



# The viscous curtain: General formulation and finite-element solution for the stability of flowing viscous sheets

C Perdigou, Basile Audoly

## ► To cite this version:

C Perdigou, Basile Audoly. The viscous curtain: General formulation and finite-element solution for the stability of flowing viscous sheets. *Journal of the Mechanics and Physics of Solids*, 2016, 96, pp.291 - 311. 10.1016/j.jmps.2016.07.015 . hal-01431041

**HAL Id: hal-01431041**

**<https://hal.science/hal-01431041>**

Submitted on 10 Jan 2017

**HAL** is a multi-disciplinary open access archive for the deposit and dissemination of scientific research documents, whether they are published or not. The documents may come from teaching and research institutions in France or abroad, or from public or private research centers.

L'archive ouverte pluridisciplinaire **HAL**, est destinée au dépôt et à la diffusion de documents scientifiques de niveau recherche, publiés ou non, émanant des établissements d'enseignement et de recherche français ou étrangers, des laboratoires publics ou privés.

# The viscous curtain: general formulation and finite-element solution for the stability of flowing viscous sheets

C. Perdigou<sup>a</sup>, B. Audoly<sup>a,b</sup>

<sup>a</sup>*UPMC Univ Paris 06, CNRS, UMR 7190, Institut Jean Le Rond d'Alembert, F-75005 Paris, France*

<sup>b</sup>*LMS, École Polytechnique, CNRS, Université Paris-Saclay, 91128 Palaiseau, France*

---

## Abstract

The stability of thin viscous sheets has been studied so far in the special case where the base flow possesses a direction of invariance: the linear stability is then governed by an ordinary differential equation. We propose a mathematical formulation and a numerical method of solution that are applicable to the linear stability analysis of viscous sheets possessing no particular symmetry. The linear stability problem is formulated as a non-Hermitian eigenvalue problem in a 2D domain and is solved numerically using the finite-element method. Specifically, we consider the case of a viscous sheet in an open flow, which falls in a bath of fluid; the sheet is mildly stretched by gravity and the flow can become unstable by ‘curtain’ modes. The growth rates of these modes are calculated as a function of the fluid parameters and of the geometry, and a phase diagram is obtained. A transition is reported between a buckling mode (static bifurcation) and an oscillatory mode (Hopf bifurcation). The effect of surface tension is discussed.

*Keywords:* A. Buckling, B. Plates, C. Stability and bifurcation

---

## 1. Introduction

The buckling of viscous sheets is involved in a number of phenomena encompassing diverse length and time scales. In earth sciences, viscous shell models have been used to capture the periodic buckling of the subducted lithosphere on geological time scales by Ribe et al. (2007). A number of industrial processes involve thin viscous layers undergoing large deformations: in the float glass process introduced by Pilkington (1969), for example, a thin layer of molten glass floating on a bath of tin is stretched to produce thin glass panels, and the process is controlled so as to avoid wrinkling instabilities (Pfingsttag et al., 2011). In the food industry, the folding instability of a viscous sheet, such as hot chocolate or liquid ice cream, impinging a surface is commonly used to produce regular, decorating folds—a technique well-known to pastry enthusiasts. In fluid mechanics, the burst of a bubble near the surface of a viscous fluid involves the collapse of a thin spherical viscous film; the wrinkles formed by this collapse have been studied by Debrégeas (1998); da Silveira et al. (2000). A similar experiment has been studied by Boudaoud and Chaïeb (2001) where a small viscous disk at the surface of water is poked and forms wrinkles due to hoop stresses.

A classical experiment involving a sheared viscous sheet in an axisymmetric geometry has provided a powerful motivation for subsequent theoretical work on the stability on thin viscous shells. Taylor (1958) first experimented with a thin layer of viscous fluid floating on a denser fluid in an annular region enclosed between a rotating inner disk and a fixed outer cylinder, and reported wrinkles in the viscous sheet produced by the shear flow. A number of studies have analyzed this specific geometry in more detail, with the aim of characterizing the instability. Suleiman and Munson (1981) proposed a quantitative version of the experiment, and identified the dimensionless shear stress governing buckling. Benjamin and Mullin (1988) argue that thin shell models are inapplicable for this particular problem, and analyze the instability based on the equations for 3D viscous flows. Slim et al. (2012) simplify the geometry by addressing an infinite viscous strip sheared by lateral walls; they modify the thin viscous shell model to include an advection

term, and argue that the modified model captures the wrinkling instabilities more accurately. Bhattacharya et al. (2013) analyze again the stability of a sheared annular floating viscous sheet, based on a thin shell model; while they obtain good agreement with their experiments in some range of the parameters, they also point out the shortcomings of the shell model which for very thin sheets predicts very short wavelength instabilities, and a vanishingly small buckling threshold. Ribe (2012) emphasizes that even the inclusion of surface tension would not cure these shortcomings.

These stability analyses are all concerned with base flows possessing a direction of invariance, such as the axisymmetric geometry of Taylor (1958) or the infinite strip of Slim et al. (2012). In the presence of such symmetries the stability is essentially a 1D problem; it can be expressed mathematically as an eigenvalue problem for an ordinary differential equation. In this paper, we address the general case of a base flow possessing no direction of invariance: the eigenvalue problem governing stability is then formulated on a genuinely 2D domain and it involves partial differential equations. A numerical solution of the linear stability analysis is proposed based on the finite element method. To do so, we build up on the methods used for analyzing the stability of elastic plates, a mature field in which non-invariant base solutions are routinely addressed, see the work of Friedl et al. (2000); Zheng (2009); Healey et al. (2013); we also build up on the classical numerical methods for the stability dissipative bulk materials (Triantafyllidis and Leroy, 1994). A specificity of the annular geometry studied in previous work is that the fluid trajectories are closed and periodic; by contrast, we address the more general case of an open flow, *i.e.* with fluid elements continuously entering and exiting the domain.

The main goal of this paper is to introduce the mathematical formulation and the simulation tools that allow one to address the stability of thin viscous sheets in arbitrary geometries. Viscous sheets have been much less studied than one-dimensional viscous filaments, for which mechanical models, stability analyses and numerical methods are available (Entov and Yarin, 1984; Teichman and Mahadevan, 2003; Ribe, 2004; Bergou et al., 2010; Arne et al., 2015).

As an illustration of viscous flows possessing no direction of invariance, we propose and analyze a problem inspired by a glass forming process patented by Dockerty (1967). A gutter is filled with molten glass, which overflows and forms a thin sheet of glass stretched by gravity. The glass cools down and glass panels are cut off at the bottom. Accordingly, we study the flow in a thin viscous sheet produced by injecting fluid at a horizontal slit with prescribed velocity, the sheet being stretched by gravity as it falls in a bath of fluid placed below, see figure 1. We ignore thermal effects and assume that the fluid viscosity is homogeneous. As the sheet flows, an in-plane viscous stress appears. We analyze the possibility that the sheet buckles when the stress is compressive. This system is therefore the viscous analogue of the well-studied wrinkled elastic membrane under traction, see the work of Friedl et al. (2000); Cerda et al. (2002); Zheng (2009); Healey et al. (2013).

Our stability analysis is based on the viscous sheet model, a 2D model for the viscous fluid akin to plate models for elastic materials. For this 2D model to be applicable, the velocity and thickness in the base state must vary over a length scale much larger than the thickness, and the velocity must be nearly constant across the thickness. The derivation of viscous sheet models from the 3D equations for Stokean fluid is well documented in the literature; a short review is given in §2 below.

The outline of the paper is as follows. In §2 the viscous sheet model is introduced; equivalent formulations of the equations for the balance of viscous stress are proposed, both in strong form (local stress balance) and in weak form (principle of virtual work). In §3, these equations are solved in a planar geometry assuming that the flow is stationary: this yields the base state of our viscous buckling problem. The buckling modes are obtained numerically in §4 by a linear stability analysis. In §5, the effect of surface tension is included and its stabilizing effect on the curtain modes is discussed.

## 2. Model for a falling viscous sheet

In this section we present the mathematical model governing the motion of a thin viscous sheet poured from a horizontal slit, which is stretched by gravity and falls in a bath of fluid. The motion in the sheet is governed by the balance of viscous stress, including both stretching and bending stress, as well as by gravity

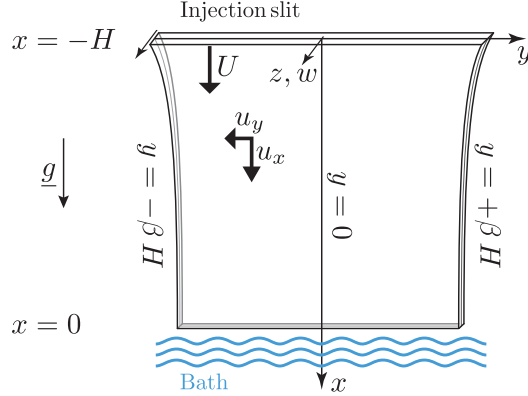


Figure 1: Thin viscous sheet stretched by gravity and falling into a bath of fluid.

and surface tension. The general, non-linear equations of motion derived in this section will be specialized to the case of a steady planar flow for the calculation of the base solution in §3, and later linearized near this base solution in §4.

### 2.1. Notations and assumptions

We use Cartesian coordinates  $(x, y, z)$ , as depicted in figure 1: the  $x$  axis is in the vertical direction and oriented downwards, the  $y$  axis is along the direction of the slit (*i.e.* across the width of the sheet), the  $z$  axis is perpendicular to the sheet. The orthonormal Cartesian basis is denoted by  $(\underline{e}_x, \underline{e}_y, \underline{e}_z)$ . We shall use a two-dimensional model for the sheet based on the theory of thin viscous plates: at time  $t$ , a generic point on the midsurface of the sheet is labelled through its coordinates  $(x, y) \in \mathcal{D}_t$  where  $\mathcal{D}_t$  is the domain obtained by projecting the mid-surface of the sheet onto the  $(x, y)$  plane. For the thin sheet model,  $(x, y)$  are Eulerian coordinates.

We shall assume throughout this paper that the effect of viscosity is dominant in the sense that velocities remain everywhere close to the injection velocity  $U \underline{e}_x$  imposed at the top slit, see equation (17a) below. As a result, the lateral edges are almost vertical and the width of the sheet is almost constant. Under this assumption of dominant viscosity, one can therefore approximate<sup>1</sup> the Eulerian domain with a rectangle that is independent of time,

$$\mathcal{D}_t = [-H, 0] \times [-\beta H, \beta H]. \quad (1)$$

Here, we denote by  $H$  the height of the sheet, *i.e.* the distance from the slit to the bath, and by  $\beta H$  its half-width, *i.e.* half the length of the slit: the quantity  $\beta$  is half the aspect-ratio of the sheet.

The top of the sheet, where the fluid is injected through a slit, corresponds to  $x = -H$ . The bottom, where the fluid makes contact with the bath at rest, is  $x = 0$ . The axis of the sheet is  $y = 0$  and the lateral edges are  $y = \pm\beta H$ . When symmetries allow, we will restrain our analysis to the half-domain  $(x, y) \in [-H; 0] \times [0; \beta H]$  and apply appropriate symmetry conditions on the axis  $y = 0$ .

Unless stated otherwise, all unknowns are functions of  $(t, x, y)$ , even though the dependence on time  $t$  is sometimes implicit for the sake of conciseness. Bars below symbols denote tensors, with the number of bars matching their rank (a single bar for vectors such as  $\underline{e}_x$ , two for matrices, etc.) A comma in a subscript indicates partial derivation, as in  $\phi_{,x} = \partial\phi/\partial x$ . We use Einstein's summation convention, whereby

<sup>1</sup>A similar simplification arises in problems of linearized elasticity, where there is no need to distinguish between reference and actual configurations in the arguments of unknown functions. In the viscous case problem studied here, there is no need to distinguish between the actual position of a particle and the position it would have arrived at if it had been advected by the constant and stationary velocity field  $U \underline{e}_x$ .

a summation is implied whenever repeated indices appear on the same side of an equality. Greek letters will be used for indices belonging in plane of the sheet:  $x_\alpha$ , for example, means  $x$  or  $y$ , but not  $z$ .

Viscous sheet models can be justified by dimensional reduction, starting from the Stokes equations for viscous fluids in 3D in the limit  $h \ll H$  where the thickness  $h$  is small compared to the size  $H$  of the sheet, see for instance Howell (1996); Ribe (2001, 2002); Pfingstag et al. (2011). Different models are applicable to viscous sheets, corresponding to different sets of scaling assumptions. One set of scaling assumptions yields the Trouton model which holds on long time scales, and describes the evolution of the thickness  $h$  as a result of stretching; see Filippov and Zheng (2010) for an application. Other sets of assumptions yield a family of models, known as Buckmaster-Nachman-Ting (BNT) models after the work of Buckmaster et al. (1975); Buckmaster and Nachman (1978): BNT models describe the bending of viscous sheets occurring on much shorter time scales. The model we use in this paper is akin to the classical small-deflection BNT model describing the incipient buckling of viscous sheets, as derived by Howell (1996) and applied by Pfingstag et al. (2011) and others. More accurately, the viscous sheet model we use in this paper is identical to the model derived in §2.2 of the paper by Howell (1996) and by Pfingstag et al. (2011) with  $m = 1$  in their notation; in the present work, however, this model is derived from different scaling assumptions than in the work of Howell and others. These differences are discussed at the end of this paper, in §6.

On the time and force scales addressed by the BNT model, the evolution of thickness is negligible. Our buckling analysis is based on a model in the BNT family. Accordingly, the thickness  $h$  is constant along trajectories. For the sake of simplicity<sup>2</sup>, we assume that the sheet is injected at the top slit with a uniform thickness: as a result, the thickness  $h$  is uniform in the entire domain.

In this paper, the evolution of the sheet is parameterized by the deflection  $w(x, y, t)$  of the sheet's mid-surface (where the deflection is defined as the  $z$ -component of the current position), and the in-plane projection of the velocity  $\underline{u}(x, y, t) = u_x(x, y, t)\underline{e}_x + u_y(x, y, t)\underline{e}_y$ . A similar set of kinematical variables is used in the dimensional reduction of Howell (1996). This choice of kinematical variables is well suited to buckling analyses, as we show. As we start from a dimensionally reduced model, the viscous sheet is represented as a surface, parameterized by the Eulerian variables  $(x, y)$  and by the time  $t$ , whose motion in the 3D Cartesian space is described by its in-plane velocity  $\underline{u}$  and out-of-plane deflection  $w$ . It is important to note that we use a *velocity*  $\underline{u} = (u_x, u_y)$  in the in-plane direction but a *displacement*  $w$  in the transverse direction.

