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# Computer-aided Mobility Analysis of Parallel Mechanisms

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## **ABSTRACT**

To determine mobility properties is one of the most challenging issue in the analysis and synthesis of parallel manipulators (PMs). However, currently, the most used methods mainly rely on experiences and manual analysis, which led to inefficient implementation. The motivation of this paper is to present an automatic mobility analysis algorithm and software package for the researchers and designers with an effective and practical means. According to the topological design theory of PMs based on position and orientation characteristic (POC) equations, this paper proposes a set of computer algorithmic rules and procedures for automated mobility analysis of PMs in the most user-friendly and efficient way. Firstly, a complete digital information model for topological structures which has a mapping relationship with the POC of a PM is proposed. This model not only describes the dimension of the motion outputs, but also includes the mapping relationship between the output orientation and the axes of the kinematic joints. Secondly, algorithmic rules are established that convert the union and intersection operations of POC into binary logical operations and then the detailed algorithmic procedures for an automatic mobility analysis are presented. In what follows, a corresponding software for automatic mobility analysis is described. The software package is equipped with a GUI that facilitates the input and allows the visualization of the results. Finally, four typical examples are provided to show the effectiveness of the software package for most of parallel mechanisms (not including some paradoxical mechanisms).

Keywords: mobility analysis; degree of freedom; position and orientation characteristic; parallel mechanisms

## 1. Introduction

Nowadays, parallel manipulators or mechanisms (in short, PMs) can be found in industrial machines because of important mechanical and kinematic properties over their serial counterparts[1], such as higher stiffness, better load weight ratio, lower inertia, higher accuracy. In their design, mobility analysis is a fundamental issue for the research on mechanical synthesis, kinematic and dynamic analysis. Here, mobility analysis includes the calculation of the number of degree of freedom (in short, DOF) and the evaluation the output motion properties of a manipulator.

It is well known that the conventional mobility analysis based on the Chebyshev-Grubler-Kutzbach (CGK) [2] criterion fails in some PMs, this is due to neglecting the geometrical relationships between joints axes. In recent years, different approaches to the mobility analysis of PMs have been studied systematically, and various formulas and methods have been derived and presented [3-26]. But many of these formulas and methods may be reducible to the same original formula and the existing methods can be grouped into four categories: methods based on screw theory [5-12], methods based on Lie group theory [13-17], methods based on the theory of Position and Orientation Characteristics (POC) [18-23], and methods based on linear transformations [2,24].

Most of existing methods for mobility analysis of PMs are performed manually. This task is often complex and difficult because much experience is required to deal with dependency or deep mathematical knowledge is required to establish the kinematic model. An advanced algorithm for the automatic mobility analysis of PMs is an effective solution to afore mentioned problems. However, as far as the authors of the paper know, computer-aided mobility analysis can be found only in very few papers [12,13,17,29,30]. In order to compare the feasibility of computerization of mobility analysis for PMs, the principle and characteristics of the existing relevant methods are listed in Table 1.

The approach using screw theory has been successfully applied in many PMs. The computation of the degree of freedom utilizes the reciprocal relation between twist and wrench defined by the mechanical constraints [5-11], and mainly involves the linear solution of systems of screws. In this sense, this method has some potential for automatic analysis. However, it is difficult to obtain the screw system automatically for the average designer. Cao [12] took the first joint of a leg as a reference to establish the system of all screws of the leg, and realized an automatic analysis of the mobility of symmetrical PMs.

The mobility analysis using Lie Group theory is based on the “multiplication” operation and the “intersection” operation of Euclidean subgroups [13-16]. There are too many operation rules involved in this process (over 107 rules [14] ). This method is therefore not suitable to be implemented directly in an automated algorithm. By translating group theory and subsets and subgroups into the Lie algebra and its vector subspaces and its subalgebras, a computer-aided method for mobility analysis was proposed by Daniel[17]. But the datum describing the infinitesimal screws associated with each kinematic pair of the kinematic chain in an arbitrary position should be entered manually.

The method based on the theory of linear transformations, relates the number of degree of freedom of a chain or a PM to the rank of the Jacobian matrix. Using this method, Angeles and Gosselin [24] obtained the mobility of several over-constraint mechanisms including the Bennet mechanism. But the Jacobian matrix, which maps a vector in joint velocity space into external velocity space, needs to develop the kinematic equations. From this point, this method is quite complicated, and to the best of the authors’ knowledge, there exist no general methods that can be automatically used for the PMs.

With the last twenty years, algebraic methods have become successful in solving problems in mechanism analysis and synthesis, and it is also called GA-based method. Husty and Schröcker [25] used methods from algebraic geometry to define the DOF of a PM as Hilbert Dimension of a set of nonlinear polynomial kinematic constraint equations. By introducing Clifford algebra, Huo et al., [26,27] proposed an analytical approach for the determination of the motions/constraints, mobility and singularity of PMs. Based on the screw theory and the Chasles’ theorem, Yang [28] presented a method for the type synthesis of 3T1R PMs with variable rotational axis. One main advantage of the GA-based method is that subspace can be added, subtracted, and intersected in its framework, which is beneficial to be carried out in automatic way. However, the establishment of algebraic forms or algebraic equations needs the designers to have the abstruse mathematical knowledge.

So-called POC theory for PMs means the topological design theory for PMs based on Position and Orientation Characteristics (POC) equations [18]. The mobility analysis method based on the POC theory, has clear formulas and

judgment criteria that are easy to use and program. To solve the mobility analysis problems, based on POC theory, a computer-aided software tool for the mobility analysis of PMs will be implemented in this work. The initial digital model and the relevant algorithms of mobility analysis were investigated in [29,30] for the PMs constituted only by revolute joints and prismatic joints. Considering the geometric properties of multi-DOF pairs and the coupling between rotations and translations, this paper presents a complete representation of topological structure of PMs, and gives the systematic rules for performing the union and intersection operations of POC. Furthermore, the algorithms of mobility analysis for serial and parallel mechanisms are proposed. Finally, a software package for automatic mobility analysis is described and some typical examples of PMs are presented to illustrate its effectiveness.

**Table 1** The methods of mobility analysis and their main features

Methods	Traditional CGK criterion	Based on screw theory	Based on displacement Subgroup/Submanifold	Based on POC set
<b>Math tool</b>	---	Screw algebra	Lie algebra	Set theory
<b>Representation</b>	Real number	Motion/constraint screw system	Symbol of subgroup /submanifold	POC Set Number of DOF
<b>DOF formula</b>	$F = \sum_{i=1}^m f_i - d(m-n+1)$	$F = d(n-g-1) + \sum_{i=1}^g f_i + v$ $v = \sum_{i=1}^p g_i - \lambda p - k$	$F = \sum_{i=1}^m f_i - \sum_{j=1}^{(n+1)} \dim(A_j^{m/f})$ $+ \dim(\prod_{j=1}^{(n+1)} A_j^{m/f})$	$F = \sum_{i=1}^m f_i - \sum_{j=1}^{(v+1)} \xi_{L_j}$ $\xi_{L_j} = \dim.((\prod_{i=1}^j M_{b_i}) U M_{b_{(j+1)}})$
<b>Algorithm</b>	Elementary algebra	Operations of screw system: • Linear operations • Independence criterion of screw systems	Operation of displacement subgroup • Union operations (107) • Intersection operations (107)	Operation of POC set • Union operations (8 linear, 2 nonlinear) • Intersection operations (12 linear, 2 nonlinear)
<b>Main features</b>	✧ Easy to perform ✧ Fail to most spatial PMs	✧ Simple linear operations ✧ Local DOF ✧ Need verification of full-cycle DOF ✧ Including paradoxical mechanisms	✧ Too many rules ✧ Complicated math tool ✧ Full-cycle DOF ✧ Excluding paradoxical mechanisms	✧ Simple linear operations ✧ Fewer rules ✧ Full-cycle DOF ✧ Excluding paradoxical mechanisms

**Table 1 (Continued)**

Methods	Based on linear transformations	Based on GA	
<b>Math tool</b>	Linear algebra	Study's kinematic mapping	Conformal geometric algebra
<b>Representation</b>	Rank of Jacobian matrix	Dimension of local tangent spaces of kinematic varieties	Linear-independent constraints
<b>DOF formula</b>	$F = \sum_{i=1}^m f_i - \sum_{j=1}^k \dim(R_{A_j}) + \dim(\prod_{j=1}^k R_{A_j})$	Hilbert dimension	Number of linear-independent constraints
<b>Algorithm</b>	Linear algebra	Study's kinematic mapping	Inner product, Outer product, Dual product, Shuffle product

<b>Main features</b>	✦ Including singular configurations	✦ Including singular configurations	✦ Simple operations
	✦ Instantaneous DOF	✦ Instantaneous DOF	✦ Including singular configurations
	✦ Need verification of full-cycle DOF	✦ Calculation of high dimensional matrix	✦ Instantaneous DOF
	✦ Solving of high dimensional equations	✦ Time consuming	✦ Need advanced algebraic knowledge
	✦ Time consuming	✦ Need advanced algebraic knowledge	

It should be pointed out that the software package described in this paper currently could not analyze some “paradoxical” mechanisms, for example Bennett and Bricard mechanisms since the mobility of these mechanisms is not only related to their geometric topological structure, but also related to the dimensional constraint parameters (i.e., link length and skew angle comply with a certain functional relationship) which can not be expressed in the matrices of topological structure proposed here.

## 2. POC theory for PMs

This section briefly introduces some basics of POC theory for PMs, which are necessary in the mobility analysis of PMs.

### 2.1 Definition and related equations of POC

#### 2.1.1 Definition of POC

To describe the relative motion characteristics between any two components in a mechanism, the POC is introduced in [18] as

$$M = \begin{pmatrix} \mathbf{t}^{\xi_{t1}}(\text{dir.}) + \{\mathbf{t}^{\xi_{t2}}(\text{dir.})\} \\ \mathbf{r}^{\xi_{r1}}(\text{dir.}) + \{\mathbf{r}^{\xi_{r2}}(\text{dir.})\} \end{pmatrix} \quad (1)$$

$$\xi = \xi_{t1} + \xi_{r1} \leq F \quad (2)$$

where

$M$  -- POC of the output motion link of the PM.

$\mathbf{t}^{\xi_{t1}}(\text{dir.})$  -- Independent output translation elements.

$\mathbf{t}^{\xi_{t2}}(\text{dir.})$  -- Non-independent output translation elements.

$\mathbf{r}^{\xi_{r1}}(\text{dir.})$  -- Independent output rotation elements.

$\mathbf{r}^{\xi_{r2}}(\text{dir.})$  -- Non-Independent output rotation elements.

$\xi_{t1}$  ( $\xi_{r1}$ ) -- Number of the independent output translation (rotation) elements ( $\xi_{t1}, \xi_{r1}=0,1,2$  or  $3$ ).

(dir.) -- Direction of output translation (or rotation) with respect to joints axes.

$\xi$  -- Number of independent output elements in the POC.

$F$  -- DOF of the mechanism.

For most mechanisms, the non-independent output element is constant (excluding the singular configurations) and therefore not considered.

#### 2.1.2 POC equation for serial mechanism

The POC of a serial leg is expressed as

$$M_L = \bigcup_{i=1}^m M_{J_i} \quad (3)$$

where

$M_L$  -- POC of the output motion link (e.g. end link of the serial leg).

$M_{J_i}$  -- POC of the  $i^{\text{th}}$  joint of the serial leg.

$m$  -- Number of joints in the serial leg.

U- "Union" operation [18].

### 2.1.3 POC equation for parallel mechanism

The POC of a parallel mechanism is

$$M_{pa} = \bigcap_{j=1}^n M_{L_j} \quad (4)$$

where

$M_{pa}$  -- POC of the output motion link (e.g. moving platform of the PM).

$n$  -- Number of serial legs of the PM.

$M_{L_j}$  -- POC of the  $j^{\text{th}}$  leg referred to the same base point.

$\cap$  -- "Intersection" operation [18].

## 2.2 Equation of mobility

A PM with  $(v+1)$  serial legs can be considered as a combination of  $v$  independent single-loop-chains (in short, SLC). The structure composition of a PM based on SLC [18] is shown in Fig.1. Two serial legs are chosen to form the first independent loop (named as  $SLC_1$ ), and the moving platform is considered as an active component of this loop. Then,  $SLC_1$  is regarded as a whole (i.e., an equivalent sub-PM), and combined with another leg to form a second independent loop (named as  $SLC_2$ ); and in the same way, the  $(v+1)^{\text{th}}$  leg is combined with  $SLC_{(v-1)}$  to form the  $v^{\text{th}}$  independent loop (named as  $SLC_v$ ).

According to the theory based on POC, the number of the mobility  $F$ , i.e., the DOF of a PM can be calculated by Eq.(5), and the property of mobility can be evaluated by POC of the moving platform with respect to the fixed platform.

$$F = \sum_{i=1}^m J_i - \sum_{j=1}^v \xi_{L_j} = \sum_{j=1}^{v+1} f_j - \sum_{j=1}^v \xi_{L_j} \quad (5)$$

where,

$$\xi_{L_j} = \dim. \left( \left( \bigcap_{i=1}^j M_{L_i} \right) \cup M_{L_{(j+1)}} \right) \quad (6)$$

$$v = m - n + 1 \quad (7)$$

where

$F$  -- DOF of the PM.

$J_i$  -- DOF of the  $i^{\text{th}}$  joint in the PM.

$m$  -- Number of joints in the PM.

$\xi_{L_j}$  -- Number (or rank) of the independent displacement equations of the  $j^{\text{th}}$  independent loop.

$v$  -- Number of the independent loops of the PM.

$f_j$  -- DOF of all joints of the  $j^{\text{th}}$  leg.

$n$  -- Number of components of the PM.

$\bigcap_{i=1}^j M_{L_i}$  -- POC of the equivalent sub-PM composed by the first  $j$  legs.

$M_{L_{(j+1)}}$  -- POC of the  $(j+1)^{\text{th}}$  leg.

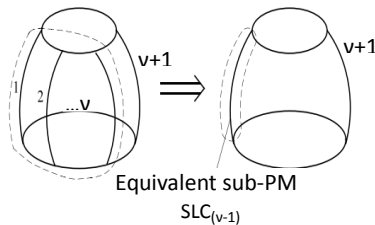


Fig. 1. Basic independent loops composed of PMs

### 2.3 Key steps for computer-aided mobility analysis

- ① The topological description of PM is one of the most significant issues in the analysis of a PM. It should include the complete topological information of the PM with the most concise representation so that it can be recognized and extracted automatically.
- ② POC description for a leg and for a PM is a fundamental problem in automatic mobility analysis. Although the symbolic description as in Eq.(1) is good for expressing the geometric meaning, it is difficult to realize the transitivity of the joint orientation relation.
- ③ The most critical issue is how to establish algorithmic rules for POCs suitable for computer programming to implement the “union” and “intersection” operations. In essential, these operations for POCs mainly involve determining spatial linear independency or not.

