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Oscillation threshold of woodwind instruments

N. Grand*, J. Gilbert[†] and F. Laloë[‡]

Abstract

We give a theoretical study of the nature of the bifurcations occurring at the oscillation threshold of woodwind instruments, or of physical systems obeying similar non-linear equations of motion. We start from the simplest description of the acoustical behavior these instruments, a mathematical model containing two equations only, one of which is linear but includes delays, while the other is non-linear but has no delay, and discuss its predictions concerning the characteristics of the small oscillations. In particular we study the nature of the bifurcation occurring at threshold; if the bifurcation is direct, the amplitude of the oscillations increases progressively when the control parameter exceeds a threshold value; but, if the bifurcation is inverse, very small oscillations are not necessarily stable and the oscillation may jump discontinuously to a finite amplitude. While direct bifurcations correspond better to what naive intuition would expect, the surprising result of our calculations is their occurrence is by no means the general rule. We also discuss the shape (spectral content) of the small oscillations, and show that they do not always become quasisinusoidal in the limit of infinitely small solutions, in contrast with what is often assumed in the literature (Worman rule). Frequency shifts are investigated as well near threshold. More generally, we show how, despite of the simplicity of the equations of motion themselves, the characteristics of the non linearities of the excitor and of those of the resonator combine to produce a variety of possible behaviors which are not necessarily intuitive.

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1 Introduction

The basic understanding of the physical processes which allow permanent oscillations to be sustained in musical instruments is due to Helmholtz [1] and Rayleigh [2]. These authors described in general terms how energy could be introduced into passive resonators, in synchronization with their natural oscillations (resonances), and how in a permanent regime the acoustical losses could be exactly compensated by these feeding mechanisms. A good illustration of these ideas can be found for instance in the book of Benade [3], who uses simple analogies to explain the mechanism producing synchronization between a linear resonator and a non linear excitor (see in particular § 20.1). Further progress towards a more detailed theory was by Worman in his thesis [4], which contains a detailed set of equations that model the physical system. Worman also made a first attempt to solve the equations in the Fourier domain but, unfortunately, the calculation was not pushed sufficiently far to reach precise conclusions. Probably, at that time, the task was more difficult than now, since the general physical ideas on the generic properties of non linear dynamical systems and their bifurcations were not as spread as they are presently. Various authors have also attempted, for instance, to define threshold conditions for equations of this type, but curiously most of them have not gone beyond considerations on energy conservation (obviously a necessary condition, but not always sufficient) so that they do not draw general conclusions on the nature of the bifurcation. The natural control parameter in the problem is the slope A of the curve giving the acoustical flow of the excitor as a function of pressure. What is often implicitly assumed is that permanent oscillations may exist as soon as A reaches a threshold value, and that the amplitude of the oscillations grows progressively if A is further increased; at the same time, from an initially almost pure sinusoidal shape, the oscillation is supposed to acquire more and more harmonic content. This latter property is sometimes called the “Worman rule”, following Benade[3] (see in particular §23.1), while its conditions of validity of the rule are not specified precisely. Of course, these predictions seem physically very natural, and one expects them to be correct, at least in some cases. On the other hand, after all, they should be derived from the equations of motion since more complex scenarios could perfectly take place; indeed, outside of musical acoustics, we know that there are many physical systems which have a complex threshold behavior, for instance jumping discontinuously from rest to a finite oscillation level.

There are nevertheless a few cases where a detailed study of the nature of the oscillations (amplitude, harmonic content) is already given in the literature. Starting from the simplified equations for musical instrument oscillations derived by Mc Intyre et al. in [5], Maganza et al [6] study a special case where the res-

onator has simple properties (a case that we will call “completely degenerate” in section 4) and show that the mathematical solution is then simply given by an iteration, so that easy complete calculations become possible. Actually, not only the oscillation threshold is then known, but also the nature of the oscillations, which turn out to be perfect square wave functions (instead of quasi sinusoidal functions). Moreover, the theory predicts the occurrence of a whole period doubling scenario for the route to the chaos. We therefore have at least one case where, at the price of stringent simplifications concerning the properties of the resonator, a complete series of predictions is obtained, but it turns out that they do not coincide at all with the naive expectations mentioned above! A more general study is therefore necessary to reconcile the two points of view, in order for instance to determine more precisely their respective domain of validity.

The purpose of this article is to reexamine the question of the nature of the oscillations in a more general context, where all particular assumptions on the characteristics of the resonator are released (no degeneracy for instance). The general spirit is not the elaboration of a detailed model including all subtleties and details of a real instrument (dynamics of the reed, local turbulence, etc.). On the contrary we attempt to focus the interest on generic properties which do not depend on the details of the modeling. For this purpose, we use a simplified physical description of a musical instrument that is hoped to contain enough physics to provide reasonably realistic results, while remaining sufficiently concise to allow analytical calculations; the model is strongly inspired of that of [5], but it may also include various generalizations. The surprising result that we will find is that, in several cases, the nature of the oscillations is actually the opposite of what a naive energy argument would provide: the threshold bifurcation may often be inverse [7], which implies that the real threshold may be different from that derived from a linear calculation (a local analysis is not sufficient). In the text below, we show how the particular case studied in [6] falls into our general scheme. We also discuss how, if necessary, the dynamics of the reed as well as the displacement flow that it creates may be incorporated in it. A preliminary description of this work can be found in the thesis of one of the authors [8] as well as in [9]; see also the lectures given in Udine by J. Kergomard [10].

The organization of this article is the following: in section 2 we give the equations of the model that are used throughout this article, first in the time domain, then in the frequency domain where the non linearities couple together different frequencies; from this, in section 3, we study the properties of small oscillations (infinitesimal solutions of the system), the values of the parameters for which they exist, as well as the oscillation frequency shifts; then, in section 4, we treat a few interesting particular cases, especially resonators having two or more impedance peaks perfectly harmonic and with exactly the same height (degenerate resonators), which allows us to make contact with the previous work of [6].

2 The equations of the model

2.1 Notation

A basic reference on the modeling of the oscillations of musical instruments is reference [5], which has in fact inspired much subsequent work by various authors. Our “minimal model”, actually already used in [6], is taken from this reference and consists of a set of two equations only, corresponding to the scheme shown in figure 1. The first equation characterizes the acoustical properties of the resonator, as seen from its excitation end; the second equation characterizes the properties of the excitator, more precisely the “valve effects” [3] that occur at the location of the reed and the very end of the mouthpiece. The system of equations is:

$$\begin{cases} p(t) &= \int d\tau G(\tau) f(t - \tau) \\ f(t) &= F[p(t), P_s] \end{cases} \quad (1)$$

where $p(t)$ is the acoustical pressure inside the resonator under the reed (defined, as usual, as the excess pressure with respect to atmospheric pressure) and $f(t)$ is the acoustical flow at this location; here, the acoustical resonator itself is characterized by an impulse response function $G(\tau)$ but, equivalently, one may characterize it by its Fourier transform, which is the acoustical impedance $Z(\omega)$, and rewrite the first equation in the form:

$$\bar{p}(\omega) = Z(\omega)\bar{f}(\omega) \quad (2)$$

where $\bar{p}(\omega)$ and $\bar{f}(\omega)$ are now the Fourier transform $p(t)$ and $f(t)$. In (1), F is a non linear function which includes the effects of reed stiffness, dynamics of the air flow between the tip of the reed and the mouthpiece, etc. P_s is the pressure inside the source (the mouth cavity of the player) which provides energy to the system; we will often omit P_s from F , since it is a fixed parameter. For a general discussion of the form of this function, see for instance [3] or [11][12]. Fig. 2 shows a typical shape for the curve representing F ; in the absence of oscillation, the solution of the equations of motion is obtained by intersecting this curve with the straight line of equation $p = Z(0)f$; we call p_0 and F_0 the corresponding values of the variables, shown in the figure. Following Worman[4], we assume that F is sufficiently regular to be expanded in a Taylor series around the equilibrium point according to:

$$F(p) = F_0 + A(p - p_0) + B(p - p_0)^2 + C(p - p_0)^3 + \dots \quad (3)$$

but, otherwise, we make no assumption on the form of this function. We choose the letter A for the coefficient of the linear term in (3), because this coefficient has the same dimension as an acoustical admittance (inverse of an impedance). This parameter plays the role of a control parameter; if A vanishes, no acoustical flow is provided by the excitator for small oscillations, and therefore no energy; no

permanent oscillation can therefore maintain itself. The question is then: what will occur when A increases progressively? The first scenario which comes to mind, already mentioned, is that an infinitely small oscillation will appear at some critical value of A , and grows more and more in size (as well as in harmonic content) for larger and larger values of A ; nevertheless, we will see below that other scenarios are perfectly possible.

2.2 Stability of the trivial solution

Obviously, the system of equations (1) always has as a trivial solution $p = p_0$ and $f = F_0$, which trivially corresponds to no oscillation at all around the equilibrium point. The question is whether or not this solution is stable against small perturbations. What happens if, for instance, the second equation is replaced by:

$$f(t) = F[p(t), P_s] + \delta u(t) \quad (4)$$

where $\delta u(t)$ is some small perturbation to the acoustical flow (it might for instance arise from microtrubulence of air around the tip of the reed)? As long as the effect of an infinitesimal perturbation remains infinitely small, the trivial solution is stable; but, if this is not the case, the solution becomes unstable.

