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Olivier Dorival, Pablo Navarro, Steven Marguet, Caroline Petiot, Michel Bermudez, et al.. Experimental results of medium velocity impact tests for reinforced foam core braided composite structures. Journal of Sandwich Structures and Materials, 2016, 10.1177/1099636216650990. hal-01325205

# HAL Id: hal-01325205 https://hal.science/hal-01325205

Submitted on 1 Jun2016

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# Experimental results of medium velocity impact tests for reinforced foam core braided composite structures

Journal Title XX(X):2–25 ©The Author(s) 2016 Reprints and permission: sagepub.co.uk/journalsPermissions.nav DOI: 10.1177/ToBeAssigned www.sagepub.com/



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### Abstract

Absorbing impact energy at sub-system level is an attractive idea that is emphasized by new composite reinforcement techniques such as stitching or pinning. This paper reports experimental results of medium velocity impact tests carried out on several arrangements of reinforced foam/braided composite structures. The tests consisted of a steel ball shot at a velocity of 110 m/s from a gas gun impacting the structures on their leading edge. Post-mortem tomography analysis delivered very rich information which shed light on the damage mechanisms that the composite structures underwent. In addition, two fast-speed cameras were used to derive the energy absorption during the impact. Absorption capabilities were also compared with those of dynamic crushing tests (reported in a companion paper) and some designs clearly exhibited promising behavior as shock absorbers.

#### Keywords

Foam/carbon-kevlar braided composite structures, impact energy absorption, gas canon ball impact tests, composite reinforcement, composite damage mechanisms, shock absorbers, sandwich structures, computed micro-tomography.

### Introduction

Over the past decades, weight optimization has been an ongoing challenge in a number of fields such as aeronautics, marine, automotive technologies, to name but a few. Multi-materials, such as composites, offer infinite possibilities to achieve this goal but, at the same time, they make predictions difficult with either analytical or numerical models. This is particularly the case in the context of composite structures which may experience impact loadings (1) due to tool drops, runway debris, bird strikes, hailstorms or ballistic loading, which make experimental analysis still a major method in both the academic and industrial research in this field. Among other lightened structures that can experience impacts during their life, foam core composite structures have shown interesting performances and have consequently raised significant interest. One key step is to identify the modes of damage raised by impact loadings. In (2) foam-skin debonding was found to start the initiation of damage under drop weight tests. In (3) composite structures similar to helicopter blades were impacted by a steel ball shot at a velocity of up to 140 m/s using a gas gun and the authors identified the following damage scenario: damage of the front edge, skin-foam delamination, damage of the roving and penetration.

In order to improve the response of foam core composite structures to impact loading, one can act on the facesheets or on the core. Concerning the facesheets, some studies have shown that incorporation of additional elements in the resin prior to molding can increase the impact resistance by a few percent (4; 5; 6). A more evident way of reinforcing facesheets relies on the reinforcement of the skin used for the external faces. For instance, Villanueva and Cantwell (7) compared fiber metal laminate reinforced skin with unidirectional glass fiber polypropylene and woven glass fiber polypropylene under medium velocity impact. Zhou et al. (8) studied the impact damage and energy absorption of 3D braided composite tubes and found a good correlation between experimental and numerical results.

However, since impact loadings mainly put a strain on the out-of-plane behavior, which is more dependent on the core materials than on the facesheets, a number of researchers have focused on the core properties of foam core sandwiches. Anderson and Madenci (9) compared damage after low velocity impact on sandwiches made of carbon fabric with two thicknesses and a core made of either Rohacell foam or honeycombs with two densities for both types of cores. Cantwell et al. (10; 11; 12)

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investigated the influence of various foam properties on the quasi-static and low/medium velocity perforation resistance of composite sandwiches. Nasirzadeh and Sabet (13) found significant influence of the foam density under medium velocity ballistic impact (100-150 m/s). They showed that medium density foams had better energy absorption and better ballistic limits than low and high density foam sandwiches. Some studies (14; 15) also investigated the impact resistance of sandwiches involving layered graded foam cores and showed that they can outperform monolithic foam core sandwiches in terms of impact resistance.