### 3. Representation of a PM and its POC

Generally, a PM consists of a moving platform connected to a fixed platform by several serial legs, and its mobility (including number and property) is completely affected by the types of kinematic joints and the geometrical relationships between the joint axes. Thus, the topology definition of a PM must be carried out before analyzing the mobility.

First of all, it needs to be emphasized that in the following the legs of the studied PMs do not include redundant kinematic pairs.

The pair types considered in the digital topological description of PMs are only 1-DOF (i.e.  $f_i=1$ ) revolute (R) and prismatic (P) pair, as any multi-DOF kinematic pair can be produced as a combination of R and P pairs. For instance, universal (U), spherical (S) and cylindrical (C) pairs can be regarded as combinations of two R pairs with orthogonal axes, three R pairs with concurrent axes, and one R pair and one P pair with collinear axes, respectively.

Mobility analysis of a PM (not including some paradoxical mechanisms) deals only with the dimensional constrained types, i.e. the geometrical relationships between joints axes. In a serial leg or between legs, there are six basic geometrical relationships proposed by the authors in [18] as: parallel, orthogonal, coaxial, spatial common point, coplanar and arbitrary, here denoted symbolically using “||”, “⊥”, “/”, “\*”, “#” and “-”, respectively.

#### 3.1 Digital representation of a serial leg and its POC

##### 3.1.1 Digital matrix of serial leg

As described in [30], a digital ordered topological structure matrix representation is proposed including not only what type of pairing the joint has, but also what pair of orientations the joint characterizes. The ordered topology matrix (L) of a serial leg is expressed as an  $f \times f$  matrix, where  $f$  is the number of 1-DOF joints in the leg.

$$L = \begin{bmatrix} J_1 & L & N_{1i} & L & N_{1f} \\ M & O & & & M \\ N_{i1} & & J_i & & N_{if} \\ M & & & O & M \\ N_{f1} & L & N_{fi} & L & J_f \end{bmatrix} \quad (8)$$

The diagonal element  $J_i$  ( $i=1 \sim f$ ) denotes the type of joint  $i$  labeled from the fixed platform to the moving platform in sequence.  $J_1$  is connected to the fixed platform and  $J_f$  is connected to the moving platform.

The non-diagonal element  $N_{ij}$  ( $i, j=1 \sim f$ ) represents the geometrical relationship between joint  $i$  and joint  $j$  (6 kinds of geometric relationship above-mentioned). So, the topological structure matrix  $L$  is symmetrical.

For convenience of programming, each element of matrix  $L$  is expressed numerically so that the ordered topological structure matrix can be handled mathematically. Here,  $N_{ij}$  is transformed into a number 0~5 respectively as shown in Table 2. In the same way, let  $J_i$ , (i.e., R and P joint) be 8 and 9. Thus, the topological structure of a leg can be expressed as an integer matrix.

Table 2. Numbers of geometrical relationships of joints axes

Geometrical relationship	Corresponding number
--------------------------	----------------------

Geometrical relationship	Corresponding number
parallel	1
orthogonal	2
coaxial	3
common point	4
coplanar	5
arbitrary	0

For example, the UP leg listed in Fig.2(a) has three pairs labeled with the numbers 8, 8, and 9. The axes relationships are  $R_1 \perp R_2$ ,  $R_1 \perp P_3$ , and  $R_2 \perp P_3$  respectively. Based on the above definition, the ordered topological matrix of this leg is

$$L = \begin{bmatrix} 8 & 2 & 2 \\ 2 & 8 & 2 \\ 2 & 2 & 9 \end{bmatrix} \quad (9)$$

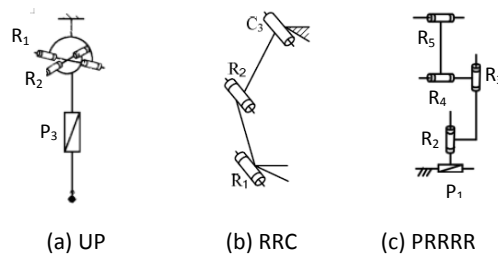


Fig. 2. Typical legs not including S pair

Generally, the multi-DOF kinematic pairs C, U and S, can be equivalent to the combinations of  $R_1^{(C)}|P_2^{(C)}$ ,  $R_1^{(U)} \perp R_2^{(U)}$  and  $R_1^{(S)} * R_2^{(S)} * R_3^{(S)}$  respectively. But the directions of the three equivalent axes of S pair should be calibrated according to the following criteria:

(1) When R pair and S pair are connected by a link, or connected by a prismatic pair or connected by a planar sub-chain, taking an equivalent  $R^{(S)}$  pair parallel to the R pair, as shown in Fig.3(a).

(2) When two S pairs are connected by a link, or connected by a prismatic pair, there is a local rotation DOF around the line connecting two points of the spherical centers. Then, one S pair is treated as two R pairs, and denoted as  $R_1^{(S)} * R_2^{(S)}$ . Meanwhile, its axes are calibrated according to the above criteria (1), as shown in Fig.3(b).

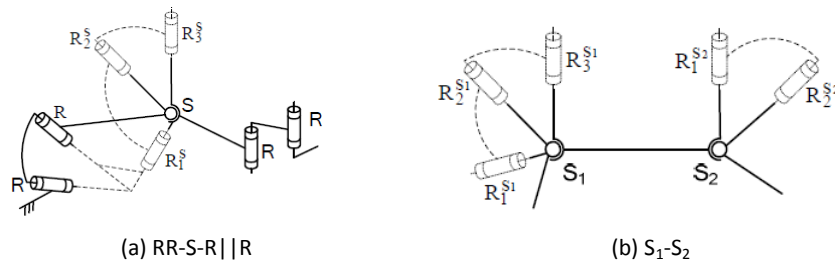


Fig. 3. Equivalent axes of S pair

For example, the RPS leg listed in Fig.4(a) with R and S pairs, can be replaced with the topologically equivalent leg  $R_1(\perp P_2) || R_3-(R_4 * R_5)$ , as shown in Fig.4(b). Based on the above definition, the ordered topological matrix of this leg is



$$L = \begin{bmatrix} 8 & 2 & 1 & 0 & 0 \\ 2 & 9 & 2 & 0 & 0 \\ 1 & 2 & 8 & 0 & 0 \\ 0 & 0 & 0 & 8 & 4 \\ 0 & 0 & 0 & 4 & 8 \end{bmatrix}_{(\#)} \quad (10)$$

where, the subscript symbol ‘#’ indicates the serial number of the spherical center. Particularly, when different S pairs (or three equivalent revolute pairs) have the same center, the serial number is the same.

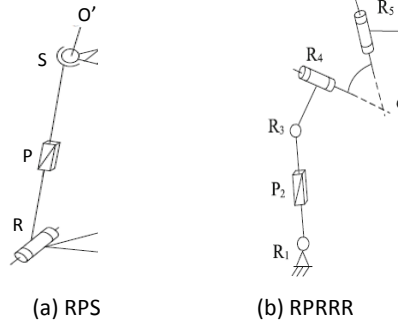


Fig. 4. Typical legs including S pair and its equivalent leg

### 3.1.2 Definition of POC matrix of leg

The POC matrix of a leg should include not only the dimension of output motion, but also the corresponding orientation of the motion. So, considering the relations between the topological structure and POC of a leg, a  $2 \times f$  POC matrix ( $M_L$ ) is developed to match with its ordered topology matrix L, that is

$$M_L = \begin{bmatrix} t_1 & L & t_1 L & t_f \\ r_1 & L & r_1 L & r_f \end{bmatrix} \quad (11)$$

The defining of  $M_L$  shall abide by the following criteria:

- (1)  $t_i (r_i)$  ( $i=1 \sim f$ ) denotes translation (rotation) output respectively, and  $t_i (r_i) = 0, 1, 2$  or 3.
- (2)  $t_i (r_i) \neq 0$  indicates the existence of independent translation (rotation) output.
- (3) when  $t_i=3$  or  $r_i=3$ , the dimension of translation or rotation output is 3. The directions are arbitrary in space and need not to be specified.

(4) The dimension of translation/rotation output is  $\xi_r = \sum_{i=1}^f r_i / \xi_t = \sum_{i=1}^f t_i$ , and the total dimension of output at the end link of the leg is  $\xi = \xi_r + \xi_t$ .

(5) Output direction of column  $i$  is related to the axis of the  $i^{\text{th}}$  joint in the ordered topological structure matrix L.

- $t_i=1$  means there is an independent translation along the direction of the  $i^{\text{th}}$  joint axis (if  $r_i=0$ ) or in the normal plane of the  $i^{\text{th}}$  joint axis (if  $r_i=1$ ).
- when  $t_i=2$ , there are two independent translations in the normal plane of the  $i^{\text{th}}$  joint axis.

Take the same UP leg as an example (Fig.2(a)). Its ordered topological structure matrix is Eq.(9), i.e.

$$L = \begin{bmatrix} 8 & 2 & 2 \\ 2 & 8 & 2 \\ 2 & 2 & 9 \end{bmatrix}. \text{ According to the definition, the POC can be expressed as } M_L = \begin{bmatrix} 0 & 0 & 1 \\ 1 & 1 & 0 \end{bmatrix}. \text{ This matrix indicates as}$$

follows,

- (1) The non-zero elements of  $M_L$  mean the existing translation output (the first row) and the existing rotation output (the second row).
- (2) The sum of the non-zero elements corresponds to the number of DOF of this leg.

(3) Each column of  $M_L$  implies the motion orientation, such as  $r_1=1$  denotes that the rotation direction corresponds to the axis of the 1<sup>st</sup> pair (i.e.  $R_1$  joint) in the leg. Similarly,  $r_2=1$  corresponds to the  $R_2$  joint and  $t_3=1$  (with  $r_3=0$ ) corresponds to the translation direction along the  $P_3$  joint axis.

### 3.1.3 POC matrix of a sub-chain

More commonly, planar sub-chains (2-DOF  $G_2$  and 3-DOF  $G_3$ ) or spherical sub-chains (2-DOF  $S_2$  and 3-DOF  $S_3$ ) may constitute a leg or part of a leg consisting of parallel or orthogonal R joints and/or P joints. It is not difficult to see that, no matter which point on the end link is chosen, the end link of a planar sub-chain has always a one-dimensional rotation and a one-dimensional or two-dimensional independent translations in the normal plane of the R joint axis. On the contrary, for a spherical sub-chain (2-DOF  $S_2$  and 3-DOF  $S_3$ ), different points of an end link have different characteristics of POC.

In order to express the motion output direction more clearly and facilitate subsequent operations of POCs, it is necessary to extract the sub-chains and to mention the specific point of the end-effectors. In accordance with the definition of a POC matrix, 9 type matrices for the 12 sub-chains is illustrated in Table 3, where POC of the sub-chains  $R_1*R_2$  and  $R_1*R_2*R_3$  have two different forms. Here, we take the sub-chain  $R_1*R_2*R_3$  as an example to illustrate what the two matrices mean and how to choose in the calculation. The specific description is given as follows,

(1) If the intersection point of three R joints' axes is selected as base point  $O'$ , there exist three independent finite rotations parallel to the axes of  $R_1$ ,  $R_2$  and  $R_3$  respectively. So the POC is written as  $\begin{bmatrix} 0 & 0 & 0 \\ 1 & 1 & 1 \end{bmatrix}$ .

(2) If base point  $O'$  is selected outside the axis of the end joint ( $R_3$ ), there are also two derivative finite translations within a plane perpendicular to radius vector  $\rho$  shown in Fig.5. Therefore, in addition to the three finite rotations mentioned above, two non-independent translations should be considered (See detailed proof in [18]). So the POC can be expressed as  $\begin{bmatrix} 0 & 0 & 0 \\ 1 & 1 & 1 \end{bmatrix} + \begin{bmatrix} 2 \\ \# \end{bmatrix}$ , where the symbol '#' denotes the No. of intersection point, and the number 2 indicates the two-dimensional derivative translations, which is non-independent.

**Table 3** Twelve Sub-chains and their POCs

No.	Type	Symbol	Dimension	Topology Matrix	POC matrix
1		R  R	2	$\begin{bmatrix} 8 & 1 \\ 1 & 8 \end{bmatrix}$	$\begin{bmatrix} 1 & 0 \\ 1 & 0 \end{bmatrix}$
2	$G_2$	R⊥P	2	$\begin{bmatrix} 8 & 2 \\ 2 & 9 \end{bmatrix}$	$\begin{bmatrix} 1 & 0 \\ 1 & 0 \end{bmatrix}$
3		P⊥R	2	$\begin{bmatrix} 9 & 2 \\ 2 & 8 \end{bmatrix}$	$\begin{bmatrix} 0 & 1 \\ 0 & 1 \end{bmatrix}$
4		R//R//R	3	$\begin{bmatrix} 8 & 1 & 1 \\ 1 & 8 & 1 \\ 1 & 1 & 8 \end{bmatrix}$	$\begin{bmatrix} 2 & 0 & 0 \\ 1 & 0 & 0 \end{bmatrix}$
5		R//R⊥P	3	$\begin{bmatrix} 8 & 1 & 2 \\ 1 & 8 & 2 \\ 2 & 2 & 9 \end{bmatrix}$	$\begin{bmatrix} 2 & 0 & 0 \\ 1 & 0 & 0 \end{bmatrix}$
6	$G_3$	P⊥R//R	3	$\begin{bmatrix} 9 & 2 & 2 \\ 2 & 8 & 1 \\ 2 & 1 & 8 \end{bmatrix}$	$\begin{bmatrix} 0 & 2 & 0 \\ 0 & 1 & 0 \end{bmatrix}$
7		R(⊥P)//R	3	$\begin{bmatrix} 8 & 2 & 1 \\ 2 & 9 & 2 \\ 1 & 2 & 8 \end{bmatrix}$	$\begin{bmatrix} 2 & 0 & 0 \\ 1 & 0 & 0 \end{bmatrix}$
8		R(⊥P)⊥P	3	$\begin{bmatrix} 8 & 2 & 2 \\ 2 & 9 & 2 \\ 2 & 2 & 9 \end{bmatrix}$	$\begin{bmatrix} 2 & 0 & 0 \\ 1 & 0 & 0 \end{bmatrix}$

