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Optimal pump excitation frequency for improvement of damage detection by nonlinear vibro acoustic modulation method in a multiple scattering sample

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Abstract

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We present a method to systematically optimize nonlinear damage detection in multiple scattering media by the nonlinear Vibro-Acoustic Modulation (VAM) technique. The latter consists here of exciting a medium simultaneously with a high frequency ultrasonic sinusoidal burst and with a low frequency continuous sinusoidal wave. Modulation of the high frequency (probe) by the low frequency (pump) is made possible by the presence of nonlinear scatterers, *i.e.* cracks, defects. A signal processing technique consisting of a closed loop system drives the automatic adaptation of the pumping frequency, yielding to the optimization of the nonlinear modulation (NM) of the output probing coda signal without *a priori* information on the medium and the scatterers. The correlation coefficient between a reference output probe signal without the pumping wave and an output modulated probe signal with a pumping wave was considered as our cost function. A multiple scattering solid beam where nonlinear scatterers can be controllably added or removed is designed and tested. The first step of this study is an empirical search of the correlation coefficient dependency on the pumping frequency to verify the performances of the proposed method. Then the implemented optimization algorithm based on genetic algorithm (GA) is used to find automatically the optimal pumping frequency. The obtained optimization results show a good agreement with the empirical study. Moreover, the genetic algorithm allowed to find the optimal pump frequency adapted to each configuration of nonlinear scatterers. This relatively fast search of the optimal nonlinear response could be extended to nonlinear scatterer imaging applications using the information on the resonant modes spatial shapes together with the associated optimal response.

Keywords: Optimal command, Genetic algorithm, Nonlinear Vibro-Acoustic Modulation, Crack detection.

1 **1. Introduction**

2 Nonlinear ultrasonic nondestructive testing has known a great develop-
3 ment during the last decades. Numerous studies have shown the possibility
4 to detect damages at earlier stages than with linear ultrasonic methods.
5 The integrity of the structure can be easily preserved by an earlier mainte-
6 nance. The presence of a nonlinear defect, such as a crack or a delamination,
7 can give rise to nonlinear phenomena, among which are the generation of
8 higher harmonics, subharmonics, wave modulation or resonance frequency
9 shift. **Several methods based on the detection of these phenomena have been**
10 **proposed [1, 2, 3, 4, 5, 6, 7].** One of the most widely studied methods is the
11 Vibro-Acoustic Modulation (VAM) [8, 9] or the Nonlinear Wave Modulation
12 Spectroscopy (NWMS) [10], which belongs to the class of nonlinear modu-
13 lation or nonlinear mixing methods. With the VAM, micro-damage can be
14 detected by following the amplitude modulation induced on a probe signal
15 (*e.g.* high frequency ultrasound) by a pump signal (*e.g.* low frequency vibra-
16 tion). Nonlinear modulation techniques have been widely used on samples
17 with simple geometries in which coherent waves propagate [11, 12]. How-
18 ever, the use of these methods has been poorly studied in environments with
19 complex geometries, which lead to multiple scattering of the waves. Among
20 the few reported results in such case, Zhang et al. have demonstrated that
21 a global inspection of multiple scattering materials is possible by combining
22 the Coda Wave Interferometry (CWI) technique with a nonlinear modulation

23 method [13, 14].

24 Generally, the pump frequency is chosen to correspond to one of the vibra-
25 tion modes of the studied sample in order to amplify the vibration response
26 [9, 15, 16, 17, 18]. A preliminary modal analysis is, then, necessary for iden-
27 tifying and selecting the resonance frequencies of the sample. It is important
28 to underline that such a procedure requires a further experimental setup and
29 time. Furthermore, a possible drawback of a modal excitation is that the
30 crack or the damaged zone can be located at a strain node of the pump or
31 probe wave, which compromises the generation of the nonlinear modulation.
32 To overcome this problem, a frequency swept pump and/or probe signals
33 have been used [13, 17, 19, 20, 21, 18, 22]. As such, several resonance modes
34 are excited and an averaged effect over a wide frequency range is detected,
35 without low sensitivity zones. Dunn *et al.* suggest that selecting the res-
36 onant frequencies as a pumping frequency is not ideal, because the system
37 nonlinearities are maximum at those frequencies and the resonant peaks shift
38 in frequency with changing amplitude [23].

39 In order to optimize and to refine the sensitivity of the nonlinear mod-
40 ulation method to detect damage, various signal processing techniques have
41 been adopted. These methods make it possible to extract the nonlinear
42 modulation information in the frequency domain [20, 21, 24, 25, 26, 27, 28].
43 The VAM was also successfully associated with the time reversal technique
44 for localizing and imaging damage in materials [29, 30, 31]. All these post-
45 processing techniques aim for improving the crack detection with fixed ex-
46 citations, while it has been proved that the effect of excitation parameters,
47 such as the frequency, can improve the VAM sensitivity [18, 22]. Pieczonka et

48 al. have used a swept sine chirp probe excitation and modal frequency pump
49 excitation associated with an advanced signal processing technique to find
50 the optimal probe frequency enhancing the VAM sensitivity [17]. Recently, a
51 novel method in which the pump frequency is omitted and the high frequency
52 is amplitude- modulated and contains 3 frequencies that interact in the pres-
53 ence of nonlinearity has been reported [32]. However, to our knowledge, no
54 study using VAM has tried to find the optimal excitation that improve the
55 sensitivity of the damage detection automatically by a closed loop system.

56 Previous results using the optimal command principle for nonlinear sys-
57 tems were initiated by Ménigot et al. [33, 34] in medical ultrasound imaging.
58 A first attempt in NDT was applied in [35] then in [36]. The originality of
59 these results is to search for the optimal input excitation parameter/shape
60 without any *a priori* knowledge on the studied system, by using known op-
61 timization algorithms. The key point of this method lies in the choice of the
62 cost function which must best write the optimization purpose.

63 The present study constitutes an extension to the NDT domain, espe-
64 cially, to the VAM technique, of the method demonstrated in [33], in which
65 the optimal command was applied to medical ultrasonic imaging. A conven-
66 tional VAM system is, then, replaced by a closed loop VAM system permit-
67 ting a real time optimization of input excitations parameters. We focused,
68 here, on the pump frequency parameter since the nonlinear modulation inten-
69 sity increase with the pump amplitude [18, 37, 38]. The aim of our study is
70 therefore to find automatically the best pumping frequency which maximizes
71 the nonlinear modulation occurring between the pump and the probe signals
72 without *a priori* information. The proposed method requires no preliminary

73 modal analysis and use a simple optimization genetic algorithm. In a first
74 time, the cost function must be adapted to the VAM. An empirical search of
75 the cost function behavior as a function of the optimization parameter was
76 achieved in order to show the relevance of our choices.

77 The remainder of the paper is organized as follow, in the Section 2, we
78 present our closed loop optimization system for the VAM technique. Section
79 3 describes the experiment including the studied sample and the setup. The
80 Section 4 is devoted to the experimental results of the empirical optimiza-
81 tion, a modal analysis for a comparison purpose, and the genetic algorithm
82 optimization. Finally, a discussion and conclusion are given in Section 5.

83 **2. Closed Loop Pump Frequency Optimization**

84 The proposed method is an optimal command method using a closed loop,
85 in order to optimize the VAM sensitivity of crack detection. Indeed, it makes
86 it possible to find the pump frequency which optimizes the nonlinear mod-
87 ulation effects. The cost function and its parameters need to be adequately
88 chosen. Here, only an iterative optimization procedure is implemented. The
89 conventional open loop system is replaced by a closed loop system in which
90 the transmitted pump frequency is modified by adding a feedback, ensuring
91 the optimization of the cost function. The closed loop system of the VAM
92 optimization is described in Fig. 1.

