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Jean-Claude Carret, Hervé Di Benedetto, Cédric Sauzeat. Multi Modal Dynamic Linear Viscoelastic Back Analysis for Asphalt Mixes. Journal of Nondestructive Evaluation, 2018, 37 (2), 10.1007/s10921-018-0491-3 . hal-02457993

HAL Id: hal-02457993 https://hal.science/hal-02457993

Submitted on 27 Oct 2020

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Multi Modal Dynamic Linear Viscoelastic Back Analysis for Asphalt Mixes

Jean-Claude Carret¹ · Herve Di Benedetto¹ · Cedric Sauzeat¹

Abstract

In this paper, an investigation was performed to determine the accuracy of a simplified viscoelastic back analysis to interpret dynamic loading tests on asphalt mixes (AM). First, quasi-static cyclic tension–compression lab tests were performed on different AM to fit the 3 dimensional 2S2P1D linear viscoelastic (LVE) model. Considering these tests on very different types of AM, a LVE material with "averaged" viscoelastic properties was obtained. Then, these "averaged" viscoelastic properties were considered to perform finite elements method numerical simulations of dynamic loading tests on a cylinder. The simulations were performed at ten different temperatures from -40 to 50 °C. The longitudinal, flexural and torsional modes of vibration are studied. The complex Young's modulus and complex Poisson's ratio were first obtained using the viscoelastic back analysis, which has the advantage of simplicity, was used to determine the elastic equivalent properties and the phase angle of the material. The results obtained directly with the 2S2P1D model and the results from the combined viscoelastic back analysis results regarding both the Young's modulus and the Poisson's ratio are discussed in the paper.

Keywords Asphalt mixes · Viscoelasticity · Dynamic loading · Finite element calculation · Back analysis

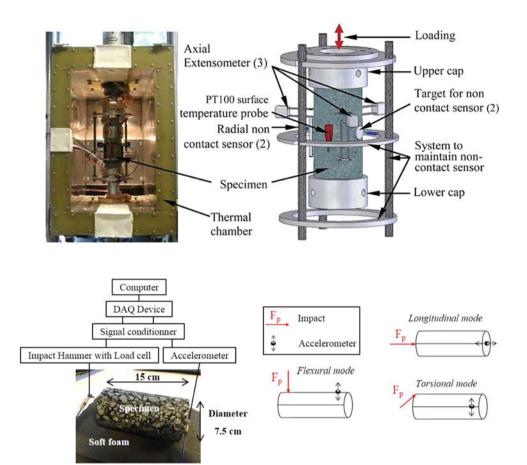
1 Introduction

Seismic measurements such as impact loadings are economical, simple to perform and are nondestructive tests. These tests seem to be a good approach for providing accurate characterization of materials properties [1]. Measurement of the flying time in wave propagation tests [2–4] and measurement of the fundamental resonance frequencies through resonance testing [5,6] have been used to determine the complex modulus of asphalt mixtures. These methods give information on the tested LVE material only for a limited number of resonance frequencies (1–3), which are, a priori, unknown. The analysis of the results of these tests is based on simplified approximate formulations used to determine a complex mod-

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¹ LTDS (UMR CNRS 5513), University of Lyon / ENTPE, Rue M. Audin, 69518 Vaulx en Velin, France ulus for each considered temperature and mode of vibration [7,8]. Ryden and Gudmarsson [9,10] applied resonant acoustic spectroscopy (RAS) to calculate resonance frequencies of cylindrical discs and beams for different modes of vibration. These tests require simplified method such as the half-power bandwidth method to evaluate the damping of the specimen. In this paper, a comparison between the direct viscoelastic analysis and a simplified back analysis of a specimen under impact loadings were performed. The differences between the two approaches and the limitations of the simplified back analysis were highlighted. LVE finite element method numerical simulations of impact loadings on a cylindrical specimen were carried out. These numerical tests were performed at 10 different temperatures between -40 and $50 \,^{\circ}\text{C}$ for three different modes of vibration: the longitudinal mode, the flexural mode and the torsional mode. The analysis of these computations were limited to the first resonance frequency for each mode. Viscoelastic properties of the specimen were estimated at the corresponding resonance frequency for each mode of vibration and for each temperature using the 2S2P1D linear viscoelastic model. Then, a combined back analysis-it is the type of method mostly used to analyze impact testing on asphalt materials-was used to evaluate the elastic equivaFig. 1 (left) Tension–compression test apparatus used for tests on asphalt mixes; (right) detailed scheme of measurement devices and sample

