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# Thermo-magnetic behaviour of AFM–MFM cantilevers

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## Abstract

Atomic force microscopy (AFM) experiments were performed to study the behaviour of AFM cantilevers under an external magnetic field  $B$  and temperature field produced by a coil with an iron core. Four cantilever types were studied. Forces were measured for different  $B$  values and at various coil-to-cantilever separation distances. The results were analysed on the basis of a phenomenological model. This model contains the contribution of two terms, one monopole–monopole interaction at short distance, and one apparent paramagnetic interaction in  $\nabla B^2$  at large distance, which represents the temperature effects. We observe a good agreement between the model and the experimental data.

Keywords: magnetic force microscopy (MFM), AFM, cantilever, coating, dc magnetic field.

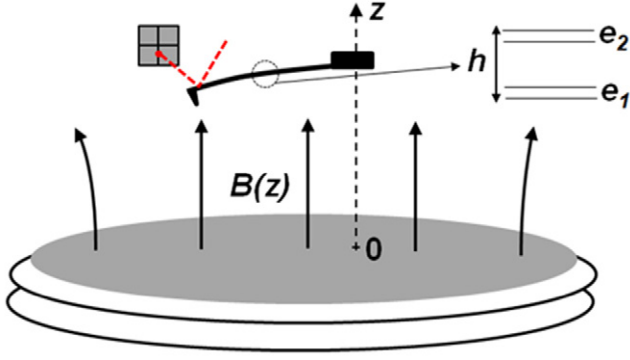
## 1. Introduction

The behaviour of AFM cantilevers under a magnetic field is primarily known through a well-established technique called magnetic force microscopy (MFM) [1, 2]. If the tip approaches the sample surface to within a distance of typically 10–500 nm, magnetic interaction of the tip with the stray field emanating from the sample becomes detectable. It is thus possible to explore magnetic domains with sub-micrometer spatial resolutions. MFM has been used to study the properties of materials for magnetic recording media [3–5], to optimize the recording modes of magnetic heads [6], to investigate the structure and properties of nanoparticles, alloys and nanocomposites [7] and thin films [8], and in the development of methods for magnetic recording with ultrahigh density [9]. MFM uses magnetized probes. In the dynamic mode, the resonance frequency or phase shifts are proportional to magnetic force gradients [3]. In order to quantify the observed effects, it has been common to use monopole and dipole approximations to describe the MFM tip–sample interactions [10]. Metal rings carrying electrical current with inner diameters between 1 and 5  $\mu\text{m}$  were used to calibrate the MFM tip's effective magnetic charge and effective magnetic moment along the

tip axis [11–13]. Quantitative MFM has been strictly limited to magnetic interactions at very short range under a very low magnetic field. This standard approach is convenient for many applications but it is not adapted to study quantitatively the local magnetic properties of condensed phases under an external magnetic field. To this end, it is important to separate the influence of the external magnetic field from the influence of the sample on the MFM probe.

Such studies are important to understand, for example, the orientation of magnetic domains (in bulk or thin films) under a dc or ac external field, or the magnetic instability of ferrofluids (peak patterning) where local response is closely related to the global excitation [14]. These studies require one to characterize the response of AFM cantilevers under an external magnetic field in the absence of samples.

In this paper, we investigate the force experienced by different cantilevers due to the combined effects of a magnetic field and a temperature field, both produced by a dc current in a millimetric coil. This work shows that the cantilevers experience repulsion or attraction far from the coil and only attraction at short distance. The force depends mainly on the coating material as well as on the bulk material of the cantilevers. We point out that effects of temperature dominate the



**Figure 1.** Schematic representation of the experimental set-up. An AFM probe is placed in an external magnetic field produced by a large coil with iron core. For a three-layer cantilever of total thickness  $h$ ,  $e_1$  is the thickness of the coating on the tip side and  $e_2$  on the reflector side.

interaction force at large distance while magnetic effects dominate at short distance. A phenomenological equivalent magnetic model that takes into account the thermal effects can be used to predict the behaviour of cantilevers at the two scales. The paper is organized as follows: in section 2 we describe the materials and methods. The experimental results are presented in section 3 and are discussed in section 4.

## 2. Materials and methods

Different commercial AFM cantilevers were placed in the vicinity of a coil of  $N_C$  turns ( $N_C = 40$ ), with a core of iron, carrying a dc current  $I_C$ , as described in figure 1.  $z$  is the distance between the top surface of the iron core and the position of the fixed end of the cantilever.  $z$  is measured by the vertical piezo-transducer of the AFM.

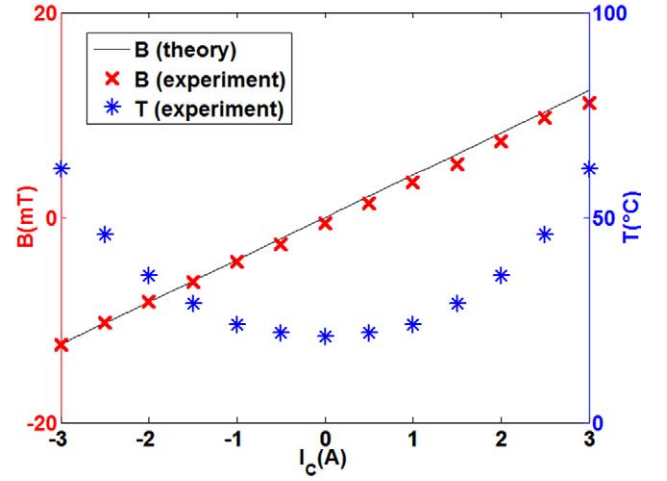
The magnetic field produced by the coil as a function of the distance  $z$  is given by:

$$B(z) = \mu_0 \frac{N_C I_C}{2h_C} \left( \frac{z}{\sqrt{R_C^2 + z^2}} - \frac{z + h_C}{\sqrt{R_C^2 + (z + h_C)^2}} \right), \quad (1)$$

where  $h_C$  and  $R_C$  are respectively the height and the radius of the coil ( $h_C \approx 4.5$  mm and  $R_C \approx 4$  mm).

