $W/t = 56$. For $W/t < 56$, the simulations indicate that the sheet can open smoothly for any length L . In this regime, there is no risk of damages by stress focusing during manipulation and self-contact. This regime is, however, difficult to explore with thin elastic sheets. With paper ($t \approx 0.1$ mm), it would require to use strips with W of the order of 5 mm. For such narrow strips, it is not possible to force a self-contact without plasticity.

The regime of focused stresses is observed for a larger range of parameters in the experiments compared to the simulations. This systematic difference could be explained by the limitations of the spring-network model or the difficulty to realize controlled experiments with thin sheets for which mechanical and physical properties are not perfectly uniform.

The simulation results are further processed to find a stability criterion for the transition. Figure 3(b) represents the longitudinal strain profile ϵ_{uu} for a sheet of $1800 \times 470 \times 2$ vertices ($L/t = 1184$, $W/t = 309$) with contact of the midlength points. The strain profile is obtained during the simulated dynamics of the sheet, just after the buckling event. The strain profile combines regions of positive stretching, to allow the sheet to open, and regions of negative stretching (compression). The existence of longitudinal compression induced by transverse curvature was proposed in a previous work [30] to satisfy a zero longitudinal net force. Here, we uncover the structure of the stretching strain and the five-band pattern. There are four points of maximal compression lying at the middle of the compression bands and longitudinally separated by a distance δ . For a postbuckled sheet, each point of maximal compression is the source point of a d -cone structure. The time-resolved simulation (video, see the Supplemental Material [41]) clearly shows that the d cones are not nucleated at the self-contact. This proves that the self-focusing transition is not due to a local indentation by the pointlike force at the self-contact.

Figure 3(c) represents the value of the maximal compression strain $\min(\epsilon_{uu})$ for a sheet as a function of the width W/t . The strain is normalized with the dimensionless factor W/t . Each data point corresponds to a unique couple (W, L), and the length L of the sheets is indicated by a horizontal bar for each data point. The plot shows two families of data points: The circle scatters are for the smooth sheets without curvature inversion, and the square or star scatters are for the postbuckled sheet with curvature inversion of the transverse profile (see inserted schematics). The data point with a star corresponds to the dimension for the strain map in (b).

The plot in Fig. 3(c) indicates a threshold value $\min(\epsilon_{uu}) \times (W/t) = -1.16$ below which a sheet at self-contact buckles. The buckling dynamics can be followed in more details with the successive profiles of the sheet during the simulation. Figure 14 in the Appendix represents the evolution of the maximal compression strain during the convergence of the mechanical equilibrium of the simulated sheet. The sheet with $1700 \times 470 \times 2$ vertices ($L/t = 1118$, $W/t = 309$) converges to a smooth solution with a minimal strain above the threshold -1.16 . For the simulation with 1750 and 1800 vertices ($L/t = 1151$ and $L/t = 1184$), the minimal strain is sufficiently low to cross over the threshold value of -1.16 and a sudden drop of the minimal strain is observed as a signature of stress focusing. For the small sheet with $W/t = 35$ and $W/t = 49$, there is no buckling transition because the compression strain remains below the threshold $1.16 \times (t/W)$ for any L/t .

Figure 3(d) represents the longitudinal spacing δ between the points of minimum strain. A value of $\delta = 0$ means that there is a total of two local minima for the compression strain located on the midlength line $u = 0$. With $\delta > 0$, there are four local minima for the compression strain as shown in Fig. 3(b).

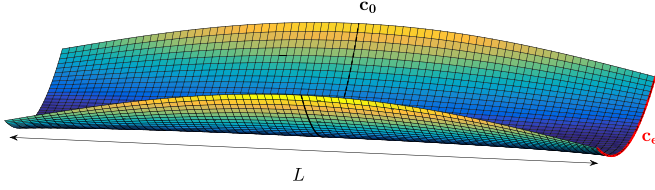


FIG. 4. View of strip of width W with the shape of a surface ruled by the curvature function $c(u)$ with $-L/2 < u < L/2$.

The number of local minima sets the profile of the first buckling event. In the simulation with $1750 \times 470 \times 2$ vertices, the stress focusing is first triggered at the four symmetric points of maximal compression strain. Then, the four points travel towards the midlength line and an abrupt increase in the maximal compression is observed when the pairs of stress focusing points merge.

V. ANALYTICAL TOY MODEL

The transition to stress focusing identified in experiments and simulations can be rationalized by a toy model based on: (i) elastic energy minimization [30] and (ii) a buckling criterion [42].

To estimate the elastic energy, we assume that the sheet at self-contact is described by its typical transverse curvature that takes the value of $c_0 \sim 1/W$ at $u = 0$ and c_e at the free ends at $u = \pm L/2$. By considering only the first-order contributions, the total elastic energy scales as

$$\frac{\mathcal{E}(c_e)}{ELWt} \sim t^2(c_0^2 + c_e^2) + t^2 \frac{W^4}{L^4}(c_0 - c_e)^2 + \frac{W^8(c_0 - c_e)^4}{L^4}, \quad (1)$$

with E as the Young's modulus. The terms on the right hand side stand for the transverse bending energy, the longitudinal bending energy, and the longitudinal stretching energy (see Appendix A). c_e is the only free parameter of the model, and its value is obtained by the minimization of the energy $\partial \mathcal{E}(c_e)/\partial c_e = 0$ that yields to an equilibrium between two bending terms and one stretching term,

$$t^2 c_e - t^2 \frac{W^4}{L^4}(c_0 - c_e) = 2 \frac{W^8(c_0 - c_e)^3}{L^4}. \quad (2)$$

We discuss here the solution of Eq. (2) in the thin sheet limit: with $W \gg t$ and $L \gg t$, the transition is such that L is larger than W and we can assume $L^4 \gg W^4$. Consequently, the term $t^2 c_e \sim t^2/W$ dominates the left hand side in Eq. (2) and the scaling for the curvature difference is $c_0 - c_e \sim t^{2/3} L^{4/3}/W^3$. According to our previous work [30], the scaling for the compression induced by a transverse bending is $\epsilon_{uu} \sim W^4(c_0 - c_e)^2/L^2$. Considering the criterion $|\epsilon_{uu}|(W_c/t) \sim 1$, we obtain

$$W_c \sim t^{1/3} L_c^{2/3}. \quad (3)$$

The exact solution of Eq. (2) confirms that the scaling (3) is valid only for large W/t (see Appendix A and Fig. 5). This exact solution also predicts an absence of buckling transition for small W/t as found in the simulations.

