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Vision-Based Microforce Measurement with a Large Range-to-Resolution Ratio using a Twin-Scale Pattern

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Abstract—Force sensors are often required in order to work at the micro-scale but existing ones rarely meet all expectations, particularly in terms of resolution, range, accuracy or integration potential. This paper presents a novel micro-force measurement method by vision, based on a twin-scale pattern fixed on a compliant structure. This approach enabled subpixelic measurement of position by the use of a micro-machined pattern based on Vernier principle. This method also presents flexibility, insensitivity to electronic noise, fast operating time and ease of calibration.

The major contribution consists in the large range-to-resolution ratio of the measurement system. With an experimental range of 50 mN and a resolution below 50 nN, a range-to-resolution ratio of 10^6 is obtained. A repeatability under 7.8 μN and a trueness under 15 μN have been experimentally measured. Finally, the method can be applied to other specifications and applications in terms of range.

Index Terms—Micro-force sensor, vision, twin-scale, range-to-resolution ratio

I. INTRODUCTION

For some decades many technologies tend toward miniaturization. A lot of improvements passed through the micro-scale issues in fields such as energy, biology, medicine, instrumentation and robotics. Societal and technological needs strongly push systems to extremely high integration levels (strong desire to have more complex systems with increasing number of functionalities), important miniaturization, smarter and more accurate systems.

Clean room microfabrication technologies and the integration of active materials are notably widespread technologies as well as the use of compliant structures [1]. Despite their strong interest and use (high resolution motion generation, high integration level, accurate fabrication, etc.) several drawbacks still have to be overcome to devise products with expected performances: (i) the behavior of active materials is highly non linear and time varying notably because of strong influence of environmental conditions; (ii) the use of compliant structures amplifies displacements but also non linearities. The integration of sensors to provide closed loop control appears as a relevant solution but several requirements have to be met simultaneously: direct measurement, small size, high bandwidth, high signal to noise ratio, multi degrees-of-freedom measurements, etc.

Force measurement is of prime importance to succeed in performing complex tasks such as in microsurgery, biology (characterization of cells, in vitro fertilization) or micro-assembly ([2], [3], [4]). Intensive researches have been conducted but designed force sensors rarely meet all expectations in terms of range, resolution, accuracy and bandwidth; as well as in terms of ease of integration, size, and interactions [5]. This limitation is mainly due to the fact that force measurement cannot be done directly. Either the displacement of a mobile element is measured (in a compliant architecture for example) or a level of stress. Force estimation is then based on a calibration step and on a force-displacement or force-stress (or its derivatives) model. Variations of system behavior and poor signal-to-noise ratio being key features at the microscale, it is then highly important to have the best initial signal quality (displacement or stress). To tackle this key lock many works propose smart, optimized or improved designs and combine it with one measurement principle among the most used: capacitive ([6], [7], [8]), piezoresistive ([9], [10], [11]), strain gauges ([12], [13]), magnetic ([14], [15]), or whether optical ([16], [10], [17]). Several research teams notably proposed compliant systems with improved shapes and reduced stiffness (notably to improve resolution) ([18], [19]). This choice notably reduces the measurement range of the whole system which may be a great limitation. Most works investigate system designs and enable range-to-resolution ratios comprised between 10^3 and 10^5 (see Fig. 1 further).

An alternative approach is to investigate displacements measurement through vision because many progresses have recently been done to provide smaller sensors, with higher image qualities, higher frequencies and because of the extensive use of cameras for microscale applications (cameras are already integrated in many platforms or systems). Among the visual methods some are based on deformation observation of the shape, as Greminger et al. in [20], Wason in [4] or Wang in [21]. Others work on a correlation principle, as Anis et al. in [22] or Chang et al. in [23]. Finally, some works use a special vision method to deal with specific applications, as Karimirad et al. [24] that observe a cell deformation during a simple contact. These works state the interest of measuring

forces through vision at the microscale but also show related limitations notably because of the severe trade-off between field of observation and achieved resolution but also because of algorithms generally used. Typical range-to-resolution ratios are about 10^3 .

To overcome such drawback, we chose in this paper a different approach in the sense that it intends to improve the visual measurement principle by itself. Some visual methods based on pseudo-periodic patterns have been proposed by different authors [25], [26], [27], [28]. These methods allow to extend the measuring range beyond the field-of-view while measuring the pattern position with sub-nanometer resolution. Recently, we proposed a concept based on a twin-scale periodic grid with two different pitch sizes that has also a large range-to-resolution ratio and fast operating time [29], [30].

In this paper, we study and demonstrate the potentiality of the combination of this visual method with a compliant structure to achieve force measurements. The principle is based on the observation of the twin-scale pattern fixed on a compliant structure. This novel visual force sensing approach aims at a very high range-to-resolution ratio.

This paper is organized as follows. The measurement principle is presented in section II, the position measurement method and its extension to force measurement are notably introduced. Section III presents the experimental setup. Section IV presents the experimental results and discusses the performances achieved in two steps: firstly the quantification of the method repeatability, resolution and trueness; and secondly the demonstration of the force measurement range. Section V allows the discussion of these results. Section VI concludes the paper.

