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Contact geometry and mechanics OPENpredict friction forces during tactile surface exploration

Marco Janko¹, Michael Wiertlewski ² & Yon Visell ³

When we touch an object, complex frictional forces are produced, aiding us in perceiving surface features that help to identify the object at hand, and also facilitating grasping and manipulation. However, even during controlled tactile exploration, sliding friction forces fuctuate greatly, and it is unclear how they relate to the surface topography or mechanics of contact with the fnger. We investigated the sliding contact between the fnger and diferent relief surfaces, using high-speed video and force measurements. Informed by these experiments, we developed a friction force model that accounts for surface shape and contact mechanical efects, and is able to predict sliding friction forces for diferent surfaces and exploration speeds. We also observed that local regions of disconnection between the fnger and surface develop near high relief features, due to the stifness of the fnger tissues. Every tested surface had regions that were never contacted by the fnger; we refer to these as "tactile blind spots". The results elucidate friction force production during tactile exploration, may aid eforts to connect sensory and motor function of the hand to properties of touched objects, and provide crucial knowledge to inform the rendering of realistic experiences of touch contact in virtual reality.

The tools we touch, grasp and manipulate often have topographical features such as ridges, bumps and texture that are meant to guide our gestures and provide crucial information on their nature and function. As an example, the rubbery curve on a quality pen provides grip and indicates where to position our fngers for optimal manipulation. These features are often recognized by touch alone from the deformation imposed by the skin in contact with the feature. While static shape perception can be attributed to the indentation (i.e. normal deformation) of the skin, the dynamic perception of these features is due to the complex interactions that result from sliding, in particular to the modifcation of frictional forces. Complex frictional forces are involved in both static and sliding contact between the fnger pad and a touched surface. Friction plays an essential role in object grasping and manipulation, interrelating the grip and load forces that are exerted by the fngers and that must be coordinated in order to prevent slipping¹.

Surface curvature, roughness, bumps, and ridges can enhance friction forces, aiding grasping and fne manipulation^{2,3}. Sensing these features is facilitated by dynamic tactile signals from thousands of mechanoreceptive afferents that innervate the finger pads⁴, allowing the brain to identify surface features and select an appropriate grasp. Friction forces are crucial for such acts of discriminatory touch. Sliding our fngers over the surface of an object, we obtain perceptual information about roughness⁵, shape⁶, adhesion⁷, and texture⁸. Local surface features, such as edges, are also reliably encoded in population responses of tactile afferents in the fingertip⁹. Variations in friction forces yield high-frequency, low-amplitude skin deformations that propagate throughout the hand¹⁰, exciting mechanoreceptive afferents within and beyond the finger pad, and further facilitating surface texture discrimination $8,11$.

While the relevance of friction forces to sensory and motor functions of the hand is well established, our understanding of how time-varying friction forces are produced during contact with non-fat surfaces is incomplete. This can partly be attributed to the complex dynamics governing the tribology of the finger pad¹², including the moisture content of the fngertip skin, which can afect the mechanics and dynamics of frictional sliding sliding^{12–14}, static to sliding transitions^{15–17}, and the presence of non-Coulombic behavior¹⁸. However, effects due

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Figure 1. Experimental apparatus and relief surfaces. (**A**) Measurement system, side view. A high speed camera, two-axis force sensor, LED light source, and tactile surface were mounted on an optical bench, and aligned ensuring accurate fronto-parallel imaging of the fnger contour. (**B**) Isometric view of the two-axis force sensor. (**C**) Frontal view of the force measurement device with force decomposition. (**D**) Surface center feature geometric specification. (**E**) Front view illustrating the surfaces; These shapes are replicated for two widths $(W=2 \text{ mm}, 4 \text{ mm})$ and heights $(H=0.2 W)$, yielding six surfaces in total.

to variations in surface geometry are also very difficult to account for. Even very controlled sliding of a finger on non-flat surfaces, such as sinusoidal gratings 1^{9-21} or braille dots²², yields forces that fluctuate in time and that vary greatly between trials. Existing mechanical models are unable to anticipate these forces from surface properties and interaction parameters, such as fnger velocities and applied forces, except for surfaces whose heights vary slowly relative to the size of a finger 2^3 .

