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Improvement of Silicon Nanotweezers Sensitivity for Mechanical Characterization of Biomolecules Using Closed-Loop Control

Nicolas Lafitte, Yassine Haddab, Yann Le Gorrec, Member, IEEE, Hervé Guillou, Momoko Kumemura, Laurent Jalabert, Dominique Collard, Member, IEEE, and Hiroyuki Fujita, Member, IEEE

Abstract—In this paper, we show that closed-loop control can be advantageously used for the characterization of mechanical prop-8 erties of biomolecules using silicon nanotweezers (SNT). SNT have already been used in open-loop mode for the characterization of 9 mechanical properties of DNA molecules. Up to now, such an ap-10 11 proach allows the detection of stiffness variations equivalent to 12 about 15 DNA molecules. Here, it is shown that this resolution is inversely proportional to the resonance frequency of the whole 13 system and that real-time feedback control with state observer 14 15 can drastically improve the performances of the tweezers used as 16 biosensors. Such improvement is experimentally validated in the 17 case of the manipulation of *fibronectin* molecules. The results are promising for the accurate characterization of biopolymers such 18 as DNA molecules. 19

Index Terms-Biosensor, MEMS tweezers, parameter detection, 20 sensitivity improvement, state feedback. 21

I. INTRODUCTION

ECENT developments in micronano manipulation tools 23 R have revealed crucial information on the mechanical be-24 25 havior of biomolecules [1]–[3]. These manipulations generally performed on a single molecule have given the quantitative 26 data needed to elucidate fundamental biological processes as 27 DNA wrapping [4] and replication [5], or cell cytoskeleton dy-28 namics through actin filament [6], and microtubule mechanical 29 responses [7]. 30

Several methods are already available for performing bio-31 logical experiments at the molecular level: magnetic tweezers 32 [8], [9], optical tweezers [10], AFM cantilevers [11], and mi-33 crofibers [2], [12]. A single filamentary macromolecule, such 34 as DNA, is first attached to a surface at one end, and to a bead 35

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are sensed by optical measurements. Nevertheless, the real-time 37 operation and the routine implementation of these techniques 38 remain difficult to achieve, as they require complex experimen-39 tal procedures. In this respect, microelectromechanical systems 40 (MEMS) offer an advantage for systematic analysis since accu-41 rate molecular level tools (actuator, end effectors, and sensor) 42 can be integrated on a MEMS platform. Furthermore, these de-43 vices can be produced at low cost with batch fabrication, as have 44 the microgrippers developed to grab microsized objects [13] or 45 cells [14]. However, up to now, the performances of such tools 46 are not sufficient to deal with single molecule characterization. 47 In previous reports, silicon nanotweezers (SNT) were designed 48 and fabricated to trap DNA molecules [15], [16] and character-49 ize DNA bundles [17] and reactions on DNA [18]. The stiffness 50 of DNA bundles was derived from the open-loop measurement 51 of the mechanical resonance frequency shift. The current sens-52 ing resolution is in the range of $1 \text{ mN} \cdot \text{m}^{-1}$, which corresponds 53 to approximatively 15 λ -DNA molecules¹ [20]. This resolution 54 is limited by the design of the tweezers and the noise of the elec-55 tronic instrumentation. To improve the overall system sensitivity 56 towards lower stiffness, the primary route consists of reducing 57 the tweezers stiffness to bring the sensor and measured object 58 characteristics close to the same range. Unfortunately, such a 59 design would result in a system that would be too fragile to be 60 processed and utilized. 61

or a flexible cantilever at the other end. Forces or displacements

Our approach consists of scaling down the resonance fre-62 quency of the closed-loop system to enhance the sensitivity to 63 SNT stiffness change. After a proper identification of the SNT 64 model, the feedback control has been simulated, optimized, and 65 then implemented. This control strategy has been evaluated with 66 thin fibronectin molecular bundles trapped between SNT tips, 67 and we propose to discuss the advantages and the limits of such 68 closed-loop operations. 69

This paper is organized as follows. Section II describes 70 the tweezers, the sensing principle, and provides an example 71 of monitoring the DNA trapping. The dynamic model of the 72 tweezers is then detailed in Section III-A. The closed-loop 73 control strategy and its potential to improve the system sen-74 sitivity are discussed in Sections III-B and III-C. Section IV 75 details the experimental implementation and provides infor-76 mation on *fibronectin* molecules measurement with different 77

 $^1\lambda\text{-DNA}$ is DNA of the bacteriophage λ having a contour length of $16.4~\mu\mathrm{m}$ and a stiffness in elastic regime of 66 μ N · m⁻¹ in low ionic solution [19].

36

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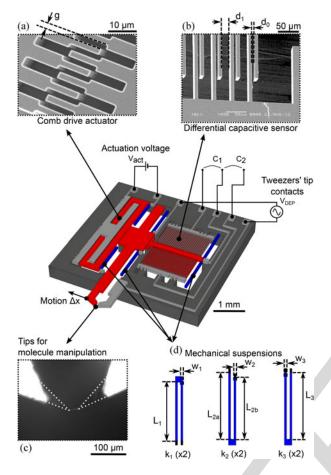


Fig. 1. SNT description. 3-D view of the device whose overall dimensions are $4.5 \times 4.5 \times 0.5$ mm. The mobile parts are shown in red and the mechanical suspensions in blue. (a) Scanning electronic microscopic (SEM) image of the actuator consisting of 880 pairs of interdigitated combs separated by a 2 - μ m gap, and organized in two series. (b) SEM image of the integrated sensor of two opposing series of 30 combs. (c) Microscope image of the tweezers tips with a DNA bundle in between. (d) Design of the three sets of suspensions, k_1 for the actuator polarization, k_2 for the mobile arm, and k_3 for central sensor plate.

closed-loop settings. Closing remarks and perspectives are givenin Section V.

II. DESCRIPTION OF THE DEVICE AND ITS USE FOR THE MECHANICAL CHARACTERIZATION OF BIOMOLECULES BUNDLES

83 A. Device Description

1) Design: Fig. 1 shows a three-dimensional illustration of 84 the SNT device. It consists of two arms terminated in sharp tips 85 that act as electrodes to trap biomolecules by dielectrophoresis 86 (DEP) forces [15]. One tip is fixed; the other one is moved by 87 an electrostatic actuator. The motion of the mobile electrode 88 89 is measured by capacitances whose value linearly varies with electrode displacement; two variable capacitances are mounted 90 in differential mode [21]. 91

2) Electrostatic Actuation: The mobile arm is actuated by
 attractive electrostatic forces generated in an interdigitated comb
 drive [see Fig. 1(a)], one of the most widely used architecture

in MEMS design [22], [23]. The maximal stroke is limited 95 compared to that achievable by a parallel plate actuator but the 96 generated force only depends on the actuation voltage and not on 97 the electrode position, thus, enabling a simpler actuation control. 98 The force is given by (1) where ε_0 is the vacuum permittivity (we 99 consider the relative permittivity of air = 1), N_a is the number 100 of comb fingers, t is the device thickness, g is the lateral gap 101 between fingers, and $V_{\rm act}$ is the actuation voltage. Due to its 102 capacitive structure, the comb drive generates attractive forces 103 between its two electrodes 104

$$F_{\rm es} = \frac{1}{2} \frac{\varepsilon_0 \times N_a \times t}{g} V_{\rm act}^2 = \alpha_{\rm es} \times V_{\rm act}^2.$$
(1)

3) Mechanical Structure: The mobile part of the tweezers 105 is linked to the silicon frame by flexible beams [see Fig. 1(d)]. 106 Commonly integrated in mechanical microsystems [23], [24], 107 folded beam springs are designed to minimize beam areas, 108 decrease their mechanical stiffness and enhance displacement 109 ranges [25]. In the SNT design of Fig. 1, three sets of folded 110 beam suspensions support the mobile part of the system (the 111 comb-drive actuator, the mobile tip, and the capacitive sensor) 112 and provide the electrical connections for the actuation and the 113 sensing. The springs are arranged symmetrically along the ac-114 tuation and sensing axis to minimize any rotation, and their sum 115 gives the total stiffness k of the device. 116

A highly compliant system is required to sense the mechanical characteristic of trapped molecules on the tweezers response. 118 On the other hand, a minimum stiffness is mandatory, 1) to endure the fabrication processes and manipulations, 2) to support 120 the mobile system weight, and 3) to prevent sticking due to attractive surface forces between the comb-drive actuator and the capacitive sensor electrodes. 123

4) Displacement/Velocity Sensing: The tip position is mea-124 sured by a capacitive sensor designed in a triplate configuration 125 with transverse combs [see Fig. 1(b)] [26]. The central elec-126 trode, linked to the mobile arm, moves in between two fixed 127 electrodes creating the two differential capacitances C_1 and C_2 128 whose difference, ΔC , is related to the displacement x. For 129 small displacements (i.e., x is much smaller than the gaps be-130 tween electrodes d_0 and d_1), ΔC is proportional to x [17], N_b 131 is the number of capacitance electrodes, and L the length of the 132 electrodes 133

$$\Delta C = C_1 - C_2 \tag{2}$$

$$\simeq 2\varepsilon_0 N_b Lt \left(\frac{1}{d_0^2} - \frac{1}{d_1^2}\right) x = \beta_{\rm C} \times x \tag{3}$$

5) *Electronic Read-Out:* The tweezers motion is sensed by 134 the measurement of the differential capacitance ΔC (2) through 135 current sensing. In dynamic mode, a dc voltage (V_{polar}) is ap-136 plied on the mobile central electrode, whose motion generates 137 dynamic currents i_1 and i_2 flowing through the capacitances C_1 138 and C_2 , respectively, [27], [28]. The resulting currents related 139 to the motion velocity are converted into voltages V_1 and V_2 by 140 two low-noise current-to-voltage (A/V) preamplifiers (Signal 141 Recovery, model 5182). The low input impedance of the pream-142 plifier (virtual ground) ensures an accurate current conversion 143 [17], [29]. 144

TABLE I NUMERICAL VALUES OF DIMENSIONS AND PARAMETERS OF THE SNT

$t \ (\mu m)$	30
Comb drive actuator	
Na	440
$g (\mu m)$	2
Mechanical suspensions	
$L_{1}, L_{2a} (\mu m)$	900
L_{2b}, L_{3} (µm)	1000
$w_1, w_2, w_3 \ (\mu m)$	15
Capacitive sensor	
N_b	30
$L(\mu m)$	585
$d_0 (\mu m)$	5
$d_1 \ (\mu \mathrm{m})$	20

The device dimensions are defined in Fig. 1.

