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A Model Reference Tracking based on an Active Fault Tolerant Control for LPV Systems

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Abstract

This work deals with the problem of a model reference tracking based on the design of an Active Fault Tolerant Control (AFTC) for linear parameter varying (LPV) systems affected by actuator faults and unknown inputs. LPV systems are described by a polytopic representation with measurable gain scheduling functions. The main contribution is to design an Active Fault Tolerant Controller whose control law is described by an adaptive Proportional Integral (PI) structure. This one requires three types of on-line informations which are: reference outputs, measured real outputs and the fault estimation provided respectively by a model reference, sensors and an Adaptive Polytopic Observer (APO). These informations are used to reconfigure the designed controller which is able to compensate the fault effects and to make the closed loop system able to track reference outputs in spite of the presence of actuator faults and disturbances. The controller and the observer gains are obtained by solving a set of linear matrices inequalities (LMIs). Performances of the proposed method are compared to an other previous method in order to underline the relevant results.

Keywords: LPV system, Adaptive Polytopic Observer, model reference tracking control, active fault tolerant control, LMIs.

1 Introduction and context

The technologies of modern systems have become more and more sophisticated, which increase the need for the stability, the safety and the reliability. These systems might be malfunctional due to faults that can affect them through their actuators, sensors or components. So, it is necessary to design control systems, as Fault Tolerant Control (FTC), in order to cancel the fault effects and to stabilize the system in spite of these malfunctions.

A fault tolerant controller has the ability to compensate the fault by adjusting some parameters to make the overall closed-loop system stable despite faults and disturbance.

In general, the Fault Tolerant Control laws can be classified into two types: Passive FTC (PFTC) and Active FTC (AFTC). In PFTC systems, controllers are fixed and are designed to be robust against a class of presumed faults which is known as a very conservative approach. In contrast to PFTC, AFTC systems react to the system component failures actively by reconfiguring control actions so that the stability and acceptable performance of the entire system can be maintained [19].

For around two decades, the researchers have been interested in developing new FTC methods. In [11], fault diagnosis and fault tolerant control methods with their application to real plants have been developed. The authors in [8] have been proposed a FTC strategy for LPV systems in the case of actuator faults based on using Unknown Input Observer (UIO). In [13], the authors have proposed an actuator fault estimation scheme based on an Adaptive Polytopic Observer (APO) for LPV descriptor systems in the presence of constant or time varying actuator faults.

The authors in [12] have extended the results published in [13]. Indeed, they have designed an APO in the objective to study the FTC problem for the same class of systems. Based on the informations provided by the APO, a new state feedback control has been developed so as to compensate the effects of constant or time varying actuator faults. The proposed control law therein has allowed the system to be stable but this one has not been able to have a good accuracy.

The development of state, parameter and fault estimation techniques have appeared in [14] and [18] where the authors have used a switched LPV observer and an adaptive observer respectively. The authors in [15] have designed a LPV state observer and state-feedback controller for a Twin Rotor MIMO System (TRMS) described by quasi-LPV model. The proposed technique has used an LPV pole placement method based on LMI regions.

In [10], an adaptive observer has been designed for uncertain multimodels affected by actuator faults and subject to unknown model parameter variations with a FTC approach. Other works as [1] and [2] concerned the Fault Tolerant Tracking Control (FTTC) for uncertain T-S models with unmeasurable premise variables and affected respectively by actuator and sensor faults and only sensor faults.

A FTC strategy using virtual actuators and sensors for LPV systems has been proposed in [16], where an active FTC strategy has been developed in order to reconfigure the virtual actuator/sensor on-line taking into account faults and operating point changes. In [17], the authors have proposed a fault-hiding approach in order to solve the problem of fault tolerant control for a four-wheeled omnidirectional mobile robot. Moreover, a switching LPV virtual actuator has been added to the control loop so as to adapt the faulty plant to the nominal switching LPV controller.

Today, the standard Proportional Integral (PI) and Proportional Integral Derivative (PID) controllers are the most used controllers in the industry. They are characterized by their simplicity and easy adaptation to the controlled system and allow to control a large number of physical quantities in various fields.

The design of FTC systems through PID controller have attracted the attention of several researchers who have interested in developing new techniques in order to tolerate faults by reconfiguring controllers or tuning their gains. Among them, the authors in [6] have proposed an intelligent-based FTC scheme with an evolutionary programming based self-tuning PID fault tolerant controller in order to solve the fault tolerant tracking control problem for unknown nonlinear multi-inputs-multi-outputs (MIMO) systems. The authors in [4] have discussed the problem of PID-FTC system design for linear systems with state and static output feedback. In [9], a FTC approach using an adaptive PID sliding-mode controller is developed for a full scale vehicle dynamic model with an active suspension system in the presence of uncertainties and actuator faults. In fact, the proposed control laws therein, have only standard structures without adding any term for the faults compensation. So, more recently, some research works have appeared in the literature, but it still lacks the on-line faults compensation through adaptive PI or PID controllers especially for LPV systems.

The main goal of the actual paper is to extend existing results so as to improve the accuracy of system outputs. For this we use an adaptive PI control law instead of a state feedback control law as in [1] and [12]. Contrary to [4], [6] and [9] which has proposed a PID controller with a standard control law structure, the proposed control law here is adapted such that becomes able to compensate perfectly the actuator fault effects.

In this paper, a new method based on an adaptive Proportional Integral (PI) control is proposed to construct an Active Fault Tolerant Control (AFTC). To achieve the controller reconfiguration, an Adaptive Polytopic Observer (APO) is used as a Fault Detection and Diagnosis (FDD) module. The proposed approach consists in designing an Active Fault Tolerant Controller which is able to track a model reference in spite of actuator faults and disturbances. Its main goal is to ensure the stability of the closed-loop LPV system with measurable gain scheduling functions.

This paper is organized as follows. The structure of the LPV system with measurable gain scheduling functions is represented and the problem statement of the paper is formulated by introducing the proposed scheme in section II. The section III is dedicated to design both the controller based on an AFTC and the Fault Detection and Diagnosis module based on an Adaptive Polytopic Observer. Finally, the section IV presents a comparison of two methods on a two-tank process, and the section V concludes the paper.

Notation: Throughout this paper, I denotes an identity matrix of appropriate dimension.

2 Problem statement

Let consider a continuous time LPV system affected by additive actuator faults and disturbances and modeled by the following state space representation [3]:

$$\begin{cases} \dot{x}(t) = A(\theta(t))x(t) + B(\theta(t))(u_f(t) + f(t)) + R(\theta(t))d(t) \\ y(t) = Cx(t) \end{cases} \quad (1)$$

where $x(t) \in \mathbb{R}^n$ is the state vector, $u_f(t) \in \mathbb{R}^p$ is the control input vector, $y(t) \in \mathbb{R}^m$ is the measured output vector, $f(t) \in \mathbb{R}^p$ denotes the actuator fault vector and $d(t) \in \mathbb{R}^q$ represents the unknown disturbance vector. C is a constant matrix but $A(\theta(t))$, $B(\theta(t))$ and $R(\theta(t))$ are continuous functions which depend on time varying parameter vector $\theta(t) \in \mathbb{R}^l$. This vector is bounded and lies into a hypercube such that:

$$\theta(t) \in \mathfrak{S} = \{\theta : \theta_{min}(t) \leq \theta(t) \leq \theta_{max}(t); \forall t \geq 0\} \quad (2)$$

By assuming an affine dependance of the parameter vector $\theta(t)$ [12], the system (1) can be described by a polytopic form, where it can be transformed into a convex combination of the vertices of \mathfrak{S} such that:

$$\begin{cases} \dot{x}(t) = \sum_{i=1}^h \rho_i(\theta(t)) [A_i x(t) + B_i(u_f(t) + f(t)) + R_i d(t)] \\ y(t) = Cx(t) \end{cases} \quad (3)$$

where $\rho(\theta(t))$ vary into the convex set Ω .

$$\Omega = \left\{ \rho(\theta(t)) \in \mathbb{R}^h, \rho(\theta(t)) = [\rho_1(\theta(t)), \dots, \rho_h(\theta(t))]^T; \rho_i(\theta(t)) \geq 0 \text{ and } \sum_{i=1}^h \rho_i(\theta(t)) = 1 \right\} \quad (4)$$

The matrices $A_i \in \mathbb{R}^{n \times n}$, $B_i \in \mathbb{R}^{n \times p}$, $R_i \in \mathbb{R}^{n \times q}$ and $C \in \mathbb{R}^{m \times n}$ are time invariant for the i^{th} model represented by a linear form. They characterize the summits \mathfrak{S}_i of the polytope which are defined such that $\Delta_i = [A_i \ B_i \ R_i \ C]$, $\forall i \in [1, \dots, h]$ where $h = 2^l$.

