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Self-adjusting, Isostatic Exoskeleton for The Human Knee Joint

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Abstract—A knee-joint exoskeleton design that can apply programmable torques to the articulation and that self-adjusts to its physiological movements is described. Self-adjustment means that the articular torque is automatically produced around the rotational axis of the joint. The requirements are first discussed and the conditions under which the system tracks the spatial relative movements of the limbs are given. If these conditions are met, the torque applied to the joint takes into account the possible relative movements of the limbs without introducing constraints. A prototype was built to demonstrate the applicability of these principles and preliminary tests were carried out to validate the design.

I. INTRODUCTION

Electro-mechanical systems used in functional rehabilitation can increasingly contribute to modern therapeutic techniques. Such systems can be employed to assist the work of therapists, to increase the mobility of patients in daily activities, or to support various rehabilitation protocols. They can also provide quantifiable measurements to help the diagnosis and monitor therapeutic process.

According to M. Hillman [8], the first active orthoses appeared in the sixties. They were used to move the paralyzed limbs of a person and support gradual rehabilitation based on mechanically assisted exercises. It is only in the late nineties, however, that such systems became truly practical.

A. Related Work

Two different approaches in the design of mechanical devices for functional rehabilitation therapy or for diagnosis have emerged. The first approach involves the use of robotic arms to guide parts of the patient's anatomy along predefined trajectories. The interaction between the driven member and the robotic arm often occurs near the extremity of the respective kinematic chains. Many systems dedicated to the rehabilitation of upper limb functions using this approach have been designed, such is the case of the MIT-Manus [9], of the MIME system [2], of the 'Braccio di Ferro' [4], or of the Nerebot system employing free-space wires [12].

Such an "external approach" yields systems that are easy to implement but can be problematic in terms of efficiency and security. The device may force the articulation of the subject to move in arbitrary directions, which can cause, for instance, hyper-extension. In addition, these devices engage

several joints simultaneously and do not allow to exercise individual joints. This limitation makes difficult diagnosis, treatment monitoring and joint-specific protocols.

The second approach involves engaging the joints individually. This function can be achieved by the coordinated control of multi-contact systems, often resulting in exoskeleton mechanical structures. Several exoskeleton systems for upper limb rehabilitation have been proposed, including the Pneu-WREX [15], the Armin [13], the Dampace [18], or the Cadenas-7 [14].

These systems, however, typically come with an important limitation due to the misalignment between the patient joints and the active mechanical joints, which combined with hyperstaticity, result in a number of deleterious effects. The transmission of forces and torques is also difficult to manage since, generally speaking, the number of actuators never exceeds the mobility of anatomical joints. As a result, it becomes impossible to gain complete control over the transmitted forces and torques.

To address this problem, authors have proposed to employ mechanisms that transmit pure torques by means of properly designed mechanisms [6], [17]. For instance, a three-slider-joint followed by three actuated rotational joints can provide such function. The transmission of pure torques to engage individual anatomical joint is the simplest solution that enables exoskeleton systems to operate safely.

B. A New Design

This article describes a novel active orthosis device. Its design is based on the determination of the number of passive degrees of freedom that the mechanism should have in order to become isostatically loaded when acting against a joint. A quasi-static analysis is employed to ensure a proper force transmission to the limbs in order to overcome muscular disturbances.

We built an active orthosis for the knee that provides a flexion-extension torque in order to exercise the joint, or to assist its movements. The device can also monitor the joint kinematics during movement through the estimation of the helical, instantaneous displacement axis [10], [20].

II. KINEMATICS OF THE KNEE

The kinematics of human joints is complex, and the knee joint is no exception. The complicated relative movements of the limbs depend on the geometry of the joint surfaces, on the load, and on the properties of the ligaments, capsules, and menisci. Within subjects, the kinematics can also vary according to a variety of conditions [11] [19].