We now proceed to introduce the main dimensionless groups associated with the various forces influencing the motion of the sheet. We do so by proposing a qualitative scaling argument, which will guide us later in rescaling the full equations, see §2.5. The surface tension  $\sigma$  between the liquid and ambient air is assumed to be constant. The volume mass of the sheet is denoted by  $\rho$ , and its dynamic viscosity by  $\mu$ . Let  $g$  be the acceleration of gravity,  $w^*$  the typical deflection,  $h$  the thickness imposed by the slit and  $H$  the height of fall. When expressed per unit volume, the weight and the capillary forces are of order  $\rho g$  and  $(\sigma w^*/H^2)/h$ , respectively (note that the typical curvature of the sheet is  $w^*/H^2$ , which gives rise to a surface density of capillary force  $\sigma w^*/H^2$ ). We compare these forces to typical force arising from viscosity, per unit volume:  $U/H$  has the dimension<sup>3</sup> of a strain rate,  $\mu U/H$  has the dimension of a viscous stress, and  $\mu U/H^2$  has the dimension of a net force per volume (it is given by the divergence of the stress). The ratio of the typical weight to the typical viscous forces is known as Jeffreys' number (Jeffreys, 1925):

$$\text{Je} = \frac{\rho g H^2}{\mu U}, \quad (2)$$

and the ratio of the typical capillary force to the typical viscous force is known as the inverse capillary number,

$$\text{Ca}^{-1} = \frac{2\sigma}{\mu U}. \quad (3)$$

<sup>2</sup>This assumption could be easily relaxed, as noted in §6.

<sup>3</sup>The 'typical' quantities identified here are simply given as a mean to identify the dimensionless groups relevant to our problems, and are not meant to be accurate estimates of the true solution. In fact, as we shall see later, the actual shear rates in the sheet are much smaller than the naïve estimate  $U/H$  proposed above, by a factor of order  $\text{Je} \ll 1$ .

Here, we have assumed that the typical deflection is comparable to the thickness,

$$w^* \sim h. \quad (4)$$

This classical scaling assumption in fact underlies the derivation of the plate model, see for instance §2.2 in the paper of Howell (1996), or Pfingstag et al. (2011).

Let  $\eta$  denote the thickness-to-height ratio of the sheet,

$$\eta = \frac{h}{H}. \quad (5)$$

We assume it to be small,

$$\eta \ll 1, \quad (6)$$

as this a necessary condition for the two-dimensional viscous sheet model to be applicable.

As explained above, we assume that the effect of viscosity is dominant compared to that of gravity and capillary forces: the motion of the sheet is close to a vertical downwards rigid-body motion with the velocity  $U$  imposed by the injection. In other words we consider the asymptotic limit

$$\text{Je} \ll 1, \quad \text{Ca}^{-1} \ll 1. \quad (7)$$

The asymptotic regime considered in this work is only partially specified by (6) and (7): as shown later on, we shall more specifically consider the distinguished limit  $\text{Je} \sim \eta^2$  and  $\text{Ca}^{-1} \sim \eta^3$ , which will follow naturally from the equations of motion of the sheet. In addition, we work in the viscous regime, assuming that Reynolds number is small,  $\text{Re} = \rho U H / \mu \ll 1$ . Accordingly, the motion is governed by a balance of forces, and inertia can be neglected.

## 2.2. Kinematics

Since our choice of kinematic variables is non-standard, we first need to derive the expression of the strain rate tensor  $d_{\alpha\beta}$  in terms of the out-of-plane deflection  $w$  and of the in-plane velocities  $u_\alpha$ . To do so, we consider the advection of an infinitesimal material segment  $\delta \underline{l}$  tangent to the mid-surface of the sheet, whose endpoints have coordinates  $(x, y, w(x, y, t)) = (x_\alpha, w(x_\beta, t))$  and  $(x + \delta x, y + \delta y, w(x + \delta x, y + \delta y, t)) = (x_\alpha + \delta x_\alpha, w(x_\beta + \delta x_\beta, t))$  at time  $t$ . Recall that we use Greek indices such as  $x_\alpha$  and  $x_\beta$  for both in-plane coordinates  $x$  and  $y$ ; by a slight abuse of notation, we use  $\beta$  as a summation index as there is no risk of confusion with the half aspect ratio  $\beta$ . Let us denote by  $D/Dt$  the convected derivative, *i.e.* the time derivative obtained by following material points. By definition of the tangent velocity, the in-plane coordinates  $(x, y)$  of the first endpoint evolves according to  $\frac{Dx_\alpha}{Dt} = u_\alpha(x_\beta, t)$ . A similar formula yields the evolution of the other endpoint. Taking the difference, we obtain the evolution of the in-plane projection of the infinitesimal segment as  $\frac{D\delta x_\alpha}{Dt} = u_{\alpha,\beta}(x_\gamma, t) \delta x_\beta$ , where we recall that a comma in subscript denotes a partial derivative and that there is an implicit sum over the repeated index  $\beta$ . The out-of-plane projection of the material segment is  $\delta z = w_{,\alpha}(x_\beta, t) \delta x_\alpha$ . Its rate of change is  $\frac{D\delta z}{Dt} = \frac{Dw_{,\alpha}}{Dt} \delta x_\alpha + w_{,\alpha} \frac{D\delta x_\alpha}{Dt} = (w_{,\alpha t} + u_\beta w_{,\alpha\beta}) \delta x_\alpha + w_{,\alpha} u_{\alpha,\beta} \delta x_\beta$ . With these two formulas, it is possible to calculate the rate of change of the squared length of the material segment as

$$\frac{1}{2} \frac{D(\delta l^2)}{Dt} = \delta x_\alpha \frac{D\delta x_\alpha}{Dt} + \delta z \frac{D\delta z}{Dt} = \delta x_\alpha \delta x_\beta [u_{\alpha,\beta} + w_{,\alpha} (w_{,\beta t} + u_\gamma w_{,\gamma\beta} + w_{,\gamma} u_{\gamma,\beta})].$$

To derive the last equality, some dummy indices have been renamed. The symmetric part of the tensor appearing in square brackets is called the membrane strain rate tensor  $\underline{d}$  and satisfies the identity  $\frac{1}{2} \frac{D(\delta l^2)}{Dt} = \underline{\delta l} \cdot \underline{d} \cdot \underline{\delta l}$ . Its components are found by identification with the above equation as:

$$d_{\alpha\beta} = u_{(\alpha,\beta)} + w_{,(\alpha} w_{,\beta)t} + u_\gamma w_{,(\alpha} w_{,\beta)\gamma} + u_{\gamma,(\beta} w_{,\alpha)} w_{,\gamma}, \quad (8a)$$

where parentheses in subscripts mean that the enclosed indices must be symmetrized, as in  $u_{(\alpha,\beta)} = (u_{\alpha,\beta} + u_{\beta,\alpha})/2$  for instance. Note that the right-hand side is the symmetric part of the tensor  $\underline{\nabla} u + \underline{\nabla} w \otimes (\underline{\nabla} w)^\Delta$ ,

where  $\underline{\underline{a}}^\Delta = \frac{D(\underline{\underline{a}})}{Dt} + \underline{\underline{a}} \cdot \underline{\underline{\nabla}} \underline{\underline{u}}$  denotes the lower-convected (covariant) time derivative of a vector: this is a co-rotational time derivative, which confirms that the membrane strain rate tensor  $\underline{\underline{d}}$  is objective. This tensor measures the rate of stretching of the mid-surface of the sheet, and determines the membrane stress via the constitutive laws for viscous stretching, which we will derive in the following section. In the definition (8a) of the membrane strain rate, the first term coming from  $\underline{\underline{\nabla}} \underline{\underline{u}}$  is the classical strain rate tensor for 2D flows, and the other terms coming from  $\underline{\underline{\nabla}} \underline{\underline{w}} \otimes (\underline{\underline{\nabla}} \underline{\underline{w}})^\Delta$  capture in-plane strain caused by the (out-of-plane) deflection: a similar geometrical nonlinearity appears in the Föppl–von Kármán equations for elastic plates, but without any time derivative.

We now proceed to introduce the second measure of strain rate, namely that associated with the rate of change of curvature of the midsurface. The deflection has been assumed to scale like  $w^* \sim h \sim \eta H \ll H$ , implying that the deflection angle stays small,  $w_{,\alpha} \sim w^*/H \sim \eta \ll 1$ . As a result, the curvature can be approximated by the the Hessian of the midsurface, and it reads  $w_{,\alpha\beta}$  in components and  $\underline{\underline{\nabla}} \underline{\underline{\nabla}} w$  in tensor notations. We define the curvature *strain rate* tensor  $\underline{\underline{k}}$  as the co-rotational derivative of the Hessian,  $\underline{\underline{k}} = (\underline{\underline{\nabla}} \underline{\underline{\nabla}} w)^\Delta$ , where we have introduced the lower-convected derivative of a second-order tensor as  $\underline{\underline{a}}^\Delta = \frac{D\underline{\underline{a}}}{Dt} + \underline{\underline{\nabla}} \underline{\underline{u}}^T \cdot \underline{\underline{a}} + \underline{\underline{a}} \cdot \underline{\underline{\nabla}} \underline{\underline{u}}$ . In components, the curvature strain rate is therefore defined as

$$\begin{aligned} k_{\alpha\beta} &= \frac{D(w_{,\alpha\beta})}{Dt} + (\underline{\underline{\nabla}} \underline{\underline{u}}^T)_{\alpha\gamma} (\underline{\underline{\nabla}} \underline{\underline{w}})_{\gamma\beta} + (\underline{\underline{\nabla}} \underline{\underline{w}})_{\alpha\gamma} (\underline{\underline{\nabla}} \underline{\underline{u}})_{\gamma\beta} \\ &= w_{,\alpha\beta t} + u_\gamma w_{,\gamma\alpha\beta} + 2 u_{\gamma,(\alpha} w_{,\beta)\gamma}. \end{aligned} \quad (8b)$$

The first term  $w_{,\alpha\beta t}$  is the partial time derivative, the second term  $u_\gamma w_{,\gamma\alpha\beta}$  makes it convective, and the last term  $2 u_{\gamma,(\alpha} w_{,\beta)\gamma}$  makes it co-rotational with the flow. This last term warrants that the strain rate tensor  $\underline{\underline{k}}$  remains zero if the sheet is passively advected by an in-plane velocity  $\underline{\underline{u}}$  describing a rigid-body rotation. It is included for the sake of completeness even though it plays no role for the linear stability analysis of the particular problem studied here, as confirmed later above equation (26). Finally, we note that other definitions of the co-rotational derivative are possible, such as the Jaumann or upper-convected derivative; these alternative definitions of the tensor  $\underline{\underline{k}}$  coincide with our particular one when the membrane strain rate is zero ( $\underline{\underline{d}} = \underline{\underline{0}}$ ), and this warrants that the corresponding viscous sheet models are all asymptotically equivalent in the limit of small thickness, as both the viscous bending modulus and the membrane strain rate are small in this limit.

The membrane strain rate tensor  $\underline{\underline{d}}$  in (8b) has the dimension of the inverse of a time; the curvature strain rate tensor  $\underline{\underline{k}}$  has the dimension of the inverse of a length, times the inverse of a time.

Having specified the strain rate measures relevant to our viscous sheet model, we proceed to write down the constitutive relations and the expressions for the virtual works.

### 2.3. Constitutive relations for viscous sheets

In a thin plate, the state of stress is described by a membrane stress tensor  $\underline{\underline{N}}$  and bending moment tensor  $\underline{\underline{M}}$ . The constitutive laws for a thin viscous sheet yield these membrane stress and bending moment as a function of the strain rate tensors introduced in the previous section:

$$N_{\alpha\beta} = 2\mu h (d_{\alpha\beta} + \delta_{\alpha\beta} d_{\gamma\gamma}), \quad (9a)$$

$$M_{\alpha\beta} = \frac{\mu h^3}{6} (k_{\alpha\beta} + \delta_{\alpha\beta} k_{\gamma\gamma}). \quad (9b)$$

Here,  $\delta_{\alpha\beta}$  denotes Kronecker's symbol, equal to 1 if the indices are equal and to 0 otherwise. In accordance with the Stokes-Rayleigh analogy (Rayleigh, 1878), the constitutive laws in equations (9) can be identified with the constitutive laws for an elastic plate made of a Hookean, isotropic, incompressible material, with  $\mu$  being the elastic shear modulus, when the membrane and bending strain *rate* tensors are replaced with the corresponding *regular* strain tensors.

The constitutive laws appearing in equations (9) are classical, see for example the work of Howell (1996) and Ribe (2002). More recently, Pfingsttag et al. (2011) has derived them in a form similar to the one used here.

#### 2.4. Virtual work expressions

Before we proceed to introduce dimensionless variables, we need to introduce some mathematical expressions which enter into the principle of virtual work. In terms of three fields  $(\hat{u}_x, \hat{u}_y, \hat{w})$  called the virtual displacements, which are all functions of  $(x, y)$  but not  $t$ , we define two linear forms, known as the internal virtual work and the external virtual work, respectively:

$$\mathcal{W}_i(\hat{u}_\gamma, \hat{w}) = \iint_{\mathcal{D}_t} (-N_{\alpha\beta} \hat{\epsilon}_{\alpha\beta} - M_{\alpha\beta} \hat{w}_{,\alpha\beta}) \, dx \, dy \quad (10a)$$

$$\mathcal{W}_e(\hat{u}_\gamma, \hat{w}) = \iint_{\mathcal{D}_t} (\rho g h \hat{u}_x - 2 \sigma \hat{\epsilon}_{\alpha\alpha}) \, dx \, dy. \quad (10b)$$

Here  $\hat{\epsilon}_{\alpha\beta}$ , known in the context of plate theory as the virtual strain increment, is another linear form reading

$$\hat{\epsilon}_{\alpha\beta}(\hat{u}_\gamma, \hat{w}) = \hat{u}_{(\alpha, \beta)} + w_{, (\alpha} \hat{w}_{, \beta)}. \quad (10c)$$

The motivation for introducing these expressions and the connection with our particular system will be presented later on, see §2.8.

In equation (10c), we remind that parentheses in subscripts indicate symmetrization of the adjacent indices, and that indices following a comma in subscripts enter into partial derivatives.

#### 2.5. Dimensionless variables

Before proceeding to write the equations for the balance of stress, we identify natural scales, similar to those introduced in §2.1, to define dimensionless quantities and rescale the equations of the sheet. The proposed rescaling is still naïve and preliminary: the rescaled equations and the dimensionless variables still depend on the small parameters  $\eta$ ,  $\text{Je}$  and  $\text{Ca}^{-1}$ , in a way which we will elucidate in §4.3 and §5.3. Considering that the aspect-ratio  $\beta$  is a quantity of order 1, we rescale both in-plane coordinates with the height,  $x_\alpha \sim H$ ; for the out-of-plane displacement, we use the scale set by the thickness  $w \sim \eta H$ ; the in-plane velocity is rescaled with the imposed velocity  $u_\alpha \sim U$ , and the time  $t$  scales with the typical advection time  $t \sim H/U$ . We use the first term<sup>4</sup> in the right-hand side of equation (8a) to define the scale for the membrane strain rate tensor,  $d_{\alpha\beta} \sim U/H$ . Based on equation (8b), we introduce the scale for the curvature  $k_{\alpha\beta} \sim \eta U/H^2$ . Using the constitutive laws (9), we finally obtain  $N_{\alpha\beta} \sim \mu U \eta$  and  $M_{\alpha\beta} \sim \mu H U \eta^4$ .

