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# Ultrasonic fatigue tests at high temperature on an austenitic steel

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## KEYWORDS

Fatigue;  
Gigacycle regime;  
High temperature;  
Austenitic steel;  
Finite element  
method;  
Crack initiation  
mechanism

**Abstract** The objective of this paper is to study the gigacycle fatigue behavior of an austenitic steel at temperatures of 600 °C and 700 °C under fully reverse loading ( $R=-1$ ). Numerical simulation by finite element method (FEM) was used to design the specimens and to analyze the effects of the variation in the dynamic Young modulus with temperature from measurements of the ultrasonic resonance frequency. Finally, new stress-life curves for this material are presented for a lifetime range from  $10^5$  to  $10^9$  cycles at room temperature, 600 °C and 700 °C.

## 1. Introduction

One of the main catastrophic causes of failure in propulsion engine is fatigue. An operative life beyond  $10^8$

cycles is required and it is necessary to investigate the fatigue behavior in the gigacycle range to improve the design. Using classical fatigue criteria, a near hyperbolic relationship between stress and fatigue life is assumed; however, new experimental results show that the fatigue fracture can occur beyond  $10^7$  cycles, and the concept of endurance limit is not correct [1]. Numerous works have shown that the fatigue behavior of different kinds of materials in the gigacycle regime can be obtained effectively by using ultrasonic fatigue testing machines, and that frequency effects are not important in the determination of the lifetime [2–4]. Many studies in this field have been reported at room temperature [5–7], but available database

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## Nomenclature

$\rho$	density
$E_{static}$	static Young modulus
$E_{dynamic}$	dynamic Young modulus
$U(x)$	amplitude of vibration
$k$	stiffness constant
$L$	bar length
$f_{reso}$	resonance frequency

$T$	temperature
$\omega$	angular frequency
$A(x)$	cross section of the specimen
$L_1$	resonance length
$L_2$	cylindrical length
$R_2$	radius in the cylindrical section of the specimen
$R_1$	radius in the central section of the specimen
$\sigma_{max}$	maximum stress
$A_0$	imposed displacement

information at high temperature is very scarce, because the time to reach  $10^9$  cycles at high temperature using traditional low cycle fatigue testing machines is an almost impossible task to perform. Under these conditions, ultrasonic fatigue testing is a new attractive technique which allows to get fatigue results in a shorter time and with lower costs compared to traditional fatigue procedures.

In this work, we present a methodology to get mechanical and fatigue properties of an austenitic steel subjected to very high cycle loadings ( $R=-1$ ) at high temperatures using the ultrasonic fatigue testing machine and numerical simulations.

## 2. Material

The material analyzed in this research is an austenitic steel because of its high strength, small Young modulus variation at high temperatures and good resistance to oxidation. The chemical composition is shown in Table 1 and the mechanical properties in Table 2.

Figure 1 shows a general view of the microstructure, where it is possible to see an austenitic structure. Two types of inclusions are presented: very fine intergranular iron and chromium carbides and more massive inclusions rich in Mn, Mo, Nb.

Since Young modulus and density are the dominant variables in ultrasonic tests, it is very important to know their values as accurately as possible. Numerical values at room temperature of the density  $\rho$ , static Young modulus  $E_{static}$ , and thermal conductivity  $k$  are shown in Table 2 as

**Table 1** Chemical composition of the analyzed steel.

Steel	C	Si	Mn	Cr	Ni	W	Nb	N
H854	0.53	0.25	9.5	21	3.8	1	2.2	0.5

**Table 2** Thermo mechanical properties of the analyzed steel.

Steel	$\rho$	$E_{static}$ /(N/mm <sup>2</sup> )	UTS /(N/mm <sup>2</sup> )	$k$ /(W/(m·K))	$\alpha$ /K <sup>-1</sup>
H854	7.7	$215 \times 10^3$	950–1150	14.5	$18.8 \times 10^{-6}$