Experiments were also performed with paper, and the transition was found for lower L/t (not shown in Fig. 2

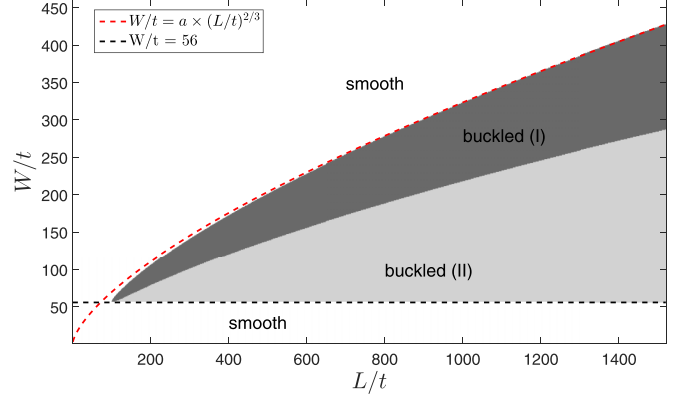


FIG. 5. Buckling criterion $\epsilon < \epsilon_c$ [Eq. (A12)] In the region *buckled I*, the buckling criterion is satisfied. Elsewhere, the sheet should be smooth except in region *buckled II* (see the text for details). The solid line is the scaling law with exponent $2/3$ and a numerical factor $a = 3.235$.

for clarity, see Appendix B). This difference is attributed to the anisotropy of paper [43]. By taking into account the anisotropy [32] with a smaller Young's modulus for the transverse direction, we show in Appendix B that the prefactor for the scaling relation is modified accordingly.

VI. CONCLUSION

The present results have evidenced a simple scaling relationship for the critical compressive strain of a sheet buckling with stress focusing, that is, $\epsilon_c \sim t/W$ with a prefactor close to unity as shown in Fig. 3(c). This criterion is used in the toy model and the state diagram obtained from the exact solution of Eq. (2) is consistent with experiments and simulations. In our previous work [30], the existence of a longitudinal compression for a transversely curved sheet was proposed to satisfy internal mechanical equilibrium in a model. Its relevance was only justified indirectly via a detailed analysis of the persistence length. Here, we show that the induced compression strain has the same effect than an external compression load applied to a shell with the same curvature: The geometric stability criterion $\epsilon_c \sim t/W$ with $1/W$ the typical curvature of the transversely curved sheet is the equivalent of the classical buckling of perfect cylindrical shells where the linear stability analysis predicts $\epsilon_c = 1/\sqrt{3(1-\nu^2)}t/R$ [42] with ν denoting the Poisson coefficient of the shell and $1/R$ its curvature. Notably, the above-mentioned formula infamously overestimates the buckling load of cylindrical shells [44] because those systems are extremely imperfection sensitive [45,46]. This affects the reliability of thin shell structures, even for precisely manufactured shells, such as soda cans [47] and space rockets [44]. In contrast, the present paper identifies a strain threshold ϵ_c that leads to robust scaling regimes observed in experiments and simulations. This hints that buckling by self-contact of a curved sheet is relatively insensitive to imperfections. We attribute this to a buckling threshold reached locally, contrarily to cans for which buckling can be triggered at any weak spot. Furthermore, although the buckling of cans by external loads leads to collapsing, buckling by self-contact offers more control with a self-regulation of the

internal stress after the transition. This provides an interesting strategy to reliably control buckling events in functional and smart materials by choosing geometries close to the transition to take advantage of the fast morphology switch.

APPENDIX A: TOY MODEL AND THEORETICAL STATE DIAGRAM

The sheet surface can be described by a curvature function $c(u)$ (see Fig. 4). For a simple scaling approach, we reduce the sheet kinematics to two degrees of freedom for the transverse curvature:

- (1) c_0 for the curvature at midlength, and
- (2) c_e for the curvature at the free end $u = \pm L/2$.

To estimate the elastic energy of the sheet, we will consider the first-order contributions from the quadratic variables obtained from the two degrees of freedom c_0 and c_e .

For the transverse bending energy, we assume that the energy comes from the two contributions,

$$\mathcal{E}_{b,\perp} \sim \alpha ELW t^3 c_0^2 + \beta ELW t^3 c_e^2, \quad (\text{A1})$$

in which E is the Young's modulus, $L \times W \times t$ are the physical dimensions of the sheet, and α, β are two numeric prefactors that could be determined with a more detailed analysis of the sheet's state. In the following, the numerical prefactors will be omitted, given that we are discussing scaling relations here.

To estimate the longitudinal bending energy, we use the relation that gives the vertical displacement of the sheet,

$$Z(u, v) = \frac{1}{2} c(u) v^2, \quad (\text{A2})$$

that gives the scaling for the typical vertical displacement $Z_0 \sim c_0 W^2$ and $Z_e \sim c_e W^2$. In a first approximation, the longitudinal curvature is

$$\frac{1}{R_{\text{long}}} \sim \frac{Z_0 - Z_e}{L^2}, \quad (\text{A3})$$

which gives an estimation for the longitudinal bending energy,

$$\mathcal{E}_{b,\parallel} \sim ELW t^3 \frac{W^4}{L^4} (c_0 - c_e)^2. \quad (\text{A4})$$

Finally, the longitudinal stretching strain is obtained by the ratio,

$$\epsilon = \frac{\sqrt{L^2 + (Z_0 - Z_e)^2} - L}{L^2}, \quad (\text{A5})$$

in which the square root estimates the excess of length at the edge of the sheet because of the relative vertical displacement $|Z_0 - Z_e|$. After a Taylor expansion for $Z_0 - Z_e \ll L$, the stretching energy is obtained with

$$\mathcal{E}_{s,\parallel} \sim ELW t \frac{W^4}{L^4} (c_0 - c_e)^4. \quad (\text{A6})$$

Using Eqs. (A1), (A4), and (A6), we get the scaling for the total energy used in the article,

$$\mathcal{E} \sim \mathcal{E}_{b,\perp} + \mathcal{E}_{b,\parallel} + \mathcal{E}_{s,\parallel}. \quad (\text{A7})$$

In Fig. 4, the sheet is represented for fairly low curvatures c_0 and c_e in order to visualize easily the sheet's shape. For self-contact, the curvature at midlength scales as $c_0 \sim 1/W$.