Besides playing on the mechanical properties of the foam core, a growing number of studies have focused on possible additional reinforcements of foam cores by means of discrete elements such as lattices, pins, stitches, etc. The idea is to stiffen the sandwich behavior in the out-of-plane direction, to delay core/facesheets debonding, while creating numerous small sources of dissipation during the crushing stage. Various hollow cores (without foam) have been developed and tested; however, foam filling is regularly found to stabilize the reinforcements thus avoiding premature buckling. It also provides additional modes of energy consumption by crushing and densification of the foam. Pitarresi et al (16) studied several reinforced sandwich structures used as walls of hollow rectangular structures. Foam reinforcements were shown to improve energy absorption during quasi-static edgewise compression as they maintain the structure in a continuous crushing process. Li and Muthyala (17) showed the improved characteristics of a sandwich made of an orthogrid stiffened syntactic foam core and glass/epoxy facesheets under low velocity impact. Zhang et al (18) studied a sandwich composed of aluminum pyramidal lattice core surrounded by polyurethane foam between unidirectional carbon/epoxy laminates. The foam is shown to improve the compression stiffness of the lattice core by improving the resistance of each truss beam to buckling. Vaidya et al. (19) tested 3D curved core piles woven into the skins with/without foam under low velocity impact. They found that the main damage mechanisms without foam were the buckling of the core pile and the rupture of the facesheets, whereas with foam, the structure mainly underwent foam core crushing with core pile failure. Wang et al. (20) determined the static properties of foam sandwich reinforced with composite columns with various reinforcement densities. Reinforcements were found to enhance the plasticity capability of the structure. Zhou et al (21) studied the behavior of PVC foam core reinforced with glass or carbon composite under quasi-static compression and low velocity impact test. They found that compressive strength and energy absorption can be optimized by choosing a medium density foam combined with thin, but stiff, carbon rods. Vaidya et al. (22) showed that foam core sandwiches with foam reinforced by titanium Zpins or glass/epoxy Z-pins, when subjected to high strain rate impact testing, experienced limited foam crushing compared to the unreinforced foam core. Nanayakkara et al. (23) reported enhancement of Z-pinned sandwich composite structures under quasi-static global compression.

Through-the-thickness stitching is another interesting reinforcement technique which brings additional stiffness and avoids core crack propagation. Stitching density was found to be a key parameter. Stanley and Adams (24) reported that stitching greatly improved stiffness, strength and energy absorption capabilities for quasi-static bending tests, out-of-plane tensile tests, core shear tests and edgewise compression tests, as well as for low velocity indentation tests. Similar results were reported in (25) and (26), except for the energy absorption; divergences may be due to the different stiffness and mass characteristics used for the foam and the reinforcements. Xia and Wu (27) also found that reinforcements enhanced the performances during low velocity impact on a polyurethane foam sandwich with plain glass fabrics reinforced by kevlar throughthickness stitches oriented at  $90^{\circ}$ . Whether the additional mass brought by the stitches finally deteriorates the specific properties (i.e. properties per unit mass) of the structure is still an open question and researchers have come to different conclusions (25; 28), again depending on the parameters used in the different studies. Stitching orientation is also a significant parameter. Angle cross-pattern stitching was found to enhance the quasi-static performances even more than 90° stitching, in particular in core shear testing (24). Influence of the stitching angle on the impact performances has been the focus of several papers. Singh and La Saponara (29) suggested that  $90^{\circ}$  stitching may not be the best choice for damage tolerance. In (30), it was shown that cross-pattern stitches improved the low velocity impact response of glass/polyurethane foam sandwiches by limiting core/skin debonding. Stitch breaking was found to be the dominating energy consumption mode. Samlal et al. (31) arrived at similar conclusions for 45° stitching under low velocity impact tests. Tekalur et al. (32) found that effects of through-thethickness stitches were also important in very high velocity impacts (1100 m/s) as stitches helped the transfer of the force to the back skin (this is not the case with unstitched foam sandwich since the dynamics are very high frequency). However, Guan et al. (28) reported that the blast resistance of stitched sandwich panels was not significantly enhanced due to the additional mass to be considered. Finally, diverging conclusions clearly suggest that there is a need for optimization of the stitching materials and parameters.