If the system (1) is linearized, which amounts to replacing $F(p) - F_0$ by $A(p - p_0)$, the scheme of figure (1-b) becomes the classical scheme of a linear feedback loop, so that Bode diagrams and the Nyquist theorem may be applied directly [13][14]. A first step is to plot on a polar diagram the variations, in the complex plane, of the complex impedance $Z(\omega)$ as a function of ω , as shown in figure 3; in a second step, it is sufficient to multiply $Z(\omega)$ by the slope A to obtain the Nyquist diagram associated with the linearized system. For an acoustical resonator resembling more or less a cylindrical (or conical) resonator, a first resonance is expected to occur, at a frequency $\omega = \bar{\omega}_1$; it corresponds to a real value $Z(\bar{\omega}_1)$ of the acoustical impedance. For a higher value $\omega = \bar{\omega}'_1$, a first anti-resonance of $Z(\omega)$ (a resonance of the admittance $A(\omega)$) will then occur and the impedance becomes real again, but with a much smaller value $Z = \bar{Z}'_1$. For a still higher value of the frequency, $\omega = \bar{\omega}_2$, a second resonance occurs and $Z(\bar{\omega}_2) = \bar{Z}_2$, and so on: a series of resonances separated by anti-resonances take place in succession.

The Nyquist theorem states in this case that the static equilibrium point is stable as long as the curve showing the product $AZ(\omega)$ does not circle around the point of coordinates 1 and 0 (on the real axis). Therefore, as long as $A\bar{Z}_{\max} < 1$, where \bar{Z}_{\max} is the largest among all values \bar{Z}_1, \bar{Z}_2 , etc. the static equilibrium point is indeed stable; it becomes unstable when A reaches the value $[\bar{Z}_{\max}]^{-1}$; when A continues to increase, it remains unstable until A reaches the value $[\bar{Z}_{\min}]^{-1}$ at which $AZ(\omega) > 1$ for all values of ω .

We remark in passing that the occurrence of stability for very small, as well as for very large, values of A is not unexpected, since there is a symmetry between small values and large values of this slope. This is because, in the equations, the pressure and the acoustical flow basically play a completely symmetrical role; indeed, the system is invariant under the transformation:

$$\begin{aligned} p(t) &\Leftrightarrow f(t) \\ Z(\omega) &\Leftrightarrow A(\omega) \\ F[p(t)] &\iff F^{-1}[f(t)] \end{aligned} \quad (5)$$

where F^{-1} is the inverse function of F ; this requires replacing the slope A by its inverse $1/A$ and implies, therefore, that very large and very small values of A play a similar role.

We now leave the trivial solution and study non-zero, but infinitesimal, solutions; the calculations will be easier in the Fourier domain of frequencies than in the time domain.

2.3 Fourier transform

The system written in (1) contains two equations; the first is linear but with a time delay, the second non-linear but without any delay. As soon as we are interested in a non trivial solution, we can no longer take a linear approximation and a more detailed calculation is required. We now study permanent oscillations of non zero amplitude and assume that the solution of the equations is periodic, with some frequency $\omega/2\pi$, which allows us to write the Fourier series of the pressure in the form:

$$p(t) - p_0 = \sum_n x_n e^{in\omega t} \quad (6)$$

Since $p(t)$ is real, we have for any value of n :

$$x_{-n} = (x_n)^* \quad (7)$$

(the star indicates complex conjugation). From (2), we can directly obtain the Fourier components of the acoustical flow as¹:

$$f_n = A_n x_n + \delta_{n,0} F_0 \quad (8)$$

where the A_n are defined for all integer (positive or negative) values of n by:

$$A_n = \frac{1}{Z(n\omega)} \quad (9)$$

The A_n 's are nothing but the acoustical admittances at an harmonic ‘‘comb of frequencies’’ that are contained in the periodic solution under study. Incidentally,

¹The term $\delta_{n,0} F_0$ arises because of the term $-p_0 = -F_0/A_0$ in the left hand side of (6).

for the moment, we do not fix the frequency of oscillation, but consider it as a free variable, to be determined later.

Since the first equation of system (1) is already included in (8), the only equation that we now take into account is the second. Calculating the Fourier transform of the square and the cube of a function, we obtain for every harmonic $n\omega$ a different equation; for instance, if $n = 0$, we get:

$$\begin{aligned} [A_0 - A - Bx_0 - C(x_0)^2] x_0 = & 2B [|x_1|^2 + |x_2|^2 + |x_3|^2 + \dots] + \\ & + 3Cx_0 [2|x_1|^2 + 2|x_2|^2 + 2|x_3|^2 \dots] + \\ & + 3C [(x_1)^2 x_2^* + 2x_1 x_2 x_3^* \dots + c.c.] \end{aligned} \quad (10)$$

(in the last line, *c.c.* is for complex conjugate). In the same way, the second equation is obtained for $n = 1$ and reads:

$$\begin{aligned} (A_1 - A')x_1 = & 2B [x_2 x_1^* + x_3 x_2^* + x_4 x_3^* + \dots] + \\ & + 3Cx_1 [|x_1|^2 + 2|x_2|^2 + 2|x_3|^2 + \dots] + \\ & + 3C [2x_0 x_1^* x_2 + 2x_0 x_2^* x_3 + (x_2)^2 x_3^* + \dots] \end{aligned} \quad (11)$$

where the coefficient A' is defined² as:

$$A' = A + 2Bx_0 + 3C(x_0)^2 \quad (12)$$

The equations for $n = 2$ is:

$$\begin{aligned} (A_2 - A')x_2 = & B'(x_1)^2 + 2B [x_3 x_1^* + x_4 x_2^* + \dots] + \\ & + 3Cx_2 [2|x_1|^2 + |x_2|^2 + 2|x_3|^2 + \dots] + \\ & + 3C [2x_0 x_1^* x_3 + 2x_1 x_2^* x_3 + (x_3)^2 x_4^* + \dots] \end{aligned} \quad (13)$$

where:

$$B' = B + 3Cx_0 \quad (14)$$

while the equations for $n = 3$ and $n = 4$ are:

$$\begin{aligned} (A_3 - A')x_3 = & 2B [x_1 x_2 + x_3 x_0 + x_4 x_1^* + x_5 x_2^* + \dots] + \\ & + Cx_1^3 + 3Cx_3 [2|x_1|^2 + 2|x_2|^2 + |x_3|^2 + 2|x_4|^2 \dots] + \\ & + 3C [(x_2)^2 x_1^* + 2x_0 x_1 x_2 + \dots] \end{aligned} \quad (15)$$

²This coefficient is nothing but the slope of the non linear function F around an new equilibrium point for the pressure, displaced by the amount x_0 with respect to the initial equilibrium (in the absence of oscillation). The distinction between A and A' applies only if acoustical impedance at zero frequency Z_0 does not vanish; if it does, A_0 becomes infinite and $x_0 = 0$, so that A' and A merely coincide.

and:

$$\begin{aligned}
(A_4 - A')x_4 = & B' (x_2)^2 + 2Bx_1x_3 + 2B [x_5x_1^* + x_6x_2^* + \dots] + \\
& + 3Cx_4 [2|x_1|^2 + 2|x_2|^2 + 2|x_3|^2 + |x_4|^2 + \dots] + \\
& + 3C [(x_1)^2 x_2 + (x_2)^2 x_0 + 2x_0x_1x_3 + 2x_1^*x_2x_3 + \dots]
\end{aligned} \tag{16}$$

Of course, the series of equations continues beyond $n = 4$: the non linear equation, initially in the time domain, is equivalent to an infinite system of equations in the frequency domain. Notice that, while A , A' , B and C are real coefficients (Taylor expansion of a real function), the A_n 's are in general complex. We have not written the equations corresponding to the negative frequencies, since they are merely the complex conjugate of those for positive frequencies, and therefore do not contain any additional information.

3 Small oscillations

When no oscillation is taking place, all x_n 's vanish and all equations are trivially satisfied. We now study the situation where an oscillation of very small amplitude is sustained; what we have to do, mathematically, is to determine the properties of the non trivial solutions of the infinite system of equations, assuming that they are infinitely small.

