Euromech 516 - Nonsmooth contact and impact laws in mechanics

# Performance of time-marching techniques dedicated to nonsmooth systems A nonlinear modal analysis approach

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## Introduction

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### 2 Modal analysis

Linear framework Nonlinear framework

### 3 Non-smooth application

Unilateral contact without friction Invariant periodic solution Linear rod with contact Modal motions Invariant manifolds

### 4 Time-stepping techniques

Central differences – Implicit contact  $\theta$ -method Comparison



Future work

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Context				

## Framework

- Flexible structures
  - finite-element models, multibody
  - possibly with many degrees-of-freedom
- Dynamics and vibration
  - transient
  - steady-state periodic, quasi-periodic regimes
- Interface nonlinearities contact, friction

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# Framework

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## Nonlinear modal analysis

- nonlinear modes: periodic motion of an automonous system, *invariant manifold* on which oscillations take place
- extension of linear modal analysis, i.e. provides an essential signature of the dynamics
- numerical framework
- useful for engineering applications

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Challen	ges			

- Large scale engineering systems (e.g. FE models, multibody) call for (efficient) numerical methods adapted to generic nonlinearities (i.e. not only polynomial)
  - Space semi-discretization
  - Time stepping approaches (incl. shooting for BVP)
  - Galerkin methods in time (e.g. harmonic balance, finite-element, etc.)
  - Space-time discretization

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- Non-smooth systems (e.g. featuring interface interactions) require specific treatments and methods
  - Contact interface laws
  - Optimization strategies

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- Non-smooth systems (e.g. featuring interface interactions) require specific treatments and methods
  - Contact interface laws
  - Optimization strategies
- Beyond nonlinear modal analysis, the dynamic regimes of such systems have to be fully qualified including their stability
  - equilibria
  - periodic or quasi-periodic solutions
  - chaos

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## Validation

- Assessment of performances of time-stepping techniques dedicated to flexible mechanical systems featuring unilateral contact conditions
- How are such techniques capable of finding periodic orbits of automonous (and conservative) systems with non-smooth nonlinearities?
- · Preservation of invariant quantities

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# Time-integration algorithms

# Validation

- Assessment of performances of time-stepping techniques dedicated to flexible mechanical systems featuring unilateral contact conditions
- How are such techniques capable of finding periodic orbits of automonous (and conservative) systems with non-smooth nonlinearities?
- Preservation of invariant quantities

## Extension

- Description of the dynamics around these invariant quantities
- Stability analysis

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Linear	framework	< (1)		

• Linear 1<sup>st</sup>-order ODE with constant coefficients

 $\dot{z} + Dz = 0$ 

Assumed solution

$$\mathbf{z}(t) = \mathbf{Z} \mathbf{e}^{\lambda t} \quad \Rightarrow \quad (\mathbf{D} - \lambda \mathbf{I})\mathbf{Z} = \mathbf{0}$$
 (2)

(1)

- generally complex eigen elements  $(\lambda_i, Z_i)$
- uncouped equations = invariant subsets
- ▶ superposition principle  $\equiv$  eigen modes span the configuration space
- Invariant set formulation
  - Master coordinates  $(x_1, y_1) = (u_1, v_1)$
  - Functional dependence

$$x_i(t) = a_{1i}u_1(t) + a_{2i}v_1(t)$$
  

$$y_i(t) = b_{1i}u_1(t) + b_{2i}v_1(t)$$
  

$$i = 1, 2, \dots, N$$
(3)

Substitution into equation of motion (1) to get  $a_{ik}$ 's and  $b_{ik}$ 's

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Linear	framework	(2)		

• Equations of motion in real modal space

$$\begin{pmatrix} \dot{u}_k \\ \dot{v}_k \end{pmatrix} = \begin{bmatrix} -\xi_k \omega_k & \omega_k \sqrt{1 - \xi_k^2} \\ \omega_k \sqrt{1 - \xi_k^2} & -\xi_k \omega_k \end{bmatrix} \begin{pmatrix} u_k \\ v_k \end{pmatrix} \qquad k = 1, 2, \dots, N$$
 (4)

- Reconstruction: superposition principle
- System with two degrees-of-freedom: first linear mode

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# Nonlinear framework (1)