Although energy absorption strongly depends on the damage mechanisms, precise investigations about the damage pattern are seldom studied by means of non-intrusive techniques such as tomography or X-ray techniques. Feng and Aymerich (33) analyzed PVC foam/carbon epoxy sandwiches under low velocity impact tests at several impact energies and showed a good agreement with FE modeling. Damage facesheets were carefully analyzed by X-radiography.

In this paper, we report experimental results of medium velocity impact tests performed on several designs of foam based composite structures that may be involved in several aircraft parts such as rotor blades, wing parts, fuselages, etc. The various specimens were designed with the idea of developing deformation and damage mechanisms of high convenience for absorbing the projectile kinetic energy during the impact. Following this idea, all the specimens were designed with the same external shape as illustrated in Fig. 1. The same medium density foam and the same external braidings were used. However the designs differ by various reinforcement techniques involving inner composite walls and/or discrete reinforcement techniques such as the use of inner walls, stitching and sewing. All the designs were provided by Airbus Group according to Airbus standards or Airbus specifications. For cost reasons, the number of samples for each design was limited. We chose to challenge each design under

various loading conditions, including dynamic crushing tests and gas gun tests. For these reasons, two samples were tested in gas gun tests under the same conditions to check the repeatability, and eventually a third one was performed to validate in case the test results varied due to experimental vagaries. In this paper, we report only the comparisons of impact tests on the leading edge with 20 mm steel ball. The damage scenario identified on the most promising sample is also validated with a more energetic 30 mm impact test.

This paper is organized as follows: after the experimental set-up is described, postmortem analyses of each specimen are reported based on tomography analysis. They provide a very precise understanding of the damage mechanisms developed during the impacts. A more energetic impact test is then reported concerning the design which was clearly identified as the most promising, in order to validate the damage scenario assumed. Energy absorption values are then given which allows us to emphasize the strengths and weaknesses of each design. Finally, these results are compared with that of low velocity dynamic crushing tests detailed in a complementary paper (34). Although the two types of loadings are basically different, some reinforcement techniques look very promising in terms of their capability to absorb impact energy either during a large crushing event or a medium velocity localized impact.



Figure 1. Dimensions of the external shape for any design.

#### Experimental set-up

A gas canon was used to shoot a 20 mm diameter steel ball at a speed of 110 m/s. The principle was to accelerate the ball in a tube in which a pressure wave was suddenly created. With this in mind, the pressure in a pressurized tank was increased to a given value and was released when the trigger started. The pressure then pushed the ball, which was stabilized in a foam part. The velocity of 110 m/s was reached when the ball exited from the tube. The projectile impacted the composite structure on its leading edge with



Figure 2. Experimental set-up.

an impact energy of 198 J. During the tests, the structure was simply supported on a solid frame as illustrated in Fig. 2. The aim was to avoid any external force or dissipation phenomenon due to the supporting tool, in order to consider the specimen's edges as free during the energy analysis. For this, the contact areas between the specimen and the frame were small and located close to the sides of the specimen.

Post-treatment of these tests was composed of several parts. Firstly, 3D computed tomography (CT) was used to scan each specimen for damaged zones. This provided a very precise understanding of the damage and led us to suggest a damage scenario for each type of reinforced structure. Apart from the observation of the damage, it was interesting to look at the energy absorption during the impact. For this, two fast cameras were used, see Fig. 2: the first fast camera recorded the motion of the ball before the impact in order to get the initial kinetic energy; the second camera recorded the motion of the specimen and the ball (in case it exited from the specimen) after the impact, in order to get the output energy of the specimen. From the recordings, the motion was tracked based on a DIC technique described in (35), and velocities were used to complete an energy balance analysis.

