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# Geometrical control of dissipation during the spreading of liquids on soft solids

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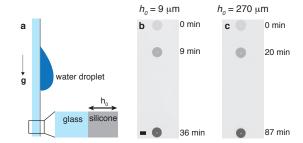
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Gel layers bound to a rigid substrate are used in cell culture to control differentiation and migration and to lower the friction and tailor the wetting of solids. Their thickness, often considered a negligible parameter, affects cell mechanosensing or the shape of sessile droplets. Here we show that the adjustment of coating thickness provides control over energy dissipation during the spreading of flowing matter on a gel layer. We combine experiments and theory to provide an analytical description of both the statics and the dynamics of the contact line between the gel, the liquid and the surrounding atmosphere. We extract from this analysis a hitherto unknown scaling law that predicts the dynamic contact angle between the three phases as a function of the properties of the coating and the velocity of the contact line. Finally, we show that droplets moving on vertical substrates coated with gel layers having linear thickness gradients drift towards regions of higher energy dissipation. Thus thickness control opens the opportunity to design a priori the path followed by large droplets moving on gel-coated substrates. Our study shows that thickness is another parameter, besides surface energy and substrate mechanics, to tune the dynamics of liquid spreading and wetting on a compliant coating, with potential applications in dew collection and free-surface flow control.

Wetting | Soft Materials | Interfacial Science |

Gels are soft complex materials made of a polymeric scaffold in which a liquid may be embedded (1). They can be obtained from a broad catalogue of macromolecules and solvents such as poly(dimethylsiloxane) (PDMS) or water-based solutions of poly(vinyl alcohol) (PVA). In turn, gels offer great flexibility with respect to the tuning of their mechanical and physicochemical properties. These materials find uses in a diverse range of applications, such as cell culture (2), regenerative medicine, drug delivery (3), or the modification of solid surfaces to control friction, wetting and heterogeneous nucleation (4–6). Finally, gels as coatings have attracted renewed interest in recent years due to the opportunity they offer to generalize existing descriptions of the wetting of solids to arbitrary materials (7–19).

Most of the reports dealing with wetting involve a sessile droplet sitting on a solid with a large shear modulus  $\mu_0 \simeq \mathcal{O}(\mathrm{GPa})(20,\,21)$ . In this context, the normal component  $\vec{F}_{cap}^\perp$  of the resulting capillary force at the contact line between the droplet, the rigid substrate and the atmosphere deforms the interfacial region of the solid on molecular scales. As a result, this contribution is neglected in the rationalization of wetting on solids, and the problem is usually described with the Young-Dupré equation. In contrast, gels can be soft enough  $(1 \leq \mu_0 \leq 10^4 \,\mathrm{Pa})$  that  $\vec{F}_{cap}^\perp$  deforms their surface on micrometer scales, much larger than the size of the polymers and of the solvent molecules. Due to progresses in experimental methods,



**Fig. 1.** Sliding of a droplet on a soft layer. (a) A water droplet is deposited on the surface of a glass slide covered with a poly(dimethylsiloxane) layer of thickness  $h_0$ . The glass slide is inclined to the vertical at t=0. (b,c) Motion of  $2\mu$ L droplets on glass slides covered with PDMS layers 9  $\mu$ m and 270  $\mu$ m thick. Labels: time in minutes elapsed since inclination. Scale bar: 1mm.

many recent studies have investigated how the presence of this deformation, known as a ridge, may alter the force balance at the contact line and modify the shape of both the sessile drop and the substrate (13, 14, 16–18, 22).

The presence of a soft coating also modifies wetting dynamics. The velocity V at which a contact line between a solid, a liquid and a gas moves on a rigid substrate coated with a soft elastomeric layer is smaller than the velocity of the contact line on the bare substrate. This velocity reduction, known as viscoelastic braking (23), results from an additional mechanism of energy dissipation due to the viscoelasticity of the gel. As a consequence, the spreading of liquids and the motion

#### Significance Statement

Soft solids such as gels are routinely used to coat solid surfaces to control wetting and adhesion. While wetting statics on these coatings has received a lot of attention, the dynamics has been hardly studied. Here we show that energy dissipation during spreading depends not only on the viscoelasticity of the gel but also on the thickness of the coating. We provide both an analytical model and a scaling law that predict the dynamic contact angle, *i.e.* the signature of dissipation in the droplet-substrate-atmosphere system, as a function of the mechanical properties of the gel and the thickness of the coating. Finally, we show that droplets moving on coatings having thickness gradients drift towards regions of higher dissipation.

