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Analysis of jet instability in flute-like instruments by means of image processing : effect of the channel geometry on the jet instability.

Patricio de la Cuadra¹, Benoit Fabre², Nathalie Henrich², Thomas Robin²

CCRMA, Center for Computer Research in Music and Acoustics, Stanford University¹
LAM, Laboratoire d'acoustique musicale, Université Paris 6, 75252 Paris Cedex 05, France².

pdelac@stanford.edu¹, fabreb@ccr.jussieu.fr².

Abstract

We are studying experimentally the behavior of jets in flute-like instruments. At the time being most models for jet oscillations are based on linear jet instability analysis described by Rayleigh. But there is no clear model for the initial perturbations induced by transverse acoustic velocity. Aiming to understand and describe the origin of the perturbation, we are currently investigating the jet motion under transverse acoustic perturbation generated by loudspeakers. The jet behavior is analyzed through flow visualization and image processing using a cross-correlation technique. The paper will present results for different flue channel geometries. The linear instability is investigated. Experimental results are compared to a theoretical linear model.

1. Introduction

Instruments from the flute family share a principle of operation that consists of an unstable jet issuing from either a channel or the lips of a player. The jet flows in the direction of a sharp edge that we call the labium. The acoustic field due to the presence of the resonator triggers the jet instability and therefore the jet oscillates at the frequency of the acoustic field. The interaction of the perturbed flow with the labium provides the necessary acoustic energy to sustain the acoustic oscillation in the pipe.

Flutes present an immense variety of geometries and playing techniques. Among them, those that present a fixed geometry have particularly interested researchers in the last decades, mostly because of experimental simplicity.

The work presented here was triggered by the differences observed in the flue exit of various instruments. Organ pipes may show a flue channel ended by a 90 angle geometry. Recorders show a 45 cut at the flue exit and recorder makers insist on not rounding the edges of the flue exit, claiming that this would kill the instrument. This may seem contradictory since lip-played instruments obviously show a rounded flue exit.

The present paper explores the influence of the geometry of the flue channel exit (flue exit) on the oscillating

behavior of a jet. Analysis are carried out using a non-intrusive technique based on image analysis of flow visualization obtained by the Schlieren method [1]. We concentrate in this paper on the estimation of the convection velocity of perturbations on the jet for different channel geometries.

2. Jet formation and oscillation

The analysis of the oscillating jet is normally split into three parts: perturbation of the jet by the acoustic field (receptivity), propagation of the perturbation along the jet (instability) and generation of acoustic energy by the perturbed jet (sources).

Most receptivity models (e.g. [2], [3]) are developed with the assumption of a potential flow where vorticity is conserved. Vorticity is generated in the boundary layers of the channel and injected into the flow at the flue exit. Perturbations are assumed to act locally at the point of flow separation, modulating the vorticity in the shear layers of the jet. None of the current models provide a description good enough to take into account the effect of the channel geometry (length and exit shape) on the jet perturbation.

The central speed of the jet U_b at the channel exit can be estimated using Bernoulli's equation:

$$U_b = \sqrt{\frac{2p_f}{\rho_0}} \quad (1)$$

Where p_f is the pressure in the cavity before the channel, and ρ_0 the air density.

Both the length of the channel and the flue exit geometry have an important influence in the velocity profile [4]. Studying the frequency of edge-tone as well as visualizing transient attacks [5], it has been observed that the convection velocity is higher on jets issuing from a short channel, corresponding to jets with sharper velocity profiles.

Due to the intrinsic instability of the jet, the oscillation is amplified while being convected downstream. And it travels at a velocity about one half of the jet speed [6]. In a linear model, we consider an amplification described by:

$$\psi = \varphi(y)e^{i(\omega t - \alpha x)} \quad (2)$$

Where φ is the complex amplitude of the stream function ψ , ω the angular pulsation, t time and α the wave number. Spatial analysis has been shown to provide a better fit to measurements, and can be written as a real ω , while looking for a complex wave number $\alpha = \alpha_r + i\alpha_i$. The velocity of perturbation corresponds to $c_p = \omega/\alpha_r$ while the spatial amplification is given by the coefficient α_i .