93 *2.1. Cost function*

94 As mentioned above, the goal of our study is to find automatically the
95 best pump frequency f_p^* , which maximizes the Nonlinear modulation (NM)

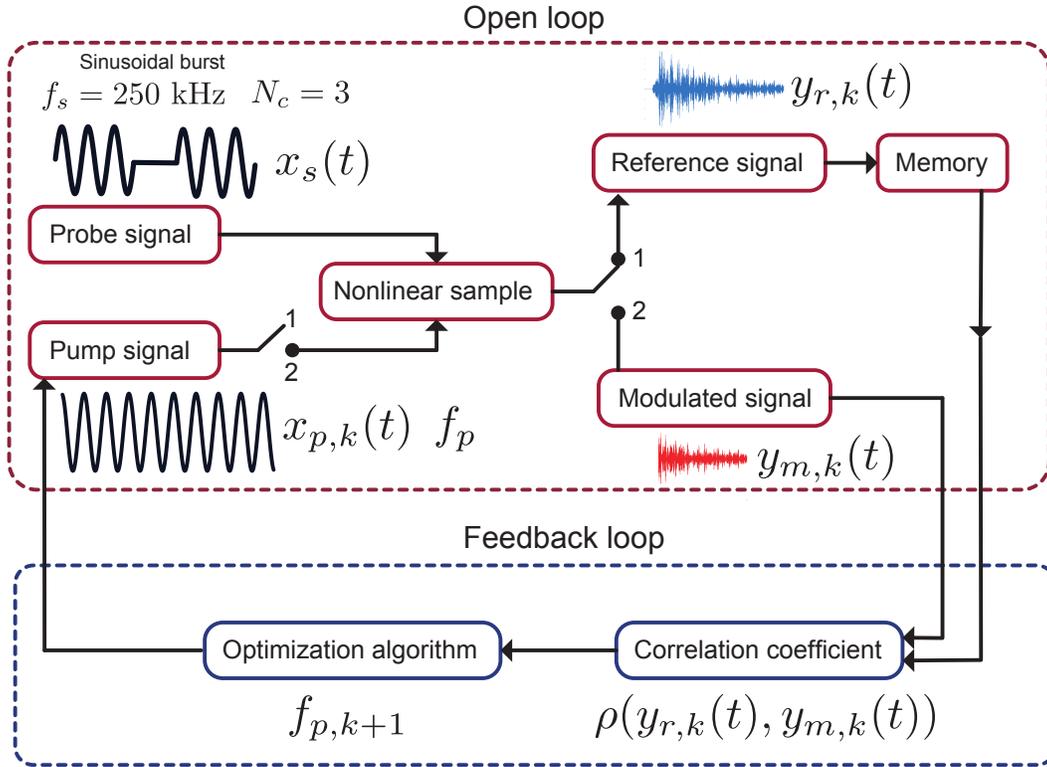


Figure 1: Bloc diagram of closed loop optimization of pump frequency for the VAM technique. The same probe excitation signal $x_s(t)$ is transmitted to the sample without the pumping excitation. The received reference probe coda signal $y_{r,k}(t)$ is recorded for a next use(switch position 1). In a second time (switch position 2), a pumping excitation $x_{p,k}(t)$ at a frequency $f_{p,k}$ is transmitted simultaneously with the probe excitation to the medium. The modulated coda probe signal $y_{m,k}(t)$ is also recorded. The added feedback consists of evaluating and minimizing the correlation coefficient ρ between $y_{r,k}(t)$ and $y_{m,k}(t)$. The optimization algorithm allowed to find a new pumping frequency $f_{p,k+1}$.

96 effects. Usually, to quantify the resulting NM effects in the context of mul-
 97 tiple scattering media, the Coda Wave Interferometry (CWI) in the time
 98 domain is used [13, 14, 39, 40]. We have chosen to proceed similarly, and
 99 extract the nonlinear modulation information in the time domain as it seems

100 to be more accessible. Indeed, the frequency response is very complex in
 101 multiple scattering media, and the modulation sidebands are difficult to see
 102 (see section 3.3).

103 The correlation coefficient ρ between the received probe coda signal with-
 104 out pumping and with the pump excitation constitutes our cost function.
 105 With the presence of a nonlinear scatterer in the sample, the probe signal is
 106 expected to be modulated by the pump excitation which induces to a decor-
 107 relation between the two signals. A correlation coefficient equal to unity
 108 means that there is no influence of the pump on the probe wave, *i.e.*, no non-
 109 linear modulation effect in the sample. On the contrary, for the same pump
 110 amplitude value, a deviation of ρ from 1 indicates the presence of nonlinear
 111 damage leading to a nonlinear modulation effect. In a theoretical point of
 112 view, the problem consists in calculating:

$$f_p^* = \arg \min_{f_p} (\rho(f_p)). \quad (1)$$

113 During the optimization process, for each iteration k , the same probe
 114 excitation signal $x_s(t)$ is transmitted to the sample without the pumping
 115 signal (see Fig.1 switch position 1), and the received reference probe coda
 116 signal $y_{r,k}(t)$ is recorded for a next use. In a second time (switch position 2), a
 117 pump excitation $x_{p,k}(t)$ at a frequency $f_{p,k}$ is transmitted simultaneously with
 118 the probe excitation to the medium. The modulated coda probe signal $y_{m,k}(t)$
 119 is also recorded. The added feedback consists in evaluating and optimizing
 120 a cost function, which is, in our case, the correlation coefficient between
 121 the two recorded received signals $\rho(y_{r,k}(t), y_{m,k}(t))$ within the time interval

122 $[t_1, t_2]$ such as:

$$\rho(y_{r,k}(t), y_{m,k}(t)) = \frac{\int_{t_1}^{t_2} (y_{r,k}(t) - \bar{y}_{r,k}(t)) (y_{m,k}(t) - \bar{y}_{m,k}(t)) dt}{\sqrt{\int_{t_1}^{t_2} (y_{r,k}(t) - \bar{y}_{r,k}(t))^2 dt \int_{t_1}^{t_2} (y_{m,k}(t) - \bar{y}_{m,k}(t))^2 dt}} \quad (2)$$

123 where $\bar{y}_{r,k}(t)$ is the average of the reference signal within the interval $[t_1, t_2]$,
 124 and $\bar{y}_{m,k}(t)$ that of the modulated signal. This coefficient quantifies the re-
 125 semblance between the reference probe signal and the modulated probe signal
 126 by the pump signal. The higher the nonlinear modulation between the probe
 127 and the pump signals is, the lower the value of the correlation coefficient
 128 is. An optimization algorithm is required to find a new pumping frequency
 129 $f_{p,k+1}$, at the iteration $k+1$, which maximizes the NM effect and therefore
 130 minimizes the correlation coefficient. The frequency is then modified and
 131 all the process described above is reiterated until the algorithm converges
 132 toward the best solution. In our study, the genetic algorithm was applied.

133 2.2. Genetic Algorithm

134 To find iteratively the pumping frequency giving the global minimum of
 135 the correlation coefficient, the genetic algorithm is used [41]. It is a search
 136 optimization technique based on the principles of genetics and natural se-
 137 lection. The genetic algorithm allows a population composed of a set of
 138 pumping frequencies to evolve under specified selection rules to a state that
 139 maximizes the "fitness" (*i.e.*, minimizes the correlation coefficient)[34, 42].
 140 The first step (called generation 1) consists of choosing randomly N pumping
 141 frequencies from a uniform distribution on a given frequencies interval. In
 142 our case, we have chosen $10\text{Hz} \leq f_p \leq 900\text{Hz}$. This choice is directly related

143 to the empirical optimization for a comparison purpose. Indeed, a largest
 144 initial population could be chosen. The correlation coefficient is evaluated
 145 for each pumping frequency value and sorted in descending order. To prepare
 146 the next step, the $N/2$ best pumping frequencies that minimize the corre-
 147 lation coefficient are kept for the next generation $k+1$ and become parents.
 148 $N/2$ new pumping frequencies named offspring are generated following the
 149 expressions [41, 42]:

$$\begin{aligned} \mathbf{offspring1} &= f_{pm} - \beta[f_{pm} - f_{pd}] \\ \mathbf{offspring2} &= f_{pm} + \beta[f_{pm} - f_{pd}] \end{aligned} \quad (3)$$

150 where β is a random value between 0 and 1. the subscripts m and d discrim-
 151 inates between the *mom* and the *dad* pumping frequencies. A percentage
 152 of the samples is mutated to obtain a robust optimization. There are some
 153 algorithm parameters that must be chosen such the population size N and
 154 the mutation rate R . In our case, $N= 12$ and the mutation rate $R=40\%$ [34].
 155 Finally, after cost function evaluation, the pumping frequency with the lower
 156 correlation coefficient is the best solution of generation $k+1$. **The genetic**
 157 **algorithm is adapted to global optimization problem; it means that it will be**
 158 **able to find the global minimum even if the function presents local minima.**
 159 **The GA is good for at identifying promising area of the search space but less**
 160 **efficient at fine-tuning the approximation to the minimum [43].**

161 3. Material

162 3.1. Sample description

163 An aluminium bar (600 mm \times 15 mm \times 3mm), with density $\rho=2700$
 164 kg/ m^3 , Poisson ratio $\sigma=0.33$, and Young modulus $E=69$ GPa [44], is used as

165 the specimen for our experiments. The probe wave propagation velocity in
166 the bulk of the bar is estimated to 4838 m/s. 10 tapped holes with a 4 mm
167 diameter and localized at distances of 10 mm, 30 mm, 50 mm, 100 mm and
168 200 mm from each sides of the bar center are drilled (see Fig. 2 from S1 to S10
169 from left to right). Identical screws ($m=4.42$ g) can be placed in the tapped
170 holes to mimic nonlinear solid contacts (cracks) such as in [45, 39, 40, 46].
171 These holes constitute linear scatterers when no screw is present. According
172 to the number of nonlinear scatterers on the bar, different levels of "effective"
173 damage can be obtained.