M. Z., J. D, T. N., F. L., L. L. and M. R. designed researched; M. Z, performed the experiments; M. Z., J. D. and M. R. analysed the data, M. Z., J. D., F. L. and L. L. developed the model. All authors contributed to the writing of the manuscript

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of droplets are slowed down. Nonetheless, this description of the problem is incomplete: Figures 1b-c demonstrate that the velocity of a water droplet moving down a vertical glass slide coated with a soft viscoelastic silicone layer ( $\mu_0=1.1$  kPa, used for all experiments in this study, see Materials for a rheological characterization) also decreases as the elastomer thickness increases. The purpose of this paper is to uncover the link between substrate thickness and the motion of a contact line.

#### 1. Results

As we want to understand a dynamic wetting observation, we must characterize the reference state, i.e. the statics of wetting. We investigated the properties of the region surrounding the contact line between a sessile droplet and a silicone elastomer layer using both side-view microscopy to measure the apparent contact angle at equilibrium  $\theta_{eq}$  and a non-invasive optical method based on Schlieren photography (24) (Fig. 2a). We find that  $\theta_{eq}$  depends on the contact radius R of the droplet on the elastomer at constant sample thickness  $h_0$  but it is independent of  $h_0$  for constant R (Fig. 2b-c), accounting for measurement uncertainties. For large droplets,  $\theta_{eq}$  tends towards a saturation at a value  $\theta_{\infty} \simeq 106^{\circ}$ . The Schlieren results show that the connection between the ridge and the flat elastomer surface far from the droplet is monotonically decreasing for large thicknesses while the surface profile shows a sub-micrometre surface depression in the vicinity of the ridge at small thicknesses, known as a micro-trough (13, 14). The micro-trough results from the incompressibility of the gel, which has to accommodate the Poisson effect at small thicknesses under the zero-motion constraint at its basis due to its bonding to a rigid substrate. Its depth  $\zeta$  increases with a decrease of  $h_0$  and an increase of the droplet radius R (see Supplementary Materials).

We interpret these observations with an analytical theoretical description of the statics of elasto-wetting in the limit of linear elasticity developed recently by Dervaux and Limat (25) that we extend to account for finite-depth effects (see Supplementary Materials). The model (and the model used in the dynamic context later) considers a rivulet, i.e. a column of liquid whose axis of symmetry is parallel to the surface of the gel, sitting on the surface of the coating. As the angles that we have measured are rather close to  $\frac{\pi}{2}$ , we assume that the interfacial tensions of the solid with the liquid and the gas are identical,  $\gamma_{SL} = \gamma_{SV} = \gamma_s$ , and this quantity is the only fitting parameter. The model is able to capture the surface profiles of the elastomer for all the thicknesses that we investigated (Fig. 2d) as well as the evolution of  $\zeta$  as a function of  $h_0$  and R (see Supplementary Materials). The model indicates that  $\gamma_s = 40$ mN m<sup>-1</sup>, a value comparable to those reported earlier(17, 26). To sum up, the surface profile of the elastomer layer in the vicinity of the contact line depends on layer thickness. In contrast, the apparent equilibrium contact angle  $\theta_{eq}$  is independent of the geometry of the gel layer; it depends only on the surface energies of the materials, a feature that is predicted by the model (See Supplementary Materials). The analysis of the statics of wetting allows us to define the reference state of the study of wetting dynamics as a function of the thickness of the elastomer layer to which we now turn.