3.1 A first calculation

Among all coefficients x_n , some at least must have non zero values for a non trivial solution to be obtained. Let us assume that x_1 is not vanishing and consider this coefficient as a first order quantity; we then assume that all other Fourier coefficients are at least of the same order, and examine which order they actually are (we come back to this assumption in section 3.2.2). In equation (11), we notice that all terms on the right hand side are at least second order, if not more; on the left hand side, for the orders to match, it is therefore necessary that $(A_1 - A')$ should be infinitely small as well; this, in turn implies that $(A_1 - A)$, which differs from the preceding quantity only by higher order corrections, is also infinitely small. Consequently, if we choose the parameter A as a control parameter, the critical value is given by:

$$A = A_1 \tag{17}$$

This, in passing, shows that at threshold, A_1 is necessarily real, and provides a condition for the possible values of the frequency; we nevertheless postpone this discussion for the moment. If, in addition, we assume that A_1 is different from all the other A_n 's (non degeneracy of the resonator), it is easy to see that all other x_n 's are necessarily of higher order; for instance, the fact that $(A_0 - A)$ does not

vanish at threshold inside the left hand side of (10) shows that x_0 is second order in x_1 ; the fact that $(A_2 - A)$ does not vanish either shows that x_2 has the same property; in the same way, the inspection of (15) shows that x_3 is third order in x_1 , while x_4 fourth order, and so on.

Knowing that, we can solve the system of equations up to third order in x_1 . To second order in this quantity, (13) provides:

$$x_2 \simeq \frac{B(x_1)^2}{A_2 - A} \quad (18)$$

(to this order, the distinction between A and A' in the denominator, or B and B' in the numerator, is irrelevant) while, to the same order, (11) provides:

$$x_0 \simeq \frac{2B|x_1|^2}{A_0 - A} \quad (19)$$

We can now insert these results into (11) and obtain an equation in x_1 :

$$\left(A_1 - A - 2B \frac{2B|x_1|^2}{A_0 - A} \right) x_1 = 2B \frac{B(x_1)^2 x_1^*}{A_2 - A} + 3C|x_1|^2 x_1 + 0 [(x_1)^4] \quad (20)$$

This is an equation of degree three, which actually reduces to second degree after simplification by x_1 (this is permitted since we are looking for a solution of the system with a non vanishing value of the first Fourier coefficient) and reduces to:

$$|x_1|^2 \left\{ \frac{2B^2}{A_2 - A} + \frac{4B^2}{A_0 - A} + 3C \right\} = A_1 - A \quad (21)$$

Finally, the calculation provides the result:

$$|x_1|^2 = \frac{(A_0 - A)(A_1 - A)(A_2 - A)}{2B^2(A_0 + 2A_2 - 3A) + 3C(A_0 - A)(A_2 - A)} \quad (22)$$

For the sake of simplicity, in what follows, we may assume that the acoustical impedance of the resonator Z_0 at zero frequency is small³, and merely replace it by a zero value. This is of course an approximation, but this impedance is indeed small since the continuous impedance of a pipe of a diameter of the order of one centimeter (or more) and of a length of the order of a meter (or less) arises only from weak viscosity effects (Poiseuille flow); if necessary, this approximation could be released without any special difficulty, but just at the price of writing more cumbersome results. When $Z_0 = 0$, the admittance A_0 becomes infinite, and (22) reduces to:

$$|x_1|^2 = \frac{(A_1 - A)(A_2 - A)}{2B^2 + 3C(A_2 - A)} \quad (23)$$

³More precisely, and as the calculation will show, we assume here that Z_0 is negligible as compared to the (real) resonance values of the impedance which sustain the acoustical oscillations.

This result, combined with (18) and (19), gives the characteristics of the small oscillations of the physical system. Obviously, for this oscillation to exist, the right hand side of (23) must be real and positive.

3.2 Discussion

3.2.1 Nature of the bifurcation

Equation (23) allows us to make predictions on the nature of the bifurcation at threshold, i.e. on what phenomenon takes place when the control parameter A is increased beyond the critical value $A = A_1$. For the moment, in order to simplify the physical discussion as much as possible, we only discuss the case where A_2 is also real (we have already seen that, at the frequency of oscillation near threshold, A_1 has to be real), in other words the case where the two resonance frequencies are perfectly harmonic ; this condition is of course not necessarily fulfilled for any resonator and this motivates, in section 3.3, a more general treatment. With this simplification, it is simple to write the criterion for a “direct bifurcation” [7] to occur (a direct bifurcation takes place when small oscillations exist for values of the control parameter for which the trivial solution is unstable; if, on the opposite, the trivial solution is stable, the bifurcation is called “inverse”). If we assume that A_1 corresponds to the highest impedance peak, that is if the small oscillations that we are studying correspond to the lowest threshold⁴ (in terms of A values), we know from the study of section 2.2 that the trivial solution $|x_1| = 0$ is stable when $A < A_1$, unstable for higher values of A . Under these conditions, the positivity of the right hand side of (23) when $(A - A_1) > 0$ provides the inequality:

$$(A_2 - A_1) [2B^2 + 3C(A_2 - A_1)] < 0 \quad (24)$$

In the plane of the variables $(A_2 - A_1)$ and C , this corresponds to two regions limited by two boundaries, a hyperbola and a straight line (the horizontal axis), as shown in figure 4 where the letter D is used for “direct”, the letter I for “inverse”. The two other regions of the plane correspond to an “inverse bifurcation”, a case where the oscillation vanishes instead of appearing when A increases through the value $A = A_1$. Figure 4 is somewhat analogous to a phase diagram and, in what follows, we will sometimes refer to it in these terms. Condition (24) may be also written as a condition on coefficient C :

$$C < -\frac{2B^2}{3(A_2 - A_1)} \quad (25)$$

In figure 5, we represent schematically the two possible bifurcations in a different way by plotting the amplitude of the oscillation (characterized, for instance, by

⁴If this is not the case, oscillations at twice the frequency would be possible for a smaller value of the control parameter A .

$|x_1|$) as a function of the control parameter A . A direct bifurcation (region D in figure 4) corresponds to an amplitude of oscillations which grows progressively when the control parameter A increases from the threshold value $A = A_1$. In this case, the general properties of the Hopf bifurcation [7][15][16] imply that the small oscillations are stable. On the other hand, an inverse bifurcation (region I in figure 4) may correspond to a more complicated behavior: the small oscillations, obtained in this case only when the control parameter is smaller than the critical value, that is when the trivial solution is stable, are now unstable [15][16]; this does not mean that stable oscillations are never possible, but, if they exist, they necessarily occur for larger values of the oscillation amplitude, as symbolized by the dashed lines of figure 5 (their existence depends on higher order terms in the non linear function [15]). If this is the case, the system now has a threshold value $A = A_t$ which is actually smaller than A_1 ; at this value, the oscillation amplitude may jump discontinuously towards a finite level. Clearly, this kind of behavior can not be studied within the perturbative techniques that we have used in this article.

Naively, it would seem natural to expect direct bifurcations to occur in most cases, but figure 4 shows that this is far from being the rule. The interesting part of the figure is actually mostly the upper half, since the lowest part is useful only if we are studying oscillations which do not correspond to the “lowest threshold”, as noted above. When $A_2 - A_1$ is positive, figure 4 shows that, if $C = 0$ (the acoustical flow is a purely quadratic function of the pressure difference), the bifurcation is always inverse. Direct bifurcation can take place only when the cubic coefficient C has a negative value; the larger the quadratic coefficient B , the smaller the range of direct bifurcations. In other words we see that only some specific class of non linear excitators will give rise to a direct bifurcation, a somewhat unexpected result⁵.

At this point, some comparison between our preliminary results and the literature may be useful. An early reference on the subject is the article by Benade and Gans [17], who start from equations which are similar to ours, but do not derive precise mathematical conclusion on the existence, or non existence, of small oscillations. As already noted in the introduction, the work of Worman [4] is actually a precursor to the present study since it contains accurately all equations in the Fourier domain that are our starting point, but unfortunately without any precise study of the threshold (no inverse bifurcation). In more recent literature, one can nevertheless find equations which give qualitative predictions on oscillation threshold, and which are therefore comparable to (22) and (23). For instance equation (14.27) of the classical book by Fletcher and Rossing [18] has exactly the same structure but, curiously, does not include factors 2 and 3 in the de-

⁵In [6], a similar result was obtained by a completely different, graphical, method; for the highly degenerate resonator considered in this reference, the existence of small oscillations turns out to be also controlled by the sign of C (see also § 4.4). Our calculation here shows that the property can be generalized to non-degenerate resonators.

nominator. More importantly, the coefficients which appear in this equation are not the Taylor expansion coefficients of a non linear flow function, but defined in (14.20) as coefficients “which are not just numbers but rather operators involving phase shifts that are frequency dependent”, a comment which certainly does not apply to our coefficients. So, no direct comparison of the two analyses is possible; after studying it, the present authors admit that they do not fully understand this section of reference [18]. In an article on the functioning of brass instruments [19], Elliott and Bowsher obtain a “regeneration condition” which has, as (24), the form of an inequality, but since they do not give a general form to their non linear function, they just obtain one point in the phase diagram of figure 3 and, therefore, do not provide a complete discussion of its features. The adaptation of our calculations to a more specific (and therefore realistic) model of woodwind instruments has been made by Kergomard [10]; in this model, the control parameter is the source pressure P_s instead of the slope A , but all coefficients A , B and C are in fact a function of P_s . A general mathematical study of the nature of bifurcations in a system of equations similar to (1) can be found in [16]; this author excludes as special cases the “degenerate resonators” that we consider below, but treats multidimensional cases that we have not investigated.