Generally, the LTI models, which compose the polytopic representation of the LPV system, are blended through on-line measurable scheduling functions that depend on the input, the output of the system, or an external parameter [7].

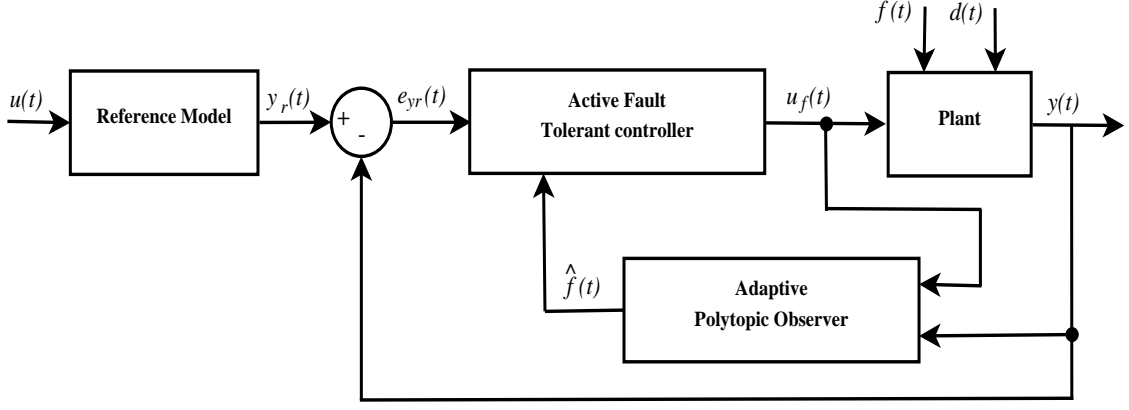


Figure 1: Model reference tracking control Scheme

The main objective of this work is to develop a model reference tracking based on an Active Fault Tolerant Control method. The proposed methodology needs the use of an Adaptive Polytopic Observer (APO) in order to synthesise a feedback control law. The proposed method aims to ensure the asymptotical convergence of the real output to the referential output despite the presence of actuator faults and unknown inputs. The proposed model reference tracking control scheme is illustrated by the figure 1. In fact, the proposed scheme is inspired from the work of (Aouaouda et al.) in [1] which treated the tracking problem of uncertain T-S fuzzy continuous systems with unmeasurable premise variables and affected by unknown inputs. A Proportional Integral (PI) fuzzy observer is used to estimate both constant faults and faulty states so as to synthesize an FTC law.

Let us consider a reference polytopic LPV model given by:

$$\begin{cases} \dot{x}_r(t) = \sum_{i=1}^h \rho_i(\theta(t)) (A_i x_r(t) + B_i u(t)) \\ y_r(t) = C x_r(t) \end{cases} \quad (5)$$

where $x_r(t) \in \mathbb{R}^n$, $u(t) \in \mathbb{R}^p$ and $y_r(t) \in \mathbb{R}^m$ are respectively the state, the control input and the measured output vectors of the model reference.

According to [13], the Adaptive Polytopic Observer (APO) is described by the following structure:

$$\begin{cases} \dot{z}(t) = \sum_{i=1}^h \rho_i(\theta(t)) [N_i z(t) + G_i u_f(t) + L_i y(t) + B_i \hat{f}(t)] \\ \hat{x}(t) = z(t) + T_2 y(t) \\ \hat{y}(t) = C \hat{x}(t) \\ \dot{\hat{f}}(t) = \Gamma \sum_{i=1}^h \rho_i(\theta(t)) \Phi_i (\dot{e}_y(t) + \sigma e_y(t)) \\ e_y(t) = y(t) - \hat{y}(t) \end{cases} \quad (6)$$

where $z(t)$ is the observer state vector, $\hat{x}(t)$ the estimated state vector, $\hat{y}(t)$ is the estimated output vector, $\hat{f}(t)$ is the estimated actuator fault and $e_y(t)$ is the output estimation error. $N_i \in \mathbb{R}^{n \times n}$, $G_i \in \mathbb{R}^{n \times p}$, $L_i \in \mathbb{R}^{n \times m}$, $\Phi_i \in \mathbb{R}^{p \times m}$ and $T_2 \in \mathbb{R}^{n \times m}$ are unknown matrices to be determined afterwards. The matrix $\Gamma \in \mathbb{R}^{p \times p}$ is a symmetric positive definite learning rate matrix. And σ is a positive scalar.

In the remainder of this note, the only necessary assumptions are the following:

Assumption 1

$$\text{rank}(CB_i) = \text{rank}(B_i) = p; \forall i = [1, \dots, h] \quad (7)$$

Assumption 2 The polytopic LPV system (5) is observable, ie:

$$\text{rank} \begin{pmatrix} C \\ CA_i \\ \vdots \\ CA_i^{n-1} \end{pmatrix} = n; \forall i = 1, \dots, h \quad (8)$$

Assumption 3 The polytopic LPV system (5) is detectable, ie:

$$\text{rank} \begin{pmatrix} sI_n - A_i \\ C \end{pmatrix} = n; \forall i = 1, \dots, h \text{ where } s \in C \quad (9)$$

Assumption 4 The input vector $u(t)$ is bounded as $\|u(t)\| \leq \alpha_1$. The fault $f(t)$ satisfies $\|f(t)\| \leq \alpha_2$, the norm of its derivative is bounded i.e. $\|\dot{f}(t)\| \leq \alpha_3$ and the vector $d(t)$ satisfies also $\|d(t)\| \leq \alpha_4$, where $\alpha_1, \alpha_2, \alpha_3, \alpha_4 \geq 0$.

In the following, the aim is to design an Active Fault Tolerant Controller ensuring the tracking trajectory performance of the faulty LPV system to the reference model. The structure of this FTC law is given by:

$$u_f(t) = \sum_{i=1}^h \rho_i(\theta(t)) \left(K_{Pi} e_{yr}(t) + K_{Ii} \int_0^t e_{yr}(\tau) d\tau - H_i \hat{f}(t) \right) \quad (10)$$

where $K_{Pi} \in \mathbb{R}^{m \times n}$, $K_{Ii} \in \mathbb{R}^{m \times n}$ and $H_i \in \mathbb{R}^{m \times p}$ are control gain matrices to be designed. Contrary to (Aouaouda et al.) in [1] where the developed approach is based on the design of a controller whose FTC law is considered as an adaptive state feedback control law, the proposed AFTC law (10) is described by an adaptive Proportional Integral (PI) control. This control law improves both the stability and accuracy performances and compensates the fault effects on the studied system.

3 Tracking Fault Tolerant Control

3.1 Reconfiguration analysis

Let define the reconfigured model as an augmented model which includes principally the state vector $x(t)$, the state of the tracking error $e_t(t)$, the state of the estimation error $e_s(t)$ and the fault estimation error $e_f(t)$. These errors are described by:

$$e_t(t) = x_r(t) - x(t) \quad (11)$$

$$e_s(t) = x(t) - \hat{x}(t) \quad (12)$$

$$e_f(t) = f(t) - \hat{f}(t) \quad (13)$$

According to [4], we consider a new state vector $x_t(t)$ which is defined by:

$$x_t(t) = \int_0^t e_t(\tau) d\tau \quad (14)$$

such that its dynamic is expressed as follows:

$$\dot{x}_t(t) = e_t(t) \quad (15)$$

By using the state space representations (3) and (5) and the equation (11), the expression of $e_{yr}(t)$ becomes:

$$\begin{aligned} e_{yr}(t) &= y_r(t) - y(t) \\ &= Ce_t(t) \end{aligned} \quad (16)$$

Now, tacking into account the equations (13), (14) and (16), the AFTC law (10) can be rewritten as:

$$u_f(t) = \sum_{i=1}^h \rho_i(\theta(t)) (K_{P_i}Ce_t(t) + K_{I_i}Cx_t(t) + H_ie_f(t) - H_if(t)) \quad (17)$$

Substituting (17) into (3), it may be possible to rewrite the dynamic of the state vector $x(t)$ as follows:

$$\begin{aligned} \dot{x}(t) &= \sum_{i,j=1}^h \rho_i(\theta(t)) \rho_j(\theta(t)) [A_ix(t) + B_iK_{P_j}Ce_t(t) + B_iK_{I_j}Cx_t(t) + B_iH_je_f(t) \\ &\quad - H_{ij}f(t) + R_id(t)] \end{aligned} \quad (18)$$

where,

$$H_{ij} = B_iH_j - B_i \quad (19)$$

The state dynamic of the tracking error is:

$$\dot{e}_t(t) = \dot{x}_r(t) - \dot{x}(t) \quad (20)$$

Substituting the expressions of $\dot{x}_r(t)$ and $\dot{x}(t)$ respectively from (5) and (18) into (20), which becomes:

$$\begin{aligned} \dot{e}_t(t) &= \sum_{i,j=1}^h \rho_i(\theta(t)) \rho_j(\theta(t)) [(A_i - B_iK_{P_j}C)e_t(t) - B_iK_{I_j}Cx_t(t) - B_iH_je_f(t) \\ &\quad + B_iu(t) + H_{ij}f(t) - R_id(t)] \end{aligned} \quad (21)$$