Wetting dynamics is characterized by the measurement of the dependence of the difference  $\Delta\theta = \theta_{eq} - \theta_{dyn}$ , the latter

being the dynamic contact angle, on the velocity of the contact line V (21). As  $\theta_{eq} > \frac{\pi}{2}$ , we studied a receding contact line surrounding a droplet with an initial contact radius R >> 1mm that we deflated at a constant flow rate. We recall that  $\theta_{eq}$  reaches its saturation value  $\theta_{\infty}$  for such large droplets. For all thicknesses,  $\Delta\theta$  increases with an increase of V (Fig. 3a). The dynamic contact angle differs by almost 30° from the static equilibrium contact angle for the largest velocities we investigated. The two curves show that  $\Delta\theta$  is also a function of thickness: it grows by 30% with an augmentation of  $h_0$ from 10 to around 100  $\mu$ m (Fig 3b).  $\Delta\theta$  saturates to its velocity-dependent value when  $h_0 > 100 \ \mu \text{m}$ . To interpret these observations, we recall that, in general, large energy dissipation induces large values for  $\Delta\theta$  at fixed contact line velocity (21). In this context, our results show that the thinner the elastomer is, the more energy dissipation is reduced. Voué et al. reported a similar observation for a silicone oil wetting a silicone elastomer (27). As the liquid can permeate through the gel in their system, the wetting dynamics they report are difficult to interpret, especially since the thickness of their elastomer layers is of the order of 10  $\mu$ m. Permeation does not occur in our system. We now unveil how thickness and dissipation are related in elastowetting.

To identify the mechanism relating dissipation to the thickness of the coating, we developed an analytical model based on linear viscoelasticity that is a generalization to large velocities and arbitrary rheology of a framework proposed by Long et al. (10). In this model, dissipation in the solid balances the energy that sets the contact line in motion. We assume that the shear relaxation modulus of the soft layer is described by the Chasset-Thirion model,  $\mu(\omega) = G'(\omega) + iG''(\omega) = \mu_0(1 + (i\omega\tau)^m)$  (10). We find this assumption to be valid for our material with  $\mu_0$  = 1085  $\pm$  124 Pa, m = 0.66  $\pm$  0.04 and  $\tau$  = 15.4  $\pm$  0.4 ms (see Materials). We also use the surface tension of the solid  $\gamma_s = 40 \text{ mN m}^{-1}$  obtained from the statics. The model predicts that the function describing the dependence of the dynamic contact angle on the properties of the substrate and the velocity of the contact line  $G(\theta_{dyn}) = \frac{\cos{(\theta_{eq})} - \cos{(\theta_{dyn})}}{\sin^2{(\theta_{dyn})}}$ is proportional to  $V^m$ , in excellent agreement with the data (Fig. 3a), with an adjusted exponent  $m_{adj} = 0.62 \pm 0.02$  close to the experimental value of  $m = 0.66 \pm 0.04$  we obtained from rheology (inset in Fig. 3a). The full model also captures the dependence of  $G(\theta_{dyn})$  on  $h_0$  at constant velocity of the contact line (Fig. 3b). Finally, we note that we did not observe effects related to the Laplace pressure inside the droplet. Thus the 2D results obtained with the model are valid in 3D (see Supplementary Materials).

We can extract from the model scaling laws that describe the asymptotic behavior of  $G(\theta_{dyn})$  for thick and thin coatings. We assume that the problem is invariant with a translation along the contact line. Our considerations are based on the Fourier transform of the problem. The power per unit length of the contact line injected by the driving capillary force is:

$$\mathcal{P}_{in} \propto \gamma V(\cos\left(\theta_{dyn}\right) - \cos\left(\theta_{eq}\right))$$
 [1]

where  $\gamma$  is the interfacial tension of the liquid-gas interface. Power  $\mathcal{P}_{in}$  is balanced by dissipation in the gel. Using the Chasset-Thirion model, we estimate the power dissipated in the gel per unit volume:

$$p_{dis} \propto \mu_0(\omega \tau)^m \epsilon^2 \omega$$
 [2]