174 In the present study, three configurations have been more particularly
175 studied: the first one (Config 1) corresponds to the case where no screw is
176 placed in the sample bar; the reference. Nevertheless, one fixation screw is
177 placed in the middle to insure the link between the shaker and the bar (S0
178 in Fig. 2). Configuration 2 (Config 2) corresponds to the case where only
179 2 screws are positioned at S2 and S9, and a third configuration (Config 3)
180 is the case where all screws are placed (at S1, S2, S3, S4, S5, S6, S7, S8,
181 S9, and S10). Note that the nonlinearity level in the sample depends on the
182 screw number and the nuts tightening.

183 *3.2. Experimental Setup*

184 The closed loop pump frequency optimization process requires the exper-
185 imental setup depicted in Fig. 3. Two **broadband** piezoelectric transducers
186 with **250kHz** central frequencies for transmitting and receiving the probe sig-
187 nal are glued to the ends of the sample. For the transmission of the probe sig-
188 nal, a 100 mVpp sinusoidal burst of 3 periods of 250 kHz frequency, repeated
189 every 20 ms, was emitted by a function generator (AFG3022, Tektronix,

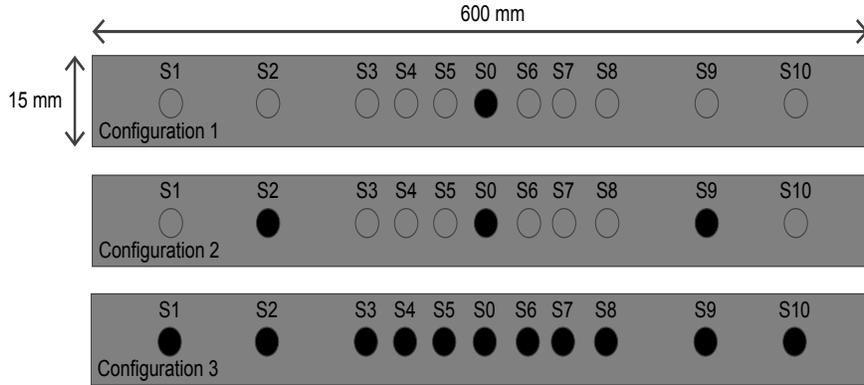


Figure 2: Specimen schemes of the three configurations. One Aluminium bar (600 mm \times 15 mm \times 3mm) containing 10 tapped holes. Config 1: no screws are placed. Config 2: 2 screws are placed at S2 and S9. Config 3: all the screws are placed on the bar.

190 Beaverton, Oregon, USA) and amplified to 60 dB (100Vpp) by a power am-
 191 plifier (Type 2713, Bruël & Kjær, Nærum, Denmark). Simultaneously, a
 192 lower frequency continuous sine pump signal is generated by the computer-
 193 controlled function generator (Tektronix, AFG3021C, Beaverton, Oregon,
 194 USA) to change the excitation frequency during the closed-loop optimiza-
 195 tion process. The pump signal is amplified by a power amplifier (PA100E,
 196 Bruël & Kjær, Nærum, Denmark), and transmitted to the shaker (LDS
 197 V406, Bruël & Kjær, Nærum, Denmark) which is connected to the center
 198 of the sample by a screw. The coda probe signal is detected by the receiving
 199 transducer and amplified by a preamplifier (Ciprian, Saint ISMIER, France),
 200 then, transmitted to an oscilloscope (LT 264ML, Lecroy, Chestnut Ridge,
 201 NY, USA). In order to improve the signal to noise ratio, an average of 300
 202 successive acquisitions is carried out, and a coda averaged signal is recorded.
 203 Each measurement lasts about 10s. Both the function generator and the
 204 oscilloscope are controlled by MATLAB (Mathworks, Natick, MA, USA).

205 The pump signal is desynchronized from the probe so that the nonlinear ef-
 206 fects are correctly distributed over the successive acquisitions and averaged
 over the acquired signal. By comparison, An experimental modal analysis

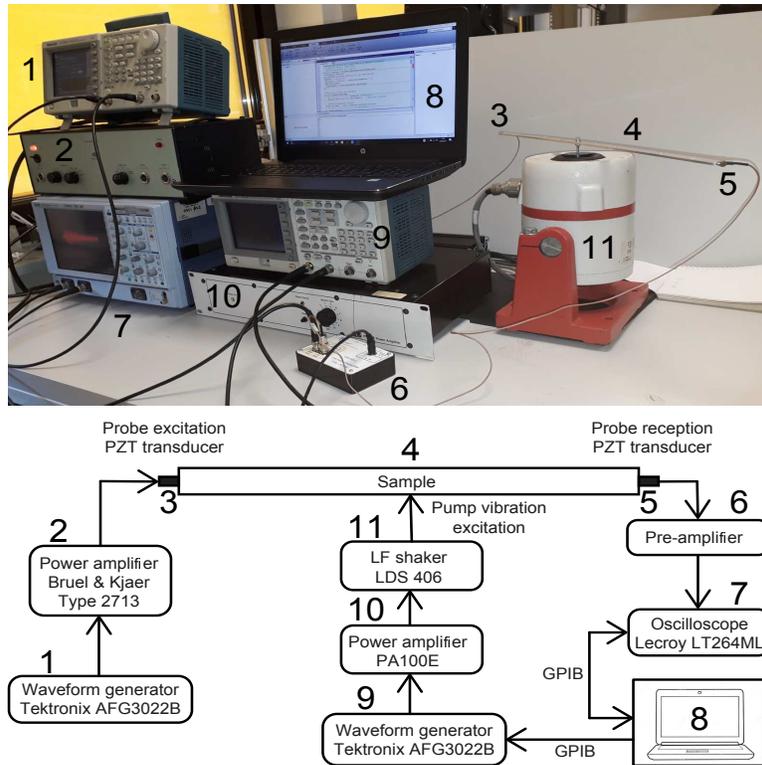


Figure 3: Experimental setup for the closed loop Vibro-Acoustic Modulation (VAM) technique.

207

208 is achieved to identify the frequencies of the first structural resonances of
 209 the sample. For this purpose, a frequency swept sine signal from 5Hz to
 210 1000Hz is generated using a signal analyzer (SR785, Stanford Research Sys-
 211 tems Inc., Sunnyvale, CA, USA). The signal is amplified by a PA100E power
 212 amplifier ,then, transmitted to a LDS406 shaker. The data are acquired by
 213 an accelerometer (352C23, PCB Piezotronics Inc, Depew, NY, USA) then

214 transmitted to the signal analyzer. Both the input excitation and the out-
215 put response are used to obtain the averaged Frequency Response Function
216 (FRF) of the sample. The excitation and the acquisition are controlled by a
217 GPIB connection via MATLAB.

218 *3.3. Nonlinear modulation effects on the probe coda signal*

219 In order to show the NM effects on the temporal coda signal, two con-
220 figurations are considered. The first one corresponds to the case without
221 screws (Config 1), and the second one corresponds to the Config 2. The
222 pump frequency is 150 Hz. Fig. 4 shows the received probe coda signal from
223 the Config 1 (Fig. 4a) and Config 2 (Fig. 4d). For each case, early and late
224 temporal windows are shown with and without the presence of the pump
225 excitation (Figs. 4b, c and e, f). The pump presence causes no detectable
226 change on the probe coda signal obtained from the intact sample, for early
227 and late windows: there is no nonlinear modulation of the probe by the pump
228 with a correlation coefficient of 1. On the contrary, for the damaged sample,
229 it can easily be noted that the two coda signals recorded with and without
230 the pumping wave are different for the late window $t=[1,1.05]$ ms (Fig. 4f),
231 with a correlation coefficient $\rho = 0.976$. It indicates that a NM has occurred
232 between the probe and the pump waves. Moreover, the nonlinear effects are
233 more visible in the late window because, at this time, the wave has crossed
234 the medium and interacted with the nonlinear scatterers several times. From
235 previous implementations of the nonlinear modulation of a coda wave by a
236 pump wave in a multiple scattering medium [13, 14, 46], the decorrelation is
237 either due to a localized velocity variation, to an amplitude-dependent dissipa-
238 tion effect or to an amplitude-dependent scattering effect. By introducing

239 internal solid contacts in the medium, between the screws and the holes,
240 amplitude- dependent clapping, tapping and frictional effects are expected.
241 These results consolidate our choice of quantifying the nonlinear modulation
242 effects by calculating the correlation coefficient between the probe coda sig-
243 nals obtained with and without the presence of the pump excitation, in a
244 time interval between $t_1=1\text{ms}$ and $t_2=1.2\text{ ms}$. It is important to notice that
245 the choice of the central time and the width of the temporal window in which
246 the correlation coefficient is estimated, obeys to some constraints [47]. The
247 characteristic scattering time t^* is estimated according to the method pro-
248 posed in [48], and is found to be around $3\ \mu\text{s}$ for our specimen. The starting
249 time of our window $t_1=1\text{ms}$ satisfies the condition $t_1 \gg t^*$. This ensures a
250 global inspection of the entire studied sample. Moreover the time interval
251 width is set to $200\ \mu\text{s}$, such as it includes enough signal periods (50 periods
252 of the probe signal at $250\ \text{kHz}$). The choice of the window position is also
253 limited by the signal quality. Indeed, the more t is increased, the more the
254 signal to noise ratio is degraded and the higher the uncertainty on the result
255 is.