In order to study the state estimation error $e_s(t)$ defined by (12), we assume that there exist $T_1 \in \mathbb{R}^{n \times n}$ and $T_2 \in \mathbb{R}^{n \times m}$ which verify the following equation:

$$T_1 + T_2C = I_n \quad (22)$$

Therefore the above equation can be rewritten as:

$$\begin{bmatrix} T_1 & T_2 \end{bmatrix} \begin{bmatrix} I_n \\ C \end{bmatrix} = I_n \quad (23)$$

A particular solution of the matrices T_1 and T_2 is computed as:

$$\begin{bmatrix} T_1 & T_2 \end{bmatrix} = \begin{bmatrix} I_n \\ C \end{bmatrix}^+ \quad (24)$$

Thus, by using (6) and (22), the equation (12) will be rewritten as:

$$e_s(t) = T_1x(t) - z(t) \quad (25)$$

The the state dynamic of the estimation error (25) is then described as follows:

$$\dot{e}_s(t) = T_1 \dot{x}(t) - \dot{z}(t) \quad (26)$$

By using the expressions of $\dot{x}(t)$ and $\dot{z}(t)$ respectively from (3) and (6), the last equation (26) can be rewritten as:

$$\begin{aligned} \dot{e}_s(t) = \sum_{i=1}^h \rho_i(\theta(t)) & [(T_1 A_i + E_i C - N_i) x(t) + (T_1 B_i - G_i) u(t) + N_i e_s(t) + B_i e_f(t) \\ & + M_i f(t) + \bar{R}_i d(t)] \end{aligned} \quad (27)$$

where,

$$E_i = N_i T_2 - L_i \quad (28)$$

$$M_i = T_1 B_i - B_i \quad (29)$$

$$\text{and} \quad \bar{R}_i = T_1 R_i \quad (30)$$

We assume that for all $i = 1, \dots, h$, the following conditions hold true:

$$T_1 A_i + E_i C - N_i = 0 \quad (31)$$

$$\text{and} \quad T_1 B_i - G_i = 0 \quad (32)$$

Therefore, by taking into account (31) and (32), the state dynamic of the estimation error (27) holds:

$$\dot{e}_s(t) = \sum_{i=1}^h \rho_i(\theta(t)) [N_i e_s(t) + B_i e_f(t) + M_i f(t) + \bar{R}_i d(t)] \quad (33)$$

The fault estimation error dynamic can be expressed as follows:

$$\dot{e}_f(t) = \dot{f}(t) - \dot{\hat{f}}(t) \quad (34)$$

3.2 Stability analysis

Before beginning the stability analysis, we consider the following lemma:

Lemma 1 *Given a symmetric positive matrix P and a positive scalar μ , the following inequality holds [10]:*

$$2x^T y \leq \frac{1}{\mu} x^T P x + \mu y^T P^{-1} y; x, y \in \mathbb{R}^n \quad (35)$$

Lemma 2 *Let X , Y and Z be real matrices of appropriate dimensions with Δ satisfying $\Delta^T \Delta \leq I$, then [7]:*

$$X + Y \Delta Z + Z^T \Delta^T Y^T \prec 0 \quad (36)$$

if and only if there exists a positive scalar $\psi > 0$ such that:

$$X + \psi Z^T Z + \frac{1}{\psi} Y Y^T \prec 0 \quad (37)$$

or equivalently:

$$\begin{bmatrix} X & Y & \psi Z^T \\ Y^T & -\psi I & 0 \\ \psi Z & 0 & -\psi I \end{bmatrix} \prec 0 \quad (38)$$

For effective Active Fault Tolerant Control, the reconfigured model presented in the last section must be stable in order to prove that the states of the faulty LPV system converge asymptotically to the states of the reference model. The following theorem provides sufficient conditions to fulfill this goal.

Theorem 1 *Let consider the polytopic system (3), the Active Fault Tolerant Control (AFTC) (10), the reference model (5) and the Adaptive Polytopic Observer (AOP) (6). Given positive scalars σ , μ , β and ψ and a symmetric and positive definite matrix Γ , the faulty state $x(t)$, the state of the tracking error $e_t(t)$, the state of the estimation error $e_s(t)$ and the fault estimation error $e_f(t)$ are bounded if there exist symmetric and positive definite matrices $X_1 = P_1^{-1}$, $X_2 = P_2^{-1}$, $X_3 = P_3^{-1}$, Q_1 , Q_2 , Q_3 and Q_4 and matrices $W_{Pj} = K_{Pj}CX_2$, $W_{Ij} = K_{Ij}CX_2$, $S_i = E_iCX_3$, H_i and Φ_i such that the following LMIs are satisfied for all $i, j, k = 1, \dots, h$:*

$$\begin{pmatrix} \bar{X}_{ijk} & Y_i & \psi Z_k^T & P \\ * & -\psi I & 0 & 0 \\ * & * & -\psi I & 0 \\ * & * & * & -Q \end{pmatrix} \prec 0 \quad (39)$$

where,

$$\bar{X}_{ijk} = \begin{pmatrix} \Pi_i & B_i W_{Pj} & B_i W_{Ij} & 0 & B_i H_j \\ * & \Omega_{ij} & X_2 - B_i W_{Ij} & 0 & -B_i H_j \\ * & * & -\beta I & 0 & 0 \\ * & * & * & \Theta_i & B_i \\ * & * & * & * & \Sigma_{ik} \end{pmatrix} \quad (40)$$

$$Y_i = \begin{pmatrix} 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & -\Phi_i C & 0 & 0 \end{pmatrix} \quad (41)$$

$$Z_k^T = \begin{pmatrix} 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & \Xi_k^T & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 \end{pmatrix} \quad (42)$$

$$P = \begin{pmatrix} X_1^T & 0 & 0 & 0 \\ 0 & X_2^T & 0 & 0 \\ 0 & 0 & I & 0 \\ 0 & 0 & 0 & X_3^T \\ 0 & 0 & 0 & 0 \end{pmatrix} \quad (43)$$

and

$$Q = \begin{pmatrix} \frac{2}{\mu} Q_1 & 0 & 0 & 0 \\ * & \frac{3}{\mu} Q_2 & 0 & 0 \\ * & * & \beta I & 0 \\ * & * & * & \frac{2}{\mu} Q_3 \end{pmatrix} \quad (44)$$

with,

$$\Pi_i = A_i X_1 + X_1 A_i^T \quad (45)$$

$$\Omega_{ij} = A_i X_2 + X_2 A_i^T - B_i W_{Pj} - W_{Pj}^T B_i^T \quad (46)$$

$$\Theta_i = T_1 A_i X_3 + X_3 A_i^T T_1^T + S_i + S_i^T \quad (47)$$

$$\Xi_k = X_3 + \frac{1}{\sigma} (T_1 A_k X_3 + S_k) \quad (48)$$

$$\Sigma_{ik} = -\frac{1}{\sigma} (\Phi_i C B_k + B_k^T C^T \Phi_i^T) + \frac{3}{\mu\sigma} Q_4 \quad (49)$$

The controller and the observer gains are given by:

$$K_{Pj} = W_{Pj} (C X_2)^{-1} \quad (50)$$

$$K_{Ij} = W_{Ij} (C X_2)^{-1} \quad (51)$$

$$N_i = T_1 A_i + E_i C \quad (52)$$

$$L_i = N_i T_2 - E_i \quad (53)$$

$$\text{and} \quad G_i = T_1 B_i \quad (54)$$

■

Proof 1 In this proof, we use the Lyapunov's method to prove the stability of the reconfigured model whose states are $x(t)$, $e_t(t)$, $x_t(t)$, $e_s(t)$ and $e_f(t)$. Moreover, we will formulate the LMIs in order to solve the optimisation problem and determinate the unknown matrices of both the controller and the observer.