Fig. 2 (left) Dynamic impact tests set-up (ENTPE/LTDS laboratory), case of the longitudinal mode; (right) Position of the accelerometer and of the impact for the three considered modes of vibration



lent properties and the phase angle of the material. Finally, results obtained directly with the 2S2P1D model and from the combined back analysis were compared.

2 Materials and Experimental Tests

2.1 Materials and Considered Material for Modelling

38 specimens are first considered in this study. They were previously tested (cyclic tension-compression test—see following section-) during 4 different Ph.D. theses performed at the University of Lyon / ENTPE, LTDS laboratory [11–17]. The wide variety of these specimens represents a good range of asphalt mixes types and makes it possible to consider a material with averaged viscoelastic properties. This material was given the average values of the constants of the 2S2P1D LVE model obtained for the 38 specimens. All the numerical simulations and analyses that are presented in this paper were performed on this average material.

2.2 Cyclic Tension–Compression Tests

The tension–compression tests were used to determine the complex Young's modulus and the complex Poisson's ratio of asphalt mixes. The tests were performed on cylindrical sam-

ples (150 mm high with a 75 mm diameter). Cyclic sinusoidal loadings were applied using a hydraulic press, as shown in Fig. 1, in strain-controlled mode with an amplitude of around $5 * 10^{-5}$ m/m.

The axial stress $(\sigma_1 = \sigma_0.sin(\omega t - \phi_E))$ was measured with a load cell with a ± 25 kN maximum load and a 25N accuracy. The axial strain ($\varepsilon_1 = \varepsilon_{01}.sin(\omega t)$) was obtained by means of three extensioneters placed at 120° from each other (Fig. 1). The radial strain ($\varepsilon_2 = \varepsilon_{02}.sin(\omega t + \phi_\nu)$) was deduced from measurements of two non-contact transducers (Fig. 1). The complex modulus and the complex Poisson's ratio at different loading frequencies (from 0.01 to 10 Hz) and different temperatures (from - 30 to 50 °C) are calculated according to Eqs. 1 and 2 where ϕ_E and ϕ_ν are the phases of the complex modulus and of the complex Poisson's ratio respectively.

$$E^*(\omega) = \frac{\sigma_1^*}{\varepsilon_1^*} = \left| E^*(\omega) \right| e^{i\Phi_E} \tag{1}$$

$$\nu^*(\omega) = -\frac{\varepsilon_2^*}{\varepsilon_1^*} = \left|\nu^*(\omega)\right| e^{i\Phi_\nu} \tag{2}$$

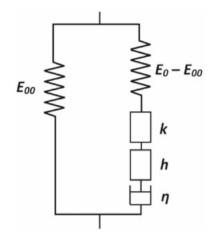


Fig. 3 Representation of the 2S2P1D LVE model in the 1-dimension case

2.3 Dynamic Impact Tests and Conditions for the Numerical Simulations

The sample geometry used for tension-compression test (7.5 cm diameter and 15 cm height) was considered for dynamic test. In this paper only numerical tests are presented but the associated lab test set-up of the dynamic impact tests is shown on the left of Fig. 2. The loading is manually applied on the specimen with an impact hammer equipped with a load cell (PCB model 086E80). The standing wave modes are measured with an accelerometer (PCB model 353B15) glued to the specimen. During the test, the specimen lays on soft foam to ensure free boundary conditions [6]. The impact hammer and the accelerometer are connected to the data acquisition system that consists of a signal conditioner (PCB model 482C15), a data acquisition device (NI DAQ USB-6251) and a computer (see Fig. 2). It is possible to excite different modes of vibration depending on the position of the impact and of the accelerometer [18,19]. The configurations for the three different modes of vibration considered in this study are described on the right of Fig. 2. These configurations were used for the FEM numerical simulations (see Sect. 3.2).