No.	Type	Symbol	Dimension	Topology Matrix	POC matrix
9		P(LP) <sub>LR</sub>	3	$\begin{bmatrix} 9 & 2 & 2 \\ 2 & 9 & 2 \\ 2 & 2 & 8 \end{bmatrix}$	$\begin{bmatrix} 0 & 0 & 2 \\ 0 & 0 & 1 \end{bmatrix}$
10		P(LR) <sub>LP</sub>	3	$\begin{bmatrix} 9 & 2 & 2 \\ 2 & 8 & 2 \\ 2 & 2 & 9 \end{bmatrix}$	$\begin{bmatrix} 0 & 2 & 0 \\ 0 & 1 & 0 \end{bmatrix}$
11	S <sub>2</sub>	R*R	2	$\begin{bmatrix} 8 & 4 \\ 4 & 8 \end{bmatrix}$	$\begin{bmatrix} 0 & 0 \\ 1 & 1 \end{bmatrix} + \begin{bmatrix} 2 \\ \# \end{bmatrix}$
12	S <sub>3</sub>	R <sub>1</sub> *R <sub>2</sub> *R <sub>3</sub> (Not coplanar)	3	$\begin{bmatrix} 8 & 4 & 4 \\ 4 & 8 & 4 \\ 4 & 4 & 8 \end{bmatrix}$	$\begin{bmatrix} 0 & 0 & 0 \\ 1 & 1 & 1 \end{bmatrix} + \begin{bmatrix} 2 \\ \# \end{bmatrix}$

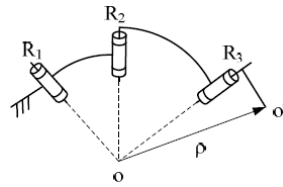


Fig. 5. The sub-chain of R<sub>1</sub>\*R<sub>2</sub>\*R<sub>3</sub>

### 3.1.4 Topology matrix and POC matrix of some typical legs

Table 4 displays seven typical legs and their topological matrices and the corresponding POC matrices, among which the structures of legs No.3~No.6 are shown in Fig.2(a)-(c) and Fig.4(a). Clearly, the representation of the topological structure and the POC of a leg exhibits an orderly one-to-one correspondence among the sequence of the pairs and the motion output orientations.

Table 4. Topology matrix and POC matrix of typical legs

No.	Symbol	Topology Matrix	POC matrix	Commentary
1	R	L=[8]	$M_L = \begin{bmatrix} 0 \\ 1 \end{bmatrix}$	Rotation around R axis
2	P	L=[9]	$M_L = \begin{bmatrix} 1 \\ 0 \end{bmatrix}$	Translation along P axis
3	UP (R <sub>1</sub> R <sub>2</sub> P <sub>3</sub> )	$L = \begin{bmatrix} 8 & 2 & 2 \\ 2 & 8 & 2 \\ 2 & 2 & 9 \end{bmatrix}$	$M_L = \begin{bmatrix} 0 & 0 & 1 \\ 1 & 1 & 0 \end{bmatrix}$	<ul style="list-style-type: none"> <li>• One-translation along P<sub>3</sub> axis</li> <li>• Two-rotations about axes of R<sub>1</sub> and R<sub>2</sub> joints</li> </ul>
4	RRC (R <sub>1</sub> R <sub>2</sub> R <sub>3</sub> P <sub>4</sub> )	$L = \begin{bmatrix} 8 & 1 & 1 & 1 \\ 1 & 8 & 1 & 1 \\ 1 & 1 & 8 & 1 \\ 1 & 1 & 1 & 9 \end{bmatrix}$	$M_L = \begin{bmatrix} 3 & 0 & 0 & 0 \\ 1 & 0 & 0 & 0 \end{bmatrix}$	<ul style="list-style-type: none"> <li>• Three-translations along arbitrary directions in space</li> <li>• One-rotation about R<sub>1</sub> joint axis</li> </ul>
5	P <sub>1</sub> R <sub>2</sub> R <sub>3</sub> R <sub>4</sub> R <sub>5</sub>	$L = \begin{bmatrix} 9 & 2 & 2 & 1 & 1 \\ 2 & 8 & 1 & 2 & 2 \\ 2 & 1 & 8 & 2 & 2 \\ 1 & 2 & 2 & 8 & 1 \\ 1 & 2 & 2 & 1 & 8 \end{bmatrix}$	$M_L = \begin{bmatrix} 3 & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 1 & 0 \end{bmatrix}$	<ul style="list-style-type: none"> <li>• Three-translations along arbitrary directions in space</li> <li>• Two-rotations about R<sub>2</sub> and R<sub>4</sub> joints axes</li> </ul>

No.	Symbol	Topology Matrix	POC matrix	Commentary
6	RPS (R <sub>1</sub> P <sub>2</sub> R <sub>3</sub> R <sub>4</sub> R <sub>5</sub> )	$L = \begin{bmatrix} 8 & 2 & 1 & 0 & 0 \\ 2 & 9 & 2 & 0 & 0 \\ 1 & 2 & 8 & 0 & 0 \\ 0 & 0 & 0 & 8 & 4 \\ 0 & 0 & 0 & 4 & 8 \end{bmatrix}$	$M_L = \begin{bmatrix} 2 & 0 & 0 & 0 & 0 \\ 3 & 0 & 0 & 0 & 0 \end{bmatrix} + \begin{bmatrix} 1 \\ \# \end{bmatrix}$	<ul style="list-style-type: none"> <li>• Two-translations in the normal plane of R<sub>1</sub> axis</li> <li>• Three--rotations around the point #</li> <li>• One derivative translation in the plane perpendicular to radius vector ρ<sub>#</sub></li> </ul>
7	UPS leg (R <sub>1</sub> R <sub>2</sub> P <sub>3</sub> R <sub>4</sub> R <sub>5</sub> R <sub>6</sub> )	$L = \begin{bmatrix} 8 & 2 & 2 & 0 & 0 & 0 \\ 2 & 8 & 2 & 1 & 0 & 0 \\ 2 & 2 & 9 & 2 & 0 & 0 \\ 0 & 1 & 2 & 8 & 0 & 0 \\ 0 & 0 & 0 & 0 & 8 & 4 \\ 0 & 0 & 0 & 0 & 4 & 8 \end{bmatrix}$	$M_L = \begin{bmatrix} 3 & 0 & 0 & 0 & 0 & 0 \\ 3 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$	<ul style="list-style-type: none"> <li>• Three-translations along arbitrary directions in space</li> <li>• Three--rotations around arbitrary axes in space</li> </ul>

### 3.2 Description of a PM and its POC

As mentioned previously, the mobility of a PM depends not only on the geometrical axes relationships in one leg, but also on the geometrical axes relationships between legs. Therefore, in addition to the topological structures of all legs, the topological relations between the legs of a PM are needed to be considered in digital form for the mobility analysis.

In most cases, the geometrical axes relationships between legs depend on the invariant geometric relations between joints axes respectively on the moving platform and on the fixed platform.

To make information concise, the representation of PMs is developed to clearly represent the composition of legs and the relationships of axes pairs on the two platforms.

#### 3.2.1 Description of PM

The representation of the topological structure for a given PM with n legs is expressed in the following four steps.

**Step 1** Label all n legs comprising the PM in turn with numbers 1~n.

**Step 2** Obtain the ordered topological matrix L<sub>i</sub> (i=1~n) of all n legs.

**Step 3** Define two virtual legs MP and BP constituted by the pairs on the moving platform and on the fixed platform in the order of legs.

**Step 4** Obtain the ordered topological matrix L<sub>MP</sub> and L<sub>BP</sub> of the virtual legs MP and BP.

Then, the topological composition of the PM can be expressed as

$$PM=(L_1, L_2, \dots, L_n, L_{MP}, L_{BP}) \quad (12)$$

It should be pointed out that if a leg is connected to the fixed platform or to the moving platform by a P pair of a planar sub-chain, the normal of the sub-chain is represented rather than the axis of the P pair. This is because the output direction of planar sub-chain is represented by the axis of the R pair which is perpendicular to the P pair ( See No. 3, 6, 9 and 10 in Table 3).

Three examples are given here to illustrate how the proposed representation works.

(1) 3-UPS & 1-UP PM

The PM in Fig.6(a) has 3-UPS legs and 1-UP leg. First, replace composite pairs S and U with three R pairs with concurrent axes and two R pairs with orthogonal axes respectively. Then, label the legs as shown in Fig.6(a). In what follows, obtain the two virtual legs MP and BP as R-R-R-R and R-R-R-P, and indicate that the axes relationships of all joints on the two platforms are arbitrary.

Accordingly, the topological composition of this PM is PM=(L<sub>1</sub>, L<sub>2</sub>, L<sub>3</sub>, L<sub>4</sub>, L<sub>MP</sub>, L<sub>BP</sub>).

The topology matrix L<sub>i</sub> (i=1~3) for 3-UPS legs and the matrix L<sub>4</sub> for the UP leg are expressed as follows.

$$L_i = \begin{bmatrix} 8 & 2 & 2 & 0 & 0 & 0 \\ 2 & 8 & 2 & 1 & 0 & 0 \\ 2 & 2 & 9 & 2 & 0 & 0 \\ 0 & 1 & 2 & 8 & 0 & 0 \\ 0 & 0 & 0 & 0 & 8 & 4 \\ 0 & 0 & 0 & 0 & 4 & 8 \end{bmatrix} \quad (i=1\sim 3), \quad L_4 = \begin{bmatrix} 8 & 2 & 2 \\ 2 & 8 & 2 \\ 2 & 2 & 9 \end{bmatrix}$$

Matrix  $L_{MP}$  and  $L_{BP}$  are

$$L_{MP} = \begin{bmatrix} 8 & 0 & 0 & 0 \\ 0 & 8 & 0 & 0 \\ 0 & 0 & 8 & 0 \\ 0 & 0 & 0 & 9 \end{bmatrix}, \quad L_{BP} = \begin{bmatrix} 8 & 0 & 0 & 0 \\ 0 & 8 & 0 & 0 \\ 0 & 0 & 8 & 0 \\ 0 & 0 & 0 & 8 \end{bmatrix}$$

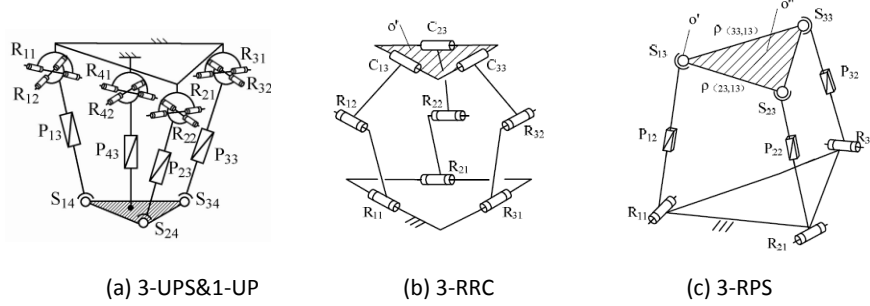


Fig.6. Four PMs

### (2) 3-RRC PM

The 3-RRC PM in Fig.6(b) has three identical RRC legs. First, replace composite pairs C with one R joint and one P joint with collinear axes. Then, label the legs as shown in Fig.6(b) and obtain the two virtual legs MP and BP as P-P-P and R-R-R, and the axes relationships of all joints on the two platforms are coplanar.

Accordingly, the topological composition of 3-RRC PM is  $PM=(L_1, L_2, L_3, L_{MP}, L_{BP})$ .

The topological structure matrices  $L_i(i=1\sim3)$  for 3-RRC legs are

$$L_i = \begin{bmatrix} 8 & 1 & 1 & 1 \\ 1 & 8 & 1 & 1 \\ 1 & 1 & 8 & 1 \\ 1 & 1 & 1 & 9 \end{bmatrix} \quad (i=1\sim3)$$

Matrix  $L_{MP}$  and  $L_{BP}$  are

$$L_{MP} = \begin{bmatrix} 9 & 5 & 5 \\ 5 & 9 & 5 \\ 5 & 5 & 9 \end{bmatrix} \quad \text{and} \quad L_{BP} = \begin{bmatrix} 8 & 5 & 5 \\ 5 & 8 & 5 \\ 5 & 5 & 8 \end{bmatrix}$$

### (3) 3-RPS PM

The 3-RPS PM in Fig.6(c) has three identical RPS legs. First, replace composite pairs S with one R joint and two intersecting R joints. Then, label the legs as shown in Fig.6(c) and obtain the two virtual legs MP and BP both as R-R-R, and the axes relationships of all joints on the two platforms are arbitrary.

Thus, the topological composition of 3-RPS PM is  $PM=(L_1, L_2, L_3, L_{MP}, L_{BP})$ .

The topological structure matrices  $L_i(i=1\sim3)$  for 3-RPS legs are

$$L_i = \begin{bmatrix} 8 & 2 & 1 & 0 & 0 \\ 2 & 9 & 2 & 0 & 0 \\ 1 & 2 & 8 & 2 & 2 \\ 0 & 0 & 2 & 8 & 4 \\ 0 & 0 & 2 & 4 & 8 \end{bmatrix}_{(i)} \quad (i=1\sim3)$$

where, the subscript i represents the No. of intersection point of the last two revolute joints axes.

Matrix  $L_{MP}$  and  $L_{BP}$  are

$$L_{MP} = \begin{bmatrix} 8 & 0 & 0 \\ 0 & 8 & 0 \\ 0 & 0 & 8 \end{bmatrix} \quad \text{and} \quad L_{BP} = \begin{bmatrix} 8 & 0 & 0 \\ 0 & 8 & 0 \\ 0 & 0 & 8 \end{bmatrix}$$

### 3.2.2 Description of POC of a PM

According to the intersection property of sets, the direction of translation/rotation output of a PM is related to one or more joints in one of the legs. Thus, the number of this leg should be recorded for reference. So, the POC of a PM can be completely expressed by adding the number of reference to the legs on the basis of POC of a serial leg. It is defined as

$$M_{pa} = \begin{bmatrix} t_1 & L & t_i L & t_{nj} \\ r_1 & L & r_i L & r_{nj} \end{bmatrix} [L_j], (n_j \leq 6) \quad (13)$$

where,  $L_j$  is the number of the reference leg specifying the direction of translation/rotation output,  $n_j$  is the number of 1-DOF pairs in leg  $L_j$ .

