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Theoretical prediction of Reynolds stresses and velocity profiles for barotropic turbulent jets

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Abstract –It is extremely uncommon to be able to predict the velocity profile of a turbulent flow. In two-dimensional flows, atmosphere dynamics, and plasma physics, large scale coherent jets are created through inverse energy transfers from small scales to the largest scales of the flow. We prove that in the limits of vanishing energy injection, vanishing friction, and small scale forcing, the velocity profile of a jet obeys an equation independent of the details of the forcing. We find another general relation for the maximal curvature of a jet and we give strong arguments to support the existence of an hydrodynamic instability at the point with minimal jet velocity. Those results are the first computations of Reynolds stresses and self consistent velocity profiles from the turbulent dynamics, and the first consistent analytic theory of zonal jets in barotropic turbulence.

Theoretical prediction of velocity profiles of inhomogeneous turbulent flows is a long standing challenge, since 2 the nineteenth century. It involves closing hierarchy for 3 the velocity moments, and for instance obtaining a relaл tion between the Reynolds stress and the velocity profile. Since Boussinesq in the nineteenth century, most of the approaches so far have been either empirical or phenomenological. Even for the simple case of a three dimensional 8 turbulent boundary layer, plausible but so far unjustified 9 similarity arguments may be used to derive von Kármán 10 logarithmic law for the turbulent boundary layer (see for 11 instance [1]), but the related von Kármán constant [2] has 12 never been computed theoretically. Still this problem is a 13 crucial one and has some implications in most of scientific 14 fields, in physics, astrophysics, climate dynamics, and en-15 gineering. Equations (6-7), (9), and (11) are probably the 16 first prediction of the velocity profile for turbulent flows, 17 and relevant for barotropic flows. 18

In this paper we find a way to close the hierarchy of the velocity moments, for the equation of barotropic flows with or without effect of the Coriolis force. This two dimensional model is relevant for laboratory experiments of fluid turbulence [3], liquid metals [4], plasma [5], and is a key toy model for understanding planetary jet formation [6] and basics aspects of plasma dynamics on Tokamaks in relation with drift waves and zonal flow formation [7]. It is also a relevant model for Jupiter troposphere organization [8]. Moreover, our approach should have future implications for more complex turbulent boundary layers, which are crucial in climate dynamics in order to quantify momentum and energy transfers between the atmosphere and the ocean.

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It has been realized since the sixties and seventies in 33 the atmosphere dynamics and plasma communities that in 34 some regimes two dimensional turbulent flows are strongly 35 dominated by large scale coherent structures. Jets and 36 large vortices are often observed in numerical simulations 37 or in experiments, but the general mechanism leading to 38 such an organization of the flow at large scales is subtle 39 and far from being understood. For simplicity, we con-40 sider in this paper the case of parallel jets favored by 41 the β effect, however without β effect both jets and vor-42 tices can be observed [4, 9, 10]. When a large scale struc-43 ture is created by the flow, a quasilinear approach may 44 be relevant. Such a quasilinear approach requires solving 45 a coupled equation for the mean flow and the Lyapunov 46 equation that describes the fluctuations with a Gaussian 47 approximation, just like the Lenard–Balescu equation in 48 plasma kinetic theory. Numerical approaches and theo-49 retical analysis have been systematically developed for fif-50

teen years in order to solve and study such quasilinear or 51 related approximations [6, 11]. In a recent theoretical pa-52 per, the range of validity of such an approach has been 53 established by proving the self consistency of the approxi-54 mations of weak forcing and dissipation limit [12]. While 55 this work gave theoretical ground to the approach, explicit 56 formula for the Reynolds stress cannot be expected in gen-57 eral. However, in a recent work [13], an expression for the 58 Reynolds stress has been derived from the momentum and 59 energy balance equations by neglecting the perturbation 60 cubic terms in the energy balance (this follows from the 61 quasilinear approach justification [12]), but also neglect-62 ing pressure terms (not justified so far, see [14]). This ap-63 proach surprisingly predicts a constant velocity profile for 64 the outer region of a large scale vortex in two dimensions 65 that does not depend on the detailed characteristics of the 66 stochastic forcing but only on the total energy injection 67 rate ϵ expressed in m²s⁻³. Another analytic expression 68 for the Reynolds stress has also been derived in the par-69 ticular case of a linear velocity profile U in [15]. For the 70 case of dipoles for the 2D Navier–Stokes equations, the pa-71 pers [16,17] following a computation analogous to the one 72 given in [15], shows that if the vorticity is passively ad-73 vected (the third term in equation (5) below is neglected), 74 then the expression for the Reynolds stress discussed in 75 [10] is recovered ([17] also discusses other interesting as-76 pects related to parts of the flow for which this relation 77 is not correct). What are the criteria for the validity of 78 these results? Can we reconcile the different results giving 79 a full theoretical justification and extend these for more 80 general cases? 81

