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ON THE NONLINEAR DYNAMICS OF A BI-STABLE PIEZOELECTRIC ENERGY HARVESTING DEVICE

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Abstract. *This work deals with the dynamics of a nonlinear piezoelectric energy harvesting device, intending to map configuration of parameters able to provide chaotic and non-chaotic response behavior. The dynamics in explored changing forcing amplitude and excitation frequency. Bifurcation diagrams and basins of attraction are computed, and their analysis allow to identify regions of chaotic and regular dynamics.*

Keywords: *non-linear dynamics, energy harvesting, bi-stable system, piezo-magneto-elastic beam, chaos*

1. INTRODUCTION

Energy harvesting is a process in which a small amount of energy from an abundant source is collected via a device called harvester, stored into a battery or capacitor, and then used by another device with low power demands (Priya and Inman, 2009; Caliò *et al.*, 2014). Although conceptually simple, this technology is capable of providing energetic solutions to several interesting applications that goes from nanosystems (Koka *et al.*, 2014; Seol *et al.*, 2013; Wang and Song, 2006) to wireless networks (Kausar *et al.*, 2014).

Among all the energy harvesting devices that exist today, those based on bi-stable configurations (Cottone *et al.*, 2009; Erturk *et al.*, 2009) are among the most promising from the efficiency point of view, being widely studied in the literature (Leite *et al.*, 2016; Peterson *et al.*, 2016; Lopes *et al.*, 2017).

This work intend to study in deep the nonlinear dynamics of the piezo-magneto-elastic energy harvesting device proposed by Erturk *et al.* (2009). For this purpose, the dynamical system is investigated by means of bifurcation diagrams and basins of attraction, seeking to analyze the qualitative nature of the system response, to identify configurations associated to chaotic behavior and sensibility to the initial conditions.

Next section is devoted to physical and mathematical modelling; third one brings numerical experiments and results discussions. Finally, last section presents work main conclusions and future perspectives.

2. NONLINEAR DYNAMICAL SYSTEM

2.1 Physical model

The physical system of interest in this work is depicted in Figure 1. It consists of a rigid base, where it is fixed a ferromagnetic beam undergoing lateral vibrations, driven by an external periodic force. A pair of magnets, placed in lower part of the rigid base, induces large amplitude movements. At the top of structure, next to beam fixed edge, a pair of piezoelectric material plates is responsible for conversion of mechanical energy into electrical potential.

2.2 Mathematical model

The dynamic of the system of interest is described by the following system of ordinary differential equations

$$\ddot{x} + 2\xi\dot{x} - \frac{1}{2}x(1 - x^2) - \chi v = f \cos \Omega t, \quad (1)$$

$$\dot{v} + \lambda v + \kappa\dot{x} = 0, \quad (2)$$

where x represents beam's extreme displacement, ξ , the mechanical damping ratio, χ , a piezoelectric coupling term in mechanical equation, f , the amplitude of excitation, Ω , the forced excitation frequency, and v , the output voltage;

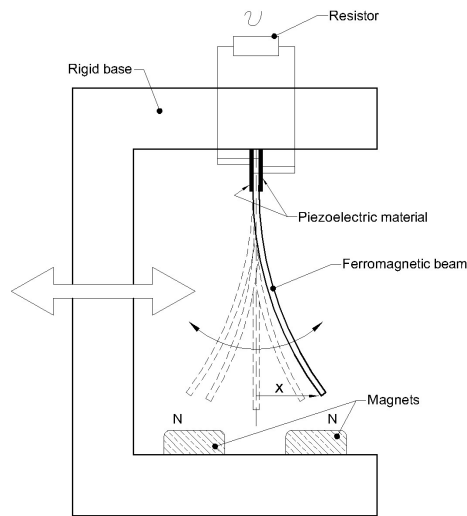


Figure 1. Schematic representation of the piezo-magneto-elastic energy harvesting device (Erturk *et al.*, 2009).

in electrical circuit equation, κ means a piezoelectric coupling term and, finally, λ is a reciprocal time constant. All parameters are dimensionless, assumed initially as $\Omega = 0.8$, $\xi = 0.01$, $\chi = 0.05$, $\kappa = 0.5$ and $\lambda = 0.05$.