II. PRINCIPLE AND POTENTIAL CAPABILITIES OF THE METHOD

A. Estimation of force through vision

Forces are extensive physical properties which can not be directly measured. A common way to measure forces is to measure the deformation of a test specimen. For instance, the Hooke's law states that the force F exerted on an ideal spring is proportional to its deformation, i.e.,

$$F = K \cdot \delta \quad (1)$$

with K the stiffness of the spring and δ the relative displacement from its relaxed position.

Measuring micro-scale forces often involves to use a very low stiffness. For example, the typical stiffness of an atomic force microscope is about 0.1 N.m^{-1} [31]. Alternative solutions rely on higher stiffness structures (few hundreds of N.m^{-1}) combined with high-sensitive displacement sensors (see an example in [32]).

Unfortunately, obtaining a high resolution in displacement measurement is generally detrimental to the allowed range of measurement. Common values of the range-to-resolution ratio of force sensors are about 10^3 , sometimes 10^4 and rarely 10^5 (as illustrated in Fig. 1).¹

¹The metrologic terms used in this paper are those defined by the Joint Committee for Guides in Metrology [33].

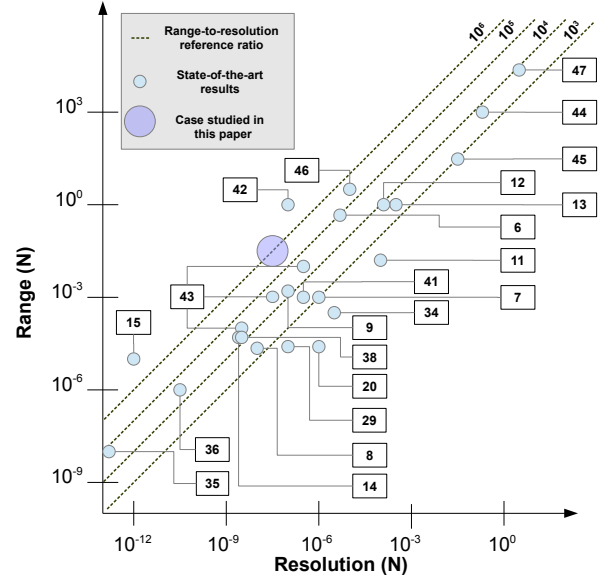


Fig. 1. Expected range-to-resolution performance (dashed line) with regards to state-of-the-art works and commercial force sensors. Numbers correspond to the references of the articles and sensors.

B. Twin-scale vision based measurement

Recently, we proposed a novel vision-based method that allows displacement measurements with improved range-to-resolution ratios. This method is an extension of phase processing methods of periodic image features [48]. Its working principle and the evaluation of these performances are detailed in [30] as well as discussions about the influence of the various parameters. This section summarizes the core content of this article.

The principle of this phase measurement method relies on the phase-to-displacement relationship that is a scalar product in the frequency domain:

$$\mathcal{F}(f(x - \delta)) = e^{-2\pi i \delta \xi} \cdot \mathcal{F}(f(x)) \quad (2)$$

where \mathcal{F} stands for the Fourier transformation. $f(x)$ is the space function considered (here the intensity of the pixels), x the spatial coordinate, δ the spatial displacement and ξ the reciprocal variable of x (or transform variable). A target displacement δ induces only a phase shift Φ in the frequency domain with $\Phi = 2\pi\delta\xi$.

Thanks to this linear relationship, we can retrieve object displacements with a very high precision through phase processing of the recorded images. In order to get efficient computation times and high signal-to-noise ratios, a high-contrast periodic-pattern can be placed on the target of interest (see Fig. 2). Then, instead of performing a complete Discrete Fourier Transform (DFT), we can compute a single-frequency spectral component; that corresponds to the spatial frequency of the periodic pattern used. In the present case, a complex analysis vector $Z(k)$ defined by a Gaussian window and a periodic signal at the period L of the stripe set is used:

$$Z(k) = e^{-\left(\frac{k-N/2}{N/4.5}\right)^2} \cdot e^{-\left(\frac{2i\pi(k-N/2)}{L}\right)} \quad (3)$$

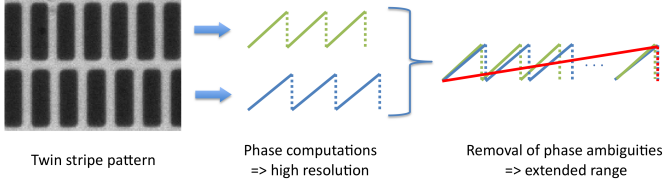


Fig. 2. Working principle of the twin-scale visual measurement. See [30].

where k is the pixel index and N is the image width in pixels.

The expected phase Φ is then given by the argument of the dot product between vector Z and the vector of pixel intensities. The target displacement can then be determined by:

$$\delta = \frac{\Phi \cdot L}{2\pi} + nL \quad (4)$$

where n is an unknown integer standing for an entire number of stripe periods. Indeed due to the stripe periodicity, the displacement value is obtained modulo L since different positions distant from an entire number of periods produce indistinguishable images.