It is unclear whether these force fuctuations are attributable to fnger pad dynamics, contact mechanics, or to other tribological efects, but one indication of what is occurring may be the development of complex patterns of contact between the finger and the surface^{15,17,24}, which arise due to the limited elasticity of fingertip tissue. The resulting multi-contact interfaces vary with position and pressure²⁵, and can evolve rapidly in time. The signifcance of these multi-contacts has also been noted in perception studies, where it was observed that they could account for the dominant efect of high relief features on roughness perception during tactile exploration of rectangular gratings²⁴. However, the evolving regions of contact are difficult to capture during tactile exploration, and there has been little prior empirical investigation of the role of contact geometry in shaping forces felt by the fnger.

Tis problem of predicting how dynamic friction forces refect the topography of touched surfaces is also highly relevant to the design of human interactive technologies, especially for the expanding category of surface haptic displays – touch screen displays and interfaces whose friction coefficients can be electronically controlled in order to provide tactile feedback via ultrasound^{26–28} or electrostatic adhesion^{29–31}. Existing simulation methods are based on reproducing either pre-recorded or synthetically generated friction forces^{32–34}, yielding feedback that is either rigidly specifed or unphysical and artifcial, unlike those that would be produced during sliding on any real surface. Tus, here too, there is a need for a greater understanding of friction forces produced during tactile exploration of non-flat surfaces.

Several challenges are encountered in modeling the mechanics of fnger-surface interactions, including the complex dynamics of finger pad friction¹², the nonlinear stiffness and viscoelastic characteristics of the tissues^{35,36}. Analytical modeling of the fngertip has yielded useful insight for quasi-static or dynamic indentation problems35–38, but the results have limited utility for understanding dynamic deformation of fnger tissues induced during frictional sliding. Of the many studies of frictional properties of the primate fngerpad, few have addressed sliding contact on realistic surfaces. Volumetric numerical simulations of fngertip sliding on diferent surfaces can predict a restricted range of observed phenomena³⁹⁻⁴¹, such as the development of nonlinear oscillations, but fail to capture the variability of sliding on real surfaces⁴², and are strongly simulation dependent – i.e., they cannot predict force production from surfaces that have not been simulated. Local surface descriptors, such as contact orientation of the surface at a contact point^{6,43,44}, have been proposed in order to account for the effect of surface shape on force production, but have not been applied to distributed contact with the fnger pad, where the slope may vary within a single contact region, and may not be defned at locations where contact is broken.

In order to account for friction force production between the fnger pad and non-fat surfaces, we measured forces and contact interfaces during frictional sliding on relief surfaces, using synchronized force measurements and a fronto-parallel high-speed video capture confguration. Using image processing methods, we tracked the geometry of contact interfaces from the latter, and used the resulting data to estimate parameters of a spatially distributed frictional model. As we show, this model, which accounts for surface shape and contact mechanical efects, could accurately predict forces that were felt during sliding on a variety of surfaces at diferent speeds.

Results

We captured forces, fnger location and deformation during sliding contact of the index fnger of two individuals on relief surfaces at three diferent speeds (40, 80, or 120 mm/s), using force sensing, high speed video capture (Fig. 1A–E), and video analysis (see Methods). We used image analysis to track the center of the fnger, *x*(*t*), during sliding (see Methods). The six surfaces were macroscopically flat with localized features in one of three shapes (a bump, a rising step, or a falling step) and either of two scales (width 2mm or 4mm).

Figure 2. Example patterns of fnger-surface contact, extracted via high-speed video capture and analysis (see Methods), at three successive instants (**A**–**C**) for each of the six relief surfaces used in the study, exemplifying three different contact conditions. At initial finger contact (A) with the relief feature, the contact interface C_1 of the fnger with the fat region to the lef of the relief feature is large, and (**B**) decreases as the fnger traverses the inclined region, and becomes supported on the relief feature, where a second disconnected contact interface C_2 , develops. For the bump surfaces, this is followed by the development of a third disconnected contact region, C_3 , which forms as the fnger descends a second slanted region.