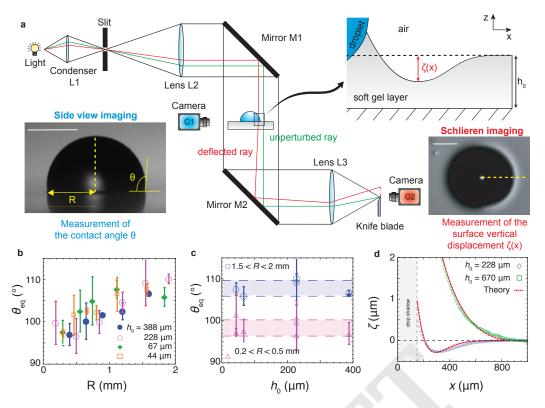


Fig. 2. Experimental set-up and characterization of the static three-phase contact line. (a) The region surrounding the contact line between water, the elastomer and the atmosphere is visualised simultaneously from the side with camera C1 and from the bottom with camera C2 that captures the output of a Schlieren photography set-up. The former allows us to measure the contact angle  $\theta$  and the contact radius between the droplet and the solid R. The latter is used as a surface profilometer along the yellow dashed line. Scale bar on the images: 0.5 mm. (b,c) Dependence of the equilibrium value of the contact angle  $\theta_{eq}$  on R for four different layer thicknesses and on the thicknesse  $h_0$  of the soft layer for sub and supra-millimetric droplets. In the latter graph, the dashed lines represent a spread of one standard deviation around the average value of  $\theta_{eq}$  over all thicknesses. (d) Dependence of the surface position  $\zeta$  on the distance to the contact line for two different thicknesses. The origin of the x axis is arbitrary; we use it to note that part of the contact line is hidden to the Schlieren set-up by the droplet as  $\theta_{eq} > \pi/2$ . Dashed lines: predictions of our model.

where  $\omega$  is a typical frequency that we assume to be of the order of  $\frac{V}{\ell}$ , with  $\ell$  a length scale to be determined. The dissipated power per unit length of the contact line is obtained by integrating Eq. 2 over  $\ell^2$ , as this length scale also corresponds to the typical vertical displacement experienced by the solid as well as the horizontal extent of the moving ridge. Taking  $\epsilon = \frac{\gamma}{\gamma_s} \sin{(\theta_{dyn})}$  as a typical scale for strains:

$$\mathcal{P}_{dis} \propto \mu_0 \left(\frac{V\tau}{\ell}\right)^m V\ell \left(\frac{\gamma}{\gamma_s} \sin\left(\theta_{dyn}\right)\right)^2$$
 [3]

Equating Eqs. 1 and 3, we find that:

$$G(\theta_{dyn}) \propto \frac{\gamma}{\gamma_s^2} \mu_0 \ell \left(\frac{V\tau}{\ell}\right)^m$$
 [4]

The length  $\ell$  is related to the balance between capillarity and elasticity in the system. Its expression depends on whether the coating has a finite thickness or not. We can express  $\ell$  in the limit of thin and thick samples by investigating the asymptotics of the amplitude A of the Fourier transform of the Fourier transform of the surface vertical displacement field  $\tilde{\zeta}(k)$  (see Eq. 20 in the Supplementary Materials):

$$A(k) = \frac{1}{\gamma_s k^2 + \mu_0 K(k)^{-1}}$$
 [5]

where k is the wavenumber and

$$K(k) = (2k)^{-1} \frac{\sinh(2h_0k) - 2h_0k}{2h_0^2k^2 + \cosh(2h_0k) + 1}.$$
 [6]

For large thicknesses,  $h_0 \to \infty$ , and we obtain that  $K(k) \to (2k)^{-1}$ . Then:

$$A(k) \to \frac{1}{\gamma_s k^2} \frac{1}{1 + 2\frac{\mu_0}{\gamma_s k}}.$$
 [7]

The characteristic length scale in Eq. 7 is  $\ell = \frac{\gamma_s}{\mu_0}$ , i.e. the elastocapillary length. Injecting this value of  $\ell$  in Eq. 4, we find the scaling predicted 20 years ago by Long *et al.*(10):

$$G(\theta_{dyn}) \propto \frac{\gamma}{\gamma_s} \left(\frac{\mu_0 V \tau}{\gamma_s}\right)^m$$
. [8]