We show this representation with three PMs shown in Fig.6. The PM shown in Fig.6(a) has 2-rotational outputs and 1-translational output, and the POC matrix of this PM can be expressed as  $M_{pa} = \begin{bmatrix} 0 & 0 & 1 \\ 1 & 1 & 0 \end{bmatrix} [4]$ . The right element "4" indicates that the 4<sup>th</sup> leg (UP leg) is the reference leg for the output directions of the moving platform, which means that this mechanism has a translation output along the  $P_{43}$  joint axis, and the directions of rotation outputs are around  $R_{41}$  and  $R_{42}$  joint axes.

The 3-RRC PM shown in Fig.6(b) has 3-translations and its POC is  $M_{pa} = \begin{bmatrix} 3 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix} [1]$ , which shows that 3-RRC has a 3-dimensional translation output and the element '1' indicates the 1<sup>st</sup> leg is the output reference. In fact, 3-dimensional translation output can translate along arbitrary space direction and needs not to be specified.

For the PM in Fig.6(c), point O' on the moving platform is selected as the base point, and its POC is  $M_{pa} = \begin{bmatrix} 2 & 0 & 0 \\ 3 & 0 & 0 \end{bmatrix} [1]$ , which shows that 3-RPS has a 3-dimensional rotation outputs and 2-dimensional translation outputs perpendicular to the axis  $R_{11}$ . In fact, since DOF of the PM is three, the POC has just three independent elements, and the other two elements are non-independent.

## 4. Automatic analysis of POC and dimension of independent displacement equations

### 4.1 Automatic POC analysis of a leg

#### 4.1.1 Algorithm principle

Based on Eq.(3), the POC of a leg is the union operation on POCs of all joints.

Each leg can be considered as a combination of a series of independent constituent units, such as plane sub-chains ( $G_3$  and  $G_2$ ), spherical sub-chains ( $S_3$  and  $S_2$ ) and the remaining 1-DOF R and P joints. Each component unit has a unique corresponding POC matrix (Listed in Table 3 and Table 4). Thus, according to the topological structure matrix defined above, calculating the POC of a leg can be mapped into simply combining the POC matrices of these independent constituent units.

Table 3 and Table 4 show that the POC matrices of these independent constituent units have different dimensions. For convenience of programming, POC matrices of these units (named as  $M_{G_3}$ ,  $M_{G_2}$ ,  $M_{S_3}$ ,  $M_{S_2}$  and  $M_J$ ) will be supplemented to a 2×6 matrix to be dimensionally consistent.

$$M_{*(sp)} = \begin{bmatrix} t_1 & t_2 & t_3 & t_4 & t_5 & t_6 \\ r_1 & r_2 & r_3 & r_4 & r_5 & r_6 \end{bmatrix} \quad (14)$$

The symbol '\*' is to be used to represent  $G_3$ ,  $G_2$ ,  $S_3$ ,  $S_2$  or J.

The specific complement rule must preserve the order of the constituent units in the leg. Take a  $P_1R_2R_3R_4R_5$  leg shown in Fig.2(c) as an example.

In Fig.2(c), the two serial sub-chains  $G_3 (P_1 \perp R_2 // R_3)$  and  $G_2 (R_4 // R_5)$  can be extracted. Their original POC matrices and the supplemented POC matrices are as follows:

$$M_{G3} = \begin{bmatrix} 0 & 2 & 0 \\ 0 & 1 & 0 \end{bmatrix} \Rightarrow M_{G3(sp)} = \begin{bmatrix} 0 & 2 & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 & 0 & 0 \end{bmatrix}$$

$$M_{G2} = \begin{bmatrix} 1 & 0 \\ 1 & 0 \end{bmatrix} \Rightarrow M_{G2(sp)} = \begin{bmatrix} 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 \end{bmatrix}$$

Similarly, for the leg of RPS in Fig.4(a), sub-chains  $G_3$  ( $R_1 \perp P_2 // R_3$ ) and  $S_2$  ( $R_4 * R_5$ ) can be extracted. If point  $O'$  is selected as the base point, their original POC matrices and the supplemented POC matrices are described by

$$M_{G3} = \begin{bmatrix} 2 & 0 & 0 \\ 1 & 0 & 0 \end{bmatrix} \Rightarrow M_{G3(sp)} = \begin{bmatrix} 2 & 0 & 0 & 0 & 0 & 0 \\ 1 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$$

$$M_{S2} = \begin{bmatrix} 0 & 0 & 0 \\ 1 & 1 & 0 \end{bmatrix} \Rightarrow M_{S2(sp)} = \begin{bmatrix} 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 1 & 0 \end{bmatrix}$$

It can be seen that the sequence of joints is retained while expressing the POC. Thus, the POC of a leg can be calculated as

$$M_{L(sp)} = M_{G3(sp)} || M_{G2(sp)} || M_{S3(sp)} || M_{S2(sp)} || M_{J(sp)} \quad (15)$$

where  $M_{G3(sp)}$ ,  $M_{G2(sp)}$ ,  $M_{S3(sp)}$ ,  $M_{S2(sp)}$  and  $M_{J(sp)}$  are the supplemented forms of the POC matrices of  $G_3$ ,  $G_2$ ,  $S_3$ ,  $S_2$  and 1-DOF joints. The symbol ' $||$ ' denotes the logical 'or' operation in each bit of the matrices. As long as one corresponding bit is 1, the result is '1'; otherwise, is '0'.

Hence, one can reason that the process of POC analysis of a leg will be converted into the process of extracting sub-chains and supplementing POC matrices.

#### 4.1.2 Analysis algorithm procedure for the POC of a leg

Here, based on the topology matrices of legs and the extracted sub-chains and 1-DOF joints, an automatic POC analysis method of a leg is presented.

**Step 1** Determine the topology matrix of the leg, and extract successively  $G_3$ ,  $G_2$ ,  $S_3$ ,  $S_2$  and the remaining 1-DOF joints.

**Step 2** Select base point  $O'$ .

**Step 3** Determine the supplemented matrix of the extracted  $G_3$ ,  $G_2$ ,  $S_3$ ,  $S_2$  and the matrix of remaining 1-DOF joints.

**Step 4** Carry out the bitwise "or" operation on the supplemented matrices of  $G_3$ ,  $G_2$ ,  $S_3$ ,  $S_2$  and 1-DOF joints, and obtain the initial POC matrix, which is denoted as  $M_L^0 = \begin{bmatrix} t_1 & t_2 & t_3 & t_4 & t_5 & t_6 \\ r_1 & r_2 & r_3 & r_4 & r_5 & r_6 \end{bmatrix}$ ,  $\sum_{i=1}^6 r_i = \xi_r^0$ ,  $\sum_{i=1}^6 t_i = \xi_t^0$ .

**Step 5** Generate the final POC matrix by converting redundant parallel revolute output into translation output according to the following three rules. The result is recorded as  $M_L = \begin{bmatrix} t_1 & t_2 & t_3 & t_4 & t_5 & t_6 \\ r_1 & r_2 & r_3 & r_4 & r_5 & r_6 \end{bmatrix}$ ,  $\sum_{i=1}^6 r_i = \xi_r$ ,  $\sum_{i=1}^6 t_i = \xi_t$ .

- If  $\xi_r^0 > 3$ , then correct  $\xi_r = 3$  and  $\xi_t = \xi_t^0 + \xi_r^0 - 3$ , and the translation direction newly augmented lies in the normal plane of the parallel revolute joints.

- If  $\xi_r^0 \leq 3$ , the final POC matrix is  $M_L = M_L^0$ , i.e.,  $\xi_t = \xi_t^0$  and  $\xi_r = \xi_r^0$ .

- If  $\xi_t = 2$  (or 3), and the leg exists a two-dimensional derivative translations, then the two-dimensional translations will be reduced to one (or zero) dimension.

#### 4.1.3 Examples of POC analysis of a leg

**Example 1** Determine the POC of the leg  $P \perp R // R \perp R // R$ , which is shown in Fig.2(c).

**Step 1** Recognize and sequentially extract sub-chains. There is one  $G_3$  sub-chain  $P \perp R // R$  and one  $G_2$  planar sub-chain  $R // R$ .

**Step 2** Select base point  $O'$  outside the axis of the end  $R$  pair.

**Step 3** Generate the supplemented POC matrices of  $G_3$  and  $G_2$  as  $M_{G3} = \begin{bmatrix} 0 & 2 & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 & 0 & 0 \end{bmatrix}$ ,  
 $M_{G2} = \begin{bmatrix} 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 \end{bmatrix}$ .

**Step 4** Carry out bitwise “or” operation on  $M_{G3}$  and  $M_{G2}$ , thus the initial POC matrix is  $M_L^0 = \begin{bmatrix} 0P0 & 2P0 & 0P0 & 0P1 & 0P0 & 0P0 \\ 0P0 & 1P0 & 0P0 & 0P1 & 0P0 & 0P0 \end{bmatrix} = \begin{bmatrix} 0 & 2 & 0 & 1 & 0 & 0 \\ 0 & 1 & 0 & 1 & 0 & 0 \end{bmatrix}$ , and  $\xi_r^0=2$ , and  $\xi_t^0=3$ .

**Step 5**  $\xi_t^0=3$  and  $\xi_r^0=2$ , so  $M_L=M_L^0$ , that means  $M_L = \begin{bmatrix} 3 & 0 & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 1 & 0 & 0 \end{bmatrix}$ , and  $\xi_t=3$  indicates that the leg has three independent translation outputs along arbitrary space directions,  $\xi_r=2$  indicates that it has two rotational outputs,  $r_2=r_4=1$  means the two rotations axes are around the joints of  $R_2$  and  $R_4$ , respectively.

**Example 2** Determine the POC of the RPS leg shown in Fig.4(a).

**Step 1** Recognize and sequentially extract sub-chains. There is one  $G_3$  sub-chain R1P//R and one  $S_2$  sub-chain R\*R.

**Step 2** Select base point O' outside the axis of the end R pair.

**Step 3** Generate the supplemented POC matrices of  $G_3$  and  $G_2$  as  $M_{G3} = \begin{bmatrix} 2 & 0 & 0 & 0 & 0 & 0 \\ 1 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$ ,  
 $M_{S2} = \begin{bmatrix} 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 1 & 0 \end{bmatrix} + \begin{bmatrix} 2 \\ \#1 \end{bmatrix}$ .

**Step 4** Carry out bitwise “or” operation on  $M_{G3}$  and  $M_{G2}$ , thus the initial POC matrix is  $M_L^0 = \begin{bmatrix} 2P0 & 0P0 & 0P0 & 0P0 & 0P0 & 0P0 \\ 1P0 & 0P0 & 0P0 & 0P1 & 0P1 & 0P0 \end{bmatrix} + \begin{bmatrix} 2 \\ \#1 \end{bmatrix} = \begin{bmatrix} 2 & 0 & 0 & 0 & 0 & 0 \\ 3 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} + \begin{bmatrix} 2 \\ \#1 \end{bmatrix}$ , and  $\xi_r^0=3$ , and  $\xi_t^0=2$ ,  $\begin{bmatrix} 2 \\ \#1 \end{bmatrix}$ .

**Step 5**  $\xi_r^0=3$ ,  $\xi_t^0=2$  and  $\begin{bmatrix} 2 \\ \#1 \end{bmatrix}$ , so  $M_L = \begin{bmatrix} 2 & 0 & 0 & 0 & 0 & 0 \\ 3 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} + \begin{bmatrix} 1 \\ \#1 \end{bmatrix}$ . Therefore, the leg has two independent translational outputs in the normal plane of axis  $R_1$  and three rotational outputs around the spherical center #1, and  $\begin{bmatrix} 1 \\ \#1 \end{bmatrix}$  shows that the effector has one derivative translation within the plane perpendicular to radius vector  $\rho_1$ .

#### 4.2 Automatic POC analysis of a PM

POC of a PM is the result of the intersection operations on all legs in the PM. Among them, the translation output of the PM is the intersection of translational outputs of all legs, and the same works for the rotation outputs.

According to the kinematic characteristics of rigid body, the rotational outputs has no effect on the translational output, but the translational outputs affect the position of the rotation axis. So intersection rules for rotation outputs of POC will be more complicated.

Given are two legs  $L_1$  and  $L_2$ , numbered as  $l_1$  and  $l_2$  respectively. The POC matrices of these legs are described as

$M_{Li} = \begin{bmatrix} t_{i1} & t_{i2} & t_{i3} & t_{i4} & t_{i5} & t_{i6} \\ r_{i1} & r_{i2} & r_{i3} & r_{i4} & r_{i5} & r_{i6} \end{bmatrix}$ . According to Eq.(11), the dimensions of translation and rotation outputs will

be  $\xi_{ti} = \sum_{j=1}^6 t_{ij}$  and  $\xi_{ri} = \sum_{j=1}^6 r_{ij}$  ( $i=1, 2$ ), and the non-zero elements of  $M_{Li}$  imply the corresponding directions.

For convenience of description, the translation and rotation output of leg  $l_i$  are denoted by  $G_i=(\xi_{ti}, e_i)$  and  $H_i=(\xi_{ri}, s_i)$ , where  $e_i$  and  $s_i$  are the directions of translation/rotation output of this leg. According to the above definition,  $e_i/s_i$  can be divided into four different types:

- (1) if  $\xi_{ti}=0/\xi_{ri}=0$ , then  $e_i/s_i=\Phi$  (empty set);
- (2) if  $\xi_{ti}=1/\xi_{ri}=1$ , then  $e_i/s_i$  is a spatial line;
- (3) if  $\xi_{ti}=2/\xi_{ri}=2$ , then  $e_i/s_i$  is a plane;
- (4) if  $\xi_{ti}=3/\xi_{ri}=3$ , then  $e_i/s_i$  is denoted as “-” (arbitrary in space).



Suppose the intersection result is described as  $M_{pa(1\sim 2)} = \begin{bmatrix} t_1 & t_2 & t_3 & t_4 & t_5 & t_6 \\ r_1 & r_2 & r_3 & r_4 & r_5 & r_6 \end{bmatrix} [l_j]$  ( $j=1$  or  $2$ ), where  $j$  is the number of reference leg.

Similarly, the dimension of translation/rotation output is denoted by  $\xi_{t(1\sim 2)} = \sum_{i=1}^6 t_i / \xi_{r(1\sim 2)} = \sum_{i=1}^6 r_i$ .

The translation and rotation output can be expressed as  $G_{(1\sim 2)} = (\xi_{t(1\sim 2)}, e_{(1\sim 2)})$  and  $H_{(1\sim 2)} = (\xi_{r(1\sim 2)}, s_{(1\sim 2)})$ , where  $e_{(1\sim 2)}$  and  $s_{(1\sim 2)}$  are the corresponding directions.