Our method to overcome this limitation consists in the use of a twin-scale. The use of a second stripe set with a slightly different period provides complementary and independent phase data that can be used for the removal of phase ambiguities. The principle is illustrated in Fig. 2. Thanks to the progressive mismatch between the two stripe sets, phase ambiguities can be removed and the unambiguous range switches from a single period to a new value Λ given by:

$$\Lambda = \frac{L_1 \cdot L_2}{|L_1 - L_2|} \quad (5)$$

where L_1 and L_2 are the twin-stripe set periods.

The last step consists in using synthetic data to determine the correct $2m\pi$ constant to apply to the phase shift observed for either stripe set:

$$\delta = \frac{\Phi_1 \cdot L_1}{2\pi} + m L_1 + p \Lambda \quad (6)$$

in which Φ_1 is the phase shift for the smallest stripe set, m the number of periods derived from the synthetic phase and p an unknown number of periods Λ that represents the new ambiguity range. In practice, if the range of displacement is below Λ , p does not change and can be set arbitrary to 0. It is important to mention that the actual periods L_1 and L_2 of the patterns are accurately known, so the pattern features provide a known size reference in images allowing a direct conversion from pixels to meters. The method is thus self-calibrating concerning the position measurement and does not depend on experimental parameters such as magnification or field of view. Moreover the process of phase measurement induces a large spectral filtering, that makes it possible to get away from problems of dynamic range and high-frequency spatial noises. As described in [49], the image processing technique presents also a low sensitivity to the recorded image sharpness as long as the spatial frequencies associated to the twin-scale patterns

TABLE I
PERFORMANCES OF TWIN-SCALE VISUAL MEASUREMENT WITH A 8 μm PERIODIC PATTERN AND 640X480 8-BITS CAMERA (FROM [30]).

Property	Value
Travel range	168 μm
Resolution	55 μm
Experimental repeatability (3- σ)	5 nm
Bandwidth	>1500 Hz

are imaged with a sufficient contrast. Therefore the resolution of the microscope is not a critical parameter.

Table I gathers the performances of the method using a micro-machined pattern with 8 μm and 8.4 μm twin scales periods, corresponding to an unambiguous range of 168 μm . The range-to-resolution ratio of the displacement measurement method is $3 \cdot 10^6$ with a 8-bits camera [30].

Applying this method to measure the displacement of an elastic structure appears as very promising to design force sensors with large range-to-resolution ratios.

C. Potentiality in force measurement

A twin-scale pattern is fixed onto a compliant structure whose stiffness is known. The idea is to estimate the force applied to it by measuring its displacement. For example, using the 8/8.4 μm pattern attached to a compliant structure with a stiffness of 1000 $\text{N}\cdot\text{m}^{-1}$ may lead to a force sensor able to measure up to 168 mN with a resolution of 55 nN in theory (performances deduced from Table I). Thus, the potential range-to-resolution ratio of such a sensor could be 3.10^6 .

Comparing these expected performances with state-of-the-art works and commercial force sensors, the proposed method could constitute an improvement of at least one order of magnitude in term of range-to-resolution ratio. Fig. 1 allows for a comparison between the presented method and different sensor principles (optical, capacitive, piezo...). The placement criteria are range and resolution. The larger circle represents the experimental performances of the method, investigated latter with the use of a basic 8-bits camera. Other performances (for example repeatability and signal-to-noise ratio) depend on experimental conditions and are not processed in this graph.

This figure illustrates that we can expect better than most of the state-of-the-art sensor performances. Moreover according to the desired range and resolution in the force domain, a couple stiffness/size can be chosen to cover the desired specifications. Thus the sensor could be adapted to many scales, from the micro-scale to the macro-scale as illustrated in Fig. 3. It is important to notice that such scale-adjustments don't affect the range-to-resolution ratio. Nevertheless, the range-to-resolution ratio could be increased by the use of a more efficient camera.

III. EXPERIMENTAL SETUP AND STIFFNESS DETERMINATION

To validate experimentally this potential, we use a setup built around a compliant structure on which a pattern is fixed.

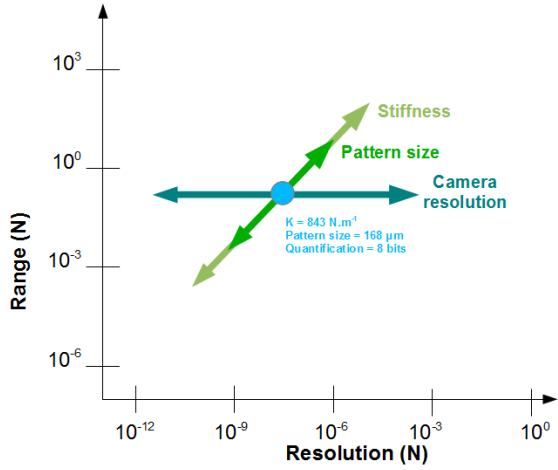


Fig. 3. Graphical representation of the potential evolutions in term of range-to-resolution of the presented method (we assume that environmental disturbances are neglected).