3 Linear Viscoelastic Analysis

3.1 2S2P1D LVE Isotropic Model

The tension–compression tests results were analyzed using the 2S2P1D LVE model developed at ENTPE [20–22]. It consists of a combination of two springs, two parabolic creep elements (also called fractional-derivative model [23]) and one dashpot as schematized in the 1-dimension case in Fig. 3. The introduced model has seven parameters and the complex modulus of the 2S2P1D model is expressed according to Eq. 3,

$$E_{2S2P1D}^{*}(\omega) = E_{00} + \frac{E_0 - E_{00}}{1 + \delta(j\omega\tau)^{-k} + (j\omega\tau)^{-h} + (j\omega\beta\tau)^{-1}}$$
(3)

where *j* is complex number defined by $j^2 = -1$, ω is the angular frequency ($\omega = 2\pi f$, where *f* is the frequency), E_{00} is the static modulus when $\omega \rightarrow 0$, E_0 is the glassy modulus when $\omega \rightarrow +\infty$, δ is a dimensionless constant, *k* and *h* are dimensionless exponents such as 0 < k < h < 1, β is a dimensionless constant related to the Newtonian viscosity η by $\eta = (E_0 - E_{00})\beta\tau$ and τ is the characteristic time, function of temperature. The time-temperature superposition principle (TTSP) is verified for asphalt mixes in the linear and nonlinear domains [16,24,25] using:

$$\tau(T) = a_T(T)\tau_{ref} \tag{4}$$

where a_T is the shift factor at the temperature T defined by the Williams–Landel–Ferry (WLF) equation [26]:

$$\log(a_T) = -\frac{C_1(T - T_{ref})}{C_2 + T - T_{ref}}$$
(5)

where C_1 and C_2 are the two constants of the WLF equation and T_{ref} is the reference temperature. The WLF parameters used in this study are listed in Table 1 and the corresponding shift factors are plotted in Fig. 4. Di Benedetto et al. [27] extended the 2S2P1D model to characterize the complex Poisson's ratio as given in Eq. 6,

$$\nu_{2S2P1D}^{*}(\omega) = \nu_{00} + \frac{\nu_{0} - \nu_{00}}{1 + \delta(j\omega\tau_{\nu})^{-k} + (j\omega\tau_{\nu})^{-h} + (j\omega\beta\tau_{\nu})^{-1}}$$
(6)

where ν_{00} is the low frequency Poisson's ratio when $\omega \rightarrow 0$ and ν_0 is the high frequency Poisson's ratio when $\omega \rightarrow +\infty$. The same values of the parameters δ , k, h and β are used to determine both the complex modulus and complex Poisson's ratio while τ is determined also for the Poisson's ratio and is therefore labeled τ_{ν} . The constants of the 2S2P1D LVE model for the material used in the numerical simulations (see Sect. 2.1) are given in Table 1. The complex modulus (E^{*}) and the complex Poisson ratio (ν^*) of the average material are plotted in Fig. 5 (master curves) and Fig. 6 (normalized cole–cole curves).

3.2 Finite Element Method Simulations

Finite element method is widely used on pavement for different applications [28–32]. In this study, the behavior of

Table 12S2P1D constants at the reference temperature of 10 $^{\circ}$ C for the material with "averaged" viscoelastic properties of asphalt mixes used for
the numerical simulations

$E_{00}(MPa)$	$E_0(MPa)$	ν_0	ν_{00}	δ	k	h	β	$\tau_{10^\circ C}(s)$	$\tau_{v}(s)$	C_1	C_2
105	35,000	0.19	0.42	2.15	0.17	0.525	505	0.1	3.165	30	210