In the limit of thin coatings, we carry out the Taylor expansion of K(k) around  $h_0 = 0$  and we keep the first non-vanishing term:

$$K(k) \sim h_0^3 k^2 \tag{9}$$

We find that the characteristic length scale is  $\ell = \left(\frac{\gamma_s h_0^3}{\mu_0}\right)^{1/4}$  in this case. Injecting  $\ell$  in Eq. 3, we obtain the following expression for the dissipated power:

$$\mathcal{P}_{diss} \propto \mu_0 \epsilon^2 V(V\tau)^m \left(\frac{\gamma_s h_0^3}{\mu_0}\right)^{(1-m)/4}$$
 [10]

and we obtain a new scaling law reflecting thickness effects on the dynamic contact angle for thin coatings:

$$G(\theta_{dyn}) \propto \frac{\gamma_{LV}}{\gamma_s} \left(\frac{V\tau\mu_0}{\gamma_s}\right)^m \left(\frac{h_0\mu_0}{\gamma_s}\right)^{3(1-m)/4}$$
 [11]

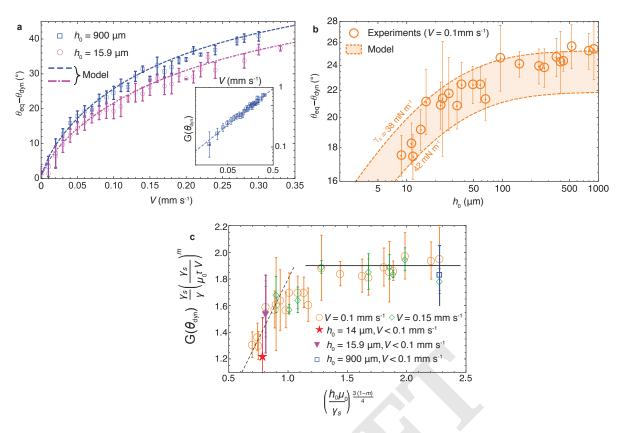


Fig. 3. Characterization of the dynamics of the receding three-phase contact line on soft elastic layers. (a) Dependence of the contact angle difference  $\theta_{eq}-\theta_{dyn}$  on the velocity of the receding contact line on two layers of thickness  $h_0=15$  and 900  $\mu$ m. Dashed lines: predictions based on our model. Inset: log-log plot of the function  $G(\theta_{dyn})$  as a function of the contact-line velocity for the same systems. Dashed line: power-law fit with an exponent  $m=0.62\pm0.02$ . (b) Dependence of  $\theta_{eq}-\theta_{dyn}$  on  $h_0$  at constant receding velocity  $V=0.1~{\rm mm\,s^{-1}}$ . Dashed lines: predictions of our model for two values of the solid surface tension  $\gamma_s$ . (c) Test of Eq. 11, relating the dynamic contact angle to the velocity of the contact line and the thickness of the sample in the small-thickness regime against experimental data. Dashed line: Eq. 11, with  $m=0.62\pm0.02$  and a slope of 2. The horizontal line is a guide for the eyes. (d) Schematic of the motion of a point of the surface of the elastomer (red dot) during the passage of the contact line.Time starts at the top left and ends at the bottom right.

Equation 11 can be tested by rescaling the experimental data as in figure 3c. We see that the data follow Eq. 11 when  $h_0 << \ell_s = \frac{\gamma_s}{\mu_0}$ . When  $h_0 >> \ell_s$ , the data is independent of  $h_0$  and it follows Eq. 8. We are the first to report the validity of Long et al.'s model (10) to the best of our knowledge. Thus, our results show that dissipation during the wetting of a soft coating by a liquid can be engineered in a predictable way by adjusting the thickness of the layer. Finally, the range of thicknesses over which we observe a dependence of dissipation on  $h_0$  corresponds to the typical range of coating thicknesses in applications.