In view of this, we define a new set of dimensionless variables, marked with a prime, by rescaling the original variables using the dynamic viscosity  $\mu$ , the injection velocity  $U$ , the height  $H$  and the aspect-ratio  $\eta$  as follows:

$$\begin{aligned} t &= \frac{H}{U} t' & d_{\alpha\beta} &= \frac{U}{H} d'_{\alpha\beta} \\ (x, y) &= H \times (x', y') & k_{\alpha\beta} &= \frac{\eta U}{H^2} k'_{\alpha\beta} \\ (u_x, u_y) &= U \times (u'_x, u'_y) & N_{\alpha\beta} &= \mu U \eta N'_{\alpha\beta} \\ w &= \eta H w' & M_{\alpha\beta} &= \mu U \eta^4 H M'_{\alpha\beta}. \end{aligned}$$

All rescaled functions take for arguments the rescaled coordinates and time  $(x', y', t')$ , and the spatial domain is now  $(x', y') \in \mathcal{D}' = [-1, 0] \times [-\beta, \beta]$ . The partial derivatives of rescaled functions are to be understood with respect to these rescaled arguments rather than the unscaled ones: for instance,  $d'_{\alpha\beta, \gamma} = \frac{\partial d'_{\alpha\beta}}{\partial x'_\gamma}$  (with a prime in the denominator).

<sup>4</sup>Note that some other terms in this right-hand side scale differently, like  $\eta^2 U/H^2$ . We will resolve this apparent asymptotic inconsistency later. For the moment, we retain all the terms entering in the definition of  $d_{\alpha\beta}$ .



We now carry out a scaling analysis of equation (10c) defining  $\hat{\epsilon}_{\alpha\beta}$ : balancing the two terms in the right-hand side, we find  $\hat{u}_\alpha/H \sim \eta \hat{w}/H$ . In view of this, we rescale the virtual displacements differently in the tangent and transverse directions (and differently than the real displacement):

$$\hat{u}_\alpha = \eta H \hat{u}'_\alpha \quad \hat{w} = H \hat{w}'.$$

Consistently, we rescale the virtual strain increments and the virtual works as

$$\hat{\epsilon}_{\alpha,\beta} = \eta \hat{\epsilon}'_{\alpha,\beta} \quad \mathcal{W}_j = \eta^2 \mu U H^2 \hat{\mathcal{W}}'_j,$$

where  $j \in \{i, e\}$  in an index denoting the type of virtual work under consideration, internal or external.

## 2.6. Dimensionless form of previous equations

From now on, we shall use exclusively the rescaled variables and, for the sake of readability, *we omit all primes*. In terms of the rescaled variables (and with the primes omitted) the definitions (8) of the strain rates read

$$d_{\alpha\beta} = u_{(\alpha,\beta)} + \eta^2 (w_{,(\alpha} w_{,\beta)t} + u_\gamma w_{,(\alpha} w_{,\beta)\gamma} + u_{\gamma,(\alpha} w_{,\beta)} w_{,\gamma}), \quad (11a)$$

$$k_{\alpha\beta} = w_{,\alpha\beta t} + u_\gamma w_{,\alpha\beta\gamma} + 2 u_{\gamma,(\alpha} w_{,\beta)\gamma}, \quad (11b)$$

the constitutive laws (9) read

$$N_{\alpha\beta} = 2 (d_{\alpha\beta} + \delta_{\alpha\beta} d_{\gamma\gamma}), \quad (12a)$$

$$M_{\alpha\beta} = \frac{1}{6} (k_{\alpha\beta} + \delta_{\alpha\beta} k_{\gamma\gamma}), \quad (12b)$$

and the linear forms involved in the principle of virtual work read

$$\mathcal{W}_i(\hat{u}_\gamma, \hat{w}) = \iint_{\mathcal{D}_t} (-N_{\alpha\beta} \hat{\epsilon}_{\alpha\beta} - \eta^2 M_{\alpha\beta} \hat{w}_{,\alpha\beta}) \, dx \, dy \quad (13a)$$

$$\mathcal{W}_e(\hat{u}_\gamma, \hat{w}) = \iint_{\mathcal{D}_t} \left( \text{Je} \hat{u}_x - \frac{\text{Ca}^{-1}}{\eta} \hat{\epsilon}_{\alpha\alpha} \right) \, dx \, dy, \quad (13b)$$

$$\hat{\epsilon}_{\alpha\beta}(\hat{u}_\gamma, \hat{w}) = \hat{u}_{(\alpha,\beta)} + w_{,(\alpha} \hat{w}_{,\beta)}. \quad (13c)$$

Note the dependence on the dimensionless groups  $(\eta, \text{Je}, \text{Ca}^{-1})$  introduced in §2.5.

## 2.7. Kinematic boundary conditions

Three types of boundary conditions are applied on the edges of the rectangular domain  $\mathcal{D}_t$ : (i) kinematic boundary conditions, which involve the velocity  $u_\alpha$  and displacement and which we derive in this section; (ii) natural (or dynamic) boundary conditions, which express the equilibrium of the boundaries in terms of the stress quantities  $N_{\alpha\beta}$  and  $M_{\alpha\beta}$ , and which we derive next from the principle of virtual work (§2.9); (iii) boundary conditions expressing symmetry along the central axis  $y = 0$  (see table 1). The complete list of boundary conditions are summarized in table 1.

Let us first address the kinematic boundary conditions applicable at the top boundary,  $x = -1$ , where the fluid is injected through a slit. There, the velocity is imposed,  $\underline{u}(-1, y, t) = U \underline{e}_x$ , the out-of-plane displacement is restrained,  $w(-1, 0, t) = 0$ , and the rotation is blocked,  $w_{,x}(-1, 0, t) = 0$ . These four conditions are similar to the classical clamping conditions applicable to an elastic plate. A fifth condition must be considered. It arises in general at boundaries which are inlets, *i.e.* which are crossed by fluid material entering into the domain. The additional condition expresses the fact that the sheet keeps the memory of the curvature it had prior to entering into the domain (no such condition exist for closed systems, such as a typical elastic plate, or at boundaries where particles exit the domain, such as the lower boundary  $x = 0$  in our system). To derive this additional condition, suppose that the sheet is planar prior to passing

through the slit, and consider the discontinuity in curvature as it crosses the slit and enters into the domain,  $\llbracket w_{,\alpha\beta} \rrbracket = w_{,\alpha\beta}((-1)^+, y, t)$ . Here  $x = (-1)^+$  denotes a line inside the domain immediately below the slit. Differentiating the clamping condition already established,  $w((-1)^+, y, t) = w_{,x}((-1)^+, y, t) = 0$ , with respect to  $y$ , one can show that  $\llbracket w_{,x\alpha} \rrbracket = 0$ : the only possible discontinuity in curvature is therefore  $\llbracket w_{,xx} \rrbracket$ . This discontinuity is associated with a fast variation of  $w_{,xx}$  as the sheet exits the slit. The instantaneous power dissipated by viscous forces is  $M_{xx} k_{xx}$ . Since  $M_{xx}$  is proportional to  $k_{xx}$  by equation (12b), this power can be re-written as  $k_{xx}^2/3$ . Integrating with respect to time, we estimate the work dissipated by viscous stress as  $\sim \llbracket w_{,xx} \rrbracket^2/(3\delta t)$  during the short time interval  $\delta t$  needed for the sheet to exit the slit (which is the time needed for  $w_{,xx}$  to go from zero to  $\llbracket w_{,xx} \rrbracket$ ). For this work to remain finite in the limit of a thin slit,  $\delta t \rightarrow 0$ , the axial curvature must be continuous across the slit,  $\llbracket w_{,xx} \rrbracket = 0$ . Therefore, we impose a fifth kinematic boundary condition on the top boundary of the domain:  $w_{,xx} = 0$ ; a similar boundary condition is used in the analysis of 1D viscous rods at their injection point in the work of Ribe (2004) and Arne et al. (2015). The five boundary conditions are summarized in the second row of table 1.

On the stressfree lateral boundaries  $y = \pm\beta$ , only natural boundary conditions are applicable. They will be derived later in §2.9.

On the lower boundary  $x = 0$  where the sheet falls in a bath of the same fluid, we assume that the lateral velocity is blocked,  $u_y = 0$ , as the viscosity of the bath is much more effective than in the thin sheet, by a bulk effect. This condition appears in the third row of table 1. We do not impose any other kinematic boundary condition at the bottom: three natural boundary conditions will be derived from the principle of virtual work in §2.9. In the absence of any restraint on the axial velocity  $u_x$ , deflection  $w$  and slope  $w_{,x}$ , they express the cancellation of the axial stress  $N_{xx}$ , the transverse force and the bending moment, see §2.9 and table 1. More accurate boundary conditions could be derived by analyzing the details of the boundary layer that forms where the sheet merges with the bath but this would require coupling with the 3D viscous flow and this is beyond the scope of the present paper. We expect that our boundary conditions are a good approximation at a large scale compared to the size of the boundary layer.

The central axis  $y = 0$  is an axis of symmetry for the domain as the problem is invariant by the mirror symmetry  $y \leftarrow (-y)$ . Later on, we will consider solutions that are either symmetric or anti-symmetric by this mirror symmetry. It is then sufficient to solve the equations on the half-domain  $[-1, 0] \times [0, \beta]$  and enforce symmetry conditions on the axis  $y = 0$ . We shall only consider symmetric planar flows, for which the kinematic condition  $u_y = 0$  holds, but we will more generally both consider symmetric and anti-symmetric buckling modes. The corresponding symmetry conditions that must be enforced read  $w_{,y} = 0$  or  $w = 0$ . These symmetry conditions are summarized in the last row of table 1.

## 2.8. Stress balance in weak form

A weak form of the balance of stress is derived in this section, based on the principle of virtual work. This formulation has the advantage of being compact and offering a natural starting point for numerical simulations using the finite-element method. So far, thin viscous sheets have been mostly studied in the fluid mechanics community, based on the strong form of the stress balance equations: the equivalence of the weak and strong formulations will be established later in §2.9.

The principle of virtual work for our plate states that a distribution of stress  $(N_{\alpha\beta}, M_{\alpha\beta})$  is balanced if, for any virtual displacement  $(\hat{u}_\alpha, \hat{w})$  satisfying the incremental kinematic boundary conditions, the sum of the internal and external virtual work vanishes,  $\mathcal{W}_i + \mathcal{W}_e = 0$ :

$$\forall(\hat{u}_\alpha, \hat{w}) \text{ k.a.}, \quad \iint_{\mathcal{D}_t} \left( -N_{\alpha\beta} \hat{\epsilon}_{\alpha\beta} - \eta^2 M_{\alpha\beta} \hat{w}_{,\alpha\beta} + \text{Je} \hat{u}_x - \frac{\text{Ca}^{-1}}{\eta} \hat{\epsilon}_{\alpha\alpha} \right) dx dy = 0. \quad (14)$$

By ‘incremental kinematic boundary conditions’, we mean the incremental version of the conditions derived in table 1: for instance, the kinematic condition for the real variable  $u_x(-1, y, t) = U$  yields the incremental condition  $\hat{u}_x(-1, y) = 0$  for the virtual displacement (recall that virtual motion does not depend on time by convention). A set of virtual displacements satisfying the incremental kinematic boundary conditions is said to be *kinematically admissible*, hence the abbreviation ‘k.a.’ in the above equation.

<i>Boundary conditions</i>	in-plane bound. cond.'s		out-of-plane bound. cond.'s	
	kinematic	natural	kinematic	natural
lateral edges (stressfree, $y = \pm\beta$ )		$N_{xy} = 0$ $N_{yy} + \frac{\text{Ca}^{-1}}{\eta} = 0$		$N_{xy} w_{,x} + N_{yy} w_{,y}$ $-\eta^2 M_{yx,x} - \eta^2 M_{yy,y}$ $+(\text{Ca}^{-1}/\eta) w_{,y} = 0$ $M_{yy} = 0$
top edge (slit, inlet, $x = -1$ )	$u_x = U$ $u_y = 0$		$w = 0$ $w_{,x} = 0$ $w_{,xx} = 0^\dagger$	
bottom edge (bath, outlet, $x = 0$ )	$u_y = 0$	$N_{xx} + \frac{\text{Ca}^{-1}}{\eta} = 0$		$N_{xx} w_{,x} + N_{xy} w_{,y}$ $-\eta^2 M_{xx,x} - \eta^2 M_{xy,y}$ $+(\text{Ca}^{-1}/\eta) w_{,x} = 0$ $M_{xx} = 0$
symmetric mode (central axis, $y = 0$ )	$u_y = 0$	$N_{xy} = 0$	$w_{,y} = 0$	$N_{xy} w_{,x} + N_{yy} w_{,y}$ $-\eta^2 M_{yx,x} - \eta^2 M_{yy,y}$ $+(\text{Ca}^{-1}/\eta) w_{,y} = 0$
antisymmetric mode (central axis, $y = 0$ )	$u_y = 0$	$N_{xy} = 0$	$w = 0$	$M_{yy} = 0$

Table 1: Summary of boundary conditions applicable on the edges of the rectangular domain. Kinematic boundary conditions are derived in §2.7. Natural boundary conditions are derived in §2.9 by integrating by parts the principle of virtual work. At each edge there are two in-plane boundary conditions overall, corresponding to the in-plane degrees of freedom  $u_x$  and  $u_y$ , and two out-of-plane boundary conditions, corresponding to the deflection and slope degrees of freedom. The top edge includes a third out-of-plane boundary condition for the curvature, labelled with the symbol  $\dagger$ , ensuring that the work done by viscous forces upon entering the domain remains finite, see §2.7. The dashed lines indicate an alternative, depending on whether symmetric or antisymmetric buckling modes are considered.

In equation (14), the virtual strain increment  $\hat{\epsilon}_{\alpha\beta}$  depends not only on the virtual displacements but also on the the current (or real) configuration  $w(x, y)$ , see equation (13c). As a result, the equations for the balance of stress are non-linear. Because of this nonlinearity, the in-plane and out-of-plane directions are coupled, much like in the Föppl–von Kármán equations for elastic plates.

Let us now proceed to justify the expressions of the virtual works. The internal virtual work  $\mathcal{W}_i$ , which first appeared in equation (10a), is applicable to both thin viscous sheets and thin elastic plates: it is geometric in essence. The virtual strain increment in equation (10c) retains the nonlinearity associated with the transverse displacement only, as does the Föppl–von Kármán model for thin plates: this is the key nonlinearity responsible for buckling, and it is important for moderate slopes. The same approximation of moderate slopes has been used to justify the approximation of the curvature tensor introduced in equation (8b).