We start from the equations for a barotropic flow on a periodic beta plane with stochastic forcing

$$\partial_t \mathbf{V} + \mathbf{V} \cdot \nabla \mathbf{V} = -r \mathbf{V} - \frac{1}{\rho} \nabla P + \beta_d y \begin{pmatrix} V_y \\ -V_x \end{pmatrix} + \sqrt{2\epsilon} \mathbf{f} \quad (1)$$

where $\mathbf{V} := \begin{pmatrix} V_x \\ V_y \end{pmatrix}$ is the two dimensional velocity field with $\nabla \mathbf{V} = 0$. r models a linear friction, and f is a stochastic force white in time, with energy injection rate ϵ , β_d is the Coriolis parameter, y the north-south coordinate. Following [12] we choose time and space units such that the mean kinetic energy is 1, and $L_x = 1$. The non dimensional equations for the vorticity $\Omega = \nabla \wedge \mathbf{V}$ are

$$\partial_t \Omega + \mathbf{V} \cdot \nabla \Omega = -\alpha \Omega - \beta V_y + \sqrt{2\alpha} \eta \tag{2}$$

where $\eta = \nabla \wedge f$, and **V** denotes from now on the nondi-82 mensional velocity. Now $\alpha = L\sqrt{\frac{r^3}{\epsilon}}$ is a nondimensional parameter although we will often refer to it as the "fric-83 84 tion". $\beta = \sqrt{\frac{r}{\epsilon}} L^2 \beta_d$ is the nondimensional Coriolis param-85 eter. Eq. (2) still has three nondimensional parameters, 86 87 α, β and K, the typical Fourier wavenumber where energy is injected. 88

Neglecting the pressure and cubic terms in the energy balance and enstrophy balance, it is straightforward to obtain the Reynolds stress expression

$$\langle uv \rangle = \frac{\epsilon}{U'} \tag{3}$$

where U' = dU/dy. This generalizes the result obtained 89 for a vortex [13] to the case of a jet with mean velocity 90 U. Is it possible to justify those hypothesis on theoretical 91 ground, uncover the validity range of (3), and to generalize 92 it? We note that detailed numerical studies of the energy 93 balanced has been discussed in several papers [10, 18, 19]. 94

In order to derive eq. (3), the key idea is to use the al-95 ready justified [12] quasilinear approximation in the limit of small forces and friction (inertial regime, $\alpha \ll 1$), and to 97 further consider the limit of small scale forcing $(K \gg 1)$, 98 with fixed β . In these limits, energy is injected at small 99 scale and is dissipated at the largest scale of the flow. 100 $\alpha \ll 1$ is the proper regime for most geophysical turbulent 101 flows, for instance for giant gaseous planets like Jupiter 102 [20,21], and many two dimensional or rotating turbulence 103 experiments. The small scale forcing limit $K \gg 1$ is the 104 most common framework for turbulence studies (see for 105 ex. [14]) and relevant for Jupiter troposphere. Also, com-106 puting the pressure from the Navier-Stokes equations in-107 volves inverting a Laplacian. It is thus natural to expect 108 the pressure term to have a power expansion in the pa-109 rameter $\frac{1}{K}$, and thus vanish in the limit of large K. The 110 main idea is then to separate the flow \mathbf{V} in two parts, 111 $\mathbf{V}(\mathbf{r},t) = U(y,t)\mathbf{e}_x + \left(\begin{array}{c} u(\mathbf{r},t) \\ v(\mathbf{r},t) \end{array}\right).$ The mean velocity 112 $U(y)\mathbf{e}_x = \frac{1}{L_x} \int \mathrm{d}x \mathbb{E}[\mathbf{V}(x,y)]$ called the mean flow or zonal 113 flow, is defined as both the zonal and stochastic average of 114 the velocity field. In the following, the bracket $\langle \rangle$ will be 115 used for this zonal and stochastic average. We are left with 116 two coupled equations, one governing the dynamics of the 117 mean flow, the other one describing the evolution of ed-118 dies. In the limit where α is small, it has been proven that 119 fluctuations are of order $\sqrt{\alpha}$ and thus it is self-consistent 120 to neglect nonlinear terms in the equation for fluctuations 121 [12]. Then one can justify [12] that, at leading order in α , 122 the full velocity field statistics are described by a quasi-123 Gaussian field (the velocity field is not Gaussian, but the 124 marginals when the zonal flow is fixed are Gaussian, jus-125 tifying a posteriori a second order closure corresponding

$$\partial_t U = -\alpha \left[\partial_y \left\langle uv \right\rangle + U \right] \tag{4}$$