Note that the legs  $L_1$  and  $L_2$ , which were used to introduce the computation algorithm, can be either the legs constituting a PM or a sub-PM composed by several legs. Similarly, the result of the above intersection operation can also be regarded as the POC of a sub-PM continuing to participate in other operations. The intersection output of matrices of two POCs can be calculated following these two rules.

#### 4.2.1 Intersection rules for translation output of POCs

In the process of intersection operation, since the rotation outputs have no influence on the translation outputs, the translation outputs can be intersected separately. Actually, the intersection operation of translations of two legs is to solve  $G_{(1\sim 2)} = G_1 \cap G_2$ , which can be obtained according the rules shown in Table 5.

**Table 5** Intersection rules for translation output of POCs

$L_2 \backslash L_1$	$\xi_{t1}=0$	$\xi_{t1}=1$		$\xi_{t1}=2$		$\xi_{t1}=3$
		$e_1    e_2$	$e_1 \nmid e_2$	$e_1    e_2$	$e_1 \nmid e_2$	
$\xi_{t2}=0$	$G_2$		$G_2$		$G_2$	$G_2$
$\xi_{t2}=1$	$G_1$	$G_2$	$\xi_{t(1\sim 2)}=0, e_{(1\sim 2)}=\Phi$	$G_2$	$\xi_{t(1\sim 2)}=0, e_{(1\sim 2)}=\Phi$	$G_2$
$\xi_{t2}=2$	$G_1$	$G_1$	$\xi_{t(1\sim 2)}=0, e_{(1\sim 2)}=\Phi$	$G_2$	$\xi_{t(1\sim 2)}=1, e_{(1\sim 2)}=e_1 \cap e_2$	$G_2$
$\xi_{t2}=3$	$G_1$		$G_1$		$G_1$	$G_1$

#### 4.2.2 Intersection rules for rotation output of POCs

Similarly, the intersection of rotational output of two POC matrices is to solve  $H_{(1\sim 2)} = H_1 \cap H_2$ . However, as stated above, translation outputs affect the position of the rotation axis. Therefore, the translational outputs of a leg must be taken into account before establishing the intersection rules of the rotational output of POCs.

It is well known, if a leg has 2-dimensional translations and 1-dimensional rotation perpendicular to the 2D translation plane, no matter where the revolute joint is, the end link of the leg can rotate around any axis to be perpendicular to the 2D plane. Likewise, if a leg has 3-dimensional translations and rotations, no matter where the rotation joints are, the end link of the leg can rotate around any axes parallel to the rotational joints. This type of rotational output is called as a fully-free output (Denoted as FF-type), and anything else is called a non-fully-free output (Denoted as non-FF-type). Generally, for the 3-DOF non-FF-type rotation type, it is usually caused by pair  $S$ , so it is considered that the three axes intersect at one point, i.e., the spherical center.

On the basis of defining the type of rotational output, the intersection operation rules for rotation output can be divided into three types and listed in Table 6. It is noted that, the symbol “\*” in the table indicates that the three axes have intersection points, and the superscript “a” indicates that the output axis will pass through the intersection point.

It should be noted that the number of the reference leg ( $l_j$ ) can be concluded directly from the result of the intersection operations. As it can be seen from Table 5 and Table 6, the result of  $G_1$  or  $G_2$  indicates that  $l_j$  is  $L_1$  or  $L_2$  correspondingly. In particular, when the result of intersection is not capable of motion, the output will be shown as ‘ $\Phi$ ’ (Empty set).

**Table 6** Intersection rules for rotation output

L <sub>1</sub> : FF-type, L <sub>2</sub> : FF-type							
L <sub>2</sub> \ L <sub>1</sub>	$\xi_{r1}=0$	$\xi_{r1}=1$		$\xi_{r1}=2$		$\xi_{r1}=3$	
		$s_1   s_2$ or $s_1   s_2$	$s_1 \uparrow s_2$ and $s_1 \uparrow s_2$	$s_1   s_2$ or $s_1   s_2$	$s_1 \uparrow s_2$ and $s_1 \uparrow s_2$		
$\xi_{r2}=0$	H <sub>1</sub>		H <sub>2</sub>		H <sub>2</sub>		H <sub>2</sub>
$\xi_{r2}=1$	H <sub>1</sub>	H <sub>1</sub>	$\xi_{r(1\sim2)}=0,$ $s_{(1\sim2)}=\Phi$		H <sub>2</sub>	$\xi_{r(1\sim2)}=0,$ $s_{(1\sim2)}=\Phi$	H <sub>2</sub>
$\xi_{r2}=2$	H <sub>1</sub>	H <sub>1</sub>	$\xi_{r(1\sim2)}=0,$ $s_{(1\sim2)}=\Phi$		H <sub>2</sub>	$\xi_{r(1\sim2)}=1,$ $s_{(1\sim2)}=s_1 \cap s_2$	H <sub>2</sub>
$\xi_{r2}=3$	H <sub>1</sub>		H <sub>1</sub>		H <sub>1</sub>		H <sub>2</sub>

L <sub>1</sub> : FF-type, L <sub>2</sub> : non-FF-type							
L <sub>2</sub> \ L <sub>1</sub>	$\xi_{r1}=0$	$\xi_{r1}=1$		$\xi_{r1}=2$		$\xi_{r1}=3$	
		$s_1   s_2$ or $s_1   s_2$	$s_1 \uparrow s_2$ and $s_1 \uparrow s_2$	$s_1   s_2$ or $s_1   s_2$	$s_1 \uparrow s_2$ and $s_1 \uparrow s_2$		
$\xi_{r2}=0$	H <sub>1</sub>		H <sub>2</sub>		H <sub>2</sub>		H <sub>2</sub>
$\xi_{r2}=1$	H <sub>1</sub>	H <sub>1</sub>	$\xi_{r(1\sim2)}=0,$ $s_{(1\sim2)}=\Phi$		H <sub>2</sub>	$\xi_{r(1\sim2)}=0,$ $s_{(1\sim2)}=\Phi$	H <sub>2</sub>
$\xi_{r2}=2$	H <sub>1</sub>	H <sub>1</sub> <sup>a</sup>	$\xi_{r(1\sim2)}=0,$ $s_{(1\sim2)}=\Phi$		H <sub>2</sub>	$\xi_{r(1\sim2)}=1,$ $s_{(1\sim2)}=s_1 \cap s_2$	H <sub>2</sub>
$\xi_{r2}=3^*$	H <sub>1</sub>		H <sub>1</sub> <sup>a</sup>		H <sub>1</sub> <sup>a</sup>		H <sub>2</sub>

L <sub>1</sub> : non-FF-type, L <sub>2</sub> : non-FF-type							
L <sub>2</sub> \ L <sub>1</sub>	$\xi_{r1}=0$	$\xi_{r1}=1$		$\xi_{r1}=2$		$\xi_{r1}=3^*$	
		$s_1   s_2$	$s_1 \uparrow s_2$	$s_1   s_2$	$s_1 \uparrow s_2$	$s_1   s_2$	$s_1 \uparrow s_2$
$\xi_{r2}=0$	H <sub>1</sub>		H <sub>2</sub>		H <sub>2</sub>		H <sub>2</sub>
$\xi_{r2}=1$	H <sub>1</sub>	H <sub>1</sub>	$\xi_{r(1\sim2)}=0,$ $s_{(1\sim2)}=\Phi$		H <sub>2</sub>	$\xi_{r(1\sim2)}=0,$ $s_{(1\sim2)}=\Phi$	H <sub>2</sub>
$\xi_{r2}=2$	H <sub>1</sub>	H <sub>1</sub>	$\xi_{r(1\sim2)}=0,$ $s_{(1\sim2)}=\Phi$		H <sub>2</sub>	$\xi_{r(1\sim2)}=0,$ $s_{(1\sim2)}=\Phi$	H <sub>2</sub> <sup>a</sup>
$\xi_{r2}=3^*$	H <sub>1</sub>	H <sub>1</sub> <sup>a</sup>	$\xi_{r(1\sim2)}=0,$ $s_{(1\sim2)}=\Phi$		H <sub>1</sub>	$\xi_{r(1\sim2)}=0,$ $s_{(1\sim2)}=\Phi$	H <sub>2</sub>

#### 4.2.3 Analysis algorithm procedure for the POC of a PM

In accordance with the analysis above, the fully-automatic POC analysis procedure of PMs is as follows.

**Step 1** Input the number of legs  $n$  and POC matrix of each leg  $M_{Li}$  ( $i=1\sim n$ ) including two virtual legs of the two platforms  $M_{MP}$  and  $M_{LP}$ .

**Step 2** Choose the 1<sup>st</sup> and the 2<sup>nd</sup> legs to form a sub-PM denoted as sub-PM<sub>(1~2)</sub>.

**Step 3** Calculate the POC matrix  $M_{pa(1\sim2)}$  of sub-PM<sub>(1~2)</sub>.

- Obtain intersection of translation outputs  $G_{(1\sim2)}$
- Obtain intersection of rotation outputs  $H_{(1\sim2)}$
- Obtain the result of POC  $M_{pa(1\sim2)}$  of the sub-PM<sub>(1~2)</sub>

**Step 4** Repeat Step 2 and Step 3 for constructing sub-PM<sub>(1~3)</sub> (consisted by sub-PM<sub>(1~2)</sub> and the 3<sup>rd</sup> leg), ..., sub-PM<sub>(1~n)</sub> (consisted by sub-PM<sub>(1~(n-1))</sub> and the n<sup>th</sup> leg), and calculation of the corresponding M<sub>pa(1~3)</sub>, ..., M<sub>pa(1~n)</sub>.

**Step 5** Output the POC result of PM: M<sub>pa</sub>=M<sub>pa(1~n)</sub>.

Taking the 3-RRC PM for example, the structural sketch is shown in Fig.6(b). The topology matrices of legs, and joints on the two platforms have been described in Section 3.2.1. The POC of 3-RRC PM can be analyzed as follows.

**Step 1** Input the number of legs, and POC of each leg.

Obviously the number of legs is n=3.

According to Eq.(15), the topology matrices of 3-RRC legs can be determined as

$$\begin{aligned} M_{L_i} &= \begin{bmatrix} 2 & 0 & 0 & 0 & 0 & 0 \\ 1 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} \parallel \begin{bmatrix} 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} \\ &= \begin{bmatrix} 2P0 & 0P0 & 0P0 & 0P1 & 0P0 & 0P0 \\ 1P0 & 0P0 & 0P0 & 0P0 & 0P0 & 0P0 \end{bmatrix} = \begin{bmatrix} 2 & 0 & 0 & 1 & 0 & 0 \\ 1 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} \quad (i=1,2,3). \\ &= \begin{bmatrix} 3 & 0 & 0 & 0 & 0 & 0 \\ 1 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} = \begin{bmatrix} G_i \\ H_i \end{bmatrix} \end{aligned}$$

Thus,  $\xi_{ti}=3$  means that there are arbitrary translations in space, and  $\xi_{ri}=1$  is a one-dimension rotation around the R<sub>i1</sub> axis in the i<sup>th</sup> leg, i.e.  $s_i=R_{i1}$ .

**Step 2** Calculate the POC matrix M<sub>pa(1~2)</sub> of sub-PM<sub>(1~2)</sub> constituted by the 1<sup>st</sup> and the 2<sup>nd</sup> legs.

According to Table 5,  $\xi_{t1}=\xi_{t2}=3$ , then  $\xi_{t(1~2)}=3$ , and  $G_{(1~2)}=G_2$ .

Since  $\xi_{r1}=\xi_{r2}=1$ ,  $s_1=R_{11}$ ,  $s_2=R_{21}$ , and  $R_{11} \perp R_{21}$ , we obtain  $s_1 \perp s_2$ . According to Table 6, then  $\xi_{r(1~2)}=0$ , and  $S_{(1~2)}=\Phi$ .

$$\text{Then, the POC matrix of sub-PM}_{(1~2)} \text{ is } M_{pa(1~2)} = \begin{bmatrix} 2 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} = \begin{bmatrix} G_{(1~2)} \\ H_{(1~2)} \end{bmatrix}.$$

**Step 3** Calculate the POC matrix M<sub>pa(1~3)</sub> of sub-PM<sub>(1~3)</sub> constituted by the sub-PM<sub>(1~2)</sub> and the third RRC leg.

According to Table 5,  $\xi_{t(1~2)}=\xi_{t3}=3$ , then  $\xi_{t(1~3)}=3$ , and  $G_{(1~3)}=G_3$ .

According to Table 6,  $\xi_{r(1~2)}=0$ ,  $\xi_{r3}=1$ , and  $s_{(1~2)}=\Phi$ ,  $s_3=R_{31}$ , then,  $\xi_{r(1~3)}=0$ ,  $s_{(1~3)}=\Phi$ .

$$\text{Thus, the POC matrix of sub-PM}_{(1~3)} \text{ is } M_{pa(1~3)} = \begin{bmatrix} 2 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} = \begin{bmatrix} 3 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} = \begin{bmatrix} G_{(1~3)} \\ H_{(1~3)} \end{bmatrix} = \begin{bmatrix} G_3 \\ \Phi \end{bmatrix}$$

**Step 4** Output the result of the POC of this PM

$$M_{pa} = M_{pa(1~3)} = \begin{bmatrix} G_3 \\ \Phi \end{bmatrix}$$

The POC of the 3-RRC PM is a 3-dimensional translation, and this result is consistent with the result in [18].

### 4.3 Calculating the number of independent displacement equations

Eq.(6) shows that solving the number of independent displacement equations of an independent loop involves the union operation of POC matrices. For the legs L<sub>1</sub> and L<sub>2</sub> mentioned above, suppose the resulting matrix of the union operation to be

$$M_{L(IU2)} = \begin{bmatrix} t_1 & t_2 & t_3 & t_4 & t_5 & t_6 \\ r_1 & r_2 & r_3 & r_4 & r_5 & r_6 \end{bmatrix}$$

The dimension of independent output is

$$\xi_{Lt(IU2)} = \sum_{i=1}^6 t_i / \xi_{Lr(IU2)} = \sum_{i=1}^6 r_i$$

The translation output is  $G_{(1U2)} = (\xi_{Lt(1U2)}, e_{(1U2)})$  and the rotation output is  $H_{(1U2)} = (\xi_{Lr(1U2)}, s_{(1U2)})$ , where  $e_{(1U2)}/s_{(1U2)}$  is the direction of translation/rotation. Thus, the number of independent displacement equations of the independent loop is  $\xi_{L(1U2)} = \xi_{Lt(1U2)} + \xi_{Lr(1U2)}$ .

