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Design of a Wooden Five-bar Mechanism

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Abstract

Eco-design of robots is a field of research which has been rarely explored in the past. In order to considerably decrease the environmental impact of robot during the design phase, metal or carbon composite parts can be replaced by bio-sourced materials, such as wood. Indeed, wood has interesting mechanical properties, but its performance / dimensions will vary with the atmospheric conditions / external solicitations and with the conditions in which trees have grown. This paper deals with the design of a stiff and accurate wooden five-bar mechanism. First, a control-based design problem is formulated. This problem aims at finding the optimal parameters of the robot, taking into account the nature of the desired control (sensor-based control). Then, reliable topology optimization methodology is proposed, taking into account the variability of the wood performance and that will allow the definition of robot architectures (shape of the links) for which the impact of this variability in terms of deformation is minimal. Finally, the optimal design variables are given and are used for the realisation of industrial prototype of a wooden five-bar mechanism.

1 Introduction

The Climate Change Mitigation (CCM) has become a priority in the world, as shown during the COP21. It becomes urgent to decrease the Environmental Impact (EI) of the human activities. Robotics, which takes an increasing place in our everyday life, must also be part of this effort. A recent study (Fiz) shows that, contrary to what could be imagined, a large percentage (around 50 %) of the EI during the entire robot life is due to the use of metallic materials (steel or aluminum, even if they are recycled) for the design of their links. The percentage is even worth if carbon composites are used. The rest of the EI is globally due to the robot energy consumption during use. A method to reduce EI of robot design, is to replace the polluting metals and carbon composites with bio-sourced materials (BSM) that have a little (even no) EI (Thakur (2013)). Moreover, BSM have

good stiffness performance, and their quantity is only limited by their growing time (Kretschmann (2010)). In this work, developed in the scope of the French project RobEcolo (Rob), we are interested in using wood instead of metals to design the robot links.

Using wood in machines is not a new idea. For instance, it is used in chassis of cars (Mor). It found also applications in the design of buildings due to its stiffness and low cost (Kretschmann (2010)). It is also used in Robotics for the design of mock-ups and proof-of-concept prototypes (Quigley et al. (2011)). However, a detailed study of these works shows that wood is never used in critical parts ensuring accuracy. Indeed, the wood performance / dimensions will vary with the atmospheric conditions / external solicitations and with the conditions they have grown (Kretschmann (2010)); thus, new robot design issues appear: How to be sure that an industrial robot made with wood can be accurate and stiff even if wood properties vary? A first attempt to introduce wood in industrial robot design was presented in Laurent et al. (2010). The results showed that the approach was valid enough to compete with usual materials. However, this study did not deal with the aforementioned issues.

In this paper we introduce a design methodology leading to the manufacture of a stiff and accurate wooden five-bar mechanism. This methodology is based on two optimization problems that can be solved in cascade. First, control-based design approach is used. This approach allows to take into account the specificities of the controller used to servo the robot motions, and aims at finding the length of the links of the five-bar mechanism. Then the shape of the robot links using the reliable topology optimisation is developed. This problem is subject to a set of constraints related to the elastostatic and dynamic performance of the robot. Finally, the optimal length and shape of the wooden five-bar mechanism links are given and are used for the development of an prototype.

2 Design Process

The planar five-bar mechanism is shown in Fig. 1a. It is a two degrees-of-freedom (*dof*) parallel robot able to achieve two translations in the plane ($\mathbf{x}_0\mathbf{Oy}_0$). The end-effector is located at point C . The mechanism, is composed of five revolute joints, two of them located at point A_1 and A_2 are active while the others are passive. The links at points A_i and B_i ($i = 1,2$) are called the proximal links and between points B_i and C the distal links. In this work, our objective is to design a wooden five-bar mechanism, the specifications that the robot should satisfy are summed up in Tab. 1. The robot should be as compact as possible due to some industrial constraints. Moreover, in order to ensure the accuracy of the robot even of the dimensions of the wooden links may vary with the atmospheric conditions, sensor-based controllers will be used. The most common approach consist of the direct