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$$\partial_t \omega + U \partial_x \omega + (\beta - U'')v = -\alpha \omega + \eta \tag{5}$$

where we have introduced $\omega = \partial_x v - \partial_y u = \Delta \psi$, the 129 vorticity of the fluctuations. Eq. (4) shows that the typical 130 time scale for the evolution of the mean flow U is $\frac{1}{\alpha}$ which 131 is, following our assumption $\alpha \ll 1$, much larger than the 132 time scale for the evolution of eddies. Using this time scale 133 separation, we will consider that U is a constant field in 134 the second Eq. (5), and we will always solve $\omega(t)$ for a 135 given U. We follow the strategy: 136

to the quasilinear approximation). Using also the incom-

pressibility condition, we obtain the quasilinear model

- First we solve the linear Eq. (5) and compute the 137 stationary distribution $\langle \omega^2 \rangle$ as a functional of U. 138
- The enstrophy balance for the fluctuations allows us 139

- to relate $\langle \omega^2 \rangle$ to the divergence of the Reynolds stress tensor (see the Supplementary Material [22]).
- Last we can use this expression to close the first Eq. (4), and discuss possible stationary profiles U.

To reach the first objective, we take advantage of the asymptotic regimes $\alpha \to 0$ and $K \to \infty$. When we take those two limits, it is natural to ask whether they commute or not, and which nondimensional parameter will govern the difference between $\alpha \to 0$ first or $K \to \infty$ first. Our asymptotic calculations show that the key parameter is the ratio between $\frac{U''}{K}$ and α . Taking the limit $\alpha \to 0$ first amounts to saying that $\frac{U''}{\alpha K}$ is very large. We first take the limit $K \to \infty$ while keeping α small

We first take the limit $K \to \infty$ while keeping α small but finite. The idea is to write Eq. (5) in an integral form using the Green function of the Laplacian, and use the fact that the Green function decreases very fast when K is large, which implies that the evolution of the flow is local in space. At this stage of the calculation, α is small but finite, and the expression for the Reynolds stress depends both on α and on the properties of the stochastic forcing. The complete calculation is reported in the Supplementary Material [22]. We emphasize that as long as α is kept finite, the Reynolds stress depends on the Fouri²⁷ spectrum of the stochastic forcing η . The result shows that the Reynolds stress can be expressed analytically as

$$\langle uv \rangle = \frac{1}{2\alpha} \chi \left(\frac{U'}{2\alpha} \right),$$
 (6)

where the explicit expression of χ is a parametric integral, see [22]. The stationary profile U thus verifies $\frac{U''}{2\alpha}\frac{1}{2\alpha}\chi'\left(\frac{U'}{2\alpha}\right) = -U$, which can be integrated using a primitive X of the function $x \to x\chi'(x)$ in

$$X\left(\frac{U'}{2\alpha}\right) + \frac{1}{2}U^2 = C,\tag{7}$$

where C is the integration constant. It is in itself remarkable that for some range of the parameters, the flow of the barotropic quasilinear model can be computed from a Newtonian equation like (7).

In Eq. (7), X plays the role of a potential as if the equa-156 tion would describe a particle moving in a one-dimensional 157 potential. The constant in the right-hand side is set by 158 U'(0) and depending on the value of this constant, there 159 can be one, two or three solutions as shown in Fig. (1). 160 If $C > X_{max}$, there is one solution for which U never 161 vanishes. As the total flow momentum is zero, such so-162 lutions with either U > 0 or U < 0 are not physical. If 163 $X_0 < C < X_{max}$, there are three possible solutions, one is 164 periodic, the other two diverge. The periodic solution corresponds to $\frac{U'}{2\alpha}$ confined in the well of X. In that case the 165 166 flow is periodic and the solution exchanges kinetic energy $\langle \cdot \rangle$ 167 in the term $\frac{1}{2}U^2$ with potential energy $X\left(\frac{U'}{2\alpha}\right)$. Outside 168 the well, the solutions are diverging, one corresponds to 169 an increasing U and the other to a decreasing U. A linear 170 stability analysis of the periodic solution of (7) shows that 171