Consider a quadratic Lyapunov function defined as follows:

$$V(t) = x^T(t) P_1 x(t) + e_t^T(t) P_2 e_t(t) + x_t^T(t) P_2 x_t(t) + e_s^T(t) P_3 e_s(t) + \frac{1}{\sigma} e_f^T(t) \Gamma^{-1} e_f(t) \quad (55)$$

where P_1 , P_2 , P_3 and Γ^{-1} are symmetric positive definite matrices with appropriate dimensions. The time derivative of the above Lyapunov function (55) yields:

$$\begin{aligned} \dot{V}(t) = & \dot{x}^T(t) P_1 x(t) + x^T(t) P_1 \dot{x}(t) + \dot{e}_t^T(t) P_2 e_t(t) + e_t^T(t) P_2 \dot{e}_t(t) + \dot{x}_t^T(t) P_2 x_t(t) \\ & + x_t^T(t) P_2 \dot{x}_t(t) + \dot{e}_s^T(t) P_3 e_s(t) + e_s^T(t) P_3 \dot{e}_s(t) + \frac{1}{\sigma} \dot{e}_f^T(t) \Gamma^{-1} e_f(t) + \frac{1}{\sigma} e_f^T(t) \Gamma^{-1} \dot{e}_f(t) \end{aligned} \quad (56)$$

By taking into account the expressions (18) of $\dot{x}(t)$, (21) of $\dot{e}_t(t)$, (15) of $\dot{x}_t(t)$, (33) of $\dot{e}_s(t)$ and (34) of $\dot{e}_f(t)$ and using the expression of $\hat{f}(t)$ in (6), the equation (56) becomes:

$$\begin{aligned} \dot{V}(t) = & \sum_{i,j=1}^h \rho_i(\theta(t)) \rho_j(\theta(t)) [x^T(t) (A_i^T P_1 + P_1 A_i) x(t) + 2x^T(t) P_1 B_i K_{Pj} C e_t(t) \\ & + 2x^T(t) P_1 B_i K_{Ij} C x(t) + 2x^T(t) P_1 B_i H_j e_f(t) - 2x^T(t) P_1 H_{ij} f(t) + 2x^T(t) P_1 R_i d(t) \\ & + e_t^T(t) (A_{ij}^T P_2 + P_2 A_{ij}) e_t(t) + 2e_t^T(t) (P_2 - P_2 B_i K_{Ij} C) x_t(t) - 2e_t^T(t) P_2 B_i H_j e_f(t) \\ & + 2e_t^T(t) P_2 H_{ij} f(t) + 2e_t^T(t) P_2 B_i u(t) - 2e_t^T(t) P_2 R_i d(t) + e_s^T(t) (N_i^T P_3 + P_3 N_i) e_s(t) \\ & + 2e_s^T(t) (P_3 B_i - C^T \Phi_i^T) e_f(t) + 2e_s^T(t) P_3 M_i f(t) + 2e_s^T(t) P_3 \bar{R}_i d(t) \\ & - \frac{1}{\sigma} \dot{e}_s^T(t) C^T \Phi_i^T e_f(t) - \frac{1}{\sigma} e_f^T(t) \Phi_i C \dot{e}_s(t) + \frac{2}{\sigma} e_f^T(t) \Gamma^{-1} \dot{f}(t) \Big] \end{aligned} \quad (57)$$

where,

$$A_{ij} = A_i - B_i K_{Pj} C \quad (58)$$

Now, substituting $\dot{e}_s(t)$ by its expression (33), the equation (57) will be rewritten as follows:

$$\begin{aligned}
\dot{V}(t) = & \sum_{i,j,k=1}^h \rho_i(\theta(t)) \rho_j(\theta(t)) \rho_k(\theta(t)) [x^T(t) (A_i^T P_1 + P_1 A_i) x(t) + 2x^T(t) P_1 B_i K_{P_j} C e_t(t) \\
& + 2x^T(t) P_1 B_i K_{I_j} C x_t(t) + 2x^T(t) P_1 B_i H_j e_f(t) - 2x^T(t) P_1 H_{ij} f(t) + 2x^T(t) P_1 R_i d(t) \\
& + e_t^T(t) (A_{ij}^T P_2 + P_2 A_{ij}) e_t(t) + 2e_t^T(t) (P_2 - P_2 B_i K_{I_j} C) x_t(t) - 2e_t^T(t) P_2 B_i H_j e_f(t) \\
& + 2e_t^T(t) P_2 H_{ij} f(t) + 2e_t^T(t) P_2 B_i u(t) - 2e_t^T(t) P_2 R_i d(t) + e_s^T(t) (N_i^T P_3 + P_3 N_i) e_s(t) \\
& + 2e_s^T(t) \left(P_3 B_i - C^T \Phi_i^T - \frac{1}{\sigma} N_k^T C^T \Phi_i^T \right) e_f(t) + 2e_s^T(t) P_3 M_i f(t) + 2e_s^T(t) P_3 \bar{R}_i d(t) \\
& - \frac{1}{\sigma} e_f^T(t) (\Phi_i C B_k + B_k^T C^T \Phi_i^T) e_f(t) - \frac{2}{\sigma} e_f^T(t) \Phi_i C M_k f(t) + \frac{2}{\sigma} e_f^T(t) \Phi_i C \bar{R}_k d(t) \\
& + \frac{2}{\sigma} e_f^T(t) \Gamma^{-1} \dot{f}(t) \Big]
\end{aligned} \tag{59}$$

By applying the Lemma 1 and using the Assumption 4, the following term from the equation (59) can be bounded as follows:

$$\begin{aligned}
-2x^T(t) P_1 H_{ij} f(t) & \leq \frac{1}{\mu} x^T(t) Q_1 x(t) + \mu f^T(t) H_{ij}^T P_1 Q_1^{-1} P_1 H_{ij} f(t) \\
& \leq \frac{1}{\mu} x^T(t) Q_1 x(t) + \eta_{1ij}
\end{aligned} \tag{60}$$

In the same manner, the other terms may be bounded as:

$$2x^T(t) P_1 R_i d(t) \leq \frac{1}{\mu} x^T(t) Q_1 x(t) + \eta_{2i} \tag{61}$$

$$2e_t^T(t) P_2 B_i u(t) \leq \frac{1}{\mu} e_t^T(t) Q_2 e_t(t) + \eta_{3i} \tag{62}$$

$$2e_t^T(t) P_2 H_{ij} f(t) \leq \frac{1}{\mu} e_t^T(t) Q_2 e_t(t) + \eta_{4ij} \tag{63}$$

$$-2e_t^T(t) P_2 R_i d(t) \leq \frac{1}{\mu} e_t^T(t) Q_2 e_t(t) + \eta_{5i} \tag{64}$$

$$2e_s^T(t) P_3 M_i f(t) \leq \frac{1}{\mu} e_s^T(t) Q_3 e_s(t) + \eta_{6i} \tag{65}$$

$$2e_s^T(t) P_3 \bar{R}_i d(t) \leq \frac{1}{\mu} e_s^T(t) Q_3 e_s(t) + \eta_{7i} \tag{66}$$

$$-\frac{2}{\sigma} e_f^T(t) \Phi_i C M_k f(t) \leq \frac{1}{\mu} e_f^T(t) Q_4 e_f(t) + \eta_{8ik} \tag{67}$$

$$\frac{2}{\sigma} e_f^T(t) \Phi_i C \bar{R}_k d(t) \leq \frac{1}{\mu} e_f^T(t) Q_4 e_f(t) + \eta_{9ik} \tag{68}$$

$$\frac{2}{\sigma} e_f^T(t) \Gamma^{-1} \dot{f}(t) \leq \frac{1}{\mu} e_f^T(t) Q_4 e_f(t) + \eta_{10} \tag{69}$$

with,

$$\eta_{1ij} = \mu\alpha_2^2 \lambda_{\max} (H_{ij}^T P_1 Q_1^{-1} P_1 H_{ij}) \quad (70)$$

$$\eta_{2i} = \mu\alpha_4^2 \lambda_{\max} (R_i^T P_1 Q_1^{-1} P_1 R_i) \quad (71)$$

$$\eta_{3i} = \mu\alpha_1^2 \lambda_{\max} (B_i^T P_2 Q_2^{-1} P_2 B_i) \quad (72)$$

$$\eta_{4ij} = \mu\alpha_2^2 \lambda_{\max} (F_{ij}^T P_2 Q_2^{-1} P_2 F_{ij}) \quad (73)$$

$$\eta_{5i} = \mu\alpha_4^2 \lambda_{\max} (R_i^T P_2 Q_2^{-1} P_2 R_i) \quad (74)$$

$$\eta_{6i} = \mu\alpha_2^2 \lambda_{\max} (M_i^T P_3 Q_3^{-1} P_3 M_i) \quad (75)$$

$$\eta_{7i} = \mu\alpha_4^2 \lambda_{\max} (\bar{R}_i^T P_3 Q_3^{-1} P_3 \bar{R}_i) \quad (76)$$

$$\eta_{8ik} = \mu\alpha_2^2 \lambda_{\max} (M_k^T C^T \Phi_i^T Q_4^{-1} \Phi_i C M_k) \quad (77)$$

$$\eta_{9ik} = \mu\alpha_4^2 \lambda_{\max} (\bar{R}_k^T C^T \Phi_i^T Q_4^{-1} \Phi_i C \bar{R}_k) \quad (78)$$

$$\eta_{10} = \mu\alpha_3^2 \lambda_{\max} (\Gamma^{-1T} Q_4^{-1} \Gamma^{-1}) \quad (79)$$