With the self-contact condition, the only free parameter in Eq. (4) is c_e . The local optimal for the energy is obtained from the cancellation of the derivative with respect of c_e which reads

$$t^2 c_e - t^2 \frac{W^4}{L^4} (c_0 - c_e) - 2 \frac{W^8}{L^4} (c_0 - c_e)^3 = 0. \quad (\text{A8})$$

With the variable $\delta_c = c_0 - c_e$, we get

$$2 \frac{W^8}{L^4} \delta_c^3 + t^2 \left(1 + \frac{W^4}{L^4} \right) \delta_c - t^2 c_0 = 0. \quad (\text{A9})$$

Equation (A9) is a third-order polynomial that may be written $\delta_c^3 + p\delta_c + q = 0$ to fits Cardano's formula with

$$p = \frac{L^4 t^2}{2W^8} \left(1 + \frac{W^4}{L^4} \right), \quad (\text{A10})$$

$$q = -\frac{L^4 t^2}{2W^9}, \quad (\text{A11})$$

with the condition $c_0 = 1/W$. The solution of Eq. (A9) is then,

$$\delta_c = \sqrt[3]{-\frac{q}{2} + \sqrt{\frac{q^2}{4} + \frac{p^3}{27}}} + \sqrt[3]{-\frac{q}{2} - \sqrt{\frac{q^2}{4} + \frac{p^3}{27}}}.$$

The stretching energy involves region of positive (extension) and negative (compression) strain with the same scaling magnitude,

$$\epsilon = \frac{W^4 \delta_c^2}{L^2}$$

obtained after the Taylor expansion of Eq. (A5).

If we use a stability criterion found in the simulations and consistent with the buckling of cans under axial load,

$$\epsilon < \epsilon_c, \quad \text{with } \epsilon_c = \alpha \frac{t}{W}, \quad (\text{A12})$$

with $\alpha = -0.43$, we find the phase diagram presented in Fig. 5.

The diagram $(L/t, W/t)$ shows a dark filled area (buckled I) for which the buckling criteria (A12) is satisfied. The buckled region *buckled (I)* has a tonguelike structure. Away from this region, the sheet's shape should be smooth. This is the case almost everywhere in experiments and simulations, except in the region identified by *buckled II*. Although the model does not predict stress focusing for this region, we suspect that the sheet should be buckled here as well: For each point in the region *buckled II*, one can find a point for the same W/t and a smaller length L/t in the effectively buckled region *buckled I*. Formulated the other way around: From a sheet effectively buckled in region *buckled (I)*, the sheet should remain buckled if it was longer (larger L for same W). We think that the absence of buckling in region *buckled (II)* is a flaw of our model that oversimplifies the curvature decay profiles and how L affects the scaling of the energy terms in Eq. (A7).

This model captures the thick-sheet regime and the fact that buckling does not exist below a threshold value W_c/t identified by a dashed line in the figure. The value of the prefactor $\alpha = -0.43$ sets the value of the lower bound for the transition $W_c/t = 56$ and the value of the prefactor a .

An additional red solid line is added for the transition regime in the thin-sheet limit $W/t = a \times (L/t)^{2/3}$ with $a =$

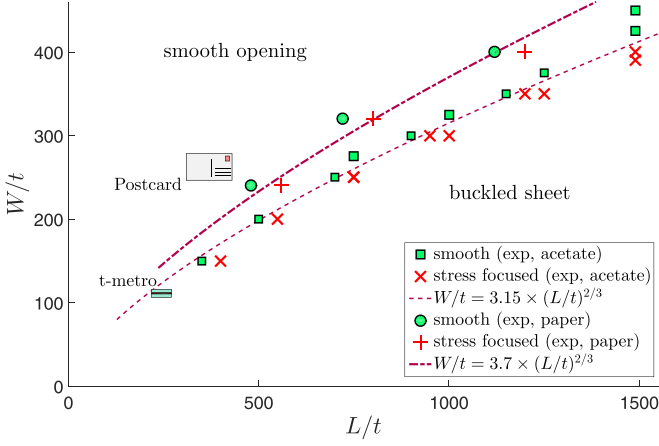


FIG. 6. State of rectangular sheets of dimensions L, W, t with contact of the midlength points. The plot represents experiments with acetate sheets and paper. The scaling law $W/t \sim (L/t)^{2/3}$ is represented with two different numerical prefactors.

3.235. This regime consistently matches with the region *buckled I* for W/t and L/t sufficiently large. This regime can be found from the model if the term for the longitudinal stretching is neglected (see the main article for the scaling argument with $L^4 \gg W^4$). This also justifies that for thick sheets, both numerical and experimental data points should shift towards smaller values of W/t .

APPENDIX B: EXPERIMENTS WITH PAPER

The self-contact experiment was performed with acetate sheets (see Fig. 1 of the main article) and standard paper sheets of thickness $t = 0.125$ mm. Figure 6 presents the state of a rectangular sheet curved by self-contact for acetate sheets [squares and crosses (x)] and paper [circles and crosses (+)].