Fig. 1: In the limit of small scale forcing, the mean flow can be computed analytically from the Newtonian Eq. (7). We show here that the situation is analogous to a particle moving in a one-dimensional potential. The blue curve displays the potential X appearing in Eq.(7). If the constant of motion is less than X_{max} , there are two classes of solutions: the solution can be confined in the central well and is thus regular and periodic in space, or it is outside the well, and in this latter case it diverges.

this solution is unstable whereas the diverging solution is stable. Thus, the periodic regular solution is not a suitable 173 candidate for the stationary mean velocity profile U(y). If 174 we now take the limit of vanishing α in expression (6), 175 a straightforward calculation using the explicit expression 176 of χ (given in [22]) allows us to recover expression (3) for 177 the Reynolds stress because $U' \langle uv \rangle \xrightarrow[\alpha \to 0]{} 1$ which is Eq. 178 (3) with dimensional units. The physical interpretation of 179 the limiting case (3) is very enlightening. The term $U' \langle uv \rangle$ 180 can be interpreted as the rate of energy transferred from 181 small scale to large scale, therefore expression (3) is con-182 sistent: with the limit of large K, the evolution of eddies 183 becomes local as if the perturbation only sees a region of 184 width $\frac{1}{K}$ around itself, and thus the different parts of the 185 flow are decoupled. The other limit of small α forces the 186 energy to go to the largest scale to be dissipated because 187 the dissipation at small scales becomes negligible. 188

Let us study the other limit where α goes to zero first. The techniques used in this second case are very different than the previous one. We assume in this section that the linearized dynamics has no unstable modes. The calculation involves Laplace transform tools that were used in [23] to study the asymptotic stability of the linearized Euler equations. In the limit $\alpha \ll 1$, using [12], we derive in the Supplementary Material [22] the relation between the Reynolds stress divergence and the long time behavior of a disturbance $\omega(y, 0)$ carried by a mean flow U(y). This is an old problem in hydrodynamics, one has to solve the celebrated Rayleigh equation

$$\left(\frac{d^2}{dy^2} - k^2\right)\varphi_{\delta}(y,c) + \frac{\beta - U''(y)}{U(y) - c - i\delta}\varphi_{\delta}(y,c) = \frac{\omega(y,0)}{ik(U(y) - c - i\delta)},$$
(8)

where $\varphi_{\delta}(y,c) := \int_{0}^{\infty} dt \psi(y,t) e^{-ik(c+i\delta)t}$ is the Laplace 189 transform of the stream function, and k is the x- 190 component of the wavevector. The Laplace transform φ_{δ} 191 is well defined for any non zero value of the real variable 192

 δ with a strictly negative product $k\delta$. c has to be understood as the phase speed of the wave, and $k\delta$ is the 194 (negative) exponential growth rate of the wave. Involved 195 computations are then required to give the explicit expres-196 sion of the Reynolds stress. Let us just mention that the 197 difficulty comes from the fact that we have to take the 198 limit $\delta \to 0$ first in Eq. (8) before $K \to \infty$. Then we can 199 relate $\omega(y, \infty)$ to the Laplace transform φ_{δ} taken at $\delta = 0$. 200 The expression for the Reynolds stress in the inertial limit 201 involves an integral with the profile U in the denominator 202 of the integrand. The integral is defined only in regions 203 of the flow where U' does not vanish, or to state it more 204 precisely, where the parameter $\frac{KU'}{U''}$ is large. Therefore, there exists a small region of size $\frac{1}{K}$ around the maximum 205 206 where the asymptotic expansion breaks down. In the outer 207 region away from the extrema, we recover the expression 208 (3) in the inertial limit $\alpha \to 0$ first. For strictly monotonic 209 velocity profiles U, we conclude that the inertial limit and 210 the small scale forcing limit commute and that expression 211 (3) is expected to be valid. 212

Using expression (3) for the Reynolds stress, we can solve Eq. (4) for the stationary profile. It writes

$$\frac{\mathrm{d}}{\mathrm{d}y}\left(\frac{\epsilon}{U'}\right) + U = 0. \tag{9}$$