The forcing amplitude interval and initial conditions for displacement, velocity and voltage, respectively denoted by x_0 , \dot{x}_0 and v_0 , are specified in next section.

3. RESULTS AND DISCUSSION

3.1 Bifurcation diagrams

In this section the dynamics of the system of interest is explored through bifurcation diagrams. Two different methodologies, here referred as forward and backward, are employed to compute these diagrams. Forward ones considers an increasing sequence a control parameter (excitation frequency), while backward a decreasing one. Is assumed a set of 1200 different values uniformly distributed from a minimum frequency of $\Omega = 0.3$ until a limit value of $\Omega = 1.4$. In all the cases initial conditions are $x_0 = 1$, $\dot{x}_0 = 0$ and $v_0 = 0$, and the first ninety percent of the time-series is neglected (to avoid the transient).

In the Figure 2 the reader can see bifurcations diagrams which shows the beam displacement as function of the excitation frequency, for a fixed excitation amplitude of $f = 0.05$. Computing sequence clearly interferes in results obtained: an bifurcation can be observed only in backward case, between $\Omega = 0.72$ and $\Omega = 0.76$. Despite of this, the system response remains regular, in both situations, for entire interval.

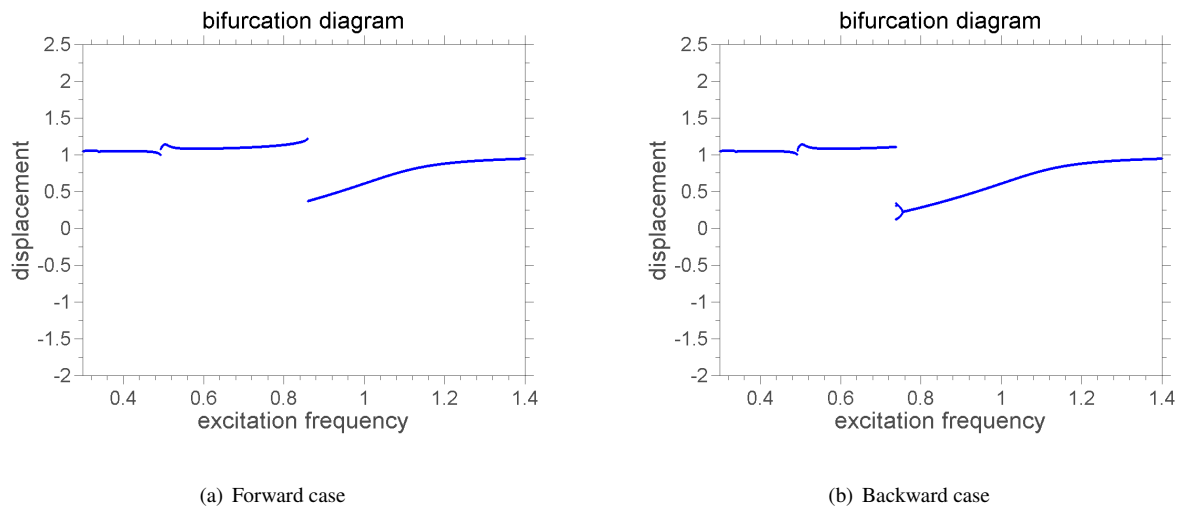


Figure 2. Bifurcation diagrams for displacement as function of excitation frequency with $f = 0.050$.

In the Figure 3 are presented other two bifurcation diagrams, now, for $f = 0.083$. A chaotic dynamic, represented by a blurred region, can be observed both forward and backward cases nearby $\Omega = 0.8$. Although, regular pattern between 0.36 and 0.78 presents an opposite behavior in forward case in comparison with backward one; last methodology also allows to detect a small set of regular solutions around $\Omega = 0.4$.