The relation between  $B$  and  $I_C$  is linear as presented in figure 2. For typical values of  $I_C$  ranging from 0 to 3 A, the field is constant in time and varies from 0 to 12 mT. The field calibration has been realized by means of a Hall effect Gaussmeter (GM07 from Hirst Magnetic Instruments Ltd) positioned in contact with the core at its centre. We have also measured a remanent magnetic field of 0.4 mT in the centre of the iron core when the current is switched off. In parallel, the temperature  $T$  of the iron core increases in  $I_C^2$  by Joule heating (figure 2). Consequently, in our experiments, the production of a magnetic field is systematically associated to a thermal field.

AFM experiments were performed with the Agilent™ 5500 in order to measure the force applied to cantilevers of different types when a dc current of various intensity is passing through the coil. For each cantilever, we determined the sensitivity



**Figure 2.** Iron core temperature and magnetic field measured by a Hall effect Gaussmeter in contact with the iron core and comparison with the theoretical values for  $z = 100$   $\mu$ m.

( $\text{nm V}^{-1}$ ) by measuring the curve of voltage deflection versus vertical displacement when the tip comes in contact with the iron core or aluminium plate (contact mode). No lateral deflection was observed in our experiments. Moreover, we checked that the stiffness calibrated by the thermal tuning method is in agreement with the value given by the manufacturers. Hence, it is straightforward to obtain the force applied to the cantilever in presence of a magnetic and thermal field from the stiffness and the sensitivity. The cantilevers are shown in figure 3. MFMR probes from Nanoworld™ are designed for MFM. They are made from highly doped monolithic silicon and their average stiffness  $k$  is equal to  $2.8 \text{ N m}^{-1}$ . On the tip-side of the cantilever, a coating of approximately 40 nm thick cobalt alloy is deposited. On the reflector side, a 30 nm thick aluminum coating enhances the reflectance and prevents light from interfering within the cantilever. MESP from Bruker™ are also MFM probes where both tip and reflector sides are coated with a 50 nm thick CoCr layer. The cantilever stiffness is  $k \approx 3 \text{ N m}^{-1}$ . Hydra 6V from AppNano™ are V-shaped soft cantilevers ( $k \approx 0.08 \text{ N m}^{-1}$ ) made of  $\text{Si}_3\text{N}_4$ . The tip is made of silicon and the reflector side is gold coated. Finally, we also used  $3.7 \text{ N m}^{-1}$  stiff, non-coated Si cantilevers (Fort from AppNano™). No further information was available on the composition and fabrication of coatings.

The experiments were realized with cantilevers taken from two or three different batches of the same type. Due to the fabrication process, cantilevers of the same batch have slightly different geometrical and material properties. For this reason, each measurement was repeated ten times with different cantilevers. The force values reported on the curves in this paper correspond to the average of these measurements given with a maximum standard deviation of 15%. The accuracy of the distance controlled by the piezo-transducer and of the magnetic field is  $\pm 1$   $\mu$ m and  $\pm 10^{-2}$  mT, respectively.

To separate the effects of the magnetic field with those of the thermal field, we designed a small aluminium plate ( $1 \times 1 \text{ cm}^2$ ) heated by a straight Cu wire. No magnetic field

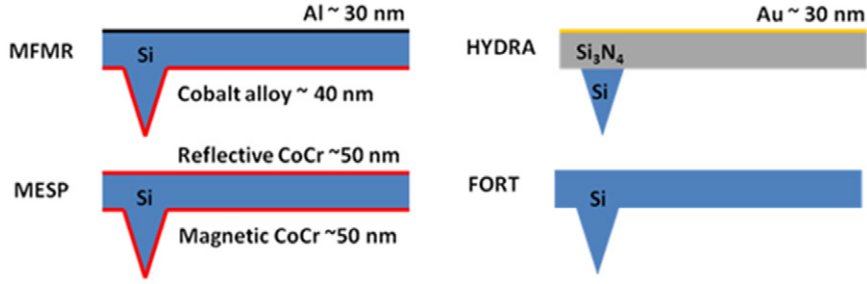


Figure 3. Commercial AFM cantilevers used in the experiments.