Finally, in harmonic mode, when the tweezers is moved by a sinusoidal actuation, a lock-in amplifier (NF, model LI 5640) performs the low-noise detection of the differential signal (amplitude and phase of $V_1 - V_2$) at the actuation frequency. The reference signal is the motion frequency imposed by the actuation voltage.

All dimensions and parameters that has been considered for a proper model of the SNT are summed up in the TableI. Furthermore, Young's modulus and the density of the silicon are, respectively, 165 GPa and 2329 kg \cdot m⁻³, and the permittivity ε_0 is 8.85×10^{-12} kg⁻¹ \cdot m⁻³ \cdot A² \cdot s⁴.

156 B. Monitoring Biomolecules Manipulation With SNT

This Section presents the way that the SNT are used as biosensors and points out the measurement sensitivity requirements. This example is illustrated with DNA but the same technique has been applied to gelatin [30] and microtubules [31].

1) DNA Bundle Trapping by DEP: The trapping is achieved by applying an ac electric field ($E = 1 \text{ MV} \cdot \text{m}^{-1}$) between the two opposing tips of the tweezers [32] once partially immersed in the solution. The retrieving of the DNA bundle is routinely performed from a droplet solution [15], [17], and single molecule trapping has even been demonstrated with a pulsed DEP signal [16].

In [20], an improved method is proposed in which the tips are introduced into an open fluidic cavity optimized to guaranty a stable meniscus. The SNT position is fixed and the cavity is mounted on a *XYZ* stage controlled by programmable robot for repeatable and controlled tip immersion, as illustrated in Fig. 2(a)–(c).

2) Real-Time Monitoring: The SNT characterization allows 174 monitoring of the DNA trapping in real time. The characteriza-175 tion of the SNT in real time allows the monitoring of the DNA 176 trapping. The number of trapped DNA molecules during the 177 DEP is controlled by the evolution of the resonance frequency 178 and quality factor of the system (Tweezers+DNA bundle). The 179 SNT are modeled by a linear second-order model as depicted in 180 Fig. 2(d). The resonance frequency, $f_{\rm R}$, and the quality factor 181

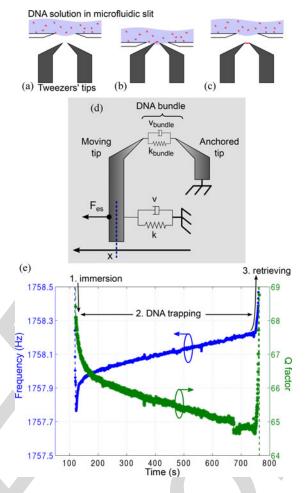


Fig. 2. λ -DNA molecules trapping with SNT. The first three sketches illustrate the key phases of the experiment: (a) the immersion of the tips in the DNA solution, (b) the DNA trapping by DEP, and (c) the retrieving of DNA bundle between the tips. (d) Equivalent dynamic model of the SNT with bundle. (e) Evolution of the *SNT*+*bundle* resonance frequency and *Q* factor during the trapping.

Q of the SNT+bundle damped oscillator are defined by

$$f_{\rm R}(t) = \frac{1}{2\pi} \sqrt{\frac{(k + k_{\rm bundle}(t))}{M}} \tag{4}$$

$$Q(t) = \frac{\sqrt{(k + k_{\text{bundle}}(t)) \times M}}{(\nu + \nu_{\text{bundle}}(t))}$$
(5)

where M is the mass of the movable tip, k is the stiffness of the suspensions, and ν is the equivalent viscous losses. k_{bundle} and ν_{bundle} are the time varying bundle stiffness and viscosity; the mass of the bundle is neglected compared to M.

As shown in Fig. 2(e), as the DEP trapping progresses, the 187 system moves to higher resonance frequencies indicating an increase in the bundle stiffness according to (4). At the same time, 189 the signal amplitude at resonance, proportional to the quality 190 factor Q, is decreasing, revealing higher viscous losses ν_{bundle} 191 as the bundle forms, in accordance with (5). 192

In this experiment, the resonance frequency experiences a 193 shift of 0.4 Hz that corresponds to an increase in the stiffness of 194 $12 \text{ mN} \cdot \text{m}^{-1}$, i.e., a bundle containing $\sim 200 \lambda$ -DNA molecules 195

196 [19]. (For the sake of clarity, it should be noted that the SNT197 used in this trapping experiment has different characteristics198 than those used in the feedback approach of Sections III and IV)

199 3) Sensor Resolution: Accurate biosensing by SNT requires high-resolution measurements. Thus, special care in signal con-200 ditioning (detection of small currents < 10 pA), noise reduction, 201 and control of the experimental conditions (e.g., the stability 202 of the meniscus air/biological liquid/tweezers) have allowed to 203 sense resonance frequency shifts of 25 mHz and quality factor 204 205 changes of 0.2. Based on this resolution, the minimum change that can be sensed due to the stiffness variation is in the range 206 of 15 molecules of λ -DNA [20]. 207

After trapping, the bundle of molecules is mechanically stimulated (in air or in a biological solution) with a sinusoidal signal at the resonance frequency of the system. Slight changes in the frequency response permit a fine characterization of the evolution of the mechanical properties of the molecule bundle revealing biological interactions.

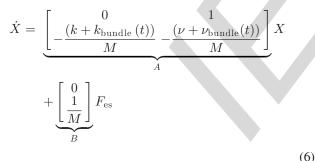
214 III. DYNAMIC MODELING AND IMPROVEMENT OF THE 215 PARAMETRIC SENSITIVITY BY A FEEDBACK APPROACH

In this Section, we describe how to reduce the closed-loop resonance frequency of the tweezers in order to improve its sensitivity to molecule stiffness variations.

219 A. Dynamic Modeling and Open-Loop Sensitivity

220 to Stiffness Variations

1) Modeling: Newton's second law applied to the damped oscillator formed by the *SNT+bundle* of Fig. 2(d) can be formulated by



224 and

$$y = \underbrace{\left[\begin{array}{c} 0 \ 1 \end{array}\right]}_{C} X \tag{7}$$

where $A \in \mathbb{R}^2, B \in \mathbb{R}^{(2,1)}, C \in \mathbb{R}^{(1,2)}, X$ is the state vector (displacement & velocity, $X = \begin{pmatrix} x \\ \dot{x} \end{pmatrix}$), and y is the output vector or the measurement.

2) *Identification:* Prior to the development of an accurate control strategy, the model parameters [M, k, v], actuation, and sensor gains of (1)–(5)] need to be extracted from device measurements. These parameters are identified through standard recursive approach with least squares method from SNT responses to small signal actuation with a 20 V offset. Identified parameters are summarized in Table II.

TABLE II
THEORETICAL AND IDENTIFIED MODEL PARAMETER VALUES OF THE SNT

Parameters	Theoretical	Identified
Mechanical parameters		
M (kg)	360×10^{-9}	360×10^{-9}
$k (N \cdot m^{-1})$	49.3^{a}	24.9
$\nu (Ns \cdot m^{-1})$	-	100×10^{-6}
Comb-drive actuator		
α_{es} (N · V ⁻²)	29.2×10^{-9a}	$35.5 imes 10^{-9}$
Capacitive sensor		
$\beta_{\rm C} ({\rm F} \cdot {\rm m}^{-1})$	412.2×10^{-9a}	227.5×10^{-9}

^a These values are calculated from the designed springs and electrodes widths and does not take into account the significant beam narrowing during etching processes.