1,E+11

 WLF parameters
 1,E+09

 1,E+07
 1,E+05

 1,E+03
 1,E+01

 1,E-01
 1,E-03

 1,E-05
 1,E-05

 Fig. 5 Master curves at 10 °C
 100000

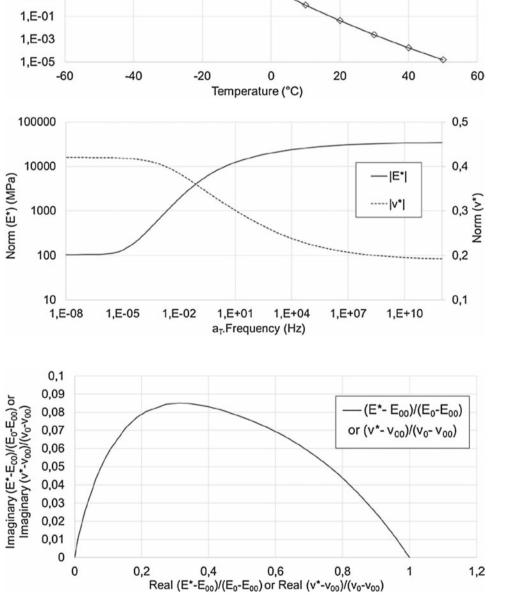
 for the norm of the complex
 100000

 Young's modulus (solid line)
 100000

 and for the norm of the complex
 100000

Fig. 4 Shift factor versus

temperature for the considered



T_{ref} = 10°C

 $C_1 = 30$

 $C_2 = 210$

WLF Equation:

 $\log(a_{\tau})$

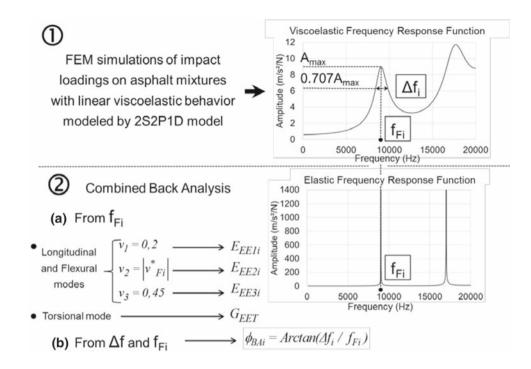
 $\frac{C_1(T-T_{ref})}{C_2+T-T_{ref}}$

Fig. 6 Cole–Cole plot for the normalized complex Young's modulus and for the normalized complex Poisson's ratio for the material used for the numerical simulations

material used for the numerical

simulations

Fig. 7 FEM calculation and principle of the combined back analysis



the material under dynamic loadings was calculated with FEM numerical simulations. The 2S2P1D model complex modulus and complex Poisson's ratio (Table 1) were used as input for the material properties and the material was considered as isotropic. The Comsol software was used to solve the three-dimensional equation of motion in the frequency domain (Eq. 7):

$$-\rho\omega^2 \mathbf{u} - \nabla \boldsymbol{.}\boldsymbol{\sigma} = \mathbf{F}_{\mathbf{p}} e^{i\boldsymbol{\Phi}} \tag{7}$$

where ω is the angular frequency, ρ is the density, **u** is the displacement vector, ∇ is the gradient tensor operator, *i* is complex number defined by $i^2 = -1$ and σ is the Cauchy stress tensor. **F**_p is the unity load and ϕ is the phase of the cyclic load.

The unity load has been applied to the corresponding point of the hammer impact while the response has been determined at the position of the accelerometer for the three considered modes of vibration (see Fig. 2). The simulations were performed at 10 different temperatures every 10 °C from -40 to 50 °C, in a frequency range from 100 to 20,000 Hz considering steps of 20 Hz. Finer analysis with steps of 1 Hz were also carried out around the first resonant frequency for each temperature and for each mode of vibration in order to obtain a better evaluation of the first resonance frequency and of the damping of the material. The geometry used in this study was a cylinder with a 75 mm diameter and a 150 mm height. The mesh consists of tetrahedral elements with a maximum element size of 2.5 cm that was determined through a convergence study.