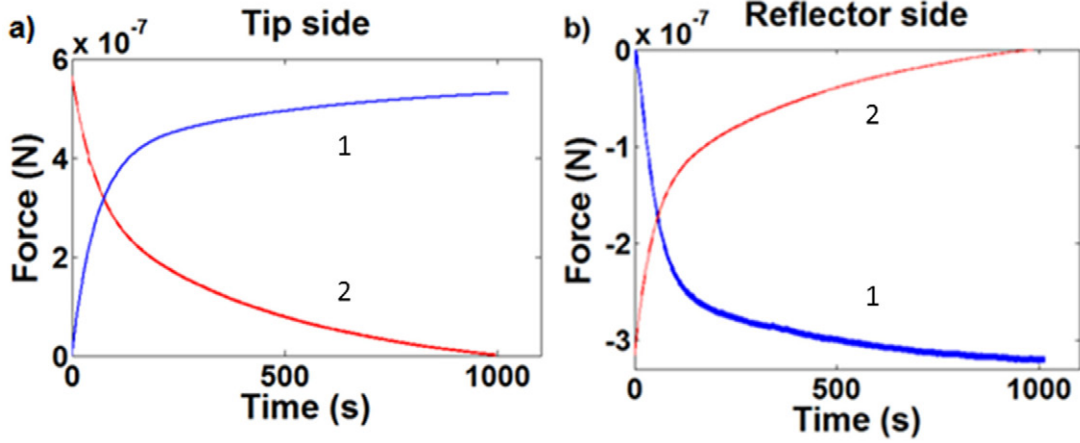


Figure 4. Typical transient regimes observed at  $z = 100 \mu\text{m}$  for MFMR cantilevers when the magnetic field ( $B = 10\text{mT}$ ) is turned on (curve 1) and then turned off (curve 2). (a) tip side configuration and (b) reflector side configuration.

was produced and the temperature could be varied from room temperature up to  $\sim 40^\circ\text{C}$ .

### 3. Experimental results

#### 3.1. Effects of the magnetic and thermal fields

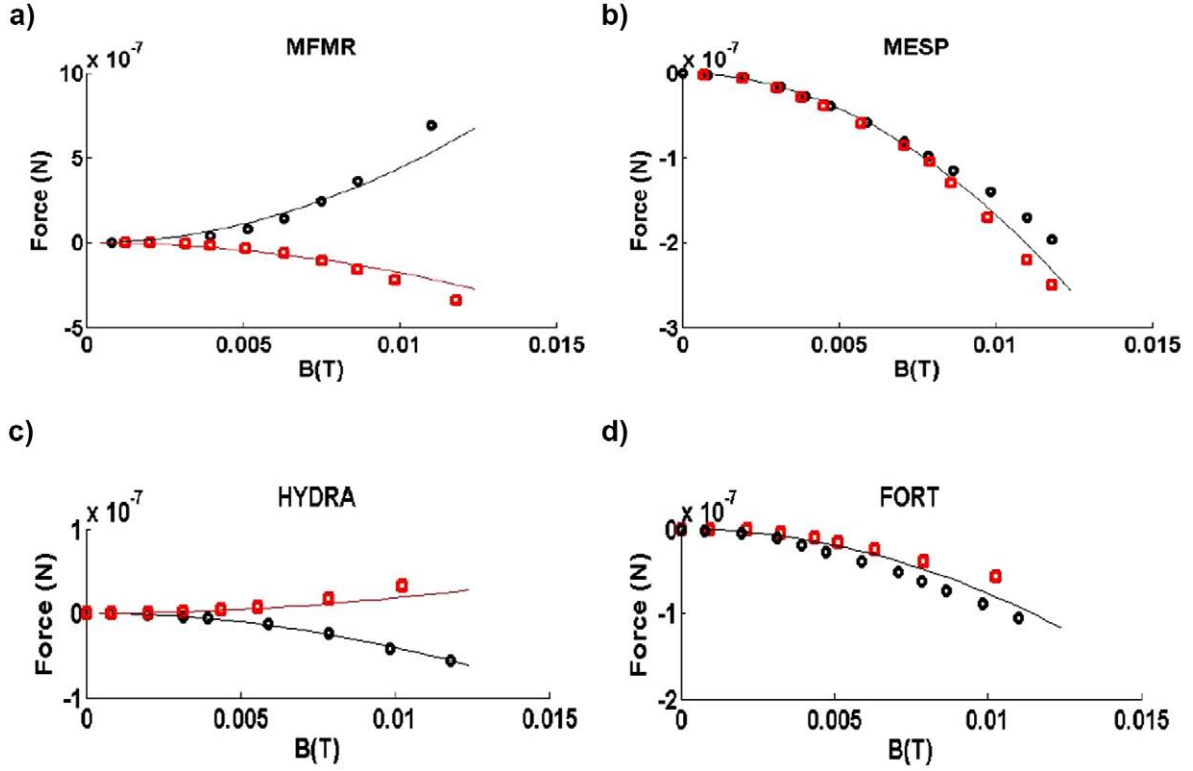
We consider, as a first example, the response at long probe-to-surface distance ( $z = 100 \mu\text{m}$ ) of an MFMR cantilever in a standard configuration, i.e. with the tip directed towards the surface. We refer this configuration as ‘tip side’. In the presence of a dc magnetic field ( $B = 10\text{mT}$  and  $T = 46^\circ\text{C}$ ), the force versus time curve exhibits both short and long characteristic times,  $\tau_1$  and  $\tau_2$  respectively (figure 4(a)). The force is positive, which means repulsion, and reaches relatively high values around  $600\text{nN}$ . After  $1000\text{s}$ , the current is switched off.

We observe a decay that is characterized by a slightly different rate of variation. When the MFMR cantilever is turned upside down (backside directed towards the surface, ‘reflector side’ configuration), the force becomes attractive and its time variation shows the same characteristic times (figure 4(b)). We point out that the same transient regimes occur for both configurations, when the field is turned on and off. However, the magnitude of the force differs in the reflector side and tip side cases by a factor 2. The main result is that the force applied on the cantilever depends on the orientation of the cantilever. Applying a dc potential between the cantilever and the iron core, we also verified that the interaction force is not due to static electrical charges.