The external virtual work  $\mathcal{W}_e$ , which first appeared in equation (13c), follows the usual definition: it is the work done by the external gravitational and capillary forces when the sheet is displaced virtually, as we show now. The first contribution captures the effect of gravity and is self-explanatory. The second term captures capillary forces. Such forces derive from a potential energy which we approximate as  $2\sigma$  times the area of the mid-surface<sup>5</sup>: noting that the trace  $\hat{\epsilon}_{\alpha\alpha} dx dy$  is the virtual change of area of a rectangle on the mid-surface, one sees that the proposed contribution  $\iint (-2\sigma \hat{\epsilon}_{\alpha\alpha}) dx dy$  is indeed the opposite of the virtual variation of this potential energy—hence the virtual work done by the capillary forces.

Inertia can be neglected, as discussed earlier, and this is why there is no virtual work of acceleration in equation (14).

### 2.9. Stress balance in strong form

The strong form of the balance of stress (2.9) is given here for the sake of completeness: it is not used in this paper since our numerical simulations of both the base planar flow and of the linear stability analysis

<sup>5</sup>The factor 2 comes from the fact that there is a fluid-air interface on each side of the mid-surface. Also, for a thin sheet we can neglect the area contribution coming from the lateral, rounded edges at  $x = \pm\beta$ .

are based on the weak formulation.

Integrating by parts equation (14), making use of the incremental kinematic boundary conditions to get rid of as many boundary terms as we can, and canceling the coefficients of the virtual displacement in the final expressions, we obtain (i) the missing (natural) boundary conditions listed in the third and fifth columns of table 1, and (ii) the equations for the balance of stress in the interior. The details of the integration by parts are omitted: this calculation is very similar to that for elastic plates, see the book by Landau and Lifshitz (1970) for example. The equations in the interior derived in this way are the classical equilibrium equations for thin sheets undergoing moderate rotations (they are applicable to Föppl–von Kármán elastic plates in particular):

$$N_{x\beta,\beta} + \text{Je} = 0 \quad (15a)$$

$$N_{y\beta,\beta} = 0 \quad (15b)$$

$$(N_{\alpha\beta} w_{,\alpha})_{,\beta} - \eta^2 M_{\alpha\beta,\alpha\beta} + \frac{\text{Ca}^{-1}}{\eta} w_{,\alpha\alpha} = 0 \quad (15c)$$

The first two equations correspond to a balance of stress in the vertical ( $x$ ) and transverse ( $y$ ) directions, respectively; gravity enters in the first equation through Jeyffreys' number  $\text{Je}$ . The third equation is an out-of-plane balance of forces. It couples the membrane stress determined by the tangent equilibrium, with the bending stress associated with out-of-plane displacement. The last term in the left-hand side of (15c) accounts for the stabilizing effect of capillary forces which tend to bring the sheet back into a planar configuration. Note that capillary forces also enter into the equations through boundary conditions, see table 1.

### 2.10. Summary

We have obtained a complete set of equations for the dynamics of thin viscous sheets. The kinematic boundary conditions are listed in table 1. The definition of strain rate tensors have been derived in equation (11), and the constitutive laws in equation (12). The balance of stress has been obtained in two equivalent forms: (i) in weak form, the principle of virtual work in equation (14) must be solved for the real stress distribution  $N_{\alpha\beta}$  and  $M_{\alpha\beta}$  and for arbitrary virtual displacements  $\hat{u}_\alpha(x, y)$  and  $\hat{w}(x, y)$  satisfying the incremental kinematic boundary conditions (this weak form will serve as the starting point of the numerical solution by the finite element method in the forthcoming section); (ii) in strong form, the stress balance is expressed by (15) in the interior, and by the natural boundary conditions in table 1 (this strong form has been used in previous work on the stability of viscous sheets).

This formulation of the viscous sheet model is modular: a strict separation has been enforced between (i) geometrical and kinematic equations (including the calculation of strain rates), (ii) equations for the balance of forces, and (iii) the constitutive law. This modular formulation makes it straightforward to extend the model. Possible extensions are discussed at the end of the conclusion in §7. They include relaxing the geometrical assumptions of moderate slope underlying equation (8b) to allow for rotations of arbitrary magnitude, or using more general constitutive laws.

## 3. Base solution: steady planar flow

In this section, we derive a steady and planar solution for the velocity field in the falling viscous sheet. In particular, we are able to calculate the distribution of membrane stress in this planar solution: we will show that it can be compressive in some conditions. This motivates the stability analysis with respect to out-of-plane buckling, which we carry out later in §4.

We ignore surface tension for the moment, by setting

$$\text{Ca}^{-1} = 0. \quad (16)$$

Surface tension will be taken into account later in §5, and its influence on both the planar solution and on its stability will be assessed.

### 3.1. Equations for the base flow

We focus on planar solutions by setting  $w(x, y, t) = 0$  in the previous set of equations. The bending stress then cancels,  $M_{\alpha\beta} = 0$ , the transverse equilibrium is automatically satisfied and we can therefore ignore the virtual deflection,  $\hat{w}(x, y) = 0$ . We seek a steady solution: the remaining unknowns are the in-plane velocity components,  $u_x(x, y) = u_x^{(0)}(x, y)$  and  $u_y(x, y) = u_y^{(0)}(x, y)$ , where quantities pertaining to the base solution are systematically labelled with a superscript (0).

In the absence of gravity, the solution to our problem would be a uniform velocity field  $u_\alpha(x, y, t) = U \delta_{x\alpha} = \delta_{x\alpha}$ , corresponding to a rigid translation with vertical velocity  $U \underline{e}_x = \underline{e}_x$  imposed by the slit at the top boundary, with  $U = 1$  in our rescaled units; the corresponding internal stress  $N_{\alpha\beta}$  is zero. Since we work in the limit of small gravity,  $\text{Je} \ll 1$ , we seek a solution by perturbing this rigid motion using  $\text{Je}$  as an expansion parameter,

$$u_\alpha^{(0)} = \delta_{\alpha x} + \text{Je} \bar{u}_\alpha^{(0)} \quad (17a)$$

$$N_{\alpha\beta}^{(0)} = \text{Je} \bar{N}_{\alpha\beta}^{(0)}. \quad (17b)$$

Inserting these expressions into equations (11a), (12a), (13a–13b), (14) and identifying the linear terms in  $\text{Je}$  on both sides of these equations, we obtain the incremental constitutive laws for membrane stress,

$$\bar{N}_{\alpha\beta}^{(0)} = 2 (\bar{u}_{(\alpha,\beta)}^{(0)} + \delta_{\alpha\beta} \bar{u}_{\gamma,\gamma}^{(0)}), \quad (18)$$

as well as the linearized principle of virtual work:

$$\forall \hat{u}_\alpha \text{ k.a.}, \quad \iint_{\mathcal{D}_t} (-\bar{N}_{\alpha\beta}^{(0)} \hat{u}_{\alpha,\beta} + \hat{u}_x) dx dy = 0. \quad (19)$$

The in-plane kinematic boundary conditions in the second column of table 1 are applicable (in their original form for the real velocities  $u_\alpha$ , and in incremental form for the virtual displacement  $\hat{u}_\alpha$ ). In the integrand of equation (19), the first term represents the viscous stress in the sheet and the second term the effect of the gravity in rescaled units.

By the Stokes-Rayleigh analogy, equations (18–19) are the equations for a hanging elastic rectangle in 2D, when the in-plane velocity  $\bar{u}_\alpha^{(0)}$  is reinterpreted as a displacement. Solving for the base flow is therefore equivalent to solving a problem of linear elasticity in 2D. In this framework, the unicity of the solution is warranted. This implies that the (unique) solution is symmetric with respect to the central axis  $y = 0$ . In view of this, we solve the base flow on the half-domain  $[-1, 0] \times [0, \beta]$  and enforce the kinematic in-plane boundary conditions  $u_y = 0$  along the axis  $y = 0$ , see the two bottom rows of table 1.

### 3.2. Numerical solution

When cast in strong form, the problem for the base planar flow presented above involves a partial differential equation having constant coefficients in a rectangular domain: Ribe (2008) derived a series solutions to this problem. We have tried this approach but have found it to be impractical as series converge very slowly near the corners of the domains. Therefore, we have turned to a numerical solution using the finite element method, based on the weak formulation given in §3.1. As noted above, the problem is effectively a linear elasticity problem in 2D: its numerical implementation using the finite element method is standard. All the finite element simulations presented in this paper are based on the general-purpose, open-source finite element library `libmesh` documented in the paper of Kirk et al. (2006). The in-plane velocity  $\bar{u}_\alpha^{(0)}$  is discretized using the same type of  $C^1$  elements as we use later for the deflection  $w$ , see §4.2 for details.  $C^1$ -smoothness is not required for the calculation of the base solution, but it has the advantage of suppressing spurious oscillations typically associated with lower-order finite elements. For an aspect-ratio  $\beta = 0.4$ , we start from a uniform grid containing 25 rectangular elements, which we refine adaptively five times, ending up with 8525 elements and 136400 degrees of freedom. The adaptive refinement is important to better resolve the neighborhood of the corners.

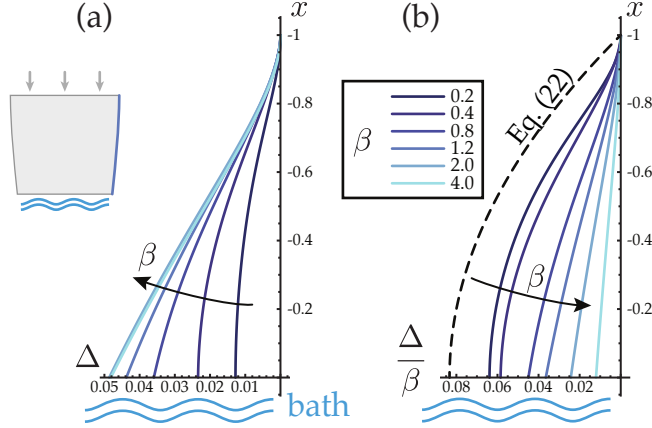


Figure 2: Retraction of the lateral free edge, based on the numerical solution for the base flow in the absence of surface tension. (a) Profile of the edge  $\Delta(x)$  as a function of aspect-ratio, based on equation (20). The retraction is exaggerated: the actual one is given by  $\text{Je} \Delta(x)$  with  $\text{Je} \ll 1$ . By Stokes-Rayleigh analogy, this retraction is similar to Poisson’s effect in an elastic band. The retraction is more pronounced for wider curtains (*i.e.* for larger values of  $\beta$ ). (b) Retraction scaled by aspect ratio  $\beta$ : for small  $\beta$ , the retraction is well approximated by the simple model of equation (22), shown by a dashed line.

### 3.3. Features of the planar solution

Jeffreys’ number  $\text{Je}$  has been scaled out from the equations for the base flow established in §3.2, and the only parameter left is the half aspect-ratio  $\beta$ : the numerical calculation of the base solution yields a family of base flows  $\bar{u}_\alpha^{(0)}(\beta, x, y)$  indexed by  $\beta$ . The dependence of  $\bar{u}_\alpha^{(0)}$  on  $\beta$  will be implicit from now on.

The shape of the free edge can then be reconstructed as follows. In our rescaled units, the descent velocity is  $U = 1$ , up to terms of order  $\text{Je} \ll 1$ . The position of a particle on the lateral edge on the right-hand side is  $(-1, \beta)$  when it exits from the slit. After a time interval  $\tau$ , its vertical coordinate is  $x = -1 + \tau$  and its horizontal velocity is  $u_y^{(0)}(-1 + \tau, \beta) = \text{Je} \bar{u}_y^{(0)}(-1 + \tau, \beta)$ . Therefore, the equation for the free edge reads<sup>6</sup>, at dominant order in  $\text{Je}$ ,  $y_{\text{fe}}(x) = \beta + \text{Je} \int_{-1}^x \bar{u}_y^{(0)}(x', \beta) dx' = \beta - \text{Je} \Delta(x)$  where  $\Delta(x)$  denotes the dimensionless retraction of the free edge,

$$\Delta(x) = \int_{-1}^x -\bar{u}_y^{(0)}(x', \beta) dx'. \quad (20)$$

Based on this formula and on the numerical solution for  $\bar{u}_\alpha^{(0)}$ , the shape of the free edge is shown in figure 2 for various aspect-ratios  $\beta$ .

Before we proceed to calculate the stress in the base solution, we come back to the expression of the in-plane strain rate which we obtained in equation (11a) based on a ‘naïve’ rescaling of the in-plane velocities by  $U$ , and of the in-plane lengths by  $H$ . The expression we obtained was  $d_{\alpha\beta} = u_{(\alpha,\beta)} + \eta^2 (w_{,(\alpha} w_{,\beta)t} + \dots)$ . We can now explain the apparent asymptotic discrepancy in the orders of magnitude of the various terms. In view of equation (17a), the velocity  $u_\alpha$  is of order 1 (*i.e.* of order  $U$  in physical units) but the dominant contribution is a rigid-body motion, so the velocity gradients are much smaller, of order  $\text{Je} \ll 1$  (*i.e.* of order  $\text{Je}U/H$  in physical units). This means that the first term  $u_{(\alpha,\beta)} \sim \text{Je}$  in the right-hand side of equation (11a) can actually be balanced with the other terms, which are of order  $\eta^2$ , provided  $\text{Je} \sim \eta^2$ . We will indeed adopt this scaling assumption  $\text{Je} \sim \eta^2$  later in the analysis of the buckling modes, see §4.

Let us now proceed to plot the rescaled membrane stress  $\bar{N}_{\alpha\beta}^{(0)} = N_{\alpha\beta}^{(0)}/\text{Je}$  which, like  $\bar{u}^{(0)}_\alpha(x, y)$  depends only on  $(x, y)$  and  $\beta$ . We are particularly interested in negative eigenvalues of  $\bar{N}_{\alpha\beta}$ , which correspond to

<sup>6</sup>Recall that we are working in dimensionless units where in-plane lengths have been rescaled by  $H$  and velocities by  $U$ , see §2.5: in original (physical) units, the retraction is  $H(\beta - \text{Je} \Delta)$ .