# Analysis of damage mechanisms in impacted designs

Reference design D1



Figure 3. Reference design D1.

**Design D1 is the reference**, see Fig. 3. It is composed of a foam block wrapped in a braided composite. The foam used is polymethacrylimide foam (density 75 kg/m<sup>3</sup>,

elastic modulus 105 MPa, shear modulus 42 MPa, compressive strength 1.7 MPa, tensile strength 2.2 MPa, shear strength 1.4 MPa). The composite consists of three layers of 2D-braiding composed of aramid 12K fibers (area weight 176 g/m<sup>2</sup>, tensile strength 2.92 GPa, tensile modulus 109 GPa) injected with HexFlow RTM6 resin (density 1110 kg/m<sup>3</sup>, tensile modulus 2.89 GPa, tensile strength 75 MPa, flexural modulus 3.3 GPa, flexural strength 132 MPa) and oriented at  $\pm 45^{\circ}$ , see Fig. 1. The braidings consisted of a four harness satin (crowfoot). The theoretical thickness of one layer was supposed to be 0.3 mm, but due to manufacturing vagaries, the actual thickness was found to be about 0.6 mm for one layer. Note that the same external shape and the same materials were used for all the designs presented.

The impact on design D1 was very clear to understand from the camera pictures and direct observation of the damage was simple. For this reason no tomography analysis was carried out on this design. The damage scenario observed was as follows: the projectile penetrated the leading edge by breaking the resin and the fibers of the front braidings and crushing the front foam under the braidings. It did not deviate and advanced in a constant direction inside the foam core by crushing and densifying the foam in a very localized zone (a very straight and narrow hole was observed with diameter almost equal to that of the ball). The ball reached the rear face and exited by breaking the resin and fibers. Once again, it did not deviate when passing through the rear braidings.

#### Effect of using a second, similar braiding



Figure 4. Design D2.

Starting from the same external shape, **design D2** consists in using a second braiding inside the foam as illustrated in Fig. 4. For this, an inner foam block with a similar shape and smaller dimensions is used. It is covered with three layers of semi-impregnated 3D-Interlock braiding in aramid 12K fibers ( $\pm 45^{\circ}$ , 1 mm thick). This assembly is surrounded with foam which is composed of two bonded parts. The whole group is covered with three layers of 3D-Interlock braiding of aramid 12K fibers and is then injected. The internal braid was intended to remain a dry net in order to keep its softness during impact. However, micro-cuts performed revealed that, unfortunately, some resin infused into the internal net.

Impact damage was checked under a micro-computed tomography non-destructive investigation device. Fig. 5 and 6 present respectively, 2D views for two orientation



Figure 5. 2D views along x-y of design D2.



**Figure 6.** 2D views along x-z of design D2.

planes, x-y and x-z, of the impacted configuration D2 sample with details of the impacted region (x is the direction of the ball's motion, see Fig. 1). Image filtering was used in order to reveal damage in the areas with less density (e.g. foam core). The views correspond to cross-sectional views in the plane containing the bullet or close to this plane. The left and right pictures present results with two contrasting parameters in order to emphasize either foam damage or braiding damage.

The scans illustrate the following damage mechanisms:

- in the leading edge, external and internal skins were perforated and the bullet created a cavity in the foam (see Fig. 5 and 6);
- when penetrating the design, the bullet slightly deviated towards the inferior part of the sample ; the bullet was finally stopped at approximately 2/3 of the specimen's total length;

- as damage in the foam was observed behind the final bullet position (see Fig. 5 and 6 on the right), the most probable scenario is that the ball compressed the foam much more during the impact with the consequence that the compressed foam broke the back face of the internal braided envelop (see Fig. 6). During testing, an out-of-plane deformation was actually observed with the high speed video camera recording the back face of the sample;
- the foam relaxation finally pushed the ball back to approximately the middle of the sample;
- the failure of the back face internal braided envelop shows that limited further absorbing capabilities are foreseen with configuration D2.