3) Open-Loop Sensitivity to Stiffness Variation: The minimum number of molecules that can be discerned by resonance frequency shifts due to the molecular stiffness variation is linked to the SNT sensitivity to stiffness variation Δk , i.e., 238

$$\frac{\partial f_{\rm R}}{\partial k} = \frac{1}{4\pi\sqrt{kM}} \text{ or } \frac{1}{8\pi^2 M f_{\rm R}}.$$
 (8)

This sensitivity is equal to $26.4 \,\mathrm{Hz}/(\mathrm{N} \cdot \mathrm{m}^{-1})$ with the iden-239 tified parameters of Table II. It appears that for a given molecu-240 lar stiffness change, Δk , the effect on the resonance frequency 241 variation $\Delta f_{\rm R}$ is enhanced with low SNT stiffness, k, and, there-242 fore, for low resonance frequency $f_{\rm R}$. In relation to Fig. 2, the 243 current experimental resolution in the resonant frequency shift 244 measurement is close to 25 mHz, a value that corresponds to 245 $1 \text{ mN} \cdot \text{m}^{-1}$ or 15 λ -DNA molecules. From (8), a reduction of 246 $f_{\rm R}$ by one order of magnitude; i.e. 2 orders of magnitude for 247 k, would improve the detection threshold down to the single 248 molecule level. 249

The development of a new MEMS device having a very low 250 stiffness ($< 0.2 \,\mathrm{N} \cdot \mathrm{m}^{-1}$ remains extremely challenging in terms 251 of fabrication and operation. Thus, we propose to reduce the 252 overall stiffness by closed-loop control. In such a control strategy, the physical system (i.e., the SNT) remains the same but the 254 closed-loop system *SNT combined with the feedback law* will 255 have the desired designed dynamic behavior. 250

B. Improvement of the Parametric Sensitivity257by a Feedback Approach258

1) Control Strategy: The control design strategy is depicted 259 in the flowchart of Fig. 3. The aim of the feedback is to reduce the 260 resonance frequency of the closed-loop system. To this end, we 261 use a state feedback eigenstructure assignment in order to assign 262 both the closed-loop resonance frequency and damping factor. 263 This state feedback is then implemented using a Luenberger-264 kind observer. The originality of the observer we use is that it 265 has to be designed in order to neither deteriorate the closed-loop 266 sensitivity nor amplify the noise. 267

2) Eigenstructure Assignment Using State Feedback: The 268 system is under its controllable canonical state-space represen-269 tation (A, B, C) given by (6) and (7). The state feedback gain 270 vector $L = (l_1, l_2) \in \mathbb{R}^{(1,2)}$ is designed to assign the desired 271

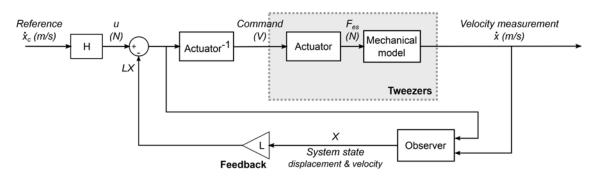


Fig. 3. Feedback control scheme. Gray box, called *Tweezers*, comprises the mechanical and the actuator models. Actuator square law needs to be inverted in order to linearize the control signal. The state feedback gain vector L and the feedforward gain H are designed to assign the desired closed-loop poles and to ensure unitary static gain. The observer is a Luenberger observer reconstructing the state of the system X for the state feedback. The sensor model is not shown here. The device output voltage is converted into the tip velocity by dividing by the sensor gain $1/\beta_C$.

TABLE III Pole Placement of the Closed-Loop System and State Feedback Gain L According to the Resonance Frequency Reduction \boldsymbol{n}

Freq.	$L \ \left([\mathrm{N} \cdot \mathrm{m}^{-1}], [\mathrm{N} \mathrm{s} \cdot \mathrm{m}^{-1}] \right)$	Poles
$f_{\rm R}$	(0, 0)	$-138.9 \pm 8.3 \times 10^{3} i$
$f_{\rm R} / 1.2$	$(-7.6, -16.7 \times 10^{-6})$	$-115.7 \pm 6.9 \times 10^{3} i$
$f_{\rm R} / 2.0$	$(-18.7, -50.0 \times 10^{-6})$	$-69.4 \pm 4.2 \times 10^{3} i$
$f_{\rm R} / 3.2$	$(-22.5, -68.8 \times 10^{-6})$	$-43.4 \pm 2.6 \times 10^3 i$
$f_{\rm R} / 5.0$	$(-23.9, -80.0 \times 10^{-6})$	$-27.8\pm1.7\times10^{3}i$
$f_{\rm R} / 10$	$(-24.7, -90.0 \times 10^{-6})$	$-13.9 \pm 0.8 \times 10^{3} i$

closed-loop poles. The actuation of the system $F_{\rm es}$ is equal to u - LX, and the closed-loop system model becomes

$$\dot{X} = \begin{bmatrix} 0 & 1\\ -\frac{k+l_1}{M} & -\frac{\nu+l_2}{M} \end{bmatrix} X + \begin{bmatrix} 0\\ 1\\ \frac{1}{M} \end{bmatrix} u.$$
(9)

Thus, the closed-loop resonance frequency and sensitivity are given by

$$f_{\rm R-cl} = \frac{1}{2\pi} \sqrt{\frac{(k+l_1)}{M}}$$
 (10)

$$\frac{\partial f_{\rm R-cl}}{\partial k} = \frac{1}{4\pi\sqrt{(k+l_1)\,M}} = \frac{1}{8\pi^2 M f_{\rm R-cl}}.$$
 (11)

The state feedback gain L is, then, determined in order to assign the closed-loop poles in such a way that the system resonates at a lower frequency than the natural frequency of the tweezers (see Table III). L is computed to scale down this resonant frequency (initially 1323.6 Hz), by a given factor depending on the desired sensitivity enhancement while keeping the damping factor unchanged.

283 When the closed-loop resonance frequency is scaled down by 284 a factor n compared to the SNT natural resonance frequency, 285 the sensitivity to small stiffness variation is improved with the 286 same ratio according to (11). Fig. 4 shows the computed fre-287 quency shift of the closed-loop system caused by a signifi-288 cant stiffness variation for different frequency scaling down 289 factor, n.

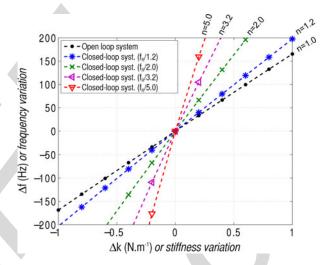


Fig. 4. Evolution of the resonance frequency shift $\Delta f_{\rm R}$ and $\Delta f_{\rm R-cl}$ with the added stiffness Δk variation for different frequency reduction ratio n.

C. Observer Design

The implementation of the aforementioned state feedback is 291 done through the use of a Luenberger observer. The observer is 292 designed such that it preserves the good closed-loop sensitivity 293 to parametric variations and it does not amplify too much noise. 294

The poles of the observer have then to be carefully chosen. 295 We used the following guidelines. 296

- The observer is designed to be faster than the original 297 device—with poles at least two times faster than the poles 298 of the system in open loop (cf., Table III).
- (2) The observer poles are chosen not too fast in order to 300 avoid excessive noise amplification. 301
- (3) The final location of the observer poles is chosen in order to preserve the sensitivity of the system to stiffness variations (by using root locus).
 304

In the following section, simulations illustrate 1) the relevance 305 of the method and 2) the importance of the design of the observer 306 towards the enhancement of the sensitivity. 307

Closed loop f_R/5.0 (Δf=125 Hz) Closed loop f_R/2.0 (Δf=50 Hz) Open loop f_R (Δf=25 Hz) 100 Magnitude(dB) 80 60 40 20 90 Phase(deg.) 45 0 -45 1000 10000 100 Frequency (Hz)

Fig. 5. Simulated frequency results for open-loop driven system and two different closed-loop systems. The three systems undergo a variation of k the system stiffness. k vary from -0.5 to 0.5 N/m by 0.1 N/m step. In black line, the tweezer transfer function, i.e., the open-loop system. In blue line, the implemented (n = 2) closed-loop system transfer function. The system resonates at a frequency twice lower. In red line, the implemented (n = 5) closed-loop system transfer function. The system resonates at a frequency twice lower.

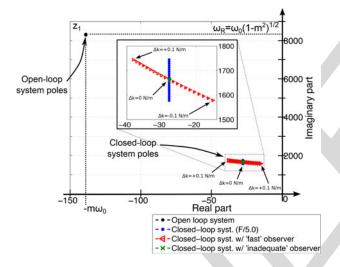


Fig. 6. Root locus of the closed-loop systems with n = 5 and with different observers. Complex conjugate poles are plotted in the positive imaginary half s plan. The parameter k varies from -0.1 to 0.1 N/m by 0.01 N/m steps. The black dots show the poles of the open-loop driven system. The ideal pole path of the closed-loop system due to the k variations are plotted with blue dots (i.e., no observer). For different observers designs, poles dependencies (and equivalent resonance frequency $\omega_{\text{R}} = 2\pi f_{\text{R}}$ and damping m) on the tweezers stiffness k are shown by red triangle and green cross plots. NB: Green cross dots path are imperceptible since they are at this scale all concentrate in the middle.

308 D. Simulation Results

From parameters identified in Table II, the feedback scheme 309 (see Fig. 3) has been first programmed and tested under MAT-310 LAB/Simulink without implementing the observer (i.e., all the 311 state variables are supposed to be measured). Fig. 5 compares 312 the resonance frequency shift between the open-loop driven sys-313 tem and the closed-loop systems computed for two resonance 314 frequency reduction factors set to n = 2 and 5. As expected, the 315 sensitivity of the closed-loop system is improved proportionally 316 to *n*. 317

In a second instance, we implemented two different kinds of observers (with different dynamics); Fig. 6 illustrates two cases.