256 In parallel, we illustrate in Fig. 5, the frequency spectra (right) of the
257 coda probe signals (left), obtained for Config 2, with and without the presence
258 of the pump wave excitation at $f_p=150\ \text{Hz}$. The selected time window is $\Delta t=$
259 $[1,1.2]\text{ms}$. We can notice that both frequency spectra are centered on 250
260 kHz , and include peaks due to the multiple scattered probe signal. The
261 modulation sidebands are difficult to see in this case, and the only difference
262 between the two spectra is an amplitude variation and a slight frequency
263 shift. Moreover, the frequency resolution is not sufficient to observe the

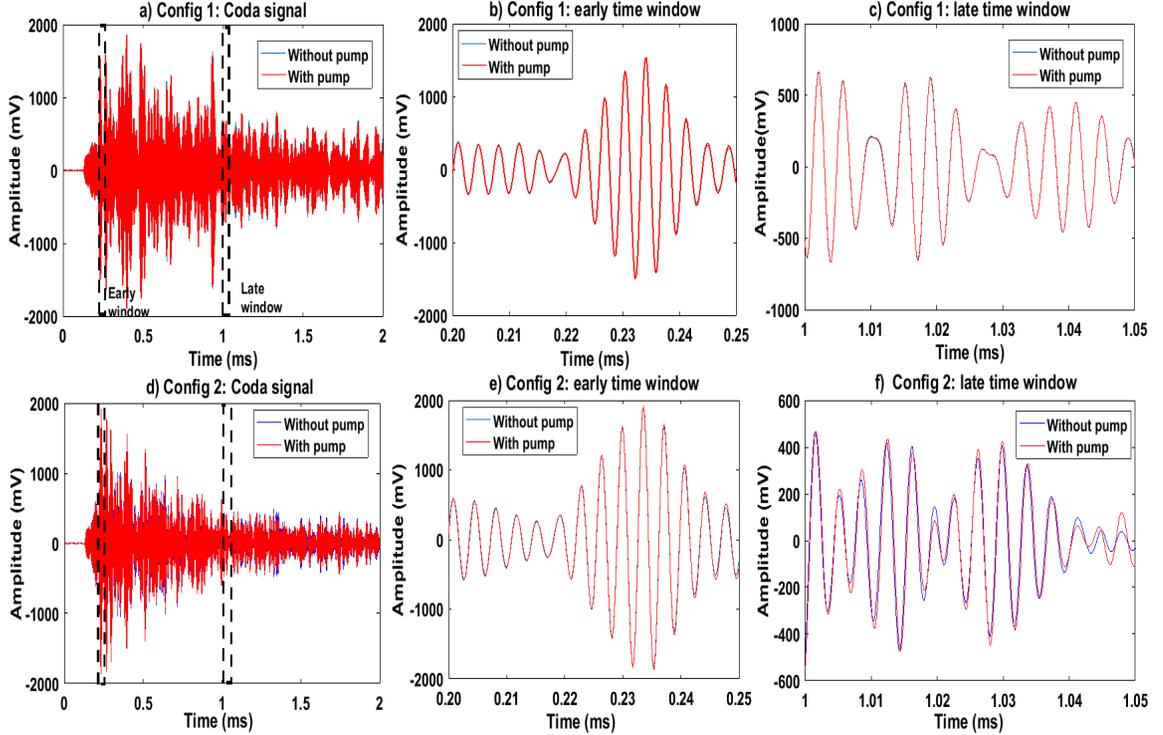


Figure 4: Experimental temporal coda probe signals obtained from the Config 1 (without screws): (a), (b) and (c), and for the Config 2 (with screws): (d), (e) and (f), with and without the pump excitation. The pump excitation is a continuous sine signal at a resonance frequency $f_p=150$ Hz. (b) and (e) are captures of the coda signals (a) and (d), respectively, during an early time interval between $[0.2, 0.25]$ ms. (c) and (f) are captures of (a) and (d), respectively, during a late interval between $[1, 1.05]$ ms.

264 modulation sidebands when $f_{pump} \ll f_{probe}$. Therefore, we choose to extract
 265 to modulation effects information in the time domain.

266 During all this study, it should be noticed that the decorrelation values
 267 are not observed higher than 2.5%. We recall, here, that we want to test our
 268 approach in a rather unfavorable configuration, *i.e.*, when the variation of

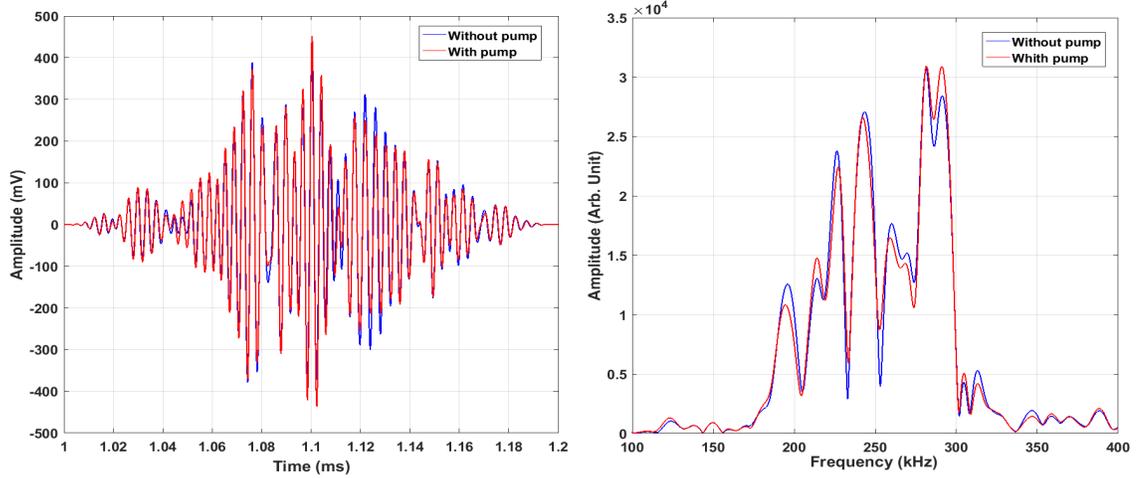


Figure 5: (left) Experimental probe coda signals obtained from the Config 2, with and without the presence of the pump. The pump excitation is a continuous sine wave at 150Hz. The coda signal is filtered with a hanning window for a duration $\Delta t=[1,1.2]$ ms. (right) The frequency spectra of the coda signals.

269 the correlation coefficient is small. Indeed, a stronger nut tightening and a
 270 higher pump amplitude would have given higher levels of decorrelation.

271 4. Experimental Results

272 In this section, the main experimental results are presented. First an
 273 experimental modal analysis is performed in order to obtain the resonance
 274 modes of the sample for a comparison purpose. Then, an empirical research of
 275 the correlation coefficient ρ behavior *versus* the pump frequency is presented
 276 and compared to the modal analysis results. Finally, a closed loop optimiza-
 277 tion of the correlation coefficient by the genetic algorithm is performed and
 278 presented.

279 *4.1. Empirical optimization: Correlation coefficient dependency on the pump*
280 *frequency*

281 The frequency response function (FRF) is obtained using the experimen-
282 tal input and output data obtained from the experiment described in the
283 section 3.2. This preliminary experiment allows identifying the frequency
284 resonance modes of the sample in the Configs 2 and 3. Figs. 6a and 6c
285 show the FRFs obtained for Configs 2 and 3, respectively, in a frequency
286 range from 5Hz to 900 Hz. We can see that the magnitude of the resonance
287 peaks and the corresponding frequencies are different for the two configura-
288 tions. The results show that there is a shift in the natural frequencies and
289 an overall change in the frequency response due to the effect of the added
290 screws.