To summarize the results of this preliminary study: assuming that the resonator is non-degenerate (non coincidence of the values of impedances at the various harmonics), we find results that agree with the predictions of the “Worman rule” [3]; see equation (18) for instance, which shows explicitly the property in question, while a generalization for higher harmonics is elementary:

$$\begin{aligned}
 x_3 &\simeq \frac{C(x_1)^3}{A_3 - A} \\
 x_4 &\simeq \frac{B(x_1)^4}{(A_4 - A)(A_2 - A)} \left[\frac{B^2}{A_2 - A} + C \frac{2A_2 + 3A_3 - 5A}{A_3 - A} \right] \quad . \quad (26) \\
 x_5 &\simeq \text{etc..}
 \end{aligned}$$

while $|x_1|$ itself is proportional to $\sqrt{|A - A_1|}$. But, to be valid, these results require very specific conditions from the excitor, related to the existence of a cubic correction to the quadratic non-linearity. Somewhat paradoxically, we find that an inverse bifurcation scenario, with a discontinuous jump in the amplitude of oscillation, is exactly as generic as (if not more than) that of a direct bifurcation. These predictions may be related to the results of Idogawa and coll. [20] who find experimentally that artificial blowing of a clarinet leads to sudden transitions between different regimes, but this is only a conjecture and we have not attempted to make a precise connection.

3.2.2 Other types of oscillations

At the beginning of section 3.1 we assumed that, among all x_n 's, the coefficient of lowest order is x_1 , all other coefficients being a least of the same order; from this assumption, we derived that they are actually higher order. But we have not yet examined what happens if the non-vanishing coefficient of lowest order is not x_1 , but x_2 for instance; we now study this possibility. From $x_2 e^{2i\omega t}$, the quadratic term in B generates frequencies 4ω and 0 ; the cubic term in C generates in addition 6ω , 2ω and -2ω , and these terms will act as "source terms" in the infinite system of equations (10), (11), (15), etc. As a consequence, we will now find as lowest order expansions:

$$x_0 \simeq \frac{2B |x_2|^2}{A_0 - A} \quad (27)$$

and:

$$x_4 \simeq \frac{B (x_2)^2}{A_4 - A} \quad (28)$$

etc. All these harmonics are even. To higher orders, they will in turn combine inside the non linear function to generate higher harmonics, but clearly all of them will still remain even: no "source term" will ever appear for x_1 , x_3 , etc. What we merely find is that the initial "comb of frequencies" is too tight, since one out of two of the frequencies that it contains is actually irrelevant. In other words, we can simply rename x_2 as x_1 , x_4 as x_2 , etc. and ignore the other frequencies, a process which reduces the new analysis to that of the preceding section. No really new phenomenon then occurs; we just have to redefine the value of the fundamental frequency.

Finally, the same simplification applies if, instead of x_2 , higher coefficients define the lowest order term. For instance if, instead of x_2 , we had chosen x_3 as a first order coefficient, the same kind of reasoning would show that only harmonics at multiples of three are present, and the same conclusion would be obtained again; redefining x_3 as x_1 reduces the problem to that treated above.

In conclusion of this section, we find that a scenario of "subharmonic buildup" can never take place for a resonator which has no degeneracy, so that the analysis of the preceding section is sufficient. This does not mean that, for degenerate resonators, the situation remains the same; for instance, if $A_2 = A_3$, a subharmonic buildup scenario where x_1 is progressively built from x_2 and x_3 is indeed possible, as discussed in § 4.

3.3 General case; frequency shifts

Here we treat a more general case, where we do no longer assume that A_1 and A_2 become simultaneously real for some value of the frequency $\omega/2\pi$. We call ω_0

a value of ω for which A_1 is indeed real, in other words a pole of the imaginary part $[A_1(\omega_0)]_I$ of the function $A_1(\omega)$:

$$[A_1(\omega = \omega_0)]_I = 0 \quad (29)$$

We note A_1^0 the (real) value of $A_1(\omega)$ at this frequency:

$$A_1(\omega_0) = [A_1(\omega_0)]_R = A_1^0 \quad (30)$$

We now assume that the frequency of oscillation remains close to ω_0 , and write the oscillation frequency as:

$$\omega = \omega_0 + \delta\omega \quad (31)$$

while the value of the control parameter A , which is close to the threshold value (A_1^0) obtained in the last section, is written:

$$A = A_1^0 + \delta A \quad (32)$$

In the same way, we may write the value of $A_1(\omega)$ in the form:

$$A_1(\omega) = A_1^0 + \delta A_1(\omega) \quad (33)$$

while if $\delta\omega \ll \omega_0$, we have⁶:

$$\delta A(\omega) = A_1^0 \left[2iQ \frac{\delta\omega}{\omega_0} \right] \quad (34)$$

where Q is the dimensionless quality factor of the impedance peak around which the oscillation takes place. As for $A_2(\omega)$, it may a priori have real and imaginary parts, so that we just write it as:

$$A_2(\omega) = [A_2^0]_R + i [A_2^0]_I + 0(\delta\omega) \quad (35)$$

(it turns out that, for our calculation, the correction in $0(\delta\omega)$ is not needed, so that we do not write it explicitly here). Rewriting equation (23) with these notations provides, when limiting the calculation to lowest order in $\delta\omega$ and δA :

$$|x_1|^2 = \frac{\left[2iA_1^0Q \frac{\delta\omega}{\omega_0} - \delta A \right] [A_{21}^0 + i\bar{A}_2^0]}{2B^2 + 3C [A_{21}^0 + i\bar{A}_2^0]} \quad (36)$$

where we have introduced the notation:

$$A_{21}^0 = [A_2^0]_R - A_1^0 \quad (37)$$

⁶More precisely, the validity of (34) requires that $\delta\omega \ll \omega_0/Q$.

and:

$$\overline{A}_2^0 = [A_2^0]_I \quad (38)$$

Now, writing that the imaginary part of the right hand side of (36) is zero provides the equality:

$$\frac{\delta\omega}{\omega_0} = \frac{\delta A}{2QA_1^0} \times \frac{2B^2\overline{A}_2^0}{2B^2A_{21}^0 + 3C \left[(A_{21}^0)^2 + (\overline{A}_2^0)^2 \right]} \quad (39)$$

which relates the frequency shift $\delta\omega$ to the excess value δA of the control parameter; if $C = 0$ (purely quadratic non linear function), this relation simplifies into:

$$\frac{\delta\omega}{\omega_0} = \frac{\overline{A}_2^0}{A_{21}^0} \times \frac{\delta A}{2QA_1^0} \quad (40)$$

We can also insert (39) into (36) in order to obtain the dependence of the square of the amplitude of oscillation as a function of δA ; after a little algebra we find:

$$|x_1|^2 = -K \left[2B^2A_{21}^0 + 3C \left[(A_{21}^0)^2 + (\overline{A}_2^0)^2 \right] \right] \delta A \quad (41)$$

where K is a positive numerical coefficient which plays no role in our discussion.

We have therefore obtained several results. First, equation (41) shows that the results of section 3.2 remain basically valid; the bifurcation is direct if:

$$2B^2A_{21}^0 + 3C \left[(A_{21}^0)^2 + (\overline{A}_2^0)^2 \right] < 0 \quad (42)$$

The only difference with (24) is that the coefficient of $3C$ is now increased by the term $(\overline{A}_2^0)^2$; as a consequence, the “phase diagram” of figure 4 is modified, and the border between the domains of direct and inverse bifurcations is no longer made of two separate lines, but of only one single cubic curve, as shown in figure 6. We also see in this figure that some inharmonicity of the acoustical resonator favors the range of direct bifurcations; but, as in section 3.2, in the case where the (real) impedance at ω is larger than that at 2ω , negative values of the cubic coefficient C are still required for getting a direct bifurcation. The second result is that, as soon as this inharmonicity occurs, a frequency shift appears, which increases linearly with the square of the amplitude of oscillation; the amplitude of the shift is inversely proportional to the quality factor Q of the impedance peak at ω , a natural result since a very narrow first resonance peak is likely to determine the frequency more accurately than a broad resonance; the sign of the shift depends of that of A_{21} and of C . The physical origin of the shift is nothing but a “frequency pulling” effect of the second resonance peak onto the first, in analogy with similar effects occurring in other oscillators in physics (lasers for instance); see [21], section 4.2 for an experimental study on single reed instruments.

3.4 Generalizations

In this section we show that the “minimal model”, which we have used as a starting point of all our calculations, may easily be generalized to include more physical effects than initially. In fact, one may directly apply all conclusions of the preceding sections to more complicated situations, just at the price of simple redefinitions of the parameters of the model. In this article, we only discuss one example: the dynamics of the reed and/or the air flow passing under it, but in ref. [8] more examples are treated explicitly. At the end of this section, in order to also give an idea of the limits of the model, we also discuss another generalization that can not be incorporated as easily into the equations, but requires more substantial rewriting of the algebra.