TABLE IV POLE PLACEMENT OF THE OBSERVER FOR STATE FEEDBACK IMPLEMENTATION

Open-loop system (Tweezers)	$z_{1,2} = -138.9 \pm 8.3 \times 10^3 i$
closed-loop system $(n = 5)$	$z_{1,2} = -27.8 \pm 1.7 \times 10^3 i$
Observer 1 (fast)	$z_{1,2} = -277.8 \pm 0.2 \times 10^3 i$
Observer 2 (slow)	$z_{1,2} = -27.8 \pm 0.2 \times 10^3 i$
Observer 3 (inadequate)	$z_{1,2} = -277.8 \pm 16.7 \times 10^3 i$

This Figure shows that the root locus (due to k parameter variations) is deformed by the observer dynamics. The sensitivity of 321 the resonance frequency to stiffness variations is given by the 322 amplitude of the imaginary axis variations. Table IV reports the 323 poles of the observers for which simulation results are presented. 324

In the insert of Fig. 6, the poles of the system implemented 325 with the observer named *inadequate* do not move compared to 326 the ideal root locus (ideal state feedback case). If the poles of 327 the observer are inappropriately chosen, the sensitivity of the 328 poles to variations of k is significantly reduced. Observer poles 329 close to the real axis, i.e., with an imaginary part close to 0, 330 are the more appropriate to reconstruct the displacement state 331 and to preserve the sensitivity of the system to variations of k. 332 The socalled *fast* observer allows us to increase the resonance 333 frequency shift ($\omega_{\rm R} = 2\pi f_{\rm R}$ in Fig. 6) close to the performances 334 theoretically expected. 335

IV. EXPERIMENTAL APPLICATION FOR EXTRA-STIFFNESS 336 CHARACTERIZATION 337

The previous method has been experimentally tested with the 338 mechanical characterization of biomolecules of *fibronectin*. The 339 physical sensor is the SNT, while the feedback controller and 340 the observer are implemented in a dSPACE prototyping board. 341

A. Experimental Protocol

Fibronectin is a protein, 100 nm in length, of the extracellular matrix which plays a major role in cell adhesion, growth, 344

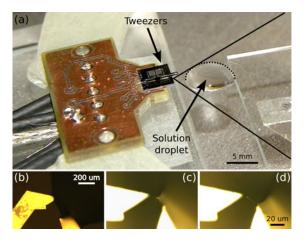


Fig. 7. Experimental setup for the trapping of *fibronectin* molecules. (a) The SNT are fixed. A droplet of *fibronectin* solution is dropped on a glass slice which is mounted on the XYZ stage. (b)–(d) Sequences of the molecule combing. Under optical microscopy, the tweezers' tips are retrieved from the solution meniscus. A bundle of *fibronectin* is released between the tips.

migration, and differentiation [33]. These molecules are used to 345 polymerize the fibrillar matrix at the surface of the cell. We have 346 used this property to trap by combing a bundle of *fibronectin*. 347 After immersion of the tips in the solution, they are retrieved by 348 the side forming a polymerized bundle of molecules (see Fig. 7). 349 350 The closed-loop systems are evaluated with *fibronectin* bundle in air, in dry condition. This stable condition was preferred 351 to the immersed one as we can avoid any effect of liquid surface 352 tension stability that can interfere with the sole system response. 353 For each case, a sinusoidal reference of $0.02 V_{AC}$ with an offset 354 of $20 V_{DC}$ is applied to the system and the oscillation frequen-355 cies are spanned from 1000 to 2000 Hz with a step of 2 Hz. 356 Then, the measurement is performed with the lock-in amplifier. 357 Two sets of experiments have been performed with two differ-358 ent bundles. From the resonance frequency shifts in open loop, 359 the stiffness of the first bundle is $11.8~\mathrm{N}\cdot\mathrm{m}^{-1},$ and $4.3~\mathrm{N}\cdot\mathrm{m}^{-1}$ 360 361 for the second one. In total, 22 resonance responses are recorded and compared. 362

363 B. Results and Discussions

Experimental Results: Fig. 8 shows the frequency responses of four different systems with and without a bundle of *fibronectin*.

In Fig. 8(a), the responses of the open-loop driven SNT are 367 given. The dashed-line curve is the reference curve, which cor-368 responds to the frequency response of the SNT alone driven in 369 open loop. The solid line is the response of the tweezers with 370 the bundle trapped in between the tips. The resonance frequency 371 increases and the amplitude decreases due to the additional stiff-372 ness and loss contributions of the molecules. In Fig. 8(b)-(d), 373 the results of the closed-loop systems with desired reduction 374 ratios are of 1.1, 1.2 and 1.5. 375

Resonance responses are fitted with a second-order resonator model and characteristics such as the resonance frequency are extracted. The Table V sums up the resonance frequencies for the experiment with/without the first bundle for five different

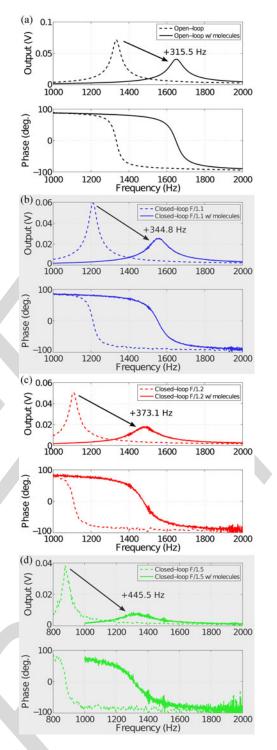


Fig. 8. Frequency responses of different implemented closed-loop systems for 0.02 V sinusoidal signal with an offset of 20 V. In dotted lines, the responses of the systems with the bundle of molecules. In plain lines, the responses of the systems without molecules. (a) Open-loop driven systems. (b) Closed-loop systems implemented with $f_{\rm R}/1.1$. (c) Closed-loop systems implemented with $f_{\rm R}/1.2$. (d) Closed-loop systems with $f_{\rm R}/1.5$.

systems. Through reducing the resonance frequency of the system, the measured results prove the increase of the frequency shift due to the extra-stiffness of the bundle. 382

Moreover Fig. 9 tends to 1) confirm with second set of experiments, the enhancement of the sensitivity through the proposed 384

TABLE V EXPERIMENTAL RESULTS FOR THE BUNDLE 1 (RELATED TO FIG. 8)

System (n)	Freq. w/o bundle (Hz)	Freq. w/ bundle (Hz)	Freq. shift (Hz)
Open loop (1.0)	1333.6	1649.1	+315.5
Closed-loop (1.1)	1211.1	1555.9	+344.8
Closed-loop (1.2)	1109.1	1482.2	+373.1
Closed-loop (1.5)	881.9	1327.4	+445.5
Closed-loop (2.0)	653.9	1175.6	+521.7

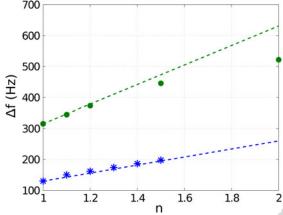


Fig. 9. Synthesis of the experimental results. In abscissa, is informed the reduction factor (n) applied to the resonance frequency for the closed-loop system. The shift Δf is the resonance frequency difference due to the presence of the bundle of molecules in between the tips. The dotted lines represent the theoretical improvement expected. The two sets of results obtained with two different bundles are plotted. The green dots correspond to the experiment of Fig 8 and TableV (bundle of $11.8 \text{ N} \cdot \text{m}^{-1}$ stiffness). The blue stars corresponds to the second bundle of $4.3 \text{ N} \cdot \text{m}^{-1}$ stiffness.

feedback strategy and 2) show a good agreement with the theory. 385 Small differences are discussed in the following section. 386

2) Discussion: The synthesis of Fig. 9 demonstrates the en-387 hancement of the frequency shifts in agreement with the theory. 388 However, the enhancement for closed-loop system with a reduc-389 390 tion factor of 2.0 is lower than expected, and the implementation 391 for a higher reduction factor has not been achieved. In this case, the system tends to become unstable. We mainly associated this 392 phenomenon to the observer implementation and to the presence 393 of some delays in the feedback loop. 394

395 We have demonstrated previously that observer's dynamic changes the root locus of system's poles, the behavior of the 396 closed-loop system under parameter variations and impacts the 397 performances of the method. Furthermore, delays have been 398 characterized due to D/A and A/D converters dead times and to 399 the phase shift brought by the current amplifiers. Current works 400 401 are devoted to the precise characterization of such delays and to their integration in the control design process. 402

V. CONCLUSION AND FUTURE WORK 403

The present study investigates the relevance of a feedback 404 approach in the improvement of the performances of MEMS 405 tweezers for the detection and the mechanical characterization of 406 407 biological molecules. An improvement of 50% on the sensitivity

to mechanical stiffness variations has been reached theoretically 408 and experimentally on *fibronectin* bundles through the emulation 409 of a more compliant system. The stiffness have been brought 410 down to 10.5 N/m instead of originally 24.5 N/m. 411

The main drawback of this approach is the sensitivity of 412 the observer to the stiffness variations. Unusually, the poles of 413 the observer are uncontrollable resulting from the separation 414 principle. During the sensing of the molecules stiffness, the 415 separation principle is not satisfied anymore and the observer 416 poles can become unstable. We are actually working on self-417 scheduled observers aiming at adapting the observer dynamics 418 with respect to sensed parametric variations. It should solve the 419 main drawback of this promising approach and allows us to go 420 one step further into the theoretical closed-loop performances. 421