Measurements of oscillating jets with short channel and squared exit have already been reported by Nolle [7] using a hot-wire. One of the benefits from choosing a visualization technique, as in the present paper, is that it is non-intrusive.

3. Experimental setup

Four flue exits are studied, of which the three with long channel are shown in Figure 1 (a fourth with squared exit and short channel is also included).

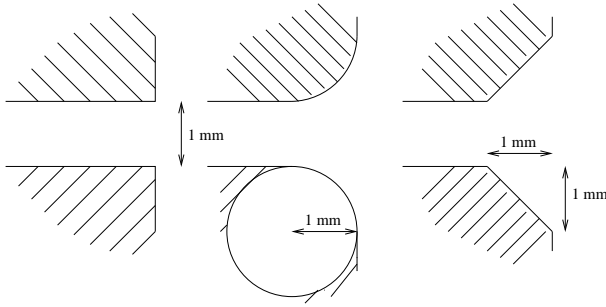


Figure 1: Geometry of the three flue exits used, from left to right: Squared, Rounded, 45 degrees.

Sequential images of the jet are taken with a digital camera [PCO,Sensicam, fast shutter] with exposure time set to one microsecond. Since the frequencies of jet oscillation are much higher than those provided by any modern camera, and because we want to be able to have an arbitrary number of images per cycle, a stroboscopic light is used. Its frequency is set such that multiples of its frequency are slightly out of phase with the excitation frequency providing an aliased representation of the oscillation. By changing this phase we can produce as many images per cycle as we want. Certainly they come from different cycles of the jet, but they can be collapsed to the same cycle since the oscillation is stationary.

Frequency of the camera was set to 14 fps while the acoustic excitation frequency was ranging from 70 to 1400 Hz to cover a *Strouhal* range from approximately 0.05 to 1.15 ($Strouhal = fh/U_b$, with h the channel height and U_b the jet speed at the exit deduced using eq.1). Approximately 100 images are taken covering two

cycles of jet oscillation. Images are captured in raw bmp files, with size 1280 x 448, and 8 intensity bits.

The speed of the jet is controlled with a manual valve measuring the pressure in the cavity p_f just before the jet formation. A *Reynolds* number based on the channel height h of 500 was chosen ($Reynolds = U_b h/\nu$, ν : kinematic viscosity) giving a jet velocity = 7.5 m/s which corresponds to normal blowing conditions for a recorder and assures a laminar behavior.

The amplitude of the acoustic excitation is measured through the acoustic velocity amplitude, which is created with two speakers in opposed phase, providing a more uniform field. A fixed amplitude was chosen for the whole experiment, and it was set to a value of 0.5% of the jet velocity. This is considerably lower than the 10% observed in real flutes but allow us to cover an interesting range of frequencies without over exciting the speakers.

The phase and amplitude of the acoustic excitation are measured using a velocity sensor (Microflown Technologies) placed as shown in Fig.2. Due to its non-flat frequency response, phase and amplitude are compensated.



Figure 2: Experimental setup. Jet exit, speakers and acoustic velocity sensor.

4. Image processing/data analysis

4.1. Cross-correlation method

Our goal is to determine the transverse displacement of the center of the jet as function of the distance from the flue exit for every image (Fig.3). One image is defined as a reference, normally the first of the sequence. Then, the position of all other images are calculated as distances from that reference, obtaining a relative position for the whole sequence. Finally the position of the reference is determined and subtracted from the relative position to obtain the actual position.