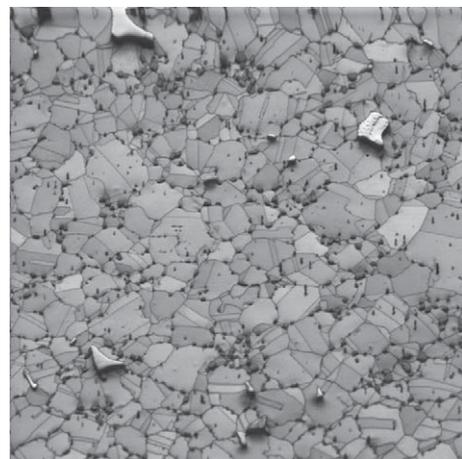
well as the thermal expansion  $\alpha$  at 500 °C. The dynamic Young modulus at different temperatures was determined following the procedure presented in Section 5.

## 3. Experimental procedure

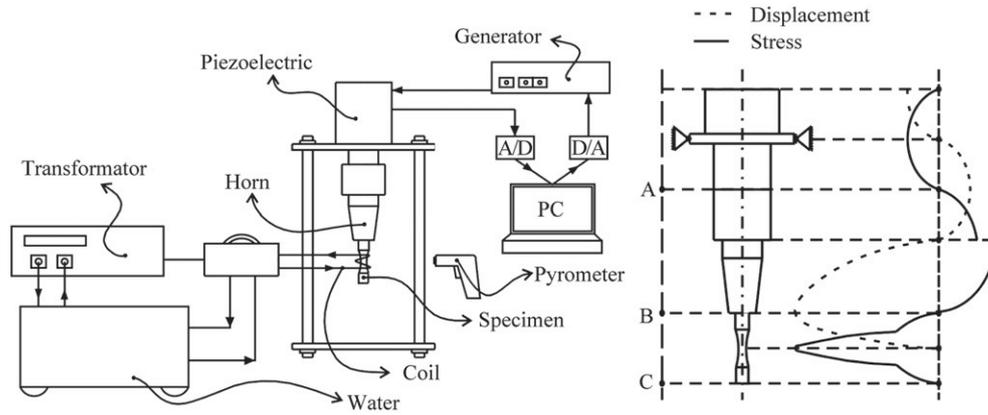
The first ultrasonic fatigue testing machine was introduced by Mason [8]. In the last two decades, research advances in this area have been very important. There are several designs of ultrasonic machines, but they always have three main components [9] (Figure 2):

- 1) An electric power generator which converts 50 or 60 Hz voltage into ultrasonic 20 kHz electrical sinusoidal signal.
- 2) A piezoelectric transducer which transforms the electrical signal into mechanical vibration of the same frequency.
- 3) An ultrasonic horn that amplifies the vibration coming from the transducer in order to obtain the required stress amplitude in the middle section of the specimen.

The generator is connected to a PC using an A/D acquisition card for controlling the generator and the piezoelectric actuator that induced the mechanical movement. The transducer generates a mechanical wave



**Figure 1** Austenitic microstructure of the analyzed material.



**Figure 2** Fatigue testing machine and vibratory stress and displacement field.

with an intrinsic frequency of 20 kHz and for this reason, the specimen must be designed with this same natural frequency in its first mode of vibration. In this way, the specimen reaches a maximum displacement (null stress) at the end and maximum stress (null displacement) in the central section, see Figure 2.

This type of machine is used for fatigue life analysis at more than  $10^5$  cycles. The operative frequency range goes from 19.5 to 20.5 kHz, and if it decreases below 19.5 kHz or increases above 20.5 kHz, the test is stopped automatically. In contrast with the low and mega cycle fatigue tests, where the machine has a natural frequency different from the specimen, in the ultrasonic fatigue machines the specimen-machine forms a resonance system in which the displacement amplitudes are kept constant during the test at a pre-selected value. In the tests, the machine stops automatically when a crack propagates rapidly causing a change in the resonance frequency of the system.