In terms of reproducibility, field-on or field-off repeated measurements showed a similar tendency with different MFMR cantilevers coming from the same batch. The values of  $\tau_1$  and  $\tau_2$  vary between  $45\text{--}55\text{s}$  and  $400\text{--}500\text{s}$ , respectively. These two times are larger by some orders of magnitude than characteristic spin relaxation times. In consequence, the transient effects do not seem to be linked to magnetism.

We have verified that  $\tau_1$  does not match the time constant of the equivalent RL circuit. The inductance and the resistance of the coil are  $L = \frac{\mu_0 N^2 \pi R^2 c}{hc}$  and  $R = \frac{\rho_w L_w}{S_w}$ , where  $\rho_w = 17 \times 10^{-9} \Omega\text{m}$ ,  $L_w = 0.1\text{m}$  and  $S_w \approx 10^{-9}\text{m}^2$  are the resistivity, the length and the section area of the wire, respectively. The electrical time constant  $\tau = L/R$  is equal to a few  $\mu\text{s}$ , which is much lower than  $\tau_1$ . This time seems to be related to the convective effects due to direct heating of ambient air by the coil (the magnitude of the air velocity is of the order of  $\sqrt{g\beta\Delta T l}$ , where  $\beta$  is the thermal expansion coefficient,  $\Delta T$  the temperature difference between coil and ambient air and  $l$  is the height of the coil). Furthermore, the second time constant  $\tau_2$  corresponds to the thermal diffusion characteristic time corresponding to temperature equilibrium of the system:  $\tau_2 = \frac{C_p}{2hS}$ , where  $h$  is the natural convection coefficient corresponding to heat transfer between the iron core and air ( $h \approx 18\text{W K}^{-2}$  estimated from [15]),  $S$  is the area of the iron core ( $S = 6.3 \times 10^{-5}\text{m}^2$ ) and  $C_p$  its heat capacity ( $C_p = 1.1\text{J K}^{-1}$ ).

The variations of force with the magnetic field for the four cantilever types are presented in figure 5. For each value of



**Figure 5.** Force versus magnetic field at  $z = 100 \mu\text{m}$ . Experimental data: (○) tip side configuration; (□) reflector side configuration. MFMR (a), MESP (b), HYDRA (c) and FORT (d). Solid lines: comparison with the model (equations (8) and (9)).

magnetic field, a delay of 10 min is observed before the force is measured at  $z = 100 \mu\text{m}$ . All the results show that the interaction force varies as a function of  $B^2$ . In the case of MFMR (figure 5(a)), we clearly distinguish strong repulsive and attractive behaviours on tip and reflector side, respectively.

The dependence of the interaction force on the magnetic field is represented in figures 5(b)–(d) for MESP, HYDRA and FORT cantilevers, respectively. MESP cantilevers have the same coating on the tip and reflector sides and show the same attractive behaviour on both sides. HYDRA cantilevers are attracted on the tip side which is uncoated and repulsed on the reflector side which is gold coated. Finally, FORT cantilevers, which are uncoated, are attracted on both sides. We note however a deviation between the results for the two orientations due to differences in the geometrical and material properties of the two sides. The forces are of the same order for MFMR, MESP and FORT cantilevers. On the other hand, the forces measured with HYDRA cantilevers are one order of magnitude lower. The attractive and repulsive behaviours are summarized in table 1.

### 3.2. Cantilever–coil distance effects

In this section, we present the variation of force as a function of distance for different dc magnetic fields. The results for the MFMR cantilevers are presented in figure 6. For each value of the magnetic field, the force exhibits two different behaviours for short range and long range distances: at short distance, the force is attractive whatever the orientation of the cantilever and scales as  $-1/z^2$ . The dependence is not measurable below

a few  $\mu\text{m}$  because the cantilever deflection is too large. At long distance, the force reaches an asymptotic value. It is repulsive for tip side cases and attractive for reflector side cases. The asymptotic value varies with the magnitude of  $B$ , as shown in figure 5(a).

The MESP probes display similar behaviour as MFMR, i.e. a  $-1/z^2$  force dependence at short distance and an asymptotic force at long distance, which is now attractive for both tip side and reflector side orientations. These results are in agreement with figure 5(b).

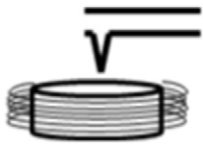
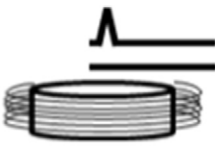
HYDRA cantilevers (figure 8) show the same short distance trend as the other cantilever types. The asymptotic force values are coherent with figure 5(c). It was difficult to measure forces above 5 mT in the tip side configuration because of the low cantilever stiffness, which implies very large deflections. For example, considering  $k \sim 0.08 \text{ N m}^{-1}$  and a sensitivity factor  $\chi \sim 100 \text{ nm V}^{-1}$  between the signal of the photodetector and the deflection, the limit of 10V of the photodetector is exceeded at forces above 80 nN. Experimental results with FORT cantilevers are represented in figure 9. Tip and reflector side responses present a similar attractive behaviour within the experimental uncertainties.