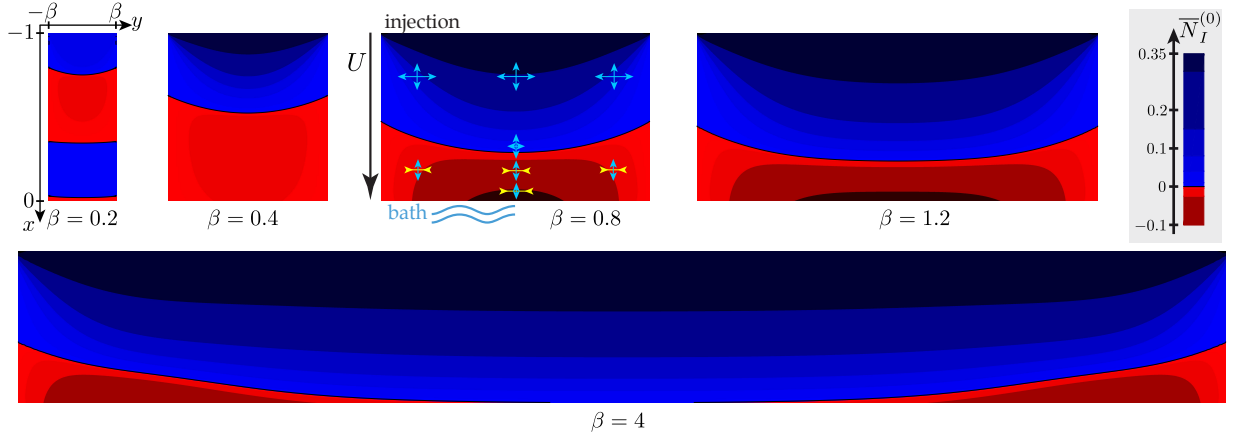


Figure 3: Contour map of the most negative eigenvalue,  $\bar{N}_I^{(0)}$ , of the scaled membrane stress  $\bar{N}_{\alpha\beta}^{(0)} = N_{\alpha\beta}^{(0)}/\text{Je}$  in the base solution for different aspect-ratios  $\beta$ ; surface tension is zero. In blue regions the base flow is tensile in all directions ( $\bar{N}_I^{(0)} > 0$ ); in red regions, some directions are compressive ( $\bar{N}_I^{(0)} < 0$ ). In the third map ( $\beta = 0.8$ ), the directions of the principal stresses are shown with arrows: the largest eigenvalue is almost vertical (along  $x$ ) and the smallest (or most negative) eigenvalue is almost horizontal (along  $y$ ); blue and yellow arrows correspond to a positive (tensile) and negative (compressive) principal stress, respectively. The orientation of the principal stress directions is similar for all the values of  $\beta$  (arrows shown for  $\beta = 0.8$ ). The small retraction of the free edge is not shown here, see figure 2. Note that the color scale (see legend) is designed to enhance the changes of sign of  $\bar{N}_I^{(0)}$ —in spite of the large contrast, the amplitude of oscillations of  $\bar{N}_I^{(0)}$  with  $x$  in the first figure  $\beta = 0.2$  are small.

directions of compression: in the presence of a negative principal stress,  $\bar{N}_I^{(0)} < 0$  (red regions in figure 3), a buckling instability is possible if the thickness is small enough. The viscous sheet is stretched vertically but compressed horizontally in the red regions ( $\bar{N}_I^{(0)} < 0$ ). By contrast, it stretches both in the horizontal and vertical directions in the blue regions ( $\bar{N}_I^{(0)} > 0$ ). We observe that, everywhere along the sheet, the principal stress directions remain almost aligned with the  $x$  and  $y$  axis (arrows shown for  $\beta = 0.8$  in the figure, but not shown for other values of  $\beta$ ). This will be explained based on a simplified one-dimensional model for the flow, see §3.4 below.

The stress maps reveal a strong dependence on the aspect-ratio  $\beta$ . In the range  $0.4 \leq \beta \leq 1.2$ , there is a single region of compression at the bottom of the sheet near the bath. For smaller values of  $\beta$ , this compressive region moves upstream and progressively invades the domain. For  $\beta = 0.2$ , it is made up of two components: a small region near the bottom, and a large region in the middle of the sheet — this corresponds to a transverse stress  $N_{yy}$  which is oscillatory with  $x$  along the central axis. For larger aspect-ratio,  $\beta \geq 4$ , the flow is tensile almost everywhere, except in the vicinity of the lower corners where the free lateral sides meet with the bath. This stress distribution is similar to that obtained for stretched elastic plates by Friedl et al. (2000), a problem which has been further studied by Cerda et al. (2002); Zheng (2009); Puntel et al. (2011); Healey et al. (2013). It is also similar to that obtained by Filippov and Zheng (2010) using a viscous membrane model (with the added simplification that the motion is close to a rigid translation and that the variation of thickness can be neglected here). The alternation of tensile and compressive regions, the genuine 2D character of the flow, and the sensitivity of the stress distribution to the single geometrical parameter  $\beta$  make this system a perfect one for analyzing the stability of viscous sheets. This is the goal we pursue in §4.

### 3.4. A simple approximation for the base flow

In this section, we propose a simple approximation for the base flow which captures the retraction of the lateral edges shown in figure 2. We do not claim that this approximation is exact in any asymptotic sense.

It is only intended to provide a simple picture of the actual solution. This section is independent of the rest of the paper and can be skipped.

Inspired by the one-dimensional example of a bar hanging under its own weight, we assume a membrane stress  $\underline{\underline{N}}^{(0)} \approx -\text{Je } x \underline{e}_x \otimes \underline{e}_x$ , which satisfies the equilibrium in the interior (15a–15b). Inverting the constitutive law for membrane stress (9a), we obtain the membrane strain rate as  $\underline{\underline{d}}^{(0)} \approx -\text{Je } \frac{x}{3} (\underline{e}_x \otimes \underline{e}_x - \frac{1}{2} \underline{e}_y \otimes \underline{e}_y)$ , or  $\underline{\underline{d}}^{(0)} \approx -\frac{x}{3} (\underline{e}_x \otimes \underline{e}_x - \frac{1}{2} \underline{e}_y \otimes \underline{e}_y)$  in dimensionless form. Here, the coefficient  $1/2$  accounts for Poisson's effect. This is only an approximation of the exact solution as we have ignored both some of the boundary condition, and the condition that the strain rate is geometrically compatible—it is not, as evident from the fact that  $\bar{d}_{yy}^{(0)}$  depends on  $x$ , although the shear component  $\bar{d}_{xx}^{(0)}$  is zero. The vertical component  $\bar{d}_{xx}^{(0)} = -x/3$  can be integrated into a vertical velocity  $\bar{u}_x^{(0)}(x, y) \approx -\text{Je}(\frac{x^2}{6} + f(y))$  where the constant of integration  $f$  is set by the condition  $\bar{u}_x = 0$  at the top boundary as

$$\bar{u}_x^{(0)}(x, y) \approx \frac{1 - x^2}{6}. \quad (21a)$$

Integrating now  $\bar{d}_{yy}^{(0)}$ , we obtain  $\bar{u}_y^{(0)} = \frac{xy}{6} + g(x)$ . The symmetry condition yields the constant of integration as  $g(x) = 0$ ,

$$\bar{u}_y^{(0)}(x, y) \approx \frac{xy}{6}. \quad (21b)$$

By comparing to the numerical solution of §3.2–3.3, we have checked that equations (21) are a reasonable approximation of the base flow, especially for wide curtains (large  $\beta$ ).

This approximation predicts a retraction of the lateral edge

$$\Delta \approx \int_{-1}^x \bar{u}_y^{(0)}|_{y=-\beta} dx' = \int_{-1}^x \left( -\frac{\beta x'}{6} \right) dx' = \frac{\beta}{12} (1 - x^2), \quad (22)$$

which is in reasonable agreement with the exact solution, as shown as a dashed curve in figure 2b.

In this approximation, one principal stress is tensile and in the vertical direction,  $\bar{N}_{xx}^{(0)} = -x/3 = -|x|/3 \geq 0$ , and the other one is neutral and in the horizontal direction,  $\bar{N}_{yy}^{(0)} = 0$ . This is consistent with the numerical finding that the principal stress directions remain everywhere close to horizontal and vertical, see the light blue and yellow arrows in figure 3. However, the approximation cannot predict the sign of the most compressive eigenvalue of the stress  $\bar{N}_I^{(0)}$ , which is crucial for the stability analysis. Still, it explains why  $\bar{N}_I^{(0)}$  takes on numerically small values, especially for small  $\beta$ , see the stress maps in figure 3. This introduces numbers in the problem whose absolute values are much less than 1 even though they are mathematically of order 1, *i.e.* they do not tend to zero in any asymptotic limit. Such small values of  $\bar{N}_I^{(0)}$  are responsible for the numerically small values appearing in the vertical axes of the rescaled phase diagrams, see later in figures 5 and 8.

#### 4. Linear stability analysis: curtain modes

In this section, we analyze the stability of the planar solutions derived in the previous section with respect to out-of-plane perturbations. By means of a finite element discretization, the buckling analysis is formulated as a generalized matrix eigenvalue problem. Due to the presence of advection, the buckling analysis differs significantly from that of an elastic plate, and it involves a non-Hermitian problem.

##### 4.1. Linearized equations for thin sheets

We perturb the base flow with a small time-dependent deflection  $w^{(1)}(x, y, t)$ :

$$u_\alpha(x, y, t) = u_\alpha^{(0)}(x, y) + 0 + \dots \quad (23a)$$

$$w(x, y, t) = 0 + w^{(1)}(x, y, t) + \dots, \quad (23b)$$



where  $^{(0)}$  denotes the base solution, and  $^{(1)}$  the first order perturbation. We may seek a purely transverse perturbation to first order, see equation above, as the base solution is invariant by reflection with respect to the  $(xy)$  plane.

From equations (13–14), the linearized principle of virtual work reads

$$\forall \hat{w} \text{ k.a.}, \quad \iint_{\mathcal{D}_t} \left( -\text{Je} \bar{N}_{\alpha\beta}^{(0)} \hat{\epsilon}_{\alpha\beta}^{(1)} - \eta^2 M_{\alpha\beta}^{(1)} \hat{w}_{,\alpha\beta} \right) dx dy = 0. \quad (24)$$

In equation (24),  $\hat{\epsilon}_{\alpha\beta}^{(1)} = w_{,\alpha}^{(1)} \hat{w}_{,\beta}$  is obtained by linearization of equation (13c) with respect to the real, incremental deflection. The out-of-plane perturbation does not change the membrane strain rate to first order,  $d_{\alpha\beta}^{(1)} = 0$ , so  $N_{\alpha\beta}^{(1)} = 0$  by the constitutive law as well: this is why the term  $-\iint N_{\alpha\beta}^{(1)} \hat{\epsilon}_{\alpha\beta}^{(0)} dx dy = 0$  is absent from equation (24).

Rewriting the constitutive law (12b) as  $M_{\alpha\beta} = L_{\alpha\beta\gamma\tau} k_{\gamma\tau}$  with

$$L_{\alpha\beta\gamma\tau} = \frac{1}{6} (\delta_{\alpha\gamma} \delta_{\beta\tau} + \delta_{\alpha\beta} \delta_{\gamma\tau}), \quad (25)$$

and noting that  $k_{\alpha\beta}^{(1)} = w_{,\alpha\beta t}^{(1)} + u_{\gamma}^{(0)} w_{,\alpha\beta\gamma}^{(1)} = w_{,\alpha\beta t}^{(1)} + U w_{,\alpha\beta x}^{(1)}$  (up to small corrections of order  $\text{Je}$ ), with  $U = 1$  in our rescaled variables, this becomes

$$\iint_{\mathcal{D}_t} \left( -\text{Je} \bar{N}_{\alpha\beta}^{(0)} w_{,\alpha}^{(1)} \hat{w}_{,\beta} - \eta^2 L_{\alpha\beta\sigma\tau} \left[ w_{,\gamma\tau t}^{(1)} + w_{,\gamma\tau x}^{(1)} \right] \hat{w}_{,\alpha\beta} \right) dx dy = 0.$$

The square bracket contains the linearized version of the bending strain rate  $\underline{k}$ : the last (co-rotational) term present in the general definition (8b) of  $\underline{k}$  has disappeared and it plays no role in the particular problem studied here.

Since the base flow is stationary, we seek modes  $w^{(1)}(x, y, t)$  having an exponential dependence on time,

$$w^{(1)}(x, y, t) = W(x, y) e^{-i\omega t}, \quad (26)$$

where the angular velocity  $\omega$  is a complex eigenvalue. The weak form of the equations for the linear modes becomes

$$\iint_{\mathcal{D}_t} \left( -\bar{N}_{\alpha\beta}^{(0)} W_{,\alpha} \hat{w}_{,\beta} - \frac{\eta^2}{\text{Je}} L_{\alpha\beta\gamma\tau} W_{,\gamma\tau x} \hat{w}_{,\alpha\beta} \right) dx dy = -i \frac{\omega \eta^2}{\text{Je}} \iint_{\mathcal{D}_t} L_{\alpha\beta\gamma\tau} W_{,\gamma\tau} \hat{w}_{,\alpha\beta} dx dy \quad (27)$$

In terms of the pre-stress distribution  $\bar{N}_{\alpha\beta}^{(0)}(x, y)$  available from §3.3, this equation defines an eigenvalue problem for  $W(x, y)$  and  $\omega$ , which we now solve numerically. This type of eigenvalue problem arises generally in the stability analysis of rate-independent systems, see for instance Triantafyllidis and Leroy (1994).

#### 4.2. Discretization of the buckling problem

One may notice that the weak form of the out-of-plane problem involves third order derivatives, which are due to advection terms. These terms give rise to fifth order derivatives in the strong form of the balance of stress, meaning that high-order elements are required in the finite-element method.  $C^1$ -type finite elements have been designed for plate-like problem, and can be applied directly to variational problems involving second order derivatives in both the test and trial functions (hence to problems involving into fourth-order derivatives in strong form). In our problem, an additional, third-order derivative of the test function is present. We take care of it using a classical method, described for instance by Xu and Shu (2008). The main idea is to use a redundant unknown  $\Theta(x, y)$  to represent the gradient  $W_{,x}(x, y)$  of the deflection unknown  $W$ : the original third-order derivative  $W_{,\gamma\tau x}$  are then available as the second-order derivative of the new variable  $\Theta$ . This effectively decreases the order of derivation to two in the weak formulation, so that standard  $C^1$ -smooth elements can be used. This comes at the price of adding a new variable. A constraint must be set up, so that the new variables  $\Theta$  matches  $W_{,x}$ , as intended. This constraint is handled in our implementation using a simple penalty method.

We define a new complex unknown field  $\Theta(x, y)$  representing the slope  $W_{,x}(x, y)$ . The modified variational formulation is as follows: one seeks the complex fields  $W(x, y)$  and  $\Theta(x, y)$  and the complex eigenvalue  $\omega$  such that, for any virtual displacement  $(\hat{w}, \hat{\theta})$  satisfying the incremental kinematic boundary conditions,

$$\iint_{\mathcal{D}_t} \left( -\bar{N}_{\alpha\beta}^{(0)} W_{,\alpha} \hat{w}_{,\beta} - \frac{\eta^2}{\text{Je}} L_{\alpha\beta\gamma\tau} \Theta_{,\gamma\tau} \hat{w}_{,\alpha\beta} - \Lambda (\Theta - W_{,x}) \hat{\theta} \right) dx dy = -i \frac{\omega \eta^2}{\text{Je}} \iint_{\mathcal{D}_t} L_{\alpha\beta\gamma\tau} W_{,\gamma\tau} \hat{w}_{,\alpha\beta} dx dy. \quad (28)$$

An additional penalty term has added to the weak form (27). The penalty parameter  $\Lambda$  is chosen large enough for the equality  $\Theta = W_{,x}$  to be satisfied to a prescribed accuracy.