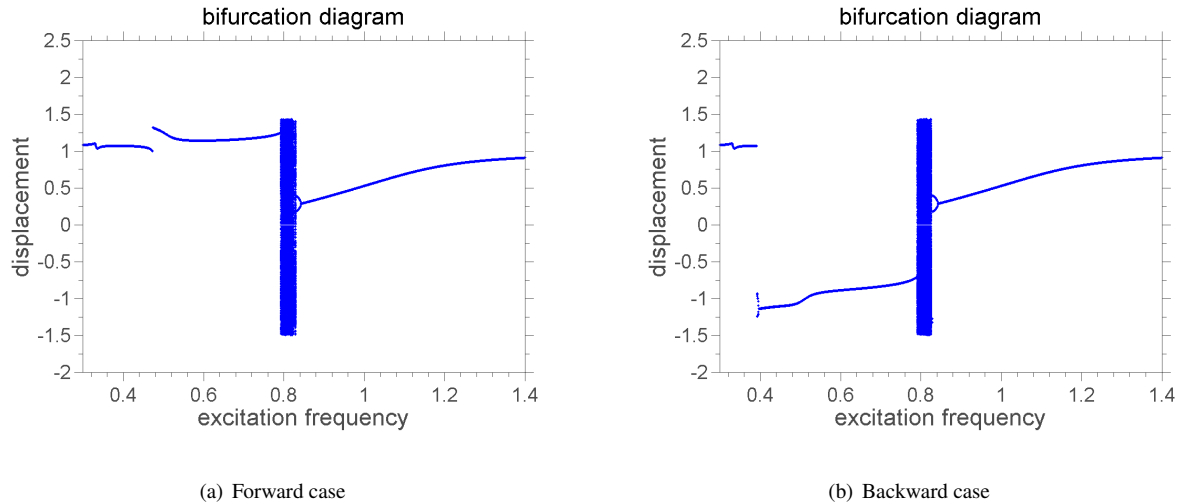


Figure 3. Bifurcation diagrams for displacement as function of excitation frequency with $f = 0.083$.

Figure 4 brings results for $f = 0.1$. Here obtained diagrams change completely from a methodology to another. For entire analyzed interval, forward solution provides a regular dynamics for system response. Discontinuities are observed around $\Omega = 0.44$ and $\Omega = 0.78$. Backward case allows to identify a chaotic region for excitation frequencies from approximately 0.78 to 0.88, while a single discontinuity can be observed around $\Omega = 0.36$.

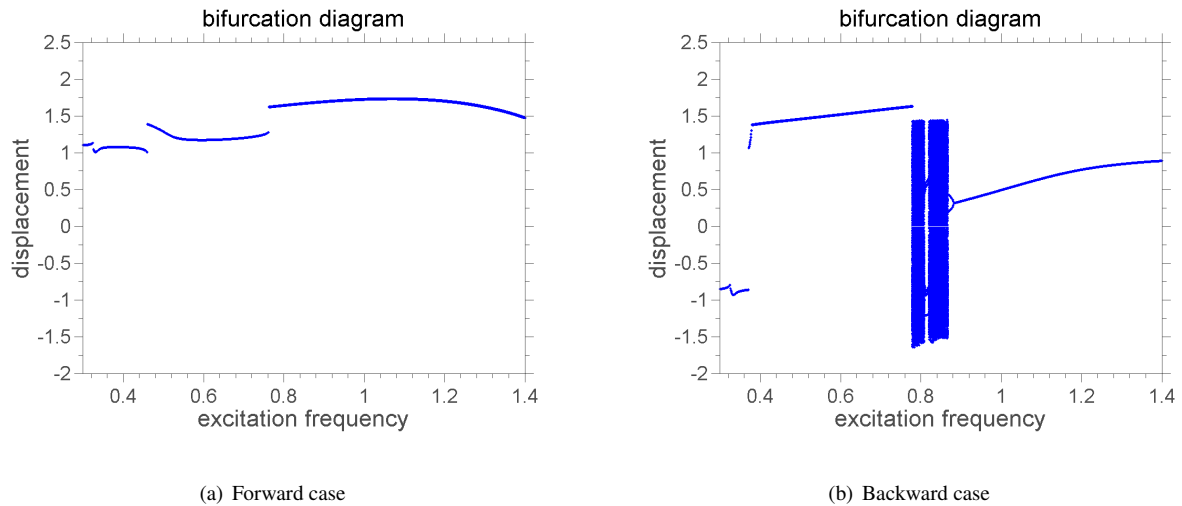


Figure 4. Bifurcation diagrams for displacement as function of excitation frequency with $f = 0.100$.