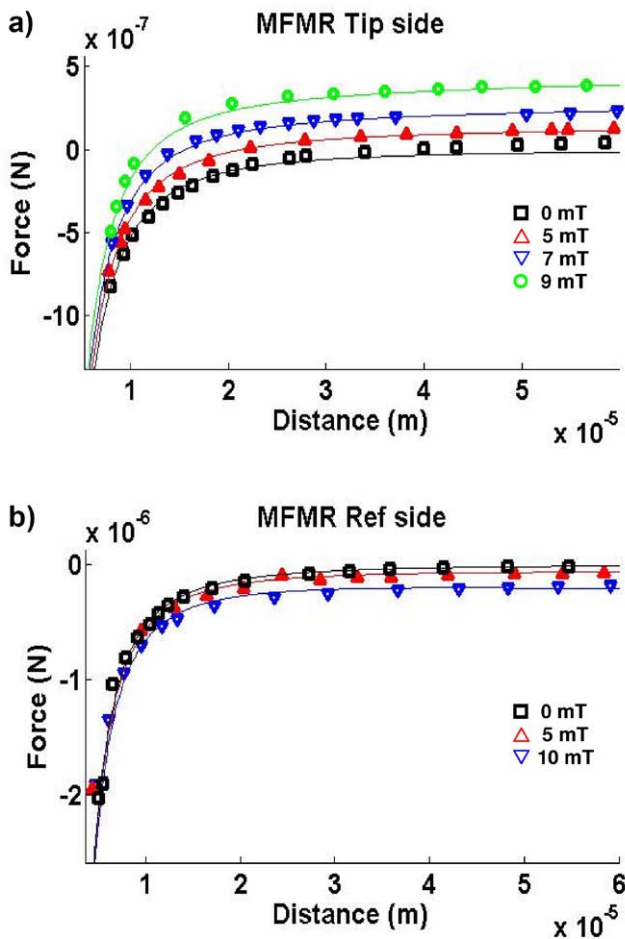
## 4. Discussion

### 4.1. Long range force model

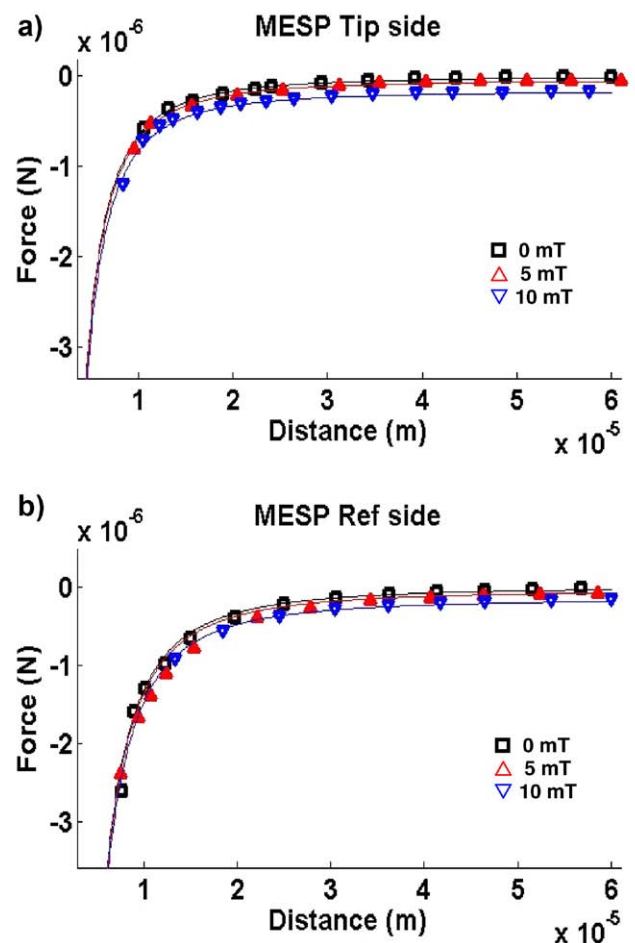
In order to analyse the effects of large variations of temperature, we studied the long range response of the cantilevers using the aluminium plate heated by the Joule effect without

**Table 1.** Long distance (at  $z = 100 \mu\text{m}$ ) attractive (A) or repulsive (R) response of the four cantilever types for tip side and reflector side configurations. Apparent magnetic coefficients were obtained by fitting the experimental data.

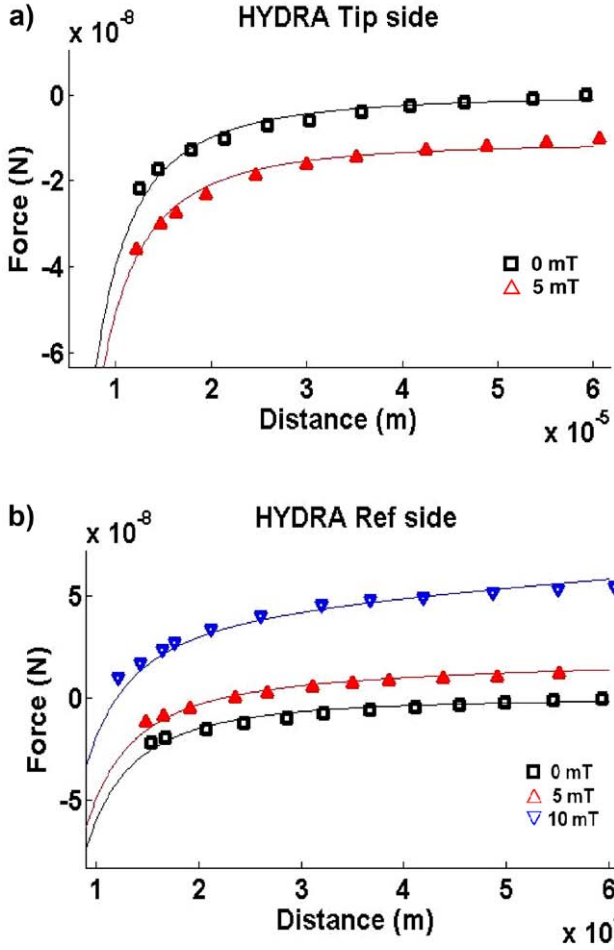
Probe reference			Apparent magnetic coefficients
MFMR	R	A	$\Gamma_1 \cong -2.93 \times 10^8$ $\Gamma_2 \cong 3.91 \times 10^8$ $\Gamma_{\text{bulk}} \cong 186$
MESP	A	A	$\Gamma_1 = \Gamma_2 \cong 2.84 \times 10^4$ $\Gamma_{\text{bulk}} \cong 186$
HYDRA	A	R	$\Gamma_1 = 0$ $\Gamma_2 \cong -1.66 \times 10^8$ $\Gamma_{\text{bulk}} \cong 8.31 \times 10^6$
FORT	A	A	$\Gamma_1 = 0$ $\Gamma_2 = 0$ $\Gamma_{\text{bulk}} \cong 186$