Figure 3: Jet issued from a squared exit, $Re: 500$, Jet speed: 7.5 m/s, Excitation freq.: 532 Hz, Str: 0.44 . Visualized with Schlieren technique: the two shear layers appear with opposite light refractive gradient, so that the upper shear layer gets darker while the lower shear layer gets lighter than the average gray level. The jet oscillating movement shows a very small transverse deviation at the beginning of the jet, slowly growing, up the point where the flow rolls up and finally breaks into vortices and later turbulence.

Cross-correlations are taken for the intensity values from the k^{th} column of reference image X_{ref}^k , with that of the k^{th} column of all other images from the sequence X_t^k .

$$Y^k(l, t) = \sum_{i=1}^N X_{ref}^k(i) * X_t^k(l + i),$$

$$l = -N + 1, \dots, 0, \dots, N - 1 \quad (3)$$

$$k = 1, \dots, P$$

$$t = 1, \dots, M$$

where, N is the number of pixels per column, P the number of columns per image, and M the number of images per sequence.

Then the position of the peak of $Y^k(l, t)$ is found, and the distance from the center calculated, giving the relative displacement. In order to estimate the jet deflection with a resolution better than 1 pixel, a parabolic interpolation is used, fitting $y(x) = a * (x - p)^2 + b$ to the three highest points, and obtaining p as the interpolated value.

This method uses all the variability of the image sequence as valuable information to find the position of the jet. Fig.4 illustrates the results obtained with it.

4.2. Processing data

All columns of the image oscillates in time at the frequency of the excitation. Therefore, for each column we fit a sinusoid at that frequency:

$$Y^k(f) = \frac{1}{N} \sum_{i=0}^{N-1} X_i^k e^{-\frac{j2\pi f i}{f_s}} \quad (4)$$

The amplitudes $|Y^k(f)|$ and phases $\angle Y^k(f)$ are shown in Fig 5. The linear model described by eq.2, predicts exponential growth described by α_i and a constant

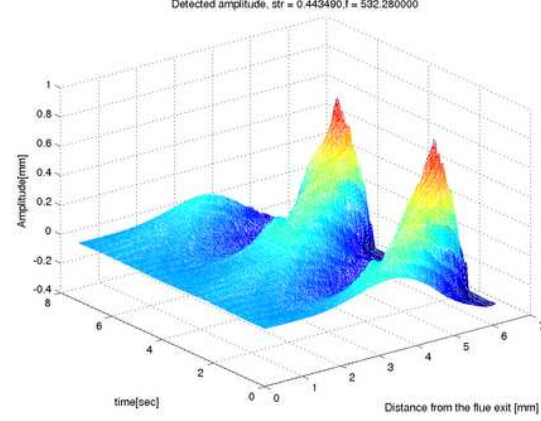


Figure 4: Position of jet found with cross-correlation method, excitation frequency = 532.28 Hz, flue exit = squared. Time axis corresponds to frame times in seconds.

velocity c_p . α_i can be obtained from the amplitude fit (Fig.5 up, thin curve), while c_p corresponds to the slope of the phase fit (Fig.5 down, thin line).

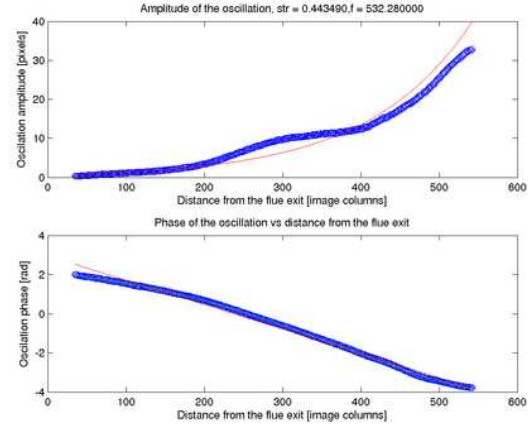


Figure 5: Up: (thick) Amplitude of the detected sinusoids, (thin) Exponential fit. Down: (thick) Phase of detected sinusoids, (thin) Linear fit. Excitation frequency = 532.28 Hz, flue exit = squared