By taking into account the inequalities (60)-(69), $\dot{V}(t)$ can be bounded as follows:

$$\begin{aligned} \dot{V}(t) \leq & \sum_{i,j,k=1}^h \rho_i(\theta(t)) \rho_j(\theta(t)) \rho_k(\theta(t)) \left[x^T(t) \left(A_i^T P_1 + P_1 A_i + \frac{2}{\mu} Q_1 \right) x(t) \right. \\ & + 2x^T(t) P_1 B_i K_{Pj} C e_t(t) + 2x^T(t) P_1 B_i K_{Ij} C x_t(t) + 2x^T(t) P_1 B_i H_j e_f(t) \\ & + e_t^T(t) \left(A_{ij}^T P_2 + P_2 A_{ij} + \frac{3}{\mu} Q_2 \right) e_t(t) + 2e_t^T(t) (P_2 - P_2 B_i K_{Ij} C) x_t(t) \\ & - 2e_t^T(t) P_2 B_i H_j e_f(t) + e_s^T(t) \left(N_i^T P_3 + P_3 N_i + \frac{2}{\mu} Q_3 \right) e_s(t) + 2e_s^T(t) (P_3 B_i - C^T \Phi_i^T \\ & \left. - \frac{1}{\sigma} N_k^T C^T \Phi_i^T) e_f(t) - \frac{1}{\sigma} e_f^T(t) \left(\Phi_i C B_k + B_k^T C^T \Phi_i^T + \frac{3}{\mu\sigma} Q_4 \right) e_f(t) \right] + \delta \end{aligned} \quad (80)$$

where the scalar δ is the maximum value over i, j and k such that:

$$\delta = \max_{i,j,k} (\eta_{1ij} + \eta_{2i} + \eta_{3i} + \eta_{4ij} + \eta_{5i} + \eta_{6i} + \eta_{7i} + \eta_{8ik} + \eta_{9ik} + \eta_{10}) \quad (81)$$

Then, the inequality (80) can be rewritten under the following form:

$$\dot{V}(t) \leq \tilde{x}^T(t) \left(\sum_{i,j,k=1}^h \rho_i(\theta(t)) \rho_j(\theta(t)) \rho_k(\theta(t)) \Lambda_{ijk} \right) \tilde{x}(t) + \delta \quad (82)$$

where $\tilde{x}(t) = [x^T(t) \quad e_t^T(t) \quad x_t^T(t) \quad e_s^T(t) \quad e_f^T(t)]^T$ is an augmented system and Λ_{ijk} is a matrix defined as follows:

$$\Lambda_{ijk} = \begin{pmatrix} \tilde{\Pi}_i & P_1 B_i K_{Pj} C & P_1 B_i K_{Ij} C & 0 & P_1 B_i H_j \\ * & \tilde{\Omega}_{ij} & P_2 - P_2 B_i K_{Ij} C & 0 & -P_2 B_i H_j \\ * & * & 0 & 0 & 0 \\ * & * & * & \tilde{\Theta}_i & \tilde{\Xi}_{ik} \\ * & * & * & * & \Sigma_{ik} \end{pmatrix} \quad (83)$$

with,

$$\tilde{\Pi}_i = A_i^T P_1 + P_1 A_i + \frac{2}{\mu} Q_1 \quad (84)$$

$$\tilde{\Omega}_{ij} = A_{ij}^T P_2 + P_2 A_{ij} + \frac{3}{\mu} Q_2 \quad (85)$$

$$\tilde{\Theta}_i = N_i^T P_3 + P_3 N_i + \frac{2}{\mu} Q_3 \quad (86)$$

$$\tilde{\Xi}_{ik} = P_3 B_i - C^T \Phi_i^T - \frac{1}{\sigma} N_k^T C^T \Phi_i^T \quad (87)$$

$$\text{and} \quad \Sigma_{ik} = -\frac{1}{\sigma} (\Phi_i C B_k + B_k^T C^T \Phi_i^T) + \frac{3}{\mu\sigma} Q_4 \quad (88)$$

If the following constraint holds:

$$\sum_{i,j,k=1}^h \rho_i(\theta(t)) \rho_j(\theta(t)) \rho_k(\theta(t)) \Lambda_{ijk} \prec 0 \quad (89)$$

we can obtain:

$$\dot{V}(t) \leq -\varepsilon \|\tilde{x}(t)\|^2 + \delta \quad (90)$$

where ε is a positive scalar defined by:

$$\varepsilon = \min \lambda_{\min} \left(- \sum_{i,j,k=1}^h \rho_i(\theta(t)) \rho_j(\theta(t)) \rho_k(\theta(t)) \Lambda_{ijk} \right) \quad (91)$$

and it can be bounded as follows:

$$\varepsilon \leq \min_{i,j,k} \lambda_{\min} (-\Lambda_{ijk}) \quad (92)$$

Consequently, we can say that $\dot{V}(t) \prec 0$, if $\varepsilon \|\tilde{x}(t)\|^2 \succ \delta; \forall t \geq 0$. Based on the Lyapunov stability theory, this proves that the augmented system $\tilde{x}(t)$ is stable meaning that the faulty state vector $x(t)$, the state vector $x_t(t)$, the state of the tracking error $e_t(t)$, the state of the estimation error $e_s(t)$ and the fault estimation error $e_f(t)$ are bounded.

By considering the constraint (89), we define a matrix:

$$X = \begin{pmatrix} P_1^{-1} & 0 & 0 & 0 & 0 \\ * & P_2^{-1} & 0 & 0 & 0 \\ * & * & P_2^{-1} & 0 & 0 \\ * & * & * & P_3^{-1} & 0 \\ * & * & * & * & I \end{pmatrix} \succ 0 \quad (93)$$

such that $\Psi_{ijk} = X \Lambda_{ijk} X \prec 0$.

Then, considering the following change of variables:

$$X_1 = P_1^{-1} \quad (94)$$

$$X_2 = P_2^{-1} \quad (95)$$

$$X_3 = P_3^{-1} \quad (96)$$

and computing $\Psi_{ijk} \prec 0$ that holds:

$$\Psi_{ijk} = \begin{pmatrix} X_1 \tilde{\Pi}_i X_1 & B_i K_{Pj} C X_2 & B_i K_{Ij} C X_2 & 0 & B_i H_j \\ * & X_2 \tilde{\Omega}_{ij} X_2 & X_2 - B_i K_{Ij} C X_2 & 0 & -B_i H_j \\ * & * & 0 & 0 & 0 \\ * & * & * & X_3 \tilde{\Theta}_i X_3 & X_3 \tilde{\Xi}_{ik} \\ * & * & * & * & \Sigma_{ik} \end{pmatrix} \prec 0 \quad (97)$$

where,

$$X_1 \tilde{\Pi}_i X_1 = \Pi_i + \frac{2}{\mu} X_1 Q_1 X_1 \quad (98)$$

$$X_2 \tilde{\Omega}_{ij} X_2 = \Omega_{ij} + \frac{3}{\mu} X_2 Q_2 X_2 \quad (99)$$

$$X_3 \tilde{\Theta}_i X_3 = \Theta_i + \frac{2}{\mu} X_3 Q_3 X_3 \quad (100)$$

$$X_3 \tilde{\Xi}_{ik} = B_i - X_3 C^T \Phi_i^T - \frac{1}{\sigma} X_3 N_k^T C^T \Phi_i^T \quad (101)$$

with,

$$\Pi_i = X_1 A_i^T + A_i X_1 \quad (102)$$

$$\Omega_{ij} = X_2 A_{ij}^T + A_{ij} X_2 \quad (103)$$

$$\text{and } \Theta_i = X_3 N_i^T + N_i X_3 \quad (104)$$

For all positive scalar $\beta > 0$, replacing the zero in the diagonal of the matrix (97) by the term $(\beta I - \beta I = 0)$ and using the expression (101) of $(X_3 \tilde{\Xi}_{ik})$ so as to rewrite (97) as this way:

$$\Psi_{ijk} = X_{ijk} + Y_i I Z_k + Z_k^T I Y_i^T \prec 0 \quad (105)$$

where,

$$X_{ijk} = \begin{pmatrix} X_1 \tilde{\Pi}_i X_1 & B_i K_{Pj} C X_2 & B_i K_{Ij} C & 0 & B_i H_j \\ * & X_2 \tilde{\Omega}_{ij} X_2 & X_2 - B_i K_{Ij} C X_2 & 0 & -B_i H_j \\ * & * & \beta I - \beta I & 0 & 0 \\ * & * & * & X_3 \tilde{\Theta}_i X_3 & B_i \\ * & * & * & * & \Sigma_{ik} \end{pmatrix} \quad (106)$$

$$Y_i = \begin{pmatrix} 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & -\Phi_i C & 0 & 0 \end{pmatrix} \quad (107)$$

and,

$$Z_k = \begin{pmatrix} 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & \Xi_k & 0 \\ 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 \end{pmatrix} \quad (108)$$

with,

$$\Xi_k = X_3 + \frac{1}{\sigma} N_k X_3 \quad (109)$$

Applying the Lemma 2, the inequality (105) holds true if and only if there exists a positive scalar $\psi \succ 0$ such that:

$$X_{ijk} + \psi Z_k^T Z_k + \frac{1}{\psi} Y_i Y_i^T \prec 0 \quad (110)$$

or equivalently:

$$\begin{pmatrix} X_{ijk} & Y_i & \psi Z_k^T \\ * & -\psi I & 0 \\ * & * & -\psi I \end{pmatrix} \prec 0 \quad (111)$$