Figure 3. Normalized proportion of time of contact between the fnger and surface locations (both subjects, all trials). At the highest level on the scale (1, white), the fnger was in contact with the surface for the highest proportion of time, while at the lowest level on the scale (0, black), the surface was never contacted by the fnger. In each trial, fnite width regions of every surface satisfed this last condition (two regions in the case of bumps); we refer to them as "tactile blind spots". Their widths ranged from 0.47 mm (Step Up surface, 2 mm scale) to 1.71mm (Right Bump surface, 4mm scale).

The contact interface between the finger and surface varied with the time-dependent position of the finger (Fig. 2). Initially, finger-surface contact was restricted to a flat region C_1 to the left of the relief feature. As the finger came into contact with the curved region, additional, disconnected contacts developed, due to fnite stifness of finger tissues, which limit its ability to conform to curved areas of the surface. A second contact, C_2 , disconnected from the frst, thus developed as the fnger ascended (Bump and Step Up surfaces) or descended (Step Down surfaces) the incline. For the bump surfaces, this was followed by the development of a third disconnected contact region, C_3 , which formed as the finger descended the raised feature. The disconnections were maintained at all times, for every trial, sliding speed, and subject. Consequently, despite the slow change in height of these surfaces, whose maximum slope angle did not exceed 0.56 radians (32degrees), there were convex regions of each, nearest to the curved region, that were never contacted by the fnger of either participant (shaded black in Fig. 3). We refer these as "tactile blind spots", because they remained "invisible" to touch contact in all trials. Teir widths ranged from 0.47mm (Step Up surface, 2 mm scale) to 2.6mm (Bump surface, 4mm scale). While the smallest bumps created a maximum of three simultaneous contacts, all other surfaces yielded no more than two at any time.

We computed the lengths L_i of each contact region C_i in each frame, as measured via fronto-parallel video, in order to characterize how they evolved during sliding. The resulting patterns were highly stereotyped for each surface (Fig. S1), varying little between speeds. There were modest differences between subjects (Table S1), likely due to variations in fnger shape or stifness. As the fnger traversed the curved section of the surface, the lengths of the first contour (C_1) and last $(C_2$ or C_3), on or off of the curved region, increased or decreased in nearly identical fashion on every trial for each surface, remaining restricted to a tightly bounded range (Fig. S1), and further highlighting the relatively invariant relation between surface shape and fnger-surface contact evolution.

Figure 4. Measured forces $F_T(x)$ grouped by surface and subject (15 trials per case). Mean of 5 trials at 40 mm/s in red, mean of 5 slides at 80mm/s in blue and mean of 5 trials at 120mm/s in orange. Shaded regions: 1 standard deviation. The forces at each speed are offset by 25 mN to aid readability. (For a superimposed force plot, see Supplementary Information, Fig. S6).

Figure 5. Contact between the finger and surface yields distinct regions of contact (lengths L_i), and local surface stresses σ_p and σ_r normal and tangential to the surface at a point within each region. Variations in surface height $h(x)$ are accompanied by changes in slope $\alpha(x)$, with $dh(x)/dx = \tan \alpha(x)$.

Force components $F_T(x)$ tangent to the surface fluctuated from trial to trial (Fig. 4), but yielded spatial variations in force that varied consistently with the geometry of the touched surface, with some inter-subject diferences. Raised features produced sharp increases in force magnitude, the latter rising on the order of 50 mN within about 1mm, at locations when the fnger came in contact with the surface feature in the region of positive surface slope, $dh/dx > 0$, where $h(x)$ is the surface height function. High speed video analysis revealed the increases in force to coincide with the compression of the leading side of the fnger as it contacted the ascending slope of the surface (Fig. 2A). Tere were typically small but consistent transient decreases in force (approximately 5 to 20mN), followed by increases (5 to 50mN), as the fnger descended incline features, possibly due to a transient increase in normal force, and contact area, of the fngerpad against the recessed fat region (Fig. 2C). Forces were similar at diferent sliding speeds (40, 80, and 120 mm/s), with slightly higher variability at higher speeds, indicating that hysteresis friction played only a modest role. Consequently, in modeling force production, we omitted any explicit dependence on sliding speed.