The mechanical system works in the elastic regime, and consequently the relationships between displacement, strain and stress are linear. In the machine used in this work, the electrical voltage is also linear and proportional to the displacement. The ultrasonic fatigue testing machine calibration is required to make the system vibrate in resonance at an ultrasonic frequency of about 20 kHz. The calibration was performed with a cylindrical specimen and a laser displacement sensor situated at one specimen end, which registers displacements from 1  $\mu\text{m}$  to 199.9  $\mu\text{m}$  with a resolution of 0.1  $\mu\text{m}$ .

In the room temperature tests, the central part of the specimen is cooled with fresh air.

An induction coil technique was used in the tests at high temperatures. The coil was made of a spirally shaped copper tube situated in the middle of the specimen, in order to heat the specimen around its middle section at a given temperature. An optical pyrometer was used during the tests to measure and control the temperature at the center of the specimen. Its emissivity was calibrated with a reference instrument. Thermocouples have also been used to calibrate the pyrometer and to measure the temperature during the tests. The

pyrometer presents some practical advantages compared to thermocouples, because it does not require any welding to the specimen allowing measurements at different points in a very easy way.

#### 4. Analytical and numerical models for designing a specimen

In dynamic loading, by taking a bar and neglecting the transversal contraction, the natural frequency is given by:

$$f_{\text{reso}} = \frac{1}{2L} \sqrt{\frac{E_d}{\rho}} \quad (1)$$

where  $E_d$  is the dynamic Young modulus which considers the dynamic effects and it is usually different from the static Young modulus,  $\rho$  the density and  $L$  the bar length.

##### 4.1. Experimental procedure for dynamic Young modulus determination

So, the knowledge of the dynamic modulus  $E_d$  is essential and the relationship between  $E_d$  and temperature,  $E_d = E_d(T)$  is therefore required for designing a specimen in an ultrasonic load regime at high temperature. We have determined the dynamic Young modulus at the resonant frequency of a cylindrical specimen, first at room temperature and later at high temperature, using the gigacycle fatigue testing machine and numerical simulations by FEM.

In order to determine  $E_d$  at room temperature, a cylindrical specimen initially of length  $L = (2f_{\text{reso}})^{-1} (E_{\text{static}}/\rho)^{0.5}$  (see Eq. (1)) was designed with the static Young modulus (Table 2) and a resonance frequency  $f_{\text{reso}} = 20$  kHz. Since tests displayed a natural frequency lower than 20 kHz, the specimen was cut until obtaining experimentally  $f_{\text{reso}} = 20$  kHz. The final dimension of the cylindrical bar was 132.1 mm. Finally, the dynamic Young modulus was obtained as  $E_d = \rho(2L20.000)^2$ .

**Table 3** Temperature and frequency values measured on cylindrical specimen (unit: °C).

Cases	I	II	III	IV	V	VI	VII
$L_1=15$ mm	20	53	100	96	130	150	210
$L_2=32$ mm	20	76	152	200	240	330	420
$L_3=64$ mm	20	100	200	300	400	500	550
$L_4=94$ mm	20	80	155	250	340	430	500
$L_5=113$ mm	20	65	130	200	300	380	400
Freq./kHz	19.98	19.89	19.82	19.73	19.64	19.54	19.50
FEM/kHz	20.00	19.89	19.819	19.734	19.65	19.52	19.52
Error/%	0.1	0	0.005	0.02	0.05	0.1	0.12

An induction coil was used for heating a cylindrical specimen. Temperature was measured with an optical pyrometer as well as with thermocouples connected along the length of the specimen. Seven tests were conducted, registering the temperatures in different points of the specimen together with its resonance frequency, as shown in Table 3. Here,  $L_i$  is the length from the top of the specimen to the measured point. The maximum temperature is located at point  $L_3$ . Measurements of resonance frequency were performed in the gigacycle fatigue testing machine for a cylindrical specimen at seven different temperature conditions (cases from I to VII). By adjusting coefficients of a quadratic function, the dynamic Young modulus variation in terms of temperature was finally approximated as:

$$E_d(T) = 2.01134 \times 10^{11} - 1.09468 \times 10^7 T - 7417.87 T^2 \quad [\text{Pa}] \quad (2)$$

where  $T$  is given in °C (see Figure 3).