4 Combined Back Analysis

Back analysis methods are used for different applications on pavement [33,34]. The proposed combined back analysis is based on the analysis of the first resonance peak for each temperature and for each mode of vibration. Figure 7 explains the principle of the back analysis for one temperature. In the first step (see Fig. 7), the first resonance frequency of each mode of vibration f_{Fi} (i = L for the longitudinal mode, i = Ffor the flexural mode and i = T for the torsional mode) was deduced from the FEM simulations. The frequency bandwidth Δf_i was also determined for each mode of vibration from the FEM simulations using the half-power bandwidth method (see Fig. 3). For the longitudinal and flexural modes of vibration, the frequencies f_{Fi} were used in the 2S2P1D model to calculate the corresponding complex modulus E_{Fi}^* , and Poisson's ratio, ν_{Fi}^* , according to Eqs. 8 and 9:

$$E_{Fi}^* = E_{2S2P1D}^* (2\pi f_{Fi}) \tag{8}$$

$$\nu_{Fi}^* = \nu_{2S2P1D}^* (2\pi f_{Fi}) \tag{9}$$

For the torsional mode of vibration, the frequency f_{FT} was used in the 2S2P1D model to calculate the corresponding complex shear modulus G_{FT}^* according to Eq. 10:

$$G_{FT}^* = G_{2S2P1D}^*(2\pi f_{FT}) \tag{10}$$

The phase angle used in the back analysis ϕ_{BAi} was deduced using the following relationship suggested by Clough and Penzien [35]:

$$\phi_{BAi} = Arctan\left(\frac{\Delta f_i}{f_{Fi}}\right) \tag{11}$$

In the second step of the combined back analysis (see Fig. 7), 2 different approaches were used. For the longitudinal and flexural modes of vibration, three cases corresponding to three different real values of Poisson's ratio of 0.2, $|v_{Fi}^*|$, and 0.45 were studied. Three elastic equivalent moduli E_{EE1i} , E_{EE2i} and E_{EE3i} associated to the three cases were backcalculated through elastic FEM numerical simulations for these two modes of vibration. For the torsional mode of vibration, the shear modulus is independent of the Poisson's ratio value. Therefore, only one elastic equivalent shear modulus G_{EET} was back-calculated through elastic FEM numerical simulations. Then ϕ_{BAi} was considered as the corresponding phase angle for the three considered modes of vibration. Three complex moduli E^*_{BA1i} , E^*_{BA2i} and E^*_{BA3i} were determined according to Eqs. 12-14 for the longitudinal and flexural mode and one complex shear modulus G_{RAT}^* was determined according to Eq. 15.

$$E_{BA1i}^* = E_{EE1i} e^{i\phi_{BAi}} \tag{12}$$

$$E_{BA2i}^* = E_{EE2i} e^{i\phi_{BAi}} \tag{13}$$

$$E_{BA3i}^* = E_{EE3i} e^{i\phi_{BAi}} \tag{14}$$

$$G_{BAT}^* = G_{EET} e^{i\phi_{BAT}} \tag{15}$$

5 Evaluation of the Proposed Combined Back Analysis

5.1 Principle of the Comparison

The procedure described in the previous section was used to determine the values of f_{Fi} , E_{Fi}^* (norm and phase angle ϕ_{Fi}), $|\nu_{Fi}^*|$, ϕ_{BAi} , E_{EE1i} , E_{E2i} and E_{EE3i} for the longitudinal and flexural modes of vibration and the values of f_{FT} , G_{FT}^* (norm and phase angle ϕ_{FT}), ϕ_{BAT} and G_{EET} . All these values were calculated for the 10 considered temperatures. Table 2 details the name of the variables evaluated in the back

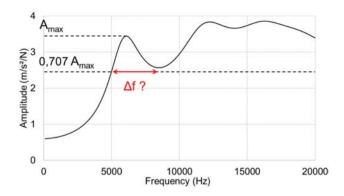


Fig. 8 Example of the half-power bandwidth method limitations at high temperatures where it was not possible to obtain Δf (numerical simulation for the longitudinal mode at 40 °C)

analysis in the different cases studied for clarity. Results are given in Tables 3, 4 and 5 in appendix.

For temperatures higher than 30 $^{\circ}$ C, it was not possible to estimate the phase angle of the material for the longitudinal and the torsional modes of vibration. This is due to too much damping in the material for such temperatures that made the estimation of the frequency bandwidth impossible with the half-power bandwidth method (see Fig. 8).