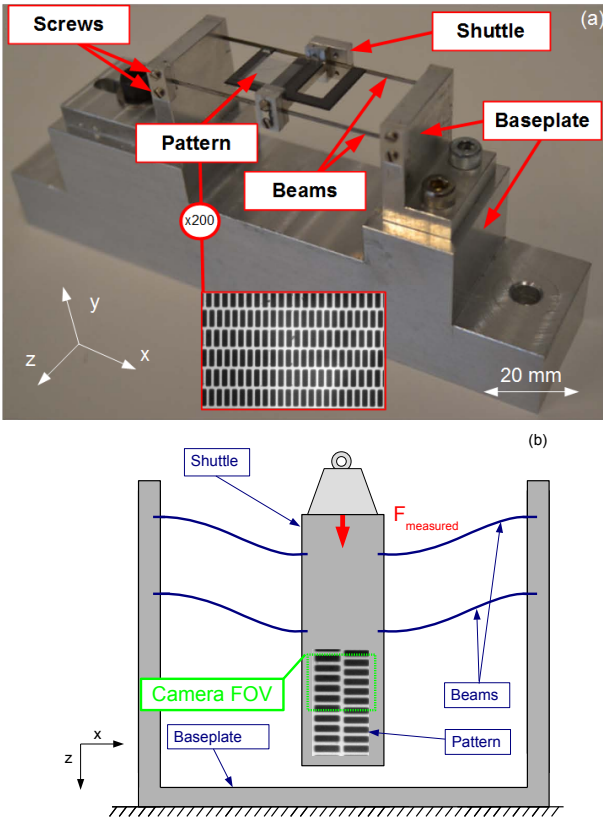


Fig. 4. The compliant structure on which a pattern is attached. The link between the shuttle and the baseplate is four beams allowing the translation of the shuttle. The camera's field of view (FOV) is focused on the pattern. (a) Photo of the compliant setup. The insert shows a typical recorded image. (b) Schema of its operating principle.

In addition to that, a reference sensor that allows the validation of our measurements is used.

A. Force measurement device

The compliant structure is presented in Fig. 4. It is composed of a mobile shuttle connected to its baseplate by four

thin compliant beams, restricting the motions along a single axis (z). The principle of this setup is extracted from [50]. From the knowledge of the stiffness between the shuttle and the baseplate, we derive the forces applied to the shuttle from the measurement of its displacements. On this compliant structure, we attached a twin-scale pattern on the shuttle in order to measure its position and displacements along the z axis.

The beam stiffness notably depends on the screws clamping the beams on the baseplate (see Fig. 4(a)). The screws allow adding a preload in the beams. This preload increases the stress in the beam and thus the stiffness of the compliant structure. Experiments show that the stiffness can be tuned from 200 to 1000 N.m^{-1} . This flexibility allows to adapt the measurement range to the application. It was used in the experiments to be as close as possible to the reference sensor range.

The visual setup is composed of a FireWire camera (Allied Vision Technology Pike F-032B, 8 bits, 640×480 pixels) equipped with a $10\times$ microscope lens. The computer used to process images is a common computer. The different elements of the setup are fixed on an antivibration table, a paramount element to minimize the impact of mechanical noises.

B. Stiffness measurement procedure

As mentioned above, the stiffness of the structure has to be known accurately to derive forces from measured displacements. At this stage, the use of a calibration method is mandatory.

The compliant structure stiffness K is measured from the processing of the free oscillations of the shuttle in response to a starting pulse:

$$w_0 = \sqrt{\frac{K}{m}} \quad (7)$$

where m is the mass of the mobile part and w_0 its natural frequency. Fig. 5 gives an example of free oscillations as reconstructed by the displacement measurement method used.

The stiffness can thus be derived from the free oscillation frequency. The unknown mass m is determined by repeating this recording after the introduction of an additional mass m_δ ; *i.e.* once unladen (mass m) and once with an additional calibrated mass m_δ laid on the shuttle:

$$m = \frac{m_\delta \cdot w_{0,m+m_\delta}^2}{w_{0,m}^2 - w_{0,m+m_\delta}^2} \quad (8)$$

Beforehand, the mass m_δ is measured accurately. Then the mass m of the mobile part can be deduced, and therefore the stiffness K . In this way we dispose of a fast and easy stiffness calibration procedure.

By way of an example, an additional mass $m_\delta = 1.107$ g was used to do the calibration. This value was measured with a calibrated balance (resolution : 10^{-4} g). We observed $w_{0,m} = 318.9 \text{ s}^{-1}$ and $w_{0,m+m_\delta} = 268.2 \text{ s}^{-1}$. Finally, the mass of the shuttle is identified as $m = 2.673$ g, which allows to calculate the stiffness $K = 271.9 \text{ N.m}^{-1}$. From that moment it's faster to re-calibrate the system because the mass m is known, so we only need to measure $w_{0,m}$ to know the stiffness.