Paper sheets buckle more easily than acetate sheets: In the region between the two scaling relations $W/t = 3.15(L/t)^{2/3}$ and $W/t = 3.7(L/t)^{2/3}$ in Fig. 6, paper sheets are buckled but acetate sheets open smoothly. This difference can be rationalized by introducing anisotropy and two different Young's moduli in Eq. (A7) for the energy,

$$\tilde{\mathcal{E}} \sim \tilde{\mathcal{E}}_{b,\perp} + \tilde{\mathcal{E}}_{b,\parallel} + \tilde{\mathcal{E}}_{s,\parallel}, \quad (\text{B1})$$

in which

$$\tilde{\mathcal{E}}_{b,\perp} \sim LWtE_{\perp}t^2(c_0^2 + c_e^2) \quad (\text{B2})$$

is the transverse bending energy,

$$\tilde{\mathcal{E}}_{b,\parallel} \sim LWtE_{\parallel}t^2\frac{W^4}{L^4}(c_0 - c_e)^2 \quad (\text{B3})$$

is the longitudinal bending energy and

$$\tilde{\mathcal{E}}_{s,\parallel} \sim LWtE_{\parallel}\frac{W^8}{L^4}(c_0 - c_e)^4 \quad (\text{B4})$$

is the longitudinal stretching energy $E_{\parallel}$ and $E_{\perp}$ are two Young's moduli with $E_{\perp} < E_{\parallel}$ in which $\parallel$ refers to the direction parallel to the paper fibers and $\perp$ the direction perpendicular (see Matsumoto *et al.*, Ref. [32]). Here, the paper sheets are cut such as the fibers are aligned with L which

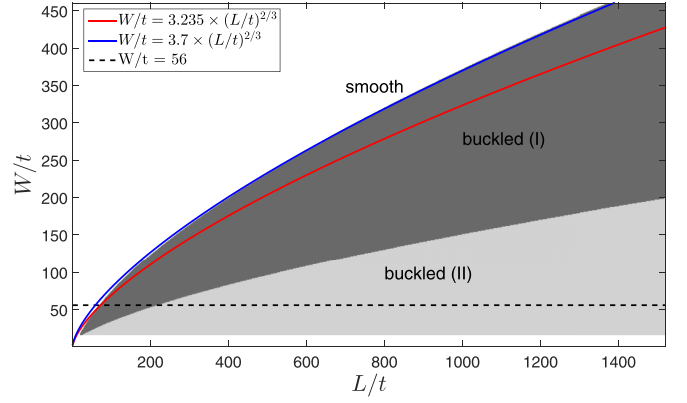


FIG. 7. State diagram predicted by the toy model with anisotropy [Eq. (B1), $E_{\perp}/E_{\parallel} = 0.71$]. The dashed line and the solid red line (with factor 3.235) are the same as in Fig. 5 for the isotropic model $E_{\parallel} = E_{\perp}$. The solid blue line is the scaling law with a prefactor 3.7 in good agreement with the experiments performed with paper sheets.

means that the smaller modulus $E_{\perp}$ appears in the transverse bending energy term. Figure 7 represents the phase diagram obtained with $E_{\perp}/E_{\parallel} = 0.71$. This value is chosen to match the thin-sheet scaling regime with prefactor 3.7 found in the experiments.

The anisotropy of paper has an impact on the buckling threshold and consistently on the postbuckling morphology. Figure 8 illustrates two morphologies obtained with a paper sheet of dimension $L = 21$, $W = 6$ cm for the paper cut with (a) the fibers in the longitudinal direction (same orientation as in Fig. 6) and (b) the fibers in the transverse direction. From a visual comparison, the sheet (a) with the fibers parallel to L is a higher state of crumpling than (b). This is confirmed by the number of d cones [four for (a) and two for (b)] and recess angle of the longitudinal centerline $v = 0$.

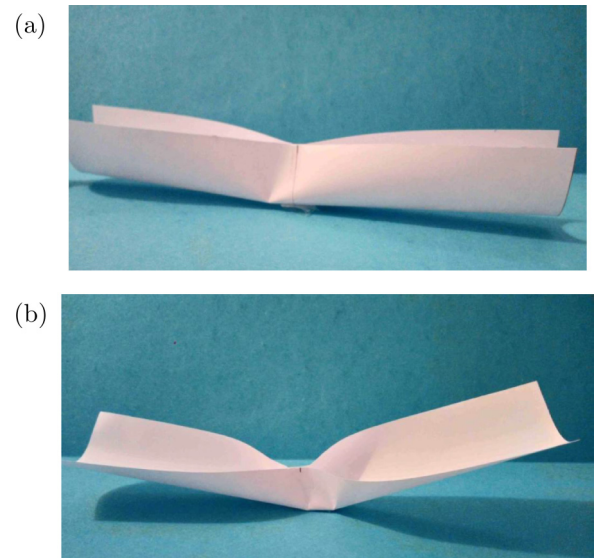


FIG. 8. Morphology of a paper sheet with midlength points contact for a sheet with (a) the paper fibers in the longitudinal direction and (b) the fibers in the transverse direction. An small adhesive strip is used to maintain the paper in self-contact.

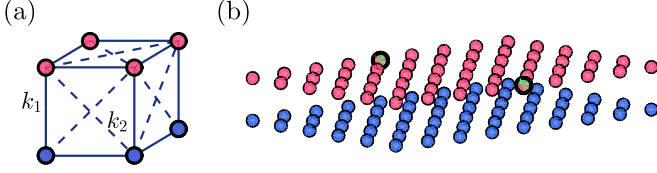


FIG. 9. (a) Schematics of the unit cell with $2 \times 2 \times 2$ vertices. The two colors indicate the upper and lower layers. The first neighbors are connected by linear springs k_1 with length 1 and the second neighbors on faces are connected by springs k_2 of length $\sqrt{2}$. (b) Representation of a sheet with $9 \times 5 \times 2$. The midlength vertices are highlighted in green.

APPENDIX C: SPRING-NETWORK MODEL

Figure 9(a) represents a unit cell made of eight vertices arranged in a cubic structure. Two types of linear springs are used

- (1) springs of length 1 with a spring constant k_1 for the first neighbors (solid lines)
- (2) springs of length $\sqrt{2}$ with a spring constant k_2 for the second-nearest neighbor (dashed lines).