In order to evaluate the accuracy of the proposed combined back analysis, the different values obtained were compared. For the longitudinal and flexural modes of vibration, the norm and the phase angle of the 2S2P1D model complex modulus E_{Fi}^* obtained at the frequency f_{Fi} (Eq. 8) were compared to the norm and to the phase angle of the three complex moduli E_{BA1i}^* , E_{BA2i}^* and E_{BA3i}^* (Eqs. 12–14) obtained from the simplified back analysis. The three studied cases allow to consider the effect of the Poisson's ratio value. For the torsional mode of vibration, the norm and the phase angle of the 2S2P1D model complex shear modulus G_{FT}^* obtained at the frequency f_{FT} (Eq. 10) were compared to the norm and to the phase angle of the complex shear modulus G_{BAT}^* (Eq. 15) obtained from the simplified back analysis.

Mode of vibration	Case studied	Elastic equivalent modulus	Phase angle	Complex modulus
Longitudinal	$v_1 = 0.2$	E _{EE1L}	ϕ_{BAL}	E^*_{BA1L}
	$\nu_2 = \nu_{2S2P1D}^*(f_L) $	E _{EE2L}		E_{BA2L}^*
	$v_3 = 0.45$	E _{EE3L}		E_{BA3L}^*
Flexural	$v_1 = 0.2$	E _{EE1F}	$\varphi_{\rm BAF}$	E_{BA1F}^{*}
	$\nu_2 = \nu^*_{2S2P1D}(f_F) $	E _{EE2F}		E_{BA2F}^{*}
	$v_3 = 0.45$	E _{EE3F}		E_{BA3F}^*
Torsional	Independent of v	G _{ET}	φ_{BAT}	G_{BAT}^{*}

 Table 2
 Summary of the variables evaluated in the different cases studied for the back analysis

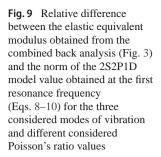
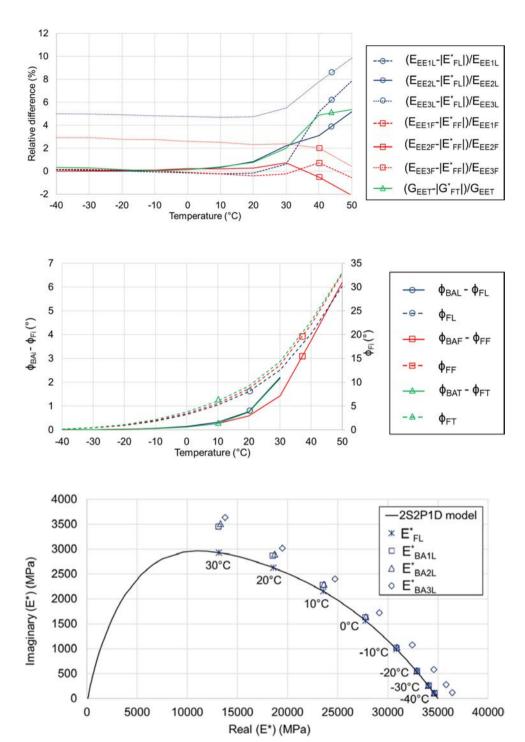


Fig. 10 2S2P1D model phase angle obtained at the first resonance frequency (Eqs. 8–10) (dot lines) and difference with the phase angle obtained from the combined back analysis (Fig. 3) (solid lines) for the three considered modes of vibration

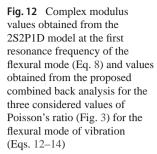
Fig. 11 Complex modulus values obtained from the 2S2P1D model at the first resonance frequency of the longitudinal mode (Eq. 8) and values obtained from the proposed combined back analysis for the three considered values of Poisson's ratio (Fig. 3) for the longitudinal mode of vibration (Eqs. 12–14)