291 The first performed experiment is to check the existence of global or lo-
292 cal minima for the correlation coefficient *versus* the pump frequency. This
293 step is called "empirical optimization". For time saving, a pump frequency
294 sine sweeping from 10 Hz to 900 Hz with a step of 10 Hz is achieved. This
295 allows a global view of the correlation coefficient dependency on the pump
296 frequency. For this first pass, the prominent observation is the presence of
297 local minima peaks localized at specific vibration frequencies corresponding
298 well to the resonances of the bar, and $\rho \simeq 1$ elsewhere. To refine the results
299 and to get a better accuracy, a second sweeping was achieved with a finer
300 frequency step of 1 Hz, over the frequency regions where ρ deviates from
301 1 (Figs. 6b and 6d). The procedure cited in the section 2.1 is adopted to
302 calculate the correlation coefficient for each pump frequency. As illustrated
303 in Fig. 6, the pump frequencies that give minima peaks of ρ coincide well

304 with the resonance frequencies of the sample for the Configs 2 and 3. This
 305 confirms the expected effects that at a resonance frequency of the sample,
 306 the vibration amplitude is naturally amplified by constructive interferences,
 307 leading to a more efficient nonlinear modulation effect. Also, we can note a
 308 frequency shift to the lower frequencies with the increase of the number of
 309 screws in the plate, as it can be seen in Figs. 6b and 6d. Additionally, we
 310 can see clearly that for the same resonance mode, the correlation coefficient
 311 values vary from one configuration to another. In fact, the second minimum
 312 peak corresponding to $f_p=150$ Hz is the most important for the Config 2;
 313 $\rho= 0.975$ (Fig. 6b) while $\rho= 0.981$ at $f_p=144$ Hz for the Config 3 (Fig. 6d).
 314 We can also notice the third ρ peak is very close to 1 ($\rho=0.998$) for the
 315 Config 2 (Fig. 6b) and equal to 0.986 for the Config 3. These observations
 316 confirm the well-known fact that in the pump resonance configuration, the
 317 NM efficiency on the probe wave by the pump wave depends on the pump-
 318 ing frequency/mode and on the exact position of the nonlinear scatterers.
 319 Based on the obtained curves, it can be stated that for each configuration, it
 320 exists an optimal pump frequency that maximizes the nonlinear modulation
 321 effects and, thus, the damage detection sensitivity. This optimal frequency
 322 corresponds to the frequency giving the lowest minimum of ρ . The empiri-
 323 cal experiment also shows that the correlation coefficient is a good indicator
 324 to evaluate the nonlinear wave mixing in the sample. For an easier com-
 325 parison of results obtained from Fig. 6, table 1 summarizes the correlation
 326 coefficient minima peaks values and the corresponding pumping frequencies
 327 obtained from the empirical optimization (EO) and those obtained from the
 328 FRF for both configurations.

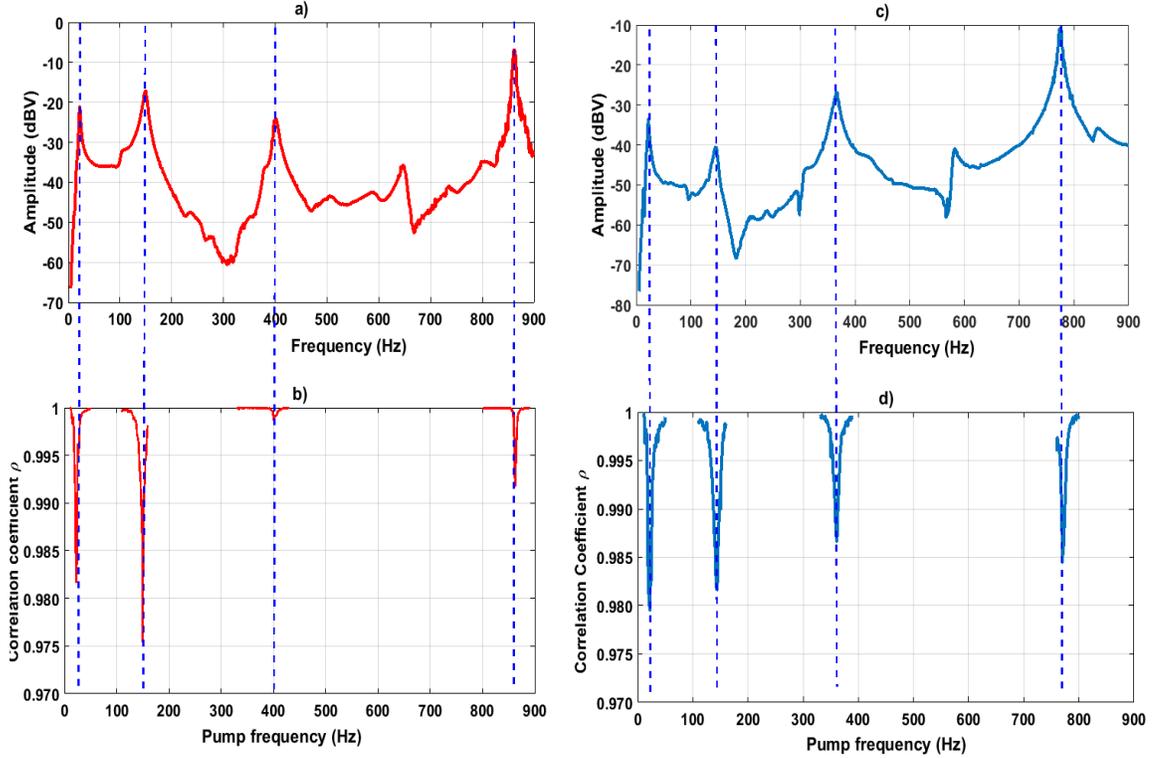


Figure 6: Comparison between the resonance frequencies obtained from the experimental FRF of the Config 2 (a) and the Config 3 (c). Experimental results of correlation coefficient *versus* pump frequency at the sample resonance frequencies areas for the two sample configurations: b) Config 2 and d) Config 3. The FRF peaks coincide with the minimum peaks of the correlation coefficient *versus* pumping frequency.

329 4.2. Optimal pumping frequency excitation

330 Fig. 7 shows the results of the pump frequency optimization for the Con-
 331 fig 2 and Fig. 8 shows those obtained for the Config 3 when using the genetic
 332 algorithm optimization. For more clarity, the result of the empirical optimiza-
 333 tion is represented in Figs. 7a and 8a with a zoom on the GA convergence
 334 area. The initial population was included in a frequency range between 10 Hz

| Modes | Config 2 | | | Config 3 | | |
|----------------------|-------------|-----------------|------------------|-------------|-----------------|------------------|
| | ρ_{EO} | $f_{p,EO}$ (Hz) | $f_{p,FRF}$ (Hz) | ρ_{EO} | $f_{p,EO}$ (Hz) | $f_{p,FRF}$ (Hz) |
| 1 st Mode | 0.981 | 23 | 22.1 | 0.979 | 22 | 22.1 |
| 2 nd Mode | 0.975 | 150 | 148.7 | 0.981 | 144 | 145.7 |
| 3 rd Mode | 0.998 | 400 | 401.0 | 0.986 | 361 | 365.8 |
| 4 th Mode | 0.991 | 862 | 861.3 | 0.984 | 770 | 773.9 |

Table 1: The correlation coefficient ρ and the f_p values corresponding to the minimum peaks obtained from the empirical optimization (EO) and the resonance frequencies obtained from the Frequency Response Function (FRF) for Config 2 and 3.