Now, we dissociate the term $\frac{2}{\mu} X_1 Q_1 X_1$ from the inequality (111) which may be reformulated as follows:

$$\begin{pmatrix} \tilde{X}_{ijk} & Y_i & \psi Z_k^T \\ * & -\psi I & 0 \\ * & * & -\psi I \end{pmatrix} - \bar{X}_1^T \left(-\frac{2}{\mu} Q_1 \right) \bar{X}_1 \prec 0 \quad (112)$$

where,

$$\tilde{X}_{ijk} = \begin{pmatrix} \Pi_i & B_i K_{P_j} C X_2 & B_i K_{I_j} C X_2 & 0 & B_i H_j \\ * & X_2 \tilde{\Omega}_{ij} X_2 & X_2 - B_i K_{I_j} C X_2 & 0 & -B_i H_j \\ * & * & \beta I - \beta I & 0 & 0 \\ * & * & * & X_3 \tilde{\Theta}_i X_3 & B_i \\ * & * & * & * & \Sigma_{ik} \end{pmatrix} \quad (113)$$

and,

$$\bar{X}_1 = \begin{pmatrix} X_1 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & \dots \\ \dots & 0 & 0 & 0 & 0 & 0 \end{pmatrix} \quad (114)$$

Applying the modified Schur Lemma in the above inequality (112) and then we repeat the same work successively for the terms $\frac{3}{\mu} X_2 Q_2 X_2$, $I(\beta I)I$ and $\frac{2}{\mu} X_3 Q_3 X_3$ so as to obtain:

$$\begin{pmatrix} \bar{X}_{ijk} & Y_i & \psi Z_k^T & P \\ * & -\psi I & 0 & 0 \\ * & * & -\psi I & 0 \\ * & * & * & -Q \end{pmatrix} \prec 0 \quad (115)$$

where,

$$\bar{X}_{ijk} = \begin{pmatrix} \Pi_i & B_i K_{P_j} C X_2 & B_i K_{I_j} C X_2 & 0 & B_i H_j \\ * & \Omega_{ij} & X_2 - B_i K_{I_j} C X_2 & 0 & -B_i H_j \\ * & * & -\beta I & 0 & 0 \\ * & * & * & \Theta_i & B_i \\ * & * & * & * & \Sigma_{ik} \end{pmatrix} \quad (116)$$

$$P = \begin{pmatrix} X_1^T & 0 & 0 & 0 \\ 0 & X_2^T & 0 & 0 \\ 0 & 0 & I & 0 \\ 0 & 0 & 0 & X_3^T \\ 0 & 0 & 0 & 0 \end{pmatrix} \quad (117)$$

and

$$Q = \begin{pmatrix} \frac{2}{\mu} Q_1 & 0 & 0 & 0 \\ * & \frac{3}{\mu} Q_2 & 0 & 0 \\ * & * & \beta I & 0 \\ * & * & * & \frac{2}{\mu} Q_3 \end{pmatrix} \quad (118)$$

This completes the proof of the theorem.

In order to solve numerically the theorem1, the following algorithm is proposed:

Algorithm 1 *Algorithm to solve the LMI optimization problem.*

- *Step 1: Compute the matrices T_1 and T_2 from (24).*
- *Step 2: Choose positive values for the scalars σ , μ , β and ψ and a symmetric and positive definite matrix Γ .*
- *Step 3: Solve the LMIs (39) by using the LMI Toolbox and obtain the unknown matrices: X_1 , X_2 , X_3 , Q_1 , Q_2 , Q_3 , Q_4 , W_{Pj} , W_{Ij} , H_j , S_i and Φ_i .*
- *Step 4: Compute the matrices E_i by: $E_i = S_i(CX_3)^{-1}$.*
- *Step 5: Compute the gain matrices K_{Pj} , K_{Ij} , N_i , L_i and G_i from (50)-(54).*

for all $i, j = 1, \dots, h$.

4 Application on two-tank process

4.1 Two-tank process

The two-tank process, presented in the figure 2, consists in two liquid tanks that can be filled with two identical, independent pumps that deliver the liquid flows $Q_1(t)$ and $Q_2(t)$. The tanks are interconnected to each other through a pipe whose flow is $Q_{12}(t)$, while the outflow from the system is located at the second tank and provides a flow $Q_{N2}(t)$ to the consumer [16]. According

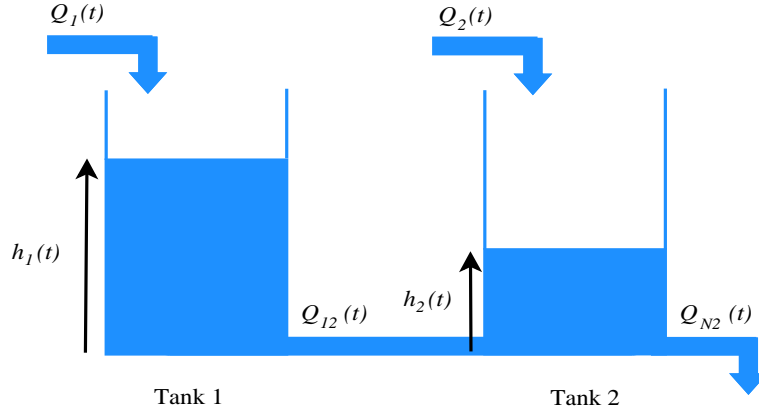


Figure 2: Two-tank process

to [16], using both the mass balance and the Torricelli's law and under the assumption that the system is operating with $h_1(t) > h_2(t)$, the two-tank process can be modeled by the following non-linear state space.

$$\begin{cases} \frac{dh_1(t)}{dt} = \frac{1}{A} (Q_1(t) - Q_{12}(t)) \\ \frac{dh_2(t)}{dt} = \frac{1}{A} (Q_2(t) + Q_{12}(t) - Q_{N2}(t)) \end{cases} \quad (119)$$

with,

$$Q_{12}(t) = c_{12} \sqrt{h_1(t) - h_2(t)} \quad (120)$$

$$\text{and, } Q_{N2}(t) = c_2 \sqrt{h_2(t)} \quad (121)$$

where $h_1(t)$ and $h_2(t)$ are the liquid levels of the tank 1 and the tank 2 respectively, which are used as state variables. A denotes the area of the cylindric tanks. c_{12} and c_2 are the constant flows of the interconnecting pipe and of the outflow pipe respectively. The two-tank process's parameters are presented in the table I.

Symbol	Parameters	Values	tableTwo-tank
A	area of the two tanks	$1.54 \times 10^{-2}m^2$	
c_{12}	flow constant of the interconnecting pipe	$6 \times 10^{-4}m^{5/2}/s$	
c_2	flow constant of the outflow pipe	$13 \times 10^{-4}m^{5/2}/s$	
process's parameters			

Considering:

$x(t) = [x_1(t) \ x_2(t)]^T = [h_1(t) \ h_2(t)]^T$,
 $u(t) = [u_1(t) \ u_2(t)]^T = [Q_1(t) \ Q_2(t)]^T$ and $y(t) = x(t)$ are respectively the state, the control input and the measured output vectors, the two-tank process can be described by the following LPV model:

$$\begin{cases} \dot{x}(t) = A(\theta(t))x(t) + Bu(t) \\ y(t) = Cx(t) \end{cases} \quad (122)$$

where the matrices $A(\theta(t))$, B and C are given by:

$$A(\theta(t)) = \begin{pmatrix} -a_{11}\theta_1(t) & 0 \\ a_{11}\theta_1(t) & -a_{22}\theta_2(t) \end{pmatrix};$$

$$B = \begin{pmatrix} b_{11} & 0 \\ 0 & b_{22} \end{pmatrix} \text{ and } C = \begin{pmatrix} 1 & 0 \\ 0 & 1 \end{pmatrix} \quad (123)$$

with, $a_{11} = \frac{c_{12}}{A}$, $a_{22} = \frac{c_2}{A}$ and $b_{11} = b_{22} = \frac{1}{A}$