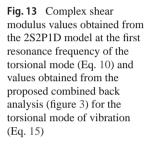


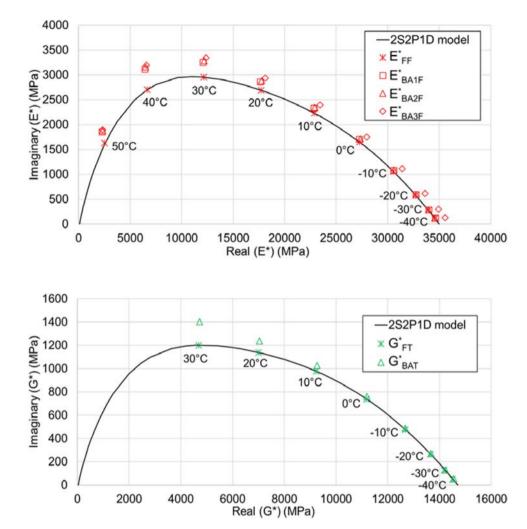
5.2 Results and Discussion

Figure 9 presents the relative difference between the elastic equivalent moduli E_{EE1i} , E_{EE2i} and E_{EE3i} and the norm of the complex modulus $|E_{Fi}^*|$ calculated using the 2S2P1D model at the resonance frequencies f_{Fi} . Lines with a circle (in blue) are for the longitudinal mode of vibration while lines with a square (in red) are for the flexural mode of vibration.

Also on Fig. 6, the relative difference between the elastic equivalent shear modulus G_{EET} and the norm of the complex shear modulus $|G_{FT}^*|$ calculated using 2S2P1D model at the frequency f_{FT} is plotted (green line with a triangle). The solid lines on Fig. 6 clearly indicate that there is a good correlation between the elastic equivalent modulus E_{EE2i} and the norm of the complex modulus from the 2S2P1D model $|E_{Fi}^*|$ and also between the elastic equivalent shear modu-







lus and the norm of the complex shear modulus from the 2S2P1D model $|G_{FT}^*|$. This is particularly true for temperatures below 30 °C with a maximum relative difference of less than 2.5%. At higher temperatures, the error increases for the three modes of vibration but it remains less than 6%. It can also be seen on Fig. 6 that the Poisson's ratio value has an impact on the calculation of the elastic equivalent moduli. The relative difference is clearly higher for E_{EE3i} (around 3% for the flexural mode and around 5% for the longitudinal mode) that is calculated for a Poisson's ratio of 0.45, a value away from the two other studied cases. Nevertheless, the impact of the Poisson's ratio value remains limited.

Figure 10 presents the phase angle ϕ_{Fi} given by the 2S2P1D model at the frequency f_{Fi} (dot lines) and also the difference in degree (°) with the combined back analysis phase angle ϕ_{BAi} for the three considered modes of vibration. Figure 4 shows a good agreement between the back analysis phase angle ϕ_{BAi} and the 2S2P1D model phase angle ϕ_{Fi} with a maximum difference of around 2° at 30 °C for the longitudinal and torsional modes. For the flexural mode, it can be seen that the difference significantly increases for

temperatures above 30 °C to reach 6° at 50 °C. This proves the inaccuracy of the half-power bandwidth method at high temperature: when there is too much damping in the material, the damping is overestimated and so is the phase angle.

For the complex modulus or complex shear modulus as well as for the phase angle, differences between values obtained from the 2S2P1D model at the first resonance frequency and values obtained with the combined back analysis increase with the temperature but remain limited. Figures 11, 12 and 13 present the different complex moduli in a Cole-Cole diagram for the three considered modes of vibration and the three different values of Poisson's ratio. On Figs. 7 and 8 corresponding to the longitudinal mode and to the flexural mode respectively, it can be seen that the values of E_{BA1i}^* or E^*_{BA2i} and E^*_{Fi} are rather close, especially at low temperatures. E^*_{BA3i} does not fit so well with E^*_{Fi} . It is due to the effect of the Poisson's ratio value as already noted on Fig. 6. Figure 9 corresponding to the torsional mode shows a good agreement between G_{BAT}^* and G_{FT}^* , especially for the low temperatures. All the observations from Figs. 11, 12 and 13 confirm the conclusions already raised from Figs. 9 and 10.