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Whatever the value of the free parameter U'(0), all pro-213 files U are diverging in finite length. An example of such 214 a profile U is given in Fig. (2a). In red, we have plotted 215 different solutions of Eq. (9) together such that the mean 216 velocity profile U is composed of many diverging jets. In 217 blue, we have drawn at hand what we expect qualitatively 218 from a real velocity profile. An example of a real profile 219 obtained by numerical simulations in [24] is displayed in 220 Fig.(2b). The fact that Eq. (9) predicts diverging pro-221 files U shows that the expression for the Reynolds stress 222 (3) is not valid everywhere in the flow, but it holds only 223 in the spatial subdomains where the flow is monotonic, 224 not at the extrema. We observe that both divergences 225 are regularized, by a cusp at the eastward jet maximum, 226 and by a parabolic profile at the westward jet minimum. 227 The second aim of this paper is to explain the asymmetric 228 regularization of the eastward and westward jets. 229

Numerical simulations like the one performed in Fig. (2b) show that the mean velocity profile is regularized at a very small scale at its maximum. As we explained previously, there exists a region of typical size $\frac{1}{K}$ around the maximum where the asymptotic expansion for the Reynolds stress breaks down. It is thus natural to choose the ansatz $\tilde{U}(y) := U\left(\frac{y}{K}\right)$. The scaling in $\frac{1}{K}$ implies that the ratio $\frac{U''}{K\alpha}$ is very large at the cusp because $U'' \propto K^2$. The cusp is then described by the inhomogeneous Rayleigh Eq. (8). If we put the ansatz \tilde{U} in (8) and consider the limit of large K, we get

$$\left(\frac{d^2}{dy^2} - \tan^2\theta\right)\varphi_{\delta}(y,c) - \frac{\tilde{U}''(y)}{\tilde{U}(y) - c - i\delta}\varphi_{\delta}(y,c) = \frac{e^{i\sin\theta y}}{\tilde{U}(y) - c - i\delta},\tag{10}$$



Fig. 2: (a) Jet profiles obtained as an analytic solution of (9) (red curve) together with a qualitative real velocity profile (blue curve). (b) Real velocity profile obtained in numerical simulations [24]. The analytic solution for U has divergences at the extrema that are regularized by a cusp at the eastward jet and a parabola at the westward jet.

where $\cos \theta = \frac{k}{K}$. The solution of this equation with $\delta \to 0$ gives us the Laplace transform of the stream func-231 tion with which we can express the Reynolds stress. The 232 β -effect disappears completely from the equation of the 233 cusp because in the region of the cusp, the curvature is 234 so large that it overcomes completely the β -effect, as can 235 be seen on the green curve of Fig. (2b). Eq. (10) to-236 gether with the equations linking the Reynolds stress with 237 φ have a numerical solution, which proves that our scaling 238 $U(y) := U\left(\frac{y}{K}\right)$ is self-consistent. This solution can not be 239 expressed analytically and depends on the Fourier spec-240 trum of the stochastic forcing. An example of a numerical 241 integration of the Reynolds stress divergence $-\partial_y \langle uv \rangle$ for 242 the cusp is displayed in Fig. (3) for $\widetilde{U}(y) = -\frac{y^2}{2}$ and a 243 stochastic forcing with a semi annular Fourier spectrum 244 where θ ranges between $-\frac{\pi}{3}$ and $\frac{\pi}{3}$. In Fig. (3), the red 245 curve is the asymptote $\frac{\widetilde{U}''}{\widetilde{U}'^2}$ obtained from formula (3) with $\epsilon = 1$. When a jet is in a stationary state, the cusp profile 246 247 joins smoothly the outer region of the jet where the result 248 (3) is valid. 249

Another physical phenomenon at the maximum of the jet is called "depletion at the stationary streamlines" and has first been observed in [23]. It means that at critical latitudes where U' = 0, any vorticity perturbation of the flow ω_0 has to asymptotically vanish with time. One main consequence of this phenomenon is the relation

$$U(y_{cr}) = -\frac{\epsilon K^2}{r U''(y_{cr})},\tag{11}$$

where y_{cr} is the latitude of the extremum. From (11) we learn that, even if the velocity profile of the cusp depends on the details of the forcing, the maximal curvature of the profile satisfies a more general relation, and it would therefore be very interesting to check it in full numerical 254



Fig. 3: The Reynolds stress divergence $-\partial_y \langle uv \rangle$ (blue curve) computed for a parabolic profile $\tilde{U}(y) = -\frac{y^2}{2}$ around its maximum y = 0. It is satisfying to observe that the red curve $\frac{\tilde{U}''}{\tilde{U}'^2}$ matches the blue curve for large y according to expression (3).

simulations with different types of forcing spectra.