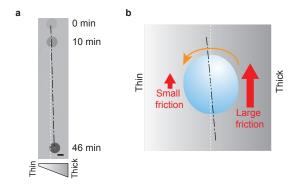
#### 2. Discussion

The prediction that the dynamic contact angle at a contact line moving on a soft coating is closer to the value of the static contact angle as the thickness of the coating decreases may seem counter-intuitive at first. To gain more insight, we shall now compare contact line motion for a rivulet on a gel layer and on a liquid substrate of similar thickness  $h_l$ . In the latter case, continuity of stresses and velocities is enforced at all the interfaces. Momentum diffuses from the droplet to the bulk of the liquid substrate through the droplet-substrate interface and stresses and dissipation do not diverge at the contact line. Energy dissipation occurs in the bulk of both the droplet and the substrate. In the latter, the dissipation power per unit

length along the contact line between the rivulet and the gel scales as  $\mathcal{P}_{vis} \propto \eta V^2 \frac{R_0(t)}{h_l}$  (28), where  $R_0(t)$  is the radius of the spreading rivulet; we assume that  $h_l$  remains constant over time. Thus, dissipation increases with a decrease of  $h_l$  (29, 30). In contrast, at the interface between a gel coating and a liquid, we enforce the no-slip boundary condition. Thus velocities are zero everywhere along this interface, and the contact line is the locus of most dissipation. In this context, we have shown that dissipation follows Eq. 10. As m < 1, dissipation decreases with a decrease of the thickness  $h_0$  of the sample.

We also computed the prediction for the dependence of  $G(\theta_{dyn})$  using another model proposed recently by Karpitschka et al.(26). This model captures well the data for  $\Delta\theta(V)$  at large thickness provided we adjust the surface tension of the solid to a value different from the one we obtain in our study of the statics of wetting. Nonetheless, it does not capture the decreasing trend of  $\Delta\theta$  at small thicknesses (see Supplementary Materials). In their model, Karpitschka et al. impose that capillary forces are at equilibrium at the moving contact line at all times. The failure of their model to capture our data suggests that this constraint is not true anymore for droplets moving on thin coatings. The question of the validity of this assumption at all thicknesses remains open as its answer lies in a second-order analysis of the dynamics of elastowetting

that has yet to be performed.



**Fig. 4.** Drifting of a droplet moving down a vertical glass slide covered with a soft layer of increasing thickness. (a) a  $2\text{-}\mu\text{L}$  droplet moving over a coating with a thickness gradient  $\nabla h \sim 70~\mu\text{m}$  mm $^{-1}$ . Scale bar: 1 mm. (b) Schematic explaining the mechanism leading to droplet drift during forced motion. Black dashed line is the trajectory of the droplet, white dashed line is the vertical. The orange arrow represents the rotation of the droplet as it moves down the coating and experiences differences in dissipation along its contour, in a fashion analogous to a car experiencing braking only on one of its sides.

We take advantage of our findings to control the trajectory of liquids moving at the surface of solid substrates. We coat a glass slide with a soft elastomeric layer with a thickness gradient (Fig. 4a). We place a water droplet at one extremity of the glass slide above the thinnest part of the coating. Then we incline the glass slide to the vertical, the thickness gradient being perpendicular to gravity. The droplet slides down and it drifts from the thinnest regions towards the thickest ones. Drift results from the dependence of dissipation in the gel on  $h_0$ : the droplet experiences a larger braking over the thick part of the coating than other the thin region (Fig. 4b). Thus, the droplet is subject to a torque that leads to a global drift towards the thick region. In our system, drift will be observed if the gradient starts from  $h_0 < 100 \ \mu \text{m}$ . If the thin part of the thickness gradient is thicker than 100  $\mu$ m, we expect no drift, as is observed in the experiments. Finally, we observe no spontaneous motion of droplets when the glass slide remains horizontal for a long time. This fact highlights the difference between our results, whose origin lies in the variation of the dynamic contact angle around the droplet, and those reported by Style et al. (31) concerning droplet durotaxis, which occurs due to changes of the static contact angle along the droplet perimeter.

Our findings opens the possibility to design soft coatings whose mechanical dissipation can be adjusted not only by formulation but also by an appropriate choice of thickness. We envision improvements in applications such as dew collection, where the use of soft layers has been shown to improve the nucleation rate of condensation droplets(5); the use of thin layers will help increase the up-time of the condensation cell. We also anticipate the use of our results to control free-surface flows such as droplet impact or gravity currents passively with thickness-gradient coatings.