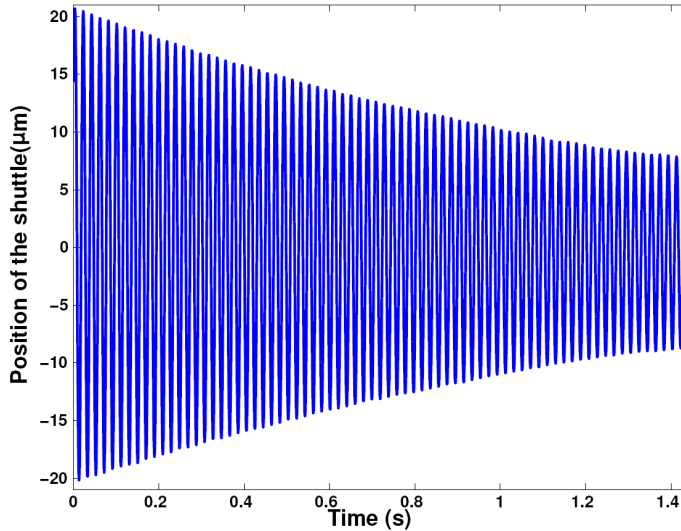


Fig. 5. Free shuttle oscillations in response to a starting pulse as measured by vision at 1389.5 fps.

The accuracy of the stiffness evaluation is dependent on the quality of additional mass characterization and on frequency measurement. However the following experiments illustrate the quality of the calibration method.

IV. EXPERIMENTAL PERFORMANCE EVALUATION

The performances of the proposed force sensing system were experimentally quantified in two steps. The first one, presented in section “Microscale experiment”, evaluates repeatability, resolution and trueness. The second one focuses on sensing range, and is developed in “Large range experiment”.

A. Microscale experiment

In order to determine the precision of the method (with repeatability, resolution and trueness), we realized measurements using a FemtoTools microforce sensing probe (model FT-S270) as reference sensor (see Fig. 6). This capacitive sensor has a force range of 2 mN and a resolution of 0.4 μN (datasheet). Moreover we corrected its non-linearity with its calibration data (full curve provided by the manufacturer).

The sensor is attached to a nanopositioning XY stage (Piezosystem Jena PXY 200 D12) with a 200 μm displacement range and integrated capacitive sensors with a 0.4 nm resolution. To reduce environment noises, the setup was located in a room that is controlled in temperature ($\Delta T < 1^\circ\text{C}$), in humidity, and mechanically isolated from the rest of the building to minimize mechanical disturbances.

The experiment consists in the observation of 50 sinusoidal cycles performed by the nanopositioning stage along the z axis. The stage pushes on the shuttle via the reference sensor. 150 measurements (20 s per cycle) are realized during each period of the sinusoid. The stiffness of the compliant structure during this experiment is set as $240 \text{ N}\cdot\text{m}^{-1}$: a low stiffness allows to measure little forces for a corresponding large displacement.

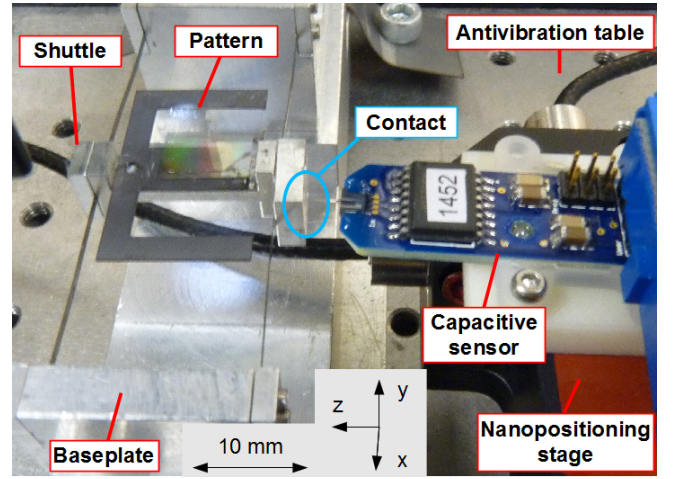


Fig. 6. Experimental setup used for the evaluation of the repeatability. A motorized nanopositioning stage is used to move the reference sensor along z relatively to the mobile shuttle.

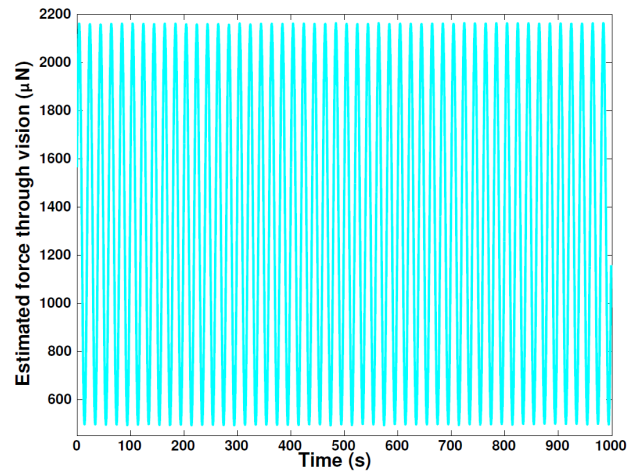


Fig. 7. Experimental force measurement vs time for sinusoidal motions of the nanopositioning stage.