## Effect of using a transverse wall

The idea of inserting a transverse wall was done by using two foam blocks covered with a layer of braiding. To achieve the same number of layers, the assembly was wrapped with two layers of braiding before injection. The side voids were filled with an aramid dry tow injected with the same resin. The final **Design D3** is illustrated in Fig. 7.





The CT scan performed on specimen D3 revealed the following damage scenario as illustrated in Fig. 8 and 9:

- delamination and failure of the leading edge skin was caused by the impact projectile probably under skin bending loading condition;
- prior to, and during, the leading edge skin failure, skin bending and failure firstly created compaction, followed by damage in the foam core underneath the leading edge skin, leaving a conical cavity in the foam core (see Fig. 9);
- the bullet then hit the inner wall at 90°, again creating an out-of-plane displacement associated with the bending of the inner wall;
- in a similar way to the first strike, the bent skin compacted the foam core on the bent side of the inner wall, leaving a cone-shaped cavity (see Fig. 9);
- the bullet (or the densified foam pushed by the bullet) then perforated the inner wall and migrated following a tunnel-like path up to the center of the second foam block;

- a certain rebound is visible close to the sample outer face which shows a remaining cavity. The cavity was created by the foam compaction. Further damage in the foam close to the back face outer wall are also visible;
- the braidings located on the back face do not show any damage.



Figure 8. Specimen D3 shown through reconstructed 3D scans after a impact perpendicular to its leading edge.



Figure 9. Specimen D3 after impact 2D scan view after filtering.

Effect of reinforcing the front bumper in design D3



Figure 10. Design D3R.

**Design D3R** is similar to the D3 design in which the front bumper is reinforced by manual continuous sewing, as illustrated in Fig. 10. The front foam block was predrilled (diameter 1.5 mm) with several holes. The sewing used doubled tow of carbon 6K for each hole with an increment of  $7.5^{\circ}$  between the direction of each hole. The pitch between two planes for sewing was 6 mm. The whole ensemble was injected with resin.



Figure 11. Design D3R shown through reconstructed 3D CT scans after impact.

The CT scan performed on design D3R after impact revealed the following damage scenario as illustrated in Fig. 11 to 13:

- failure of the leading edge skin was caused by the impact projectile under skin bending loading condition;
- prior to, and during, the leading edge skin failure, skin bending and failure firstly created foam compaction, followed by damage to the reinforced foam core underneath the leading edge skin;



**Figure 12.** Design D3R after an impact perpendicular to its leading edge : *(left)* cross-sectional view of 3D CT scan view after filtering, *(right)* 2D CT scan view showing a multiple failure of a stitch by effect of stitch bending along the ball's path.



**Figure 13.** Design D3R 2D and 3D CT scan views showing several stitch head failures and the foam tunnel left by the ball's path.

• the foam reinforcements were totally broken (see Fig. 12) along the axis of the bullet's trajectory. Also the stitches located on the lateral side from the impact axis were forced to bend while the ball was passing through the sample;

- the lateral stitch bending generated multiple fragmentations of some of the stitches (see Fig. 12);
- the bullet then hit the inner wall at 90°, again creating an out-of-plane displacement associated to a limited bending of the inner wall due to the joint through the stitches to the external envelope;
- the bent skin compacted the foam core on the bent side of the inner wall, leaving a cone-shaped cavity;
- the bullet then perforated the inner wall and migrated following a tunnel-like path up to the back face of the second block;
- the outer wall located on the back face partially failed. The bullet also remained in the sample. The upper bumper stitch attachment on the inner wall probably limited the inner wall bending deflection before its failure on the lower bumper foam core.

From the tomography analysis, one may conclude that the reinforced design D3R did not bring a significant improvement compared to design D3. However, an energy analysis, which will be given in a coming section, showed a slight improvement in the energy absorption that can be explained by additional dissipation capabilities offered by the possibility of stitch breakage.

### Effect of using a longitudinal wall



Figure 14. Design D4.