335 and 900 Hz, with a population size of 6 for the Config 2 and 12 for the Config
336 3. We can see that in both cases, the optimal pump frequency is reached
337 after 4 generations for the Config 2 with an optimal pump frequency of 153.2
338 Hz and $\rho=0.991$ (Figs. 7b and 7c), and after 7 generations for the Config 3
339 with $f_p=22.17$ Hz and $\rho=0.971$ (Figs. 8b and 8c). As it is illustrated in Figs.
340 7a and 8a the obtained results are quite similar with those obtained by the
341 empirical optimization. For Config 2, the optimal pump frequency is reached
342 by the genetic algorithm with a slight discrepancy in the pump frequency and
343 the corresponding correlation coefficient. Indeed, we can see in Fig. 7a, that
344 the GA converges nearly to the global minimum at $f_p=153.2$ Hz, in a pump
345 frequency range between 10 and 900 Hz, including four (4) local minima of
346 the cost function, but the accurate value of this minimum, at $f_p=150$ Hz, is
347 not reached. We suppose that a greater number of iterations should improve
348 this result. Moreover, it is important to notice that for the empirical search,
349 the step between two pump frequencies is chosen empirically, this may be

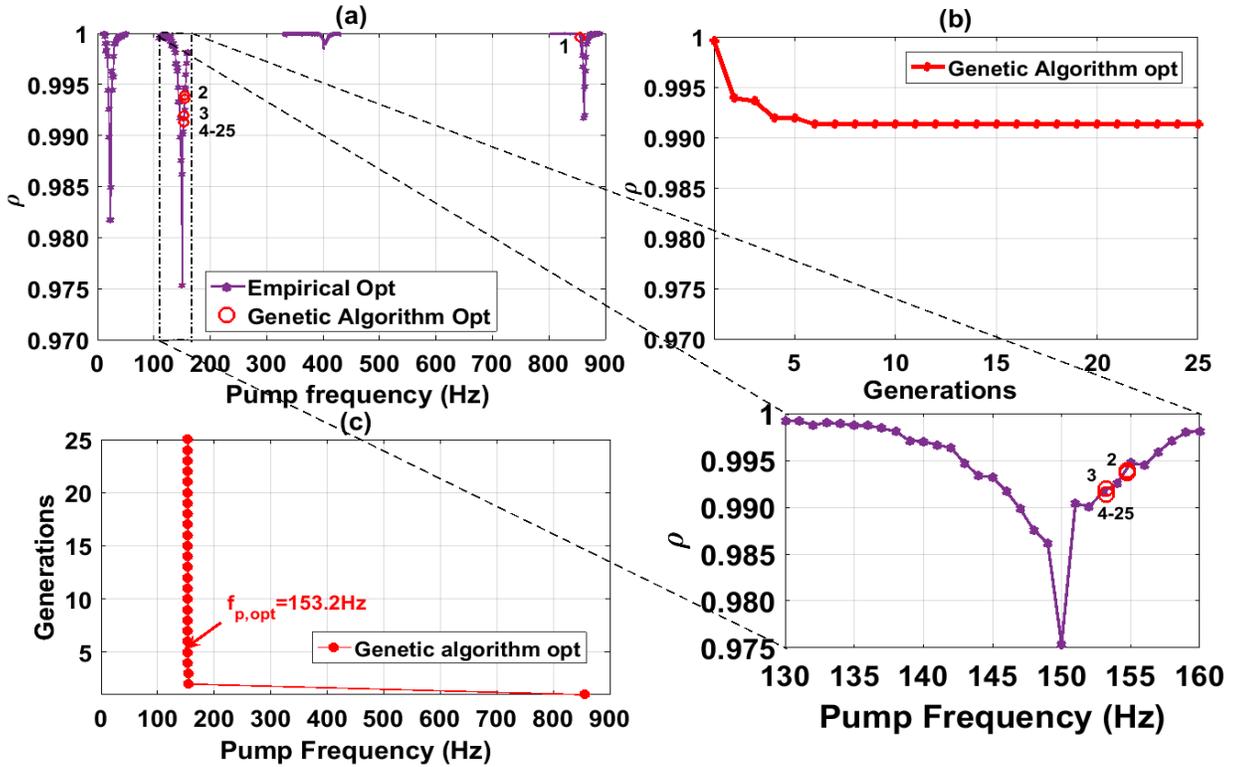


Figure 7: Genetic algorithm (GA) optimization results for the Config 2: (a) Empirical optimization and GA iteration numbers (red circles), a zoom on the GA convergence area is illustrated. (b) The correlation coefficient ρ versus generations. (c) The corresponding pump frequency versus generations. The GA converges to $f_{p,opt}=153.2$ Hz after the 4th generation.

350 the origin of the discrepancy observed between the empirical and the auto-
 351 matic optimization. For the Config 3, the optimal pump frequency is nearly
 352 reached. For both cases, the pump frequency converges while ρ remains vari-
 353 able for the same value of the pump frequency. This can be clearly seen in
 354 Figs. 8b and 8c where the pump frequency converges to $f_p=22.17$ Hz, and
 355 the corresponding ρ still varies slightly. This observation is directly due to

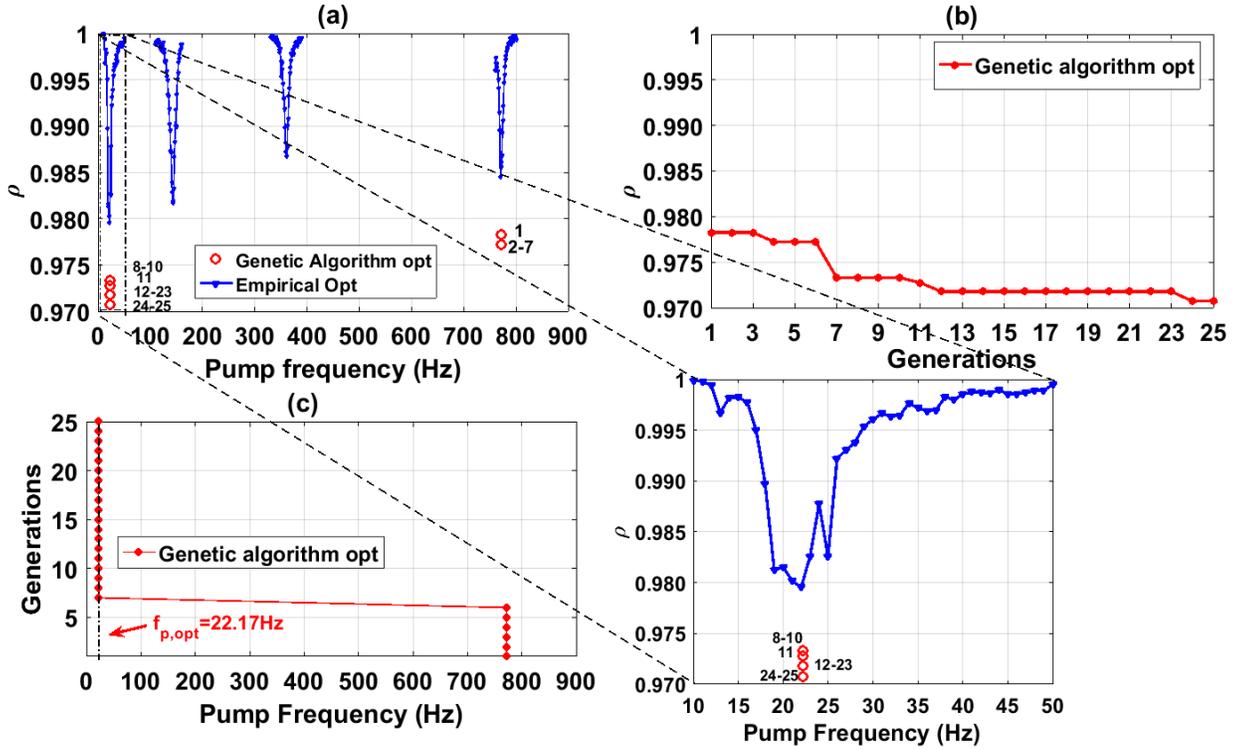


Figure 8: Genetic algorithm (GA) optimization results for the Config 3: (a) Empirical optimization and GA iteration numbers (red circles), a zoom on the GA convergence area is illustrated. (b) The correlation coefficient ρ versus generations. (c) The corresponding pump frequency versus generations. The GA converges to $f_{p,opt} = 22.17$ Hz after the 7th generation.

356 the experimental conditions since the coda signal is very sensible to small
 357 temperature changes. Moreover, the correlation coefficient value reaches a
 358 lower value than that obtained by the empirical optimization (Fig. 8a). It
 359 is important to mention that the empirical cost function is not a continuous
 360 function, and a measurement is made with a pump frequency step of 1Hz. We
 361 think, that a finer pump frequency step between two successive cost function

| | Config 3 | | Config 2 | |
|------------------------|-----------------|------------------|-----------------|------------------|
| | ρ | $f_{p,opt}$ (Hz) | ρ | $f_{p,opt}$ (Hz) |
| Genetic Algorithm | 0.971 | 22.17 | 0.991 | 153.2 |
| Empirical optimization | 0.979 | 22.00 | 0.975 | 150.0 |

Table 2: Experimental optimization results of the genetic algorithm convergence and the empirical values of the global minimum obtained by the empirical search, for the Config 2 and 3.

