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# Thermo-mechanical modelling of the aircraft tyre cornering

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## Abstract

Some numerical models are proposed for simulate the aircraft tyre behaviour on the ground in critical situations. Fully coupled thermo-mechanical analysis procedures taking into account finite deformation, dynamics and frictional contact are studied. The proposed models are compared with experimental data such as contact area, lateral and longitudinal stiffnesses, surface temperature evolution due to friction, self-alignment torque, and lateral friction coefficients. A good agreement is observed at low speed (less than 70 km/h). Thus, the model is extrapolated to high speeds and seems able to predict results in cases for which it is not possible to obtain realistic experimental data.

**Keywords:** aircraft tyre, Finite Element Method, thermo-mechanical modelling

## 1. Introduction

Landing and take-off operations are the most critical phases of an aircraft flight. Indeed, the high speed, the violent impact when the aircraft touches the ground, the need of braking efficiently, the ground irregularities, the climatic conditions (crosswind, contaminations, etc.) as well as other parameters can make these ground manoeuvres very complex.

Knowing at each time the forces in the contact patch allows controlling the behaviour at the tire/ground interface. For aeronautical applications, the experiments realized on real tire, at moderated speeds (less than 70 km/h), help to analyse the tire/ground behaviour in this reduced field. However, numerical models based of Finite Element method are fundamental methods to predict the tyre behaviour at higher speed, taking into account the high non-linearities related

to materials, and coupled phenomena (contact with friction, thermal-mechanical couplings, and internal dissipation).

## 2. Modelling of tire aircraft by Finite Element Methods

In this study, finite element analyses were performed in several steps. In the first step, the tyre was modeled under inflation pressure using an axisymmetric 2-D model. A 3-D model was then developed for performing static vertical loading simulations. To study the lateral, torsional and longitudinal stiffnesses of tire, a quasi-static analysis was performed after the static analysis. Recently a fully coupled thermo-mechanical 3-D model was also realized to predict the tire behaviour during landing and taking-off operations: an improved geometry description was introduced, materials behaviours were identified, experimental determination of rubber properties was validated, and Mooney-Rivlin model was used to characterize the hyper-elasticity of rubber. The visco-elasticity properties of rubber were also taken into account.

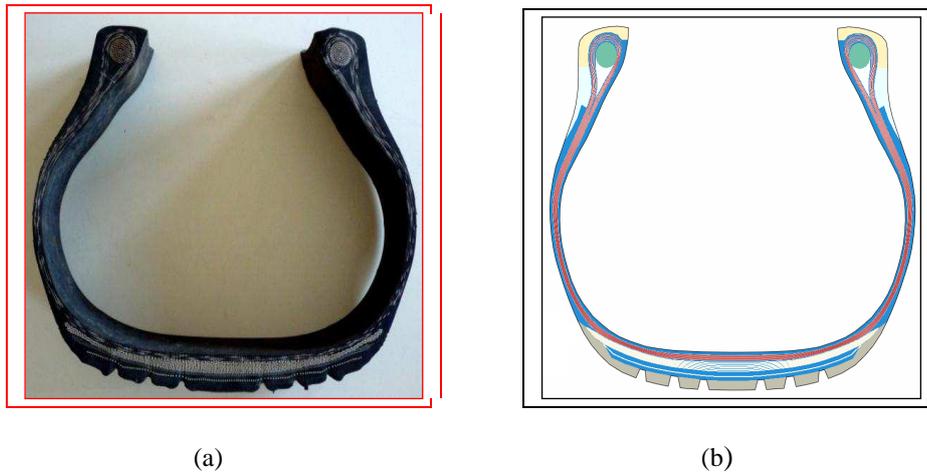


Fig.1. 2-D tire structure: (a) real cross-section of tire (b) 2-D axisymmetric FE model

All the finite element computations were realized using ABAQUS/Standard Version 12-EF on an HP server with Two Xeon quad-core processors cadenced at 3 GHz, using the parallel code version.