A description of the kinematics of the knee joint based on a six-degrees-of-freedom model composed from three screw joints was proposed by [5], see Fig. 1. Its use is nowadays recommended by the International Society of Biomechanics [21]. It comprises:

- 1) A rotational axis for the femur that passes through the centers of both femoral condyles. The joint displacements along this axis are named 'flexion-extension' and 'medial-lateral', respectively.
- 2) The axis of the tibia. The joint displacements along this axis is named 'internal rotation' and 'proximal-distal displacement'.
- 3) The varus axis that is orthogonal to the first two axes. The joint displacements along this axis are named 'varus-valgus rotation' and 'anterior-posterior displacement'.

Figures 2 and 3 show the results of previous kinematics measurements of the knee. These measurements were performed using a passive 6 DOFs electromechanical goniometer. Further details can be found in [3].

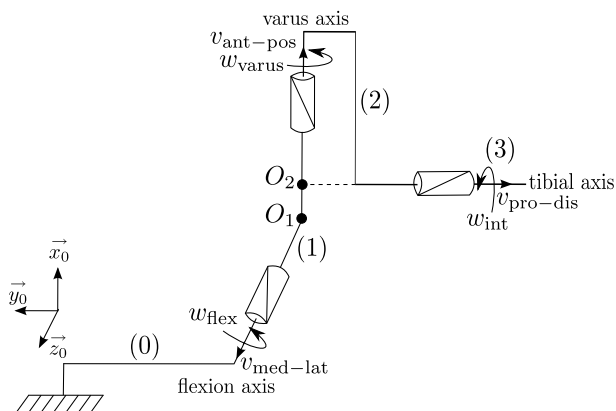


Fig. 1. Model of the knee joint by 6 degrees of freedom [5].

As we can see from Figure 3, the knee joint has a variable instantaneous axis during movement. Its location varies up to 2 centimeters during flexion due to anterior-posterior displacements. The varus-valgus and internal rotation angles vary up to 5° and 10° . Notice also that the internal rotation angle can become active when the knee is in full flexion, and can vary up to 30° .

These results highlight the fact that the knee is a spatial joint. For orthosis design, it should not be modeled as a hinge joint in flexion-extension. In an active or passive orthosis design, passive joints are crucial to free up the constraints due to the misalignment between the instantaneous rotation axes of the knee and that of the orthosis.

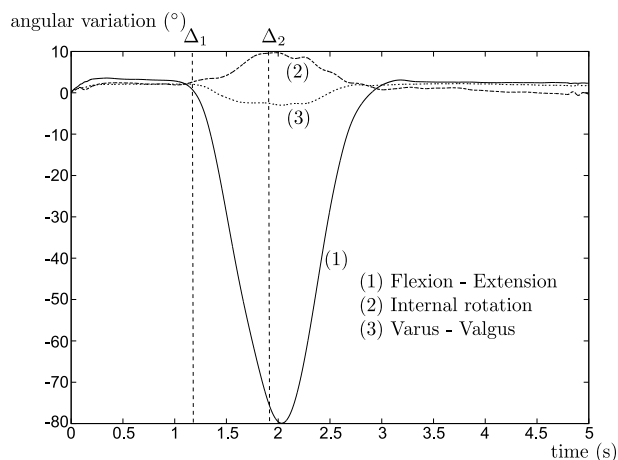


Fig. 2. The angular variation of the knee joint during flexion-extension movement. Calculations are done using the model of [5] illustrated in the figure 1.

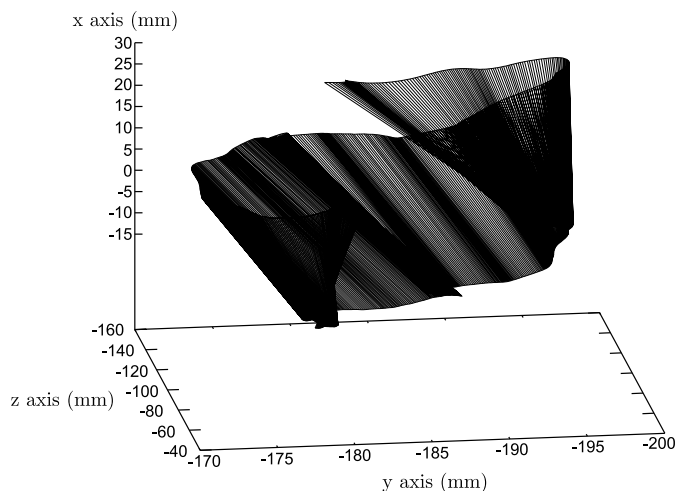


Fig. 3. The instantaneous helical axis of the knee joint measured by an electro-goniometer [3]. Calculations are done between the moments Δ_1 and Δ_2 as shown in the figure 2.