3.4.1 Including reed dynamics in the model

In the initial model, we have assumed that the acoustical air flow is a function, with no delay, of the pressure difference across the reed. This means that the reed itself follows instantaneously this pressure difference and that, in turn, the hydrodynamic flow of air under the reed also adapts in the same way. In other words, inertia effects are ignored as well in the reed and in the air flow that it determines. Of course, this is an approximation, and one could for instance try and improve the model by adding one more independent variable, the position $y(t)$ of the reed; $y(t)$ may for instance be defined as the distance between the tip of the reed and that of the mouthpiece, so that at any time it corresponds to the width of the channel that is available to the flow of air. The latter may then be taken equal to:

$$f(t) = F[y(t), P_s] \quad (43)$$

where P_s is the (constant) pressure inside the mouth of the player. We can now close the system of equations by relating the changes in time of the position $y(t)$ to the pressure $p(t)$; we do this by writing a differential equation:

$$\mu \frac{d^2}{dt^2} y(t) + \gamma \frac{d}{dt} y(t) + \xi y(t) = Sp(t) \quad (44)$$

which introduces new physics since the position $y(t)$ now follows the pressure variation with some phase shift; μ is the effective mass of the vibrating part of the reed, S its area, γ a coefficient characterizing its damping; ξ depends on the stiffness of the reed. The exact type of differential equation that we choose is irrelevant for the present discussion; here, in order to make things more concrete, we select the familiar case of a second order differential equation, but a differential equation of higher order would be treated exactly in the same way (provided it remains linear). Together with the second line of (1), or with (2), equations (43) and (44) form a system of three coupled equations, which a priori may have a more general behavior at threshold than (1). We will see that this is not the case.

To show this, it is sufficient to notice that, since the Fourier version (44) is merely:

$$\bar{p}(\omega) = C(\omega)\bar{y}(\omega) \quad (45)$$

where $\bar{y}(\omega)$ is the Fourier transform of $y(t)$, and where:

$$C(\omega) = S^{-1} [\mu\omega^2 + i\gamma\omega + \xi] \quad (46)$$

We may therefore eliminate p by condensing (2) and (44) into a single equation:

$$\bar{f}(\omega) = C(\omega)A(\omega)\bar{y}(\omega) \quad (47)$$

Finally, instead of taking three independent dynamical variables, p , f and y , we may eliminate the pressure and treat the problem with two variables only. In the new version of the elementary model, the flow f still plays the same role, but the role of the acoustical pressure is now played by the position of the reed $y(t)$; under these conditions, the equations which relate the variables are exactly of the same type, with the only change that, according to relation (47), that the acoustical admittance $A(\omega)$ should now be replaced by the product $C(\omega)A(\omega)$. With this substitution, all reasonings of the preceding sections immediately apply; the only change is summarized in the equation⁷:

$$A_n \Rightarrow A'_n = S^{-1} [\mu n^2 \omega^2 + i\gamma n \omega + \xi] A_n \quad (48)$$

The role of the cubic term in the non linear function, for instance, is therefore the same as discussed above. We nevertheless note that, since the relevant admittances are now defined in (48), an “effective inharmonicity” is introduced even if the acoustical resonator itself has perfectly harmonic resonances; even if all the A_n ’s become real for the same frequency $\omega/2\pi$, this is no longer true in general for the A'_n ’s, because reed inertia effects introduce phase shifts. Under these conditions, formula (39) may be used to evaluate frequency shifts arising from the frequency pulling effects arising from the reed. For instance, if the resonance frequency of the reed was chosen close to 2ω , it is clear that pulling effect of the second peak could be strongly reinforced, a direct consequence of (48); a generalization to a higher harmonic than $n = 2$ is straightforward.

In conclusion of this section, we see that our preceding results can easily be adapted in order to incorporate more general cases. In the next section, we give an opposite example, where a more substantial adaptation is needed.

⁷If, instead of a second order equation as in (44), we write a higher order equation, the correction between brackets in (48) becomes a polynomial of higher degree than two, but otherwise all conclusions remain the same.

3.4.2 Effects that require a new calculation

In order to show at least one example of a physical situation that does not reduce almost trivially to the “minimum model”, we now consider a generalization of equation (43) in the form:

$$f(t) = F [y(t), p(t) - P_s] \quad (49)$$

This is a natural generalization since the flow of air under the reed moves under the influence of the pressure difference $P_s - p(t)$ so that, if $p(t)$ is not much smaller than P_s , relation (49) is more appropriate than (43). If we now keep a relation such as (44) between $y(t)$ and $p(t)$, we see that the non linear function becomes a function of variables which include various phase shifts instead of keeping its instantaneous character. In this case, there is no simple redefinition procedure which brings us back to the minimum model, and we really have to calculate the Fourier series of a product of several different Fourier series to extend the analysis. Of course, there is no special difficulty in doing so, but the equations become more cumbersome. This is done in the appendix of ref [8] and, for brevity, we do not reproduce the results here. We just mention that the results are somewhat similar formally, while more delicate to interpret physically since more and more physical effects interfere at the same time.

4 Degenerate resonators and special cases

It is clear that the analysis of the equations that we have given so far is not general: we have always assumed that, when the control parameter A becomes equal to one admittance A_n for $n = n_0$, it remains different from all the other A_n 's. In other words, in the left hand side of the Fourier equations written in § 2.3, we have assumed that only one of the brackets $(A - A_n)$ can vanish at one time; consequently, among all threshold conditions for A , one corresponds to the lowest values and therefore determines the position of the threshold. We now study the case where two threshold conditions degenerate into one, that is where two impedances become real and equal for a particular frequency. We first assume that $A_2 = A_1$ and examine how the analysis should be modified in this case; we will examine later the case where $A_3 = A_1$, which is more realistic for a clarinet, as well as the case where $A_2 = A_3$, which is interesting because it allows subharmonic generation. Finally, we will go to the other extreme and assume that an infinite number of A_n 's are equal (complete degeneracy), which corresponds to the model used by Maganza and coll. [6].

4.1 Degeneracy 1-2

If⁸ $A_2 = A_1$, we expect that x_1 and x_2 will be of the same order, and we take them as a first order quantity. Equations (10) then provides, to lowest order:

$$x_0 \simeq 2B \frac{|x_1|^2 + |x_2|^2}{A_0 - A} \quad (50)$$

while relations (15) and (16) lead to:

$$x_3 \simeq \frac{2Bx_1x_2}{A_3 - A} \quad (51)$$

and:

$$x_4 \simeq \frac{B(x_2)^2}{A_4 - A} \quad (52)$$

All these Fourier coefficients are second order.

We now insert these results into (11) and (13), and obtain, when retaining terms up to third order:

$$\left(A_1 - A - 4B^2 \frac{|x_1|^2 + |x_2|^2}{A_0 - A} \right) x_1 = 2Bx_2x_1^* + \frac{4B^2}{A_3 - A} x_1 |x_2|^2 + 3C \left[|x_1|^2 + 2|x_2|^2 \right] x_1 \quad (53)$$

and:

$$\left(A_1 - A - 4B^2 \frac{|x_1|^2 + |x_2|^2}{A_0 - A} \right) x_2 = B(x_1)^2 + 2B^2 \left[\frac{2|x_1|^2}{A_3 - A} + \frac{|x_2|^2}{A_4 - A} \right] x_2 + 3C \left[2|x_1|^2 + |x_2|^2 \right] x_2 \quad (54)$$

We immediately see that these equations are markedly different from equation (20), where the right hand side was entirely of third order; here we have second order as well as third order. It makes thus sense to limit ourselves to second order and to simplify the non-linear system into:

$$\begin{aligned} (A_1 - A) x_1 &= 2Bx_2x_1^* \\ (A_1 - A) x_2 &= B(x_1)^2 \end{aligned} \quad (55)$$

Assuming that x_1 does not vanish (we are not interested in a trivial solution), we can rewrite the first equation of this system as:

$$\frac{2Bx_2}{(A_1 - A)} = \frac{x_1}{x_1^*} = e^{2i\varphi} \quad (56)$$

⁸ A_1 being real near threshold, A_2 is therefore also real. In other words, we are dealing here with two harmonic resonances which, moreover, have the same height for their resonance peaks.

where φ is the phase of x_1 . We therefore obtain:

$$x_2 = \frac{A_1 - A}{2B} e^{2i\varphi} \quad (57)$$

which, inserted in the second equation of the system, gives in turn:

$$x_1 = \pm \frac{A_1 - A}{B\sqrt{2}} e^{i\varphi} \quad (58)$$

(in these equations, the phase factor $e^{i\varphi}$ as well as the \pm can be cancelled by a change of the time origin).

Our results are significantly different from those of section 3.1. What is still true is that oscillations occur at frequencies close to a zero of the imaginary part of the admittance (or, equivalently, of the impedance); this is because A itself is real, and because (57) and (58) require that $A_1 - A$ be infinitely small for our calculations to be consistent. But, now, there is no condition on the sign of this difference; oscillations are possible whether it is positive or negative. In other words, when two threshold conditions $A = A_1$ and $A = A_2$ coalesce into one, they give rise to a completely different bifurcation scheme where the amplitude is now directly proportional to the excess control parameter and where small oscillations are possible independently of the sign (as before, it is natural to expect that the oscillations are stable only when $A > A_1$). What is the explanation for a behavior which is so different from that obtained in the absence of degeneracy? Mathematically, the crucial point is the presence of the two second order terms $2Bx_2x_1^*$ and $b(x_1)^2$ in the right hand side of the equations; in other words, the situation is different because the two waves at ω and 2ω can beat together and, directly, bring non-linear corrections to the other.