The previous discussion successfully explained the jet 256 regularization of the eastward jet cusp. As clearly ob-257 served in Fig. (2b), and from Jupiter data, westward jets 258 do not produce cusps. At first sight, it may seem that all 259 the theoretical arguments used so far, the energy balance, 260 the asymptotic expansions, and the results (3) and (11), 261 do not break the symmetry between eastward and west-262 ward jets, as β disappears from all these computations. 263 However, as clearly stressed in [12], the whole theoretical 264 approach relies on an assumption of hydrodynamic stabil-265 ity for U. The asymmetry is clearly visible in the Rayleigh-266 Kuo criterion, that states that when $\beta - U''$ change sign, an 267 instability may develop. We will now argue that the tur-268 bulent flow is constantly oscillating between a stable and 269 unstable solution, in order to control the westward jet be-270 havior. This means that the flow is not linearly stable, but 271 only marginally stable. As shown in the following, the in-272 stability is localized at the extremum of the westward jet, 273 and the unstable mode has a very small spatial extension. 274 That's why the flow can be considered as stable away from 275 the westward extremum of the jet and the assumptions of 276 expression (3) are satisfied. 277

To check this marginal stability hypothesis, we solved numerically the homogeneous PDE for a perturbation carried by a mean flow

$$\partial_t \omega + ikU\omega + ik(\beta - U'')\psi = 0.$$
(12)

We chose a parabolic mean velocity with a small perturbation at its minimum $U(y) = \gamma \frac{y^2}{2} - \eta e^{-\frac{y^2}{\sigma^2}}$. The main curvature γ is chosen to be slightly smaller than β , and we performed simulations using different values of η and σ . The result of one of those simulations is displayed in figure (4). The red curve shows the Rayleigh-Kuo criterion



Fig. 4: Amplitude of the perturbation $|\omega^2(y,t)| - 1$ (blue curve) carried by a mean flow U. The Rayleigh-Kuo criterion $\beta - U''$ is displayed in red. The simulation shows that the perturbation grows exponentially with the complex rate c.

 $\beta - U''$ which is locally violated around y = 0. The blue curve shows the amplitude of the perturbation $|\omega^2(y,t)|-1$ from which we can then express the Reynolds stress divergence. We check that the growth rate of the perturbation is indeed exponential with time with a complex rate c.

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In this paper we gave a global consistent picture of how 289 a zonal jet is sustained in a steady state through con-290 tinuous energy transfer from small scale to large scale. 291 Eq. (7) and (9) are probably the most striking results of 292 this work, showing for the first time that it is possible to 293 find closed equations for the velocity profile of turbulent 294 flows. It illustrates that, although far from equilibrium, 295 turbulent flow velocity profiles may be described by self 296 consistent equations, as density of other macroscopic pro-297 files can be described in condensed matter physics. This 298 is a fundamental property which existence is far from ob-299 vious, and that no other approach was able to establish 300 so far. In this paper, we have considered the case of a 301 Rayleigh friction as the mechanism for removing energy 302 that is transferred to the largest scale. Rayleigh friction 303 is a rather ad-hoc type of damping (although it can be 304 justified in certain cases involving, e.g., Ekman pumping). 305 If one would consider other kinds of friction, for instance 306 scale-selective damping, provided that this damping actu-307 ally acts on the largest scales of the flow, in a correspond-308 ing inertial limit we expect most of our results to easily 309 generalize (the proper dissipation operator should then re-310 place Rayleigh friction in equation (4)). Indeed the pro-311 cesses that explain the computation of the Reynolds stress 312 are inertial in nature and independent from the dissipation 313 mechanisms. As an example, the case of viscous dissipa-314

tion has been discussed in section 5.1.2 of [12], showing that while the regularization by viscosity is very different 316 from the regularization by linear friction, the inertial re-317 sults for the Reynolds stresses do coincide. This is true as 318 far as the shear is non zero and equation (3) is concerned; 319 by contrast the regularization of the cusp is dissipation 320 dependent. In some specific cases, changing the dissipa-321 tion mechanism may induce specific instability modes, like 322 boundary layer modes due to viscous dissipation, however 323 such effects are expected to be non generic. Extension and 324 generalizations of our approach can be foreseen for other 325 geometries (on the sphere), for more comprehensive quasi-326 geostrophic models of atmosphere jets, and for classes of 327 flows dominated by a strong mean jet, for instance in some 328 instances of boundary layer theory. 329

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