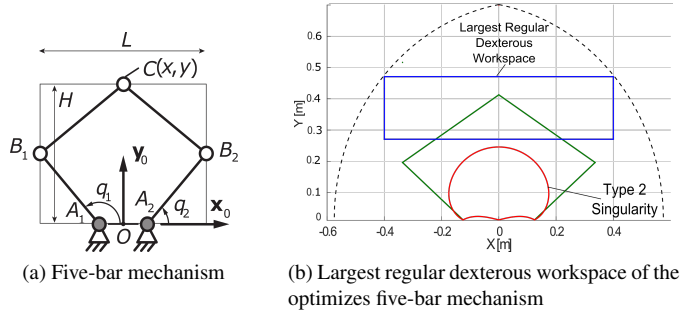


Figure 1: Five-bar mechanism and the optimized one

Table 1: Specifications for the Wooden Five-bar Mechanism

Absolute Positioning accuracy e_{max}	0.5 mm
Regular workspace size	800 mm \times 200 mm
Deformations under forces $\mathbf{f}_1 = [50\text{ N } 0\text{ N } 1\text{ Nm}]^T$ and $\mathbf{f}_2 = [0\text{ N } 50\text{ N } 1\text{ Nm}]^T$	0.5 mm 0.25 mrad

observation of the end-effector pose. In some cases however it may be difficult or unwise to observe the end-effector of the robot, e.g. in the case of a five-bar mechanism the end-effector is a point. For this reason we decided to define a controller based on the observation of the distal links, as in Andreff et al. (2007). However, as shown in Briot et al. (2015) this controller is not free of singularities, that effect its accuracy performance which that not only depend on the camera localisation and specifications but also on the robot dimensions as shown in Kaci et al. (2016). Thus the optimization process must take all this parameters into account.

2.1 Link Length Optimization

The first design optimization problem aims at finding the optimal lengths of the robot links that minimize the footprint size of the five-bar mechanism in the plane of motion for a prescribed regular dexterous workspace, taking into account the controller accuracy performance. Four cameras with 1920×1200 pixels resolution are supposed to observe the two links attached to end-effector to control the robot end-effector pose. We introduced in Kaci et al. (2016) a procedure to define the error model between a single camera and the end-effector and these to define the robot optimal design variables. This method is adapted to the case of four cameras to optimally design a five-bar mechanism. Based on Kaci et al. (2016), we solved

Table 2: Optimal Design Solution of the Wooden Five-bar Mechanism

A [m ²]	l_0 [m]	l_1 [m]	l_2 [m]
0.1372	0.125	0.280	0.400

Table 3: Optimal Position and Orientation of Cameras

Pos/Ori	x_c [m]	y_c [m]	z_c [m]	ϕ [rad]	θ [rad]	ψ [rad]
Camera3	0.01	0.2291	0.75	0	π	0
Camera4	0.02	0.100	0.100	0	π	0

the following optimization problem:

$$\begin{aligned}
 & \text{minimize} && A = LH \\
 & \text{over} && \mathbf{x} \\
 & \text{subject to} && \ell_{W_x} > \ell_{W_{x0}} \text{ and } \ell_{W_y} > \ell_{W_{y0}}
 \end{aligned} \tag{1}$$

Where: $\mathbf{x} = [\ell_0 \ell_1 \ell_2 \mathbf{x}_{c1} \mathbf{x}_{c2} \mathbf{x}_{c3} \mathbf{x}_{c4}]$ with $\ell_0 = \ell_{A_1A_2}$, $\ell_1 = \ell_{A_1B_1} = \ell_{A_2B_2}$ and $\ell_2 = \ell_{B_1C} = \ell_{B_2C}$, and \mathbf{x}_{ci} ($i = 1$ to 4) is a vector of dimension 6 defining the position and orientation of the cameras. $A = LH$ is the robot footprint (Figure 1a). ℓ_{W_x} and ℓ_{W_y} are the dimensions along \mathbf{x}_0 and \mathbf{y}_0 of the rectangular dexterous workspace Merlet (2006). The robot must have a rectangular dexterous workspace of rectangular shape with dimensions along \mathbf{x}_0 bigger than $\ell_{W_{x0}} = 800$ mm and along \mathbf{y}_0 of $\ell_{W_{y0}} = 200$ mm in which the following properties are certified:

- no Type 2 singularities of the robot,
- the end-effector is within the image frame and thus the legs can be observed,
- knowing the resolution of the camera, the resolution of the end-effector position should be lower than 0.5 mm.
- the static forces exerted into the passive joints are proportional to $1/\sin \xi$, ξ being the angle between the distal links (Briot et al. (2013)). Consequently, it is decided that $\sin \xi$ should be higher than 0.1 to avoid excessive efforts in the joints.