The matrix $A(\theta(t))$ depends on time-varying parameters vector:

$$\theta(t) = (\theta_1^T(t) \ \theta_2^T(t))^T \quad (124)$$

These two time varying parameters $\theta_1^T(t)$ and $\theta_2^T(t)$ depend on the measured states $x_1(t)$ and $x_2(t)$. So, they are said measurable as it is shown in their expressions:

$$\theta_1(t) = \frac{\sqrt{x_1(t) - x_2(t)}}{x_1(t)} \quad (125)$$

$$\theta_2(t) = \frac{\sqrt{x_2(t)}}{x_2(t)} \quad (126)$$

We consider that the studied process operates with:

$x_1(t) \in [0.6m; 1.8m]$ and $x_2(t) \in [0.1m; 0.3m]$. In this defined operating range, the time-varying parameters are bounded as follows:

$$0.6804m^{-1/2} \leq \theta_1(t) \leq 1.1785m^{-1/2}$$

$$\text{and } 1.8257m^{-1/2} \leq \theta_2(t) \leq 3.1623m^{-1/2} \quad (127)$$

We consider that additive actuator faults occur in the process's pumps such that the control inputs $u_1(t)$ and $u_2(t)$ of the faulty pumps are respectively affected by the faults $f_1(t)$ and $f_2(t)$ which compose the actuator fault vector $f(t) = (f_1^T(t) \ f_2^T(t))^T$. Moreover, an unknown input $d(t)$ with a distribution matrix represented by $R = \begin{bmatrix} 0 \\ 1 \end{bmatrix}$ is considered in this process.

Consequently, the whole process can be modeled by the following polytopic LPV model:

$$\begin{cases} \dot{x}(t) = \sum_{i=1}^4 \rho_i(\theta(t)) (A_i x(t) + B(u(t) + f(t)) + R d(t)) \\ y(t) = Cx(t) \end{cases} \quad (128)$$

where the gain scheduling functions are described as follows:

$$\rho_1(\theta(t)) = \frac{\theta_1(t) - \underline{\theta}_1}{\overline{\theta}_1 - \underline{\theta}_1} \frac{\overline{\theta}_2 - \theta_2(t)}{\overline{\theta}_2 - \underline{\theta}_2} \quad (129)$$

$$\rho_2(\theta(t)) = \frac{\theta_1(t) - \underline{\theta}_1}{\overline{\theta}_1 - \underline{\theta}_1} \frac{\theta_2(t) - \underline{\theta}_2}{\overline{\theta}_2 - \underline{\theta}_2} \quad (130)$$

$$\rho_3(\theta(t)) = \frac{\overline{\theta}_1 - \theta_1(t)}{\overline{\theta}_1 - \underline{\theta}_1} \frac{\theta_2(t) - \underline{\theta}_2}{\overline{\theta}_2 - \underline{\theta}_2} \quad (131)$$

$$\rho_4(\theta(t)) = \frac{\overline{\theta}_1 - \theta_1(t)}{\overline{\theta}_1 - \underline{\theta}_1} \frac{\overline{\theta}_2 - \theta_2(t)}{\overline{\theta}_2 - \underline{\theta}_2} \quad (132)$$

and the matrices $A_{1,...,4}$ are given as follows:

$$A_1 = \begin{pmatrix} -a_{11}\underline{\theta}_1 & 0 \\ a_{11}\underline{\theta}_1 & -a_{22}\underline{\theta}_2 \end{pmatrix} \quad (133)$$

$$A_2 = \begin{pmatrix} -a_{11}\underline{\theta}_1 & 0 \\ a_{11}\underline{\theta}_1 & -a_{22}\overline{\theta}_2 \end{pmatrix} \quad (134)$$

$$A_3 = \begin{pmatrix} -a_{11}\overline{\theta}_1 & 0 \\ a_{11}\overline{\theta}_1 & -a_{22}\underline{\theta}_2 \end{pmatrix} \quad (135)$$

and,

$$A_4 = \begin{pmatrix} -a_{11}\overline{\theta}_1 & 0 \\ a_{11}\overline{\theta}_1 & -a_{22}\overline{\theta}_2 \end{pmatrix} \quad (136)$$

with, $\underline{\theta}_i$ and $\overline{\theta}_i$ are respectively the minimum and the maximum values of time-varying parameter θ_i for all $i = 1, 2$.

The above gain scheduling functions $\rho_1(\theta(t))$, $\rho_2(\theta(t))$, $\rho_3(\theta(t))$ and $\rho_4(\theta(t))$ are plotted in the figure 3.

4.2 Result simulation

Solving the LMI (39) by using the LMI Toolbox give the unknown matrices of the controller (10) and the observer (6) through the expressions (50)-(54). For $\mu = 1$, $\sigma = 2$, $\psi = 2$ and $\beta = 2$, we obtain the matrices K_{Pj} , K_{Ij} , H_j , N_i , L_i , G_i and ϕ_i ($\forall i, j = 1, \dots, 4$) which are used in the simulation.

In the simulation, we consider that the flow to the consumer Q_{N2} is bounded such that:

$$1.3 \times 10^{-4} m^3/s \leq Q_{N2} \leq 3.9 \times 10^{-4} m^3/s \quad (137)$$

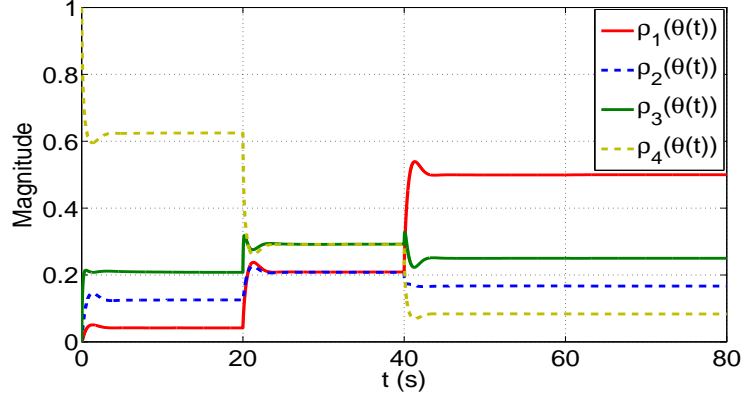


Figure 3: Gain scheduling functions

The references of liquid level in the two tanks are expressed as follows:

$$h_{r1}(t) = \begin{cases} 0.8m & \text{for } 0s \leq t \leq 20s \\ 1.1m & \text{for } 20s < t \leq 40s \\ 1.4m & \text{for } t > 40s \end{cases} \quad (138)$$

and,

$$h_{r2}(t) = \begin{cases} 0.15m & \text{for } 0s \leq t \leq 20s \\ 0.20m & \text{for } 20s < t \leq 40s \\ 0.25m & \text{for } t > 40s \end{cases} \quad (139)$$

The faults $f_1(t)$ and $f_2(t)$ affecting the pumps, are considered as incipient faults such that they can be expressed as follows:

$$f_1(t) = \begin{cases} 0m^3/s & \text{for } 0s \leq t < 10s \\ -1 \times 10^{-4}(t-10)m^3/s & \text{for } 10s \leq t \leq 50s \\ -4 \times 10^{-3}m^3/s & \text{for } t > 50s \end{cases} \quad (140)$$

and

$$f_2(t) = \begin{cases} 0m^3/s & \text{for } 0s \leq t < 30s \\ -1 \times 10^{-4}(t-10)m^3/s & \text{for } 30s \leq t \leq 60s \\ -3 \times 10^{-3}m^3/s & \text{for } t > 60s \end{cases} \quad (141)$$

The unknown input $d(t)$ disturbing the second pump through the distribution matrix R , is assumed as a rectangular signal such that:

$$d(t) = \begin{cases} 1 \times 10^{-3}m^3/s & \text{for } 15s \leq t \leq 25s \\ 0m^3/s & \text{elsewhere} \end{cases} \quad (142)$$

The above level references $h_{r1}(t)$ and $h_{r2}(t)$, the faults $f_1(t)$ and $f_2(t)$ and the unknown input $d(t)$ are applied on the closed loop of the two-tank process whose simulation gives the following results.

The control inputs $u_{f1}(t)$ and $u_{f2}(t)$ described by the adaptive PI control law (10) and provided by the proposed active fault tolerant controller for faulty system, are compared to their for healthy system (under no fault) and illustrated in the figure 4.