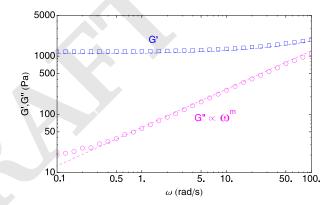
#### **Materials and Methods**

**Gel preparation and characterization.** All the experiments were performed with Millipore-Q purified water ( $\gamma=72~\text{mN/m}$ , ) on polydimethylsiloxane (PDMS) substrates made from the commercial

elastomer Sylgard 527 (Dow Corning, USA). Sylgard 527 is a two-component elastomer kit composed of a base and a cross-linker. The two liquids were mixed at a manufacturer-recommended 1:1 ratio, to respect crosslinking stoichiometry, with a magnetic stirrer for approximately 30 minutes. The mixture is then let to degas under vacuum ( $P \simeq 50$  mbar) for at least one hour. We obtained thin flat PDMS gel films (thickness  $h_0 < 100 \mu \text{m}$ ) by spin-coating directly the degassed mixture on glass slides. Thickness was controlled by tuning the spinning speed from 100 to 3400 RPM. For thick flat films (thickness >  $100 \mu \text{m}$ ), the thickness was tuned by changing the elastomer volume poured onto the glass slide mounted with movable walls.

For the thickness gradient experiment, the Sylgard mixture was poured into a mold containing a titled glass slide. Next, the elastomer was cross-linked at  $65^{\circ}\mathrm{C}$  in an oven for 24 hours. This protocol produced transparent PDMS films strongly bound to glass slides. The thickness of the films was measured with a 3D Profiler (FOCAL 3D Pilot, FOCAL Nanotech, France) operating in white light scanning mode with a precision of the order of a nanometer.

Figure 5 shows the rheological data obtained for Sylgard 527 and measured with a strain-controlled rheometer (Physica MCR 500, Anton Paar, Austria). The elastomer was cross-linked directly between the Peltier plate and the plate tool (PP20-MRD, d=20 mmAnton Paar, Austria) at  $T=65^{\circ}$  C for 4 hours. The sample was then let to cool down to  $25\pm0.2^{\circ}$  C before running the test. The gap was set at 0.6 mm and the strain was fixed at 1%. A fit of the Chasset-Thirion model,  $\mu(\omega)=G'(\omega)+iG''(\omega)=\mu_0(1+(i\omega\tau)^m)$ , to the experimental data provides the parameters for the frequency-dependent mechanical response of the gel. We also characterized



**Fig. 5.** Small-amplitude oscillatory shear measurement of the mechanical properties of the Sylgard 527 elastomer with a strain-controlled rheometer (Physica MCR 500, Anton Paar, Austria). Dashed line: best fit of the Chasset-Thirion model,  $\mu(\omega)=G'(\omega)+iG''(\omega)=\mu_0(1+(i\omega\tau)^m)$ , to the experimental data.  $\mu_0=1085\pm124$  Pa,  $m=0.66\pm0.04$  and  $\tau=15.4\pm0.4$  ms

the contact angle hysteresis of water on Sylgard 527 and we found it to be of the order of  $3\text{-}4^{\circ}$ .

Quantitative Schlieren optics. We built an observation platform for the measurements of out-of-plane deformations of the soft film based on Schlieren optics. A white LED light source (Luxeon, Lumileds, USA) is focused on a mechanical slit (VA100/M, Thor Labs, USA, size  $1 \times 13.6$  mm) by a condenser L1. The light beam from the slit is then collimated by a lens L2 (diameter  $d_{L2} = 25.4$  mm; focal length = 101.6 mm; MPD 149-P01, Thor Labs, USA). The parallel light beam goes through the sample between mirror prisms M1 and M2. It is then collected by a third lens L3 (LB1374-B, Thor Labs, USA). To obtain a Schlieren set-up, a knife blade is placed at the focal point of lens L3 and hides part of the focal region. The blade filters those light beams that have been deviated because of surface deformations of the gel sample due to the presence of the droplet. Images of the filtered focal point are captured with a digital camera (DFK 23UX174, Imaging Source, Germany) equipped with a lens (Avenir TV Zoom Lens F1.8, JAPAN). The CCD sensor of the camera has a dimension of  $1280 \times 960$  pixel, resulting in a lateral spatial resolution of 10.2  $\mu m px^{-1}$ 