362 measurements should give a better concordance between the empirical and
363 the automatic search by the GA. Another possible explanation is that for
364 the empirical optimization, one measurement of the cost function has been
365 made for each pump frequency value between 10-900Hz, while for the GA
366 optimization, and after convergence, several measurements have been made
367 at the same optimal pump frequency (resonance frequency). In fact, exciting
368 a sample at its resonance frequency during a long time (conditioning), could
369 gives rise to a greater decorrelation between the reference coda signal and
370 the modulated coda signal. Indeed, for the modal excitation, the vibration
371 response is amplified such as the medium does not reach the equilibrium and
372 the reference signal is still affected by the previous iterations. This may be
373 linked to "slow dynamics" effects [49].

374 We can also note that the algorithm convergence is faster when the pop-
375 ulation size is lower. The strong point of our approach is that the algorithm
376 converged to the global minimum of the correlation coefficient without *a pri-*
377 *ori* information. For more clarity, table 2 summarizes the results obtained
378 by both the empirical and the GA optimization.

379 5. Discussion and Conclusion

380 In this work, a widely studied nonlinear ultrasonic nondestructive testing
381 method has been combined with the optimal command method: an open loop
382 Vibro Acoustic Modulation system has been enriched with a feedback sys-
383 tem. The main idea was to achieve an optimal damage detection by finding
384 automatically the best input pump frequency. This input frequency maxi-
385 mizes the nonlinear modulation effects induced by the presence of nonlinear
386 scatterers and, therefore, should increase the sensitivity of the VAM method.
387 The best pump frequency adjusted to each configuration has been success-
388 fully found by the feedback method. This method is based, principally, on
389 the good choice of a so called "cost function" and the optimization parame-
390 ter. Moreover, the experiment setting is easy and not user dependent. The
391 genetic algorithm was tested on two sample configurations with different lev-
392 els of nonlinearity to demonstrate the adaptivity of the proposed method. It
393 is found able to determine automatically the optimal pump frequency over
394 a wide frequency range including 4 resonance modes. Even if the genetic
395 algorithm is not always efficient to find the **accurate optimal value**, it guar-
396 antees to find the global optimum region quite fast since four (4) generations
397 were sufficient in our case. A preliminary empirical search was achieved to
398 check for the behavior of the cost function, the correlation coefficient as a
399 function of the optimization parameter (the pump frequency). The results of
400 this empirical search, requiring a long blind experimental search, have been
401 presented in the form of correlation coefficient *versus* pump frequency. The
402 main outcome is the existence of frequencies, which correspond to some of the
403 resonance frequencies of the sample, giving a maximum of decorrelation (or

404 a minimum correlation). It constitutes a first suitable result which confirms
405 our hypothesis of choosing the correlation coefficient of the probe coda sig-
406 nal, with and without pumping to quantify the NM effects, and the pumping
407 frequency as an optimization parameter. The quantitative variation of ρ with
408 the pump resonance frequencies, for the different configurations is related to
409 the modal vibration shapes and the associated strain amplitude changes from
410 a resonance mode to another, at the location of the nonlinear scatterers. For
411 different arrangements of nonlinear scatterers, we observe differences in the
412 values of the correlation coefficient, because a given resonant mode does not
413 necessarily excites differently located nonlinear scatterers with the same effi-
414 ciency. Similarly, for a given medium configuration, but when comparing two
415 different resonant modes, the nodes and anti-nodes positions are not located
416 identically, and they do not excite with the same efficiency a given nonlinear
417 scatterer. As such, using a single pump frequency leads to the apparition of
418 blind zones for the VAM, *e.g.* when a single scatterer is at a strain node. For
419 this reason, a preliminary modal analysis to determine a resonance modes of
420 the studied medium seems to be not sufficient to ensure an increase of the
421 VAM sensitivity detection.

422 The proposed method, in a resonance configuration, ensures to avoid the
423 low sensitivity regions associated with the strain nodes, which could compro-
424 mise the nonlinear scatterer detection. Moreover, for a fixed pump ampli-
425 tude, the adjusted pump frequency excitation may bring maximum detection
426 sensitivity for nonlinear damages, which can be very useful to detect early
427 stage damage in materials. Furthermore, the advantage of the automatic
428 optimization is a time gain compared to the empirical optimization which

429 takes about 15 times more then our method.

430 The optimal command can be extended to other NDT methods by defin-
431 ing the right cost functions adapted to the optimization purpose. Also, as
432 a perspective, the genetic algorithm outcomes could be used as the initial-
433 ization for a gradient descent algorithm in order to refine the results. In
434 addition, information on the local minima of the correlation coefficient may
435 be relevant for locating the nonlinear scatterers: each resonant mode consti-
436 tutes a spatial sensitivity kernel for the method and information stacking for
437 several modes may offer imaging capabilities.

438 **References**

- 439 [1] O. Buck, W. Morris, J. M. Richardson, Acoustic harmonic generation at
440 unbonded interfaces and fatigue cracks, *Applied Physics Letters* 33 (5)
441 (1978) 371–373.
- 442 [2] I. Y. Solodov, Ultrasonics of non-linear contacts: propagation, reflection
443 and nde-applications, *Ultrasonics* 36 (1-5) (1998) 383–390.
- 444 [3] P. B. Nagy, Fatigue damage assessment by nonlinear ultrasonic materials
445 characterization, *Ultrasonics* 36 (1-5) (1998) 375–381.
- 446 [4] Y. Zheng, R. G. Maev, I. Y. Solodov, Review/sythèse nonlinear acoustic
447 applications for material characterization: a review, *Canadian Journal*
448 *of Physics* 77 (12) (2000) 927–967.
- 449 [5] K.-A. Van Den Abeele, J. Carmeliet, J. A. Ten Cate, P. A. Johnson, Non-
450 linear elastic wave spectroscopy (*NEWS*) techniques to discern material

- 451 damage, part *II*: Single-mode nonlinear resonance acoustic spectroscopy,
452 Journal of Research in Nondestructive Evaluation 12 (1) (2000) 31–42.
- 453 [6] V. V. Kazakov, A. Sutin, P. A. Johnson, Sensitive imaging of an elas-
454 tic nonlinear wave-scattering source in a solid, Applied Physics Letters
455 81 (4) (2002) 646–648.
- 456 [7] G. Renaud, S. Callé, M. Defontaine, Remote dynamic acoustoelastic
457 testing: Elastic and dissipative acoustic nonlinearities measured under
458 hydrostatic tension and compression, Applied Physics Letters 94 (1)
459 (2009) 011905.
- 460 [8] D. M. Donskoy, A. M. Sutin, Vibro-acoustic modulation nondestructive
461 evaluation technique, Journal of intelligent material systems and
462 structures 9 (9) (1998) 765–771.
- 463 [9] L. Pieczonka, A. Klepka, A. Martowicz, W. J. Staszewski, Nonlinear
464 vibroacoustic wave modulations for structural damage detection: an
465 overview, Optical Engineering 55 (1) (2015) 011005.
- 466 [10] K.-A. Van Den Abeele, P. A. Johnson, A. Sutin, Nonlinear elastic wave
467 spectroscopy (*NEWS*) techniques to discern material damage, part *I*:
468 nonlinear wave modulation spectroscopy (*NWMS*), Journal of Research
469 in Nondestructive Evaluation 12 (1) (2000) 17–30.
- 470 [11] M. Meo, U. Polimeno, G. Zumpano, Detecting damage in composite
471 material using nonlinear elastic wave spectroscopy methods, Applied
472 composite materials 15 (3) (2008) 115–126.