When using this expression in the modal analysis, the computed vibration frequencies approximated the experimental values with an error less 0.12% as shown in Table 3.

#### 4.2 Specimen design

In a typical ultrasonic fatigue specimen the length of the cylindrical part of the specimen is adjust to ensure the resonance of the system.

Then, the resonance length  $L_1$  can be shown to be:

$$L_1 = \frac{1}{k} \tan^{-1} \left\{ \frac{1}{k} [\beta \coth(\beta L_2) - \alpha \tanh(\alpha L_2)] \right\} \quad (3)$$

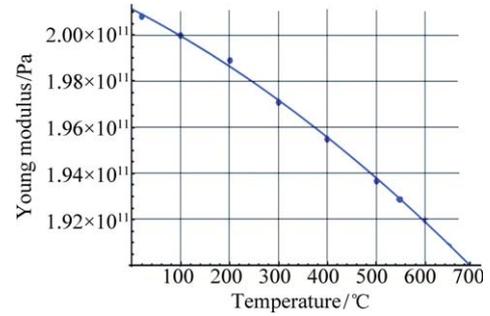
where

$$\beta = \sqrt{\alpha^2 - k^2} \quad (4)$$

$$\alpha = \frac{1}{L_2} \cosh^{-1}(R_2/R_1) \quad (5)$$

$$k = \omega \sqrt{\frac{\rho}{E_d}} \quad (6)$$

For the material used in this work (see Tables 1 and 2 together with Eqs. (3–6))  $L_1$  at 20 kHz becomes 23.75 mm.

**Figure 3** Dynamic Young modulus vs. temperature for the analyzed steel.

A modal analysis with an axisymmetric FEM model was also performed in order to make a comparison. The resonance frequency in the FEM model using the geometry obtained with the analytical equations was 20.0188 kHz. The resonance length was modified until obtaining a resonance frequency of 20 kHz. The final dimension of  $L_1$  proposed by FEM was 23.8 mm. We used the geometry obtained by FEM calculation for the tests; nevertheless, we should mention that the solution of both methods is in good agreement. The dimensions of the specimen used for room temperature tests are shown in Figure 5.

Experimental results obtained on the gigacycle testing machine showed that the designed specimen vibrated with a natural frequency of 19.985 kHz.

Afterwards, the stress and displacement fields were computed using both FEM mechanical analysis and the analytical equations.

When a specimen is heated, the resonance frequency is affected by geometric changes and by the decrease of the Young modulus produced by the thermal effects. Consequently, the specimen geometry has to be modified in order to keep a natural frequency of vibration at the desired temperature as close as possible to 20 kHz. In other words, the resonance length of the specimens used at high temperature should be shorter than that used for room temperature [10,11].

The temperature distribution on the specimen was determined experimentally using a pyrometer and thermocouples on a specimen whose dimensions were not the definitive ones for the tests at high temperatures.



heterogeneity and stress concentration related to persistent slip bands (PSB) formation.