III. DESIGN METHOD

When the mechanism is fixed to the two limbs, it forms a closed kinematic chain with the anatomical joint. As will be demonstrated in section IV, if the device is properly designed, it mainly applies a torque around the knee joint. In these conditions, if the limbs fixations are properly designed, the muscle movements are limited and can be neglected in a first step. Then the mobility of the entire closed chain can be determined using

$$m = d(b - 1) - \sum_{i=1, n} u_i \quad (1)$$

where, m is degree of mobility or degree of freedom of the mechanism, n is number of elementary joints in the mechanism, d is the dimension of the space in which the mechanism operates ($d = 3, 6$), b is total number of bodies in the mechanism, including 1 fixed body and $b - 1$ moving

bodies, and u_i is number of elementary constraints in the joint i .

When $m > 0$, the mechanism is isostatic, the number of actuations required is equal to m . When $m \leq 0$, the mechanism becomes over-constrained. There is theoretically no possible movement of the mechanism, except in cases where a singularity appears, for example when the axis of rotation of the mechanism aligns with the articulation of the subject. This phenomenon can, in the best case, cause discomfort due to sliding movements of the attachments, and in the worst case, long term injury. If the torque provided by the system is significant, interaction with the device can cause pain. This problem has already been noticed by several authors [16], [17], [7].

Design rules for active functional rehabilitation devices can be established for an anatomical joint with k mobilities, which satisfy the isostatic condition, see Table I.

TABLE I
DETERMINATION OF THE DOF'S AND ACTUATORS.

	Planar joint	Spatial joint
Total freedoms (m)	$m \geq 3$	$m \geq 6$
Actuators (a)	$a = k + m - 3$	$a = k + m - 6$
Passive DOF's	$3 - k$	$6 - k$

IV. DESCRIPTION OF THE KNEE EXOSKELETON

The CAD model of our first knee exoskeleton prototype is shown in Fig. 4. The device comprises a total of 7 links forming a 6 degrees-of-freedom kinematic chain. The first three rotational joints are intersecting at point O . They are connected to a sliding joint that allows the mechanism to adapt to different limb sizes. The last two rotational joints are used to compensate for the offset between axes 2 and 3 and the axis of the tibia and the varus–valgus axis.

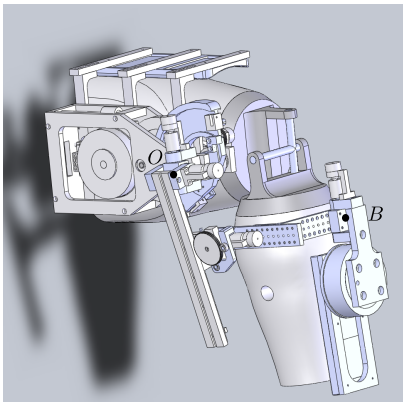


Fig. 4. The computer model of the knee exoskeleton

The device can provide a torque of 40 Nm through the use of a brushless motor and a two-stages, backdrivable, 100:1 transmission. It comprises a high-speed friction drive followed by a low speed cable-drive. Each stage gives a torque gain equal to 10. The friction drive operates through

direct contact between the motor shaft and a disk drive, see Fig. 5. Contact is regulated by a compression spring pushing on rollers so that slip does not occur. This system allows for adjusting the slipping threshold, and thus provides an extra level of safety.