The main result of this section, therefore, is that degeneracy helps obtaining direct bifurcations⁹. In addition to good ‘‘collaboration’’ between the impedance peaks (good harmonicity), emphasized by Bouasse [22] and Benade [3], we see that it is also important that the values of the impedances themselves match properly. Can this fact be useful in practice for designing instruments, since direct bifurcations are presumably desirable musically for the instruments (to allow pianissimo playing)? Of course, in reality the mathematical degeneracy condition will never occur with infinite precision, so that no real resonator will ever satisfy $A_1 = A_2$ exactly. But this does not mean that our preceding discussion does not apply in practice, since all physical phenomena behave continuously as a function of $A_1 - A_2$. If A_1 and A_2 are only slightly different, there will be a very small range of the control parameter where the discussion of section 3.1 (no degeneracy) will apply; but, as soon as the amplitude of oscillations increases

⁹If, in figure 4, we assume that $A_2 = A_1$, the point representing the system falls on the horizontal axis, which is the border between direct and inverse regions; it is therefore not easy to decide which type the bifurcation is.

slightly, a crossover will occur and the system will behave as if A_1 and A_2 were equal, that is as discussed in this section. In other words, as soon as one has left a very small domain of inverse bifurcation, it will resemble a direct bifurcation.

4.2 Degeneracy 1-3

In the same spirit, we now study the effects of a degeneracy between the first and the third impedance peaks; we therefore assume that, for some frequency:

$$A_1 = A_3 \quad (59)$$

When x_1 and x_3 are taken as first order quantities, the Fourier equations of section 2.3 provide:

$$x_0 = 2B \frac{|x_1|^2 + |x_3|^2}{A_0 - A} \quad (60)$$

as well as:

$$x_2 = \frac{B(x_1)^2 + 2Bx_3x_1^*}{A_2 - A} \quad (61)$$

and:

$$x_4 = \frac{2Bx_1x_3}{A_4 - A} \quad (62)$$

If we insert these equations into (11) and (15), we get:

$$\left[A_1 - A - 4B^2 \frac{|x_1|^2 + |x_3|^2}{A_0 - A} \right] x_1 = 2B^2 \left[\frac{(|x_1|^2 + 2|x_3|^2)x_1 + 3x_3(x_1^*)^2}{A_2 - A} + \frac{2x_1|x_3|^2}{A_4 - A} \right] + 3C \left[|x_1|^2 + 2|x_3|^2 \right] x_1 \quad (63)$$

and:

$$\left[A_1 - A - 4B^2 \frac{|x_1|^2 + |x_3|^2}{A_0 - A} \right] x_3 = 2B^2 \left[\frac{(x_1)^2 + 2x_3x_1^*}{A_2 - A} x_1 + 2 \frac{|x_1|^2 + |x_3|^2}{A_0 - A} x_3 + \frac{2|x_1|^2 x_3}{A_4 - A} \right] + C(x_1)^3 + 3C \left[2|x_1|^2 + |x_3|^2 \right] x_3 \quad (64)$$

Here the beating notes between ω and 3ω do not allow to reproduce directly these initial frequencies; we therefore expect the situation to be more similar to that of a non degenerate resonator. To show this in the general case would probably require a numerical solution of equations (63) and (64). Here, in order to make a simple analytical calculation possible, we make additional assumptions: we assume that the impedances of the resonator at all frequencies but ω and

3ω are negligible, in other words that A_0 , A_2 as well as A_4 are infinite (alternatively, we could assume that the coefficient B has a negligible value). Under these conditions, the system reduces to:

$$\begin{aligned}(A_1 - A) x_1 &= 3C \left[|x_1|^2 + 2|x_3|^2 \right] x_1 \\ (A_1 - A) x_3 &= 3C \left[2|x_1|^2 + |x_3|^2 \right] x_3\end{aligned}\tag{65}$$

Both equations can then be simplified by eliminating a trivial (zero) solution and we obtain:

$$|x_1|^2 = |x_3|^2 = \frac{A_1 - A}{9C}\tag{66}$$

(curiously, at this stage of approximation, only $|x_1|$ and $|x_3|$ are fixed, not their phase). Indeed, what we now find is strongly reminiscent of the results of §3.2: the bifurcation is direct if the sign of the cubic constant C is negative.

The next step would be to assume finite values for A_0 , A_2 and A_4 , and treat the terms in B of (63) and (64) to first order; we do not give this calculation here.

4.3 Degeneracy 2-3; subharmonic generation

Another interesting situation is degeneracy between the second and the third peak:

$$A_2 = A_3 = \text{real number}\tag{67}$$

We then examine whether the 2-3 degeneracy allows subharmonic generation, while it was shown in § 3.2.2 to be impossible in the absence of degeneracy. We then obtain:

$$x_1 = 2B \frac{x_3 x_2^*}{A_1 - A}\tag{68}$$

and similar relations for x_0 and x_5 , which are also second order in x_2 and x_3 . We can then write the relations:

$$\left[A_2 - A - 4B^2 \frac{|x_2|^2 + |x_3|^2}{A_0 - A} \right] x_2 = 4B^2 \left[\frac{1}{A_1 - A} + \frac{1}{A_5 - A} \right] |x_3|^2 x_2 + 3C \left[|x_2|^2 + 2|x_3|^2 \right] x_2\tag{69}$$

as well as:

$$\left[A_2 - A - 4B^2 \frac{|x_2|^2 + |x_3|^2}{A_0 - A} \right] x_3 = 4B^2 \left[\frac{|x_2|^2 + |x_3|^2}{A_0 - A} + \frac{|x_2|^2}{A_1 - A} + \frac{|x_2|^2}{A_5 - A} \right] x_3 + 3C \left[2|x_2|^2 + |x_3|^2 \right] x_3\tag{70}$$

The situation is then similar to that of § 4.2. If we assume that $B = 0$, the solutions of the equations are:

$$|x_2|^2 = |x_3|^2 = \frac{A_2 - A}{9C} \quad (71)$$

and a similar situation occurs (direct bifurcation if C is negative). Inserting this result into (68) directly provides:

$$|x_1|^2 = \left[\frac{2B(A_2 - A)}{9C(A_1 - A)} \right]^2 \quad (72)$$

(here also, within the approximations that have been made, the phase of the Fourier components remain undetermined; a higher order calculation would be necessary to determine them). One could easily proceed in the same way and obtain higher order corrections in B . We therefore see that small oscillations can indeed happen if two peaks are degenerate, and beat non linearly to create a subharmonic frequency.

4.4 Complete degeneracy

After studying the effect of partial degeneracy of two peaks, we now come to the extreme case where an infinite number of peaks are degenerate. More precisely, we will assume that all resonances are perfectly harmonic, with equal values of the impedance separately for the even and for the odd resonances. This situation corresponds precisely to that studied by Maganza and coll. [6], since these authors assume that the impedance of the acoustical resonator is given by:

$$Z(\omega) = Z_\infty \frac{\varepsilon + i \tan \omega L/c}{1 + i \varepsilon \tan \omega L/c} \quad (73)$$

Here Z_∞ is the impedance of air in an infinite space without boundaries, L the length of the resonator and c the velocity of sound; a small number ε accounts for weak losses in the model, with no attempt of treating them in a realistic way (assuming $\varepsilon \neq 0$ is a simple way of avoiding divergencies in the formulas). When $\omega = \pi c/4L$, which is the resonance condition for this resonator, odd harmonics correspond to an infinite value of the tangent, while even harmonics correspond to a zero value, so that (73) provides:

$$\begin{cases} A_1 = A_3 = A_5 = \dots = \varepsilon/Z_\infty \\ A_0 = A_2 = A_4 = \dots = 1/\varepsilon Z_\infty \end{cases} \quad (74)$$

These equations imply an exact degeneracy of all the resonant peaks at $\omega, 3\omega, 5\omega$, etc. As recalled in the introduction, the interesting feature of this mathematical model is that it allows a complete and exact solution, not only for small oscillations, but also for finite oscillations; this includes domains where they undergo

period doubling and chaos. Here we examine how these degeneracy conditions affect the solution of the equations in the Fourier domain, (10), (11), (13), etc.