The 3-D model was created by rotating the axisymmetric 2-D model around the tire axis. In the 2-D model (fig.1 (b)) CAX4T(H) and CAX3T elements from

ABAQUS element library were selected to represent all the rubber parts of the structure. These are four and three-node coupled temperature-displacement elements without twist and with constant pressure. The reinforcement materials were modeled by SFMAX1 Rebar elements (two-node linear axisymmetric surface element with twist).



Fig.2.3-D fully coupled thermo-mechanical model

### **3. Static simulation**

#### **3.1. Cross section mesh**

In previous papers (Kongo at al. 2012, Kongo et al. 2013), some simulations are presented and analyzed in detail. The main defect of these simulations is the rather low number of elements constituting the mesh. In this part a new refined mesh was realized for the 2-D Element Finite model of tire (Fig.3, second part). This operation was necessary to analyze the mesh sensibility, thus the number of elements was increased in the contact area, and a higher number for circumferential elements was used for the 3-D modelling. Total run time for a complete static analysis with this refined cross section mesh is still little expensive for industrial applications. This new strategy will be adopted in all our future analyses, both for static and dynamic simulations.

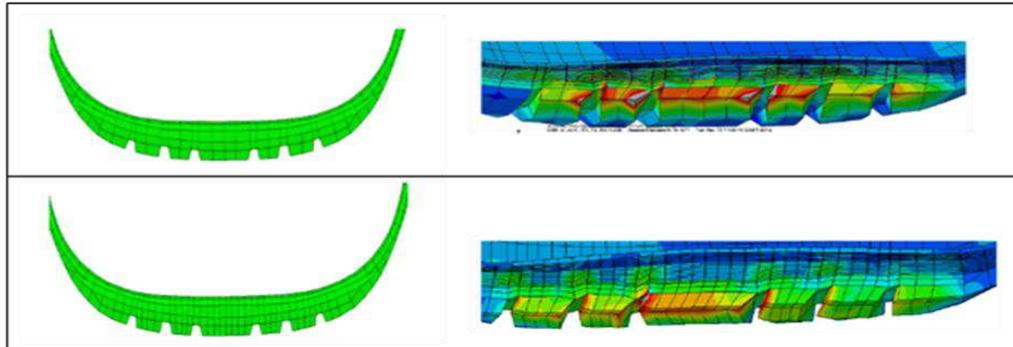


Fig.3. Comparison between old (Kongo et al. 2012) and new refined cross mesh for 2-D FEM model.

At left side: cross section mesh before and after refinement.  
 At right side: cross tread stress before and after refinement.

### 3.2. Contact area

In this study, the inflation analysis of the axisymmetric new model was followed by a 3-D quasi-static analysis of a vertically loaded tyre. Simulations were performed under static vertical loads  $F_z$  ranging from 0.125 to 0.75 % of  $f_0$ . The inflation pressure  $p_0$  was equal to 17.2 bars and the static friction coefficient  $\mu$  was equal to 0.65.  $f_0$  stands for a reference value of the vertical load. The (Fig.4) shows the numerical evolution of the contact area with regard to the vertical loading in comparison with the experimental results.