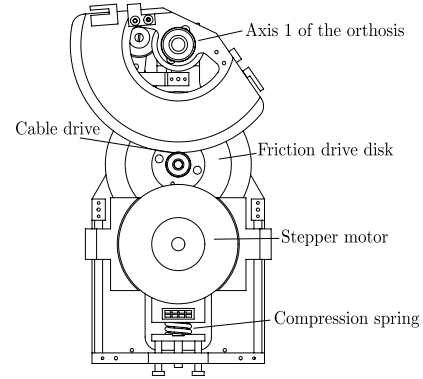


Fig. 5. View of the actuator unit

Each joint is equipped with a precision potentiometer. A force-torque sensor is inserted in the load path at the extremity of the mechanical chain for the control of force. Some links can be manually adjusted to optimize performance and ensure that the mechanical chain remains far from its singularities. Figure 6 shows the frame assignment of the system using the Denavit-Hartenberg convention. In the following sections, in order to facilitate analysis, all calculations are done in the reference frame R_2 .

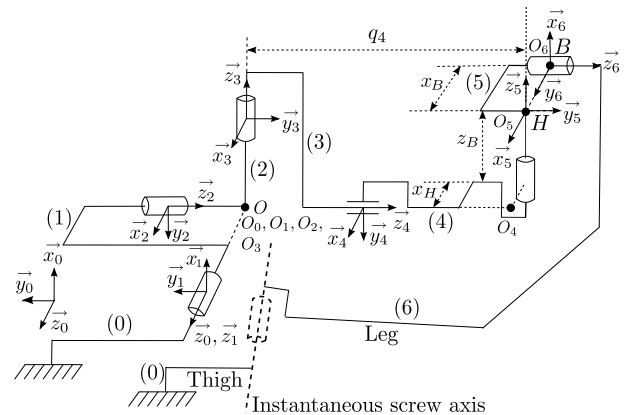


Fig. 6. System frame assignment using the Denavit-Hartenberg notation.

A. Singularities

To determine the singular configurations, we write the Jacobian matrix at point O in the frame $R_2, (O, x_2, y_2, z_2)$ so that the matrix becomes block-triangular,

$$\begin{aligned} \mathbf{J}(\mathbf{O})_{R_2} &= \begin{pmatrix} \mathbf{J}w_1 & \mathbf{J}w_2 \\ \mathbf{J}v_1 & \mathbf{J}v_2 \end{pmatrix} \\ &= \begin{pmatrix} C(q_2) & 0 & 0 & 0 & 0 & -S(q_{35}) \\ -S(q_2) & 0 & -1 & 0 & -1 & 0 \\ 0 & 1 & 0 & 0 & 0 & C(q_{35}) \\ 0 & 0 & 0 & -S(q_3) & Jv_{15} & Jv_{16} \\ 0 & 0 & 0 & 0 & Jv_{25} & Jv_{26} \\ 0 & 0 & 0 & C(q_3) & Jv_{35} & Jv_{36} \end{pmatrix} \end{aligned}$$

where $q_{35} = q_3 + q_5$. The determinant of the matrix is

$$\begin{aligned} \det(\mathbf{J}(\mathbf{O})_{R_2}) &= \det|\mathbf{J}w_1| \det|\mathbf{J}v_2| \\ &= -q_4 C(q_2) (-q_4 S(q_5) + x_H C(q_5) + x_B). \end{aligned}$$

Singularities appear when $\det(\mathbf{J}(\mathbf{O})_{R_2}) = 0$, i.e. when,

$$q_4 = 0, \quad q_2 = \pm\pi/2, \quad -q_4 S(q_5) + x_H C(q_5) + x_B = 0.$$

If the mechanism is properly attached to the limbs (i.e. axis z_1 close to the flexion axis after visual inspection) and if it is correctly tuned (i.e. using the adjustments of links (0),(4) and (5)). See Fig. 4 and Fig. 6), the principal movements are flexion and extension. The angle q_2 , which in this case roughly corresponds to the internal rotation along the axis of the tibia, is then smaller than $\pm\pi/2$ and the length q_4 is greater than zero. The orthosis remains far from its singular positions during, except when q_5 is positive and much greater than zero.