Modeling frictional forces. To clarify the relation between contact geometry and force production during tactile exploration of relief surfaces, we developed a simplifed analytical model that could be used for comparison with our measurements. As we demonstrate, this model agrees with the observed spatial variations in friction forces. It accounts for two main features of fnger-surface interactions, namely the geometry of contact between the skin and surface, and the development of frictional forces at the boundary between the two. Within the two-term non-interacting model, the frictional force *F* can be assumed to result from the sum of interfacial *Fint* and deformation F_{def} components^{45,46}

$$
F = F_{int} + F_{def} \tag{1}
$$

The interfacial component F_{int} corresponds to the energy dissipated in the breaking of molecular bonds formed transiently between the sliding surfaces due to short range attractive forces. For rough surface contacts, the applied load per unit area $\sigma_p(x)$ acts in the direction normal to the surface at *x*, while the homogenized interfacial stress $\sigma_r(x)$ is tangent to the surface in the projection plane; the angle of the surface with the horizontal is $\alpha(x)$ (see Fig. 5). The normal and tangential stress are related by a Coulomb-Amonton term $\sigma_r = \mu \sigma_p$, with a coefficient of friction μ that is independent of the applied load. While this assumption is common, with some theoretical and empirical support $46,47$, it has some limitations (see Discussion). The net interfacial force in the direction of motion is obtained by projecting the local stress components onto the *x*-direction and integrating over the contact area, A . This yields

Figure 6. Comparing measured and modeled frictional forces. Measured forces $F_T(x)$, mean of 15 trials in each condition (black). Model estimates $\overline{F}_T(x)$, mean of 15 trials in each condition (blue). Shaded regions: 1 standard deviation. Inset provides further detail for the Step Down surfaces, which elicited smaller forces.

$$
F_{int} = \int_{\mathcal{A}} \sigma_p(x) \left\{ \sin(\alpha(x)) + \mu \cos(\alpha(x)) \right\} dx dz \tag{2}
$$

The deformation component *F_{def}* is due to the energy dissipated by subsurface viscoelastic deformation near the leading edge of the contact. As refected in our measurements (Fig. 4), this is largest when there is interlocking between the contacting bodies, corresponding to a raised surface feature, and yields a local friction force component proportional to the product of the net vertical stress σ _{*y*} and the tangent of the angle α (*x*) of the surface with the horizontal²⁵, which is also the slope, tan $\alpha(x) = dh(x)/dx$, where $h(x)$ is the height function (see Fig. 5). We integrated local deformation stresses across the contact interface, and projected these onto the *x*-direction, since this yielded the best ft to our observations, yielding a force term of the form

$$
F_{def} \propto \int_{\mathcal{A}} \frac{dh(x)}{dx} \left\{ \sin(\alpha(x)) + \mu \cos(\alpha(x)) \right\} dx dz \tag{3}
$$

To account for hyperelastic tissue deformation, we investigated nonlinear dependencies on slope *dh*(*x*)/*dx*, but these did not yield better agreement with our observations (see Fig. S2, Supplementary Information). For simplicity, we modeled normal stress $\sigma_p(x)$ as a constant pressure p_0 , drawing on an analogy with a fluid-filled membrane in equilibrium that has proven useful in prior fingertip models³⁶. Since the relative importance of interfacial and deformation components is unknown, we introduce a constant weight, p_1 , for the latter. To match our fronto-parallel measurement confguration, we interpret the area integral as spanning an efective contact length, i.e. as a sum of integrals over N_c disconnected contact regions \mathcal{C}_i of lengths L_i with effective size D in the *z* direction. For simplicity, we assume *D* to be a constant and, without loss of generality, absorb it into the parameters p_0 , p_1 introduced above. Putting these together, we obtain

$$
F = F_{int} + F_{def} = \sum_{i=1}^{N_C} \int_{\mathcal{C}_i} \left(p_0 + p_1 \frac{dh(x)}{dx} \right) \left\{ \sin(\alpha(x)) + \mu \cos(\alpha(x)) \right\} dx, \tag{4}
$$

We sought to evaluate this explanatory mechanical model by comparing it with the observations from our experiments. The height functions $h(x)$ of our surfaces are known, as are the slope $dh(x)/dx$ and angle $\alpha(x) = \text{atan}(dh/dx)$ (see Methods). The contact surfaces \mathcal{C}_i are obtained from the video analysis. Because the participants explored the surfaces with their fingers unconstrained, the pressure, represented by p_0 , and contact conditions, which involve p_1 , could vary from trial to trial. We estimated them from the measurements in each trial (see Methods). We also estimated a constant value of the friction coefficient μ for each subject, because we took measures to control friction during the experiment.