Finally, the Figure 5 presents diagrams for $f = 0.115$. As in last case, backward methodology presents a richer dynamics regards to chaotic response for same region. A set of points appears nearby $\Omega = 0.36$, representing multiples solutions for this excitation amplitude values region. Forward case diagram shape does not presents significant changes beyond discontinuities position in comparison with that obtained for $f = 0.100$.

Results present in Figures 4 and 5 reinforce system dynamics complexity: for the same excitation frequency and forcing amplitudes values, in each case, obtained diagrams change significantly depending on the way analysis interval is evaluated. Even in the earlier cases, presented by Figures 2 and 3, small dynamics phenomena are captured, in general, for backward analysis while forward suppresses them.

3.2 Basins of attraction

This section presents results of the analysis of basins of attraction in displacement versus velocity plane using an uniform 1400 x 1400 grid of points.

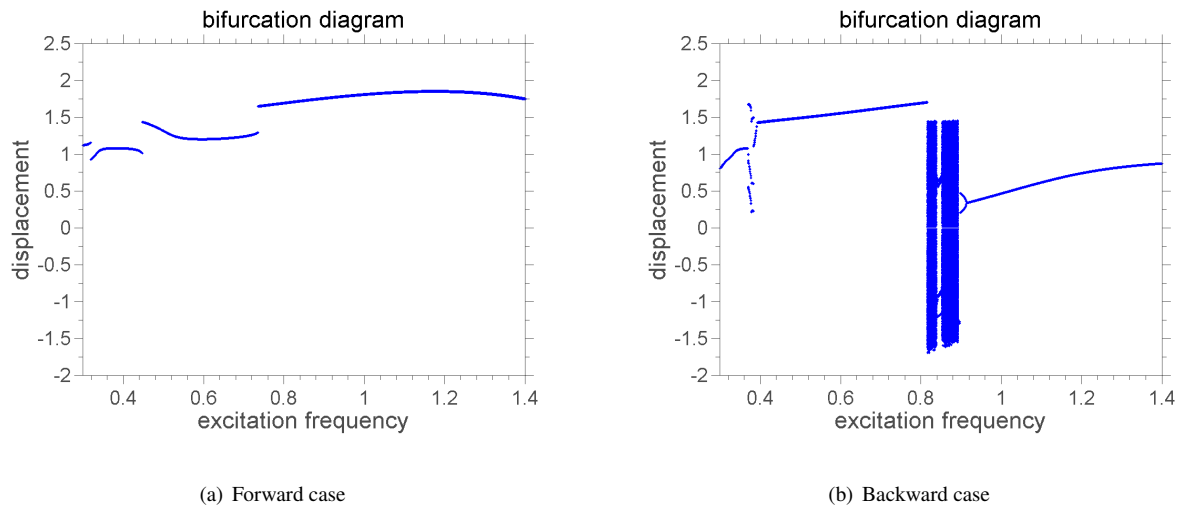


Figure 5. Bifurcation diagrams for displacement as function of excitation frequency with $f = 0.115$.

In Figure 6 the reader can observe the basins of attraction for $f = 0.05$ and the corresponding attractors, which corresponds to regular behavior. It is important to notice that a minimum change in the set of initial conditions can lead to a completely different attractor, characterizing deep sensitivity to these parameters.

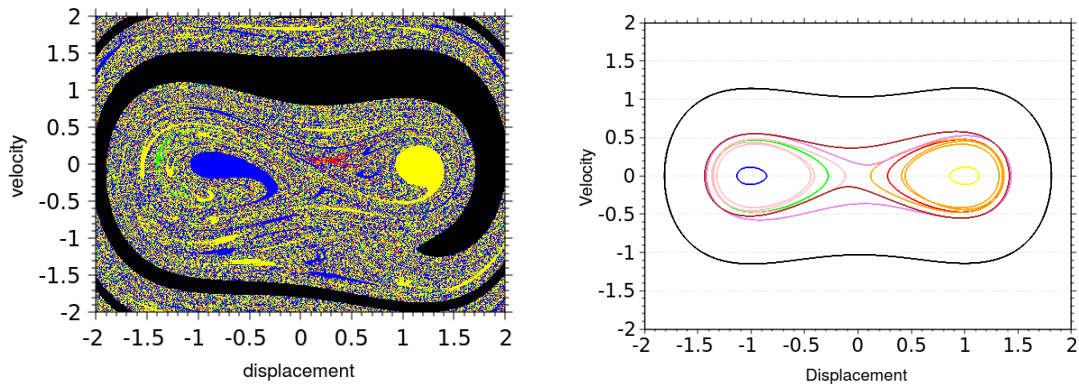


Figure 6. Basins of attraction in displacement versus velocity plane for $f = 0.05$ and the corresponding attractors.