1. Structure

Figure 9(b) represents a small sheet made of $9 \times 5 \times 2$ vertices. The structure is at rest without additional constraints and the elastic energy is nil (all springs have a length 1 or $\sqrt{2}$). The vertices at midlength are highlighted on the upper face. To force self-contact and bending, a supplementary spring linking the two midlength points is added.

2. Spring constants

The value of the springs constants k_1 and k_2 is set to have the same force-displacement relation for the stretching along the sides and the diagonals of the cubic cells. Figure 10 presents the displacements of two points at a distance r for an applied force F and $-F$ in the cases where the two points are in the longitudinal direction $\vec{x}$ and in the diagonal direction $\vec{x} + \vec{y}$. The superimposition of the simulation results for the force applied in the longitudinal and the diagonal directions is obtained for $k_1/k_2 = \sqrt{2}$.

3. Mechanical thickness

The thickness of the spring network is calibrated by the simulation of stretching and bending tests. Figure 11 represents a sheet made of $81 \times 31 \times 2$ vertices submitted to (a) an in-plane stretching force F_S and to (b) an out-of-plane bending force F_B .

For an elastic plate of dimension $L \times W \times t$, the stretching force-displacement relation is given by

$$F_S = E \frac{Wt}{L} \delta_S, \quad (C1)$$

with δ_S as the displacement parallel to the applied force F_S and t the thickness.

The measurement of the displacement orthogonal to the stretching direction δ_S^* defines the Poisson ratio $\nu = \delta_S^*/\delta_S =$

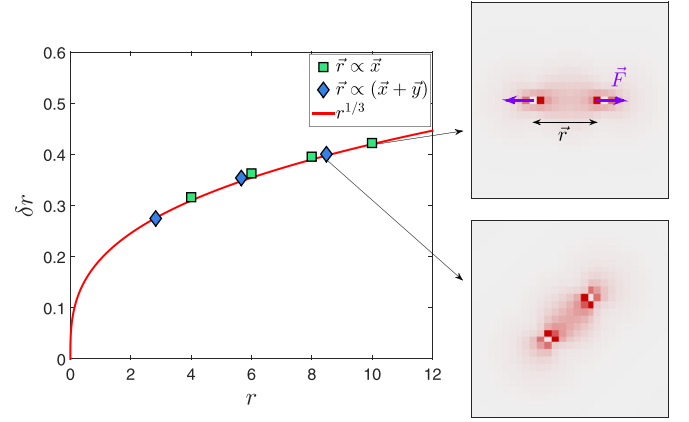


FIG. 10. Displacement δr of a pair of points separated by a distance r for a stretching force of magnitude F . The size of the sheet is $81 \times 81 \times 2$ vertices, and r is small enough ($r \leq 10$) to avoid finite-size effect of the sheet. The displacement is computed for the force F either in a longitudinal direction (squares, F parallel to springs k_1) or in a diagonal direction (diamonds, F parallel to springs k_2). The inserted images represent the elastic energy for the 15×15 vertices in the middle of the sheet.

0.29. The bending force-displacement relation reads

$$F_B = \frac{3EI}{L^3} \delta_B \quad \text{with } I = \frac{EWt^3}{12(1-\nu^2)}. \quad (C2)$$

Combining Eqs. (C1) and (C2), one can get the ratio,

$$\frac{F_B}{F_S} = \frac{1}{4(1-\nu^2)} \left(\frac{t}{L} \right)^2 \frac{\delta_B}{\delta_S}. \quad (C3)$$

One obtains the effective thickness of the two-layer sheet,

$$t = \sqrt{4(1-\nu^2)L^2 \frac{F_B}{F_S} \frac{\delta_S}{\delta_B}}. \quad (C4)$$

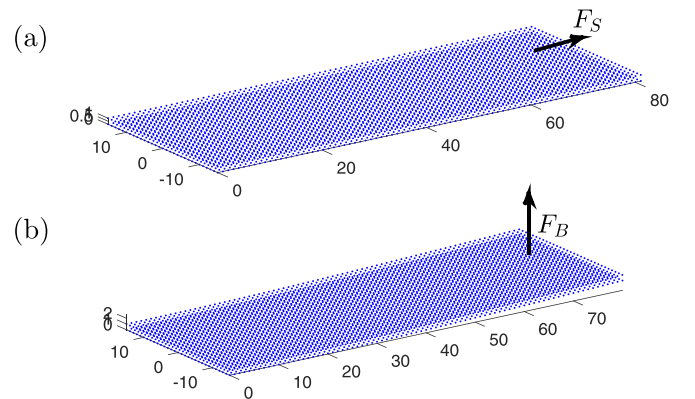


FIG. 11. Sheet made of $81 \times 31 \times 2$ vertices submitted to (a) a stretching deformation and (b) a bending deformation. The forces are as follows: such as $F_S/F_B = L^2$ ($L = 80$), and they are distributed on the width of the strip (only one force vector is represented for clarity). The displacements for the extension (a) and the deflection (b) are used to calibrate the effective thickness of the sheet according to Eq. (C4).

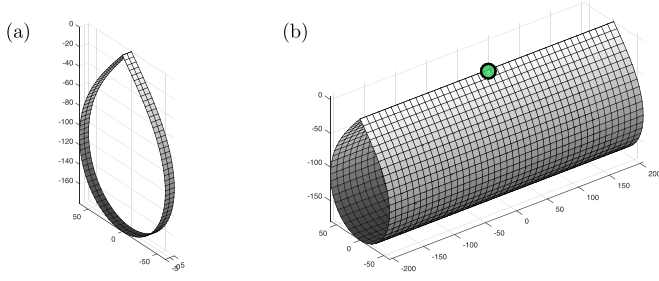


FIG. 12. (a) Short-length sheet with $11 \times 471 \times 2$ vertices with midlength points contact. (b) Sheet obtained by longitudinal replication of a transverse profile $1 \times 471 \times 2$ from (a). The sheet has a total of $1701 \times 471 \times 2$ vertices.