B. Force transmission

The force transmission analysis can be done by using the quasi-static model,

$$\mathbf{T}_j = \mathbf{J}^\top(\mathbf{B})\mathbf{T}_e(\mathbf{B}) \quad (2)$$

where \mathbf{T}_j is the vector of joint torques and forces, which in our case is $\mathbf{T}_j = (C, 0, 0, 0, 0, 0)^\top$, where C is the motor torque. The matrix $\mathbf{J}^\top(\mathbf{B})$ is the transpose Jacobian of the system written at the point B and $\mathbf{T}_e(\mathbf{B})$ is the vector of external torques and forces applied to the system at point B , which is defined by,

$$\mathbf{T}_e(\mathbf{B}) = \begin{pmatrix} \mathbf{M}_{\text{ext}} \\ \mathbf{F}_{\text{ext}} \end{pmatrix} = (M_x, M_y, M_z, F_x, F_y, F_z)^\top. \quad (3)$$

The knee movement mainly influences the movement of the first axis of rotation. The other mechanical joints of the orthosis only play a role of alignment between the first rotation axis and the instantaneous axis of the knee. As a result, they are close to zero. So the terms q_2, q_3, q_5, q_6 may be assumed to be close to zero. The matrix $\mathbf{J}^\top(\mathbf{B})$, in the ideal case, becomes:

$$\mathbf{J}^\top(\mathbf{B})_{R_2} = \begin{pmatrix} 1 & 0 & 0 & 0 & -q_4 & -z_B \\ 0 & 0 & 1 & z_B & -x_H - x_B & 0 \\ 0 & -1 & 0 & -q_4 & 0 & -x_H - x_B \\ 0 & 0 & 0 & 0 & 0 & 1 \\ 0 & -1 & 0 & 0 & 0 & -x_B \\ 0 & 0 & 1 & 0 & 0 & 0 \end{pmatrix}. \quad (4)$$

From (2), (3), and (4), we obtain the following system of equations,

$$\begin{aligned} C &= M_x - q_4 F_y - z_B F_z, \\ 0 &= M_z + z_B F_x - (x_H + x_B) F_y, \\ 0 &= -M_y - q_4 F_x - (x_H + x_B) F_z, \\ 0 &= F_z, \\ 0 &= -M_y - x_B F_z, \\ 0 &= M_z. \end{aligned}$$

The solution of this system of equations give us directly $M_x = C$ and $M_y = M_z = F_x = F_y = F_z = 0$. When the mechanical chain is aligned with the knee instantaneous axis, only the motor torque is transmitted to the leg and there isn't any residual force or torque transmitted.

In the general case, the six components of the external force vector expressed in the frame R_2 are obtained from the following expressions,

$$\begin{aligned} M_y &= 0, \quad F_x = 0, \quad F_z = 0, \\ F_y &= -\frac{M_z}{Jv_{22}}, \quad M_z = M_x \frac{S(q_{35})}{C(q_{35})}, \\ M_x &= C \frac{Jv_{22} C(q_{35})}{Jv_{22} C(q_{235}) - Jv_{21} S(q_{35})}. \end{aligned}$$

where $q_{35} = q_3 + q_5$ and $q_{235} = q_2 + q_3 + q_5$. Jv_{22} and Jv_{21} are terms of the Jacobian $J(\mathbf{B})$ of the system written at the point B . These force and torque components can then be expressed in frame R_6 using the homogeneous matrix,

$$\begin{aligned} \begin{pmatrix} M_{x_6} \\ M_{y_6} \\ M_{z_6} \end{pmatrix}_{R_6} &= \mathbf{R}_{62} \begin{pmatrix} M_x \\ 0 \\ M_z \end{pmatrix} \\ &= \begin{pmatrix} C(q_{35})S(q_6)M_x + S(q_{35})S(q_6)M_z \\ C(q_{35})C(q_6)M_x + S(q_{35})C(q_6)M_z \\ -S(q_{35})M_x + C(q_{35})M_z \end{pmatrix} \\ \begin{pmatrix} F_{x_6} \\ F_{y_6} \\ F_{z_6} \end{pmatrix}_{R_6} &= \mathbf{R}_{62} \begin{pmatrix} 0 \\ F_y \\ 0 \end{pmatrix} = \begin{pmatrix} -F_y C(q_6) \\ F_y S(q_6) \\ 0 \end{pmatrix}. \end{aligned}$$

The term $-S(q_{35})M_x + C(q_{35})M_z = 0$, i.e. the component of the moment along the axis z_6 is equal to zero ($M_{z_6} = 0$). The component M_{x_6} is minimized when q_6 is near to zero, as for the component F_{y_6} . Thus, there are essentially two components (M_{y_6}, F_{x_6}) which are transmitted to the leg.