- Theoretical tools
  - Central manifold theorem
  - Lyapunov theorem
  - Normal form theorem
- Assumed solution

$$\mathbf{z}(t) = \sum_{n} \mathbf{Z}_{n} \mathbf{e}^{n\lambda t} \quad \Rightarrow \quad \mathbf{F}(\lambda, \mathbf{Z}) = \mathbf{0}$$

$$\mathsf{Z} = (\mathsf{Z}_1, \mathsf{Z}_2, \dots, \mathsf{Z}_n)$$

- continuation technique:  $Z(\lambda)$
- geometry of the invariant manifold
- Invariant set formulation
  - Master coordinates  $(x_1, y_1) = (u, v)$
  - (Nonlinear) functional dependence

$$x_i = X_i(u, v)$$
  

$$y_i = Y_i(u, v)$$
  

$$i = 1, 2, \dots, N$$

(5)

(6)

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# Nonlinear framework (2)

Time differentiation

$$\dot{x}_{i} = \frac{\partial X_{i}}{\partial u} \dot{u} + \frac{\partial X_{i}}{\partial v} \dot{v}$$

$$\dot{y}_{i} = \frac{\partial Y_{i}}{\partial u} \dot{u} + \frac{\partial Y_{i}}{\partial v} \dot{v}$$
(7)
(7)

Substitution into equation of motion (1) — i=1,2,...,N

$$Y_{i} = \frac{\partial X_{i}}{\partial u} v + \frac{\partial X_{i}}{\partial v} f_{1}(u, X_{1}, \dots, X_{n}, v, Y_{2}, \dots, Y_{N})$$
  
$$f_{i}(u, X_{1}, \dots, X_{n}, v, y_{2}, \dots, Y_{N}) = \frac{\partial Y_{i}}{\partial u} v + \frac{\partial Y_{i}}{\partial v} f_{1}(u, X_{1}, \dots, X_{n}, v, Y_{2}, \dots, Y_{N})$$
(8)

- Power series expansions, nonlinear Galerkin technique...
- Reduced-dynamics on the invariant manifold

$$\dot{u} = v 
\dot{v} = f_1(u, X_1, \dots, X_n, v, Y_2, \dots, Y_N)$$
(9)

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Nonline	ear framew	vork (3)		

- energy-dependent frequency
- perturbation techniques
- stability analysis

## Cubic nonlinearity: first nonlinear mode

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# Unilateral contact without friction

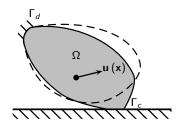
- Autonomous and conservative system
  - contact interface :  $\Gamma_c$
  - no external forcing
  - no friction, no structural damping
- Gap function on  $\Gamma_c$

$$g(\mathbf{u}) = \mathbf{u}(\mathbf{x}) \cdot \mathbf{n} - g_0(\mathbf{x})$$

- n: outward normal
- g<sub>0</sub>: initial gap
- Contact conditions  $\forall \mathbf{x} \in \Gamma_c$

$$\tau_N \ge 0$$
,  $g(\mathbf{u}) \le 0$ ,  $g(\mathbf{u}) \cdot \tau_N = 0$ 

•  $\tau_N = \sigma \cdot \mathbf{n}$ : contact pressure



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# Invariant periodic solution

• Boundary value problem (Dirichlet and Signorini conditions)

$$\begin{split} \rho \ddot{\mathbf{u}} - \operatorname{div} \sigma(\mathbf{u}) &= 0 \quad \text{on } \Omega \times \mathbb{R}^+_* \\ \mathbf{u} &= 0 \quad \text{on } \Gamma_d \times \mathbb{R}^+_* \\ \mathbf{g}(\mathbf{u}) &\leq 0, \ \tau_N &\leq 0 \ \text{and} \ \mathbf{g}(\mathbf{u}) \cdot \tau_N &= 0 \quad \text{on } \Gamma_c \times \mathbb{R}^+_* \end{split}$$

• Frequency-domain displacement field

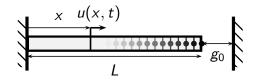
$$\mathbf{u}(t) = \sum_{n \in \mathbb{Z}} \hat{\mathbf{u}}_n e^{jn\omega t}$$
 with  $\hat{\mathbf{u}}_n = \frac{1}{T} \int_T \mathbf{u}(t) e^{-jn\omega t} dt$ 