The simulation presented in Fig. 11 is performed with $F_B/F_S = (1/L)^2$ with $L = 80$ for the 80 intervals between the 81 vertices on the longitudinal direction. The amplitudes obtained in the simulation are $\delta_S = 1.0897$ and $\delta_B = 1.7224$. According to Eq. (C4), the thickness of the spring model is $t = 1.52$.

4. Dynamics of the vertices and equilibrium

The dynamics of the vertices is numerically solved by iteration of the forces on each vertex. The equation of motion for each vertex is as follows:

$$m \frac{d^2 \vec{r}_i}{dt^2} = -\Gamma \frac{d\vec{r}_i}{dt} - \Sigma_j \vec{F}_{ij}, \quad (\text{C5})$$

with $\vec{r}_i$ as the location of the vertex i , $m = 1$ as the mass of the vertex, Γ as a damping parameter, and Σ_j as the force from the springs k_1 and k_2 connecting the neighbors vertices j to the vertex i . The damping coefficient Γ is in the range of 10^{-4} to 0.5, depending on the network size.

A simple first-order integration method is used to solve the equation of motion (C5). This method is known to produce inaccurate time resolution of differential equations. However, the main objective of this simulation is to identify the final state of the sheet. To minimize the number of iterative steps to converge towards equilibrium, a large integration step is taken with $\Delta t = 1$ for spring constants $k_1 = 0.3568$ and $k_2 = 0.2523$, satisfying the condition $k_1 = \sqrt{2}k_2$. A very small random noise is added to the position (on the order of 10^{-6} as compared to 1 the typical distance between two vertices) in order to avoid symmetry-breaking degeneracy issues.

The sheet profiles are first computed for short-length sheets. Figure 12(a) presents the stationary configuration of a sheet made of $11 \times 471 \times 2$ vertices after multiple iteration of the equation of motion for each vertices (2×10^4 iteration steps). In the limit of small length, the sheet configuration matches with Euler's elastica model for slender structures (pure bending limit). Figure 12(b) presents the initial state of the sheet's configuration with $1701 \times 471 \times 2$ obtained by replication of the transverse profile in Fig. 12(a). The simulation of the equation of motion is performed by maintaining a contact between the two midlength points marked by a circle in Fig. 12(b).

Figure 13 presents the configuration of a sheet of dimensions $1401 \times 401 \times 2$ vertices during the simulation after

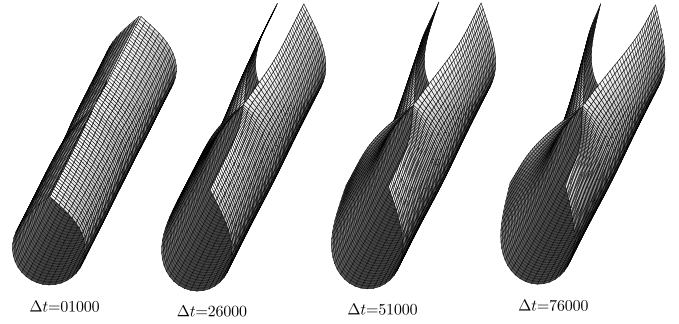


FIG. 13. Sequence showing the evolution of a sheet of $1401 \times 401 \times 2$ vertices during the simulation of the spring-network model. Time-step $\Delta t/1000 = 1, 26, 51, 76$.

a number of iteration steps Δt . In principle, the simulations should be run until equilibrium is reached. Because simulations are time consuming, they are stopped when buckling and stress focusing appear: The goal of the simulation is to identify the existence of buckling and not to follow in detail the postbuckling equilibrium shape. In Fig. 3(c) of the main article, the simulation points for the smooth opening (circles) are obtained for sheets at mechanical equilibrium. For the data points corresponding to buckled sheets (squares and star), equilibrium is not reached for some of the largest sheets. It does not affect the validity of the threshold since the minimal stretching keeps on decreasing during stress focusing.

5. Dynamics and threshold

Figure 14 presents the evolution of the minimal strain $\epsilon_{uu} \times (W/t)$ for three sheet sizes as a function of the simulation time step. The three sheets have 471 vertices in the transverse direction and 1701, 1751, and 1801 vertices in the

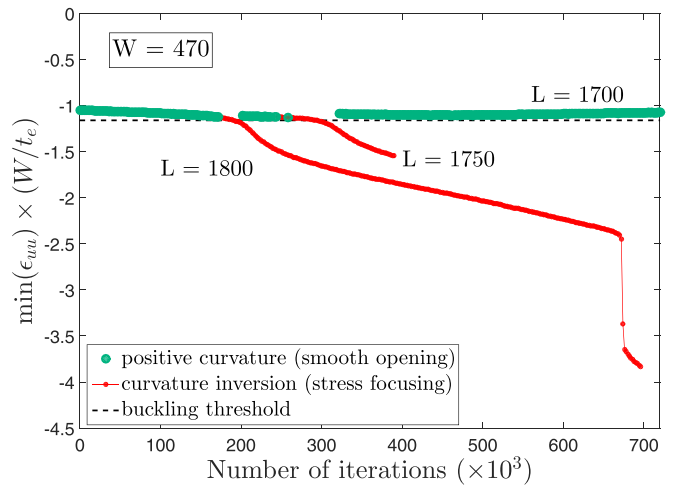


FIG. 14. Evolution of the minimal strain (longitudinal compression) during three simulations with 471 vertices for the transverse direction and 1701, 1751, and 1800 vertices for the longitudinal direction. The dashed line is the threshold value -1.16 . A video file, see the Supplemental Material [41], shows the evolution of the sheet with $L = 1800$. The abrupt transition corresponds to the transition between four d cones and two d cones.