According to our simulation results, when the motor transmits a torque of 15 Nm, this torque is essentially transmitted to the leg along the knee axis of rotation. A residual force around 30 N is also transmitted along the axis x_6 perpendicular to the leg. This residual force may cause slipping movements of the attachments with the thigh when the knee is in full flexion. Thus it should be minimized by adjusting the value q_{35} so that this latter is close to zero. This is done by tuning the geometry of link (5) using the adjustments shown on Fig. 4.

A similar result is obtained for the analysis of force transmission on the thigh. In the ideal case where there is

no misalignment between the orthosis actuated axis and the knee instantaneous screw axis, two opposing torques of equal value are transmitted on the two limbs, thereby mobilizing the knee joint in flexion or in extension.

V. EXPERIMENTAL RESULTS



Fig. 7. View of the prototype.

Tests were carried out to validate the usability and performances of the device. We first implemented a robotized enhancement of the static progressive stretch for the treatment of knee stiffness. Such exercise was originally designed to restore mobility of joints previously immobilized for a long postoperative period. In this experiment, instead of using a passive articulated splint [1], the device is used to pull the subject knee from an angular position to a new one. The subject was seated during the experiment.

Since there is no significant constraint on accuracy, a simple proportional controller was used to control the closed loop system. The actuation unit was controlled in torque to follow a reference trajectory θ_{ref} using a proportional controller,

$$\tau_p = -K(\theta_{measured} - \theta_{ref}). \quad (5)$$

The reference trajectory is chosen to progressively stretch the knee joint. It can be e.g. a square or triangular q_1 signal with growing amplitude. The total amount of torque applied on the system is equal to the sum of this virtual spring torque plus the user's torque minus the resistance of the device measured by the force/torque sensor, hence the following control scheme:

To make the interaction 'orthosis-leg' more natural, the system was controlled in practice by speed and not by torque. The virtual spring torque was set at 0.5 Nm/deg. The results of this experiments are shown in Figs. 9 and 10.

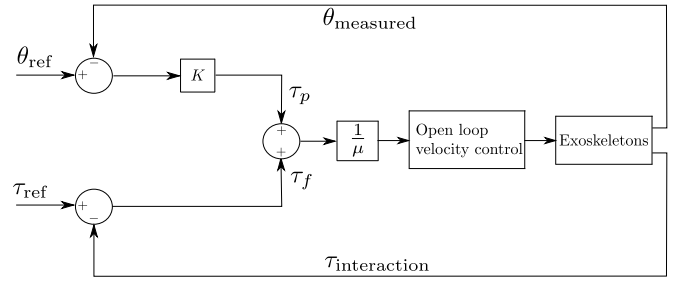


Fig. 8. Scheme of the control of the muscular stretching application.

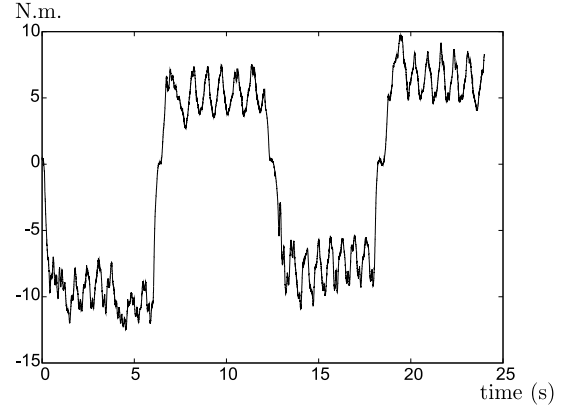


Fig. 9. The resistance torque of the knee during stretching.