Tis yielded model predictions for friction forces patterns that varied with fnger position, and approximately matched the measured forces in all conditions, for both subjects (Fig. 6). The highest force values, produced during the initial contact phase between the fnger and surface, were overestimated for the coarsest surfaces, but at other instants, and in all other conditions, the model predictions $\hat{F}(x)$ were within one standard deviation of the measured values $F(x)$. The modeled forces exhibited fluctuations, due to variations in finger-surface contact, that were similar in scale to the measurements (Fig. 6, shaded regions). A Pearson correlation analysis between modeled and measured force patterns yielded values *ρ* that were greater than 0.75 in all but one subcondition, accounting for all 180 trials (Table 1). Results were similar at all three speeds tested. Correlations were lowest for the Bump surfaces (both spatial scales), refecting the variable quality of the correspondence between model and measurements (Fig. 6).

Table 1. Measured and estimated forces, F_T and \hat{F}_T , were significantly similar in all conditions (Pearson's correlation value, *ρ*), for both surface heights, both subjects. Data from all three speeds is grouped.

Discussion

We investigated the frictional sliding of the fngertip against relief surfaces, using high-speed video and force measurements, and analyzed the results with the aid of a friction force model that accounted for interfacial and deformation efects. We observed that the forces vary greatly from one sliding interaction to the next (Fig. 4), with patterns that do not bear an obvious relation to the surface geometry. This is consistent with previous measurements of sliding contact forces between a fingertip and high relief features in sinusoidal gratings²⁰ or braille dots²². The spatial patterns of the forces that we observed depended on the shape of the surface, and to a lesser degree on the speed of sliding.

By tracking the finger and surface using high speed video, we observed the spatiotemporal pattern of finger-surface contact during sliding. The geometry of contact evolved during the course of motion in a manner that was highly stereotyped between trials (Fig. S1). Multiple regions of contact developed between fnger and surface near surface concavities (Fig. 2), due to the stifness of the fnger tissues. Despite the modest curvature of the surfaces, there were regions of each that were never contacted by the fnger of either subject (Fig. 3). Tese regions were comparable in size to the surface relief features themselves. We refer to these as "tactile blind spots", because they represented surface regions that could not be felt by the fnger during sliding, due to the absence of contact. Although such regions do not directly contribute to friction, it can be concluded that they afect the distributions of frictional forces at the fngertip. From a perceptual standpoint, Lederman and Taylor previously remarked upon the relevance of contact gaps for the perception of roughness during the tactile exploration of high relief surface textures^{24,48}, although the mechanical explanation for this remained largely unexplored. While subjects may or may not be consciously aware of these contact gaps (and we suspect not), their presence undoubtedly afects both friction and stress distributions in the fngerpad. As such, they may signifcantly afect the tactile perception of relief surfaces. For example, a gap in contact could increase stress gradients, thereby augmenting the perception of surface features. Future investigations of the saliency of contact gaps to mechanics and perception could help to elucidate these issues. From an applications standpoint, it is also noteworthy that existing surface haptic displays are unable to reproduce such features.

We used the tracked contact interfaces and force measurements to investigate the geometric and mechanical origin of sliding friction forces, by developing a model (Eq. 4) that accounts for distributed fnger-surface contact, and contributions of interfacial and deformation contact stresses. While deterministic, the model reproduces force fluctuations due to variations in finger-surface contact, including at multi-contact interfaces. The measurements indicated that even small changes in contact, such as at the leading edge of a raised feature, could elicit large changes in friction forces. Modeled force fuctuations were also similar in scale to the measurements.

Friction forces in the experiment varied with the shape and scale of the surfaces. Initial contact with highest features elicited the largest forces. During contact with surface features, the force variations extended over a length scale of about 10mm (Fig. 4), considerably larger than that of the surface features (2mm and 4mm), due to the fnite size of the fnger. While the fnger-surface contact areas were very consistent from trial to trial, force variations were much larger, with standard deviations reaching 50% of the force magnitude (Fig. 4). Forces produced during sliding on the Step Down surfaces were only about 20% as large in magnitude as the Step Up surfaces, which were geometrically identical, but reversed in spatial orientation. This difference can be attributed to the absence of deformation forces produced via interlocking between the fnger and raised features when traversing the Step Down surfaces. This interpretation appears to be consistent with the predictions of the model, which suggested that deformation forces F_{det} were almost an order of magnitude larger than interfacial forces F_{int} during contact with raised features (Step Up, Bump), and were negligibly small for the Step Down surfaces (see Fig. S3).