The optimal design variables of problem (1) are given in Tab. 2 and Tab. 3. The architecture of the optimized five-bar mechanism and the position of the dexterous regular workspace is illustrated in Fig. 1b. It should be mentioned that, in order to reduce the number of decision variables in the optimization problem, only the position/orientation of cameras 3 and 4 are optimized. Then, the position/orientation of the two other cameras is obtained by symmetry with respect to the \mathbf{y}_0 axis.

2.2 Link Topology Optimization

As mentioned in Tab. 1, the deformation of the wooden five-bar mechanism should be lower than 0.5 mm under external forces. To reach the desired stiffness

of the robot, topology optimization of the robot proximal links is used. Moreover, in order to decrease the dimensional variability of wood due to humidity, we decided to design our links with a novel type of wood named acetylated beech (Hill (2006)). This wood is known to have many advantages over raw wood species in terms of durability and dimensional stability. However, this acetylated beech, as any other type of wood, presents a normal distribution of its Young's modulus: here, the mean value is equal to 1277.2 MPa and the standard deviation is equal to 2043.5 MPa. To deal with this variation of the mechanical properties of acetylated beech, reliable topology optimization is developed. It should be mentioned that in what follows, only the shape of the robot proximal link are optimized. Indeed, the controller (Andreff et al. (2007)) impose that the distal links are cylindrical, for reason of facility of observation.

Modeling of the linkages elastic behavior Topology optimization uses the same physical model as in the FEM for modelling of a link, except that an interpolation scheme is used in order to define an artificial material. This method is called the Solid Isotropic Material with Penalization (SIMP, Bendsoe and Sigmund (1999)) and is known to be the most effective and the most widely used material interpolation scheme. This interpolation scheme is adopted in order to avoid having optimization results with too much intermediate material density, i.e. in order to have a final design solution with densities only equal to $\rho_{ij} = 1$ (which represent the presence of materials) or $\rho_{ij} = 0$ (which represent the absence of materials) without too many intermediate values ($0 < \rho_{ij} < 1$) that are difficult to manage by the designer. The SIMP scheme is used to parameterize the Young's modulus associated with the stiffness matrix of the finite element ij and it is defined as follows:

$$E_{ij} = E_{\min} + \rho_{ij}^p (E_0 - E_{\min}), \text{ with } \rho_{ij} \in [0, 1] \quad (2)$$

where E_0 is the Young's modulus of the material, E_{\min} is a very small stiffness value assigned to void regions in order to prevent the stiffness matrix from becoming singular, p (typically $p = 3$) is the penalization factor, and E_{ij} is the Young's modulus of element j of the body i corresponding to the density variable ρ_{ij} . Then, based on this definition of the Young's modulus for the element ij , we build its stiffness matrix. Finally, once all elementary matrices are defined, the computation of the body and linkage stiffness matrices is the same as in the traditional methodology (Shabana (2005)). Once the linkage stiffness is obtained, the elastic performance of the mechanism can be defined, in our case we will consider the deformation of the five-bar mechanism end-effector. For dealing with reliability of the robot performance due to the uncertainty of the wood mechanical properties, reliable topology optimisation approach must be used. We take advantages of the work Asadpoure et al. (2011) to define the following constraint included in the

design process:

$$E(\|\mathbf{u}_e\|) + k\sigma(\|\mathbf{u}_e\|) \leq u_{max} \quad (3)$$

Where: $E(\cdot)$ is the expectation operator, $\sigma(\cdot)$ is the standard deviation operator, k is a positive real and \mathbf{u}_e is the deformation vector at the node located at the center of the end-effector under a giving loading and u_{max} is the maximum desired deformation. This constraint, is derived from the well-known Bienaymé-Tchebichev theorem, which states that the probability, for a stochastic variable X , to have $|X - E(X)| \geq k$, is lower than $\frac{\sigma(X)^2}{k^2}$. Thus, by increasing the value of k in (3) we reduce the probability to have a robot whose deformation under the loading given in Tab. 1 will be bigger than 0.5 mm. For instance, with $k = 3$, the probability to have a robot whose deformations are acceptable is almost 90 %. Approximate formulas are provided in Asadpoure et al. (2011) to compute $E(\|\mathbf{u}_e\|)$ and $\sigma(\|\mathbf{u}_e\|)$ and their derivatives, which are necessary in topology optimization process.