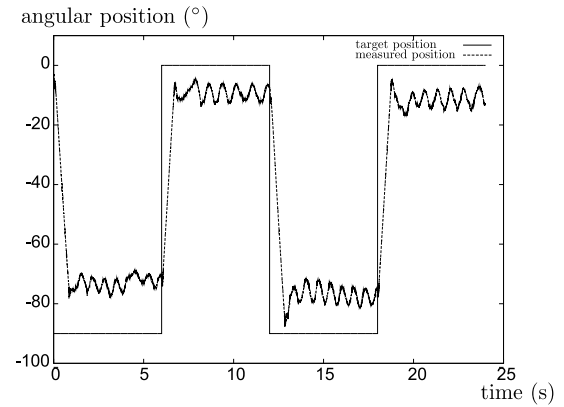


Fig. 10. The angular position and velocity of the actuated rotational axis of the orthosis during stretching.

The second test aimed at assisting the knee flexion and/or extension while the subject walked in a crouched or with a stiff knee gait. Such gaits frequently affect cerebral palsy children. Assisting the knee moment along quantified gait analysis may help to distinguish the part of the lack of muscle strength (or command) from the other possible causes of the gait deviation.

A torque control law was firstly implemented to allow an unconstrained knee motion to occur during the whole gait cycle (see Fig. 11). The interaction torque was minimized by setting the torque reference to zero (Fig. 12 and 13).

To assist the knee flexion during the swing phase, the

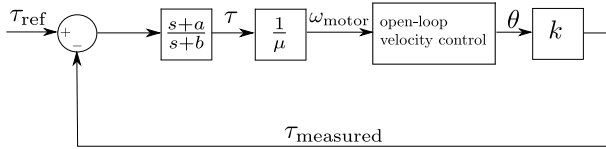


Fig. 11. Torque control scheme.

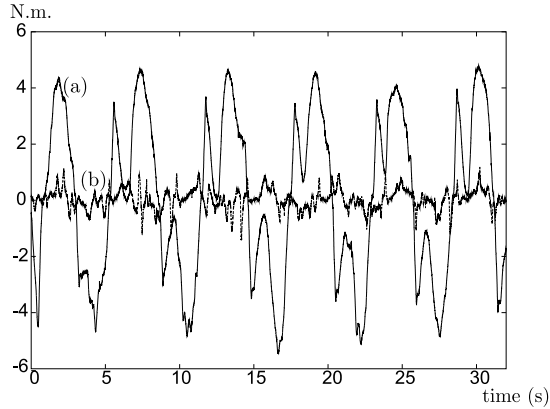


Fig. 12. Interaction torque between the mechanism and the leg during flexion-extension movement of the subject. (a) Result without force control. (b) Result with force control, the reference torque is set at zero.

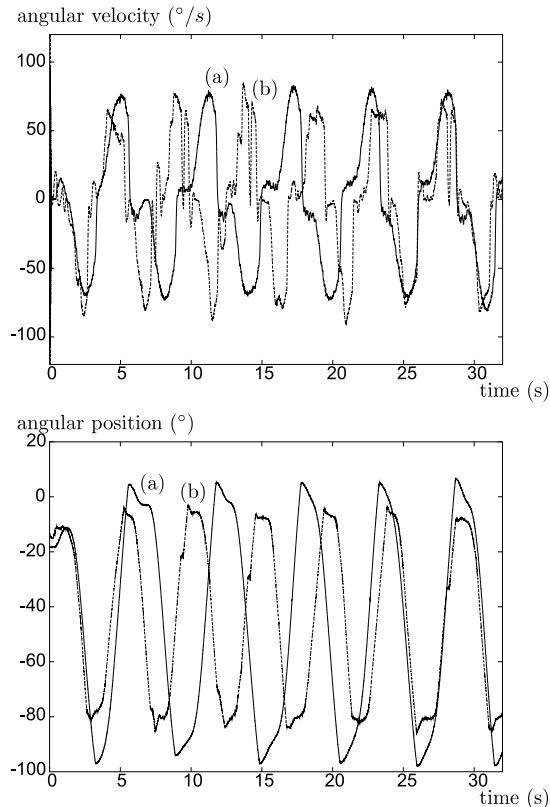


Fig. 13. The velocity and angular position of the actuated rotational axis of the orthosis during the zero force interaction experience. (a) Result without force control. (b) Result with force control, the reference torque is set at zero.

algorithm detected the period during which the flexion movement occurs based on the estimation of the instantaneous

angular velocity of the actuated joint. Flexion torque could therefore be improved by using the torque control law with a torque reference greater than zero (see Table II). A triangular signal was used as the reference torque so that the transition between the empowering phase and the zero force movement tracking phase occurred as smoothly as possible.