The eigenvalue problem (28) concerns a dissipative system and is not Hermitian, as can be checked directly.

#### 4.3. Numerical eigenvalue analysis

In equation (28), the unknowns are the spatial part  $(W, \Theta)$  of the eigenmodes, and the eigenvalue is the complex frequency  $\omega$  or, equivalently, the stability exponent  $(-i\omega)$ . We proceed to discretize this problem using the finite element method. The variables  $W$  and  $\Theta$  are discretized using  $C^1$  elements to comply with the presence of second-order spatial derivatives in equation (28). Specifically, we use a Hermite polynomials basis with Bogner-Fox-Schmidt element types, a popular choice for elastic plate problems Rustein (2008). We use the open source C++ finite elements library `libmesh`, leveraging the powerful numerical tools PETSC (Balay et al., 1997, 2014b,a) for matrix algebra and SLEPC (Hernandez et al., 2005) for eigenvalue analysis. Upon discretization of equation (28), we obtain a generalized matrix eigenvalue problem in the form

$$[\hat{w}, \hat{\theta}] \cdot \underline{\underline{A}} \left( \beta, \frac{\eta^2}{\text{Je}} \right) \cdot [W, \Theta] = i \frac{\omega \eta^2}{\text{Je}} [\hat{w}] \cdot \underline{\underline{B}} \cdot [W], \quad (29)$$

where the assembly of the matrices  $\underline{\underline{A}}$  and  $\underline{\underline{B}}$  is described below, and the eigenvalue  $\frac{\omega \eta^2}{\text{Je}}$  is the scaled complex frequency. The matrix  $\underline{\underline{A}}$  is assembled from all the terms in the variational form (28) that include no time derivative; by contrast, the matrix  $\underline{\underline{B}}$  gathers all the terms involving a time derivative—all of which come from viscous bending. As the term  $-\bar{N}_{\alpha\beta}^{(0)} W_{,\alpha} \hat{w}_{,\beta}$  in equation (28) is taken care of by  $\underline{\underline{A}}$ , the pre-stress  $\bar{N}_{\alpha\beta}^{(0)}$  must be determined prior to assembling  $\underline{\underline{A}}$ ; this is why  $\underline{\underline{A}}$  depends on the aspect-ratio  $\beta$ .  $\underline{\underline{A}}$  depends on the parameter  $\eta^2/\text{Je}$  as well because of the second term in the integrand in the left-hand side of (28). This parameter enters as a multiplicative coefficient in the right-hand side of (28) and has been factored out from  $\underline{\underline{B}}$ .

The step following the assembly of the matrices  $\underline{\underline{A}}$  and  $\underline{\underline{B}}$  is to solve the discrete eigenvalue problem (29). We are specifically interested in finding the vector  $(\bar{W}, \bar{\Theta})$  associated with the largest real growth rate ( $\text{Im } \omega$ ) as this is the most unstable mode. This corresponds to finding the eigenvalue  $i\omega\eta^2/\text{Je}$  with the smallest real part. For maximal efficiency, we use a solver based on an iterative Krylov-Schur method combined with a spectral transformation method (shift-and-invert), which can return a few eigenvectors corresponding to the largest growth rates. Some of the eigenmodes are jagged, *i.e.* undulating with a length scale comparable to the mesh size (grid modes), and are discarded. Typical eigenvectors corresponding to the largest growth rate are shown in figure 4 for different values of  $\beta$ , and for values of  $\frac{\eta^2}{\text{Je}}$  close to the onset of the instability.

One can notice that the eigenmodes are mostly concentrated in the regions where the base flow is compressive (red regions in figure 3 and 4), as expected. As apparent from the plot for  $\beta = 0.2$ , the eigenmodes decay along the direction the flow (downwards) in the regions where the pre-stress  $\bar{N}_{\alpha\beta}^{(0)}$  is tensile in all directions, hence locally stable (blue regions). The modes are undulating in the transverse direction  $y$ , much like a hanging curtain: this is because a negative principal stress in the base solution always takes place in a nearly transverse direction, see the yellow arrows in figure 3.

The wavelength of the curtain is significantly smaller than the size of the system. This contrasts with buckling problems involving a homogeneous base solution, where the wavelength is usually set by the size the system. Indeed, for the viscous curtain, the base flow is far from homogeneous for the problem. As a

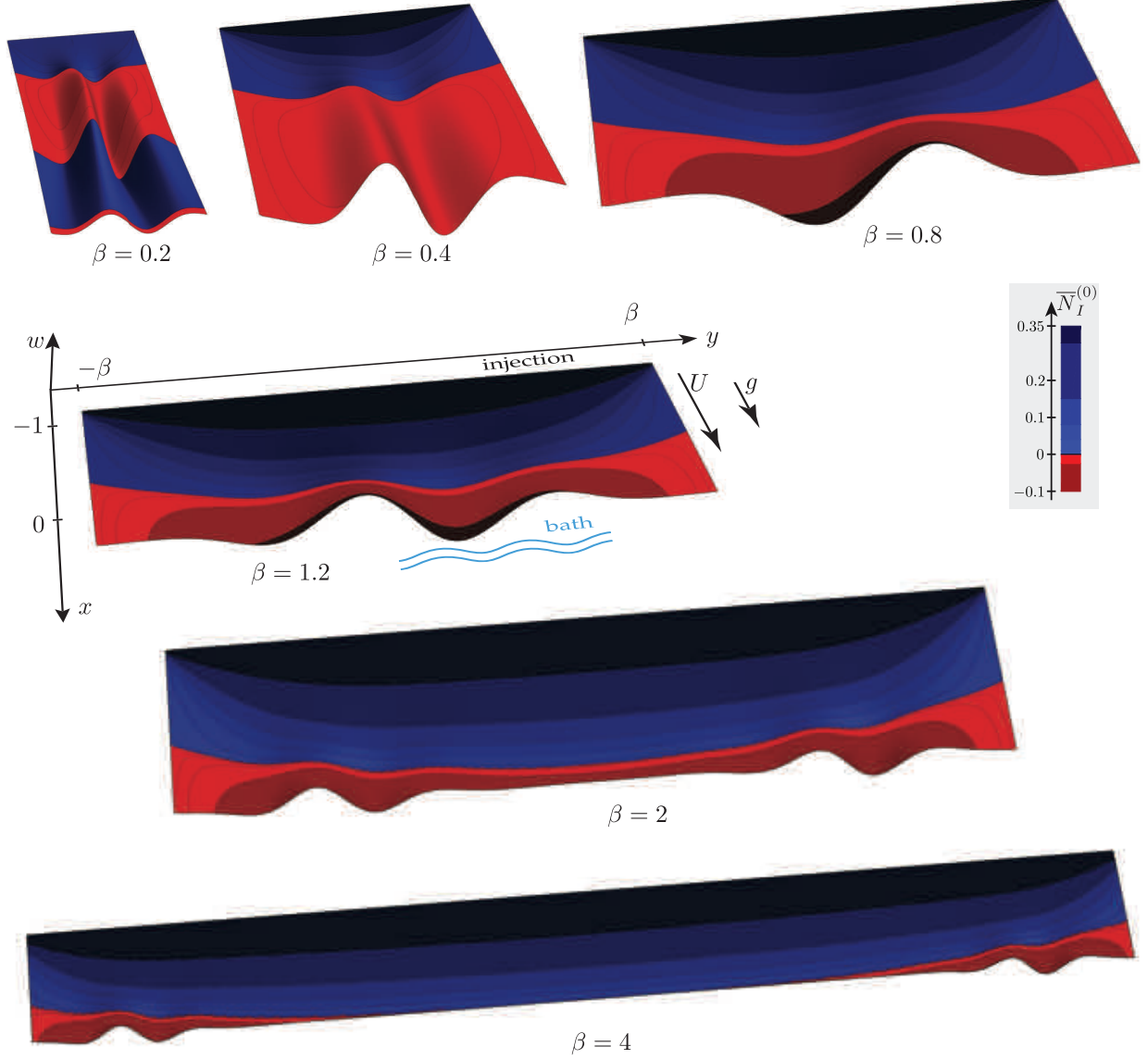


Figure 4: Shape of the most unstable buckling mode for different aspect ratios  $\beta$ , without surface tension. For every value of  $\beta$ , a different value of the parameter  $\frac{\eta^2}{\text{Je}}$  is picked such that the sheet is close to the limit of stability: the different plots correspond to  $(\beta, \log_{10} \frac{\eta^2}{\text{Je}}) = (0.2, -6.9), (0.4, -5.8), (0.8, -4.5), (1.2, -4.5), (2, -4.9)$  and  $(4, -5.0)$ , respectively, as indicated by a star in the phase diagram in figure 5. The four modes with  $\beta \leq 1.2$  are not oscillatory ( $\text{Re}\omega = 0$ ), while the two ones with  $\beta \geq 2$  are oscillatory ( $\text{Re}\omega \neq 0$ ). The colors encode the most negative eigenvalue of the stress in the base flow, as in figure 3: the instability develops in compressive regions, shown in red.

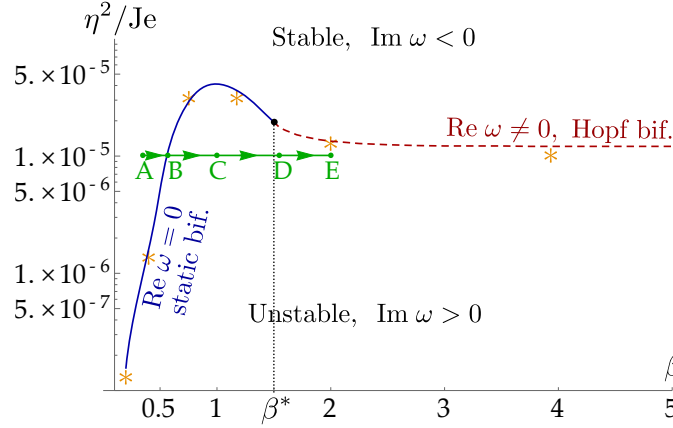


Figure 5: Phase diagram without surface tension: critical scaled thickness squared,  $\frac{\eta^2}{J_e}$ , as a function of the aspect-ratio  $\beta$ . The curves of marginal stability are obtained as the locus of  $\text{Im } \omega = 0$ , where  $\omega$  is the complex frequency of the most unstable eigenmode. The dashed curve corresponds to an oscillatory mode ( $\text{Re } \omega \neq 0$ ) obtained by a Hopf bifurcation, and the solid curve corresponds to a non-oscillatory, buckling mode ( $\text{Re } \omega = 0$ ) obtained by a static bifurcation. The green path going from A to E is used to track the bifurcation from a static to an oscillatory mode, as studied in detail in figure 6;  $\beta^*$  is the critical value of aspect ratio where oscillatory modes appear. The star symbols refer to the eigenmodes shown in figure 4. The numerical values on the vertical axis of this phase diagram are small because the base flow is only mildly compressive, even in rescaled variables, as noted at the end of §3.4.

result, the wavelength of the instability is essentially set by the length scale at which the stress varies in the base solution, rather than by the size of the system. The eigenmodes are reminiscent of those obtained for elastic plates in the work of Friedl et al. (2000); Kim et al. (2012); Nayyar et al. (2011); Zheng (2009); Puntel et al. (2011); Fischer et al. (2000). Note that the present system is non-conservative, because of viscosity and of the open character of the flow: as a result, oscillatory linear modes are possible ( $\text{Im } \omega = 0$  and  $\text{Re } \omega \neq 0$ )—the modes  $\beta = 2$  and  $\beta = 4$  shown in figure 4 are indeed oscillatory, as discussed below. For those oscillatory modes, the plots in figure 4 are based on the real part of the eigenmodes and the imaginary part is not shown.

#### 4.4. Phase diagram

In this linear analysis, an instability takes place when the growth rate ( $\text{Im } \omega$ ) of the eigenmodes shown in figure 4 becomes positive. This happens when the sheet is thin enough, *i.e.* when the dimensionless thickness  $\eta$  is small enough. More accurately, the stability of the planar configuration is determined by the *scaled* thickness squared  $\eta^2/J_e$  and by the aspect ratio  $\beta$ , see equation (29). By analyzing the sign of the largest growth rate  $\text{Im } \omega$  as a function of these parameters, a phase diagram can therefore be plotted in the  $(\beta, \frac{\eta^2}{J_e})$  plane, see figure 5. According to this phase diagram, the linear stability depends on the parameter  $\frac{\eta^2}{J_e} = \frac{\mu U h^2}{\rho g H^4}$ : the planar flow is linearly stable above a critical value of this parameter depending solely on the aspect-ratio  $\beta$ , and linearly unstable below. In terms of the physical parameters, the flow is therefore stable when the thickness  $h$  is above a critical value depending on the physical parameters of the problem, and unstable below, as usual in plate buckling instabilities: increasing the thickness corresponds to moving upwards in the phase diagram.

When the flow is marginally stable ( $\text{Im } \omega = 0$ ), the mode can either be static ( $\text{Re } \omega = 0$ ) or oscillatory ( $\text{Re } \omega \neq 0$ ), corresponding to the solid and dashed curves in figure 5, respectively. A static bifurcation is obtained for small aspect-ratios  $\beta < \beta^*$  and an oscillatory (flutter-like, or Hopf) bifurcation is obtained for larger aspect-ratios,  $\beta > \beta^*$ . The change in behavior occurs for  $\beta^* = 1.5$  (this number depends only on the boundary conditions). Note that the presence of oscillatory modes is made possible by the non-Hermitian character of the eigenvalue problem, as a consequence of both viscous dissipation and of the open flow.

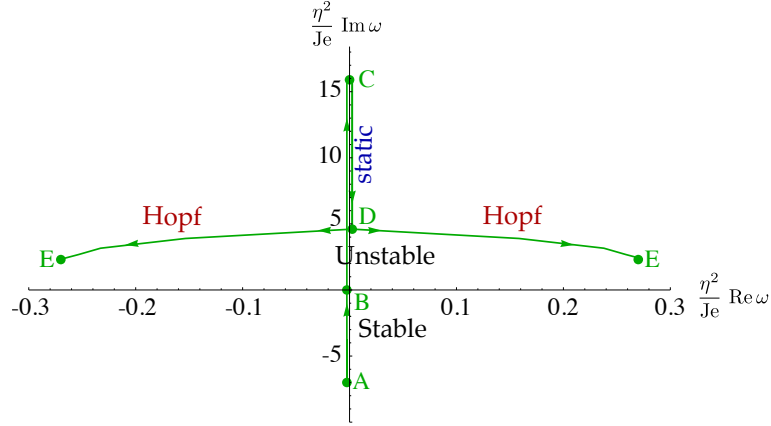


Figure 6: Analysis of the transition from static to oscillatory modes: motion of the growth rate  $\omega$  in the complex plane when the aspect ratio  $\beta$  is varied, for fixed  $\eta^2/Je = 10^{-5}$ . The curves are the paths traced out by  $\omega$  as the aspect ratio  $\beta$  is varied from  $\beta = 0.4$  to  $\beta = 2$ , corresponding to the path joining the points labelled A and E in figure 5. Point B corresponds to the value of the aspect ratio  $\beta$  at which the system becomes unstable, *i.e.* where the green line in figure 5 intersects the curve of marginal stability. At point C the system has the largest possible growth rate for this particular choice of parameters. At point D the instability becomes oscillatory, as the double eigenvalue  $\omega$  splits into a pair of eigenvalues having opposite real parts.