As an intermediate function, we introduce a periodic square function $G(t)$, with period $2\pi/\omega$, which has value -1 when $-(\pi/\omega) < t < 0$, and $+1$ when $0 < t < (\pi/\omega)$. Its Fourier coefficients are the y_n 's given by:

$$\begin{aligned} y_n &= 0 \quad \text{if } n \text{ is even} \\ y_n &= \frac{2i}{n\pi} \quad \text{if } n \text{ is odd} \end{aligned} \quad (75)$$

A property of $G(t)$ is that:

$$\begin{aligned} [G(t)]^2 &= 1 \\ [G(t)]^3 &= G(t) \end{aligned} \quad (76)$$

The first of these equations implies, by a calculation which is similar to that leading to (10), (11), (13), etc., the following relations between the coefficients y_n :

$$\begin{cases} 1 = 2 [|y_1|^2 + |y_3|^2 + |y_5|^2 + \dots] \\ 0 = 0 \\ 0 = (y_1)^2 + 2 [y_3 y_1^* + y_5 y_3^* + \dots] \\ 0 = 0 \\ \text{etc.} \end{cases} \quad (77)$$

while the second provides:

$$\begin{cases} 0 = 0 \\ y_1 = 3y_1 [|y_1|^2 + 2 |y_3|^2 + 2 |y_5|^2 + \dots] \\ 0 = 0 \\ y_3 = (y_1)^2 + 3y_3 [2 |y_1|^2 + |y_3|^2 + 2 |y_5|^2 + \dots] \\ \text{etc.} \end{cases} \quad (78)$$

Knowing this, we may assume that $p(t)$ varies according to:

$$p(t) = x_0 + a G(t) \quad (79)$$

where a is the amplitude of the square oscillation while x_0 is the time independent offset around which the oscillation is centered; both are unknown quantities at this stage. The x_n 's are then given by:

$$x_n = x_0 \delta_{n,0} + a y_n \quad (80)$$

But it turns out that, with this value of the x_n 's, all equations (10), (11), (13), etc. are satisfied term by term provided adequate values of x and x_0 are chosen. For instance, if we insert (80) into (10), using the first relation (77) and the fact that all x_n 's are zero for even values of n , we find the relation:

$$[A_0 - A - Bx_0 - C(x_0)^2] x_0 = Ba^2 + 3Ca^2 x_0 \quad (81)$$

In the same way, relation (11) combined with the second equation (78) and the zero value of all even coefficients, leads to:

$$\left[A_1 - A - 2Bx_0 - 3C(x_0)^2 \right] ay_1 = Ca^3y_1 \quad (82)$$

Now, if we go further, we can check that no additional relation between x_0 and a is obtained. For instance, inserting (80) into (13) provides, with the help (twice) of the third relation (77):

$$0 = 3Cx_0a^2 + 3C \left[-a^2 (y_1)^2 \right] \quad (83)$$

which is an identity. In the same way, (15) now becomes, when taking into account the fourth relation (78):

$$\left[A_1 - A - 2Bx_0 - 3C(x_0)^2 \right] ay_3 = Ca^3y_3 \quad (84)$$

which is merely equivalent to (82). To generalize the reasoning to any Fourier component, it is sufficient to note that, because of (76), relation (79) implies that:

$$\begin{aligned} p^2(t) &= (x_0)^2 + a^2 + 2ax_0 G(t) \\ p^3(t) &= (x_0)^3 + 3a^2x_0 + \left[3a(x_0)^2 + a^3 \right] G(t) \end{aligned} \quad (85)$$

so that, by grouping the constant terms and those in $G(t)$ in (3), we obtain:

$$f(t) = f_0 + bG(t) \quad (86)$$

with:

$$\begin{aligned} f_0 &= F_0 + Ax_0 + B \left[(x_0)^2 + a^2 \right] + C \left[(x_0)^3 + 3a^2x_0 \right] \\ b &= aA + 2aBx_0 + 3Ca(x_0)^2 + Ca^3 \end{aligned} \quad (87)$$

Because the function $G(t)$ has only odd Fourier components which, according to (23), all correspond to the same admittance of the resonator, we merely have:

$$\begin{aligned} f_0 &= A_0x_0 \\ b &= A_1a \end{aligned} \quad (88)$$

so that the equations of motion reduce to:

$$\begin{aligned} (A - A_0)x_0 + B \left[(x_0)^2 + a^2 \right] + C \left[(x_0)^3 + 3a^2x_0 \right] &= 0 \\ (A - A_1) + 2Bx_0 + 3C(x_0)^2 + Ca^2 &= 0 \end{aligned} \quad (89)$$

These results are indeed equivalent to (81) and (82); they provide necessary and sufficient conditions on the two coefficients x_0 and a for (79) to give a solution of the equations of motion. The major difference with all equations that we have

written in the preceding sections is that they are valid for oscillations of any amplitude - not only for infinitely small oscillations.

If nevertheless we assume the amplitude to be infinitesimal, the first equation of this system provides:

$$(A - A_0) x_0 + B a^2 = 0 \quad (90)$$

If we insert the corresponding value of x_0 into the second equation, we get:

$$(A - A_1) + \left[C - \frac{2B^2}{A_1 - A_0} \right] a^2 = 0 \quad (91)$$

which shows that criterion of direct bifurcation is:

$$C - \frac{2B^2}{A_1 - A_0} \simeq C + \frac{2B^2}{A_0} < 0 \quad (92)$$

where A_0 is given in (74). We notice that, for small oscillations, the amplitude a is proportional to $\sqrt{|A - A_1|}$ while the offset x_0 is, according to (90), proportional to the difference $(A - A_1)$ itself. Condition (92) corresponds to the prediction of Maganza et al.[6], obtained by a completely different method (graphical iteration); the condition for direct bifurcation depends in general on the dissipation¹⁰ in the resonator, contained in ε (through A_0); nevertheless, for weak dissipation, A_0 becomes very large, and the condition that we obtain is merely that the cubic coefficient C should be negative.

The study of finite oscillations is possible by a simple calculation. From the second equation of (89) we obtain:

$$a^2 = -\frac{1}{C} \left[A - A_1 + 2Bx_0 + 3C(x_0)^2 \right] \quad (93)$$

which we can insert into the first equation to obtain an equation with x_0 only:

$$8C^2(x_0)^3 + 9BC(x_0)^2 + \left[C(2A + A_0 - 3A_1) + 2B^2 \right] x_0 + B(A - A_1) = 0 \quad (94)$$

The corresponding curve is a cubic; but a root of equation (94) is acceptable as a solution only if it gives a positive value to the right hand side of (93). In figure 7 we therefore show on the same graph the cubic function as well as the parabola giving the value of:

$$-C^2 a^2 = 3C^2(x_0)^2 + 2BCx_0 + C(A - A_1) \quad (95)$$

Solutions of the equations are the intersection of the cubic with the horizontal axis which fall “inside” the parabola, while for $A = A_1$ we have the limit situation where this intersection point is exactly on it (figure 7-a). What happens when A

¹⁰In [6], formula (9) gives the ε dependence of the iterated function, so that the criterion of direct bifurcation is indeed ε dependent.

increases beyond this value? The intersection point moves to the left part of the axis and the parabola moves upwards, with the consequence that the acceptable domain for x_0 is reduced, so that two different cases are possible: either we have the situation of fig. 7-b where the intersection point is still in the “good” region, which means that the bifurcation is direct; or we have the situation of figure 7-c and the bifurcation is inverse. We have already seen that (92) is the criterion for the first situation to occur. In figure 8 we show how an acceptable value of x_0 may appear for a value of A_T of A which is inferior to A_1 , providing a threshold value corresponding to the appearance of a finite level of oscillation (first order transition, a case somewhat similar to that shown in figure 4).

An interesting feature of the fully degenerate case is that the iteration method of [6] also provides predictions on the stability of solutions. For instance, if the slope of the iterated function is less than one (condition $A < A_1$), the attractor of the iteration consists of just one fixed point; therefore, if (92) is not satisfied, the oscillations obtained for $A < A_1$ are unstable and tend to decay until static equilibrium is obtained. Moreover, when $A > A_1$, the same method shows that at some point, the square wave at frequency $\omega/2\pi$ becomes unstable against a period doubling phenomenon, while the solution at the initial frequency is still a mathematical solution, but unstable. This phenomenon is reminiscent of the subharmonic generation studied in section 4.3; see also [10].

5 Conclusion

The analysis that we have performed gives a whole set of predictions concerning the characteristics of small oscillations. A variety of results has been obtained concerning the nature of the bifurcations (direct or inverse), which depends on the characteristics of both the resonator and of the excitor in a way which is not always intuitive. Numerical simulations and real experiments are possible in order to check these results as well as the stability of the solutions. We have indeed done computer simulations, which are described in [8], as well as a few experiments; for lack of space, here we summarize the experiments in a few sentences only. The general idea was to obtain reproducible permanent oscillations of an acoustical resonator with a physical system which mimics the effects of the embouchure, the reed and the mouthpiece, while being more easily controllable and reproducible than real playing. For this purpose, a fast valve was made with a commercial “shaker” in order to control the input of air from a pressurized box into a cylindrical resonator, and a microphone was used to measure the acoustical pressure inside the resonator near the excitation point; the corresponding electrical signal was filtered, controlled by a non linear electronic device creating the equivalent of F , and amplified for driving the shaker. In this way a feedback loop was created, partly acoustical, partly electrical, and by changing the characteristics of the non linear feedback loop made it possible to check condition (25)

for direct bifurcation, as well as the frequency shift formula (39). In this way it was possible to check that the mathematical predictions of this article are indeed relevant to the behavior of real physical oscillators.