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Improvement of Silicon Nanotweezers Sensitivity for Mechanical Characterization of Biomolecules Using Closed-Loop Control

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Abstract—In this paper, we show that closed-loop control can be advantageously used for the characterization of mechanical prop-8 erties of biomolecules using silicon nanotweezers (SNT). SNT have already been used in open-loop mode for the characterization of 9 mechanical properties of DNA molecules. Up to now, such an ap-10 11 proach allows the detection of stiffness variations equivalent to 12 about 15 DNA molecules. Here, it is shown that this resolution is inversely proportional to the resonance frequency of the whole 13 system and that real-time feedback control with state observer 14 15 can drastically improve the performances of the tweezers used as 16 biosensors. Such improvement is experimentally validated in the 17 case of the manipulation of *fibronectin* molecules. The results are promising for the accurate characterization of biopolymers such 18 as DNA molecules. 19

Index Terms-Biosensor, MEMS tweezers, parameter detection, 20 sensitivity improvement, state feedback. 21

I. INTRODUCTION

ECENT developments in micronano manipulation tools 23 R have revealed crucial information on the mechanical be-24 25 havior of biomolecules [1]–[3]. These manipulations generally performed on a single molecule have given the quantitative 26 data needed to elucidate fundamental biological processes as 27 DNA wrapping [4] and replication [5], or cell cytoskeleton dy-28 namics through actin filament [6], and microtubule mechanical 29 responses [7]. 30

Several methods are already available for performing bio-31 logical experiments at the molecular level: magnetic tweezers 32 [8], [9], optical tweezers [10], AFM cantilevers [11], and mi-33 crofibers [2], [12]. A single filamentary macromolecule, such 34 as DNA, is first attached to a surface at one end, and to a bead 35

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or a flexible cantilever at the other end. Forces or displacements 36 are sensed by optical measurements. Nevertheless, the real-time 37 operation and the routine implementation of these techniques 38 remain difficult to achieve, as they require complex experimen-39 tal procedures. In this respect, microelectromechanical systems 40 (MEMS) offer an advantage for systematic analysis since accu-41 rate molecular level tools (actuator, end effectors, and sensor) 42 can be integrated on a MEMS platform. Furthermore, these de-43 vices can be produced at low cost with batch fabrication, as have 44 the microgrippers developed to grab microsized objects [13] or 45 cells [14]. However, up to now, the performances of such tools 46 are not sufficient to deal with single molecule characterization. 47 In previous reports, silicon nanotweezers (SNT) were designed 48 and fabricated to trap DNA molecules [15], [16] and character-49 ize DNA bundles [17] and reactions on DNA [18]. The stiffness 50 of DNA bundles was derived from the open-loop measurement 51 of the mechanical resonance frequency shift. The current sens-52 ing resolution is in the range of $1 \text{ mN} \cdot \text{m}^{-1}$, which corresponds 53 to approximatively 15 λ -DNA molecules¹ [20]. This resolution 54 is limited by the design of the tweezers and the noise of the elec-55 tronic instrumentation. To improve the overall system sensitivity 56 towards lower stiffness, the primary route consists of reducing 57 the tweezers stiffness to bring the sensor and measured object 58 characteristics close to the same range. Unfortunately, such a 59 design would result in a system that would be too fragile to be 60 processed and utilized. 61

Our approach consists of scaling down the resonance fre-62 quency of the closed-loop system to enhance the sensitivity to 63 SNT stiffness change. After a proper identification of the SNT 64 model, the feedback control has been simulated, optimized, and 65 then implemented. This control strategy has been evaluated with 66 thin fibronectin molecular bundles trapped between SNT tips, 67 and we propose to discuss the advantages and the limits of such 68 closed-loop operations. 69

This paper is organized as follows. Section II describes 70 the tweezers, the sensing principle, and provides an example 71 of monitoring the DNA trapping. The dynamic model of the 72 tweezers is then detailed in Section III-A. The closed-loop 73 control strategy and its potential to improve the system sen-74 sitivity are discussed in Sections III-B and III-C. Section IV 75 details the experimental implementation and provides infor-76 mation on *fibronectin* molecules measurement with different 77

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 $^{^1\}lambda\text{-DNA}$ is DNA of the bacteriophage λ having a contour length of $16.4~\mu\mathrm{m}$ and a stiffness in elastic regime of 66 μ N · m⁻¹ in low ionic solution [19].

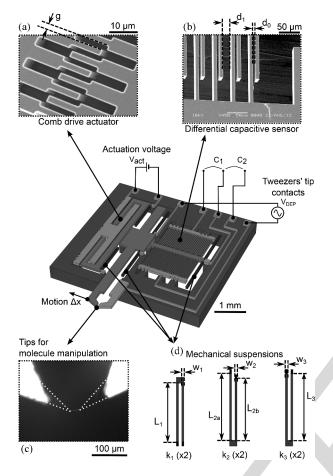


Fig. 1. SNT description. 3-D view of the device whose overall dimensions are $4.5 \times 4.5 \times 0.5$ mm. The mobile parts are shown in red and the mechanical suspensions in blue. (a) Scanning electronic microscopic (SEM) image of the actuator consisting of 880 pairs of interdigitated combs separated by a 2 - μ m gap, and organized in two series. (b) SEM image of the integrated sensor of two opposing series of 30 combs. (c) Microscope image of the tweezers tips with a DNA bundle in between. (d) Design of the three sets of suspensions, k_1 for the actuator polarization, k_2 for the mobile arm, and k_3 for central sensor plate.

closed-loop settings. Closing remarks and perspectives are given
 in Section V.

II. DESCRIPTION OF THE DEVICE AND ITS USE FOR THE MECHANICAL CHARACTERIZATION OF BIOMOLECULES BUNDLES

83 A. Device Description

1) Design: Fig. 1 shows a three-dimensional illustration of 84 the SNT device. It consists of two arms terminated in sharp tips 85 that act as electrodes to trap biomolecules by dielectrophoresis 86 (DEP) forces [15]. One tip is fixed; the other one is moved by 87 an electrostatic actuator. The motion of the mobile electrode 88 is measured by capacitances whose value linearly varies with 89 electrode displacement; two variable capacitances are mounted 90 in differential mode [21]. 91

2) Electrostatic Actuation: The mobile arm is actuated by
attractive electrostatic forces generated in an interdigitated comb
drive [see Fig. 1(a)], one of the most widely used architecture

in MEMS design [22], [23]. The maximal stroke is limited 95 compared to that achievable by a parallel plate actuator but the 96 generated force only depends on the actuation voltage and not on 97 the electrode position, thus, enabling a simpler actuation control. 98 The force is given by (1) where ε_0 is the vacuum permittivity (we 99 consider the relative permittivity of air = 1), N_a is the number 100 of comb fingers, t is the device thickness, g is the lateral gap 101 between fingers, and $V_{\rm act}$ is the actuation voltage. Due to its 102 capacitive structure, the comb drive generates attractive forces 103 between its two electrodes 104

$$F_{\rm es} = \frac{1}{2} \frac{\varepsilon_0 \times N_a \times t}{g} V_{\rm act}^2 = \alpha_{\rm es} \times V_{\rm act}^2.$$
(1)

3) Mechanical Structure: The mobile part of the tweezers 105 is linked to the silicon frame by flexible beams [see Fig. 1(d)]. 106 Commonly integrated in mechanical microsystems [23], [24], 107 folded beam springs are designed to minimize beam areas, 108 decrease their mechanical stiffness and enhance displacement 109 ranges [25]. In the SNT design of Fig. 1, three sets of folded 110 beam suspensions support the mobile part of the system (the 111 comb-drive actuator, the mobile tip, and the capacitive sensor) 112 and provide the electrical connections for the actuation and the 113 sensing. The springs are arranged symmetrically along the ac-114 tuation and sensing axis to minimize any rotation, and their sum 115 gives the total stiffness k of the device. 116

A highly compliant system is required to sense the mechanical characteristic of trapped molecules on the tweezers response. 118 On the other hand, a minimum stiffness is mandatory, 1) to endure the fabrication processes and manipulations, 2) to support 120 the mobile system weight, and 3) to prevent sticking due to attractive surface forces between the comb-drive actuator and the capacitive sensor electrodes. 123

4) Displacement/Velocity Sensing: The tip position is mea-124 sured by a capacitive sensor designed in a triplate configuration 125 with transverse combs [see Fig. 1(b)] [26]. The central elec-126 trode, linked to the mobile arm, moves in between two fixed 127 electrodes creating the two differential capacitances C_1 and C_2 128 whose difference, ΔC , is related to the displacement x. For 129 small displacements (i.e., x is much smaller than the gaps be-130 tween electrodes d_0 and d_1), ΔC is proportional to x [17], N_b 131 is the number of capacitance electrodes, and L the length of the 132 electrodes 133

$$\Delta C = C_1 - C_2 \tag{2}$$

$$\simeq 2\varepsilon_0 N_b Lt \left(\frac{1}{d_0^2} - \frac{1}{d_1^2}\right) x = \beta_{\rm C} \times x \tag{3}$$

5) *Electronic Read-Out:* The tweezers motion is sensed by 134 the measurement of the differential capacitance ΔC (2) through 135 current sensing. In dynamic mode, a dc voltage (V_{polar}) is ap-136 plied on the mobile central electrode, whose motion generates 137 dynamic currents i_1 and i_2 flowing through the capacitances C_1 138 and C_2 , respectively, [27], [28]. The resulting currents related 139 to the motion velocity are converted into voltages V_1 and V_2 by 140 two low-noise current-to-voltage (A/V) preamplifiers (Signal 141 Recovery, model 5182). The low input impedance of the pream-142 plifier (virtual ground) ensures an accurate current conversion 143 [17], [29]. 144

TABLE I NUMERICAL VALUES OF DIMENSIONS AND PARAMETERS OF THE SNT

Silicon thickness	
$t \; (\mu m)$	30
Comb drive actuator	
N_a	440
g (µm)	2
Mechanical suspensions	
$L_{1}, L_{2a} \; (\mu m)$	900
L_{2b}, L_{3} (µm)	1000
$w_1, w_2, w_3 \ (\mu m)$	15
Capacitive sensor	
N_b	30
$L (\mu m)$	585
$d_0 (\mu m)$	5
$d_1 \ (\mu \mathrm{m})$	20

The device dimensions are defined in Fig. 1.