Fig. 7 illustrates the experimental data measured by the visual method during the 50 cycles. Fig. 8 presents the average measurement of the sinusoid, each point corresponding to the average of 50 points of the Fig. 7.

We can extract from this experiment the repeatability of the force estimation, defined as three times the standard deviation of the measured force. The repeatability is presented in the zoom of Fig. 8. We obtain here a 3σ repeatability of 7.8 μN .

However this result notably includes the unknown repeatability of the nanopositioning stage. As an estimation, a nano-actuator has a repeatability about 100 times bigger than its resolution (see [51]). The resolution of this stage is 0.4 nm, so its repeatability could be evaluated to 40 nm. Multiplying by the stiffness of our device ($240 \text{ N}\cdot\text{m}^{-1}$), an estimation of the repeatability of the positioning stage in terms of applied force can be 9.6 μN . This estimated repeatability and the measured repeatability are of the same order of magnitude. We can thus conclude that the repeatability of our visual method is likely intrinsically better than the value observed experimentally; i.e.

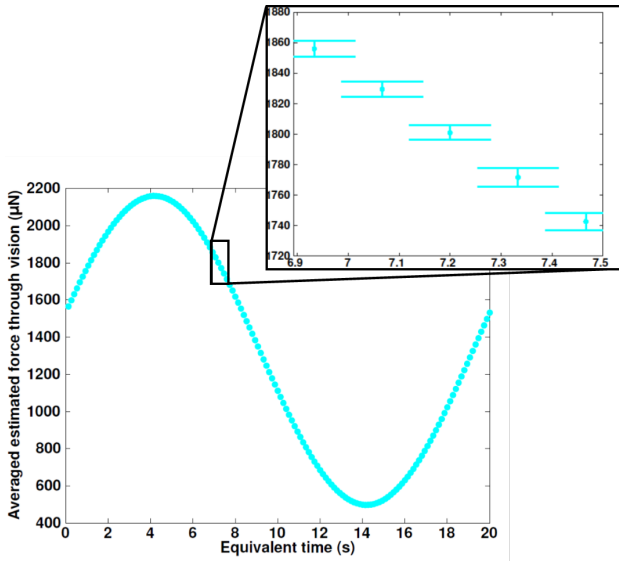


Fig. 8. Average of the force measurement during the sinusoidal displacement. The zoom presents the repeatability derived from the standard deviation of these values (3σ).

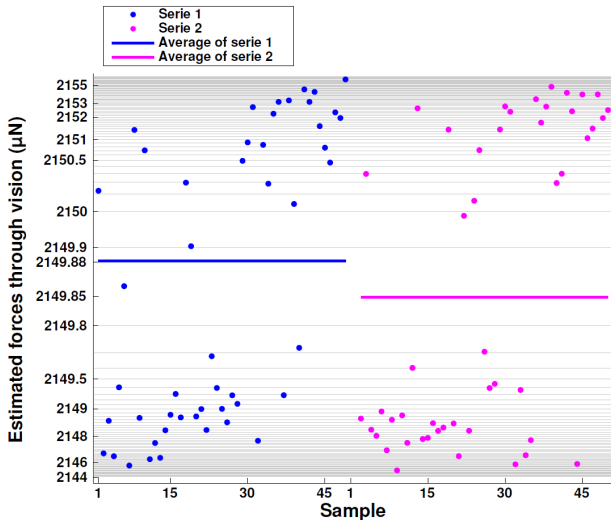


Fig. 9. Two series of measured forces, acquired respectively to an equivalent time of 3.74 s and 4.66 s (see Fig. 8). The difference between the average of the two series is the resolution, equal to 32.6 nN. The low point density in the central part is due to the non-linear scale allowing the emphasizing of the difference between the averaged values obtained.

7.8 μN .

The second characteristic that we can extract from this experiment is the resolution. The resolution is defined as “smallest change in a quantity being measured that causes a perceptible change in the corresponding indication”.

We calculated the averaged sinusoid (see Fig. 8), that allows to filter out an important part of the noise. The smallest difference between two points of this curve is of 32.6 nN (see Fig. 9), corresponding to a small but actual displacement. This little force difference is the experimental resolution, near to the theoretical resolution (given in [30]) here equal to 13.2 nN for a stiffness equal to 240 N.m^{-1} .