Triantafyllidis and Leroy (1994) studied a stability problem involving viscous and elasto-plastic dissipation but no open flow, and found only static bifurcations.

To analyze the transition from static to oscillatory modes, we consider the path A→E traced out in the phase diagram in figure 5, and track the corresponding motion of the most unstable eigenvalues  $\omega$  in the complex plane, see figure 6. At point A, the eigenvalues are located in the stable half-plane  $\text{Im } \omega < 0$ . At point B, when the path crosses the curve of neutral stability, the mode becomes marginally stable,  $\text{Im } \omega = 0$ ; the bifurcation is then static, so  $\omega = 0$ , see figure 6. Further along the path,  $\omega$  becomes real and positive (unstable mode obtained by a static bifurcation), reaches a maximum at C which is roughly where the path enters most deeply into the unstable region, and then decreases up to point D where the double real eigenvalue splits into a pair of conjugate complex eigenvalues. The corresponding modes remain unstable all the way to the end E of the path. This shows that the oscillatory mode is produced by a continuous bifurcation from the static mode: the angular frequency of the oscillatory mode  $|\text{Re } \omega|$  varies smoothly from zero when  $\beta$  is increased beyond  $\beta^*$ .

## 5. Effect of surface tension

### 5.1. Restoring surface tension

We have included the surface tension in our derivation of the governing equations (§2) but have ignored it since then by setting  $\text{Ca}^{-1} = 0$  at the beginning of §3. Here, we extend the analysis of the base flow of §3 and the linear stability analysis of §4 to restore the effect of surface tension,

$$\text{Ca}^{-1} > 0. \quad (30)$$

As we shall show, a moderate change in surface tension modifies quantitatively the base flow and its stability, but not qualitatively.

Surface tension tends to minimize the area of the fluid-air interface, which for a thin plate is twice the area of the midsurface, see equation (10c). It affects the stability of the film in two antagonistic ways. On one hand it enhances the contraction of the lateral edges of the curtain and as a result tends to make the horizontal stress  $N_{yy}^{(0)}$  more compressive, and the base flow more unstable (compare the upper halves of the flow maps in figure 7b below). On the other hand, surface tension penalizes undulations as they increase the

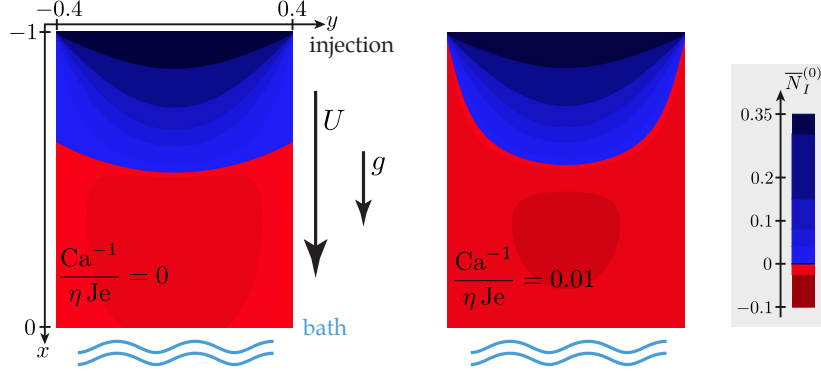


Figure 7: Influence of surface tension on the stress distribution in the base flow. The most compressive principal membrane stress  $\bar{N}_I^{(0)}$  is plotted, as in figure 3, now for a fixed aspect ratio  $\beta = 0.4$ : without surface tension (left,  $\text{Ca}^{-1}/(\eta \text{Je}) = 0$ ) and with surface tension (right,  $\text{Ca}^{-1}/(\eta \text{Je}) = 0.01$ ). Note that the sharp contrast in the color table near  $\bar{N}_I^{(0)} = 0$  emphasizes small changes in the map.

area of the midsurface, and therefore tends to stabilize the flow. A detailed analysis is therefore required to assess the net effect of surface tension.

### 5.2. Influence on the base flow

In section 3, the base flow has been found by solving equation (19), which describes the equilibrium of a planar viscous sheet. In the presence of surface tension, an additional term is present in this equation, see equation (14),

$$\forall \hat{u}_\alpha \text{ k.a.}, \quad \iint_{\mathcal{D}_t} \left( -\bar{N}_{\alpha\beta}^{(0)} \hat{u}_{(\alpha,\beta)} + \hat{u}_x - \frac{\text{Ca}^{-1}}{\eta \text{Je}} \hat{u}_{\alpha,\alpha} \right) dx dy = 0. \quad (31)$$

As evident from (13b) already, this confirms that the natural scaling for the inverse capillary number in our problem is  $\text{Ca}^{-1} \sim \eta \text{Je}$ , that is, in physical units,  $\sigma \sim \rho g H h$ . Repeating the numerical procedure of §3, one can then solve for the base flow as a function of the dimensionless parameters  $\beta$  and  $\text{Ca}^{-1}/(\eta \text{Je})$ . Such a solution is presented in figure 7b for two particular values of the parameters. Comparison of the case without surface tension in figure 7a of the figure reveals that a surface tension  $\text{Ca}^{-1}/(\eta \text{Je}) = 0.01$  has a noticeable but limited influence on the base flow. The effect is most pronounced along the sides of the curtain, where the stress tends to become more compressive.

### 5.3. Influence on the stability

Equation (24) governs the stability of the curtain. After restoring the surface tension term, it reads

$$\forall \hat{w}(x, y) \text{ k.a.}, \quad \iint_{\mathcal{D}_t} \left( -\bar{N}_{\alpha\beta}^{(0)} \hat{\epsilon}_{\alpha\beta}^{(1)} - \frac{\eta^2}{\text{Je}} M_{\alpha\beta}^{(1)} \hat{w}_{,\alpha\beta} + \frac{\text{Ca}^{-1}}{\eta \text{Je}} \hat{\epsilon}_{\alpha\alpha}^{(1)} \right) dx dy = 0, \quad (32)$$

with the same kinematic boundary conditions are earlier. By contrast, the natural boundary conditions, which express the balance of forces along the sides of the curtain, are modified by the presence of a capillary force but there is no need to derive them explicitly as the numerical implementation works directly on the weak form.

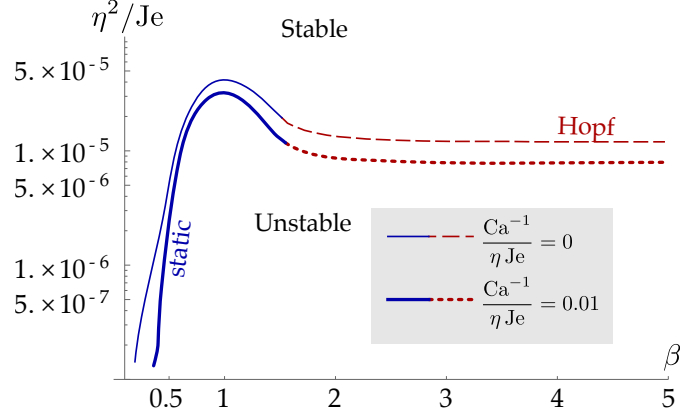


Figure 8: Influence of surface tension on the phase diagram. The upper set of curves is the same set of data as in figure 5. Surface tension lowers the curves of marginal stability, meaning that it stabilizes the planar solution.

The capillary term brings in a contribution to the left-hand side of the generalized eigenvalue problem (28):

$$\forall \hat{w}(x, y) \text{ k.a., } \iint_{\mathcal{D}_t} \left[ \left( -\bar{N}_{\alpha\beta}^{(0)} + \frac{\text{Ca}^{-1}}{\eta \text{Je}} \right) W_{,\alpha} \hat{w}_{,\beta} - \frac{\eta^2}{\text{Je}} L_{\alpha\beta\gamma\tau} \Theta_{,\gamma\tau} \hat{w}_{,\alpha\beta} - \Lambda (\Theta - w_{,x}) \hat{\theta} \right] dx dy$$

$$\dots = -i \frac{\omega \eta^2}{\text{Je}} \iint_{\mathcal{D}_t} L_{\alpha\beta\gamma\tau} W_{,\gamma\tau} \hat{w}_{,\alpha\beta} dx dy \quad (33)$$

In addition to the explicit dependence on surface tension through the inverse capillary number  $\text{Ca}^{-1}$ , there is now an implicit dependence through the pre-stress  $\bar{N}_{\alpha\beta}^{(0)}$ , see figure 7.

The eigenvalue problem (33) is discretized and solved by exactly the same method as in §4.3 earlier. The complex eigenvalues  $\omega$  are now determined as a function of the aspect ratio  $\beta$ , the dimensionless thickness squared  $\frac{\eta^2}{\text{Je}}$  and the dimensionless surface tension  $\frac{\text{Ca}^{-1}}{\eta \text{Je}}$ . The modified phase diagram is shown in figure 8 for  $\frac{\text{Ca}^{-1}}{\eta \text{Je}} = 0.01$ . This value is typical of the glass forming process described in figure 1: we use  $\sigma = 250 \text{ mN/m}$  for molten glass (Parmelee et al., 1939),  $h = 2 \text{ mm}$ ,  $\rho = 2.4 \text{ g/cm}^3$ ,  $g = 9.81 \text{ m/s}^2$  and  $H = 1 \text{ m}$  to obtain  $\frac{\text{Ca}^{-1}}{\eta \text{Je}} = \frac{2\sigma}{h\rho g H} = 0.01$ .

Overall, the phase diagram reveals that surface tension makes the planar flow more stable, *i.e.* its stabilizing effect on wrinkles is more important than its tendency to make the base flow more compressive.

## 6. Discussion

In Section 2, we have proposed a set of general equations governing the viscous dynamics of thin sheets. The strain rates have been defined using objective derivatives which, in the presence of advection, effectively operate in a frame that both moves and rotates with the fluid. In a subsequent step, these general equations have been specified to the special cases of a planar flow, and of its linear stability analysis. In previous work on viscous plates, advection has been taken care of in a less systematic manner, by adding by hand a single advection term in the definition of the bending strain rate, see the work of Slim et al. (2012): the resulting time derivative is co-moving but not co-rotating with the flow, which may become problematic if the base flow involves non-zero rotation velocities with respect to the laboratory frame.

We have investigated the stability of the falling viscous curtain under the assumption of weak gravity,  $\text{Je} \ll 1$ , *i.e.* the base flow is a rigid-body translation plus a small non-rigid correction due to gravity. The initial motivation was to avoid technical complications: for weak gravity, the solution for the base

flow does not involve any free boundary and the linear stability can be formulated in a fixed rectangular domain. In addition, the assumption of weak gravity brings in an important and unexpected benefit: it leads to an asymptotically consistent thin plate problem. Equations (32) for the base flow and (33) for the linear stability involve parameters of order 1 only if we assume that the thickness-to-height ratio scales like  $\eta \sim \sqrt{\text{Je}}$  and the dimensionless surface tension like  $\text{Ca}^{-1} \sim \eta \text{Je} \sim \text{Je}^{3/2}$ . By contrast, the classical and well-studied case of a viscous layer in a Taylor-Couette flow between counter-rotating cylinders cannot be formulated in an asymptotically consistent manner. This has led to a number of difficulties: Bhattacharya et al. (2013) note that the thin viscous plate model predicts very short wavelengths, a vanishingly small buckling threshold, while Slim et al. (2012) argue that advection terms are physically important and must be retained in the model even though they are not asymptotically balanced with the other terms. None of these difficulties are present in our problem.

To describe the buckling of a viscous curtain, we have used the classical small-slope variant of the BNT equations for thin viscous sheets, as derived in §2.2 of the paper of Howell (1996) and by others. However, our scaling assumptions are different from those used by Howell and others, and we must now explain that they are consistent. Howell considers a time-scale  $t_{\text{H}}^* \sim \eta^2 H/U$  (in our own notation), while we have introduced<sup>7</sup> a typical time-scale  $t^* \sim H/U$  in §2.5. In addition, Howell considers the obvious scaling for the strain rate tensor  $d_{\text{H}}^* \sim U/H$ , while in our problem the strain rate scales<sup>8</sup> like  $d^* \sim \text{Je} U/H$ , with  $\text{Je} \ll 1$ . Both these sets of scaling assumptions are equally consistent with the BNT equations, as we show now. In non-rescaled variables and according to BNT, the balance of transverse forces governing viscous buckling contains two main terms,  $N_{\alpha\beta}^{(0)} w_{,\alpha\beta}^{(1)} \sim M_{\alpha\beta,\alpha\beta}^{(1)}$ , see the first two terms in equation (15c), or equation (24). Inserting the constitutive laws (9), this yields  $\mu(\eta H) d_{\alpha\beta}^{(0)} w^{(1)}/H^2 \sim [\mu(\eta H)^3 w^{(1)}/(t^* H^2)]/H^2$ . Therefore, the growth rate for viscous buckling is set by the BNT model as

$$\frac{1}{t^*} \sim \frac{d^*}{\eta^2}.$$

This is consistent with Howell's scaling assumptions, namely  $1/t_{\text{H}}^* \sim U/(\eta^2 H)$  and  $d_{\text{H}}^* \sim U/H$ . This is also consistent with our assumptions,  $1/t^* \sim U/H$  and  $d^* \sim \text{Je} U/H$ , when gravity is assumed to scale as  $\text{Je} \sim \eta^2$  (as we did in §4 onwards).

Let us return to the assumption of uniform thickness  $h$ . As mentioned in §2.1, the fact that thickness is uniform across the width of the curtain is a simplifying assumption which could be relaxed: both the calculation of the base solution and of the buckling modes can be extended easily to any prescribed thickness distribution  $h(y)$  imposed by the injection slit. By contrast, the fact that the thickness is constant along the trajectories, *i.e.* along the vertical direction, is a consequence of our scaling assumptions (this is a general feature of BNT models: a similar results holds for Howell's scaling assumptions, see §2.2 in his paper). Indeed, the conservation of mass in the base solution can be written in general as  $\frac{1}{h} \frac{Dh}{Dt} + u_{\alpha\beta,\beta}^{(0)} = 0$ , where  $D/Dt$  denotes the convected derivative. The second term is a divergence, and it cancels for rigid-body motions: in view of equation (17a), it scales like  $(\text{Je} U/H)$  and is therefore negligible compared to the first term  $\frac{1}{h} \frac{Dh}{Dt}$  that scales like  $1/t^* \sim U/H$ . Dropping the second term, we can therefore write the conservation of mass as  $\frac{1}{h} \frac{Dh}{Dt} = 0$ , which implies that the thickness is passively advected on the time-scale relevant to viscous buckling; for a steady flow like the base state analyzed in §3, the thickness  $h$  is therefore constant along trajectories.