Finally, in harmonic mode, when the tweezers is moved by a sinusoidal actuation, a lock-in amplifier (NF, model LI 5640) performs the low-noise detection of the differential signal (amplitude and phase of $V_1 - V_2$) at the actuation frequency. The reference signal is the motion frequency imposed by the actuation voltage.

All dimensions and parameters that has been considered for a proper model of the SNT are summed up in the TableI. Furthermore, Young's modulus and the density of the silicon are, respectively, 165 GPa and 2329 kg \cdot m⁻³, and the permittivity ε_0 is 8.85×10^{-12} kg⁻¹ \cdot m⁻³ \cdot A² \cdot s⁴.

156 B. Monitoring Biomolecules Manipulation With SNT

This Section presents the way that the SNT are used as biosensors and points out the measurement sensitivity requirements. This example is illustrated with DNA but the same technique has been applied to gelatin [30] and microtubules [31].

1) DNA Bundle Trapping by DEP: The trapping is achieved by applying an ac electric field ($E = 1 \text{ MV} \cdot \text{m}^{-1}$) between the two opposing tips of the tweezers [32] once partially immersed in the solution. The retrieving of the DNA bundle is routinely performed from a droplet solution [15], [17], and single molecule trapping has even been demonstrated with a pulsed DEP signal [16].

In [20], an improved method is proposed in which the tips are introduced into an open fluidic cavity optimized to guaranty a stable meniscus. The SNT position is fixed and the cavity is mounted on a *XYZ* stage controlled by programmable robot for repeatable and controlled tip immersion, as illustrated in Fig. 2(a)–(c).

2) Real-Time Monitoring: The SNT characterization allows 174 monitoring of the DNA trapping in real time. The characteriza-175 tion of the SNT in real time allows the monitoring of the DNA 176 trapping. The number of trapped DNA molecules during the 177 DEP is controlled by the evolution of the resonance frequency 178 and quality factor of the system (Tweezers+DNA bundle). The 179 SNT are modeled by a linear second-order model as depicted in 180 Fig. 2(d). The resonance frequency, $f_{\rm R}$, and the quality factor 181

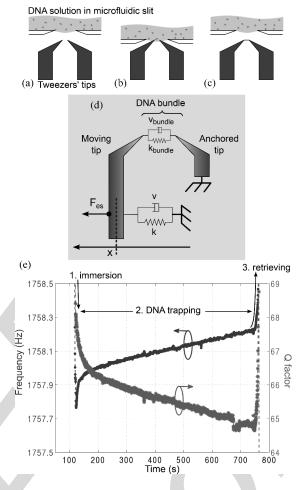


Fig. 2. λ -DNA molecules trapping with SNT. The first three sketches illustrate the key phases of the experiment: (a) the immersion of the tips in the DNA solution, (b) the DNA trapping by DEP, and (c) the retrieving of DNA bundle between the tips. (d) Equivalent dynamic model of the SNT with bundle. (e) Evolution of the *SNT*+*bundle* resonance frequency and *Q* factor during the trapping.

Q of the SNT+bundle damped oscillator are defined by

$$f_{\rm R}(t) = \frac{1}{2\pi} \sqrt{\frac{(k+k_{\rm bundle}(t))}{M}} \tag{4}$$

$$Q(t) = \frac{\sqrt{(k + k_{\text{bundle}}(t)) \times M}}{(\nu + \nu_{\text{bundle}}(t))}$$
(5)

where M is the mass of the movable tip, k is the stiffness of the suspensions, and ν is the equivalent viscous losses. k_{bundle} and ν_{bundle} are the time varying bundle stiffness and viscosity; the mass of the bundle is neglected compared to M.

As shown in Fig. 2(e), as the DEP trapping progresses, the 187 system moves to higher resonance frequencies indicating an increase in the bundle stiffness according to (4). At the same time, 189 the signal amplitude at resonance, proportional to the quality 190 factor Q, is decreasing, revealing higher viscous losses ν_{bundle} 191 as the bundle forms, in accordance with (5). 192

In this experiment, the resonance frequency experiences a 193 shift of 0.4 Hz that corresponds to an increase in the stiffness of 194 $12 \text{ mN} \cdot \text{m}^{-1}$, i.e., a bundle containing $\sim 200 \lambda$ -DNA molecules 195

[19] [19]. (For the sake of clarity, it should be noted that the SNTused in this trapping experiment has different characteristicsthan those used in the feedback approach of Sections III and IV)

199 3) Sensor Resolution: Accurate biosensing by SNT requires high-resolution measurements. Thus, special care in signal con-200 ditioning (detection of small currents < 10 pA), noise reduction, 201 and control of the experimental conditions (e.g., the stability 202 of the meniscus air/biological liquid/tweezers) have allowed to 203 sense resonance frequency shifts of 25 mHz and quality factor 204 205 changes of 0.2. Based on this resolution, the minimum change that can be sensed due to the stiffness variation is in the range 206 of 15 molecules of λ -DNA [20]. 207

After trapping, the bundle of molecules is mechanically stimulated (in air or in a biological solution) with a sinusoidal signal at the resonance frequency of the system. Slight changes in the frequency response permit a fine characterization of the evolution of the mechanical properties of the molecule bundle revealing biological interactions.

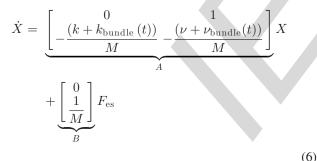
214 III. DYNAMIC MODELING AND IMPROVEMENT OF THE 215 PARAMETRIC SENSITIVITY BY A FEEDBACK APPROACH

In this Section, we describe how to reduce the closed-loop resonance frequency of the tweezers in order to improve its sensitivity to molecule stiffness variations.

219 A. Dynamic Modeling and Open-Loop Sensitivity

220 to Stiffness Variations

1) Modeling: Newton's second law applied to the damped oscillator formed by the *SNT+bundle* of Fig. 2(d) can be formulated by



224 and

$$y = \underbrace{\left[\begin{array}{c} 0 \ 1 \end{array}\right]}_{C} X \tag{7}$$

where $A \in \mathbb{R}^2, B \in \mathbb{R}^{(2,1)}, C \in \mathbb{R}^{(1,2)}, X$ is the state vector (displacement & velocity, $X = \begin{pmatrix} x \\ \dot{x} \end{pmatrix}$), and y is the output vector or the measurement.

2) *Identification:* Prior to the development of an accurate control strategy, the model parameters [M, k, v], actuation, and sensor gains of (1)–(5)] need to be extracted from device measurements. These parameters are identified through standard recursive approach with least squares method from SNT responses to small signal actuation with a 20 V offset. Identified parameters are summarized in Table II.

TABLE II THEORETICAL AND IDENTIFIED MODEL PARAMETER VALUES OF THE SNT

Parameters	Theoretical	Identified
Mechanical parameters		
M (kg)	360×10^{-9}	360×10^{-9}
$k (N \cdot m^{-1})$	49.3^{a}	24.9
$\nu (Ns \cdot m^{-1})$	-	100×10^{-6}
Comb-drive actuator		
$\alpha_{es} (N \cdot V^{-2})$	29.2×10^{-9a}	35.5×10^{-9}
Capacitive sensor		
$\beta_{\rm C} ({\rm F} \cdot {\rm m}^{-1})$	412.2×10^{-9a}	227.5×10^{-9}

^a These values are calculated from the designed springs and electrodes widths and does not take into account the significant beam narrowing during etching processes.