**Figure 6.** Magnetic force versus distance for MFMR probes, measured at different  $B$  values. (a) tip side configuration; (b) reflector side configuration. Solid lines: comparison with the model (equations (8)–(10)).



**Figure 7.** Magnetic force versus distance for MESP probes, measured at different  $B$  values. (a) tip side configuration; (b) reflector side configuration. Solid lines: comparison with the model (equations (8)–(10)).



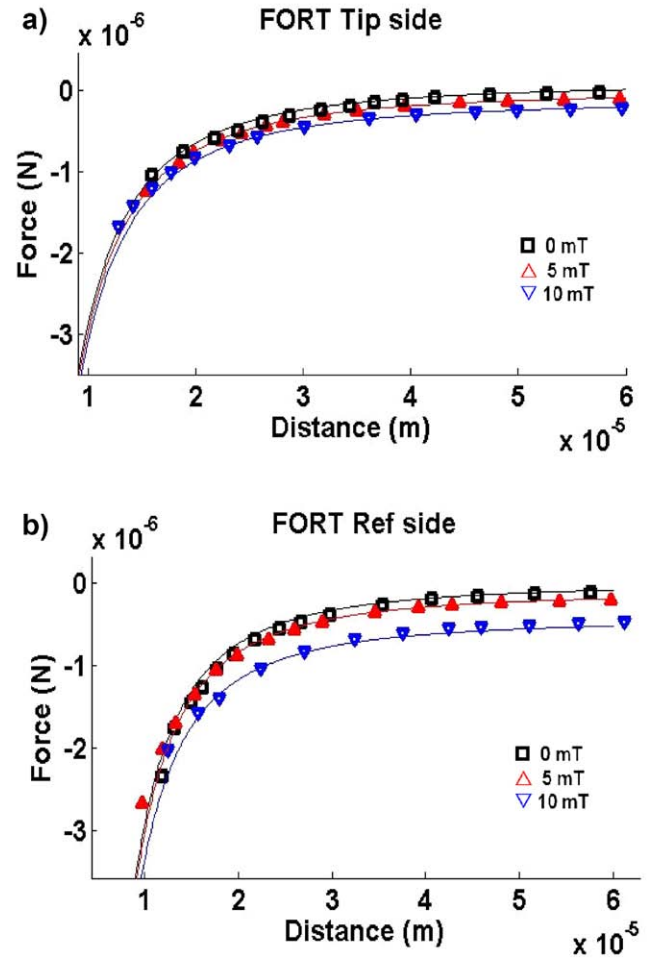
**Figure 8.** Magnetic force versus distance for HYDRA probes, measured at different  $B$  values. (a) tip side configuration; (b) reflector side configuration. Solid lines: comparison with the model (equations (8)–(10)).

a magnetic field. For all the cantilevers, we observed that the long range asymptotic forces ( $z > 100 \mu\text{m}$ ) are of the same order of magnitude than those measured with the electrical coil. In particular, we found exactly the same attractive and repulsive responses when the orientations of the cantilevers were reversed. Hence, in the magnetic experiments, the thermoelastic properties seem to control the long range behaviour by the temperature gradient effect and, for multilayer cantilevers, by the difference in the coefficients of thermal expansion.

In the framework of linear elasticity, the radius of curvature of the bent multilayer cantilever is given by:

$$R = \frac{\int E(z)z^2 dz}{\int E(z)\epsilon(z)z dz}, \quad (2)$$

where  $E(z)$  is Young's modulus at the position  $z$  in the cantilever and  $\epsilon(z)$  is the thermal deformation of the cantilever at  $z$ , which is proportional to the thermal expansion coefficients of the layers and their temperature variations [16]. When  $R \ll L$ , where  $L$  is the length of the cantilever, the deflection is  $\delta \approx \frac{L^2}{2R}$  and the bending force is  $F = k\delta$ .



**Figure 9.** Magnetic force versus distance for FORT probes, measured at different  $B$  values. (a) tip side configuration; (b) reflector side configuration. Solid lines: comparison with the model (equations (8)–(10)).