While the model was generally able to capture the observed force patterns (Fig. 6), the largest discrepancies were observed at initial contact with the highest raised features. These large forces could be due to unmodeled efects, such as static to sliding contact transitions, although these cannot be reliably identifed using our apparatus. They are not captured in the proposed model, but would be important in a complete account of tactile exploration at low speeds. Among other possible sources of discrepancies of the model predictions with the measurements are mechanical efects of sliding friction that are not accounted for, and uncontrolled aspects of the sliding interactions and stimuli. At low forces, the coefficient of friction μ between the finger and a counter surface varies with normal force¹⁸. For the magnitudes of normal force studied here (approx. (0.3 N) , the rate of change of μ appears to be slow – less than 20% change if the force is doubled or halved – but could affect the agreement with the measurements, and (more importantly) the applicability of the model to a range of force levels.

In the experiment, the fnger was unconstrained, and participants were trained to perform sliding at specifed speeds and normal forces. While this ensured that the interactions more accurately refected those that occur during natural tactile exploration, it also meant that the speed and applied forces varied (Fig. S5), due to diferences in motor performance and to forces produced during interactions between the fnger and relief features.

While this could affect the forces, our measurements did not reveal any consistent effect of sliding speed (Fig. 4). Indeed, during contact with relief features, the modeling results suggest that the forces were dominated by fngertip deformation (Fig. S3). In addition, although we controlled moisture during the experiment, this model does not account for the hydration of the fngertip. Moisture can greatly afect the compliance of the outer layers of the skin, consequently increasing friction forces, and can also introduce additional time-scales to the dynamics^{12,18,35}. Under large displacements, the fingerpad exhibits hyperelasticity $40,49$ which is also not accounted for here, nor is the variation of tissue stiffness with finger orientation⁵⁰, or viscoelasticity⁵⁰. The ambiguous effect of sliding speed on these results call into question whether viscoelastic efects contributed signifcantly, and may be consistent with previously reported results^{51,52}. Future investigations may be needed in order to more fully account for such efects. Finally, while these results are based on experiments that involving many measurements (on the order of 300,000 images, for example), the data were collected from only two participants interacting with three surfaces at three speeds. Future studies aimed at assessing these results across conditions and individuals is warranted in order to assess the generalizability of these observations.

The results of this study contribute toward our understanding of sliding friction forces during tactile exploration. They may aid efforts to understand sensory and motor functions of the hand, and to relate them to properties of touched objects. Finally, predictive modeling of contact interaction forces is needed in order to inform the rendering of realistic experiences of touch contact in virtual reality.

Methods

Measurement apparatus. The force sensor, a custom two axis load cell, consisted of an aluminum tray resting on two piezoelectric transducers (Model 9712A5, Kistler Instruments, Winterthur, Switzerland; resolution $45 \mu m$) through hemispherical contact buttons. The piezoelectric transducers were fixed to a base at 45° angles with respect to the horizontal plane. The tray was constrained in all but the horizontal and vertical directions through a compliant mechanism of fexures with stifness negligible compared to that of the piezoelectric transducers, see Fig. 1B,C. Signals from the piezoelectric transducers made it possible to compute normal F_N and tangential force F_T on the tray via simple trigonometry, i.e. $F_N = (F_1 + F_2)/\sqrt{2}$ and $F_T = (F_1 - F_2)/\sqrt{2}$. The tray was 120 mm long and 25 mm wide. The force measurement device was machined from 6010 aluminum alloy using Electrical Discharge Machining (EDM), the flexures were constructed from 0.25 mm type 1095 spring steel. The device had a nearly flat frequency response from 15 to 500 Hz. The piezoelectric transducers were powered using a compliant power supply (Model 5134, Kistler Instruments, Winterthur, Switzerland), and signals from each sensor were conditioned and digitized with a 0.1ms sample period (10 kHz sample rate) and 16bits quantization using a data acquisition hardware (NI-6229, National Instruments Inc., Austin, TX). Three zero-phase notch filters centered around 50Hz, 150Hz and 300Hz, were used to remove power supply interference.