At the bottom of the domain, the viscous sheet meets with the bath and a boundary layer is formed. In this layer, the curtains slows down and becomes thicker, as described by Entov and Yarin (1984) in a 2D setting. A detailed representation of the layer would require coupling the plate model with a volumetric simulations for viscous fluids, which is beyond the scope of this paper. At the bottom of the curtain, we

<sup>7</sup>This initial guess turned out to be the relevant time scale for our problem: later on, we found that the *rescaled* growth rate  $\omega$  scales like  $\eta^2/\text{Je}$  in (27), and have assumed that  $\eta^2/\text{Je}$  is a quantity of order 1.

<sup>8</sup>In view of the decomposition (17a) of the velocity as a rigid-body motion plus a non-rigid correction arising from gravity, the rescaled strain rate reads  $d_{\alpha\beta}^{(0)} = u_{(\alpha,\beta)}^{(0)} = \text{Je} \bar{u}_{(\alpha,\beta)}^{(0)}$ . The original strain rate comes with an additional factor  $U/H$ , see §2.5.



have simply used the natural condition  $N_{xx} = 0$  in the absence of surface tension (see table 1 for the case with surface tension). This condition can be viewed as defining the top of the boundary layer, where the axial stress passes from tensile to compressive. This simple approximation ignores the possibility that this layer destabilizes by folding back and forth as studied by Mahadevan and Keller (1999); Skorobogatiy and Mahadevan (2007); Ribe (2003); such an instability could make the entire curtain unstable.

In a separate work, we have tried to apply to the viscous curtain the general tools developed for the stability analysis of open flows in fluid mechanics by Kupfer et al. (1987); Huerre and Monkewitz (1985, 1990); Chomaz (2005). These approaches propose a global stability criterion, which in principle should give the same phase diagrams as in figures 5 and 8, based on the local stability analyses of infinitesimal horizontal slices of the curtain. This approach is appealing for two reasons: it spares one from solving the eigenvalue problem in 2D, and it allows one to understand the global instability as the result of a competition between stable and unstable regions. However, we have found after extensive tests that this approach yields very poor results for the particular problem at hand: it works well only when the pre-stress varies slowly in the direction of the flow, which is not the case here.

## 7. Conclusion and perspectives

In this paper, we have analyzed a viscous sheet (curtain) falling into a bath in an open flow geometry. The curtain is stretched by the effect of gravity. By a Poisson effect, the curtain gets narrower as it falls down and compressive viscous stress are produced in the sheet in the transverse direction. The detailed distribution of the compressive stress has been found as a function of the aspect ratio of the sheet. The hanging viscous curtain can become unstable as a result of the compressive stress, by bending in the out-of-plane direction. This instability has been analyzed based on a two-dimensional viscous plate model. Finite-element simulations have been used to determine both the base flow and its linear stability. The eigenvalue problem for the linear stability involves a partial differential equation which is fifth order in space. This difficulty has been tackled by including a dedicated unknown representing the slope in the finite element formulation. Because of the open character of the flow and of the viscosity of fluid, the system is non-conservative and the eigenvalue problem governing stability is non-Hermitian. Indeed, we have found that the curtain can become unstable either by a static (buckling-like) bifurcation, or by a Hopf (flutter-like) bifurcation. We also showed how surface tension can be included in the model. For a mild surface tension, and have found that it tends to make the planar solution more stable.

Our plate model is applicable to a thin viscous plate undergoing moderate out-of-plane rotations, and having a viscous (Stokean) constitutive law. The model has been formulated in a modular manner: a strict separation has been enforced between the geometric approximations (moderate slopes), the constitutive assumptions (viscous fluid), and the equilibrium equations. As a result, it is straightforward to modify the present model to accommodate different geometric approximations (*e.g.* to a generic shell model allowing for finite rotations, or more specifically to an axisymmetric shell model), different constitutive laws, or to include inertia. Moreover, this modular approach makes it unnecessary to start the dimension reduction over from scratch whenever one of the modeling assumptions changes. This is an important benefit, as the reduction has been done several times from scratch already, see for example the work of Howell (1996); Ribe (2002); Pfingstag et al. (2011); in the present work, the complicated details of the dimensional reduction are captured by the simple equations (9a–9b).

Two possible extensions of the present work are particular worth of interest. The first one is to couple the mechanics with the temperature field: this is important for applications to glass, the viscosity of which varies enormously with temperature. The consequence of the non-homogeneous viscosity is that the bending and stretching moduli will vary in space. The second extension concerns finite rotations. The formulation of fully non-linear viscous shells raises no difficulty in principle: it suffices to replace our definitions of the strain rates and virtual strain with their fully non-linear counterparts. Numerical simulations of viscous shell undergoing large displacements, however, are difficult: in spite of recent efforts by Batty et al. (2012), no numerical method for non-linear viscous shells has been widely accepted and validated to date. In view of this, we hope that the numerical approach of the linear stability of thin viscous sheets pursued in this paper will be useful.

## Acknowledgments

We would like to thank N. Ribe, F. Viola, F. Gallaire and P. Huerre for helpful discussions. This paper is based on the preliminary work of Pfingstag (2010) done under the supervision of A. Boudaoud and of the second author of the present paper.

- Arne, W., Marheineke, N., Meister, A., Schiessl, S., Wegener, R., 2015. Finite volume approach for the instationary cosserat rod model describing the spinning of viscous jets. *Journal of Computational Physics* 294, 20–37.
- Balay, S., Abhyankar, S., Adams, M. F., Brown, J., Brune, P., Buschelman, K., Eijkhout, V., Gropp, W. D., Kaushik, D., Knepley, M. G., McInnes, L. C., Rupp, K., Smith, B. F., Zhang, H., 2014a. PETSc users manual. Tech. Rep. ANL-95/11 - Revision 3.3, Argonne National Laboratory.
- Balay, S., Abhyankar, S., Adams, M. F., Brown, J., Brune, P., Buschelman, K., Eijkhout, V., Gropp, W. D., Kaushik, D., Knepley, M. G., McInnes, L. C., Rupp, K., Smith, B. F., Zhang, H., 2014b. PETSc web page.
- Balay, S., Gropp, W. D., McInnes, L. C., Smith, B. F., 1997. Efficient management of parallelism in object-oriented numerical software libraries. *Modern software tools in scientific computing*, 163–202.
- Batty, C., Audoly, B., Grinspun, E., 2012. Discrete viscous sheets. *Transactions on Graphics* 31 (4), 113:1–7.
- Benjamin, T. B., Mullin, T., 1988. Buckling instabilities in layers of viscous liquid subjected to shearing. *Journal of Fluid Mechanics* 195, 523.
- Bergou, M., Audoly, B., Vouga, E., Wardetzky, M., Grinspun, E., 2010. Discrete viscous threads. *Transactions on Graphics* 29 (4), 116.
- Bhattacharya, S., Craster, R. V., Flynn, M. R., 2013. Buckling of a thin, viscous film in an axisymmetric geometry. *Physics of Fluids* 25 (4), 43102.
- Boudaoud, A., Chaieb, S., 2001. Singular thin viscous sheet. *Physical Review E: Statistical, nonlinear, and soft matter physics* 64 (5), 050601.
- Buckmaster, J. D., Nachman, A., 1978. The buckling and stretching of a viscida II. Effects of surface tension. *The Quarterly Journal of Mechanics and Applied Mathematics* 31 (2), 157–168.
- Buckmaster, J. D., Nachman, A., Ting, L., 1975. The buckling and stretching of a viscida. *Journal of Fluid Mechanics* 69 (1), 1–20.
- Cerda, E., Ravi-Chandar, K., Mahadevan, L., 2002. Thin films: Wrinkling of an elastic sheet under tension. *Nature* 419 (6907), 579–580.
- Chomaz, J.-M., 2005. Global instabilities in spatially developing flows: Non-normality and nonlinearity. *Annual Review of Fluid Mechanics* 37 (1), 357–392.
- da Silveira, R., Chaieb, S., Mahadevan, L., 2000. Rippling instability of a collapsing bubble. *Science* 287 (5457), 1468–1471.
- Debrégeas, G., 1998. The life and death of 'bare' viscous bubbles. *Science* 279 (5357), 1704–1707.
- Dockerty, S. M., 1967. Sheet forming apparatus. US Patent 3,338,696.
- Entov, V. M., Yarin, A. L., 1984. The dynamics of thin liquid jets in air. *J. Fluid Mech.* 140, 91–111.
- Filippov, A., Zheng, Z., 2010. Dynamics and shape instability of thin viscous sheets. *Physics of Fluids* 22 (2), 023601.
- Fischer, F. D., Rammerstorfer, F. G., Friedl, N., Wieser, W., 2000. Buckling phenomena related to rolling and levelling of sheet metal. *International Journal of Mechanical Sciences* 42 (10), 1887–1910.
- Friedl, N., Rammerstorfer, F. G., Fischer, F. D., 2000. Buckling of stretched strips. *Computers & Structures* 78 (1-3), 185–190.
- Healey, T. J., Li, Q., Cheng, R. B., 2013. Wrinkling behavior of highly stretched rectangular elastic films via parametric global bifurcation. *Journal of Nonlinear Science* 23 (5), 777–805.
- Hernandez, V., Roman, J. E., Vidal, V., 2005. SLEPc. *ACM Transactions on Mathematical Software* 31 (3), 351–362.
- Howell, P. D., 1996. Models for thin viscous sheets. *European Journal of Applied Mathematics* 7 (04), 321–343.
- Huerre, P., Monkewitz, P. A., 1985. Absolute and convective instabilities in free shear layers. *Journal of Fluid Mechanics*, 151–168.
- Huerre, P., Monkewitz, P. A., 1990. Local and global instabilities in spatially developing flows. *Annual Review of Fluid Mechanics* 22, 473–537.
- Jeffreys, H., 1925. The flow of water in an inclined channel of rectangular section. *Philosophical Magazine* 49, 793–807.
- Kim, T.-Y., Puntel, E., Fried, E., 2012. Numerical study of the wrinkling of a stretched thin sheet. *International Journal of Solids and Structures* 49 (5), 771–782.
- Kirk, B. S., Peterson, J. W., Stogner, R. H., Carey, G. F., 2006. libmesh : a C++ library for parallel adaptive mesh refinement/coarsening simulations. *Engineering with Computers* 22 (3-4), 237–254.
- Kupfer, K., Bers, A., Ram, A., 1987. The cusp map in the complex frequency plane for absolute instabilities. *Physics of Fluids*.
- Landau, L. D., Lifshitz, E. M., 1970. *Theory of elasticity. Course of Theoretical Physics*.
- Mahadevan, L., Keller, J. B., 1999. Periodic folding of thin sheets. *SIAM Review* 41 (1), 115–131.
- Nayyar, V., Ravi-Chandar, K., Huang, R., 2011. Stretch-induced stress patterns and wrinkles in hyperelastic thin sheets. *International Journal of Solids and Structures* 48 (25-26), 3471–3483.
- Parmelee, C. W., Lyon, K. C., Harman, C. G., 1939. The surface tensions of molten glass. *University of Illinois Bulletin*.
- Pfingstag, G., 2010. étude de la dynamique des plaques visqueuses. Ph.D. thesis, Université Pierre et Marie Curie, Paris.
- Pfingstag, G., Audoly, B., Boudaoud, A., 2011. Linear and nonlinear stability of floating viscous sheets. *Journal of Fluid Mechanics* 683, 112–148.
- Pilkington, L. A. B., 1969. The float glass process. *Proceedings of the Royal Society of London. Series A* 314 (1516), 1–25.
- Puntel, E., Deseri, L., Fried, E., 2011. Wrinkling of a stretched thin sheet. *Journal of Elasticity* 105 (1-2), 137–170.
- Rayleigh, J. W. S., 1878. *The theory of sound. Vol. 2.* Cambridge University Press, London.

- Ribe, N. M., 2001. Bending and stretching of thin viscous sheets. *Journal of Fluid Mechanics* 433, 135–160.
- Ribe, N. M., 2002. A general theory for the dynamics of thin viscous sheets. *Journal of Fluid Mechanics* 457, 255–283.
- Ribe, N. M., 2003. Periodic folding of viscous sheets. *Physical Review E (Statistical, Nonlinear, and Soft Matter Physics)* 68 (3), 036305.
- Ribe, N. M., 2004. Coiling of viscous jets. *Proceedings of the Royal Society A: Mathematical, Physical and Engineering Sciences* 460 (2051), 3223–3239.
- Ribe, N. M., 2008. Linearized model for a falling viscous curtain. Private communication.
- Ribe, N. M., 2012. All bent out of shape: buckling of sheared fluid layers. *Journal of Fluid Mechanics* 694, 1–4.
- Ribe, N. M., Stutzmann, E., Ren, Y., van der Hilst, R., 2007. Buckling instabilities of subducted lithosphere beneath the transition zone. *Earth and Planetary Science Letters* 254 (1–2), 173–179.
- Rustein, T., 2008. MEK4560 the finite element method in solid mechanics II. Tech. rep., Department of Mathematics University of Oslo, Oslo.
- Skorobogatiy, M., Mahadevan, L., 2007. Folding of viscous sheets and filaments. *Europhysics Letters* 52 (5), 532–538.
- Slim, A. C., Teichman, J., Mahadevan, L., 2012. Buckling of a thin-layer Couette flow. *Journal of Fluid Mechanics* 694, 5–28.
- Suleiman, S. M., Munson, B. R., 1981. Viscous buckling of thin fluid layers. *Physics of Fluids* 24 (1), 1–5.
- Taylor, G. I., 1958. *The Scientific Papers of Sir Geoffrey Ingram Taylor*. Cambridge University Press.
- Teichman, J., Mahadevan, L., 2003. The viscous catenary. *Journal of Fluid Mechanics* 478, 71–80.
- Triantafyllidis, N., Leroy, Y. M., 1994. Stability of a frictional material layer resting on a viscous half-space. *Journal of the Mechanics and Physics of Solids* 42 (1), 51–110.
- Xu, Y., Shu, C.-W., 2008. A local discontinuous Galerkin method for the camassa–holm equation. *SIAM Journal on Numerical Analysis* 46 (4), 1998–2021.
- Zheng, L., 2009. Wrinkling of dielectric elastomer membranes. Ph.D. thesis, California Institute of Technology.