• Final formulation Find  $\{\omega, \hat{\mathbf{u}}\}$ , with  $\hat{\mathbf{u}} = \{\hat{\mathbf{u}}_n, n \in \mathbb{Z}\}$  such as:

$$\begin{aligned} -\operatorname{div} \sigma(\hat{\mathbf{u}}_n) &= (n\omega)^2 \hat{\mathbf{u}}_n & \text{ on } \Omega \times \mathbb{Z} \\ \hat{\mathbf{u}}_n &= 0 & \text{ on } \Gamma_d \times \mathbb{Z} \\ \mathbf{g}(\hat{\mathbf{u}}) &\leq 0, \ \tau_N &\leq 0 \text{ and } \mathbf{g}(\hat{\mathbf{u}}) \cdot \tau_N &= 0 & \text{ on } \Gamma_c \times [0, T] \end{aligned}$$

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# Linear rod with contact



Local equation

$$\rho \frac{\partial^2 u}{\partial t^2}(x,t) - E \frac{\partial^2 u}{\partial x^2}(x,t) = 0, \quad x \in ]0, L[$$

Boundary condition

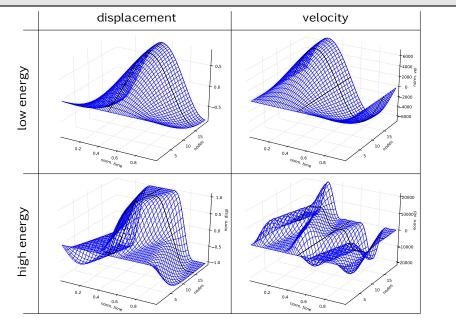
u(0,t)=0

• Unilateral contact conditions

$$u(L,t) - g_0 \leq 0, \qquad \sigma_{xx}(L,t) = EA \frac{\partial u}{\partial x}(L,t) \geq 0, \qquad (u(L,t) - g_0) \cdot \sigma_{xx}(L,t) = 0$$



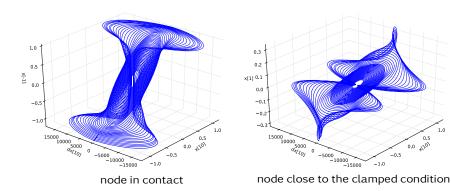
# Modal motions



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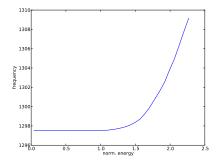
# Invariant manifolds

- · Cross section of the invariant manifold
- From low to high energy
- Energy–frequency dependence



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Invariant manifolds						

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Invaria	nt manifol	ds		

- Cross section of the invariant manifold
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- First linear mode

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# Central differences – Implicit contact

• Problem Find  $\mathbf{u}_{n+1}$  and  $\lambda_{n+1}$  such that:

$$\mathbf{M}\left(\frac{\mathbf{u}_{n+1}-2\mathbf{u}_n+\mathbf{u}_{n-1}}{\Delta t^2}\right)+\mathbf{K}\mathbf{u}_n-\mathbf{G}^{\mathrm{T}}\boldsymbol{\lambda}_{n+1}=0$$

with  $\mathbf{G}\mathbf{u}_{n+1} \ge 0$ ,  $\lambda_{n+1} \ge 0$ , and  $\lambda_{n+1}^{\mathrm{T}} \mathbf{G}\mathbf{u}_{n+1} = 0$ 

- Algorithm
  - 1. Prediction of  $\mathbf{u}_{n+1}$  by ignoring contact forces  $\lambda_{n+1}$
  - 2. If  $(\mathbf{Gu}_{n+1})_{(i)} < 0$  then  $\mathbf{u}_{n+1}$  is modified so that  $(\mathbf{Gu}_{n+1})_{(i)} = 0$
- Properties
  - conditionally stable
  - proof of convergence in 1D
  - possibilities of lumped mass
  - energy losses affect the solution after few impacts

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$\theta$ -method (1)				