Equation (2) also takes into account the effects of the temperature gradient in the cantilever. In case of a uniform temperature and a bimorph cantilever, equation (2) is similar to equation (4) in [17]. Modelling the cantilever bending requires precise values of the thicknesses and the thermoelastic film properties of the layers, and the geometry of the cantilever. With values of these parameters given in the literature, we obtain a good qualitative agreement with the experimental response of the cantilever: first, the maximum deflection scales as  $I_C^2$  and second, we find the same attractive and repulsive behaviour in tip and reflector side configurations (figure 5). However, the magnitudes of the forces are difficult to predict precisely due to the uncertainties in the physical data and the presence of the internal stresses.

We propose a simple phenomenological model based on the force balance of a multilayer cantilever that takes into account the main experimental observations at long range, i.e. the force is a square function of the electrical current and then of  $B^2$ , and the specific behaviour of each cantilever in the tip and reflector side configurations. This model was also developed because it is easier to control the magnetic field via the electrical current (Biot–Savart law) than the temperature field

due to its highly non-linear behaviour. Moreover, the thermal properties of the multilayer cantilever are not always well established.

In the model, we consider that the cantilever is characterized by its length  $L$ , width  $w$  and thickness  $h$ , and is clamped at one extremity.  $e_1$  and  $e_2$  are the thickness of the coating layers on tip and reflectors sides, respectively ( $e_1, e_2 \ll h$ ). Considering a cantilever substrate with thin film layers, equation (2) can be approximated by the differential equation governing the cantilever deflection  $y$  along the  $x$ -axis:

$$\frac{d^2y}{dx^2} = \frac{M(x)}{EI}, \quad (3)$$

where  $E$  and  $I$  are Young's modulus and the moment of inertia of the bulk substrate, and  $M$  the total bending moment.  $M$  and

$I$  are respectively given by  $M(x) = \left(\frac{\rho}{2}\right)(x-L)^2$ , where  $\rho$  is the force per unit length and  $I = \frac{wh^3}{12}$ . In the case of V-shaped cantilevers (HYDRA), this last expression must be multiplied by 2. From the conditions at the clamped end,  $y(0) = 0$  and  $dy(0)/dx = 0$ , we obtain the general expression of the deflection:

$$y(x) = \frac{\rho x^2}{24EI}(4xL - L^2 - x^2). \quad (4)$$

The deflection is measured at the laser spot,  $x = 0.95L$ . We postulate that the effects of orientation are due to the multilayer structure of the cantilevers as shown in figure 1 and  $\rho$  is a function of  $B^2$ . The force per unit length experienced by the cantilever is given by  $\rho = \rho_1 + \rho_{\text{bulk}} + \rho_2$ , where  $\rho_1$  and  $\rho_2$  are the forces per unit length in the coating layers, and  $\rho_{\text{bulk}}$  is that experienced in the bulk substrate. By analogy with paramagnetism, we assume that the forces on the cantilever are:

$$\rho_1 = \frac{we_1\Gamma_1}{2\mu_0} \frac{dB^2}{dz} \Big|_{z_1}, \quad (5)$$

$$\rho_2 = \frac{we_2\Gamma_2}{2\mu_0} \frac{dB^2}{dz} \Big|_{z_2}, \quad (6)$$

$$\rho_{\text{bulk}} = \frac{w\Gamma_{\text{bulk}}}{2\mu_0} \int_{z_1}^{z_2} \frac{dB^2}{dz} dz, \quad (7)$$

where  $\Gamma_1$ ,  $\Gamma_2$  and  $\Gamma_{\text{bulk}}$  are 'apparent magnetic coefficients' of tip side, reflector side and bulk layers respectively, and  $\mu_0$  is the vacuum magnetic permeability.  $z_1$  and  $z_2$  are the vertical positions of the coating layers defined as  $z_1 = z$  and  $z_2 = z + h$  for the tip side configuration and  $z_1 = z + h$  and  $z_2 = z$  for the reflector side configuration (figure 1).

At a constant tip-to-surface distance  $z$ , we deduce from equation (1) that  $\frac{dB^2}{dz}$  and  $\frac{d^2B^2}{dz^2}$  are proportional to  $I^2$  and therefore to  $B^2$ , in such a way that  $\rho = KB^2$ . By fitting the experimental curves presented in figure 5, it is then possible to estimate  $K$  for both tip and reflector side configurations. From

the Taylor series expansions of  $\rho_1$  and  $\rho_2$  at  $z + h/2$ , we obtain for the tip side configuration:

$$\rho = \frac{w}{2\mu_0} \frac{dB^2}{dz} \Big|_{z+\frac{h}{2}} (e_1\Gamma_1 + e_2\Gamma_2 + h\Gamma_{\text{bulk}}) + \frac{wh}{2\mu_0} \frac{d^2B^2}{dz^2} \Big|_{z+\frac{h}{2}} (e_2\Gamma_2 - e_1\Gamma_1) + O(h^2), \quad (8)$$

In the same way, the force per unit length for the reflector side configuration is:

$$\rho = \frac{w}{2\mu_0} \frac{dB^2}{dz} \Big|_{z+\frac{h}{2}} (e_1\Gamma_1 + e_2\Gamma_2 + h\Gamma_{\text{bulk}}) + \frac{wh}{2\mu_0} \frac{d^2B^2}{dz^2} \Big|_{z+\frac{h}{2}} (e_1\Gamma_1 - e_2\Gamma_2) + O(h^2). \quad (9)$$

The case of uncoated FORT cantilevers (see figure 5(d)), for which  $e_1 = 0$  and  $e_2 = 0$ , has been used to determine  $\Gamma_{\text{bulk}} \cong 186$  in silicon based cantilevers (MFMR and MESP). From this value it is possible to determine  $\Gamma_1$  and  $\Gamma_2$  coefficients for MFMR and MESP by fitting the experimental data (figure 5) with the linear system (8) and (9). In the case of HYDRA probes,  $e_1 = 0$  in equations (8) and (9). All the apparent magnetic  $\Gamma$  coefficients were calculated from the fit of the averaged experimental data by standard linear regression and are reported in table 1.