3) Open-Loop Sensitivity to Stiffness Variation: The minimum number of molecules that can be discerned by resonance frequency shifts due to the molecular stiffness variation is linked to the SNT sensitivity to stiffness variation Δk , i.e., 238

$$\frac{\partial f_{\rm R}}{\partial k} = \frac{1}{4\pi\sqrt{kM}} \text{ or } \frac{1}{8\pi^2 M f_{\rm R}}.$$
 (8)

This sensitivity is equal to $26.4 \,\mathrm{Hz}/(\mathrm{N} \cdot \mathrm{m}^{-1})$ with the iden-239 tified parameters of Table II. It appears that for a given molecu-240 lar stiffness change, Δk , the effect on the resonance frequency 241 variation $\Delta f_{\rm R}$ is enhanced with low SNT stiffness, k, and, there-242 fore, for low resonance frequency $f_{\rm R}$. In relation to Fig. 2, the 243 current experimental resolution in the resonant frequency shift 244 measurement is close to 25 mHz, a value that corresponds to 245 $1 \text{ mN} \cdot \text{m}^{-1}$ or 15 λ -DNA molecules. From (8), a reduction of 246 $f_{\rm R}$ by one order of magnitude; i.e. 2 orders of magnitude for 247 k, would improve the detection threshold down to the single 248 molecule level. 249

The development of a new MEMS device having a very low 250 stiffness ($< 0.2 \,\mathrm{N} \cdot \mathrm{m}^{-1}$ remains extremely challenging in terms 251 of fabrication and operation. Thus, we propose to reduce the 252 overall stiffness by closed-loop control. In such a control strategy, the physical system (i.e., the SNT) remains the same but the 254 closed-loop system *SNT combined with the feedback law* will 255 have the desired designed dynamic behavior. 250

B. Improvement of the Parametric Sensitivity257by a Feedback Approach258

1) Control Strategy: The control design strategy is depicted 259 in the flowchart of Fig. 3. The aim of the feedback is to reduce the 260 resonance frequency of the closed-loop system. To this end, we 261 use a state feedback eigenstructure assignment in order to assign 262 both the closed-loop resonance frequency and damping factor. 263 This state feedback is then implemented using a Luenberger-264 kind observer. The originality of the observer we use is that it 265 has to be designed in order to neither deteriorate the closed-loop 266 sensitivity nor amplify the noise. 267

2) Eigenstructure Assignment Using State Feedback: The 268 system is under its controllable canonical state-space represen-269 tation (A, B, C) given by (6) and (7). The state feedback gain 270 vector $L = (l_1, l_2) \in \mathbb{R}^{(1,2)}$ is designed to assign the desired 271

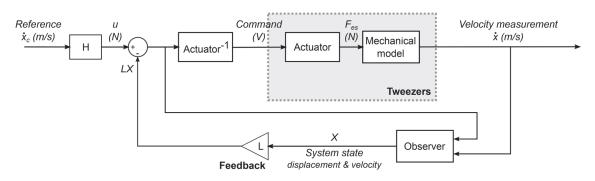


Fig. 3. Feedback control scheme. Gray box, called *Tweezers*, comprises the mechanical and the actuator models. Actuator square law needs to be inverted in order to linearize the control signal. The state feedback gain vector L and the feedforward gain H are designed to assign the desired closed-loop poles and to ensure unitary static gain. The observer is a Luenberger observer reconstructing the state of the system X for the state feedback. The sensor model is not shown here. The device output voltage is converted into the tip velocity by dividing by the sensor gain $1/\beta_C$.

TABLE III POLE PLACEMENT OF THE CLOSED-LOOP SYSTEM AND STATE FEEDBACK GAIN L According to the Resonance Frequency Reduction n

Freq.	$L~([\mathrm{N}\cdot\mathrm{m}^{-1}],[\mathrm{Ns}\cdot\mathrm{m}^{-1}])$	Poles
$f_{\rm R}$	(0, 0)	$-138.9 \pm 8.3 \times 10^{3} i$
$f_{\rm R} / 1.2$	$(-7.6, -16.7 \times 10^{-6})$	$-115.7 \pm 6.9 \times 10^{3} i$
$f_{\rm R} / 2.0$	$(-18.7, -50.0 \times 10^{-6})$	$-69.4 \pm 4.2 \times 10^{3} i$
$f_{\rm R} / 3.2$	$(-22.5, -68.8 \times 10^{-6})$	$-43.4 \pm 2.6 \times 10^3 i$
$f_{\rm R} / 5.0$	$(-23.9, -80.0 imes 10^{-6})$	$-27.8 \pm 1.7 \times 10^{3} i$
$f_{\rm R}/10$	$(-24.7, -90.0 \times 10^{-6})$	$-13.9 \pm 0.8 \times 10^{3} i$

closed-loop poles. The actuation of the system F_{es} is equal to u - LX, and the closed-loop system model becomes

$$\dot{X} = \begin{bmatrix} 0 & 1\\ -\frac{k+l_1}{M} & -\frac{\nu+l_2}{M} \end{bmatrix} X + \begin{bmatrix} 0\\ 1\\ \frac{1}{M} \end{bmatrix} u.$$
(9)

Thus, the closed-loop resonance frequency and sensitivity are given by

$$f_{\rm R-cl} = \frac{1}{2\pi} \sqrt{\frac{(k+l_1)}{M}}$$
 (10)

$$\frac{\partial f_{\rm R-cl}}{\partial k} = \frac{1}{4\pi\sqrt{(k+l_1)\,M}} = \frac{1}{8\pi^2 M f_{\rm R-cl}}.$$
 (11)

The state feedback gain L is, then, determined in order to assign the closed-loop poles in such a way that the system resonates at a lower frequency than the natural frequency of the tweezers (see Table III). L is computed to scale down this resonant frequency (initially 1323.6 Hz), by a given factor depending on the desired sensitivity enhancement while keeping the damping factor unchanged.

283 When the closed-loop resonance frequency is scaled down by 284 a factor n compared to the SNT natural resonance frequency, 285 the sensitivity to small stiffness variation is improved with the 286 same ratio according to (11). Fig. 4 shows the computed fre-287 quency shift of the closed-loop system caused by a signifi-288 cant stiffness variation for different frequency scaling down 289 factor, n.

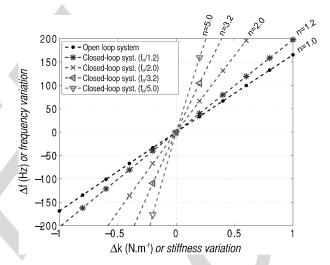


Fig. 4. Evolution of the resonance frequency shift $\Delta f_{\rm R}$ and $\Delta f_{\rm R-cl}$ with the added stiffness Δk variation for different frequency reduction ratio n.

C. Observer Design

The implementation of the aforementioned state feedback is 291 done through the use of a Luenberger observer. The observer is 292 designed such that it preserves the good closed-loop sensitivity 293 to parametric variations and it does not amplify too much noise. 294

The poles of the observer have then to be carefully chosen. 295 We used the following guidelines. 296

- The observer is designed to be faster than the original 297 device—with poles at least two times faster than the poles 298 of the system in open loop (cf., Table III).
- (2) The observer poles are chosen not too fast in order to 300 avoid excessive noise amplification. 301
- (3) The final location of the observer poles is chosen in order to preserve the sensitivity of the system to stiffness 303 variations (by using root locus).
 304

In the following section, simulations illustrate 1) the relevance 305 of the method and 2) the importance of the design of the observer 306 towards the enhancement of the sensitivity. 307

Closed loop f_R/5.0 (Δf=125 Hz) Closed loop $f_R/2.0$ (Δf =50 Hz) Open loop f_R (Δf=25 Hz) 100 Magnitude(dB) 80 60 40 20 90 Phase(deg.) 45 0 -45 90 10000 100 1000 Frequency (Hz)

Fig. 5. Simulated frequency results for open-loop driven system and two different closed-loop systems. The three systems undergo a variation of k the system stiffness. k vary from -0.5 to 0.5 N/m by 0.1 N/m step. In black line, the tweezer transfer function, i.e., the open-loop system. In blue line, the implemented (n = 2) closed-loop system transfer function. The system resonates at a frequency twice lower. In red line, the implemented (n = 5) closed-loop system transfer function. The system resonates at a frequency twice lower.

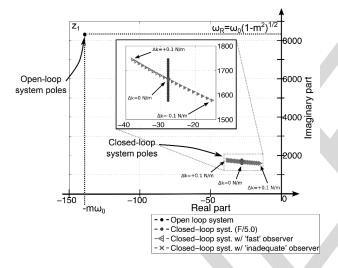


Fig. 6. Root locus of the closed-loop systems with n = 5 and with different observers. Complex conjugate poles are plotted in the positive imaginary half s plan. The parameter k varies from -0.1 to 0.1 N/m by 0.01 N/m steps. The black dots show the poles of the open-loop driven system. The ideal pole path of the closed-loop system due to the k variations are plotted with blue dots (i.e., no observer). For different observers designs, poles dependencies (and equivalent resonance frequency $\omega_{\text{R}} = 2\pi f_{\text{R}}$ and damping m) on the tweezers stiffness k are shown by red triangle and green cross plots. NB: Green cross dots path are imperceptible since they are at this scale all concentrate in the middle.

308 D. Simulation Results

From parameters identified in Table II, the feedback scheme 309 (see Fig. 3) has been first programmed and tested under MAT-310 LAB/Simulink without implementing the observer (i.e., all the 311 state variables are supposed to be measured). Fig. 5 compares 312 the resonance frequency shift between the open-loop driven sys-313 tem and the closed-loop systems computed for two resonance 314 frequency reduction factors set to n = 2 and 5. As expected, the 315 sensitivity of the closed-loop system is improved proportionally 316 to *n*. 317

In a second instance, we implemented two different kinds of observers (with different dynamics); Fig. 6 illustrates two cases.