Instead of using discrete reinforcements to improve the initial stiffness, **Design D4** was made of two oblong foam blocks covered with one layer of braiding of carbon 12K fibers, see Fig. 14. The shape was completed by a front foam block and the rear void was filled with an aramid dry tow. To ensure that the front edge included three layers of braiding, as in other designs, a core cap fabric was used to cover the front foam. The assembly was surrounded by an external braiding and was injected with resin.

As for other sample configurations, micro-computed tomography non-destructive investigation technology was used in order to identify damage scenarios after impact. Fig. 15 presents 3D views of the impacted configuration D4 sample. A cross-sectional view by means of a 2D plan was performed in the software in order to show half of specimen D4 with details of the impacted region. Fig. 15 right view corresponds to a cross-sectional view without external braidings for better visibility of the damage created in the inner wall.

Fig. 16 and 17 respectively present 2D views for plane orientation x-y and x-z of the impacted configuration D4 sample with details of the impacted region. Image filtering was used in order to reveal the damage in the areas with lower density (e.g. foam core). The views correspond to cross-sectional views in the symmetry plane of the bullet, or close to this plane.

The scans presented in Fig. 15 to 17 illustrate the following damage mechanisms:

- the leading edge skin was perforated and the bullet created a cavity in the front foam;
- the strike of the bullet on the internal braided envelope damaged it;
- consequently the dividing wall was broken and delaminated;
- the bullet was stopped by the back face of the external braided envelope of one of the sub-structures;
- the ball final location indicated that very limited further absorbing capabilities could be foreseen with sample D4 as in Fig. 17 (right picture), it seems that the back face external braided envelop slightly suffered from the shock wave.



Figure 15. Views of two different angles of 3D micro-tomography of sample D4's global impact damages.

# Effect of reinforcing rear bumpers in design D4

Starting from the idea of strengthening oblong bumpers from design D4, **Design D4R** oblong chambers and their carbon braidings were reinforced by stitching, see Fig. 18. The stitches were made of carbon 6K fibers oriented at  $\pm 60^{\circ}$  from the symmetry plane (that is  $+60^{\circ}$  in a given plane and  $-60^{\circ}$  in the next plane, the pitch between two planes was 6 mm). The distance between two stitches in a given plane was 6 mm. Note that the two oblong chambers and their carbon braidings coverage were separately stitched before being assembled. To ensure that the front edge included three layers of composite as in other designs, a core cap fabric was put on the top of the front foam. The assembly was surrounded by an external braiding and was injected with resin.



Figure 16. Four sets of 2D views along x-y of design D4.



Figure 17. 2D views along x-z of design D4.

As for other sample configurations, micro-computed tomography non-destructive investigation technology was used. Fig. 19 and 20 present 3D views of the impacted configuration D4R sample. A 2D cross-sectional view was performed in the specimen in order to show half of the D4R specimen with details of the impacted region. The views correspond to a cross-sectional views in the symmetry plane of the bullet, or close to this plane.

Fig. 21 and 22 present 2D views of the impacted configuration D4R sample with details of the impacted region. Image filtering was used in order to reveal damage to the areas with lower densities (eg. foam core). The views correspond to a cross-sectional view



Figure 18. Design D4R.



Figure 19. Partial view of 3D micro-tomography of sample design D4R.



Figure 20. Partial view of 3D micro-tomography of sample design D4R.

in the symmetry plane of the bullet, or close to this plane. The bright dots in Fig. 22 correspond to the stitches.

The scans presented from Fig. 19 to 22 illustrate the following damage mechanisms:



Figure 21. 2D views along *x*-*y* of configuration D4R.