TABLE II
ALGORITHM FOR DETECTION OF THE FLEXION MOVEMENT AND
EMPOWERING STRATEGY.

```

Dispflex = 0; ωlast = ω
Infinite Loop
// Flexion estimation
Determine ω
if (ωlast > 0) and (ω < 0) then
    Dispflex = 0
endif
if (ωlast < 0) and (ω < 0) then
    Dispflex = Dispflex + ωT
endif
// Flexion movement detection and empowering of the movement
if (Dispflex > thresholdflexion)
and (ω < -ε) and (θ > -θmax-flexion) then
    τref = τflex // reference torque greater than zero
else
    τref = 0 // movement tracking
endif

```

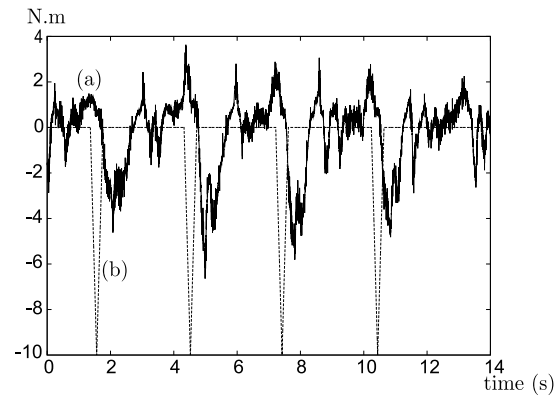


Fig. 14. Interaction torque between the mechanism and the leg during flexion-extension movement of the subject. (a) Measured torque (b) Target torque.

The results of these experiments are seen in Fig. 14 and 15. Even if the subject feels assisted by the torque provided by the orthosis during flexion of the latter, the interaction between the subject and the device is not sufficient during locomotion. We can observe in Fig. 15 that the rotational velocity of the orthosis is saturated. This in turn affects the angle of rotation which is very different from a natural one in our case. This problem could be solved by lowering the reduction ratio of the powering unit. This would decrease the maximum torque of the device but would increase its maximum rotational velocity. Moreover, a more natural reference torque has to be identified from in-vivo experimental data.

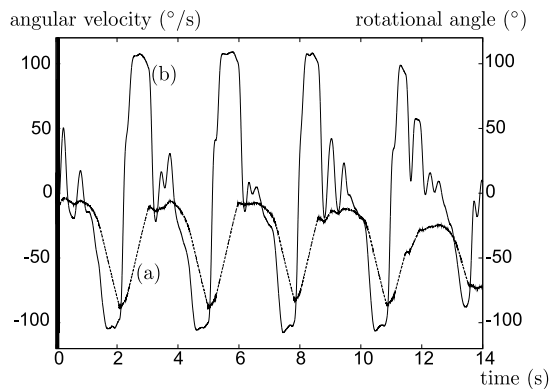


Fig. 15. The angular position (a) and angular velocity (b) of the actuated rotational axis of the orthosis during the flexion empowering experience.

VI. CONCLUSION

In this paper, we presented an isostatic exoskeleton designed for the knee joint. An active prototype was built to validate this approach. The device has six degrees of freedom, one of which is actuated. It drives the knee flexion-extension by transmitting two opposed torques to the two limbs of the user. The first experiments provide encouraging results, considering both transparent mode during which a zero interaction torque is controlled and assistance mode during which the device is used to activate the knee as a function of the subject's resistant torque.

Efforts should be made to improve the interaction between the device and the subject during very fast movements for a possible application of assistance to flexion/extension during walking. The maximum rotational velocity in particular should be improved.

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