The pictures in Figure 7 are clippings from the basins presented in Figure 6, trying to characterize the red structure seen below. It is easy to see that the aforementioned structure is very well delimited.

On the other hand, Figure 8 shows basins of attraction for different excitation amplitudes ($f = 0.083$, $f = 0.1$ and $f = 0.115$). In this case, the initial conditions painted white lead to chaotic attractors, whereas the ones painted black to a regular attractor.

4. FINAL REMARKS

This work have analyzed the nonlinear dynamics of an bi-stable energy harvesting device by means of bifurcation diagrams and basins of attraction. The dynamics behavior was addressed in a way to characterize the sensibility of the system response to the initial conditions and the influence of excitation parameters over chaos incidence. Bifurcation diagrams were built under two distinct methodologies for a set of four different forcing amplitude values, demonstrating dynamics complexity and results sensibility. Basins of attraction were computed for the same forcing amplitude values, allowing to identify initial conditions sets of values which provides chaotic responses. In the future work, the authors intend to investigate in deep the nonlinear structure of the dynamical system and implement power electronics technics to improve delivered energy quality and output power.

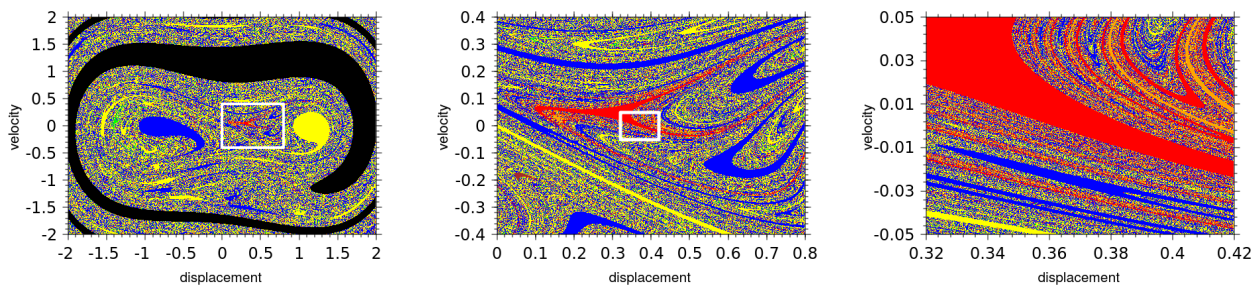


Figure 7. Clippings of the basins of attraction in displacement versus velocity plane for $f = 0.05$.

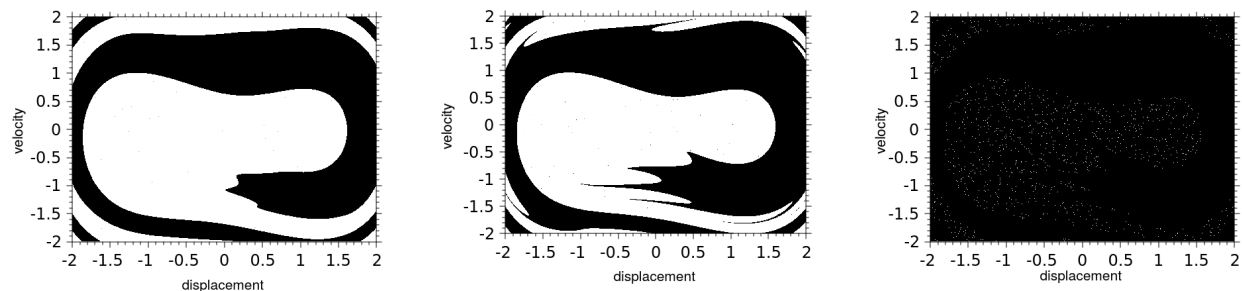


Figure 8. Basins of attraction in displacement versus velocity plane for different excitation amplitudes: $f = 0.083$ (left), $f = 0.1$ (right) and $f = 0.115$ (bottom).

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