#### 4.3.1 Rules for the dimension of translation output of SLC

Solving the dimension of translation output of an SLC means essentially to calculate  $\xi_{Lt(1U2)} = \dim(G_1UG_2)$ . The corresponding operation rules are shown in Table 7.

**Table 7.** Rules for the dimension of the translation output of SLC

$L_2 \backslash L_1$	$\xi_{t1}=0$	$\xi_{t1}=1$		$\xi_{t1}=2$		$\xi_{t1}=3$
		$e_1    e_2$	$e_1 \uparrow\uparrow e_2$	$e_1    e_2$	$e_1 \uparrow\uparrow e_2$	
$\xi_{t2}=0$	0		1		2	3
$\xi_{t2}=1$	1	1	2	2	3	3
$\xi_{t2}=2$	2	2	3	2	3	3
$\xi_{t2}=3$	3		3		3	3

#### 4.3.2 Rules for the dimension of rotation output of SLC

Similarly, the dimension of the rotation output of SLC is to calculate  $\xi_{Lr(1U2)} = \dim(H_1UH_2)$ , and its operation rules are shown in Table 8.

**Table 8.** Rules for the dimension of the rotation output of SLC

$L_2 \backslash L_1$	$\xi_{r1}=0$	$\xi_{r1}=1$		$\xi_{r1}=2$		$\xi_{r1}=3$
		$e_1    e_2$	$e_1 \uparrow\uparrow e_2$	$e_1    e_2$	$e_1 \uparrow\uparrow e_2$	
$\xi_{r2}=0$	0		1		2	3
$\xi_{r2}=1$	1	1	2	2	3	3
$\xi_{r2}=2$	2	2	3	2	3	3
$\xi_{r2}=3$	3		3		3	3

The rules described above does not only work for serial legs but also for equivalent sub-PMs. Therefore, it is emphasized once more that the legs  $L_1$  and  $L_2$ , which were used to introduce the computation algorithm, can be either the legs constituting a PM or a sub-PM composed by several legs.

## 5. Procedure of automatic mobility analysis of PMs

Based on the POC algorithm for a leg, for a PM and for the number of independent displacement equations, the steps of mobility analysis of PMs are illustrated below.

**Step 1** Input topological structure of a PM.

Number  $1 \sim (v+1)$  of legs in sequence and input the corresponding topology matrix of each leg as  $L_1, \dots, L_{(v+1)}$ . Then, input the topology matrix between legs,  $L_{MP}$  characterizes a virtual leg constituted by pairs of joints on the moving platform and  $L_{BP}$  is analogously defined on the fixed platform. Based on above, the number of joints  $f_i$  and the number of legs  $v+1$  can be obtained automatically.

It should be pointed out that for the PM to be analyzed, the legs can be numbered arbitrarily and does not affect the result of the mobility analysis. However, once the legs are numbered, the order of the two virtual legs should be basic on the number.

**Step 2** Select base point  $O'$  on the moving platform.

**Step 3** Calculate POC matrices of  $(v+1)$  legs, denoted as  $M_{L1}, \dots, M_{L(v+1)}$ .

**Step 4** Calculate the sum of all joints:  $f_{(1 \sim (v+1))} = \sum_{i=1}^{v+1} f_i$ .

**Step 5** Calculate the POC matrix  $M_{pa(1\sim j)}$  of the sub- $PM_{(1\sim j)}$  composed by the front  $j^{\text{th}}$  legs connected in parallel ( $j=2, \dots, v$ ).

- ① Calculate the translation output  $G_{(1\sim j)} = G_{(1\sim(j-1))} \cap G_{(j+1)}$ .
- ② Calculate the rotation output  $H_{(1\sim j)} = H_{(1\sim(j-1))} \cap H_{(j+1)}$ .

- ③ Get  $M_{pa(1\sim j)} = \begin{bmatrix} G_{(1\sim j)} \\ H_{(1\sim j)} \end{bmatrix}$ .

**Step 6** Calculate the number of the independent displacement equations  $\xi_j$  of the  $j^{\text{th}}$  loop  $SLC_j$ , which is composed by the sub- $PM_{(1\sim j)}$  and the  $(j+1)^{\text{th}}$  leg connected in series ( $j=1, \dots, v$ ).

- ① Calculate the dimension of the independent translation output of  $SLC_1$ :  $\xi_{jt} = \dim(G_{(1\sim j)} \cup G_{(j+1)})$ .
- ② Calculate the dimension of the independent rotation output of  $SLC_1$ :  $\xi_{jr} = \dim(H_{(1\sim j)} \cup H_{(j+1)})$ .
- ③ Calculate  $\xi_{lj} = \xi_{jt} + \xi_{jr}$ .

**Step 6** Calculate the number of DOF:  $F = f_{(1\sim(v+1))} - \sum_{i=1}^v \xi_i$ .

**Step 7** Determine the mobility property of this PM:  $M_{pa} = M_{pa(1\sim v)} = M_{pa(1\sim(v-1))} \cap M_{L(v+1)}$ .

The corresponding flow process chart of an automatic mobility analysis is shown in Fig.7.

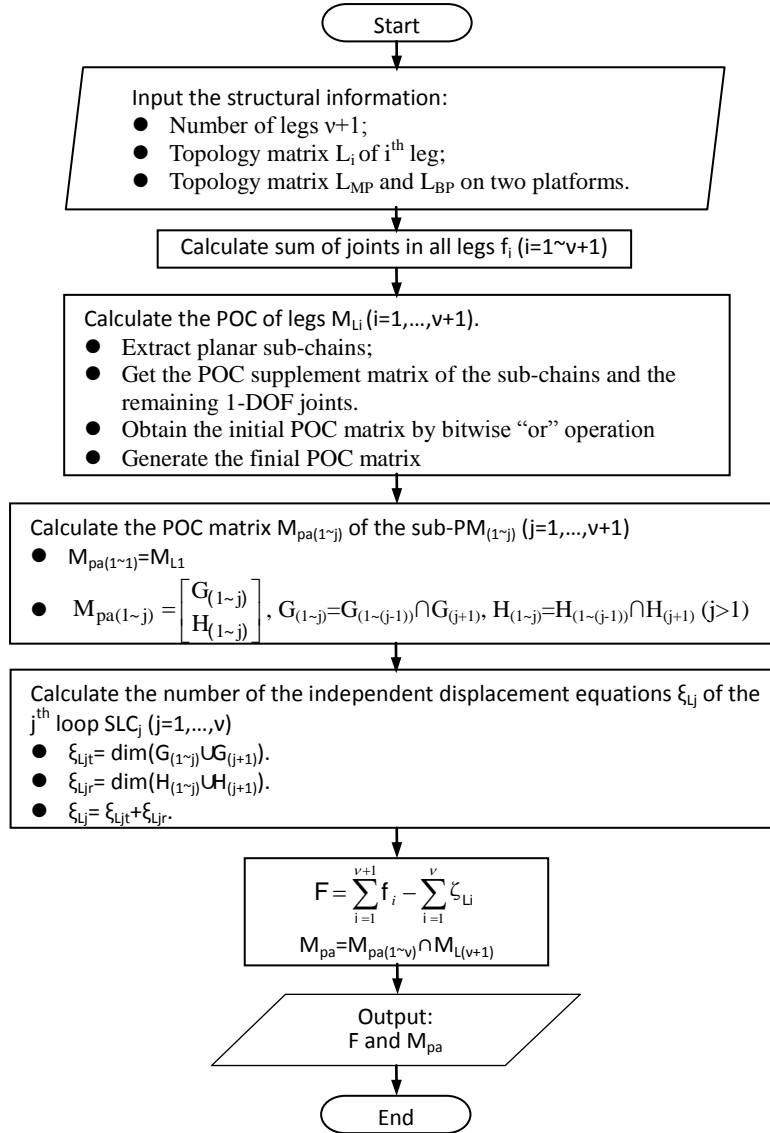


Fig. 7. Procedure of mobility analysis of a PM

## 6. Software implementation and case study

### 6.1 Software implementation

Based on the basic principles, algorithmic rules and algorithmic procedures, a program which can analyze the mobility of PMs automatically has been developed on Windows platform with Visual C++6.0 software and object-oriented programming language. Using the software, a large number of PMs listed in [18] have been analyzed automatically and the effectiveness of the presented method has been testified.

Fig.8 presents the developed human-computer interactive interface. The input information is located at the top of the interactive interface. It mainly includes the number of legs, topological structure matrices of two virtual legs on the two platforms and the topology structure matrices of all legs. It is using integer arrays and is easy to operate.

What needs to be added is that, as the topology matrices are symmetric, only upper triangular elements need to be input in the human-computer interaction interface for convenience.

In addition to DOF and POC (i.e., Number of POC Elements, T-directions and R-directions), other topological features can be obtained, such as independent loops, independent displacement equations. Due to limited space, this work does not go into all details.

### 6.2 Case study

Detailed examples of the mobility analysis of four typical PMs are presented in this section.

#### 6.2.1 Mobility analysis of Tricept PM

The Tricept non-overstrained PM shown in Fig.6(a) has 3-DOF which comprise one translation and two rotations. The legs are 3-UPS and 1-UP. The axes of the joints on the two platforms are arbitrary to each other.

(1) Input topological structure matrices of this PM

$$\text{Topology matrix of UPS leg is } L_i = \begin{bmatrix} 8 & 2 & 2 & 0 & 0 & 0 \\ 2 & 8 & 2 & 1 & 0 & 0 \\ 2 & 2 & 9 & 2 & 0 & 0 \\ 0 & 1 & 2 & 8 & 0 & 0 \\ 0 & 0 & 0 & 0 & 8 & 4 \\ 0 & 0 & 0 & 0 & 4 & 8 \end{bmatrix}, (i=1\sim 3).$$

$$\text{Topology matrix of UP leg is } L_4 = \begin{bmatrix} 8 & 2 & 2 \\ 2 & 8 & 2 \\ 2 & 2 & 9 \end{bmatrix}.$$

$$\text{Topology matrices of two virtual legs on the two platforms } L_{MP} \text{ and } L_{BP} \text{ are respectively, } L_{MP} = \begin{bmatrix} 8 & 0 & 0 & 0 \\ 0 & 8 & 0 & 0 \\ 0 & 0 & 8 & 0 \\ 0 & 0 & 0 & 9 \end{bmatrix},$$

$$L_{BP} = \begin{bmatrix} 8 & 0 & 0 & 0 \\ 0 & 8 & 0 & 0 \\ 0 & 0 & 8 & 0 \\ 0 & 0 & 0 & 8 \end{bmatrix}.$$

Number of legs  $v+1=4$ .

(2) Select an arbitrary point  $O'$  on the moving platform as the base point.

(3) Calculate POC matrices of legs  $M_{L1}, \dots, M_{L4}$ .

$$\textcircled{1} \quad M_{L_i} = \begin{bmatrix} 3 & 0 & 0 & 0 & 0 & 0 \\ 3 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} = \begin{bmatrix} G_i \\ H_i \end{bmatrix}, \xi_{ri}=3, \text{ and } \xi_{ti}=3 (i=1\sim 3)$$

$$\textcircled{2} \quad M_{L_4} = \begin{bmatrix} 0 & 0 & 1 & 0 & 0 & 0 \\ 1 & 1 & 0 & 0 & 0 & 0 \end{bmatrix} = \begin{bmatrix} G_4 \\ H_4 \end{bmatrix}, \text{ and } \xi_{t4}=1, \xi_{r4}=2, e_4=P_{43}, s_4=(R_{41}, R_{42}).$$

(4) Get the sum of all joints in this PM:  $f_{(1-4)}=6+6+6+3=21$ .

(5) Calculate the POC matrices  $M_{pa(1\sim j)}$  of sub-PM $_{(1\sim j)}$  constituted by the front  $j$  legs in parallel ( $j=1\sim 3$ ).

- Set  $M_{pa(1\sim 1)}=M_{L1}$

- Calculate the POC matrix  $M_{pa(1\sim2)}$  of sub-PM $_{(1\sim2)}$  constituted by the 1<sup>st</sup> and the 2<sup>nd</sup> leg in parallel.

①  $\xi_{ti}=3$ , as shown in Table 5,  $G_{(1\sim2)}=G_{(1\sim1)}\cap G_2=G_2$  ( $i=1,2$ ).

②  $\xi_{ri}=3$ , as shown in Table 6,  $H_{(1\sim2)}=H_{(1\sim1)}\cap H_2=H_2$  ( $i=1,2$ ).

③ So,  $M_{pa(1\sim2)}=\begin{bmatrix} 3 & 0 & 0 & 0 & 0 & 0 \\ 3 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}=\begin{bmatrix} G_2 \\ H_2 \end{bmatrix}$ .

- Calculate the POC matrix of sub-PM $_{(1\sim2)}$  composed by sub-PM $_{(1\sim2)}$  and the 3<sup>rd</sup> leg:

$$M_{pa(1\sim3)}=M_{pa(1\sim2)}\cap M_{L3}=\begin{bmatrix} G_2 \\ H_2 \end{bmatrix}\cap\begin{bmatrix} G_3 \\ H_3 \end{bmatrix}=\begin{bmatrix} G_3 \\ H_3 \end{bmatrix}.$$

(5) Calculate the number of the independent displacement equations  $\xi_{Lj}$  of SLC $_j$  ( $j=1\sim3$ ).

- Calculate  $\xi_{L1}$  of the first loop SLC $_1$  concatenated by 1<sup>st</sup> and 2<sup>nd</sup> leg,  $\xi_{L1}=\dim(M_{pa(1\sim1)}\cup M_{L2})$ .

①  $\xi_{ti}=3$ , as shown in Table 7,  $\xi_{L1t}=3$  ( $i=1,2$ ).

②  $\xi_{ri}=3$ , as shown in Table 8,  $\xi_{L1r}=3$  ( $i=1,2$ ).

③ So,  $\xi_{L1}=\xi_{L1t}+\xi_{L1r}=3+3=6$ .

- Calculate  $\xi_{L2}$  of SLC $_2$ :  $\xi_{L2}=\dim(M_{pa(1\sim2)}\cup M_{L3})$ .

①  $\xi_{t(1\cap2)}=3$ , and  $\xi_{t3}=3$ , then  $\xi_{L2t}=3$ .

②  $\xi_{r(1\cap2)}=3$ , and  $\xi_{r3}=3$ , then  $\xi_{L2r}=3$ .

③ thus,  $\xi_{L2}=\xi_{L2t}+\xi_{L2r}=3+3=6$ .