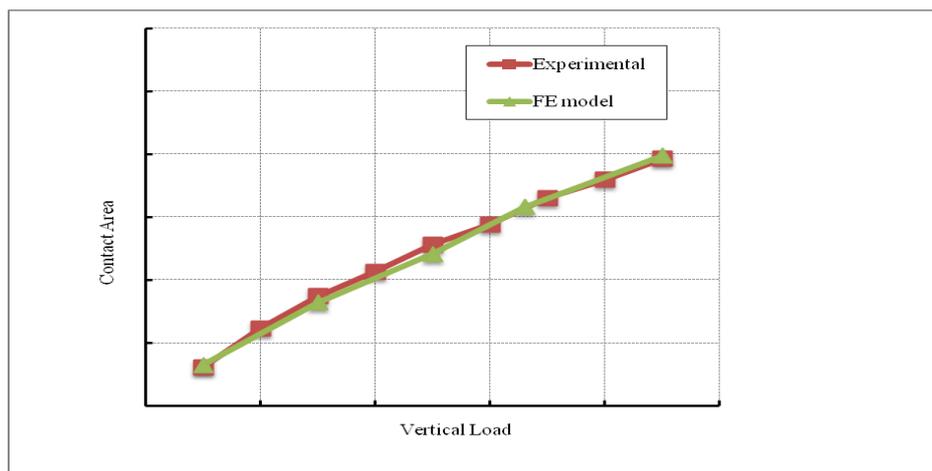
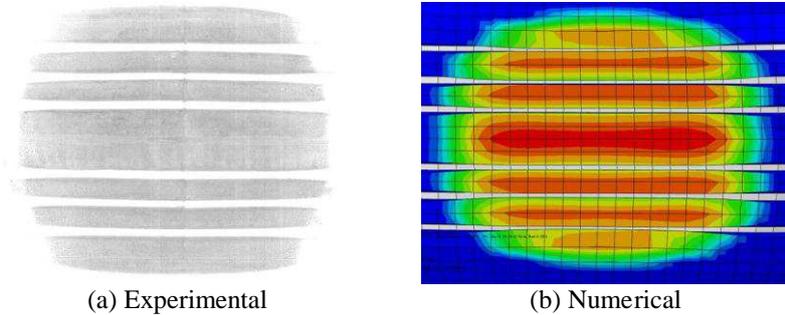


Fig.4. Evolution of contact area in the case of a grooved tire



(a) Experimental (b) Numerical  
 Fig.5. Experimental and Numerical footprints at F3 of loading

As we can see in (Fig.5), a good agreement is observed between numerical and experimental data; footprints were measured in the case of a grooved tire.

#### 4. Dynamic simulations

When we study tire yawed rolling, it is relevant to well estimate the Self Aligning Torque and the ratio, lateral force/ normal force. They respectively express the ability of a tyre to turn back to its steering direction when it is no longer steered and the lateral grip potential of the rubber/ground couple. Indeed, the ratio  $\mu_Y$  (between the lateral force and the vertical one) defines the equilibrium which exists between steering lock action and the trajectory of the vehicle. During  $\mu_Y$  upward phase, there is equilibrium; more the pilot turns the wheel, more  $\beta$  increases. It corresponds to pseudo-sliding in the contact patch up to  $\beta_{max}$  (usually around  $\beta=4$  to  $7^\circ$  for automobile pneumatics, around 15 to  $25^\circ$  for aircraft).

In this study, mechanical and fully coupled thermo-mechanical analyses are developed to predict the friction ratio  $\mu_y$  and the self-aligning torque  $M_z$ . A classical rolling (CR) method based on a Lagrangian formulation was studied, thus from an observer point of view, the tyre mesh (51 circumferential elements) is rotating and moving forward. A variable friction coefficient  $\mu(P,T)$  depending on pressure and temperature was used in the thermo-mechanical model, real viscosity parameters were also introduced in this model. Concerning the mechanical model, a fixed friction coefficient  $\mu_0$  was adopted. These simulations (for both models) require two numerical steps, an acceleration step from zero to a given velocity and a rolling cornering step at a constant given velocity and a given slip angle.

## **4.1. Experimental vs Numerical simulations**

### **4.1.1. Self aligning torque/ lateral friction coefficient**

Mechanical and thermo-mechanical analyses were investigated; these studies concerned a grooved aircraft tire. Fig.6 shows the evolution of the self aligning torque  $M_z$  versus the slip angle. In order to show the influence of the temperature on the lateral friction coefficient  $\mu_y$  and on the self-aligning torque  $M_z$ , purely mechanical, and thermo-mechanical models were performed under the same loading conditions and for the same slip angles. Results were compared with experimental data. In the case of the pure mechanical simulation without any temperature dependence, we can observe that the lateral coefficient  $\mu_y$  (Fig.7) reaches a plateau equal to  $\mu_0$  with respect to the Coulomb's friction law.  $\beta_{max}$ , the angle for which the self-aligning torque is maximum, seems to be in good agreement with experimental data. The self-aligning torque  $M_z$  increases up to  $\beta_{max}$  at which the full grip potential is reached. The self-aligning torque  $M_z$  decreases when  $\beta > \beta_{max}$ , and tends to negative values.