 TABLE IV

 POLE PLACEMENT OF THE OBSERVER FOR STATE FEEDBACK IMPLEMENTATION

Open-loop system (Tweezers)	$z_{1,2} = -138.9 \pm 8.3 \times 10^3 i$
closed-loop system $(n = 5)$	$z_{1,2} = -27.8 \pm 1.7 \times 10^3 i$
Observer 1 (fast)	$z_{1,2} = -277.8 \pm 0.2 \times 10^3 i$
Observer 2 (slow)	$z_{1,2} = -27.8 \pm 0.2 \times 10^3 i$
Observer 3 (inadequate)	$z_{1,2} = -277.8 \pm 16.7 \times 10^3 i$

This Figure shows that the root locus (due to k parameter variations) is deformed by the observer dynamics. The sensitivity of 321 the resonance frequency to stiffness variations is given by the 322 amplitude of the imaginary axis variations. Table IV reports the 323 poles of the observers for which simulation results are presented. 324

In the insert of Fig. 6, the poles of the system implemented 325 with the observer named *inadequate* do not move compared to 326 the ideal root locus (ideal state feedback case). If the poles of 327 the observer are inappropriately chosen, the sensitivity of the 328 poles to variations of k is significantly reduced. Observer poles 329 close to the real axis, i.e., with an imaginary part close to 0, 330 are the more appropriate to reconstruct the displacement state 331 and to preserve the sensitivity of the system to variations of k. 332 The socalled *fast* observer allows us to increase the resonance 333 frequency shift ($\omega_{\rm R} = 2\pi f_{\rm R}$ in Fig. 6) close to the performances 334 theoretically expected. 335

IV. EXPERIMENTAL APPLICATION FOR EXTRA-STIFFNESS 336 CHARACTERIZATION 337

The previous method has been experimentally tested with the 338 mechanical characterization of biomolecules of *fibronectin*. The 339 physical sensor is the SNT, while the feedback controller and 340 the observer are implemented in a dSPACE prototyping board. 341

A. Experimental Protocol

Fibronectin is a protein, 100 nm in length, of the extracellular matrix which plays a major role in cell adhesion, growth, 344

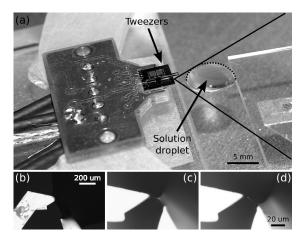


Fig. 7. Experimental setup for the trapping of *fibronectin* molecules. (a) The SNT are fixed. A droplet of *fibronectin* solution is dropped on a glass slice which is mounted on the XYZ stage. (b)–(d) Sequences of the molecule combing. Under optical microscopy, the tweezers' tips are retrieved from the solution meniscus. A bundle of *fibronectin* is released between the tips.

migration, and differentiation [33]. These molecules are used to 345 polymerize the fibrillar matrix at the surface of the cell. We have 346 used this property to trap by combing a bundle of *fibronectin*. 347 After immersion of the tips in the solution, they are retrieved by 348 the side forming a polymerized bundle of molecules (see Fig. 7). 349 350 The closed-loop systems are evaluated with *fibronectin* bundle in air, in dry condition. This stable condition was preferred 351 to the immersed one as we can avoid any effect of liquid surface 352 tension stability that can interfere with the sole system response. 353 For each case, a sinusoidal reference of $0.02 V_{AC}$ with an offset 354 of $20 V_{DC}$ is applied to the system and the oscillation frequen-355 cies are spanned from 1000 to 2000 Hz with a step of 2 Hz. 356 Then, the measurement is performed with the lock-in amplifier. 357 Two sets of experiments have been performed with two differ-358 ent bundles. From the resonance frequency shifts in open loop, 359 the stiffness of the first bundle is $11.8 \text{ N} \cdot \text{m}^{-1}$, and $4.3 \text{ N} \cdot \text{m}^{-1}$ 360 361 for the second one. In total, 22 resonance responses are recorded and compared. 362

363 B. Results and Discussions

Experimental Results: Fig. 8 shows the frequency responses of four different systems with and without a bundle of *fibronectin*.

In Fig. 8(a), the responses of the open-loop driven SNT are 367 given. The dashed-line curve is the reference curve, which cor-368 responds to the frequency response of the SNT alone driven in 369 open loop. The solid line is the response of the tweezers with 370 the bundle trapped in between the tips. The resonance frequency 371 increases and the amplitude decreases due to the additional stiff-372 ness and loss contributions of the molecules. In Fig. 8(b)-(d), 373 the results of the closed-loop systems with desired reduction 374 ratios are of 1.1, 1.2 and 1.5. 375

Resonance responses are fitted with a second-order resonator model and characteristics such as the resonance frequency are extracted. The Table V sums up the resonance frequencies for the experiment with/without the first bundle for five different

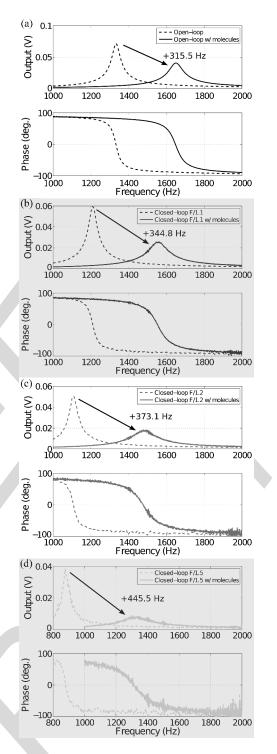


Fig. 8. Frequency responses of different implemented closed-loop systems for 0.02 V sinusoidal signal with an offset of 20 V. In dotted lines, the responses of the systems with the bundle of molecules. In plain lines, the responses of the systems without molecules. (a) Open-loop driven systems. (b) Closed-loop systems implemented with $f_{\rm R}/1.1$. (c) Closed-loop systems implemented with $f_{\rm R}/1.2$. (d) Closed-loop systems with $f_{\rm R}/1.5$.

systems. Through reducing the resonance frequency of the system, the measured results prove the increase of the frequency shift due to the extra-stiffness of the bundle. 382

Moreover Fig. 9 tends to 1) confirm with second set of experiments, the enhancement of the sensitivity through the proposed 384

TABLE V EXPERIMENTAL RESULTS FOR THE BUNDLE 1 (RELATED TO FIG. 8)

System (n)	Freq. w/o bundle (Hz)	Freq. w/ bundle (Hz)	Freq. shift (Hz)
Open loop (1.0)	1333.6	1649.1	+315.5
Closed-loop (1.1) 1211.1	1555.9	+344.8
Closed-loop (1.2		1482.2	+373.1
Closed-loop (1.5		1327.4	+445.5
Closed-loop (2.0) 653.9	1175.6	+521.7
700			
600			
500			
(ZH) 100 ↓	_	•	
300			
200	***	*	
1001	1.2 1.4	1.6 1.	8 2

Fig. 9. Synthesis of the experimental results. In abscissa, is informed the reduction factor (n) applied to the resonance frequency for the closed-loop system. The shift Δf is the resonance frequency difference due to the presence of the bundle of molecules in between the tips. The dotted lines represent the theoretical improvement expected. The two sets of results obtained with two different bundles are plotted. The green dots correspond to the experiment of Fig 8 and TableV (bundle of $11.8 \text{ N} \cdot \text{m}^{-1}$ stiffness). The blue stars corresponds to the second bundle of $4.3 \text{ N} \cdot \text{m}^{-1}$ stiffness.

feedback strategy and 2) show a good agreement with the theory. 385 Small differences are discussed in the following section. 386

2) Discussion: The synthesis of Fig. 9 demonstrates the en-387 hancement of the frequency shifts in agreement with the theory. 388 However, the enhancement for closed-loop system with a reduc-389 390 tion factor of 2.0 is lower than expected, and the implementation 391 for a higher reduction factor has not been achieved. In this case, the system tends to become unstable. We mainly associated this 392 phenomenon to the observer implementation and to the presence 393 of some delays in the feedback loop. 394

395 We have demonstrated previously that observer's dynamic changes the root locus of system's poles, the behavior of the 396 closed-loop system under parameter variations and impacts the 397 performances of the method. Furthermore, delays have been 398 characterized due to D/A and A/D converters dead times and to 399 the phase shift brought by the current amplifiers. Current works 400 401 are devoted to the precise characterization of such delays and to their integration in the control design process. 402

V. CONCLUSION AND FUTURE WORK 403

The present study investigates the relevance of a feedback 404 approach in the improvement of the performances of MEMS 405 tweezers for the detection and the mechanical characterization of 406 407 biological molecules. An improvement of 50% on the sensitivity

to mechanical stiffness variations has been reached theoretically 408 and experimentally on *fibronectin* bundles through the emulation 409 of a more compliant system. The stiffness have been brought 410 down to 10.5 N/m instead of originally 24.5 N/m. 411

The main drawback of this approach is the sensitivity of 412 the observer to the stiffness variations. Unusually, the poles of 413 the observer are uncontrollable resulting from the separation 414 principle. During the sensing of the molecules stiffness, the 415 separation principle is not satisfied anymore and the observer 416 poles can become unstable. We are actually working on self-417 scheduled observers aiming at adapting the observer dynamics 418 with respect to sensed parametric variations. It should solve the 419 main drawback of this promising approach and allows us to go 420 one step further into the theoretical closed-loop performances. 421

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