Optimization Problem The topology optimisation problem can be formalized as:

$$\begin{aligned} \min_{\rho} \quad & F = \bar{\tau}^2(\mathcal{T}^*) \\ \text{under} \quad & g_{1i} = (\delta_C^2(\mathbf{f}_i, \mathbf{q}^*) - \delta_{max}^2) / \delta_{max}^2 \leq 0, \quad i = 1, 2 \\ & g_{2i} = (\theta_C^2(\mathbf{f}_i, \mathbf{q}^*) - \theta_{max}^2) / \theta_{max}^2 \leq 0, \quad i = 1, 2 \end{aligned} \quad (4)$$

Where \mathbf{q}^* are the selected exciting configurations for the computation of the translational/rotational deformation when the loading \mathbf{f}_i is applied ($i=1,2$). Configuration \mathbf{q}^* can be found using the methodology proposed in Briot and Goldsztejn (2017). The objective is to minimize the RMS of the actuators torques τ along optimized trajectory \mathcal{T}^* selected based on the approach shown in Briot and Goldsztejn (2017). Concerning the constraints, we must ensure that wherever in the dexterous workspace defined in section 2.1, the elastic translational displacement δ_C of the end-effector under external forces $\mathbf{f}_1 = [50 \text{ N } 0 \text{ N } 1 \text{ Nm}]^T$ and $\mathbf{f}_2 = [0 \text{ N } 50 \text{ N } 1 \text{ Nm}]^T$ is lower than $\delta_{max} = 0.5$ mm and that the rotational displacement θ_C of the distal links at the end-effector is lower than $\theta_{max} = 0.25$ mrad. Here, $\delta_C = E(\|\mathbf{u}_{de}\|) + k\sigma(\|\mathbf{u}_{de}\|)$ and $\theta_C = E(\|\mathbf{u}_{re}\|) + k\sigma(\|\mathbf{u}_{re}\|)$, with \mathbf{u}_{de} is the translational deformation of the node at the end-effector and \mathbf{u}_{re} is the rotational one. Indeed, we recall that in our case $k = 3$ in order to have a design reliability of 90 %.

Result of the topology optimization The initial design domain for the proximal links is represented in Fig. 2a. Each link has two empty holes at the extremity for the joints insertion (motor shaft at points A_i , and passive rotary joints for all others. For the meshing of the links, 24132 planar elements of size 1×1 mm are used and with 50 mm of thickness. The obtained shape of proximal links is shown

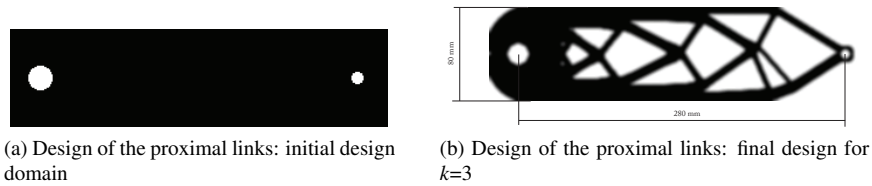


Figure 2: Evolution of the design of the five-bar links: the links are shown in gray-scale (black elements correspond to $\rho_i = 1$, white elements to $\rho_i = 0$, and gray elements to $0 < \rho_i < 1$)



Figure 3: Final prototype of a wooden five-bar mechanism.

in Figure 2b. All models and optimisation algorithms have been encoded with Matlab in the Windows 7 environment. The computation time took around 4948 sec (for a Pentium 4 2.70 GHz, 16 GB of RAM).

Based on the previous optimisation results, a prototype of a wooden five-bar mechanism was designed (Fig. 3). Our future works will concern its visual servoing and the experimental validations of the obtained theoretical results.

3 Conclusion

Robot eco-design is a field of research in Robotics that should be deeper investigated: this could contribute to the Climate Change Mitigation. In order to considerably decrease the environmental impact of robot during the design phase, we proposed to replace metal or carbon composite parts by bio-sourced materials, such as wood. Wood has interesting mechanical properties but also drawbacks: its mechanical performance encounters a large variability. In order to be able to design a stiff industrial robot, robust design approaches must be used. In this paper, we developed a wooden parallel robot, two optimization problems have been formulated and solved in cascade in order to find the robot optimal design architecture for which the impact of the wood properties variability is minimal. Finally, the CAD design and prototyping of the wooden five-bar mechanism and its key

technological solutions have been described.

Our future works, will concern the implementation of sensor-based controllers. Then the experimental validations of the theoretical results obtained in the framework of the RobEcolo project will be made soon to characterize the robot properties.

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