In the case of a thermo-mechanical approach, the dependence of the lateral friction coefficient on the temperature (Fig.7) resulted in a decrease in  $\mu_y$  beyond the  $\beta_{max}$  value. The self-aligning torque  $M_z$  (Fig.6) falls quickly toward null values.

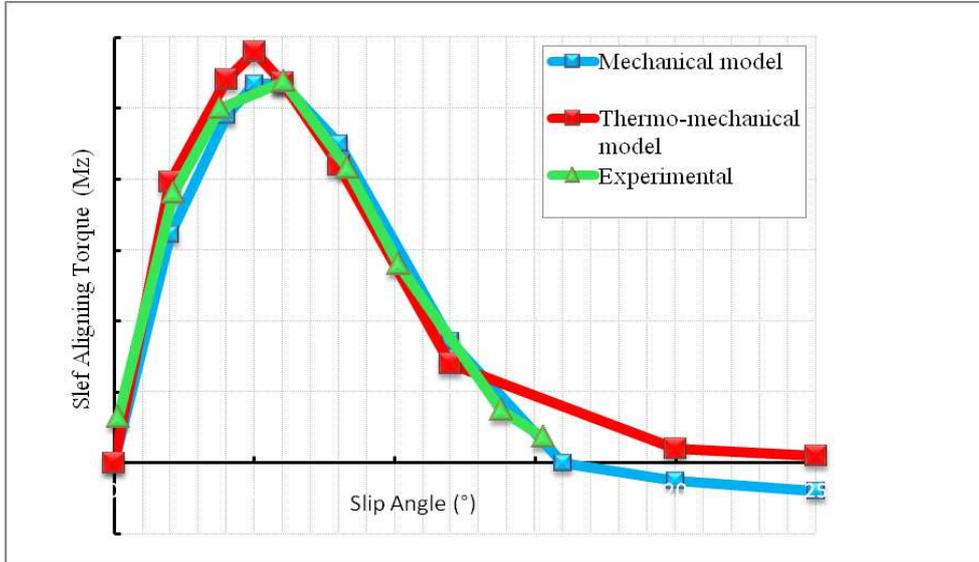


Fig. 6. Comparison between experimental data, mechanical and thermo-mechanical models  
Self aligning torque  $M_z$ ,  
Under given loading conditions (vertical load, pressure and velocity)