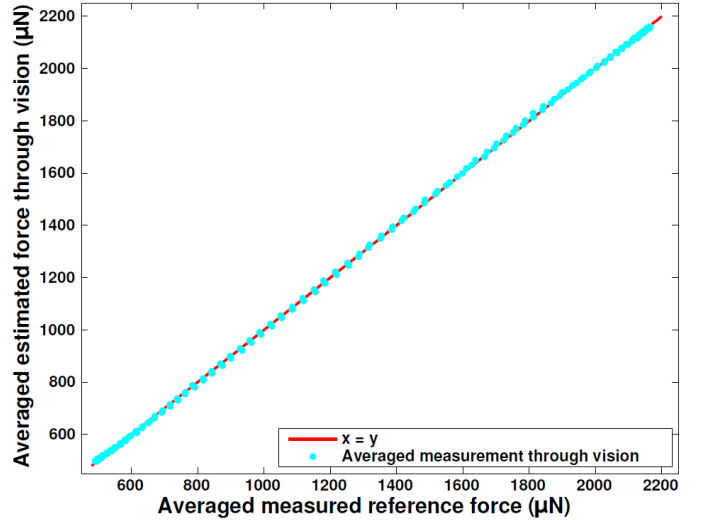


Fig. 10. Averaged force measurements versus averaged reference measurements. Each point corresponds to the average of 50 measures (for both visual and reference sensors). Forces applied were chosen in the working range of the reference sensor, i.e. $[0; 2.2 \text{ mN}]$.

In a last step, we compared the visual and reference sensor measurements to evaluate the method’s trueness, defined as the “closeness of agreement between the average of an infinite number of replicate measured quantity values and a reference quantity value” [33]. We consider a large number of measures to evaluate it. For that, we compare the averaged value of the visual force measurements with the average value provided by the reference sensor. The resulting data are plotted in Fig. 10 that shows good agreement and linearity between the visual and reference measurement tools. The error between the two curves is always inferior to 15 μN . This point allows to say that the trueness between the visual measurement and the reference sensor is 15 μN . Moreover this good agreement between the two curves validates the calibration approach (cf. sect. III-B) that doesn’t require any reference force sensor.

These experimental results demonstrate that the proposed visual method allows reliable force measurements with a repeatability under 7.8 μN , a resolution below 32.6 nN and a trueness below 15 μN .

B. Large range experiment

A second experimental setup (see Fig. 11) has been used to perform large range experiments, to quantify the range of the sensor. The vision-based force measurements are compared to those provided by a reference sensor: a calibrated balance (Mettler Toledo ML3002: repeatability $\sigma = 0.01 \text{ g} \simeq 100 \mu\text{N}$; linearity $0.02 \text{ g} \simeq 200 \mu\text{N}$). The compliant structure is attached on a micro-positioning table above this balance and is moved along z axis. In this way the shuttle applies a force on the balance when moved vertically. At equilibrium, the force applied to the balance is equal to that supported by the shuttle. In this experiment, the stiffness was evaluated equal to 843 N.m^{-1} .

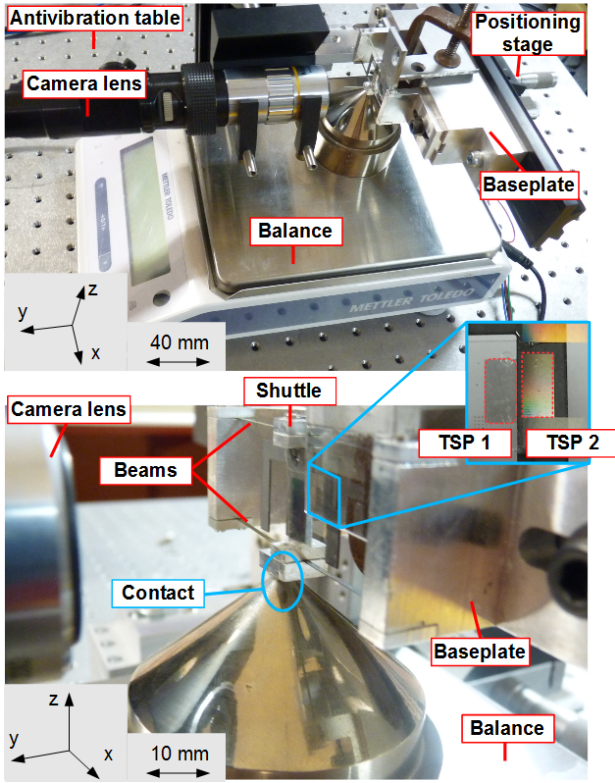


Fig. 11. Experimental setup used for quantification of the measurement range of the sensor. The compliant structure is the one presented in Fig. 4. The shuttle applies a force on the balance when displaced by means of the micro-positioning table. Two twin-scale patterns (TSP) are used: one fixed on the shuttle; the other one fixed on the baseplate. The camera's FOV is focused on the two twin-scale patterns to provide a differential measurement of the displacement proportional to the force.

To measure the shuttle displacement relatively to its baseplate, two twin-scale patterns were used: a first one fixed on the shuttle, the other one on the baseplate (respectively patterns 1 and 2 on Fig. 11). As the baseplate is moving with regards to the camera, such a differential measurement is necessary and the expected force is then proportional to the difference between the baseplate and shuttle displacements; that is also given by the difference between the positions of the two patterns.

Firstly a null force is measured (without contact) to correlate the visual sensor and the reference sensor. Then forces were randomly applied between 19.2 mN and 50.2 mN. Fig. 12 presents the visual measurement of the force and that provided by the balance, according to the balance measurement.