**Figure 22.** 2D view in the impacted area along x-z(*left*) and y-z (*right*) of design D4R after impact.

- the leading edge skin was perforated and the bullet created a cavity in the foam (see Fig. 21);
- the strike of the bullet on the internal braided envelope damaged it (see Fig. 19 to 22);
- consequently the dividing wall was broken and delaminated (see Fig. 20 and 21);
- the bullet was stopped by the stitches stabilized in the foam core in one of the substructures (see Fig. 19, 20 and 21). It is interesting to note that the projectile was stopped much sooner compared to other designs;
- the stitches contributed to the energy absorption as several stitch failures were observed (see Fig. 21);
- beneath the ball final location, a long part of reinforced foam did not seem to have suffered any damage. None of the stitches seem to have moved from the initial position (see Fig. 20 and 22-*left*). Therefore, it appears that configuration D4R has further absorbing capability potential;
- roughly half the height of the configuration D4R contributed to the energy absorption capabilities as the bullet was stopped at mid-height with no damage observed beneath.

# Test of D4R to a more energetic impact

In this section, a test was carried out with a steel ball of diameter 30 mm. For the sake of brevity, this test was only carried out on design D4R. The idea was to validate the damage scenario identified in the previous section with a different impactor. Design D4R was chosen because a change in the impactor size or the impact energy could be expected to change the damage scenario, due to the complex damage mechanisms involved. The impact speed was kept at 110 m/s, resulting in an impact energy of 667 J, which was 3.375 more energetic than with the 20 mm ball.



Figure 23. Partial view of 3D micro-tomography of design D4R after impact at 110 m/s by a 30 mm steel ball.

Fig. 23 presents a 3D view of design D4R impacted by the 30 mm steel ball. 2D crosssectional views were performed in the specimen in order to reveal details of the impacted region only. The figure reveals a clear hole in the outer braiding due to the penetration of the ball. The breakage of the outer braid was probably caused by deformation and damage of the foam beneath the braid, followed by resin damage and breakage of the fibers under tension.



**Figure 24.** 2D view in the impacted area along x-z(*left*) and x-y (*right*) of design D4R after 30 mm impact.

The inner walls, and in particular the mean wall, were completely destroyed. Stitch debris were also visible. Fig. 24 gives more details about the damage of the inner walls. The left figure shows that the path followed by the impactor up to the rear face was totally damaged on the exact diameter of the ball, suggesting that while progressing in the two oblong chambers, the ball totally destroyed the mean wall. Away from this path, the inner braidings look clearly undamaged. The right figure shows that the walls experienced delamination.





At the same time, numerous stitches were damaged. Fig. 25 gives some information about the damage mechanisms of the stitches. As the inner wall was destroyed, one may assume that the attachment to the mean wall was lost, and that the stitches folded in the direction of the external braid to allow the progress of the ball. Most of the stitches

remained attached to the outer braids. Fig. 25 also confirms that a single stitch often broke at several points. This is interesting information because this multiple failure mode dissipates more energy than a single one. Additionally, one can note that the stitch attachments remained totally undamaged. Stitch breakage essentially occurred far from the bonding to the braidings. As the impact energy was greater than for 20 mm projectile, the ball finally rebounded on the back face due to the elastic energy release of the foam and the external braid. When the ball came back, the stitches returned to their initial location due to the remaining attachment to the outer braid. These results confirm the damage mechanisms identified with the 20 mm impactor.

### Energy absorption and comparisons with crushing tests

Fast camera pictures were used to measure initial and post-impact velocities. From the velocities, kinetic energy and, as a consequence, the energy absorption ratio and specific energy absorption (SEA) could be computed. The energy absorption ratio is the ratio of the energy absorbed during the impact relative to the initial kinetic energy. SEA is the ratio of the absorbed energy to the mass of the specimen. Note that SEA values were normalized with respect to that of reference design D1. The results are reported in Table 1. The energy absorption ratio are very similar and consequently, they do not help distinguish the designs that have good potential to face impact events. However, one must pay the price for this improved potential, as the SEA indicates. Indeed, design D1, although less absorbant, shows the best SEA since it is the lighter design. On the other hand, design D4R, has the smallest SEA value, even though the other values are relatively close. Additionally and to be more precise, one can say that in D4R, the entire length of the design was not required to stop the ball. This means that the design dimensions, and in particular the reinforcement dimensions, and as a consequence its weight, could be optimized for a given impact energy.