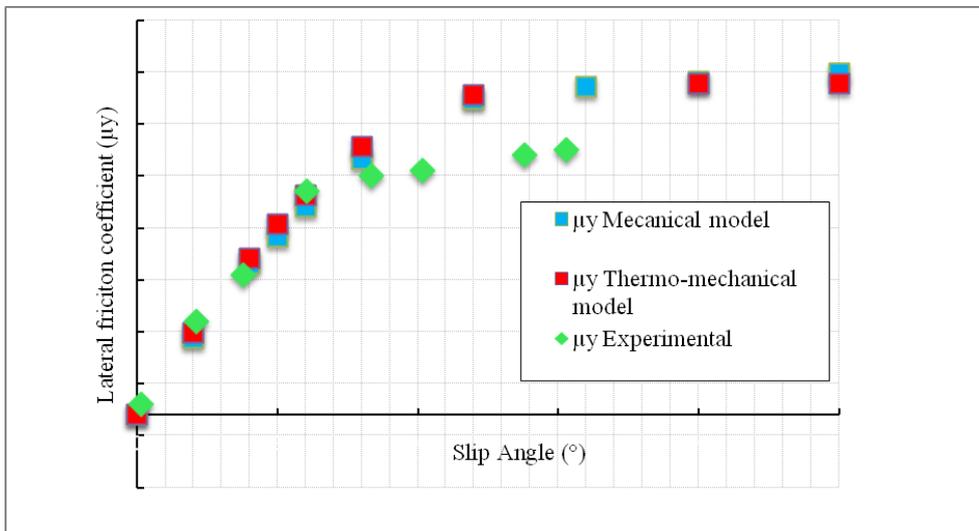


Fig. 7. Comparison between experimental data, mechanical and thermo-mechanical models  
Lateral friction coefficient  $\mu_y$ ,  
under given loading conditions (vertical load, pressure and velocity)

#### 4.1.3. Temperature evolution of an aircraft tire thermo-mechanical approach

Cornering rolling is obtained by imposing longitudinal and lateral velocities (related to a slip angle  $\beta$ ). Figure 8 shows the temperature distribution at the tyre tread.

After a few seconds of rolling in cornering, an increase of the temperature occurs on tyre tread and especially on groove sides; a warming zone is observed on the entire contact zone. However a slight diffusion of temperature is observed after a short period of rolling. We note also the role of a thermal barrier presented by the reinforced zones. Recently different experimental and numerical tests were performed; their purpose was to verify the behaviour of rubber during cornering rolling. These studies confirmed the necessity of taking into account the frictional heating of rubber and also the heating produced by energy dissipation (visco-elasticity properties).

Fig.9 shows a rolling cornering case of an aircraft tyre at a low velocity and for a small slip angle; a good agreement seems to be obtained between the thermo-mechanical model and experimental data.

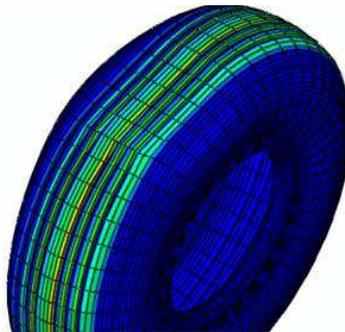


Fig.8. Temperature gradient for a cornering simulation at given velocity and slip angle

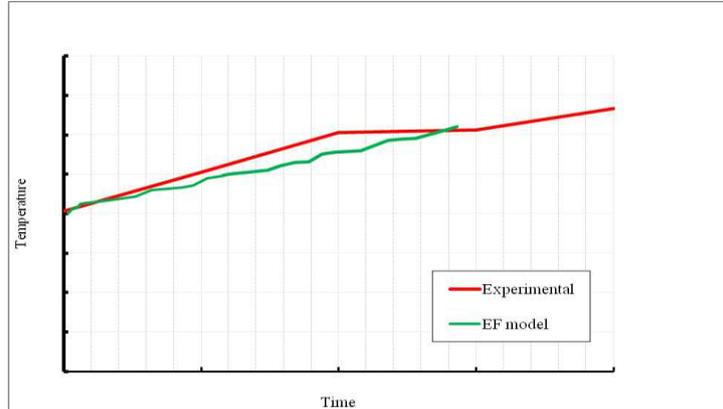


Fig.9. Temperature evolution: comparison between thermomechanical FEM and Experimental data for a cornering simulation of an aircraft tyre for given velocity and slip angle

## 6. Conclusion

In this paper, we have proposed some numerical models to simulate the aircraft tyre behaviour in cornering. Fully coupled thermo-mechanical analysis procedures taking into account finite deformation were studied. The proposed models have been compared with experimental data such as contact area, lateral and longitudinal stiffnesses, surface temperature evolution due to friction, self-alignment torque, and lateral friction coefficients. A good agreement was observed at low speeds (less than 70 km/h) and the model seems to be able to predict results at high speeds, in which case it is not possible to obtain realistic experimental data. In the future, we intend to improve the models introducing more complex constitutive equations coupling thermo-mechanical effects, particularly taking into account viscosity, internal dissipation, etc. Other kind of experimental devices will be proposed in order to confirm the validity's domain of our models in various critical situations.

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