Conceptually, probably the major result of this work is the that, in addition to good harmonicity of the resonance peaks, which relates only to the values of the resonance frequencies, the value of the peak impedance itself is also crucial to the regime of weak oscillations. Changing these peak values may result in changing a direct bifurcation into an inverse one. Moreover, we have seen in section 4.2 that the equality between the height of the first and the second impedance peak may be useful in obtaining stable small oscillations. If the first resonance peak is low, we have also discussed how oscillations may be stabilized by the creation of subharmonics, a case which is related to the functioning of saxophones. We therefore believe that the preceding analysis brings a general frame for a better understanding of the mechanism by which permanent oscillations can occur in woodwind instruments.

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FIGURE CAPTIONS

Fig. 1: Figure (a) shows a schematic representation of a woodwind instrument, including a source of flow a constant pressure P_s , an excitor E which acts as a non linear “valve” (the mouthpiece and the reed for a reed instrument), and a linear resonator R (the body of the instrument). Figure (b) symbolizes the mathematical model used in this article: the excitor is characterized by a non linear function F which, applied to the difference $P_s - p$, provides the acoustical flow f entering the resonator, while p and f are linearly related through the acoustical impedance Z of the resonator (or its admittance).

Fig. 2: A typical non-linear function giving the entering air flow F crossing the valve of fig. 1 as a function of the acoustic pressure p at the output side of the valve (at the entrance of the resonator). In the absence of oscillation, the equilibrium values are p_0 and F_0 , obtained by intersecting the curve with a straight line of slope A_0 (the admittance of the resonator at zero frequency, sometimes assumed infinite in the text so that this static equilibrium point goes to the vertical axis). When $p = P_s$, the flow changes sign. The self sustained oscillations can only occur around function points where the slope A of the curve is positive, that is when energy is fed into the resonator by the valve. In the text, we discuss the threshold condition $A = A_1$, where A_1 is the (real) value of the admittance at some frequency.

Fig. 3: Polar diagram showing the variations of the complex impedance of the resonator $Z(\omega)$ as a function of ω . The first resonance occurs when $\omega = \bar{\omega}_1$ and corresponds to a real value $Z(\bar{\omega}_1) = \bar{Z}_1$ of the acoustical impedance; then, for $\omega = \bar{\omega}'_1$, a first anti-resonance of $Z(\omega)$ occurs (resonance of the admittance $A(\omega)$) and the impedance is again real, but with a much smaller value $Z = \bar{Z}'_1$; for a still higher of the frequency, $\omega = \bar{\omega}_2$, a second resonance occurs and $Z(\bar{\omega}_2) = \bar{Z}_2$, and so on. Multiplying this complex impedance by the slope A creates the Nyquist diagram associated with the linearized system of equations around static equilibrium; as long as $A\bar{Z}_1 < 1$, the curve obtained does not circle around the point $+1$ and this static equilibrium is stable; it becomes unstable beyond this value until A reaches the value at which $AZ(\omega) > 1$ for all values of ω , a second crossover point at which the equilibrium becomes stable again.

Fig. 4: “Phase diagram” showing the regions D where the bifurcation is direct as well as those regions I where it is inverse. The horizontal axis shows the values of the coefficient C of the cubic term in the non-linear function F , the vertical axis the difference $A_2 - A_1$ between the admittance at 2ω and that at ω . If the bifurcation really corresponds to the “first threshold”, that is to the smallest value of the control parameter A for which small oscillations are obtained, then necessarily $A_2 > A_1$, so that only the upper part of the diagram is relevant. This

figure is only valid if the second resonance peak is perfectly harmonic with the first; the general case is shown in figure 6.

Fig. 5: Diagram showing the amplitude $|x_1|$ of the first harmonic of the oscillation as a function of the control parameter A . The dashed lines show extrapolations to oscillations of finite amplitudes that are not calculated in this article. The trivial solution $|x_1|$ is stable until the value $A = A_1$ is reached. Two cases are possible:

(a) direct bifurcation, where small oscillations begins to occur while A increases and crosses the threshold value A_1 .

(b) inverse bifurcation, where small oscillations disappear in the same situation. In this case, they are expected to be unstable, and the stable oscillations to be of finite amplitude. Another threshold, $A = A_T$, occurs for a smaller value of A and corresponds to a first order transition (the system jumps there towards a finite level of oscillation, shown with dashed lines).

Fig 6: Phase diagram in the general case where the second resonance peak is not in perfect harmonic relation with the first; the horizontal axis shows the value of C , while the vertical axis gives the difference A_{21}^0 between the real values of the admittances at 2ω and ω . When the imaginary part \overline{A}_2^0 of the admittance at 2ω is non zero, the range of direct bifurcations is increased (dashed regions) while the border with the region of inverse bifurcation is now one single cubic curve; for comparison, the dashed lines show the same border when \overline{A}_2^0 vanishes. When $A_{21}^0 = |\overline{A}_2^0|$, the critical value of C has its minimal value $C_m = -B^2/3|\overline{A}_2^0|$.

Fig. 7: Graphical solution of equation (94), with the constraint that the right hand side of (93) must be positive; the intersections of the cubic curve with the horizontal axis give solutions which are valid provided that they fall “outside” of the parabola. When $A = A_1$, as in figure (a), both curves intersect the horizontal axis at the origin. When $A > A_1$, if the bifurcation is direct, one has the situation of figure (b), while if it is inverse one has the situation of figure (c).

Fig.8: In the case shown in figure (7-c), a threshold for a value $A = A_T < A_1$ may be obtained by the construction shown in this figure; it corresponds to a jump of the solution to a finite level of oscillation.

References

- [1] H. Helmholtz, “On the sensation of tone”, Dover, New York (1954).
- [2] J.W.S. Rayleigh, “The theory of sound”, Dover, New York (1945).
- [3] A. Benade, “Fundamental of Musical Acoustics”, Oxford University Press (1976).
- [4] W.E. Worman, “Self sustained oscillations in clarinet like systems”, thesis, Case Western University (1971).

- [5] M.E. McIntyre, R.T. Schumacher and J. Woodhouse, *J. Acoust. Soc. Am.* **56**, 653 (1983).
- [6] C. Maganza, R. Caussé and F. Laloë, *Europhys. Lett.* **1**, 295 (1986); C. Maganza, Thèse, Paris (1986).
- [7] P. Bergé, Y. Pomeau and C. Vidal, “L’ordre dans le chaos”, Hermann, Paris (1984).
- [8] N. Grand, “Etude du seuil d’oscillation des systèmes non-linéaires de type instrument à vent”, Thèse, Paris (1994).
- [9] N. Grand, J. Gilbert et F. Laloë, “Influences des non-linéarités sur les caractéristiques des oscillations des instruments à vent”, Congrès Français d’Acoustique, Proceedings, *J. Physique III*, **C5**, 585 (1994).
- [10] J. Kergomard, “Mechanics of musical instruments”, Lecture Notes of CISM (1994), Udine summer school, Springer Verlag, to be published.
- [11] C.J. Nederveen “Acoustical aspects of woodwind instruments”, Fritz Kufn, Amsterdam (1969).
- [12] A. Hirschberg, J. Gilbert, A.P.J. Wijnands and A.M.C. Valkering, “Musical aero-acoustics of the clarinet”, Proc. of the third “Congrès français d’acoustique”, *Journ. Physique Suppl. III*, **4 C5**, 559 (1994).
- [13] A.I. Mees, “Dynamics of feedback systems”, John Wiley (1981).
- [14] R.T. Stephani, C.J. Savant, B. Shahian, G.H. Hostetter, “Design of feedback control systems”, Saunders College Publishing, §6.5 (1994).
- [15] P. Manneville, “Structures dissipatives, chaos et turbulence”, Alea Saclay (1991); “Dissipative structures and weak turbulence”, Academic Press (1990); see in particular chap. 5.
- [16] D.J. Allwright, “Harmonic balance and the Hopf bifurcation”, *Math. Proc. Camb. Phil. Soc.* **82**, 453 (1977).
- [17] A.H. Benade and D.J. Gans, *Ann. N.Y. Acad. Sc.*, **155**, 154 (1968).
- [18] N.H. Fletcher and T.D. Rossing “The physics of musical instruments”, Springer Verlag (1990).
- [19] S.J. Elliott and J.M. Bowsher “Regeneration in brass instruments”, *J. Sound Vib.* **83**, 181 (1982).

- [20] T. Idogawa, T. Kobata, K. Komuro and M. Iwaki, “Non-linear vibrations in the air column of a clarinet artificially blown”, *J.Acoust.Soc.Am.* **93**, 540 (1993).
- [21] J.P. Dalmont, B. Gazengel, J. Gilbert and J. Kergomard, “Some aspects of tuning and clean intonation in reed instruments”, *Applied Acoustics*, **46**, 19 (1995).
- [22] H. Bouasse “Instruments à vent”, Delagrave, Paris (1929 and 1930); Blanchard, Paris (1986).