In order to compare the designs, and taking into consideration the net mass used, it was convenient to give an idea of further potential for each design which would then confirm the possibility for the design to absorb a more energetic projectile. With this in mind, a "ball penetration indicator" was defined by capital letters as follows: "A" denotes a test in which the ball was stopped before reaching the middle of the sample (good potential), "B": the ball was stopped before reaching the rear braidings (medium potential), "C": the ball reached the rear braidings and the foam was crushed along the whole length (low potential), "D": the ball damaged the rear braidings but stayed trapped (almost no potential left), "E": the ball was not stopped during the test and exited the specimen (the specimen could not retain the impact at 110 m/s). The "ball penetration indicator" clearly highlights that designs D2, D3 and, above all, D4R are able to support more energetic loadings; all other designs have, at most, only the rear braiding resistance left.

Finally, Table 1 also shows the results from low velocity crushing tests reported in a companion paper (34) in order to give a cross-comparison based on two different types of impact loadings: while canon ball impact tests challenged the structures in a very localized manner, dynamic crushing tests showed their capabilities under global dynamic loading case. The results summarized in Fig. 26 clearly confirm that design D4R has

good potential as an impact absorber for both types of loadings, due to its high stitching density. However the additional mass of the stitches obviously led to degraded SEA value for both cases. In the same time, good potential and low SEA value should be interpreted as the possibility to optimize the stitch density, consequently the mass and the SEA, for a given impact energy to be absorbed.

Design		D1	D2	D3	D3R	D4	D4R
Medium velocity tests							
Energy absorption ratio Specific energy absorption Ball penetration indicator	[%] (normalized)	80.1% 1.00 E	89.3% 0.75 C	84.6% 0.79 C	86.0% 0.84 D	84.7% 0.72 D	87.8% 0.70 A
	Low velocity cru	shing test	s (results f	from ( <mark>34</mark> ))			
Energy absorption ratio Specific energy absorption Usable concept left	[%] (normalized) $[m]$	94% 1 0.024	84% 0.49 0.039	94% 0.68 0.048	91% 0.61 0.047	32% 0.22 0.016	84% 0.46 0.062

<b>Table 1.</b> Energy absorption ratio and comparisons with crushing test results from
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**Figure 26.** Comparisons between medium velocity impact tests and crushing tests. *(left):* further absorption indicator and *(right):* SEA comparisons.

# Conclusion

Several foam core composite structures reinforced by several techniques were submitted to medium velocity impact loading consisting of a 20 mm ball shot at 110 m/s on the leading edge. Computed tomography analysis clearly revealed the damage mechanisms for each design. The conclusions can be summarized as follows:

- in the reference scenario, the projectile penetrated the leading edge by breaking the front braidings (compacting the foam underneath and bending the braidings) and it advanced by crushing the foam core. It reached the rear braidings, broke them and exited the sample;
- all the reinforced designs were able to trap the projectile inside. The reinforcements
  were capable of strengthening the foam core as well as offering different modes of
  energy consumption such as inner braidings breakage/delamination, stitch multibreakage by bending, tension, compression or shear loading;
- only design D4R (involving foam stitching) showed potential for absorbing more energetic impact loadings. Indeed, half of the design D4R was clearly left undamaged. For this design, more energetic impact leads to similar damage mechanisms.

Energy analysis showed that in return for improving the impact resistance by stitching, the design must be optimized in weight and reserve potential to reach an SEA competitive to other reinforced designs. Finally, comparisons with results from crushing tests were in agreement with those of medium velocity tests and reinforced the conclusions about the potential of the designs under a different load case. This paper also suggests that an optimization of the reinforced structure should be carried out for a given range of impact energy to be absorbed. For this, reinforcement parameters, such as stitch angle, stitch density, stitch materials and diameter must be investigated. Numerical models are under development to help in this design stage. One must also keep in mind that reinforcements bring additional manufacturing costs and time, which are also to be considered.

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