5. Results and Discussion

The convection velocity of the perturbation c_p for the four cases and complete range of frequencies is presented in Fig. 6, as fraction of the center jet velocity at the flue exit U_b . In the case of the short channel with squared exit, the convection velocity is clearly faster than in the other cases, confirming the observations done by Ségoufin [5]

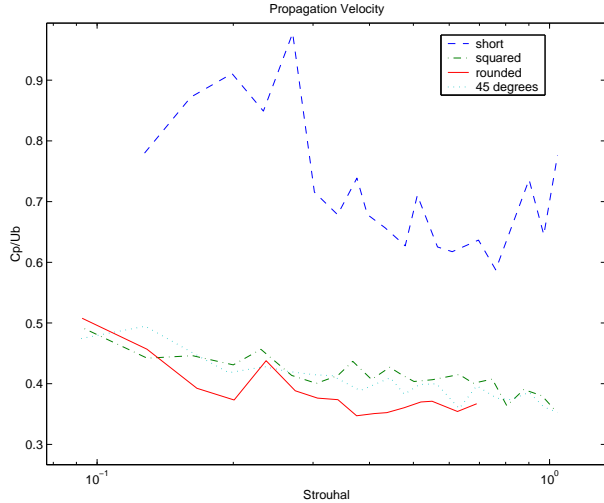


Figure 6: Dimensionless velocity, flue exit short (dashed), squared(dashdot), rounded (solid) and 45 degrees (dotted).

in the context of edge-tones. This can be interpreted, as suggested by Ségoufin, in terms of the boundary layers' thickness of the jet, which becomes thinner in the case of short channel. The magnitudes of the slowest three exits are in agreement both with the experimental measurements by Nolle [7] and theoretical values by Mattingly and Criminale [8].

For the three long channels, the convection velocity seems to be independent of the flue exit which confirms that it depends mostly on shear layer formation in the channel. A detailed analysis of the velocity field in the vicinity of the flue exit should be carried out in order to interpret these results further.

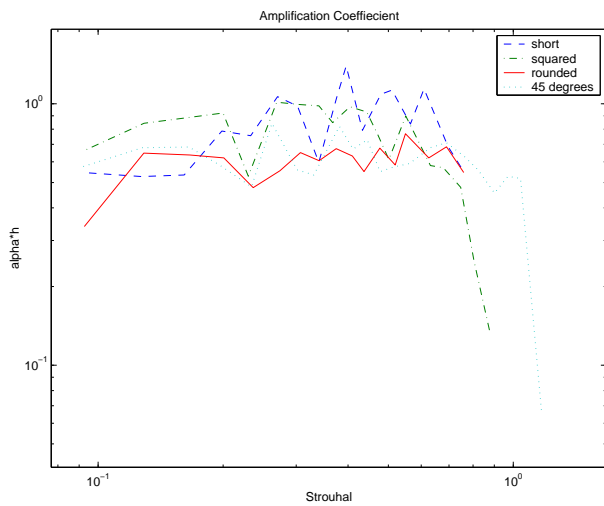


Figure 7: Dimensionless amplitude, flue exit short (dashed), squared(dashdot), rounded (solid) and 45 degrees (dotted).

The amplification coefficient α is presented dimensionless in Fig. 7. For each geometry exit, we observe an overall curved shape with a peak or preferred frequency that lies around $Str = 0.4$, which is in agreement with the theory [8]

For the moment we have no tools to determine the position of the transition from exponential growth into a discrete vortex model [9]. Only the part of the jet that oscillates linearly has to be included in our calculations. The estimation of amplification α is very sensitive to the position of that boundary, producing data with bigger dispersion than the one from the propagation speed c_p . Further work is needed in order to estimate the phase and amplitude of the excitation at the origin. An accurate model would need a detailed description of the total flow (from jet and from acoustics) at the flue exit.

Visualization of the oscillating jet together with the data analysis presented, provide a promising tool to explore some of the mysteries of the operation of flute-like instruments.

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