Fronto-parallel video of the fnger was captured using a high-speed camera (Phantom Miro M110, Vision Research Inc., Wayne, NJ), with 1 ms sample period (1 kHz sample rate) and resolution 1280 \times 720 pixels. The camera position centered on the interaction region, yielding an effective spatial resolution of 0.01 mm. Illumination was provided by a 40W LED light source (No. 5 LED, HS Vision GmbH, Ettlingen, Germany) generating a light beam that covered the entire aperture of the lens of the camera, and yielding high-speed video that consisted of the background (bright light), the contour of the surface profle (dark area) and the shaded area caused by the fngertip obstructing the light. Tis confguration (Fig. 1A) made it possible to accurately track the fngertip contour and its evolution during the sliding contact.

Relief surfaces. We fabricated solid relief surfaces by machining rectangular (120 mm \times 25 mm) aluminum plates with localized relief features. We modeled the samples parametrically and fabricated them from 6061 aluminum alloy using electrical discharge machining (Model HS-3100, Brother Intl.; Accuracy and roughness approx. 1*μ*m) yielding a macroscopically artifact-free fnish. We included three types of relief features, located at the center of otherwise fat surface samples: Bump, a Step Up edge, and a Step Down edge, see Fig. 1D,E. Two widths for each relief feature were used, 2 mm and 4 mm, yielding a total of 6 surfaces. All shapes had a raised cosine profle *h*(*x*), with heights *H* equal to 20% of their widths *W*.

Procedure. The experiments were approved by the institutional review boards of both Drexel University and Aix Marseille University, with the informed consent of all participants (experiments 1610004894 and RO-2016/36). All methods were performed in accordance with the relevant guidelines and regulations. We captured video and force data as each of two participants (male, ages 29 and 35) slid their index fnger on the surface samples (Fig. 1D,E) five times. Sliding was performed at each of three speeds (40 mm/s, 80 mm/s and 120 mm/s) for a total of 15 trials per surface per person, and 180 trials in total. Sliding speed was assessed afer the experiment (Fig. S5). The sliding direction was always the same, from the participants left to right side. The normal force was prescribed to be 0.3 N, a value that is typical of forces observed in prior studies of tactile exploration⁵³. Participants were trained to produce this force level prior to the experiment, using a force sensor, and continued until they were able to maintain the force within a range ± 50 mN. An audio metronome aided them in maintaining the prescribed average sliding speed in each trial. In the experiment, the fnger was otherwise unconstrained (Fig. 1A), in order to ensure that the interactions more accurately refected those that occur during natural tactile exploration, In order to maintain a constant level of friction, the fngers were cleaned with isopropyl alcohol before each recording trial. The surfaces were treated with a small amount of talc to reduce stick-slip effects.

Image analysis. The high speed video data for each trial consisted of a sequence of grayscale images – video frames sampled in time. Each image had a resolution of 1280×720 pixels. They were cropped after recording to the center 1260×360 pixels to remove unwanted background and then were converted to binary images using a threshold operation. Each pixel in the resulting binary images represented either background illumination (1) or

Figure 7. Image analysis procedure (illustrative, 4mm sinusoidal bump). (**A**,**B**) First frame of the video sequence without (**A**) and with (**B**) fnger, used to calibrate the length scale, identify the surface geometry, and initiate the finger contour tracking. (C,D) Tracked finger contour C_f enclosing area *A*, as the finger contacts the surface feature.

shade from the fnger and/or surface profle (0). We determined the size of one pixel in the image plane (which was approximately 23 μ m) from both the maximum height \hat{h}_0 in pixels, of the observed shape \hat{h} and the known height of the surface *H* expressed in mm, leading to a precise calibration of the spatial dimensions, see Fig. 7A.

We processed the binary images in order to extract outlines of the fnger and surface profle, by using the Moore-Neighbor tracing image boundary detection algorithm modifed by Jacob's stopping criteria54, implemented in custom software using an image processing library (Matlab, The Mathworks Inc., Natick, MA). We extracted curves that delineated the contours representing the fingertip, surface, and the contact interface between them, and tracked their motion over time.