In principle, the slit light source is perfectly refocused on the knife blade plane as illustrated by the green light beam drawn in Fig. 2 (a), as long as there is no disturbance on the light path. However, changes in the surface slope of the gel film deflect the light beam and the conjugated image is thus shifted by a distance  $\Delta a$  in the focal plane as illustrated by the red beam shown in Fig. 2 (a). Under the small angle approximation, this distance is related to the deflection  $\epsilon$  of the light beam through the Schlieren object by the formula  $\Delta a = f \epsilon$  where f is the focal length of L3. For an accurate quantification of the surface profile, we carried out a three-step measurement. First, a sequence of pictures was recorded at different vertical blade positions and a map relating the pixel grayscale value to the deflection by the knife cut-off is built. During this step, the Schlieren object is absent. Second, all the camera parameters are kept identical and pictures are taken 5 mins after the drop fully relaxes on the soft substrate (for the static deformation measurement). Then, the grayscale value of the Schlieren picture is compared to the calibration map and we obtain a deflection map. Finally, by combining the deflection relation and Snell's law, the map of the surface slope is reconstructed. Our system has a deflection angle sensitivity of 0.0007 rad, and we could measure angles up to 0.05 rad.

Data acquisition. To study the statics of wetting, we enclosed the sample in a transparent glass chamber  $(75 \times 50 \times 8 \text{mm})$  to control humidity and avoid evaporation. Its lid could easily be removed to place drops. For experiments, a reservoir of water was enclosed together with the PDMS slides to saturate the atmosphere of the glass chamber. All the measurements were performed 5 minutes after deposition. For the spreading dynamic experiments, a side view camera (DMK 23UV024 Imaging Source, Germany, C1 in Fig. 2 a) equipped with a lens (TV Lens 50mm, f1:2.8, Ricoh, Japan) and two tube sets (C Mount TV Lens Extension Tube Set 40 mm, JAPAN), was used. The spatial resolution of C1 was 5.5  $\mu$ m px<sup>-1</sup>. Using a Gaussian subpixel detection technique, the error in the contact line velocity determination is around 10  $\mu m s^{-1}$ All the drops were deposited using a syringe which was connected to a pump (PUMP 33, HARVARD APPARATUS, USA) at room temperature. For the sliding and drifting experiments on tilted slides, droplets were deposited using a micropipette (0.5-10  $\mu$ L, FinnPipette) at ambient temperature. A front view camera (DFK 23UX174, Imaging Source, Germany) equipped with a lens (TV Lens 50mm f1:2.8, Ricoh, Japan) and one tube set (C Mount TV Lens Extension Tube Set 20 mm, JAPAN), with a spatial resolution of 15.9  $\mu$ m px<sup>-1</sup> was used to track the sliding droplets.

**Model.** We use two models, one for the statics and one for the dynamics. The model describing the statics is built in the framework of linear elasticity and we account for the surface tension of the solid. We assume that the system is translation-invariant and we describe a x-y slice of the substrate subjected to the capillary force of a liquid. We model the latter as a concentrated load on an elastic half-space of depth  $h_0$ . We are interested in two situations, the single contact line and the rivulet, i.e. a column of liquid that has two straight contact lines with the substrates. We focus on the far-field vertical displacement for which possible divergences in the displacements at the contact line do not matter (25). We use Fourier transforms to solve the biharmonic equation that describes the problem, and we compare the data with the outcomes of the description of the rivulet. The static model also accounts for the small hysteresis that we measure in experiments in the form of a force tangential to the surface.

The model for the dynamics is built in the framework of linear viscoelasticity. We use the surface tension of the solid deduced from the study of the statics as an input to the model. As experiments indicate that the properties of the motion of the contact line are independent of the size of the droplet, we focus on the case of a single contact line. We compute the dependence of the vertical displacement on the velocity of the contact line using a similar procedure to the statics resorting to Fourier transforms. We then compare the power injected in the system due to capillarity with the dissipated power in the substrate due to its viscoelasticity.

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