Integral formulation

$$\int_{t}^{t+\Delta t} \mathbf{M} d\dot{\mathbf{u}} + \int_{t}^{t+\Delta t} \mathbf{K} \mathbf{u} dt - \int_{t}^{t+\Delta t} \lambda dt = 0$$

Assumptions

$$\begin{aligned} & \mathbf{u}_{t+\Delta t} - \mathbf{u}_t = \Delta t \Big( (1-\theta) \dot{\mathbf{u}}_t + \theta \dot{\mathbf{u}}_{t+\Delta t} \Big) \text{ with } \mathbf{0} \leq \theta \leq 1 \\ & \int_t^{t+\Delta t} \mathbf{M} d \dot{\mathbf{u}} = \mathbf{M} (\dot{\mathbf{u}}_{t+\Delta t} - \dot{\mathbf{u}}_t) \\ & \int_t^{t+\Delta t} \mathbf{K} \mathbf{u} dt = \Delta t \Big( (1-\xi) \mathbf{K} \mathbf{u}_t + \xi \mathbf{K} \mathbf{u}_{t+\Delta t} \Big) \text{ with } \mathbf{0} \leq \xi \leq 1 \\ & \int_t^{t+\Delta t} \lambda dt = \Delta t \lambda_{t+\Delta t} \end{aligned}$$

• Problem Find  $\mathbf{u}_{n+1}$  and  $\lambda_{n+1}$  such that:

$$\left(\frac{\mathbf{M}}{\theta\Delta t^{2}}+\xi\mathbf{K}\right)\mathbf{u}_{t+\Delta t}+\frac{\mathbf{M}}{\theta\Delta t}\mathbf{v}_{t}+\left(\frac{\mathbf{M}}{\theta\Delta t^{2}}-(1-\xi)\mathbf{K}\right)\mathbf{u}_{t}-\mathbf{G}^{\mathrm{T}}\boldsymbol{\lambda}_{t+\Delta t}=0$$

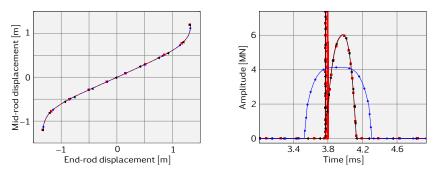
with  $\mathbf{G}\mathbf{u}_{t+\Delta t} \ge 0$ ,  $\lambda_{t+\Delta t} \ge 0$ , and  $\lambda_{t+\Delta t}^{\mathrm{T}} \mathbf{G}\mathbf{u}_{t+\Delta t} = 0$ 

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$\theta$ -meth	nod (2)			

- Algorithm
  - **1**. Steppest descent in  $\mathbf{u}_{n+1}$
  - 2. Uzawa in  $\lambda_{n+1}$
- Properties
  - Best parameters:  $\theta = \xi = 0.5$
  - Inconditonally stable in a linear framework
  - Energy conservation

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Compar	rison			

- Initial conditions on the calculated invariant manifold
- Time-stepping integration
- Comparison of the three numerical approaches: implicit , explicit —, and NLM —



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• Full unilateral and periodic formulation, ie, no regularization

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- Full unilateral and periodic formulation, ie, no regularization
- Strategy dedicated to flexible structures

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Summa	ary			

- Full unilateral and periodic formulation, ie, no regularization
- Strategy dedicated to flexible structures
- Numerical proof of the existence of these periodic motions
  - Agreement between three different numerical techniques
  - Need for mathematical proof (already existing?)

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Summa	arv			

- Full unilateral and periodic formulation, *ie*, *no* regularization
- Strategy dedicated to flexible structures
- Numerical proof of the existence of these periodic motions
  - Agreement between three different numerical techniques
  - Need for mathematical proof (already existing?)
- Capture the well-known stiffening effects observed for systems undergoing unilateral contact conditions

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• Contact interface involving more nodes

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- Contact interface involving more nodes
- Study of higher modes of vibration
  - Comparison with conservation of energy
  - Are specific modes more affected?

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- Several contacting flexible structures
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  - More complicated formulation

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  - More complicated formulation
- Development of other existing time-marching techniques

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  - More complicated formulation
- Development of other existing time-marching techniques
- Finite-element techniques in time vs. Fourier series
  - Improved description of the contact efforts
  - More DoF

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- Friction?