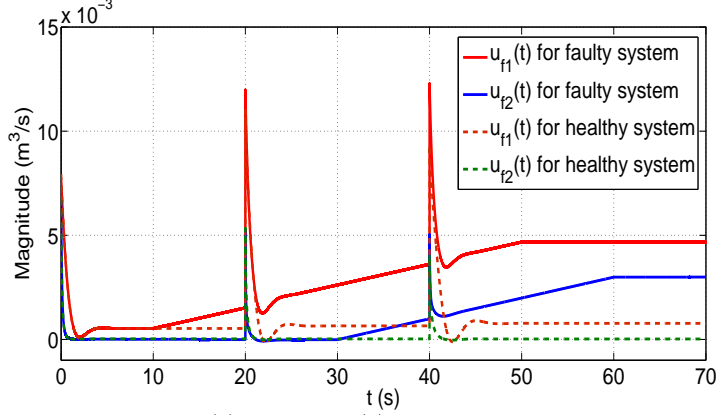


Figure 4: Comparison between the AFTC $u_{f1}(t)$ and $u_{f2}(t)$ for faulty system and their for healthy system

Note that the magnitudes of the control inputs $u_{f1}(t)$ and $u_{f2}(t)$, for the faulty system, start increasing when the faults occur. And from the time instants 50s and 60s, respectively $u_{f1}(t)$ and $u_{f2}(t)$ remain constant. These ones show the fault effects compensation.

In this paper, we compare our results to the FTC approach presented in [1], where the control law is given by:

$$u_f(t) = \sum_{i=1}^h \rho_i(\theta(t)) \left(u(t) + K_i(x_r(t) - x(t)) - K_i^f \hat{f}(t) \right) \quad (143)$$

The authors in [1] use a PI observer so as to estimate only constant faults and states. So, in order to apply their developed approach on the studied process where the faults are assumed time-varying, we must replace the PI observer by an APO. This allows us to obtain the LMI (39) and the following matrices \bar{X}_{ijk} , Y_i , Z_k , P and Q :

$$\bar{X}_{ijk} = \begin{pmatrix} \Pi_i & B_i W_j & 0 & B_i K_j^f \\ * & \bar{\Omega}_{ij} & 0 & -B_i K_j^f \\ * & * & \Theta_i & B_i \\ * & * & * & \Sigma_{ik} \end{pmatrix} \quad (144)$$

$$Y_i = \begin{pmatrix} 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 \\ 0 & -\Phi_i C & 0 & 0 \end{pmatrix} \quad (145)$$

$$Z_k = \begin{pmatrix} 0 & 0 & 0 & 0 \\ 0 & 0 & \Xi_k & 0 \\ 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 \end{pmatrix} \quad (146)$$

$$P = \begin{pmatrix} X_1^T & 0 & 0 \\ 0 & X_2^T & 0 \\ 0 & 0 & X_3^T \\ 0 & 0 & 0 \end{pmatrix} \quad (147)$$

and,

$$Q = \begin{pmatrix} \frac{3}{\mu}Q_1 & 0 & 0 \\ 0 & \frac{2}{\mu}Q_2 & 0 \\ 0 & 0 & \frac{2}{\mu}Q_3 \\ 0 & 0 & 0 \end{pmatrix} \quad (148)$$

where,

$$\bar{\Omega}_{ij} = A_i X_2 + X_2 A_i^T - B_i W_j - W_j^T B_i^T \quad (149)$$

and,

$$W_j = K_j X_2 \quad (150)$$

The comparison results are illustrated in the figure 5 and the figure 6.

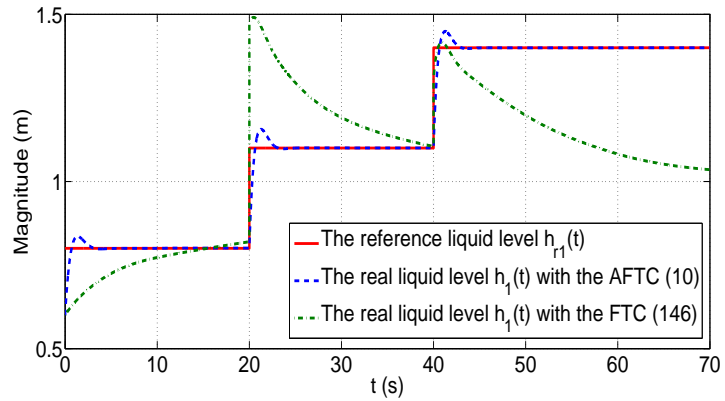


Figure 5: Comparison between the real liquid level $h_1(t)$ in the cases with the proposed AFTC (10) and with the FTC (143) and its reference $h_{r1}(t)$

The figures (figure 5 and figure 6) show that the real liquid levels of the two tanks controlled by the proposed AFTC well follow their references level in spite of the presence of faults and disturbance. In addition, the fault effects are successfully compensated by the designed control law (10). These results show the effectiveness of the proposed method that allows to ensure stability, to track reference levels and to tolerate faults.

However, the FTC (143) given by the approach developed in [1], cannot keep track the reference levels. Almost from the time 40s, the real liquid levels $h_1(t)$ and $h_2(t)$ obtained with the FTC (143) diverge. This proves that the approach of (Aouaouda et al,) is limited for the cases of the systems affected by a certain type of faults.

The proposed APO structure presented by (6) is used to estimate the magnitudes of the faults on-line. The provided informations are necessary to reconfigure the control law. Comparing the original faults and their estimations given by the proposed observer so as to show its effectiveness. The comparison results are presented in the figure 7 and the figure 8. Figures (figure 7 and figure 8) show that the designed APO estimates perfectly the original applied faults.

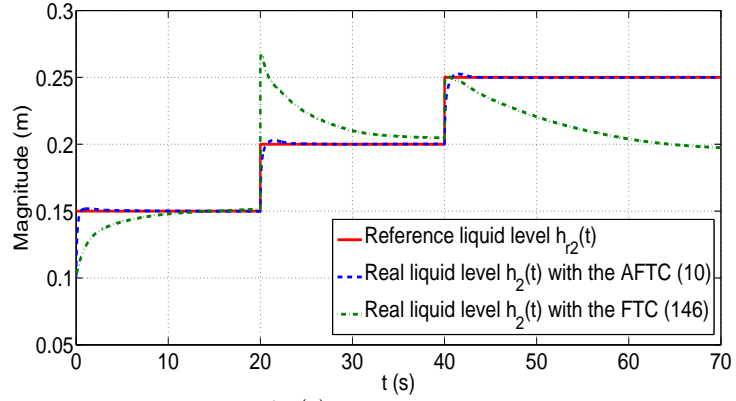


Figure 6: Comparison between the real liquid level $h_2(t)$ in the cases with the proposed AFTC (10) and with the FTC (143) and its reference $h_{r2}(t)$

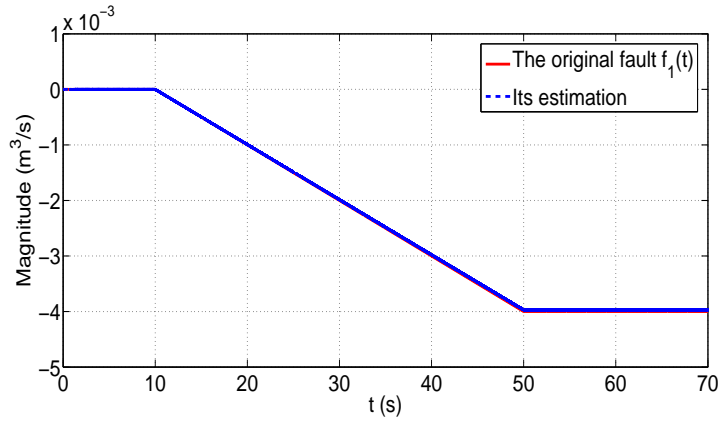


Figure 7: Comparison between the original actuator fault $f_1(t)$ and its estimation by using the APO

5 Conclusion

In this work, the problem of model reference tracking for polytopic LPV systems with measurable gain scheduling functions has been treated. The proposed scheme consists to design a closed loop system based on an active fault tolerant controller, an Adaptive Polytopic Observer (APO) and a reference model. Such controller ensures good performances like stability, trajectory tracking, accuracy and faults compensation. The controller and the APO gains are obtained by solving a set of linear matrices inequalities (LMI). A comparison with others previous methods underline the relevant results obtained through this new method.

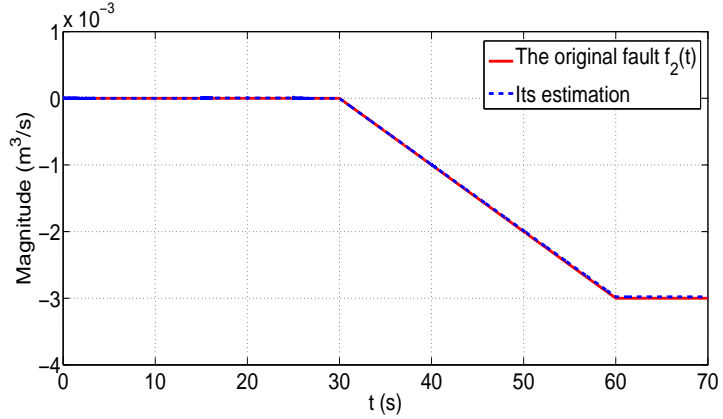


Figure 8: Comparison between the original actuator fault $f_2(t)$ and its estimation by using the APO

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