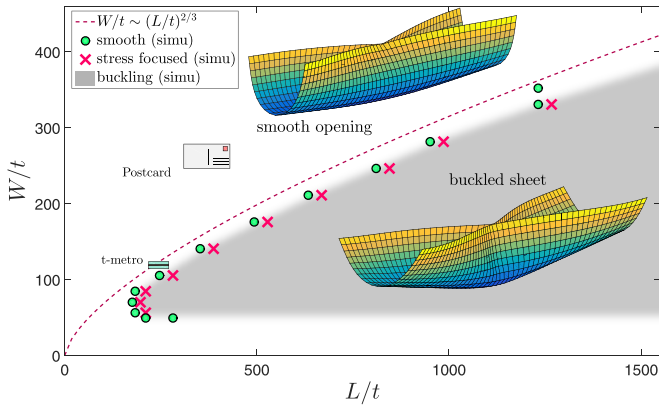


FIG. 15. Representation of the simulation results to obtain the parameter region (L/t , W/t) with stress focusing. The experimental points are not presented to avoid confusion. The dashed line is the same as in Fig. 2 of the main article.

longitudinal direction. The data for 1751 vertices is shifted horizontally for clarity.

For the shortest sheet $1701 \times 471 \times 2$, the normalized strain stays above the threshold marked by a dashed line. For 1751 and 1801, the strain is decreasing with the iteration number (opening of the sheet represented in Fig. 13). The color and symbol change indicates a sign change for the curvature of the sheet transverse profiles at $u = \delta/2$ [see criteria in Fig. 3(c) of the main article and Fig. 3(b) for the definition of δ].

The simulation with 1801 vertices explores a longer simulation time, and a second and more abrupt transition is observed after 675×10^3 time steps and the minimal strain

almost double. This second transition is associated with the traveling and merging of the stress focusing points [Fig. 3(d) with the separation δ]. The video in the Supplemental Material [41] shows the evolution of the sheet profile $1751 \times 471 \times 2$ and the two transitions (first transition around 300×10^3 and the second, more abrupt transition at 675×10^3).

Although it is close, the curvature inversion does not exactly match with the threshold -1.24 . One should expect that the compression threshold depends on the local curvature of the sheet. Since the location of the minimal strain is not self-similar [see Fig. 3(d) of the main article with $\delta(W)$], one should expect that the appropriate normalization of ϵ_{uu} is not exactly W/t but a slightly modified function that accounts for the parameter δ at the second order. Besides, the simulation includes added noise, which could explain fluctuating dynamics, notably if an instability is about to be reached (see the plot for 1851 vertices with the curvature sign that fluctuates around $50\text{--}100 \times 10^3$ time steps).

The simulation $1701 \times 471 \times 2$ was performed from an initial configuration taken from the sheet $1751 \times 471 \times 2$ by removing 25 vertices on both ends, just before the transition. This is why the minimum strain increases for the simulation $1701 \times 471 \times 2$. This strategy was performed to be sure that the smooth opening is stable for the configuration $1701 \times 471 \times 2$.

6. State diagram

The simulation of the spring-network model for different sheets sizes is used to establish the buckling region in parameters (L/t , W/t). Figure 15 is the same figure that Fig. 3 in the main article excepted that the simulation data are presented. The experimental data for the acetate sheets are removed from the plot (those data points are distributed around the dashed line).

- [1] J. W. Strutt, On the bending and vibration of thin elastic shells, especially of cylindrical form, *Proc. R. Soc. London* **45**, 105 (1889).
- [2] T. A. Witten, Stress focusing in elastic sheets, *Rev. Mod. Phys.* **79**, 643 (2007).
- [3] E. Cerda and L. Mahadevan, Conical Surfaces and Crescent Singularities in Crumpled Sheets, *Phys. Rev. Lett.* **80**, 2358 (1998).
- [4] S. Chaïeb, F. Melo, and J.-C. G  minard, Experimental Study of Developable Cones, *Phys. Rev. Lett.* **80**, 2354 (1998).
- [5] E. Cerda, S. Chaïeb, F. Melo, and L. Mahadevan, Conical dislocations in crumpling, *Nature (London)* **401**, 46 (1999).
- [6] M. Ben Amar and Y. Pomeau, Crumpled paper, *Proc. R. Soc. London., Ser. A* **453**, 729 (1997).
- [7] A. Lobkovsky, S. Gentges, H. Li, D. Morse, and T. A. Witten, Scaling properties of stretching ridges in a crumpled elastic sheet, *Science* **270**, 1482 (1995).
- [8] E. Sultan and A. Boudaoud, Statistics of Crumpled Paper, *Phys. Rev. Lett.* **96**, 136103 (2006).
- [9] C. A. Andresen, A. Hansen, and J. Schmittbuhl, Ridge network in crumpled paper, *Phys. Rev. E* **76**, 026108 (2007).
- [10] A. S. Balankin, D. S. Ochoa, I. A. Miguel, J. P. Ortiz, and M.   . M. Cruz, Fractal topology of hand-crumpled paper, *Phys. Rev. E* **81**, 061126 (2010).
- [11] A. D. Cambou and N. Menon, Three-dimensional structure of a sheet crumpled into a ball, *Proc. Natl. Acad. Sci. USA* **108**, 14741 (2011).
- [12] A. B. Croll, T. Twohig, and T. Elder, The compressive strength of crumpled matter, *Nat. Commun.* **10**, 1502 (2019).
- [13] O. Gottesman, E. Efrati, and S. M. Rubinstein, Furrows in the wake of propagating d-cones, *Nat. Commun.* **6**, 7232 (2015).
- [14] J. Chopin and A. Kudrolli, Disclinations, e-cones, and their interactions in extensible sheets, *Soft Matter* **12**, 4457 (2016).
- [15] J. A. Rogers and Y. Huang, A curvy, stretchy future for electronics, *Proc. Natl. Acad. Sci. USA* **106**, 10875 (2009).
- [16] J. Luo, H. D. Jang, T. Sun, L. Xiao, Z. He, A. P. Katsoulidis, M. G. Kanatzidis, J. M. Gibson, and J. Huang, Compression and aggregation-resistant particles of crumpled soft sheets, *ACS Nano* **5**, 8943 (2011).
- [17] S. Mao, Z. Wen, H. Kim, G. Lu, P. Hurley, and J. Chen, A general approach to one-pot fabrication of crumpled graphene-