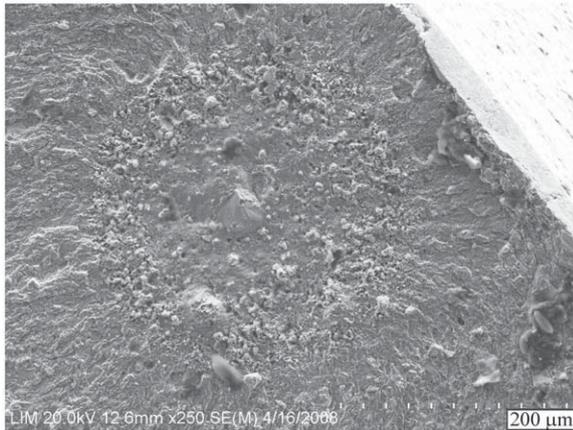
This austenitic steel is face centered cubic structure with a low stacking fault energy. In this case, the cross slip is not favoured and the twinning occurs leading to the delay of PSB formation [12]. A sample test at RT which runs out up to  $10^9$  cycles without failing was cut in the central part. A polished section has been observed in SEM (Figure 10) where we can see mechanical twins.

Electron Back Scattering Diffraction (EBSD) observations were performed on a polished section (perpendicular to the longitudinal axis) taken in the central part of the specimen or taken at the end of the specimen. For the 2 locations, we did not observe phase transformation: the microstructure remains austenitic. But, on the other hand, the volume fraction of twinning has increased between the 2 locations. It is 0.18 on the polished section at the specimen end and 0.25 in the central part specimen (Figure 11 which is the EBSD observation of Figure 1).

It seems, when the sample has not failed at  $10^9$  cycles, the PSB formation is delayed due to the twinning deformation. So, at RT, the deformation mode is either



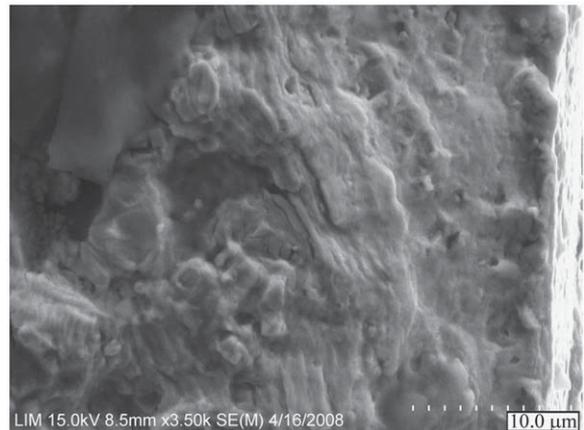
**Figure 11** EBSD observation of a polished section in the central part of the specimen (not failed at  $10^9$  cycles).



**Figure 9** Zoom amplification at inclusion.



**Figure 10** SEM image on a specimen not failed at  $10^9$  cycles.



**Figure 12** SEM image on a failed specimen at RT (a twin is visible at the bottom on the left).

twinning (no PSB formation and not failure) or gliding (with PSB formation and failure).

At high temperature, the stacking fault energy has increased, and the cross slip is always possible with formation of PSB. SEM observations of fracture surface at RT show mechanical twins (Figure 12), whereas at high temperature, no twins are visible.

## 6. Conclusions

The ultrasonic fatigue technique was developed to determine fatigue behavior of austenitic steels in a range of  $10^6$  to  $10^9$  cycles under fully reverse loading at high and room temperature. The most important conclusions obtained are summarized below:

- 1) The dynamic mechanical properties of austenitic steel subjected to high temperatures were identified

based on numerical simulations and the measured resonance frequency of the specimens.

- 2) The design of specimens for high temperature tests using FEM modal analysis, showed good agreement with the experimental tests. When the specimen reached the desired temperature, i.e., room temperature, 600 °C or 700 °C, resonance frequency was close to 20 kHz during the ultrasonic fatigue test.
- 3) At room temperature, more than 20 specimens were tested and none failed above  $2 \times 10^7$  cycles. For the non failed specimens, a deformation by twinning (delaying PSB formation) seems to be present.
- 4) The 600 °C and 700 °C *S-N* curve have shown a continuous decreasing stress-life response.
- 5) In the specimens tested, the crack initiation was observed on the surface (except for one specimen).

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