Fig. 13 illustrates the deviation between the two force measurements. For a force range of 50.2 mN the peak-to-peak error is 303 μN , so a linearity equals to $(303 \cdot 10^{-6}) / (50.2 \cdot 10^{-3}) = 0.6\%$ is measured.

V. DISCUSSION

The experiments have demonstrated a linear range of 50.2 mN and a resolution of 32.6 nN. A straightforward calculus of the range-to-resolution ratio gives $1.5 \cdot 10^6$. On closer examination, things are a bit more complicated because

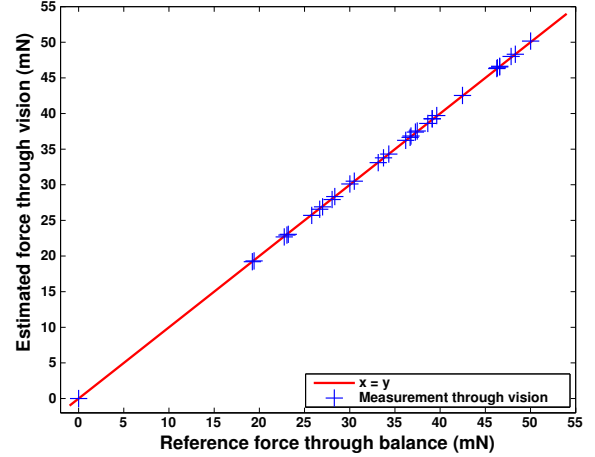


Fig. 12. Visual force estimation through vision compared with the balance measurements, used as reference sensor. Forces are applied by random vertical displacements of the compliant structure, which presses on the balance.

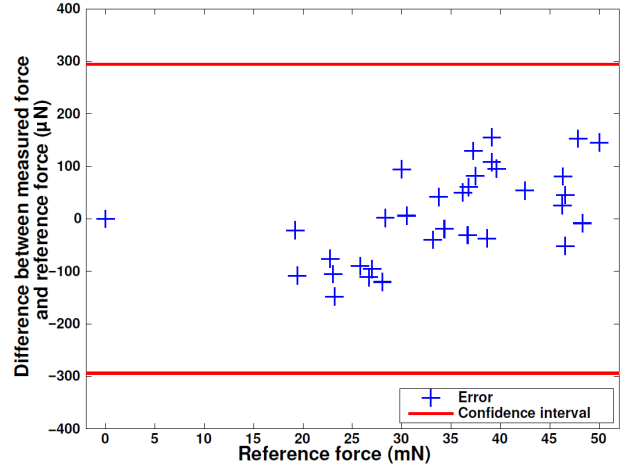


Fig. 13. Deviation between visual and balance measurements. Results remain within the balance confidence interval as defined by its repeatability and linearity.

the stiffness of the compliant structure was different for both experiments.

In the range experiments, the stiffness was 843 N.m^{-1} . So, when a force of 50.2 mN is applied, the displacement is $59.5 \mu\text{m}$. In the resolution experiments, stiffness was 240 N.m^{-1} . The force required to get a displacement of $59.5 \mu\text{m}$ is then 14.3 mN. In this case, the range-to-resolution ratio is $4.4 \cdot 10^5$.

Nevertheless, only a part of the full displacement range of the visual method is used. Indeed beyond $60 \mu\text{m}$ of deformation the force is not proportional to the displacement anymore. Anyway it is possible to increase the measurement range of forces using a non-linear model in place of the Hooke's law. As the unambiguous displacement range is $168 \mu\text{m}$, forces up to 40.3 mN could be measured with a stiffness of 240 N.m^{-1} leading to a range-to-resolution ratio of $1.2 \cdot 10^6$.

The experimental range-to-resolution ratio is below the

expected one calculated in theory but in same the order of magnitude, i.e. 10^6 .

VI. CONCLUSION

The presented method meets many expectations raised by the use of force sensors, notably at the micro-scale. The experiments with a calibrated balance have demonstrated a range of 50.2 mN with a linearity below 0.6%. The repeatability of the measurement has been evaluated to 7.8 μ N and the resolution obtained experimentally is below 32.6 nN. The comparison of the measurements with a calibrated reference force sensor showed that the trueness is 15 μ N.

These performances are at the cutting edge of the state of the art in term of range-to-resolution ratio which reaches the order of magnitude of 10^6 . Beyond these figured performances, the method shows also a flexibility welcomed at micro-scale: remote measurement, insensitivity to electronic noise, and camera self-calibration.

Further works on the design of the compliant structure would enable to further improve the performances. For instance a monolithic structure could reduce the sensitivity to the perturbations and thus improve the repeatability and the trueness. The linear part of the sensor could also be enlarged with a better design. Finally, a more effective camera would improve a lot the resolution: the use of a 12-bits camera instead of 8-bits would for example divide by 16 the theoretical resolution. All these improvements could lead together to gain another order of magnitude in the range-to-resolution ratio.

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