- 473 [12] G. Gao, D. Li, D. Shi, J. Dong, X. Shi, F. Teng, Nonlinear acoustic
474 characteristics of fatigue microcracks in al alloy plate, *JOM* 63 (2) (2011)
475 77–80.
- 476 [13] Y. Zhang, V. Tournat, O. Abraham, O. Durand, S. Letourneur,
477 A. Le Duff, B. Lascoup, Nonlinear mixing of ultrasonic coda waves with
478 lower frequency-swept pump waves for a global detection of defects in
479 multiple scattering media, *Journal of Applied Physics* 113 (6) (2013)
480 064905.
- 481 [14] Y. Zhang, V. Tournat, O. Abraham, O. Durand, S. Letourneur,
482 A. Le Duff, B. Lascoup, Nonlinear coda wave interferometry for the
483 global evaluation of damage levels in complex solids, *Ultrasonics* 73
484 (2017) 245–252.
- 485 [15] H. Hu, W. Staszewski, N. Hu, R. Jenal, G. Qin, Crack detection us-
486 ing nonlinear acoustics and piezoceramic transducers instantaneous am-
487 plitude and frequency analysis, *Smart materials and structures* 19 (6)
488 (2010) 065017.
- 489 [16] A. Klepka, W. Staszewski, R. Jenal, M. Szvedo, J. Iwaniec, T. Uhl,
490 Nonlinear acoustics for fatigue crack detection—experimental investiga-
491 tions of vibro-acoustic wave modulations, *Structural Health Monitoring*
492 11 (2) (2012) 197–211.
- 493 [17] L. Pieczonka, A. Klepka, M. Adamczyk, W. Staszewski, F. Aymerich,
494 T. Uhl, Optimal selection of parameters for impact damage detec-
495 tion in composites based on nonlinear vibro-acoustics modulations, in:

- 496 ECCM16 16th European Conference on Composite Materials, Seville,
497 Spain, 22-26 June 2014.
- 498 [18] B. Liu, Z. Luo, T. Gang, Influence of low-frequency parameter changes
499 on nonlinear vibro-acoustic wave modulations used for crack detection,
500 Structural Health Monitoring (2017) 1475921716689385.
- 501 [19] N. C. Yoder, D. E. Adams, Vibro-acoustic modulation utilizing a swept
502 probing signal for robust crack detection, Structural Health Monitoring
503 9 (3) (2010) 257–267.
- 504 [20] H. Sohn, H. J. Lim, M. P. DeSimio, K. Brown, M. Derriso, Nonlinear
505 ultrasonic wave modulation for online fatigue crack detection, Journal
506 of Sound and Vibration 333 (5) (2014) 1473–1484.
- 507 [21] B. Liu, T. Gang, C. Wan, C. Wang, Z. Luo, Analysis of nonlinear modu-
508 lation between sound and vibrations in metallic structure and its use for
509 damage detection, Nondestructive Testing and Evaluation 30 (3) (2015)
510 277–290.
- 511 [22] K. Dziedziech, L. Pieczonka, M. Adamczyk, A. Klepka, W. J.
512 Staszewski, Efficient swept sine chirp excitation in the non-linear vibro-
513 acoustic wave modulation technique used for damage detection, Struc-
514 tural Health Monitoring 17 (3) (2018) 565–576.
- 515 [23] M. Dunn, A. Carcione, P. Blanloeuil, M. Veidt, Critical aspects of
516 experimental damage detection methodologies using nonlinear vibro-
517 ultrasonics, Procedia Engineering 188 (2017) 133–140.

- 518 [24] C. R. Courtney, B. W. Drinkwater, S. A. Neild, P. D. Wilcox, Factors
519 affecting the ultrasonic intermodulation crack detection technique using
520 bispectral analysis, *NDT & E International* 41 (3) (2008) 223–234.
- 521 [25] W. Collis, P. White, J. Hammond, Higher-order spectra: the bispectrum
522 and trispectrum, *Mechanical systems and signal processing* 12 (3) (1998)
523 375–394.
- 524 [26] J. Jiao, J. Sun, N. Li, G. Song, B. Wu, C. He, Micro-crack detection
525 using a collinear wave mixing technique, *NDT & E International* 62
526 (2014) 122–129.
- 527 [27] C. Courtney, S. Neild, P. Wilcox, B. Drinkwater, Application of the bis-
528 pectrum for detection of small nonlinearities excited sinusoidally, *Jour-
529 nal of Sound and Vibration* 329 (20) (2010) 4279–4293.
- 530 [28] N. Li, J. Sun, J. Jiao, B. Wu, C. He, Quantitative evaluation of micro-
531 cracks using nonlinear ultrasonic modulation method, *Ndt & E Interna-
532 tional* 79 (2016) 63–72.
- 533 [29] A. Sutin, P. Johnson, Nonlinear elastic wave nde ii. nonlinear wave mod-
534 ulation spectroscopy and nonlinear time reversed acoustics, in: *AIP
535 Conference Proceedings*, Vol. 760, AIP, 2005, pp. 385–392.
- 536 [30] G. Zumpano, M. Meo, A new nonlinear elastic time reversal acoustic
537 method for the identification and localisation of stress corrosion cracking
538 in welded plate-like structures—a simulation study, *International journal
539 of solids and structures* 44 (11-12) (2007) 3666–3684.

- 540 [31] T. Goursolle, S. Dos Santos, O. B. Matar, S. Calle, Non-linear based
541 time reversal acoustic applied to crack detection: Simulations and ex-
542 periments, *International Journal of Non-Linear Mechanics* 43 (3) (2008)
543 170–177.
- 544 [32] A. Carcione, P. Blanloeuil, L. Rose, C. H. Wang, M. Veidt, Modulated
545 high frequency excitation approach to nonlinear ultrasonic ndt, *Journal*
546 *of Sound and Vibration* 446 (2019) 238–248.
- 547 [33] S. Ménigot, J.-M. Girault, I. Voicu, A. Novell, Optimization of contrast-
548 to-tissue ratio by frequency adaptation in pulse inversion imaging, *IEEE*
549 *transactions on ultrasonics, ferroelectrics, and frequency control* 59 (11).
- 550 [34] S. Ménigot, J.-M. Girault, Optimization of contrast resolution by genetic
551 algorithm in ultrasound tissue harmonic imaging, *Ultrasonics* 71 (2016)
552 231–244.
- 553 [35] S. Ménigot, M. Geryes, J. M. Girault, Inclusion/ flaw detection in ultra-
554 sound imaging through optimization of random transmitted wave, in:
555 *Acoustics 2013*, 2013.
- 556 [36] N. Houhat, S. Ménigot, T. Boutkedjirt, R. Draï, J.-M. Girault, Optimal
557 stochastic excitation for linear flaw detection in a solid material, *Lecture*
558 *Notes in Computer Science* 11401 (2019) 229–236.
- 559 [37] K. E. Van Den Abeele, A. Sutin, J. Carmeliet, P. A. Johnson, Micro-
560 damage diagnostics using nonlinear elastic wave spectroscopy (*NEWS*),
561 *Ndt & E International* 34 (4) (2001) 239–248.

- 562 [38] V. Zaitsev, P. Sas, Nonlinear response of a weakly damaged metal sam-
563 ple: a dissipative modulation mechanism of vibro-acoustic interaction,
564 Journal of Vibration and Control 6 (6) (2000) 803–822.
- 565 [39] B. Hilloulin, Y. Zhang, O. Abraham, A. Loukili, F. Grondin, O. Durand,
566 V. Tournat, Small crack detection in cementitious materials using non-
567 linear coda wave modulation, NDT & E International 68 (2014) 98–104.
- 568 [40] J.-B. Legland, Y. Zhang, O. Abraham, O. Durand, V. Tournat, Evalu-
569 ation of crack status in a meter-size concrete structure using the ultra-
570 sonic nonlinear coda wave interferometry, The Journal of the Acoustical
571 Society of America 142 (4) (2017) 2233–2241.
- 572 [41] A. H. Wright, Genetic algorithms for real parameter optimization, in:
573 Foundations of genetic algorithms, Vol. 1, Elsevier, 1991, pp. 205–218.
- 574 [42] R. L. Haupt, S. E. Haupt, S. E. Haupt, Practical genetic algorithms,
575 Vol. 2, Wiley New York, 1998.
- 576 [43] L. Costa, I. Santo, R. Denysiuk, E. Fernandes, Hybridization of a genetic
577 algorithm with a pattern search augmented lagrangian method, 2010.
- 578 [44] J. Krautkrämer, H. Krautkrämer, Ultrasonic testing of materials,
579 Springer Science & Business Media, 2013.
- 580 [45] J. Riviere, G. Renaud, S. Hauptert, M. Talmant, P. Laugier, P. A. John-
581 son, Probing interface elastic nonlinearity applying nonlinear resonance
582 ultrasound spectroscopy: The case of screw tightness-of-fit, Journal of
583 Applied Physics 107 (2010) 124901.

- 584 [46] G. Chen, D. Pageot, J.-B. Legland, O. Abraham, M. Chekroun, V. Tour-
585 nat, Numerical modeling of ultrasonic coda wave interferometry in a
586 multiple scattering medium with a localized nonlinear defect, *Wave Mo-*
587 *tion* 72 (2017) 228–243.
- 588 [47] C. Payan, V. Garnier, J. Moysan, P. Johnson, Determination of third
589 order elastic constants in a complex solid applying coda wave interfer-
590 ometry, *Applied Physics Letters* 94 (1) (2009) 011904.
- 591 [48] D. M. Egle, Diffuse wave fields in solid media, *The Journal of the Acous-*
592 *tical Society of America* 70 (2) (1981) 476–480.
- 593 [49] J. A. Ten Cate, T. J. Shankland, Slow dynamics in the nonlinear elas-
594 tic response of berea sandstone, *Geophysical Research Letters* 23 (21)
595 (1996) 3019–3022.