- Similarly, the number of the independent displacement equations of SLC $_3$  is  $\xi_{L3}=\xi_{L3t}+\xi_{L3r}=3+3=6$ .

(6) Get the number of DOF:  $F=f_{(1\sim4)}-\sum_{i=1}^3\xi_i=21-(6+6+6)=3$ .

(7) Get the mobility property:

$$M_{pa}=M_{pa(1\sim3)}\cap M_{L4}=\begin{bmatrix} G_3 \\ H_3 \end{bmatrix}\cap\begin{bmatrix} G_4 \\ H_4 \end{bmatrix}=\begin{bmatrix} G_4 \\ H_4 \end{bmatrix}.$$

The results show that this PM has 1T2R output, the rotation directions are around  $R_{41}$  and  $R_{42}$ , and the translation direction is along the  $P_{43}$  joint. The result generated by the software is shown in Fig.8, which is consistent with the result in [18].

Fig. 8. Mobility analysis of Tricept

### 6.2.2 Mobility analysis of 3-RRC PM

Fig.6(b) shows a 3-RRC PM with overstrains. It has three identical RRC legs connecting the two platforms, labeled with the numbers 1~3 respectively. The topological structure of this PM is  $PM=(L_1,L_2,L_3,L_{MP},L_{BP})$ . The fixed platform is a triangle, and the axes of the joints on the two platforms are all coplanar.

(1) Input topology matrices of the PM

Topology matrix of RRC leg is  $L_i = \begin{bmatrix} 8 & 1 & 1 & 1 \\ 1 & 8 & 1 & 1 \\ 1 & 1 & 8 & 1 \\ 1 & 1 & 1 & 9 \end{bmatrix}$ , ( $i=1\sim 3$ ).

Topology matrices of two virtual legs on the two platforms  $L_{MP}$  and  $L_{BP}$  are  $L_{MP} = \begin{bmatrix} 9 & 5 & 5 \\ 5 & 9 & 5 \\ 5 & 5 & 9 \end{bmatrix}$  and  $L_{BP} = \begin{bmatrix} 8 & 5 & 5 \\ 5 & 8 & 5 \\ 5 & 5 & 8 \end{bmatrix}$

respectively.

Number of legs  $v+1=3$ .

(2) Select an arbitrary point  $O'$  on the moving platform as the base point.

(3) Calculate POC matrix of legs  $M_{L1}$ ,  $M_{L2}$  and  $M_{L3}$ .

① Recognize and sequentially extract planar and spherical sub-chains: one  $G_3$  sub-chain  $R//R//R$  and one 1-DOF P joint.

② Generate the POC supplemented matrices of  $G_3$  and P as  $M_{G3} = \begin{bmatrix} 2 & 0 & 0 & 0 & 0 & 0 \\ 1 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$  and  $M_P = \begin{bmatrix} 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$ .

③ Carry out "or" operation on POC matrices in the leg  $M_{Li} = \begin{bmatrix} 2 & 0 & 0 & 1 & 0 & 0 \\ 1 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} = \begin{bmatrix} 3 & 0 & 0 & 0 & 0 & 0 \\ 1 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$ ,  $\xi_r^0=1$ , and  $\xi_t^0=3$  ( $i=1\sim 3$ )

④  $\xi_t=3$  and  $\xi_r=1$ , then  $M_{Li} = \begin{bmatrix} 3 & 0 & 0 & 0 & 0 & 0 \\ 1 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$ , ( $i=1\sim 3$ ). That is to say,  $\xi_{ti}=3$  indicates that the leg has three translation outputs along arbitrary directions,  $\xi_{ri}=1$  indicates that it has one rotation output.  $r_{i1}=1$  means that the rotation is around the axes of  $R_{i1}$ , i.e.  $s_i=R_{i1}$ .

(4) Calculate the sum of all joints in this PM:  $f_{(1-3)}=4+4+4=12$ .

(5) Calculate the POC matrix  $M_{pa(1-j)}$  of sub-PM $_{(1-j)}$  constituted by the front  $j$  legs in parallel.

● Set  $M_{pa(1-1)}=M_{L1}$

● Calculate the POC matrix  $M_{pa(1-2)}$  of sub-PM $_{(1-2)}$  with  $M_{pa(1-2)}=M_{pa(1-1)}\cap M_{L2}$ .

①  $\xi_{ti}=3$ , as shown in Table 5,  $G_{(1-2)}=G_2$  ( $i=1,2$ ).

②  $\xi_{ri}=1$ , and  $s_1 \perp s_2$ , as shown in Table 6,  $H_{(1-2)}=\Phi$  ( $i=1,2$ ).

③ So,  $M_{pa(1-2)} = \begin{bmatrix} G_2 \\ \Phi \end{bmatrix}$

(6) Calculate the number of the independent displacement equations  $\xi_{lj}$  of SLC $_j$  ( $j=1\sim 2$ ).

● Calculate  $\xi_{l1}$  of the first loop SLC $_1$  concatenated by 1<sup>st</sup> and 2<sup>nd</sup> leg.

①  $\xi_{ti}=3$ , as shown in Table 7,  $\xi_{l1t}=3$  ( $i=1,2$ ).

②  $\xi_{ri}=1$ , as shown in Table 8, and  $s_1 \perp s_2$ , then,  $\xi_{l1r}=2$  ( $i=1,2$ ).

③ So,  $\xi_{l1}=\xi_{l1t}+\xi_{l1r}=3+2=5$ .

● Calculate the number of the independent displacement equations  $\xi_{l2}$  of SLC $_2$  ( $\xi_{l2}=\dim(M_{pa(1-2)}\cup M_{L3})$ )

①  $\xi_{t3}=3$ , and  $G_{(1-2)}=G_2$ , as shown in Table 6,  $\xi_{l2t}=3$ .

②  $\xi_{r3}=1$ , and  $H_{(1-2)}=\Phi$ , as shown in Table 7,  $\xi_{l2r}=1$ .

③ Thus,  $\xi_{l2}=\xi_{l2t}+\xi_{l2r}=3+1=4$ .

(7) Get the number of DOF:  $F=f_{(1-3)}-\sum_{i=1}^2 \xi_i=12-(5+4)=3$

(8) Get the mobility property:  $M_{pa}=M_{pa(1-3)}=M_{pa(1-2)}\cap M_{L3}=\begin{bmatrix} G_3 \\ \Phi \end{bmatrix}$ .

It shows that this PM has three translations and the DOF is three also. The automatically generated result is shown in Fig.9. and the result is consistent with the result in [18].



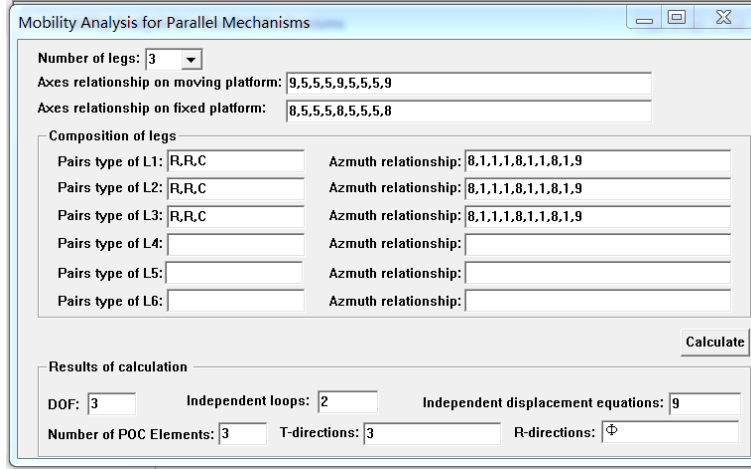


Fig. 9. Mobility analysis of 3-RRC

### 6.2.3 Mobility analysis of Exechon

Fig.6(c) shows an Exechon PM with parasitic motion. It has three identical RPS legs, labeled with the numbers 1~3 respectively. The topological structure is  $PM=(L_1, L_2, L_3, L_{MP}, L_{BP})$ . The axes of the joints on the two platforms are all arbitrary.

(1) Input topology matrices of the PM

$$\text{Topology matrix of RPS leg is } L_i = \begin{bmatrix} 8 & 2 & 1 & 0 & 0 \\ 2 & 9 & 2 & 0 & 0 \\ 1 & 2 & 8 & 0 & 0 \\ 0 & 0 & 0 & 8 & 4 \\ 0 & 0 & 0 & 4 & 8 \end{bmatrix}_{(i)} \quad (i=1\sim 3).$$

where, the subscript  $i$  represents the No. of the intersecting point of the last two revolute joints in the  $i^{\text{th}}$  leg.

$$\text{Topology matrices of two virtual legs on the two platforms are } L_{MP} = \begin{bmatrix} 8 & 0 & 0 \\ 0 & 8 & 0 \\ 0 & 0 & 8 \end{bmatrix} \text{ and } L_{BP} = \begin{bmatrix} 8 & 0 & 0 \\ 0 & 8 & 0 \\ 0 & 0 & 8 \end{bmatrix}, \text{ respectively.}$$

Number of legs is  $3(=v+1)$ .

(2) Select center  $O_{13}$  of  $S_{13}$  as the base point  $O'$ .

(3) Calculate POC matrix of legs  $M_{L1}$ ,  $M_{L2}$  and  $M_{L3}$ .

① Recognize and sequentially extract planar and spherical sub-chains: one  $G_3$  sub-chain  $R \perp P//R$  and one  $S_2$  sub-chain through center  $O_{13}$  ( $i=1\sim 3$ ) respectively.

② Generate the POC supplemented matrices of  $G_3$  and  $S_2$  as the follows,

$$\text{For the first leg: } M_{G3} = \begin{bmatrix} 2 & 0 & 0 & 0 & 0 & 0 \\ 1 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} \text{ and } M_{S2} = \begin{bmatrix} 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 1 & 0 \end{bmatrix} + \begin{bmatrix} 0 \\ \#1 \end{bmatrix}.$$

$$\text{For the second (and third) leg: } M_{G3} = \begin{bmatrix} 2 & 0 & 0 & 0 & 0 & 0 \\ 1 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} \text{ and } M_{S2} = \begin{bmatrix} 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 1 & 0 \end{bmatrix} + \begin{bmatrix} 2 \\ \#i \end{bmatrix} \quad (i=2 \text{ or } 3).$$

Among which, “# $i$ ” indicates the intersecting point of the last two R joints in the  $i^{\text{th}}$  leg, and the number 2 means two derivative translation outputs in the plane perpendicular to the line connecting point  $O'$  and  $O_{13}$ .

③ Carry out “or” operation on POC matrices

$$\text{To the first leg: } M_{L1} = \begin{bmatrix} 2 & 0 & 0 & 0 & 0 & 0 \\ 1 & 0 & 0 & 1 & 1 & 0 \end{bmatrix}_{(1)}, \xi_r^0=3, \xi_t^0=2 \text{ and } \rho_i=O'O_{13};$$

$$\text{To the second and the third legs: } M_{Li} = \begin{bmatrix} 2 & 0 & 0 & 0 & 0 & 0 \\ 1 & 0 & 0 & 1 & 1 & 0 \end{bmatrix} + \begin{bmatrix} 1 \\ \#i \end{bmatrix}, \xi_r^0=3, \xi_t^0=3 \text{ and } \rho_i=O'O_{13} \quad (i=2,3)$$

$$\text{④ Then, } M_{L1} = \begin{bmatrix} 2 & 0 & 0 & 0 & 0 & 0 \\ 3 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} \text{ and } M_{Li} = \begin{bmatrix} 3 & 0 & 0 & 0 & 0 & 0 \\ 3 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}, \quad (i=2,3). \text{ i.e., } \xi_{t1}=2, \xi_{t2}=\xi_{t3}=3, \xi_{r1}=3 \quad (i=1\sim 3),$$

and  $e_1 = \perp (R_{11})$ .

(4) Calculate the sum of all joints in this PM:  $f_{(1-3)}=5+5+5=15$ .

(5) Calculate the POC matrix  $M_{pa(1\sim j)}$  of sub-PM $_{(1\sim j)}$  constituted by the front  $j$  legs in parallel.

- Set  $M_{pa(1\sim 1)}=M_{L1}$
- Calculate the POC matrix  $M_{pa(1\sim 2)}$  of sub-PM $_{(1\sim 2)}$  with  $M_{pa(1\sim 2)}=M_{pa(1\sim 1)}\cap M_{L2}$ .
- ①  $\xi_{t1}=2$ , and  $\xi_{t2}=3$ , as shown in Table 5,  $\xi_{t(1\sim 2)}=2$ , and  $e_{(1\sim 2)}=e_1$ .
- ②  $\xi_{r1}=3$ ,  $\xi_{r2}=3$ , as shown in Table 6,  $\xi_{r(1\sim 2)}=3$ .
- ③ So,  $M_{pa(1\sim 2)} = \begin{bmatrix} G_1 \\ H_1 \end{bmatrix}$ .

(6) Calculate the number of the independent displacement equations  $\xi_{Lj}$  of SLC $_j$  ( $j=1\sim 2$ ).

- Calculate  $\xi_{L1}$  of the first loop SLC $_1$  concatenated by the 1<sup>st</sup> and the 2<sup>nd</sup> leg.
- ①  $\xi_{t1}=2$ , and  $\xi_{t2}=3$ , as shown in Table 7,  $\xi_{L1t}=3$ .
- ②  $\xi_{r1}=3$ ,  $\xi_{r2}=3$ , as shown in Table 8, then  $\xi_{L1r}=3$ .
- ③ So,  $\xi_{L1}=\xi_{L1t}+\xi_{L1r}=3+3=6$ .
- Calculate the number of the independent displacement equations  $\xi_{L2}$  of SLC $_2$  ( $\xi_{L2}=\dim(M_{pa(1\sim 2)} \cup M_{L3})$ )
- ①  $\xi_{t3}=3$ , and  $e_{(1\sim 2)}=G_1$ , as shown in Table 7,  $\xi_{L2t}=3$ .
- ②  $\xi_{r3}=3$ , and  $\xi_{r(1\sim 2)}=3$ , as shown in Table 8,  $\xi_{L2r}=3$ .
- ③ Thus,  $\xi_{L2}=\xi_{L2t}+\xi_{L2r}=3+3=6$ .

(7) Get the number of DOF:  $F=f_{(1\sim 3)} - \sum_{i=1}^2 \xi_i = 15 - (6+6) = 3$

(8) Get the mobility property:  $M_{pa} = M_{pa(1\sim 2)} \cap M_{L3} = \begin{bmatrix} G_1 \\ H_1 \end{bmatrix} \cap \begin{bmatrix} 3 & 0 & 0 & 0 & 0 & 0 \\ 3 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} = \begin{bmatrix} G_1 \\ H_1 \end{bmatrix}$ .