6 Conclusion

The proposed simplified back analysis proposed in this paper gives good results, both for the norm of the complex modulus and for the phase angle at low and intermediate temperatures. Slight differences appear for temperatures higher than 20 °C. They are mostly due to an overestimation of the phase angle. Therefore, the half-power bandwidth method used to determine the phase angle is not adapted for high temperatures because it overestimates the damping of the materials. The influence of the Poisson's ratio value was also studied. It was found that it has a limited impact on the calculation of the elastic equivalent modulus (less than 5%). The observations and findings from this study demonstrate that for temperatures below 20 °C, a simple assumption on constant real value for Poisson's ratio and the proposed simplified combined back analysis are enough to analyse accurately impact loadings in order to obtain the behaviour of asphalt mixes.. This is very interesting to obtain quickly and accurately the LVE properties at high frequencies. However, the limitation with the temperature gives only access to a small part of the master curve that can be estimated with this method. In addition, for higher temperatures corresponding to values of the phase angle higher than about 15°, the method does not provide a good evaluation of the phase angle. Therefore, for temperatures higher than 30 °C corresponding to phase angle of above 15° the half-power bandwidth could not be used because of too much damping in the material. These numerical results need to be checked and validated in further studies using laboratory measurements.

Appendix

 Table 3
 Results of the numerical simulations and the combined back analysis (longitudinal mode)

T(°C)	2S2P1D model			Combined back analysis						
	$ E_{FL}^* (MPa)$	ϕ_{FL} (°)	$ v_{FL}^* $	$f_{FL}(Hz)$	ϕ_{BAL} (°)	$E_{EE1L} (MPa)$	$E_{EE2L} (MPa)$	$E_{EE3L} (MPa)$		
-40	34,602	0.178	0.191	12,320	0.186	34,645	34,605	34,425		
-30	34,029	0.434	0.194	12,216	0.450	34,060	34,030	35,810		
-20	32,890	0.943	0.198	12,006	0.969	32,900	32,890	34,585		
-10	30,891	1.837	0.206	11,631	1.901	30,870	30,910	32,460		
0	27,813	3.232	0.219	11,031	3.372	27,775	27,850	29,200		
10	23,675	5.219	0.238	10,173	5.542	23,620	23,755	24,840		
20	18,770	8.049	0.264	9062	8.813	18,740	18,930	19,705		
30	13,475	12.575	0.294	7708	14.782	13,560	13,780	14,260		
40	8179	20.188	0.327	6080	×	8440	8625	8870		
50	3746	30.621	0.362	4160	×	3950	4065	4155		

 Table 4
 Results of the numerical simulations and the combined back analysis (flexural mode)

T(°C)	2S2P1D model			Combined back analysis						
	$ E_{FF}^* $ (MPa)	ϕ_{FF} (°)	$ v_{FF}^* $	$f_{FF}(Hz)$	ϕ_{BAF} (°)	E_{EE1F} (MPa)	E_{EE2F} (MPa)	E_{EE3F} (MPa)		
-40	34,569	0.193	0.192	7619	0.203	34,625	34,575	35,580		
-30	33,948	0.47	0.194	7549	0.486	34,000	33,960	34,940		
-20	32,720	1.019	0.199	7410	1.044	32,750	32,750	33,630		
-10	30,580	1.977	0.207	7161	2.031	30,595	30,610	31,420		
0	27,317	3.461	0.221	6764	3.587	27,290	27,380	28,030		
10	22,991	5.571	0.242	6201	5.838	22,935	23,040	23,570		
20	17,933	8.63	0.268	5472	9.218	17,860	17,980	18,350		
30	12,518	13.671	0.3	4573	15.101	12,490	12,610	12,815		
40	7177	22.12	0.334	3460	25.880	7140	7230	7325		
50	3002	32.832	0.37	2220	39.036	2940	2985	3015		

Table 5Results of thenumerical simulations and thecombined back analysis(torsional mode)

<i>T</i> (°C)	2S2P1D model		Combined back analysis					
	$ G_{FT}^* $ (MPa)	ϕ_{FT} (°)	f_{FT} (Hz)	ϕ_{BAT} (°)	$G_{EET} (MPa)$			
-40	14,502	0.192	8033	0.214	14,550			
-30	14,184	0.514	7954	0.526	14,225			
-20	13,653	1.107	7795	1.139	13,670			
-10	12,681	2.152	7512	2.203	12,685			
0	11,205	3.769	7065	3.887	11,220			
10	9284	6.051	6437	6.321	9320			
20	7090	9.227	5639	9.967	7145			
30	4834	14.371	4684	16.540	4930			
40	2711	22.839	3560	×	2850			
50	1107	33.091	2280	×	1170			

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