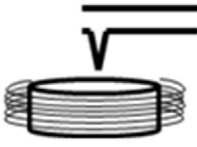

We point out that the  $\Gamma$  coefficients obtained in this phenomenological model have no link with the standard magnetic susceptibilities and show sensitivity to the total thermal field. For example, the magnetic properties of the silicon material found in the literature [18] give an interaction force much lower in magnitude than what we have observed.

In figure 5, the experimental data are fitted by the model (equations (8) and (9)). We find a good agreement for the  $B^2$  dependence of the interaction force for all the cantilevers with a unique value of  $\Gamma_{\text{bulk}}$  for a Si substrate.

#### 4.2. Full range force model

The variation of the interaction force with the distance displays two regimes, at short and long distance. At long distance, the  $z$  dependence is derived from the applied magnetic field (equations (1) and (5)–(7)). At short distance, the interaction force curves for different applied field converge to a unique one. Without any current, the cantilever is submitted to the remanent magnetic field of the iron core. When there is no magnetic field and no magnetization but heating (experiments with the aluminium plate), the force is of smaller magnitude and exhibits only capillary effects at short distance. In the limit of short distance and in the presence of a magnetic field, the force–distance relation scales as  $-1/z^2$  for all the cantilevers (figures 6–9) and can be modelled by monopole–monopole interaction. In such an approximation, the force is expressed as:

**Table 2.** Product of magnetic monopoles ( $g_m^{\text{probe}} g_m^{\text{surface}}$  in  $\text{A}^2 \text{m}^2$ ) obtained by fitting the experimental data for the four cantilever types for tip side and reflector side configurations.

Probe reference		
MFMR	$1 \times 10^{-10}$	$2 \times 10^{-10}$
MESP	$2.5 \times 10^{-10}$	$2.5 \times 10^{-10}$
HYDRA	$3 \times 10^{-10}$	$4.5 \times 10^{-10}$
FORT	$4.5 \times 10^{-10}$	$5.5 \times 10^{-10}$

$$F_{\text{monopoles}} = -\frac{\mu_0 g_m^{\text{probe}} g_m^{\text{surface}}}{4\pi z^2}, \quad (10)$$

where  $g_m^{\text{probe}}$  is the equivalent monopole associated with the probe and  $g_m^{\text{surface}}$  that associated with the sample surface. The monopole model can be understood as an effect of local magnetic field that can be decomposed in a multipolar expansion, in which the first monopole–monopole term dominates. In our experiments, the local force characterizes the magnetization of the iron surface. In other situations, this force could be measured to characterize the magnetization of a medium by an external field (such as, for instance, a ferrofluid).

From the experiments, we cannot determine the  $g_m$  coefficients in equation (10) separately. Only the product of magnetic monopole intensities  $g_m^{\text{probe}} g_m^{\text{surface}}$  can be estimated. These values are obtained by fitting the raw data and are summarized in table 2. We note that the order of magnitude of all the monopole products is  $\sim 10^{-10} \text{A}^2 \text{m}^2$ , which could be used to estimate the short distance interaction force for this iron core.

In order to determine the full range interaction force undergone by the cantilevers, equations (8)–(10) can be combined. Figures 6–9 show the comparison between this full range model and the experimental data, for which the  $\Gamma$  coefficients are identical to those calculated from figure 5. While the multilayer model predicts the correct behaviour of the cantilever, the question remains of how the chemical composition and the process of fabrication affect the thermal and magnetic properties of cantilevers. At long range, in addition to temperature, other physical effects can contribute to the bending of the cantilevers, such as magnetostriction [19–21].

Based on this study, we can propose a method to develop MFM experiments to investigate the magnetic response of a given sample to a dc or ac external magnetic field. In order to obtain a good lateral resolution, the MFM tip has to be close to the sample surface (less than 50 nm). In this case, the force on the cantilever is dominated by monopole–monopole interaction. First, we have to find a cantilever characterized by a low  $g_m^{\text{probe}}$  value and a repulsive behaviour in the tip side configuration (as MFMR) in order to increase the range of the measured force by the photodiode. Second, the cantilever stiffness must be large to limit the cantilever deflection in the presence of a large interaction force.

## 5. Conclusion

The study of the near-field behaviour of samples by AFM under an external magnetic field requires knowledge of the cantilever response in similar conditions. Therefore, we have characterized in this paper the response of the AFM cantilevers to an external dc magnetic field. A coil with an iron core was used to produce the external magnetic field. The force exerted on the cantilever was measured by its deflection. The results for four types of cantilevers have been presented and discussed. We have measured both the effects of  $B$  and temperature. At long distance, we observed that the temperature effects dominate while the magnetic effects dominate at short distance. We have proposed a phenomenological model based on two contributions: a monopole–monopole interaction at short distance and a multilayer interaction at long distance characterized by a generalized paramagnetic force proportional to  $\nabla B^2$  modelling the temperature effects. Based on this work, it is possible to define the cantilever properties that are adapted for the study of the mechanisms at the nanoscale occurring in magnetic samples under external magnetic fields.

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