The automatically generated result is shown in Fig.10 and the number of POC elements is 5. Since DOF of this PM is three, the POC has three independent elements and the other two are non-independent. The result is consistent with that in [18], where the axes of rotations outputs are selected as  $r^1_{(O13-O23)} \cup r^1_{(O13-O23)} \cup r^1_{(O13-O23)}$ .

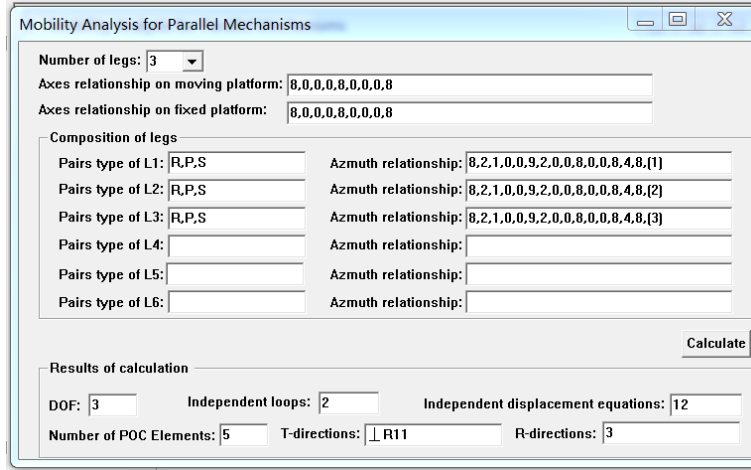


Fig. 10. Mobility analysis of Exechon

#### 6.2.4 Mobility analysis of 4-RPRRR PM

Fig.11(a) shows a kind of 4-RPRRR PM. It has four identical R( $\perp$ P)|R-R\*R legs, labeled with the numbers 1~4 respectively. The topological structure of this PM is  $PM=(L_1, L_2, L_3, L_4, L_{MP}, L_{BP})$ . The axes of the joints on the fixed platforms:  $R_{11} | R_{21}, R_{31} | R_{41}, R_{11} \perp R_{31}$ . Axes of  $R_{14}, R_{15}, R_{44}$  and  $R_{45}$  intersect at point  $O_1'$ , and axes of  $R_{24}, R_{25}, R_{34}$  and  $R_{35}$  intersect at point  $O_2'$ .

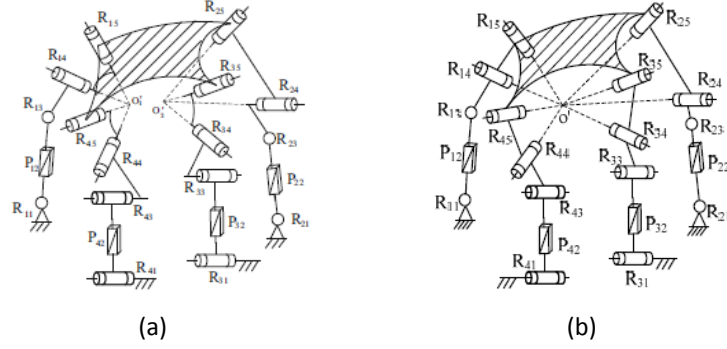


Fig. 10. Two kinds of 4-RPRRR PMs

(1) Input topology matrices of the PM

Topology matrix of the first (and the fourth) leg is,  $L_i = \begin{bmatrix} 8 & 2 & 1 & 0 & 0 \\ 2 & 9 & 2 & 0 & 0 \\ 1 & 2 & 8 & 0 & 0 \\ 0 & 0 & 0 & 8 & 4 \\ 0 & 0 & 0 & 4 & 8 \end{bmatrix}_{(1)}$ , ( $i=1, 4$ ).

Topology matrix of the second (and the third) leg is,  $L_i = \begin{bmatrix} 8 & 2 & 1 & 0 & 0 \\ 2 & 9 & 2 & 0 & 0 \\ 1 & 2 & 8 & 0 & 0 \\ 0 & 0 & 0 & 8 & 4 \\ 0 & 0 & 0 & 4 & 8 \end{bmatrix}_{(2)}$ , ( $i=2, 3$ ).

where, the subscript 1 and 2 represent the No. of intersection point of the last two revolute joints in the legs.

Topology matrices of two virtual legs on the two platforms  $L_{MP}$  and  $L_{BP}$  are  $L_{MP} = \begin{bmatrix} 8 & 0 & 0 & 4 \\ 0 & 8 & 4 & 0 \\ 0 & 4 & 8 & 0 \\ 4 & 0 & 0 & 8 \end{bmatrix}$  and

$L_{BP} = \begin{bmatrix} 8 & 1 & 2 & 2 \\ 1 & 8 & 2 & 2 \\ 2 & 2 & 8 & 1 \\ 2 & 2 & 1 & 8 \end{bmatrix}$ , respectively.

Number of legs  $v+1=4$ .

(2) Select the point  $O_1'$  on the moving platform as the base point ( $O'$ ).

(3) Calculate POC matrix of legs  $M_{L1}$ ,  $M_{L2}$ ,  $M_{L3}$  and  $M_{L4}$ .

① Recognize and sequentially extract planar and spherical sub-chains: one  $G_3$  sub-chain  $R \perp P // R$  and one  $S_2$  sub-chain  $R^*R$  respectively.

② Generate the POC supplemented matrices of  $G_3$  and  $S_2$  as the follows,

For the first (or the fourth) leg:  $M_{G3} = \begin{bmatrix} 2 & 0 & 0 & 0 & 0 & 0 \\ 1 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$  and  $M_{S2} = \begin{bmatrix} 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 1 & 0 \end{bmatrix} + \begin{bmatrix} 0 \\ \#1 \end{bmatrix}$ .

For the second (or the third) leg:  $M_{G3} = \begin{bmatrix} 2 & 0 & 0 & 0 & 0 & 0 \\ 1 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$  and  $M_{S2} = \begin{bmatrix} 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 1 & 0 \end{bmatrix} + \begin{bmatrix} 2 \\ \#2 \end{bmatrix}$ .

Among which, subscript 1 (2) indicates the intersecting points of the last two revolute joints, the number 2 means two derivative translational outputs in the plane perpendicular to the line connecting point  $O'$  and  $O_2'$ .

③ Carry out "or" operation on POC matrices

To the first (and the fourth) leg:  $M_{Li} = \begin{bmatrix} 2 & 0 & 0 & 0 & 0 & 0 \\ 3 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}_{(1)}$ , so  $\xi_r^0=3$ ,  $\xi_t^0=2$  and  $\rho_i=OO_1'$  ( $i=1,4$ ).

To the second (and the third) leg:  $M_{Li} = \begin{bmatrix} 2 & 0 & 0 & 0 & 0 & 0 \\ 3 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} + \begin{bmatrix} 2 \\ \#2 \end{bmatrix}$ , so  $\xi_r^0=3$ ,  $\xi_t^0=3$  and  $\rho_i=O'O_2'$  ( $i=2,3$ ).

④ Then  $M_{Li} = \begin{bmatrix} 2 & 0 & 0 & 0 & 0 & 0 \\ 3 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} + \begin{bmatrix} 0 \\ \#1 \end{bmatrix}$  ( $i=1,4$ ), and  $M_{Li} = \begin{bmatrix} 3 & 0 & 0 & 0 & 0 & 0 \\ 3 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$  ( $i=2,3$ ), i.e.,  $\xi_{ti}=2$ ,  $e_i=\perp(R_{i1})$  ( $i=1,4$ ), and  $\xi_{ti}=3$  ( $i=2,3$ ), and  $\xi_{ri}=3$  ( $i=1\sim4$ ).

(4) Calculate the sum of all joints in this PM:  $f_{(1-3)}=5+5+5=20$ .

(5) Calculate the POC matrix  $M_{pa(1\sim j)}$  of sub-PM $_{(1\sim j)}$  constituted by the front  $j$  legs in parallel.

- Set  $M_{pa(1\sim 1)}=M_{L1}$
- Calculate the POC matrix  $M_{pa(1\sim 2)}$  of sub-PM $_{(1\sim 2)}$  with  $M_{pa(1\sim 2)}=M_{pa(1\sim 1)}\cap M_{L2}$ .
  - ①  $\xi_{t1}=2$ , and  $\xi_{t2}=3$ , as shown in Table 5,  $\xi_{t(1\sim 2)}=2$ , and  $e_{(1\sim 2)}=e_1$ .
  - ②  $\xi_{r1}=3$ ,  $\xi_{r2}=3$ , as shown in Table 6,  $\xi_{r(1\sim 2)}=\xi_{r1}=3$ .
  - ③ So,  $M_{pa(1\sim 2)}=\begin{bmatrix} G_1 \\ H_1 \end{bmatrix}$ .
- Calculate the POC matrix  $M_{pa(1\sim 3)}$  of sub-PM $_{(1\sim 3)}$  with  $M_{pa(1\sim 3)}=M_{pa(1\sim 2)}\cap M_{L3}$ .
  - ①  $\xi_{t(1\sim 2)}=2$ ,  $e_{(1\sim 2)}=e_1$ , and  $\xi_{t3}=3$ , as shown in Table 5,  $\xi_{t(1\sim 3)}=2$ , and  $e_{(1\sim 3)}=e_1$ .
  - ②  $\xi_{r(1\sim 2)}=3$ ,  $\xi_{r3}=3$ , as shown in Table 6,  $\xi_{r(1\sim 3)}=3$ .
  - ③ So,  $M_{pa(1\sim 3)}=\begin{bmatrix} G_1 \\ H_1 \end{bmatrix}$ .

(6) Calculate the number of the independent displacement equations  $\xi_{Lj}$  of SLC $_j$  ( $j=1\sim 2$ ).

- Calculate  $\xi_{L1}$  of the first loop SLC $_1$  concatenated by the 1<sup>st</sup> leg and the 2<sup>nd</sup> leg.
  - ①  $\xi_{t1}=2$ , and  $\xi_{t2}=3$ , as shown in Table 7,  $\xi_{L1t}=3$  ( $i=1,2$ ).
  - ②  $\xi_{r1}=3$ ,  $\xi_{r2}=3$ , as shown in Table 8, then  $\xi_{L1r}=3$  ( $i=1,2$ ).
  - ③ So,  $\xi_{L1}=\xi_{L1t}+\xi_{L1r}=3+3=6$ .
- Calculate the number of the independent displacement equations  $\xi_{L2}$  of SLC $_2$  ( $\xi_{L2}=\dim(M_{pa(1\sim 2)} \cup M_{L3})$ )
  - ①  $\xi_{t3}=3$ , and  $e_{(1\sim 2)}=e_1$ , as shown in Table 7,  $\xi_{L2t}=3$ .
  - ②  $\xi_{r3}=3$ , and  $\xi_{r(1\sim 2)}=3$ , as shown in Table 8,  $\xi_{L2r}=3$ .
  - ③ Thus,  $\xi_{L2}=\xi_{L2t}+\xi_{L2r}=3+3=6$ .
- Calculate the number of the independent displacement equations  $\xi_{L3}$  of SLC $_3$  ( $\xi_{L3}=\dim(M_{pa(1\sim 3)} \cup M_{L4})$ )
  - ①  $\xi_{t4}=2$ ,  $e_{(1\sim 3)}=e_1$ , and  $e_4(\perp R_{41})$ , as shown in Table 7,  $\xi_{L3t}=3$ .
  - ②  $\xi_{r4}=3$ , and  $\xi_{r(1\sim 3)}=3$ , as shown in Table 8,  $\xi_{L3r}=3$ .
  - ③ Thus,  $\xi_{L3}=\xi_{L3t}+\xi_{L3r}=3+3=6$ .

(7) Get the number of DOF:  $F=f_{(1\sim 4)}-\sum_{i=1}^3 \xi_i=20-(6+6+6)=2$

(8) Get the mobility property:  $M_{pa}=M_{pa(1\sim 3)}\cap M_{L4}=\begin{bmatrix} G_1 \\ H_1 \end{bmatrix} \cap \begin{bmatrix} G_4 \\ H_4 \end{bmatrix}=\begin{bmatrix} (\perp R_{11})I (\perp R_{41}) \\ H_1 \end{bmatrix}$ .

The automatically generated result is shown in Fig.11 and the result is consistent with the result in [18]. Since DOF of this PM is 2, the POC has two independent elements, and the other two elements are non-independent. Without loss of generality, one translation along the common perpendicular of axes of  $R_{11}$  and  $R_{41}$ , and one rotation around  $O_1'-O_2'$  are selected as two independent elements, and the POC is written as  $M_{pa}=\begin{bmatrix} t_{(\perp R_{11}, R_{41})}^1 \\ r_{(o_1-o_2)}^1 \end{bmatrix} \cup \{r^2\}$  in [18].

(9) Discussion

If the intersecting points of  $O_1'$  and  $O_2'$  coincide together, a new PM is obtained in Fig.10(b). Select the intersecting point as the base point, and follow above steps (1) to (8). It is easy to determine that the POC is  $M_{pa}=\begin{bmatrix} G_1 I G_4 \\ H_1 \end{bmatrix}$  and DOF is four.

Although this POC has the same form as that of the original PM, it should be noted that since DOF is four, the POC has four independent elements. So, the PM has three independent rotations and one independent translation along the common perpendicular of axis of  $R_{11}$  and axis of  $R_{41}$ .

Fig. 11. Mobility analysis of the 4-RPRRR PM shown in Fig.10(a)

## 7. Conclusions

In this paper, a computer-aided method for mobility analysis of PMs is proposed. A matrix representation mapping the topological structure of a leg or a PM into their POC is established. This matrix includes not only the dimension of translation/rotation output, but also indicates the orientation of the output motion. By extracting planar and spherical sub-chains, calculating the POC for legs transforms into logical bitwise "or" operation of matrices. Algorithmic rules for computing the POCs of PMs are established without manual intervention. Based on the proposed method, the process of mobility analysis is performed by a computer program, and a human-computer interactive interface has been developed for potential designers. four typical examples are provided in detail to show the effectiveness of the developed algorithms. The relevant study of this paper is expected to benefit the automation and computerization of the mechanical creative design and analysis.

For the "paradoxical" mechanisms that the software mentioned here currently can not solve, the future work will concentrate on proposing the digital model of the dimensional constraint parameters (link length, axial distance between two joint axes, etc.) of the PMs, and then focus on establishing the mapping relationship between the model and the POC. Further, the automatic analysis for another important topological properties of PMs such as coupling degree will be presented.

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