The initial surface profile contour, a geometric curve \hat{h} , was extracted from the first frame of the video, before the fnger contacted the surface feature (e.g., Fig. 7B). To aid the tracking of fnger contours, we modeled the tissue as approximately incompressible, so that a deformation from interaction with the surface profle caused tissue to be displaced to other regions. We computed the area *A* (Fig. 7B) enclosed by the fngertip contour, and intersected it with a horizontal line located 0.4mm above the maximum height of the surface feature. For each subsequent frame, the region of interest from the fingertip was obtained as a closed geometric curve C_f resulting from the intersection between the fngertip contour and a horizontal line that was selected to ensure that the area enclosed was equal to *A* (Fig. 7C,D). The position $x(t)$ of the fingertip was estimated as the *x* coordinate of the centroid of the area enclosed by the curve C_f of every video frame.

Within each frame, the region of interfacial contact between the fngertip and the surface was estimated from the contours of the finger, C_p and the surface, \hat{h} . For the purpose of determining the contact interface, a pair of points separated by a distance of 3pixels (i.e. 70*μ*m) or less was considered to be in contact.

Force data processing. The raw signals from the piezoelectric transducers were downsampled and filtered to match the video sampling period of 0.1 ms. The tangential force $F_T(t)$ low-pass filtered using a zero-phase filter with 500 Hz cutoff frequency. The piezoelectric sensors have a high pass filtering characteristic with approximately 5 Hz cutoff frequency, and the measurements drift slightly during data collection. For each slide, lasting no more than 0.4 s, the drif could be approximated by a linear trend. We removed this artifact by computing the best linear ft to the force signal during the fat segment, and subtracted it from the force measured during the touch interaction. Then the force signals were truncated to the time period corresponding to the finger sliding at a 26mm window centered at the middle of the surface matching the location of the relief feature. We sampled spatial positions with a resolution of 0.01 mm. The estimates of fingertip position $x(t)$ were used to infer spatial patterns of the temporal signals, using the same procedure employed in our prior work^{20,55}, associating the force F_T to the position *x* at the corresponding time *t*.

Estimating model parameters. We computed the model forces by computing the integrals in Equation (4) using the contact region C_i extracted from each trial. Per-subject coefficients of friction μ were estimated from the data for each subject via grid search, yielding *μ*=0.55 and 0.6 for subjects 1 and 2 respectively. We fltered the force estimates using a median flter to remove quantization artifacts that were due to pixelation and illumination errors in the tracked fnger contours.

The parameters p_0 , p_1 of the force model were estimated via a simplex direct search method⁵⁶, which minimized the normalized mean square error $\varepsilon(F_T, \hat{F}_T)$ (NMSE) between the measured force F_T and the model estimate \hat{F}_T , where

$$
\varepsilon(F_T, \hat{F}_T) = \frac{\|\hat{F}_T - F_T\|}{\|\hat{F}_T - \overline{\hat{F}}_T\|}, \quad \overline{\hat{F}}_T = \frac{1}{L} \sum_{x=1}^L \hat{F}_T(x) \tag{5}
$$

The parameters could vary from trial-to-trial in order to accommodate variations in contact force. To compare the observations and model predictions in different conditions, we computed Pearson's correlation coefficient values, *ρ*, for the ensemble of measurements in each condition, and the *p*-values for testing the hypothesis that there is no relationship between the observed and measured forces. All correlations were computed on detrended data.

Data availability. The primary data may be retrieved from the following internet address: [https://s3.ama](https://s3.amazonaws.com/rtlab/MJ-MW-YV-2017-Data.zip)[zonaws.com/rtlab/MJ-MW-YV-2017-Data.zip](https://s3.amazonaws.com/rtlab/MJ-MW-YV-2017-Data.zip).

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Author Contributions

M.J. developed the apparatus, carried out the measurements, analyzed and modeled the data, prepared fgures, and drafed the manuscript. M.W. modeled the data, prepared fgures, edited the manuscript, and supervised the research. Y.V. analyzed and modeled data, edited the manuscript, and supervised the research. All authors conceived of the study and designed the study. All authors gave fnal approval for publication.

Additional Information

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