- based nanohybrids for energy applications, *ACS Nano* **6**, 7505 (2012).
- [18] X. Ma, M. R. Zachariah, and C. D. Zangmeister, Crumpled nanopaper from graphene oxide, *Nano Lett.* **12**, 486 (2012).
- [19] W. Yan, W.-Y. He, Z.-D. Chu, M. Liu, L. Meng, R.-F. Dou, Y. Zhang, Z. Liu, J.-C. Nie, and L. He, Strain and curvature induced evolution of electronic band structures in twisted graphene bilayer, *Nat. Commun.* **4**, 2159 (2013).
- [20] P. M. Reis, A perspective on the revival of structural (in) stability with novel opportunities for function: From buckliphobia to buckliphilia, *J. Appl. Mech.* **82**, 111001 (2015).
- [21] D. P. Holmes, Elasticity and stability of shape-shifting structures, *Curr. Opin. Colloid Interface Sci.* **40**, 118 (2019).
- [22] R. D. Schroll, E. Katifori, and B. Davidovitch, Elastic Building Blocks for Confined Sheets, *Phys. Rev. Lett.* **106**, 074301 (2011).
- [23] J. F. Fuentealba, O. Albarrán, E. Hamm, and E. Cerda, Transition from isometric to stretching ridges in thin elastic films, *Phys. Rev. E* **91**, 032407 (2015).
- [24] A. Boudaoud, P. Patrício, Y. Couder, and M. B. Amar, Dynamics of singularities in a constrained elastic plate, *Nature (London)* **407**, 718 (2000).
- [25] B. Roman and A. Pocheau, Stress Defocusing in Anisotropic Compaction of Thin Sheets, *Phys. Rev. Lett.* **108**, 074301 (2012).
- [26] E. Cerda, L. Mahadevan, and J. M. Pasini, The elements of draping, *Proc. Natl. Acad. Sci. USA* **101**, 1806 (2004).
- [27] A. E. Lobkovsky and T. A. Witten, Properties of ridges in elastic membranes, *Phys. Rev. E* **55**, 1577 (1997).
- [28] E. Cerda and L. Mahadevan, Geometry and Physics of Wrinkling, *Phys. Rev. Lett.* **90**, 074302 (2003).
- [29] H. Vandeparre, M. Piñeirua, F. Brau, B. Roman, J. Bico, C. Gay, W. Bao, C. N. Lau, P. M. Reis, and P. Damman, Wrinkling Hierarchy in Constrained Thin Sheets from Suspended Graphene to Curtains, *Phys. Rev. Lett.* **106**, 224301 (2011).
- [30] T. Barois, L. Tadríst, C. Quiliet, and Y. Forterre, How a Curved Elastic Strip Opens, *Phys. Rev. Lett.* **113**, 214301 (2014).
- [31] V. Pini, J. J. Ruz, P. M. Kosaka, O. Malvar, M. Calleja, and J. Tamayo, How two-dimensional bending can extraordinarily stiffen thin sheets, *Sci. Rep.* **6**, 29627 (2016).
- [32] D. Matsumoto, T. G. Sano, and H. Wada, Pinching an open cylindrical shell: Extended deformation and its persistence, *Europhys. Lett.* **123**, 14001 (2018).
- [33] M. Taffetani, F. Box, A. Neveu, and D. Vella, Limitations of curvature-induced rigidity: How a curved strip buckles under gravity, *Europhys. Lett.* **127**, 14001 (2019).
- [34] B. Audoly and Y. Pomeau, *Elasticity and Geometry: from Hair Curls to the Non-linear Response of Shells* (Oxford University Press, 2010).
- [35] E. M. Kramer and T. A. Witten, Stress Condensation in Crushed Elastic Manifolds, *Phys. Rev. Lett.* **78**, 1303 (1997).
- [36] H. S. Seung and D. R. Nelson, Defects in flexible membranes with crystalline order, *Phys. Rev. A* **38**, 1005 (1988).
- [37] W. A. Curtin and H. Scher, Mechanics modeling using a spring network, *J. Mater. Res.* **5**, 554 (1990).
- [38] M. Ostoja-Starzewski, Lattice models in micromechanics, *Appl. Mech. Rev.* **55**, 35 (2002).
- [39] T. Omori, T. Ishikawa, D. Barthès-Biesel, A.-V. Salsac, J. Walter, Y. Imai, and T. Yamaguchi, Comparison between spring network models and continuum constitutive laws: Application to the large deformation of a capsule in shear flow, *Phys. Rev. E* **83**, 041918 (2011).
- [40] B. A. DiDonna, Scaling of the buckling transition of ridges in thin sheets, *Phys. Rev. E* **66**, 016601 (2002).
- [41] See Supplemental Material at <http://link.aps.org/supplemental/10.1103/PhysRevE.104.014801> for a representation of the strain map and the sheet profile during a simulation of a sheet with $1801 \times 471 \times 2$ vertices.
- [42] J. W. Hutchinson, Knockdown factors for buckling of cylindrical and spherical shells subject to reduced biaxial membrane stress, *Int. J. Solids Struct.* **47**, 1443 (2010).
- [43] M. Alava and K. Niskanen, The physics of paper, *Rep. Prog. Phys.* **69**, 669 (2006).
- [44] J. P. Peterson, P. Seide, and V. I. Weingarten, Buckling of thin-walled circular cylinders, Report No. NASA-SP-8007 (NASA, 1968), <https://ntrs.nasa.gov/citations/19690013955>.
- [45] J. Singer, J. Arbocz, and T. Weller, *Buckling Experiments: Experimental Methods in Buckling of Thin-Walled Structures: Shells, Built-Up Structures, Composites and Additional Topics*, Vol. 2 (John Wiley & Sons, New York, 2002).
- [46] S. Gerasimidis, E. Virot, J. W. Hutchinson, and S. M. Rubinstein, On establishing buckling knockdowns for imperfection-sensitive shell structures, *J. Appl. Mech.* **85**, 091010 (2018).
- [47] E. Virot, T. Kreilos, T. M. Schneider, and S. M. Rubinstein, Stability Landscape of Shell Buckling, *Phys. Rev. Lett.* **119**, 224101 (2017).